A THREE-DIMENSIONAL MATHEMATICAL MODEL

OF AN AUTOMOBILE PASSENGER

by

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Evaluation of the Roadway Environment by Dynamic Analysis of the Interaction Between the Vehicle, Passenger, and Roadway

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FOREWORD

The information contained herein was developed on Research Project 2-5-69-140 entitled "Evaluation of the Roadside Environment by Dynamic Analysis of the Interaction Between the Vehicle, Passenger, and Roadway" which is a cooperative research study sponsored jointly by The Texas Highway Department and the U. S. Department of Transportation, Federal Highway Administration.

Basically, the objectives of the study are to apply mathematical simulation techniques in determining the dynamic response of vehicles and their passengers when in collision with various roadside objects or when traversing curves in the road, shoulders, or other situations. It is a three-year study with a proposed completion date of August 1971. The first year's work was presented in Research Report 140-1 entitled "Documentation of Input for Single Vehicle Accident Computer Program," July 1969.

As part of the second year's task an analytical model was developed which predicts the dynamic response of an automobile's occupant in three-dimensional space. This report presents the derivation of the passenger model, a validation study, and a description of input data to the computer program used in determining the passenger's response.

The opinions, findings, and conclusions expressed in this publication are those of the author and not necessarily those of the Federal Highway Administration.

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ABSTRACT

Engineers are currently attempting to reduce the severity of vehicle accidents by designing and building a safer roadway environment. To accomplish this, consideration must be given to the expected dynamic response of the vehicle and occupant during a collision with roadside obstacles.

The reported research presents the development of an analytical model that predicts the response of an automobile passenger during vehicle motion of a general nature, i.e., a three-dimensional path including simultaneous rotations about the three directions. This model reduces the problem of predicting the accelerations and forces acting on a passenger during a collision or violent maneuver to that of specifying the path of the vehicle as a function of time plus the deformation properties of the vehicle interior.

The vehicle occupant is defined mathematically in three-dimensions as an independent system which is then placed inside the vehicle but not connected to it. The vehicle interior contains the passenger within its boundaries by applying contact forces while the vehicle moves through space. The vehicle interior is idealized with 25 planar surfaces and includes lap and torso restraint belts.

The geometry of the vehicle occupant is idealized by 12 rigid mass segments interconnected in a pattern which reflects the articulated nature of the human body. This system has 31 degrees of freedom which correspond to the set of generalized coordinates used in Lagrange's

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equations to derive equations of motion. These equations are solved numerically with the aid of the IBM 360/65 computer.

"Spinal elasticity" is simulated with rotational springs in the back joints of the articulated body, and "muscle tone" is simulated with rotational viscous dampers in every body joint.

Validation of the model has been achieved for frontal collisions. The model's predicted response of a dummy on a test cart compared well with actual test data. Both restrained and unrestrained conditions were analyzed. There were two restrained conditions: (1) lap belt only and (2) lap and torso belt.

The model provides the highway engineer with a tool which will enable him to apply biomechanics data on human tolerance limits to the problem of modifying roadside structures such that occupant injuries during vehicle accidents can be reduced.

SUMMARY

During the second year's effort of this three-year study, a mathematical model was developed which describes the motion, accelerations, and forces experienced by an automobile's passenger during a collision or any violent maneuver of a vehicle in three dimensions. A computer program was then written to solve the governing equations of the model.

Features of this program include: automatic seating of the passenger inside the vehicle; seating the passenger at either the left front, right front, left rear, or right rear positions; a lap restraint belt at any of these four positions; and/or a torso restraint belt at any of these four positions.

The passenger program requires input of the position of the vehicle in space as a function of time plus the force-deformation relations of the various surfaces of the vehicle interior. The path of the vehicle is obtainable from the CAL Single Vehicle Accident computer program or by full-scale testing.

Output from the program is presented in three forms: (1) digital printout, (2) X-Y plots of selected parameters, and (3) orthographic views of the passenger with respect to the vehicle interior for any given time during an event.

It has been concluded that this passenger model provides the engineer with a valuable tool with which to study vehicle and roadway problems culminating in the saving of lives and the reduction of occupant injuries.

IMPLEMENTATION STATEMENT

The reported research describes a tool which supplements existing technology as related to the prediction of occupant injury during a given vehicle maneuver and therefore exhibits favorable implementation potential. The passenger model, described herein, can be used in conjunction with the CAL Single Vehicle Accident computer program or fullscale test results and with biomechenics data on human tolerance limits to provide a systematic approach to the evaluation of roadways and vehicle interiors for safety. Some of its suggested applications include:

1. the evaluation of roadway geometry, i.e., sideslopes, ditches, terrain involving a variation of vertical and horizontal alignment, etc.; roadside safety features such as the breakaway sign, energy absorbing impact cushions, etc.; roadside protective barriers such as guardrails, bridge rails, median barriers, etc.;

2. the design of the vehicle interior and restraint systems;

3. the study of the dynamic behavior of a pedestrian when struck by an automobile; and

4. the study of collisions involving more than one vehicle.

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INTRODUCTION

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Background

Engineers are currently attempting to reduce the severity of single vehicle accidents by designing and building a safer roadway environment.

To design or evaluate effectively a roadway or its immediate environment for safety, consideration must be given to the dynamic response of the vehicle and occupant during interaction with geometric features such as curves, ditches, etc., or obstacles such as guardrails, bridge rails, median barriers, sign posts, etc. Accordingly, the design of highway safety devices such as breakaway signs, energy absorbing impact cushions, earth berms (an earth embankment geometrically designed to redirect safely a vehicle which has left the roadway), etc., depends directly on the dynamic response of vehicle and passenger during collision with these objects.

The above considerations are accurately summarized in the following quotation (1)*:

"Unless the motion time history of the vehicle can be translated into the expected kinematics of the vehicle

*Numerals in parentheses refer to corresponding items in "APPENDIX I.-REFERENCES."

occupant and further translated into the nature and extent of physical damage, it is not possible to establish performance requirements for roadside structure modifications that will effect a reduction in occupant injuries during single vehicle collisions."

The reported research was aimed at providing an analytical means of supplementing existing technology as related to roadside energy conversion systems. This was accomplished by developing a mathematical model to predict the response of an automobile passenger during violent vehicle motion of a general nature, i.e., a threedimensional path including simultaneous rotations about the three directions.

Design Considerations

Usual design practice is first to determine the time history and levels of acceleration (g-levels) experienced by the vehicle during a particular maneuver or collision. These are next compared to certain tolerance limits assuming that the occupant is subjected to the same g-level. This assumption is rigorously true only if the occupant is rigidly fastened to the vehicle. In actuality the passenger is unrestrained, lap belted, or shoulder harnessed and movement is not completely restricted, such that this assumption could range anywhere from overly conservative to dangerously inadequate depending on the situation.

Another factor which influences highway safety design problems is the quantitative consideration of contact forces between vehicle occupant and vehicle interior. It is possible for an automobile passenger to suffer fatal injuries from contact forces during a vehicle maneuver which at present may appear completely tolerable from the standpoint of vehicle accelerations alone.

It is felt that an analytical model of a passenger, used in conjunction with available biomechanics data on human tolerance limits, can be of significant value in approaching highway safety design problems.

Review of Literature

A survey of the literature has shown that the mathematical modeling of a vehicle occupant has been attempted in recent years. In most cases these efforts were aimed at developing restraint systems for the occupant, but in no instance was the occupant's general dynamic response the prime consideration.

In the early 1960's, a mathematical approach to the occupant restraint problem was made by the aerospace industry (5, 16). The primary concern was the behavior of viscera for fully restrained subjects.

During 1962-63, an analytical study of occupant restraint systems was performed by Cornell Aeronautical Laboratory (CAL) (10). A seven degree-of-freedom nonlinear mathematical model of a restrained, articulated body on a test cart, for the case of a frontal collision, was formulated and programmed for an electronic computer. This study also led to the development of an eleven degree-of-freedom

passenger model completed in 1966, and is the most sophisticated yet employed for the occupant restraint problem (12, 13, 14).

In 1967, Emori (2) conducted a study whose purpose was "to understand the mechanics of the automobile collision and to establish a logical background for the injury reduction of occupants." The scope of his research precluded the use of the CAL model and a single degree-of-freedom spring mass system for the occupant, and a similar representation of the automobile was used.

Renneker (17) used a two degree-of-freedom model of an occupant characterized by hip and torso restraint to study the effect of vehicle forestructure energy absorption on occupant injury.

Martinez and Garcia (8), in 1968, developed a mathematical model to represent the motion of the head and neck during rear-end collisions to study the "whiplash" phenomenon.

In 1969, Suggs, et al., (18), considered the problem of objectionable amplitudes and frequencies in the vibration of seats using a two degree-of-freedom representation of the human, for the purpose of developing more comfortable seats.

With the exception of the CAL model (12, 13, 14), the abovementioned efforts have little in common with the reported research but are acknowledged because they were mathematical simulations of the vehicle occupant.

The CAL model provided the major guidelines for performing this research since, in this writer's opinion, the results of that study reflect an adequate representation of the vehicle occupant for the two dimensional environment considered. However, the specific equations derived by CAL were not applicable to this study since this study involves a three-dimensional formulation, although the same basic geometrical configuration and concepts were applicable.

DERIVATION OF THE EQUATIONS OF MOTION

FOR THE VEHICLE OCCUPANT

Mathematical Formulation

Body Geometry and Coordinate Systems

The vehicle occupant is defined separately from the vehicle, or as an independent system of articulated rigid mass segments in threedimensional space. Consequently, the vehicle interior can be thought of as a confining environment for the occupant.

Interaction between occupant and vehicle interior is discussed in detail in a later section.

Figure 1 shows the center lines of the 12 rigid mass segments and their connection pattern, chosen as to geometrically describe the human body. A center line is defined as a straight line which connects the ends of a segment. It is assumed that the center of mass of a segment lies on its center line. For the body extremities, i.e., head-neck segments, forearm-hand segments, and leg-foot segments below the knees, the center line is defined as the straight line extending from the connecting body joint through the center of mass of the segment. Fixed at the center of mass, of each segment n, is a right-handed cartesian coordinate system denoted by axes X_n , Y_n , and Z_n . The positive directions of these axes are defined such that when the body in Figure 1 is standing upright with arms hanging vertically (downward) X_n will be positive straight ahead, Y_n will be positive to his left, and Z_n will be positive upwards. In addition, the



FIGURE 1.-ARTICULATED BODY WITH COORDINATE SYSTEMS

coordinate system of each segment X_n , Y_n , and Z_n will be parallel to the space-fixed coordinate system X', Y', Z'.

Euler Transformation and Generalized Coordinates

The relationship between segment-fixed coordinates (X_n, Y_n, Z_n) and space-fixed coordinates (X', Y', Z') is defined, in matrix form, as

 $\begin{cases} \mathbf{X}' \\ \mathbf{Y}' \\ \mathbf{Z}' \end{cases} = \begin{bmatrix} \mathbf{T}^{\mathbf{n}} \\ \mathbf{T}^{\mathbf{n}} \end{bmatrix} \begin{cases} \mathbf{X}_{\mathbf{n}} \\ \mathbf{Y}_{\mathbf{n}} \\ \mathbf{Z}_{\mathbf{n}} \end{cases}$

The matrix $[T^n]$ is a transformation matrix that transforms the segment-fixed coordinates back to the space-fixed coordinates, which means that if each of the 12 segments of the articulated body assumes a different orientation in space at any particular time, it is still possible to relate each of them to a common reference frame.

 $[T^n]$ could be derived in a number of ways, the most common of which utilizes direction cosines of the X_n , Y_n , Z_n axes relative to the X', Y', Z' axes. However, direction cosines would not be suitable for a Lagrangian formulation (to be discussed later) since they are not independent of each other and hence are not generalized (4). However, if the transformation from one cartesian coordinate system to another is done by means of three successive rotations performed in a specific sequence, the three angles are independent of each other and can be used as generalized coordinates. These three sequential rotations, commonly referred to as the "Eulerian angles", will be used to derive [Tⁿ].

The three sequential rotations to be used are given in Figure 2. Consider the cartesian coordinate systems X', Y', Z' and X_n , Y_n , Z_n to be initially parallel, Figure 2(a), then the rotations are as follows:

- 1. Rotate about the Z' axis an amount ϕ_n to first intermediate position X_{n1} , Y_{n1} , Z_{n1} , Figure 2(b).
- 2. Rotate about the Y_{n1} axis an amount θ_n to second intermediate position X_{n2} , Y_{n2} , Z_{n2} , Figure 2(c).
- 3. Rotate about the Z_{n2} axis an amount ψ_n to final position X_n , Y_n , Z_n , Figure 2(d).

The transformation from coordinates X_{n1} , Y_{n1} , Z_{n1} to coordinates X', Y', Z', Figure 2(b), is given by

$$\begin{cases} X' \\ Y' \\ Z' \end{cases} = \begin{bmatrix} \cos\phi_n & -\sin\phi_n & 0 \\ \sin\phi_n & \cos\phi_n & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{cases} X_{n1} \\ Y_{n1} \\ Z_{n1} \end{cases}$$
 (2)

The transformation from the coordinates X_{n2} , Y_{n2} , Z_{n2} to coordinates X_{n1} , Y_{n1} , Z_{n1} , Figure 2(c), is given by

$$\begin{cases} X_{n1} \\ Y_{n1} \\ Z_{n1} \end{cases} = \begin{bmatrix} \cos\theta_{n} & 0 & \sin\theta_{n} \\ 0 & 1 & 0 \\ -\sin\theta_{n} & 0 & \cos\theta_{n} \end{bmatrix} \begin{cases} X_{n2} \\ Y_{n2} \\ Z_{n2} \end{cases}$$
 (3)



FIGURE 2.-EULERIAN ANGLES

The transformation from coordinates X_n , Y_n , Z_n to coordinates

$$\begin{cases} X_{n2}, Y_{n2}, Z_{n2}, \text{ Figure 2(d), is given by} \\ \begin{cases} X_{n2} \\ Y_{n2} \\ Z_{n2} \end{cases} = \begin{bmatrix} \cos \psi_n & -\sin \psi_n & 0 \\ \sin \psi_n & \cos \psi_n & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{cases} X_n \\ Y_n \\ Z_n \end{cases}$$

Substituting Equation 4 into Equation 3 yields

$$\begin{cases} X_{n1} \\ Y_{n1} \\ Z_{n1} \end{cases} = \begin{bmatrix} \cos\theta_{n} & 0 & \sin\theta_{n} \\ 0 & 1 & 0 \\ -\sin\theta_{n} & 0 & \cos\theta_{n} \end{bmatrix} \begin{bmatrix} \cos\psi_{n} & -\sin\psi_{n} & 0 \\ \sin\psi_{n} & \cos\psi_{n} & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{cases} X_{n} \\ Y_{n} \\ Z_{n} \end{cases} .$$
 (5)

Substituting from Equation 5 into Equation 2 and performing the triple matrix product yield $[T^n]$ when the result is compared with Equation 1. The individual elements of matrix $[T^n]$ are

$$T_{11}^{n} = \cos\phi_{n} \cos\phi_{n} \cos\psi_{n} - \sin\phi_{n} \sin\psi_{n} \qquad \dots \qquad (6a)$$

$$T_{12}^{n} = -\cos\phi_{n}\cos\theta_{n}\sin\psi_{n} - \sin\phi_{n}\cos\psi_{n} \qquad \dots \qquad (6b)$$

$$T_{13}^{n} = \cos\phi_{n} \sin\theta_{n} \qquad (6c)$$

$$T_{21}^{n} = \sin\phi_{n} \cos\theta_{n} \cos\psi_{n} + \cos\phi_{n} \sin\psi_{n} \qquad \dots \qquad \dots \qquad (6d)$$

$$T_{22}^{n} = -\sin\phi_{n}\cos\theta_{n}\sin\psi_{n} + \cos\phi_{n}\cos\psi_{n} \qquad (6e)$$
$$T_{23}^{n} = \sin\phi_{n}\sin\theta_{n} \qquad (6f)$$

$$T_{31}^{n} = -\sin\theta_{n}\cos\psi_{n} \qquad (6g)$$

(4)

$$T_{32}^{n} = \sin\theta_{n} \sin\psi_{n} \qquad (6h)$$
$$T_{33}^{n} = \cos\theta_{n} \qquad (6i)$$

It can also be shown by using Figure 2 that

$$\begin{cases} X_n \\ Y_n \\ Z_n \end{cases} = \begin{bmatrix} T^n \\ T^n \end{bmatrix}^T \begin{cases} X' \\ Y' \\ Z' \end{cases}$$
 (7)

Furthermore, matrix multiplication shows that $[T^n][T^n]^T = [I]$, the identity matrix, such that Equation 1 is an orthogonal transformation.

At this point, the minimum number of coordinates needed to define the system is 39, namely, X'_{T1} , Y'_{T1} , and Z'_{T1} (Figure 1) plus three "Eulerian angles" for each of the 12 segments. However, realizing that the elbows and knees are really pinned joints rather than ball and socket joints, this number can be reduced.

Elbow and knee constraints. The pinned nature of the elbows and knees can be incorporated by constraining them to bend such that axes Y_7 , Y_8 , Y_{11} , and Y_{12} (Figure 1) are always parallel to axes Y_5 , Y_6 , Y_9 , and Y_{10} , respectively. In the case of an arm, e.g., the left arm, this means that the orientation of segment No. 7 with respect to the space-fixed coordinate system can be uniquely specified by stating the "Eulerian angles" of segment No. 5 plus one additional angle, namely, the angular position of segment No. 7 relative to segment No. 5. The same is true for the right arm and both legs. Let m denote the segment number of an extremity (m = 7, 8, 11, or 12) and α_m represent the position angle of segment No. m relative to segment No. j, where j = m-2.



FIGURE 3.-ELBOW OR KNEE CONSTRAINT

Figure 3 shows that

$$\begin{cases} X_{j} \\ Y_{j} \\ Z_{j} \end{cases} = \begin{bmatrix} \cos\alpha_{m} & 0 & \sin\alpha_{m} \\ 0 & 1 & 0 \\ -\sin\alpha_{m} & 0 & \cos\alpha_{m} \end{bmatrix} \quad \begin{cases} X_{m} \\ Y_{m} \\ Z_{m} \end{cases}$$
 (8)

Let j = n and substitute from Equation 8 into Equation 1, thus giving

$$\begin{cases} X' \\ Y' \\ Z' \end{cases} = \begin{bmatrix} T^{n} \\ -\sin\alpha_{m} & 0 & \sin\alpha_{m} \\ 0 & 1 & 0 \\ -\sin\alpha_{m} & 0 & \cos\alpha_{m} \end{bmatrix} \begin{cases} X_{m} \\ Y_{m} \\ Z_{m} \end{cases}$$
 (9)

Rewrite Equation 9 as

Therefore the matrix $[T^{nm}]$ where m = 7, 8, 11, or 12 and n = m-2 is the transformation matrix that transforms the coordinate system fixed in segment No. m back to the space-fixed system. From Equation 9 and Equations 6a through 6i the elements of matrix $[T^{nm}]$ are

$$T_{12}^{nm} = -\cos\phi_n \cos\theta_n \sin\psi_n - \sin\phi_n \cos\psi_n \qquad \dots \dots \dots \dots (11b)$$

$$T_{13}^{nm} = \cos\phi_n \cos\theta_n \cos\psi_n \sin\alpha_m - \sin\phi_n \sin\psi_n \sin\alpha_m + \cos\phi_n \sin\theta_n \cos\alpha_m \qquad (11c)$$

$$T_{22}^{nm} = -\sin\phi_n \cos\theta_n \sin\psi_n + \cos\phi_n \cos\psi_n \qquad \dots \qquad (11e)$$

$$T_{31}^{nm} = -\sin\theta_n \cos\psi_n \cos\alpha_m - \cos\theta_n \sin\alpha_m \qquad \dots \dots \dots \dots \dots (11g)$$

$$T_{33}^{nm} = -\sin\theta_n \cos\psi_n \sin\alpha_m + \cos\theta_n \cos\alpha_m \qquad \dots \qquad (11i)$$

From Figure 3 it can be shown that

$$\begin{cases} X_{m} \\ Y_{m} \\ Z_{m} \end{cases} = \begin{bmatrix} T^{nm} \end{bmatrix}^{T} \begin{cases} X' \\ Y' \\ Z' \end{cases}$$
 (12)

Furthermore, matrix multiplication shows that $[T^{nm}][T^{nm}]^T = [I]$, the identity matrix, such that Equation 10 is an orthogonal transformation.

Constraining the elbows and knees in the manner described has eliminated eight degrees of freedom, hence the minimum number of coordinates now needed to describe the system is 31. These generalized coordinates, denoted by q_1 , q_2 , ... q_{31} , are defined as follows: $q_1 = X'_{T1}$, $q_2 = Y'_{T1}$, $q_3 = Z'_{T1}$, $q_4 = \phi_1$, $q_5 = \theta_1$, $q_6 = \psi_1$, $q_7 = \phi_2$, $q_8 = \theta_2$, $q_9 = \psi_2$, $q_{10} = \phi_3$, $q_{11} = \theta_3$, $q_{12} = \psi_3$, $q_{13} = \phi_4$, $q_{14} = \theta_4$, $q_{15} = \psi_4$, $q_{16} = \phi_5$, $q_{17} = \theta_5$, $q_{18} = \psi_5$, $q_{19} = \phi_6$, $q_{20} = \theta_6$, $q_{21} = \psi_6$, $q_{22} = \alpha_7$, $q_{23} = \alpha_8$, $q_{24} = \phi_9$, $q_{25} = \theta_9$, $q_{26} = \psi_9$, $q_{27} = \phi_{10}$, $q_{28} = \theta_{10}$, $q_{29} = \psi_{10}$, $q_{30} = \alpha_{11}$, $q_{31} = \alpha_{12}$.

The Position of a Point on a Segment in Space

Using Equations 1 and 10, it is possible to express the position of any point on the articulated body with respect to the space-fixed coordinate system for any given set of values of the generalized coordinates.

To express kinetic energy of the articulated body in terms of the generalized coordinates, it is necessary to derive equations of position for the center of mass of each segment with respect to the space-fixed coordinate system. The matrix equation of position for the center of mass of segment No. 1, from Figure 1, is

$$\begin{cases} \mathbf{X}_{1}^{\mathbf{i}} \\ \mathbf{Y}_{1}^{\mathbf{i}} \\ \mathbf{Z}_{1}^{\mathbf{i}} \end{cases} = \begin{cases} \mathbf{X}_{T1}^{\mathbf{i}} \\ \mathbf{Y}_{T1}^{\mathbf{i}} \\ \mathbf{Z}_{T1}^{\mathbf{i}} \end{cases} + \begin{bmatrix} \mathbf{T}^{1} \\ \mathbf{T}^{1} \\ \mathbf{Z}_{T1}^{\mathbf{i}} \end{bmatrix} \begin{cases} \mathbf{0} \\ \mathbf{0} \\ -\bar{\rho}_{1} \end{bmatrix} \qquad (13)$$

in which (X'_1, Y'_1, Z'_1) represents the coordinates of the center of mass of segment No. 1 and $(X'_{T1}, Y'_{T1}, Z'_{T1})$ represents the coordinates of the reference point* on the articulated body both with respect to the space-fixed system, whereas $(0, 0, -\bar{\rho}_1)$ represents the coordinates of the center of mass of segment No. 1 with respect to segment-fixed coordinates which are parallel to axes X_1 , Y_1 , Z_1 but located at the terminal end of segment No. 1.

*Such a point is needed to account for translation of the articulated body as a unit (see Figure 1).



Likewise, the matrix equation for the center of mass of segment No. 8 is

Applying the same reasoning to all 12 segments and performing the matrix multiplication yields the following equations for the coordinates of the center of mass of each segment with respect to the space-fixed system:

$$X_{1}' = X_{T1}' - \rho_{1} T_{13}^{1} \qquad (15a)$$

$$Y_{1}' = Y_{T1} - \rho_{1} T_{23}^{1} \qquad (15b)$$

$$Z_{1}' = Z_{T1}' - \rho_{1} T_{33}^{1} \qquad (15b)$$

$$X_{2}' = X_{T1}' + \rho_{2} T_{13}^{2} \qquad (15c)$$

$$Y_{2}' = Y_{T1}' + \rho_{2} T_{23}^{2} \qquad (16b)$$

$Z'_{2} = Z'_{T1} + \rho_{2}T^{2}_{33}$	•	• • •	(16c)
$X'_{3} = X'_{T1} + L_2 T^2_{13} + \rho_3 T^3_{13} $	i •	• • •	(17a)
$Y'_3 = Y'_{T1} + L_2 T_{23}^2 + \rho_3 T_{23}^3 \dots \dots$, s , s , s, € , s	• • •	(17b)
$Z'_{3} = Z'_{T1} + L_2 T^2_{33} + \rho_3 T^3_{33}$	•	• • •	(17c)
$X'_4 = X'_{T1} - L_1 T^1_{13} - \overline{\rho}_4 T^4_{13} $	· ·	•••	(18a)
$Y'_4 = Y'_{T1} - L_1 T^1_{23} - \bar{\rho}_4 T^4_{23} $	•	• • •	(18b)
$Z'_4 = Z'_{T1} - L_1 T^1_{33} - \bar{\rho}_4 T^4_{33} $	•	• • •	(18c)
$x'_{5} = x'_{T1} + L_2 T_{13}^2 + L_s T_{12}^2 - \tilde{\rho}_5 T_{13}^5 \dots$	• •	• • •	(19a)
$Y'_{5} = Y'_{T1} + L_2 T_{23}^2 + L_8 T_{22}^2 - \bar{\rho}_5 T_{23}^5 \dots$	•	• • •	(19Ъ)
$Z'_{5} = Z'_{T1} + L_2 T^2_{33} + L_s T^2_{32} - \bar{\rho}_5 T^5_{33}$	•	• • •	(19c)
$x_6' = x_{T1}' + L_2 T_{13}^2 - L_s T_{12}^2 - \overline{\rho}_6 T_{13}^6 $	P°_,●	• • •	(20a)
$Y'_6 = Y'_{T1} + L_2 T_{23}^2 - L_s T_{22}^2 - \overline{\rho}_6 T_{23}^6 \dots$	• •	• • •	(20b)
$Z_{6}' = Z_{T1}' + L_2 T_{33}^2 - L_s T_{32}^2 - \overline{\rho}_6 T_{33}^6 $	• •	•••	(20c)
$X_7' = X_{T1}' + L_2 T_{13}^2 + L_s T_{12}^2 - L_5 T_{13}^5 - \bar{\rho}_7 T_{13}^{57}$	•	•••	(21a)
$Y_7' = Y_{T1}' + L_2 T_{23}^2 + L_s T_{22}^2 - L_5 T_{23}^5 - \bar{\rho}_7 T_{23}^{57}$		• • •	(21b)
$Z_7' = Z_{T1}' + L_2 T_{33}^2 + L_3 T_{32}^2 - L_5 T_{33}^5 - \bar{\rho}_7 T_{33}^{57}$	•	• • •	(21c)

$X_8' = X_{T1}' + L_2 T_{13}^2 - L_s T_{12}^2 - L_6 T_{13}^6 - \bar{\rho}_8 T_{13}^{68} \dots$	(22a)
$Y'_8 = Y'_{T1} + L_2 T_{23}^2 - L_s T_{22}^2 - L_6 T_{23}^6 - \bar{\rho}_8 T_{23}^{68} \dots$	(22b)
$Z_8' = Z_{T1}' + L_2 T_{33}^2 - L_s T_{32}^2 - L_6 T_{33}^6 - \bar{\rho}_8 T_{33}^{68} \cdots$	(22c)
$x_{9}' = x_{T1}' - L_{1}T_{13}^{1} - L_{4}T_{13}^{4} + L_{H}T_{12}^{4} - \bar{\rho}_{9}T_{13}^{9} \dots$	(23a)
$Y'_{9} = Y'_{T1} - L_{1}T^{1}_{23} - L_{4}T^{4}_{23} + L_{H}T^{4}_{22} - \bar{\rho}_{9}T^{9}_{23} $	(23b)
$Z_{9}' = Z_{T1}' - L_{1}T_{33}^{1} - L_{4}T_{33}^{4} + L_{H}T_{32}^{4} - \bar{\rho}_{9}T_{33}^{9} \dots$	(23c)
$X'_{10} = X'_{T1} - L_1 T^1_{13} - L_4 T^4_{13} - L_H T^4_{12} - \bar{\rho}_{10} T^{10}_{13} $	(24a)
$Y'_{10} = Y'_{T1} - L_1 T^1_{23} - L_4 T^4_{23} - L_H T^4_{22} - \bar{\rho}_{10} T^{10}_{23} $	(24b)
$Z_{10} = Z_{T1} - L_1 T_{33}^1 - L_4 T_{33}^4 - L_H T_{32}^4 - \bar{\rho}_{10} T_{33}^{10} \dots$	(24c)
$x_{11}' = x_{T1}' - L_1 T_{13}^1 - L_4 T_{13}^4 + L_H T_{12}^4 - L_9 T_{13}^9 - \bar{\rho}_{11} T_{13}^{911} .$	(25a)
$Y_{11} = Y_{T1} - L_1 T_{23}^1 - L_4 T_{23}^4 + L_H T_{22}^4 - L_9 T_{23}^9 - \bar{\rho}_{11} T_{23}^{911}$.	(25b)
$Z_{11}' = Z_{T1}' - L_1 T_{33}^1 - L_4 T_{33}^4 + L_H T_{32}^4 - L_9 T_{33}^9 - \bar{\rho}_{11} T_{33}^{911} .$	(25c)
$x_{12} = x_{T1} - L_1 T_{13}^1 - L_4 T_{13}^4 - L_H T_{12}^4 - L_{10} T_{13}^{10} - \bar{\rho}_{12} T_{13}^{1012}$	(26a)
$Y_{12} = Y_{T1} - L_1 T_{23}^1 - L_4 T_{23}^4 - L_H T_{22}^4 - L_{10} T_{23}^{10} - \bar{\rho}_{12} T_{23}^{1012}$	(26b)
$Z_{12} = Z_{T1} - L_1 T_{33}^1 - L_4 T_{33}^4 - L_H T_{32}^4 - L_{10} T_{33}^{10} - \bar{\rho}_{12} T_{33}^{1012}$	(26c)

<u>Velocity squared</u>. Taking the first derivative with respect to time* of X'_1 , Y'_1 , Z'_1 through X'_{12} , Y'_{12} , Z'_{12} (Equations 15a through 26c) and squaring the results yields the following equations for components of velocity squared of the center of mass of each segment with respect to the space-fixed system:

$$(\dot{x}_{1})^{2} = (\dot{x}_{T1})^{2} - 2\bar{\rho}_{1}\dot{x}_{T1}\dot{T}_{13}^{1} + \bar{\rho}_{1}^{2}(\dot{T}_{13})^{2}$$
 ... (27a)

$$(\dot{\mathbf{x}}_{1}')^{2} = (\dot{\mathbf{x}}_{T1}')^{2} - 2\bar{\rho}_{1}\dot{\mathbf{x}}_{T1}\dot{\mathbf{x}}_{23}^{1} + \bar{\rho}_{1}^{2}(\dot{\mathbf{x}}_{23}^{1})^{2} \qquad \dots \qquad (27b)$$

$$(\dot{z}_{1})^{2} = (\dot{z}_{11})^{2} - 2\bar{\rho}_{1}\dot{z}_{11}\dot{r}_{33}^{1} + \bar{\rho}_{1}^{2}(\dot{r}_{33}^{1})^{2} \qquad \dots \dots (27c)$$

$$(\dot{z}_{12})^{2} = (\dot{z}_{11})^{2} + (L_{1}\dot{T}_{33}^{1})^{2} + (L_{4}\dot{T}_{33}^{4})^{2} + (L_{H}\dot{T}_{32}^{4})^{2} + (L_{10}\dot{T}_{33}^{10})^{2} + (\bar{\rho}_{12}\dot{T}_{33}^{1012})^{2} - 2L_{1}\dot{z}_{11}\dot{T}_{33}^{1} - 2L_{4}\dot{z}_{11}\dot{T}_{33}^{4} - 2L_{H}\dot{z}_{11}\dot{T}_{32}^{4} - 2L_{10}\dot{z}_{11}\dot{T}_{33}^{10} - 2\bar{\rho}_{12}\dot{z}_{11}\dot{T}_{33}^{1012} + 2L_{1}L_{4}\dot{T}_{33}^{1}\dot{T}_{33}^{4} + 2L_{1}L_{H}\dot{T}_{33}^{1}\dot{T}_{32}^{4} + 2L_{1}L_{10}\dot{T}_{33}^{1}\dot{T}_{33}^{10} + 2L_{1}\dot{\rho}_{12}\dot{T}_{33}^{1}\dot{T}_{33}^{1012} + 2L_{4}L_{H}\dot{T}_{33}^{4}\dot{T}_{32}^{4} + 2L_{4}L_{10}\dot{T}_{33}^{4}\dot{T}_{33}^{10} + 2L_{4}\bar{\rho}_{12}\dot{T}_{33}^{1}\dot{T}_{33}^{1012} + 2L_{4}L_{H}\dot{T}_{33}^{4}\dot{T}_{33}^{10} + 2L_{4}\bar{\rho}_{12}\dot{T}_{32}^{4}\dot{T}_{33}^{1012} + 2L_{4}\bar{\rho}_{12}\dot{T}_{33}^{1}\dot{T}_{33}^{1012} + 2L_{H}L_{10}\dot{T}_{32}\dot{T}_{33}^{10} + 2L_{H}\bar{\rho}_{12}\dot{T}_{32}\dot{T}_{33}^{1012} + 2L_{10}\bar{\rho}_{12}\dot{T}_{33}^{10}\dot{T}_{33}^{1012}$$
 (38c)

*A dot above a variable name will be taken to mean the first derivative of that variable with respect to time.

Angular Velocities

To express the rotational portion of the articulated body's kinetic energy in terms of the generalized coordinates, it was necessary to relate the angular velocities of a segment to the generalized velocities of the system. This was accomplished by realizing that angular velocities are vectors which obey the transformation relationships that were derived for Figures 2 and 3 (Equations 1 through 12).

The generalized angular velocities $\dot{\phi}_n$, $\dot{\theta}_n$, $\dot{\psi}_n$ are vectors parallel to axes Z', Y_{n1} , and Z_{n2} , respectively, Figure 2. Let ω_{xn} , ω_{yn} , and ω_{zn} be angular velocities about segment coordinate axes X_n , Y_n , and Z_n , respectively.* Similarly, define $\dot{\phi}_{xn}$, $\dot{\phi}_{yn}$, and $\dot{\phi}_{zn}$ as components of $\dot{\phi}_n$; $\dot{\theta}_{xn}$, $\dot{\theta}_{yn}$, and $\dot{\theta}_{zn}$ as components of $\dot{\theta}_n$; and $\dot{\psi}_{xn}$, $\dot{\psi}_{yn}$, and $\dot{\psi}_{zn}$ as components of $\dot{\psi}_n$ where all components are parallel to axes X_n , Y_n , and Z_n , respectively, such that

$$\begin{cases} \omega_{\mathbf{xn}} \\ \omega_{\mathbf{yn}} \\ \omega_{\mathbf{zn}} \end{cases} = \begin{cases} \dot{\phi}_{\mathbf{xn}} \\ \dot{\phi}_{\mathbf{yn}} \\ \vdots \\ \phi_{\mathbf{zn}} \end{cases} + \begin{cases} \dot{\theta}_{\mathbf{xn}} \\ \vdots \\ \theta_{\mathbf{yn}} \\ \vdots \\ \theta_{\mathbf{zn}} \end{cases} + \begin{cases} \dot{\psi}_{\mathbf{xn}} \\ \vdots \\ \psi_{\mathbf{yn}} \\ \vdots \\ \psi_{\mathbf{zn}} \end{cases}$$
 (39)

*This can be interpreted to mean that ω_{xn} , ω_{yn} , and ω_{zn} are components of the angular velocity vector of segment No. n parallel to directions X_n , Y_n , and Z_n , respectively.

From Equation 7 it is evident that

$$\begin{cases} \dot{\phi}_{xn} \\ \dot{\phi}_{yn} \\ \dot{\phi}_{zn} \end{cases} = \begin{bmatrix} T^{n} \end{bmatrix}^{T} \begin{cases} 0 \\ 0 \\ \dot{\phi}_{n} \end{cases} \qquad (40)$$

From Figure 2 it can be shown that

$$\begin{cases} X_{n} \\ Y_{n} \\ Z_{n} \end{cases} = \begin{bmatrix} \cos\psi_{n}\cos\theta_{n} & \sin\psi_{n} & -\cos\psi_{n}\sin\theta_{n} \\ -\sin\psi_{n}\cos\theta_{n} & \cos\psi_{n} & \sin\psi_{n}\sin\theta_{n} \\ \sin\theta_{n} & 0 & \cos\theta_{n} \end{bmatrix} \begin{cases} X_{n1} \\ Y_{n1} \\ Z_{n1} \end{cases}$$
 (41)

and it immediately follows that

$$\begin{cases} \dot{\theta}_{xn} \\ \dot{\theta}_{yn} \\ \dot{\theta}_{zn} \end{cases} = \begin{bmatrix} \cos\psi_{n}\cos\theta_{n} & \sin\psi_{n} & -\cos\psi_{n}\sin\theta_{n} \\ -\sin\psi_{n}\cos\theta_{n} & \cos\psi_{n} & \sin\psi_{n}\sin\theta_{n} \\ \sin\theta_{n} & 0 & \cos\theta_{n} \end{bmatrix} \begin{cases} 0 \\ \dot{\theta}_{n} \\ 0 \end{cases}$$
 (42)

Again, using Figure 2 along with the definitions for $\dot{\psi}_n$, $\dot{\psi}_{xn}$, $\dot{\psi}_{yn}$, and $\dot{\psi}_{zn}$ it can be shown that

$$\begin{cases} \dot{\psi}_{xn} \\ \dot{\psi}_{yn} \\ \dot{\psi}_{zn} \end{cases} = \begin{bmatrix} \cos\psi_n & \sin\psi_n & 0 \\ -\sin\psi_n & \cos\psi_n & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{cases} 0 \\ 0 \\ \dot{\psi}_n \end{cases} \qquad (43)$$

Substituting Equations 40, 42, and 43 into Equation 39 yields

$$\omega_{xn} = -\dot{\phi}_n \sin\theta_n \cos\psi_n + \dot{\theta}_n \sin\psi_n \qquad (44a)$$

$$\omega_{yn} = \dot{\phi}_n \sin\theta_n \sin\psi_n + \dot{\theta}_n \cos\psi_n \qquad (44b)$$

$$\omega_{zn} = \dot{\phi}_n \cos\theta_n + \dot{\psi}_n \qquad (44c)$$

For segment No. m (m = 7, 8, 11, or 12), i.e., an arm or leg extremity, define $\dot{\phi}_n$, $\dot{\theta}_n$, and $\dot{\psi}_n$ (n = m-2) as before and introduce $\dot{\alpha}_m$ as the generalized velocity vector that is parallel to both Y_n and Y_m (see Figure 3). In addition, let $\bar{\omega}_{xm}$, $\bar{\omega}_{ym}$, and $\bar{\omega}_{zm}$ be angular velocities about segment coordinate axes X_m , Y_m , and Z_m , respectively.

Using Figure 3 it can be shown that

$$\begin{cases} \overline{\omega}_{\mathbf{x}\mathbf{m}} \\ \overline{\omega}_{\mathbf{y}\mathbf{m}} \\ \overline{\omega}_{\mathbf{z}\mathbf{m}} \end{cases} = \begin{bmatrix} \cos\alpha_{\mathbf{m}} & 0 & -\sin\alpha_{\mathbf{m}} \\ 0 & 1 & 0 \\ \sin\alpha_{\mathbf{m}} & 0 & \cos\alpha_{\mathbf{m}} \end{bmatrix} \begin{cases} \omega_{\mathbf{x}\mathbf{n}} \\ \omega_{\mathbf{y}\mathbf{n}} + \dot{\alpha}_{\mathbf{m}} \\ \omega_{\mathbf{z}\mathbf{n}} \end{cases}$$
(45)

Substituting from Equations 44a, 44b, and 44c into Equation 45 and performing the matrix multiplication yields

$$\bar{\omega}_{xm} = \dot{\phi}_{n} (\sin\theta_{n} \cos\psi_{n} \cos\alpha_{m} + \cos\theta_{n} \sin\alpha_{m}) + \dot{\theta}_{n} \sin\psi_{n} \cos\alpha_{m} - \dot{\psi}_{n} \sin\alpha_{m} \qquad \dots \qquad (46a)$$
$$\bar{\omega}_{ym} = \dot{\phi}_{n} \sin\theta_{n} \sin\psi_{n} + \dot{\theta}_{n} \cos\psi_{n} + \dot{\alpha}_{m} \qquad \dots \qquad (46b)$$
$$\bar{\omega}_{zm} = \dot{\phi}_{n} (\cos\theta_{n} \cos\alpha_{m} - \sin\theta_{n} \cos\psi_{n} \sin\alpha_{m}) + \dot{\theta}_{n} \sin\psi_{n} \sin\alpha_{m} + \dot{\psi}_{n} \cos\alpha_{m} \qquad \dots \qquad (46c)$$

Lagrange's Equations

The equations of motion were derived according to the set of differential equations known as Lagrange's equations for nonconservative systems (4, 6, 19). These may be written as

$$\frac{\mathrm{d}}{\mathrm{dt}}\left(\frac{\partial U}{\partial \dot{\mathbf{q}}_{j}}\right) - \frac{\partial U}{\partial \mathbf{q}_{j}} + \frac{\partial V}{\partial \mathbf{q}_{j}} = Q_{j} \qquad (47)$$

in which t = time, U = kinetic energy of the system, V = potential energy of the system, q_j = generalized coordinates, \dot{q}_j = generalized velocities, Q_j = generalized forces acting on the system which are not necessarily derivable from a potential function, and j = 1, 2, ... 31, for this particular problem.

It was convenient to define

$$Q_{j} = (Q_{f})_{j} + R_{j}$$
 (48)

in which $(Q_f)_j$ = generalized forces resulting from externally applied loads (contact forces)* and R_j = generalized forces due to frictional resistance in the joints (viscous damping) to simulate muscle tone.*

In addition, the potential energy V is of two types, namely V_p , potential energy of position, and V_s , potential energy due to restoring springs* located in each of the two back joints (spinal elasticity), such that

$$\mathbf{v} = \mathbf{v}_{\mathbf{p}} + \mathbf{v}_{\mathbf{s}} \qquad \dots \qquad \dots \qquad \dots \qquad \dots \qquad (49)$$

therefore

$$\frac{\partial V}{\partial q_{j}} = \frac{\partial V}{\partial q_{j}} + \frac{\partial V}{\partial q_{j}} + \cdots$$
 (50)

Define $(F_p)_j$ and $(F_s)_j$ as the generalized forces due to potential energy of position and strain energy potential, respectively,

*To be discussed in detail later under separate heading.
hence

$$(\mathbf{F}_{\mathbf{p}})_{\mathbf{j}} = -\frac{\partial V_{\mathbf{p}}}{\partial q_{\mathbf{j}}} \qquad (51a)$$

$$(\mathbf{F}_{\mathbf{s}})_{\mathbf{j}} = -\frac{\partial V_{\mathbf{s}}}{\partial q_{\mathbf{j}}} \qquad (51b)$$

Substituting from Equations 48, 50, 51a and 51b into Equation 47 gives

$$\frac{d}{dt}\left(\frac{\partial U}{\partial \dot{q}_{j}}\right) - \frac{\partial U}{\partial q_{j}} = (F_{p})_{j} + (F_{s})_{j} + (Q_{f})_{j} + R_{j} \qquad (52)$$

For the articulated body under consideration, Equation 52 represents a set of 31 differential equations which can be categorized as being ordinary, of second order, simultaneous, and nonlinear.

The fact that the differential equations are nonlinear immediately dictates a solution by numerical integration, and the particular approach used was the "Runge-Kutta" method because it is inherently stable. As will be discussed in a later section, this method lends itself well to matrix manipulation, therefore, it was for convenience of solution that the 31 differential equations represented by Equation 52 were rewritten in matrix form as

 $[D] \{ \mathbf{\ddot{q}} \} = \{ \mathbf{E} \} + \{ \mathbf{F}_{\mathbf{p}} \} + \{ \mathbf{F}_{\mathbf{s}} \} + \{ \mathbf{Q}_{\mathbf{f}} \} + \{ \mathbf{R} \} \qquad . . . (53)$

in which matrices [D] and $\{E\}$ are simply a rearrangement of the terms stemming from the left side of Equation 52, i.e., [D] is the matrix of coefficients for the generalized accelerations, $\{q\}$; and $\{E\}$ is a column vector containing all remaining terms with signs reversed. The mathematical formulation of the vehicle occupant is complete and the remainder of this section will be devoted to evaluating the matrix components of Equation 53.

Contribution of Kinetic Energy

to Equations of Motion

The kinetic energy of the articulated body is given by

$$U = \frac{1}{2} \sum_{i=1}^{12} [(\dot{x}_{i})^{2} + (\dot{y}_{i})^{2} + (\dot{z}_{i})^{2}]$$

+ $\frac{1}{2} \sum_{l} [I_{xl}(\omega_{xl})^{2} + I_{yl}(\omega_{yl})^{2} + I_{zl}(\omega_{zl})^{2}]$
+ $\frac{1}{2} \sum_{k} [I_{xk}(\bar{\omega}_{xk})^{2} + I_{yk}(\bar{\omega}_{yk})^{2} + I_{zk}(\bar{\omega}_{zk})^{2}]$. . . (54)

in which l = 1, 2, 3, 4, 5, 6, 9, and 10; k = 7, 8, 11, and 12; M_i is the mass of segment No. 1; and I_{xn} , I_{yn} , and I_{zn} (n = l or k) are the mass moments of inertia about axes X_n , Y_n , and Z_n , respectively* of segment No. n. Equations 27a through 38c define $(\dot{x}_i')^2$, $(\dot{Y}_i')^2$, and $(\dot{z}_1')^2$; whereas ω_{xl} , ω_{yl} , and ω_{zl} are given by Equations 44a through 44c and $\bar{\omega}_{xk}$, $\bar{\omega}_{yk}$; and $\bar{\omega}_{zk}$ are expressed by Equations 46a through 46c.

The kinetic energy contributes matrices [D] and $\{E\}$ to the matrix equation of motion, Equation 53, in the manner prescribed by the left

*Note that no products of inertia (I_{XY} , etc.) appear in Equation 54 which means that axes X_n , Y_n , and Z_n are assumed to be a principal set.

side of Equation 52. Computing the elements of these matrices was a very tedious operation and thus is omitted from this presentation; however, the final result is given in "APPENDIX III.-EQUATIONS OF MOTION."

Contribution of Potential Energy to

Equations of Motion

As previously stated, the potential energy is of two types, namely V_p , potential energy of position, and V_s , potential energy resulting from "spinal elasticity." For clarity, these are discussed separately.

Potential Energy of Position

For the case where the vehicle occupant is in a gravitational field, the direction of the space-fixed axis, Z', was defined as being parallel to the direction of gravitational attraction with opposite sense, e.g., pointing vertically upward when on the surface of the earth. Furthermore, Z'_0 was defined as the arbitrary position of a datum plane (parallel to X'-Y' plane), such that

in which M_i is the mass of segment No. i, g is acceleration due to gravity, and Z'_i is given by Equations 15c through 26c.

The potential energy of position provided the column vector $\{F_p\}$ to the matrix equation of motion, Equation 53, in the manner dictated by Equation 51a such that the jth element of $\{F_p\}$ is

The result of Equation 56 is given in "APPENDIX III.-EQUATIONS OF MOTION."

Potential Energy due to Spinal Elasticity

To simulate in the simplest manner, the ability of the human spine to recover its initial configuration after bending, it was decided to use rotational elastic springs in the two back joints shown in Figure 1. These springs, shown in Figure 4, are defined as lying in the plane formed by the center lines of the two adjoining back segments, regardless of their spatial orientation. Also, the springs are adjustable such that their undeformed positions are assumed to be compatible with the initial configuration of the spine, i.e., the spinal configuration oscillates (in space) about its initial position. Defining the undeformed positions of the two springs as $(\pi - \beta_{10})$ and $(\pi - \beta_{20})$, respectively, the potential energy stored in rotating to positions $(\pi - \beta_1)$ and $(\pi - \beta_2)$, respectively, can be expressed as

$$V_{s} = \frac{1}{2} \kappa_{1} (\beta_{1} - \beta_{10})^{2} + \frac{1}{2} \kappa_{2} (\beta_{2} - \beta_{20})^{2} \qquad (57)$$



in which K_1 and K_2 are spring stiffnesses having units of torque per angular measure.

FIGURE 4.-SPRINGS IN BACK JOINTS

To express β_1 and β_2 in terms of the generalized coordinates $(q_j's)$, define \overline{i}_n , \overline{j}_n , and \overline{k}_n as a triad of unit vectors which coincides with the segment-fixed coordinate axes, X_n , Y_n , and Z_n , of segment No. n. Therefore, from Figure 4 and the definition of the scalar product (dot product) of two vectors, it follows that

$$\cos\beta_1 = \bar{k}_1 \cdot \bar{k}_2 \qquad \dots \qquad (58a)$$
$$\cos\beta_2 = \bar{k}_1 \cdot \bar{k}_4 \qquad \dots \qquad (58b)$$

since \bar{k}_n coincides with the segment's center line.

Before the "dot products" in Equations 58a and 58b can be performed, it is necessary that \bar{k}_1 , \bar{k}_2 , and \bar{k}_4 be resolved to a common coordinate system. Choosing the space-fixed system, shown in Figure 2, as that common reference, and defining \bar{I} , \bar{J} , and \bar{K} as a triad of unit vectors coinciding with space-fixed axes X', Y', and Z', respectively, it follows from Equation 7 that

$$\begin{cases} \overline{i}_{n} \\ \overline{j}_{n} \\ \overline{k}_{n} \end{cases} = \begin{bmatrix} T^{n} \\ T^{n} \end{bmatrix}^{T} \begin{cases} \overline{i} \\ \overline{j} \\ \overline{k} \end{cases} \qquad (59)$$

from which

Substituting this result into Equations 58a and 58b yields

$$\beta_1 = \cos^{-1}(T_{13}^1 T_{13}^2 + T_{23}^1 T_{23}^2 + T_{33}^1 T_{33}^2) \qquad (61a)$$

$$\beta_2 = \cos^{-1}(T_{13}^1 T_{13}^4 + T_{23}^1 T_{23}^4 + T_{33}^1 T_{33}^4) \qquad (61b)$$

where T_{ij}^n is a function of the generalized coordinates as given by Equations 6a through 6i.

Equation 57, the potential energy due to "spinal elasticity", contributes column vector, $\{F_s\}$, to the matrix equation of motion, Equation 53, in the manner dictated by Equation 51b such that the j^{th} element of $\{F_s\}$ is

$$(\mathbf{F_s})_j = -\mathbf{K_1}(\beta_1 - \beta_{10}) \frac{\partial \beta_1}{\partial q_j} - \mathbf{K_2}(\beta_2 - \beta_{20}) \frac{\partial \beta_2}{\partial q_j} \qquad . . . (62)$$



FIGURE 5.-LOCATIONS OF CONTACT SPHERES

contact spheres chosen for the vehicle occupant and their respective locations on the center lines of the body segments. The location of contact sphere No. i is denoted by S_i and its radius by r_i .

Force on a Contact Sphere

Whenever a contact sphere encounters a boundary of the vehicle interior, a contact force results. Assuming that the magnitude* of this force can be computed, there still remains the task of resolving it into components which correspond to the generalized coordinates $(q_j's)$ for use in the equations of motion. These contact force components are the generalized forces which compose the column vector, $\{Q_f\}$, in Equation 53, and are evaluated using the concept of virtual work (4,6,19).

Consider, as in Figure 6, a resultant contact force, F_i , acting on each contact sphere No. i (i = 1,2,...17) at point A defined by $(X'_{Ai}, Y'_{Ai}, Z'_{Ai})$ in the space-fixed coordinate system, and let each force move through a virtual displacement having components $\delta X'_{Ai}$, $\delta Y'_{Ai}$, and $\delta Z'_{Ai}$; then the total virtual work done is

 $\delta W = \sum_{i=1}^{17} \left[F_{xi} \delta X'_{Ai} + F_{yi} \delta Y'_{Ai} + F_{zi} \delta Z'_{Ai} \right] \qquad (63)$

in which F_{xi} , F_{yi} , and F_{zi} are components of force F_i with respect to the space-fixed coordinate system.

*To be discussed in the section entitled "PASSENGER-VEHICLE INTERACTION."

Assuming that F_i can be resolved into equivalent generalized forces, $(Q_f)_j$, then the virtual work is also

in which $\delta \boldsymbol{q}_{i}$ is a generalized virtual displacement.

Define \bar{r}_i as the radius vector extending from the center of contact sphere No. i to the point on the sphere at which F_i acts, i.e., point A in Figure 6.



FIGURE 6.-FORCE ON A CONTACT SPHERE

The components of \bar{r}_i are defined as r_{xi} , r_{yi} , and r_{zi} with respect to X_n , Y_n , and Z_n , respectively; i.e., the segment-fixed coordinates in Figure 6. With this definition of F_i , it is now possible to derive expressions for X'_{Ai} , Y'_{Ai} , Z'_{Ai} in terms of the generalized coordinates for use in Equation 63. From Figures 1, 5, and 6 and Equations 1 and 10,



$$\begin{cases} x_{A6}^{*} \\ y_{A6}^{*} \\ z_{A6}^{*} \\ z_{A6}^{*} \\ z_{A7}^{*} \\ z_{A17}^{*} \\ z_{A17}^{*}$$

where the elements of $[T^n]$ and $[T^{nm}]$ are given by Equations 6a through 6i and Equations 11a through 11i, respectively.

By performing the matrix multiplication indicated by Equations 65a through 65q*, expressions for X'_{Ai} , Y'_{Ai} , and Z'_{Ai} (i = 1, 2...17) are obtained as a function of the generalized coordinates, q_j (j = 1,2...31), such that $\delta X'_{Ai}$, $\delta Y'_{Ai}$, and $\delta Z'_{Ai}$ now become

*Equations 65h through 65p are not given, to avoid unnecessary illustration.

Substituting from Equations 66a through 66c into Equation 63 and rearranging the order of summation yields

$$\delta W = \sum_{j=1}^{31} \sum_{i=1}^{17} (F_{xi} \frac{\partial X'_{Ai}}{\partial q_j} + F_{yi} \frac{\partial Y'_{Ai}}{\partial q_j} + F_{zi} \frac{\partial Z'_{Ai}}{\partial q_j}) \delta q_j \qquad ... (67)$$

By comparison of Equation 64 with Equation 67, it is obvious that

$$(Q_{f})_{j} = \sum_{i=1}^{17} (F_{xi} \frac{\partial X_{Ai}}{\partial q_{j}} + F_{yi} \frac{\partial Y_{Ai}}{\partial q_{j}} + F_{zi} \frac{\partial Z_{Ai}}{\partial q_{j}}) \qquad (68)$$

which defines the terms of column vector $\{Q_f\}$, the result of which is given in "APPENDIX III.-EQUATIONS OF MOTION."

Generalized Forces due to Viscous Damping

in All Joints

The human body's muscular network is principally a source of kinetic energy for body motion, however, it can also act as a dissipator of rotational kinetic energy that is derived from an external source. The latter property of the muscular system could have a

significant effect on body kinematics in a panic situation such as a vehicle collision, since the tensing of muscles is a rather natural reaction. In an attempt to account for this phenomenon in a simple manner, rotational resistance (in the form of viscous damping) was introduced in every joint of the articulated body.

The Viscous Damper

Viscous damping is, by definition (6), a velocity dependent resistance (force or torque) to motion. Figure 7 shows the viscous damper for body joint No. i (i = 1,2...11) which joins body segment No. k to segment No. l. Each body joint contains a similar damper in that it may differ from all others only in the damping constant, J_i , which is a measure of the mechanism's resistance capability.

Note: The plane formed by the center lines of segments k and l contains the viscous damper and the angle β_i .



FIGURE 7.-TYPICAL VISCOUS DAMPER

The resisting torque, $(R_T)_i$, in joint No. i, Figure 7, resulting from the damping constant, J_i , and the angular velocity, $\dot{\beta}_i$, is defined as

Virtual Work of the Damping Torque

The total virtual work done by the instantaneous damping torques, $(R_T)_i$, acting during the virtual angular displacement, $\delta \beta_i$, of each joint No. i, can be expressed as

$$\delta W = -\sum_{i=1}^{11} j_{i} \delta \beta_{i} \qquad (70)$$

where the numbering of the 11 joints is given in Figure 8.

Employing the same reasoning as was used for the two back joints under the heading of "Potential Energy due to Spinal Elasticity," it can be shown that

$$\beta_{i} = \cos^{-1}(T_{13}^{k}T_{13}^{l} + T_{23}^{k}T_{23}^{l} + T_{33}^{k}T_{33}^{l}) \qquad \dots \qquad (71)$$

in which i = 1, 2, 3, 4, 7, 8, and 9; and k and l represent adjoining segment numbers as shown in Figure 8.

For the arm and leg extremities

$$\beta_{i} = \alpha_{m} \qquad (72)$$

in which i = 5, 6, 10, and 11; and m = 7, 8, 12, and 11, respectively, as indicated by Figure 3.



FIGURE 8.-NUMBERING OF JOINTS FOR DAMPING

Equations 71 and 72 show that β_i is a function of the generalized coordinates, q_i , therefore

$$\delta \beta_{\mathbf{i}} = \sum_{\mathbf{j}=1}^{S} \frac{\partial \beta_{\mathbf{i}}}{\partial q_{\mathbf{j}}} \delta q_{\mathbf{j}} \qquad \dots \qquad (73)$$

Substituting from Equation 73 into Equation 70 and interchanging the order of summation yield

Define R_j as the generalized damping force (torque) which corresponds to generalized coordinate, q_j , therefore the total virtual work done by all R_j 's, each acting through a virtual displacement δq_j , is

$$31$$

$$SW = \sum_{j=1}^{N} R_{j} \delta q_{j} \qquad \dots \qquad (75)$$

By comparison of Equations 74 and 75, it is obvious that

Incorporating the knee and elbow constraints described in Figure 3, Equation 76 reduces to

$$R_{j} = -\sum_{i} J_{i}\dot{\beta}_{i} \frac{\partial \beta_{i}}{\partial q_{j}} - J_{5}\dot{\alpha}_{7} \frac{\partial \alpha_{7}}{\partial q_{j}} - J_{6}\dot{\alpha}_{8} \frac{\partial \alpha_{8}}{\partial q_{j}}$$
$$- J_{10}\dot{\alpha}_{12} \frac{\partial \alpha_{12}}{\partial q_{j}} - J_{11}\dot{\alpha}_{11} \frac{\partial \alpha_{11}}{\partial q_{j}} \qquad (77)$$

in which i = 1, 2, 3, 4, 7, 8, and 9 and β_i is obtained by differentiating Equation 71 with respect to time.

Equation 77 defines the elements of column vector, {R}, in the matrix equation of motion, Equation 53, and the final result is in "APPENDIX III.-EQUATIONS OF MOTION."

Joint Stops

To prevent unrealistic angular travel of a body segment relative to its adjoining segment, stops were introduced in the joints.

These "joint stops" are achieved by simply increasing the damping coefficient of joint No. i, J_i , to some large number when β_i approaches an estimated limiting value depending upon whether $\dot{\beta}_i$ is positive or negative.

Figure 9 shows the J_i versus β_i relationship for a typical joint (all body joints except knees and elbows) in which J_i increases from a "normal operation value", $(J_N)_i$, at $(\beta_N)_i$ to a "stop value" of $(J_s)_i$ at $(\beta_s)_i$ if $\dot{\beta}_i$ is positive; where $[(\beta_s)_i - (\beta_N)_i]$ is simply a transition region (approximately five degrees) to prevent abrupt motion (14).



FIGURE 9-RELATIONSHIP FOR BODY JOINT STOPS

Figures 10 and 11 show the J_m versus α_m relationship for the elbows and knees, respectively. It should be noted that two "joint stops" are needed for these since α_m is a generalized coordinate capable of assuming both negative and positive values as opposed to β_i being always positive (dot product of two vectors), hence the subscripts 1 and 2.

This concludes the derivation of the equations of motion for the vehicle occupant.



FIGURE 11.-RELATIONSHIP FOR KNEE STOPS

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PASSENGER-VEHICLE INTERACTION

The idea of passenger-vehicle interaction is analogous to that of placing an object in a glass box, fastening the lid, then observing the motion of the object while the box is shaken. One could conclude from such an experiment that the motion of the object is totally dependent upon the forces afforded to it by the walls of the box (with the exception of gravity) and that these forces are dependent upon the path of the box in space as a function of time. Likewise, before contact forces on the passenger can be computed, it is necessary to define the path of the vehicle.

The Path of the Vehicle in Space

A tabular record of the vehicle's path in space as a function of time is sufficient for purposes of computing contact forces. This record can be fed to the computer program and, if necessary, interpolation between time stations can be performed. However, it is necessary that this tabular information conform to a predetermined standard for specifying the orientation of a body in space. There are many such standards available, one of which has already been discussed, i.e., Equation 1 and Figure 2; hence it would seem to be the logical choice. Nevertheless, a different standard was chosen for a very good reason.

There is a computer program available (9,11,15,20) for the solution of a mathematical model of an automobile capable of

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three-dimensional motion while interacting with the roadway or certain roadside features, obstacles, etc. Since the overall aim for the project, of which this passenger model is only a part, is to study the interaction between vehicle, passenger, and roadway, it is only fitting that input to the passenger program be made compatible with output from the vehicle program. Achieving this compatibility involves the introduction of two additional coordinate systems, namely, the vehicle-fixed system and the space-fixed system for the vehicle; plus a different system of "Eulerian angles," namely, the "aeronautical standard."

Coordinate Systems for the Vehicle

Let X'_v , Y'_v , and Z'_v represent a space-fixed cartesian coordinate system to which is referenced a vehicle-fixed coordinate system, denoted by X_v , Y_v , and Z_v , as shown in Figure 12.





(X'vc, Y'vc, Z'vc) - coordinates of c.g. of vehicle with respect to space-fixed system

FIGURE 12.-VEHICLE COORDINATE SYSTEMS

The vehicle-fixed system transforms back to its space-fixed system as

$$\begin{cases} X_{\mathbf{v}}^{\dagger} \\ Y_{\mathbf{v}}^{\dagger} \\ Z_{\mathbf{v}}^{\dagger} \end{cases} = \begin{bmatrix} & A \\ & A \end{bmatrix} \begin{cases} X_{\mathbf{v}} \\ Y_{\mathbf{v}} \\ Z_{\mathbf{v}} \end{cases} \qquad (78)$$

The elements of [A] are derived using the same procedure that was employed in deriving [T] (see "Euler Transformation and Generalized Coordinates") except that the "Eulerian angles" for this case are slightly different. For the "aeronautical standard" the three successive rotations are as follows:

- 1. Rotate about the Z'_v axis an amount ψ_c to first intermediate position X_{v1} , Y_{v1} , Z_{v1} . This angle is called yaw.
- 2. Rotate about the Y_{v1} axis an amount θ_c to second intermediate position X_{v2} , Y_{v2} , Z_{v2} . This angle is called pitch.
- 3. Rotate about the X_{v2} axis an amount ϕ_c to final position X_v , Y_v , Z_v . This angle is called roll.

Using the above definition, the elements of [A] are found to be

$A_{11} = \cos \psi_{c} \cos \theta_{c}$	• • • • • • •	••	• •	٠	•	•	•	•	(79a)
$A_{12} = -\sin\psi_c \cos\phi_c + c$	$\mathbf{s}_{\mathbf{c}}^{\mathbf{s}} \mathbf{s}_{\mathbf{c}}^{\mathbf{s}} \mathbf{s}_{\mathbf{c}}^{\mathbf{s}} \mathbf{s}_{\mathbf{c}}^{\mathbf{s}}$		• •	•	•	•	•	•	(79b)
$A_{13} = \sin \psi_{c} \sin \phi_{c} + co$	$\mathbf{s}\psi_{\mathbf{c}}\mathbf{sin}\theta_{\mathbf{c}}\mathbf{cos}\phi_{\mathbf{c}}$	•	•••	•	•	•	•	•	(79c)
$A_{21} = \sin\psi_{c}\cos\theta_{c}$	• • • • • • •	• •	••	•	•	•	•	•	(79d)
$A_{22} = \cos\psi_c \cos\phi_c + \sin\phi_c$	$ n\psi_{c} \sin\theta_{c} \sin\phi_{c} $	•	••	•	•	•	•	•	(79e)

$$^{A}_{23} = -\cos\psi_{c}\sin\phi_{c} + \sin\psi_{c}\sin\theta_{c}\cos\phi_{c} \qquad \dots \qquad (79f)$$

Consider a point i on or in the vehicle whose coordinates are (X_{vi}, Y_{vi}, Z_{vi}) in the vehicle-fixed system. With respect to the space-fixed system for the vehicle, the coordinates of point i become

$$\begin{cases} \mathbf{X}_{\mathbf{v}\mathbf{i}}^{\prime} \\ \mathbf{Y}_{\mathbf{v}\mathbf{i}}^{\prime} \\ \mathbf{Z}_{\mathbf{v}\mathbf{i}}^{\prime} \end{cases} = \begin{cases} \mathbf{X}_{\mathbf{v}\mathbf{c}}^{\prime} \\ \mathbf{v}\mathbf{c} \\ \mathbf{z}_{\mathbf{v}\mathbf{c}}^{\prime} \end{cases} + \left[\begin{array}{c} \mathbf{A} \\ \mathbf{Z}_{\mathbf{v}\mathbf{i}}^{\prime} \\ \mathbf{z}_{\mathbf{v}\mathbf{i}}^{\prime} \end{bmatrix} \begin{cases} \mathbf{X}_{\mathbf{v}\mathbf{i}} \\ \mathbf{Y}_{\mathbf{v}\mathbf{i}} \\ \mathbf{Z}_{\mathbf{v}\mathbf{i}} \end{cases} \right]$$
 (80)

in which $(X'_{vc}, Y'_{vc}, Z'_{vc})$ represents the coordinates of the center of gravity (c.g.) of the vehicle with respect to the space-fixed system for the vehicle, as shown in Figure 12.

To simplify the inconvenience of having two independent spacefixed coordinate systems, i.e., one for the articulated body, Figure 1, and another for the vehicle, Figure 12, the following conditions are specified:

- Axes X' and X' are parallel with the same sense which simplifies the most common initial condition of both the vehicle and passenger heading in the same direction.
- Axes Z' and Z' are parallel but of opposite sense which satisfies the fact that acceleration due to gravity should

have the same direction for both passenger and vehicle.

- 3. From 1 and 2 above it follows that Y' and Y' are parallel but of opposite sense if both systems are to be right handed.
- 4. For additional simplification, the origins of both systems are located at the same point in space.

From the above four statements it follows that

$$\begin{cases} X' \\ Y' \\ Z' \end{cases} = \begin{bmatrix} 1 & 0 & 0 \\ 0 & -1 & 0 \\ 0 & 0 & -1 \end{bmatrix} \begin{cases} X'_{v} \\ Y'_{v} \\ Z'_{v} \end{cases} \qquad (81)$$

From Equations 80 and 81 it follows that

as

$$\begin{cases} X_{i}^{\prime} \\ Y_{i}^{\prime} \\ z_{i}^{\prime} \\ z_{i}^{\prime} \end{cases} = \begin{bmatrix} 1 & 0 & 0 \\ 0 & -1 & 0 \\ 0 & 0 & -1 \end{bmatrix} \begin{cases} X_{vc}^{\prime} \\ Y_{vc}^{\prime} \\ z_{vc}^{\prime} \\ vc \end{cases} + \begin{bmatrix} 1 & 0 & 0 \\ 0 & -1 & 0 \\ 0 & 0 & -1 \end{bmatrix} \begin{bmatrix} A \\ A \\ z_{vi} \\ z_{vi} \end{bmatrix}$$
(82)

in which (X'_i, Y'_i, Z'_i) represents the coordinates of a point i, located in or on the vehicle, with respect to the space-fixed coordinate system for the passenger.

Using the fact* that $[A]^{T} = [A]^{-1}$, Equation 82 may be rewritten

$$\begin{cases} \mathbf{X}_{\mathbf{v}\mathbf{i}} \\ \mathbf{Y}_{\mathbf{v}\mathbf{i}} \\ \mathbf{Z}_{\mathbf{v}\mathbf{i}} \end{cases} = \begin{bmatrix} \mathbf{A} \\ \mathbf{A} \end{bmatrix}^{\mathrm{T}} \begin{bmatrix} \mathbf{1} & \mathbf{0} & \mathbf{0} \\ \mathbf{0} & -\mathbf{1} & \mathbf{0} \\ \mathbf{0} & \mathbf{0} & -\mathbf{1} \end{bmatrix} \begin{cases} \mathbf{X}_{\mathbf{i}}^{\dagger} \\ \mathbf{Y}_{\mathbf{i}}^{\dagger} \\ \mathbf{Z}_{\mathbf{i}}^{\dagger} \end{bmatrix} - \begin{bmatrix} \mathbf{A} \\ \mathbf{A} \end{bmatrix}^{\mathrm{T}} \begin{cases} \mathbf{X}_{\mathbf{v}\mathbf{c}} \\ \mathbf{Y}_{\mathbf{v}\mathbf{c}} \\ \mathbf{Y}_{\mathbf{v}\mathbf{c}} \\ \mathbf{Z}_{\mathbf{v}\mathbf{c}} \end{cases}$$
(83)

*Matrix multiplication shows that $[A]^{T}[A] = [I]$, the identity matrix.

Equation 83 may be interpreted as the transformation of a point i, whose coordinates are (X'_i, Y'_i, Z'_i) , from the space-fixed system for the passenger to the vehicle-fixed system; assuming that the position of the vehicle with respect to the space-fixed system for the vehicle is known, i.e., X'_{vc} , Y'_{vc} , Z'_{vc} , ψ_{c} , θ_{c} , and ϕ_{c} .

The Computation of Contact Forces

The Idealized Passenger Compartment

To facilitate the computation of contact forces the vehicle interior or passenger compartment is idealized by a series of planar surfaces. This greatly simplifies the geometry considerations for predicting contact between the articulated body and its confining environment.

Figure 13 shows the numbering of the points where coordinates are necessary for defining the geometry of the idealized passenger compartment. These points are used to express the equations of the planar surfaces and their inward normal vectors, shown in Figure 14.

The Prediction of Contact using Lines and Planes

The computer program is written such that each of the 17 "contact spheres," Figure 5, is checked for contact with each of the 25 planar surfaces of the vehicle interior, Figure 14. The technique used to predict contact is illustrated in Figure 15.



FIGURE 13.-COORDINATES AND DIMENSIONS OF THE IDEALIZED PASSENGER COMPARTMENT

UT .





FIGURE 14.-PLANAR SURFACES AND NORMALS OF THE IDEALIZED PASSENGER COMPARTMENT



FIGURE 15.-CONTACT SPHERE AND PLANAR SURFACE

Consider "contact sphere" No. n whose center has coordinates (X'_n, Y'_n, Z'_n) with respect to the space-fixed coordinate system for the passenger which becomes $(X'_{vn}, Y'_{vn}, Z'_{vn})$ with respect to the vehicle-fixed coordinate system through the use of Equation 83. The distance of this point from the planar surface No. m is found by passing a line through the point, in a direction normal to the plane, then finding the point P where the line and plane intersect, as shown in Figure 15.

The inward normal unit vector \overline{N} is given by

$$\bar{N} = A_{m}\bar{i} + B_{m}\bar{j} + C_{m}\bar{k} \qquad (84)$$

in which \overline{i} , \overline{j} , and \overline{k} form a triad of unit vectors parallel to vehiclefixed axes, X_v , Y_v , and Z_v , respectively; and A_m , B_m , and C_m are calculated from the coordinates of points given in Figure 13.

The equation of the line parallel to \overline{N} and passing through point n, of Figure 15, is

$$\frac{X_v - X_{vn}}{A_m} = \frac{Y_v - Y_{vn}}{B_m} = \frac{Z_v - Z_{vn}}{C_m} \qquad (85)$$

The equation of planar surface No. m, having \bar{N} as its unit normal is

$$A_{m}X_{v} + B_{m}Y_{v} + C_{m}Z_{v} = G_{m} \qquad (86)$$

The constant G_m in Equation 86 can be evaluated as

$$G_{m} = A_{m}X_{i} + B_{m}Y_{i} + C_{m}Z_{i} \qquad (87)$$

in which (X_{vi}, Y_{vi}, Z_{vi}) are the coordinates of any known point on planar surface No. m.

Solving Equations 85 and 86, simultaneously, produces the coordinates of intersection point P (X_{vp} , Y_{vp} , Z_{vp}), shown in Figure 15. These are

vn

$$X_{vp} = X_{vn} + A_{m}L_{xm}$$
(88a)
$$Y_{vp} = Y_{vn} + B_{m}L_{xm}$$
(88b)

$$Z_{vp} = Z_{vn} + C_{m}L_{m} \qquad (88c)$$

in which

$$L_{xm} = G_m - (A_m X_n + B_m Y_n + C_m Z_{vn})$$
 (89)

Define \overline{V}_{pn} as the vector extending from point n to point P in Figure 15. From Equations 88a, 88b, and 88c

$$\overline{\overline{V}}_{pn} = L_{xm} (A_m \overline{i} + B_m \overline{j} + C_m \overline{k}) \qquad (90)$$

in which the parenthetical expression is the unit vector, \bar{N} , therefore the distance from the "contact sphere" center to the planar surface is simply

$$\left| \vec{\mathbf{v}}_{pn} \right| = \left| \mathbf{L}_{\mathbf{xm}} \right|$$
 (91)

Contact is predicted by $L_{\rm XM}$, Equations 89 and 91, considering its sign as well as its absolute value. If $L_{\rm XM}$ is a positive quantity, then $\bar{V}_{\rm pn}$ and \bar{N} have the same sense meaning that the center of the contact sphere has traveled through the planar surface; hence contact has certainly occurred. Likewise, contact has occurred when $L_{\rm XM}$ is zero, since points n and P in Figure 15 are concurrent. When $L_{\rm XM}$ is negative, contact has occurred only if $|L_{\rm XM}|$ is less than $r_{\rm n}$, the radius of "contact sphere" No. n. The above considerations can be summarized mathematically as

$$\Delta = \mathbf{r}_{n} + \mathbf{L}_{xm} \qquad (92)$$

from which contact is assured if Δ is positive. In fact, when Δ is positive not only is contact defined but Δ is also the amount of deformation present.

Magnitude of the Contact Force

With deformation defined by Equation 92, the final step towards computing a force is that of introducing a force-deformation relation for the materials involved; and testing is obviously the only way to obtain such information.

Testing was outside the scope of this particular project, nevertheless a limited source of information (12) was available from which evolved an idealized force-deformation relation. In particular, these tests which consisted of the dynamic loading of various parts of an antropomorphic dummy against cardboard honeycombed material exposed the following pertinent characteristics:

- 1. During the loading process, force increased at a low rate in the lower range of deformation as compared to that of the upper range of deformation.
- 2. Forces were considerably less during the unloading process, thus pointing to a certain amount of energy dissipation. These observations are responsible for the proposed idealized force-deformation curve, shown in Figure 16.

In Figure 16, K_{c1} and K_{c2} are the two slopes of the bilinear curve during loading. The factors μ_1 and μ_2 represent the fraction of strain energy to be conserved during the unloading process in ranges of deformations 1 and 2, respectively; e.g., $\mu_1 = \mu_2 = 1.0$ defines a perfectly elastic collision whereas $\mu_1 = \mu_2 = 0$ defines a completely plastic collision between "contact sphere" and planar



FIGURE 16.-THE BILINEAR DISSIPATIVE SPRING

surface. It should be noted that no attempt is being made to separate body deformation from vehicle deformation, hence the stiffness coefficients (K_{c1} , K_{c2}) for a given surface in the vehicle interior are actually "lumped" parameters which include body stiffness properties.

With the use of Equation 92 and Figure 16 a contact force (\bar{F}_N) is computed when contact occurs. The direction of this force is chosen as normal to the planar surface being contacted since the deformation (Δ) is computed in the normal direction. In addition, a friction force (\bar{F}_{f}) is calculated as some fraction of the normal force. The direction of \bar{F}_{f} is tangent to the planar surface being contacted such that it opposes the motion of the "contact sphere." The forces \bar{F}_{N} and \bar{F}_{f} are added vectorially to produce a resultant force (\bar{F}) as

$$\overline{\mathbf{F}} = \overline{\mathbf{F}}_{\mathbf{N}} + \overline{\mathbf{F}}_{\mathbf{f}} \qquad (93)$$

whose components are F_{vx} , F_{vy} , and F_{vz} with respect to vehicle-fixed axes X_v , Y_v , and Z_v , respectively.

From Equations 78 and 81

$$\begin{cases} F_{xi} \\ F_{yi} \\ F_{zi} \end{cases} = \begin{bmatrix} 1 & 0 & 0 \\ 0 & -1 & 0 \\ 0 & 0 & -1 \end{bmatrix} \begin{bmatrix} A \\ A \end{bmatrix} \begin{cases} F_{vx} \\ F_{vy} \\ F_{vz} \end{cases} \qquad (94)$$

in which F_{xi} , F_{yi} , and F_{zi} are the components of \overline{F} , with respect to the space-fixed coordinate system for the passenger, acting on "contact sphere" No. i. These three components appear in column vector, $\{Q_f\}$, of the matrix equation of motion, Equation 53.

Lap and Torso Restraint Belts

Othersources of contact forces to which the vehicle occupant may be subjected are the safety belts. The lap belt has its ends anchored at arbitrary points and loops around the pelvic area ("contact sphere" No. 3). Likewise, the torso belt has its ends anchored at arbitrary points and loops around the upper torso area ("contact sphere" No. 2).

It is assumed that the center line of a belt defines a plane which contains the center of its respective "contact sphere" at all times. This facilitates the definition of the restraining force vector which, by definition, also lies in this plane, as shown in Figure 17.

The angles shown in Figure 17 are expressed as

$$\bar{\theta}_{1} = \cos^{-1}(\frac{\mathbf{r}_{1}}{\left|\bar{\mathbf{P}}_{c1}\right|}) \qquad (95)$$

$$\overline{\theta}_2 = \cos^{-1}(\frac{r_1}{|\overline{P}_{c2}|}) \qquad (96)$$

$$\overline{\theta}_{3} = \cos^{-1}\left(\frac{\mathbf{P}_{c1} \cdot \mathbf{P}_{c2}}{\left|\mathbf{\bar{P}}_{c1}\right| \left|\mathbf{\bar{P}}_{c2}\right|}\right) \qquad (97)$$

$$\beta = \pi - \frac{1}{2}(\overline{\theta}_1 + \overline{\theta}_2 + \overline{\theta}_3) \qquad \dots \qquad (98)$$

$$\alpha = \frac{1}{2} \left| \overline{\theta}_1 + \overline{\theta}_2 + \overline{\theta}_3 - \pi \right| \qquad (99)$$

$$\gamma = \frac{1}{2}(\overline{\theta}_1 + \overline{\theta}_3 - \overline{\theta}_2) \qquad \dots \qquad (100)$$

Equations 95 through 100 hold for all special cases, e.g., the sides of the belt being parallel or even pointing away from each other, etc.



FIGURE 17.-VIEW NORMAL TO PLANE CONTAINING BELT AND CENTER OF CONTACT SPHERE
The length of the belt at any time, t, is

$$L_{Bt} = |\vec{P}_{c2}| \sin \theta_2 + |\vec{P}_{c1}| \sin \theta_1 + 2r_i\beta \qquad \dots \qquad (101)$$

in which r_i is the radius of "contact sphere" No. i (i = 2 or 3).

The belt elongation at any time, t, is

$$\Delta L_{B} = L_{Bt} - L_{Bo} \qquad (102)$$

in which the initial length (L_{BO}) contains "slack" for loose-fitting belts. If $\Delta L_B^{\leq} 0$, the loop force (F_L) is zero. For $\Delta L_B^{\geq} 0$, F_L is computed from the idealized curve shown in Figure 18.



FIGURE 18.-IDEALIZED FORCE-ELONGATION CURVE FOR A RESTRAINT BELT

The magnitude of the restraining force $|\overline{F}_{R}|$ is defined as

The restraining force vector is expressed as

$$\overline{\mathbf{F}}_{\mathrm{R}} = \frac{\left|\overline{\mathbf{F}}_{\mathrm{R}}\right|}{\left|\overline{\mathbf{F}}_{3\mathrm{c}}\right|} \overline{\mathbf{P}}_{3\mathrm{c}} \qquad (104)$$

in which the coordinates of point P_3 (needed to find \overline{P}_{3c}) correspond to the components of \overline{P}_{30} , which is found by

while the vector \bar{P}_{32} is

$$\bar{P}_{32} = \frac{|\bar{P}_{32}|}{|\bar{P}_{12}|} \bar{P}_{12} \qquad \dots \qquad (106)$$

It can be shown that

$$\left|\bar{P}_{32}\right| = \frac{\left|\bar{P}_{c2}\right|_{\sin\gamma}}{\sin(\gamma + \bar{\gamma})} \qquad \dots \qquad (107)$$

in which, as shown in Figure 17

$$\overline{\gamma} = \cos^{-1}\left(\frac{\overline{\mathbf{P}}_{12} \cdot \overline{\mathbf{P}}_{c2}}{\left|\overline{\mathbf{P}}_{12}\right| \left|\overline{\mathbf{P}}_{c2}\right|}\right) \qquad (108)$$

The components of \overline{F}_R , Equation 104, with respect to the vehiclefixed axes are added to F_{vx} , F_{vy} , and F_{vz} of Equation 94, for the case of i = 2 or 3; i.e., "contact spheres" for upper torso and pelvic area, respectively. Therefore the restraint force simply becomes part of the resultant force, on a "contact" sphere, which is transformed to the space-fixed system of the passenger for use in column vector {Q} of Equation 53.

Initial Position of Passenger

The initial position of the vehicle occupant is completely arbitrary and is specified by initializing all of the 31 generalized coordinates or degrees of freedom $(q_j, j = 1, ..., 31)$. This could prove to be a difficult task in the case where the vehicle assumes a skewed initial position. To alleviate this situation, a routine was devised by which the passenger is automatically seated properly in the vehicle, regardless of the vehicle's initial position.

Seating the Passenger

To simplify matters it is assumed that, initially, the passenger is heading in the same direction as the vehicle; i.e., the Y_n axis for each of the 12 body segments (n = 1,...12) is parallel (opposite sense) to Y_v , the vehicle-fixed Y direction. However, as shown in Figure 19, the vehicle occupant's posture remains arbitrary and is defined by parameters γ_1 , γ_2 , γ_3 , γ_4 , γ_5 , α_7 ($\alpha_7 = \alpha_8$), X_{VH} , and Y_{VH} . The angle γ_n specifies the position of body segment No. n with respect to the vehicle-fixed coordinate system as shown in Figure 20.

From Figure 20 it is apparent that

$$\begin{cases} X_{\mathbf{v}} \\ Y_{\mathbf{v}} \\ Z_{\mathbf{v}} \end{cases} = \begin{bmatrix} \cos \gamma_{\mathbf{n}} & 0 & -\sin \gamma_{\mathbf{n}} \\ 0 & -1 & 0 \\ -\sin \gamma_{\mathbf{n}} & 0 & -\cos \gamma_{\mathbf{n}} \end{bmatrix} \begin{cases} X_{\mathbf{n}} \\ Y_{\mathbf{n}} \\ Z_{\mathbf{n}} \end{cases}$$
 (109)



FIGURE 19.-INPUT PARAMETERS FOR PASSENGER SEATING OPTION



FIGURE 20.-RELATION BETWEEN SEGMENT-FIXED AND VEHICLE-FIXED COORDINATE SYSTEMS

Manipulation of Equations 78, 81, and 109 produces



in which

$$[B^{n}] = \begin{bmatrix} 1 & 0 & 0 \\ 0 & -1 & 0 \\ 0 & 0 & -1 \end{bmatrix} \begin{bmatrix} A \\ A \end{bmatrix} \begin{bmatrix} \cos \gamma_{n} & 0 & -\sin \gamma_{n} \\ 0 & -1 & 0 \\ -\sin \gamma_{n} & 0 & -\cos \gamma_{\underline{n}} \end{bmatrix}$$
 (111)

"Seating the passenger" or rather defining the generalized coordinates, such that the occupant is initially sitting as shown in Figure 19, is accomplished as follows:

- 1. From the information specified for Figures 13 and 19 along with the force-deformation properties of the seat and the initial position of the vehicle, it is possible, with the use of Equation 82, to determine the proper coordinates of the reference point on the articulated body as shown in Figure 1. This initially establishes X'_{T1} , Y'_{T1} , and Z'_{T1} or generalized coordinates q_1 , q_2 , and q_3 , respectively.
- 2. Since $[T^n] = [B^n]$ at initial time*, initial values of generalized coordinates q_4 through q_{23} can be computed by equating terms of the matrices for $n = 1, \dots 6$.
- 3. From the coordinates of the heels, X_{VH} and Z_{VH} , in Figure 20, it is possible to compute the required angular positions of the leg segments. Knowing these, initial values of generalized coordinates q_{24} through q_{31} can be computed by setting $[T^n] = [B^n]$ for n = 9 and 10.

The operations described in items 1, 2, and 3 above are included in the computer program as a convenience to the user. However, the user of the program still has the option of specifying arbitrary values for the generalized coordinates if such is desired. This and other options are given in "APPENDIX IV.-DESCRIPTION OF INPUT TO THE COMPUTER PROGRAM."

*This is realized by comparison of Equations 1 and 110.

SOLUTION OF EQUATIONS

The Runge-Kutta Method

Equation 53, the matrix equation of motion for the vehicle occupant, was solved using the Runge-Kutta method for ordinary differential equations (3). This method was applicable since Equation 53 is of the form where the highest derivative, \dot{q}_j (j=1, ... 31), can be expressed as a function of the lower derivative, \dot{q}_j , the dependent variable, q_i , and the independent variable, t (time).

Define the right-hand side of Equation 53 as

$$\{\overline{E}\} = \{E\} + \{F_p\} + \{F_s\} + \{Q_f\} + \{R\}$$
 (112)

Substituting from Equation 112 into Equation 53 and solving for $\{\ddot{q}\},$ such that

$$\{\ddot{q}\} = [D]^{-1}\{\overline{E}\}$$
 (113)

Using the notation, $[D({q})]$, to indicate that [D] is a function of the elements of ${q}$, Equation 113 may be rewritten as

$$\{\ddot{q}\} = [D(\{q\})]^{-1}\{\overline{E}(\{\dot{q}\}, \{q\})\}$$
 (114)

in which $\{\overline{E}\}$ is shown to be a function of the elements of $\{q\}$ and $\{q\}$.

Solution of Equation 114 was achieved through the following successive computations, for the increment of integration, Δt :

$\{\overline{K_1}\} = \Delta t\{\dot{q}\}$
$\{\overline{L}_1\} = \Delta t[D(\{q\})]^{-1}\{\overline{E}(\{\dot{q}\}, \{q\})\}$ (116)
$\{\overline{K}_2\} = \Delta t\{\dot{q}\} + \frac{1}{2} \Delta t\{\overline{L}_1\} \qquad \dots \qquad $
$\{\overline{L}_{2}\} = \Delta t [D(\{q\} + \frac{1}{2} \{\overline{K}_{1}\})]^{-1} \{\overline{E}(\{\dot{q}\} + \frac{1}{2} \{\overline{L}_{1}\}, \{q\} + \frac{1}{2}\}$
$\{\overline{\kappa}_1\}\}$
$\{\overline{K}_3\} = \Delta t\{\dot{q}\} + \frac{1}{2} \Delta t\{\overline{L}_2\} \qquad \dots \qquad \dots \qquad \dots \qquad \dots \qquad (119)$
$\{\overline{L}_3\} = \Delta t [D(\{q\} + \frac{1}{2} \{\overline{K}_2\})]^{-1} \{\overline{E}(\{\dot{q}\} + \frac{1}{2} \{\overline{L}_2\}, \{q\} + \frac{1}{2}\}$
$\{\overline{\kappa}_2\}\}$
$\{\overline{K}_{4}\} = \Delta t\{\dot{q}\} + \Delta t\{\overline{L}_{3}\} \qquad \dots \qquad (121)$
$\{\overline{L}_{4}\} = \Delta t [D(\{q\} + \{\overline{K}_{3}\})]^{-1} \{\overline{E}(\{\dot{q}\} + \{\overline{L}_{3}\}, \{q\} + \{\overline{K}_{3}\})\} . (122)$
$\{\overline{K}\} = \frac{1}{6} \{\overline{K_1}\} + \frac{1}{3} \{\overline{K_2}\} + \frac{1}{3} \{\overline{K_3}\} + \frac{1}{6} \{\overline{K_4}\} $ (123)
$\{\overline{L}\} = \frac{1}{6} \{\overline{L}_1\} + \frac{1}{3} \{\overline{L}_2\} + \frac{1}{3} \{\overline{L}_3\} + \frac{1}{6} \{\overline{L}_4\} $ (124)
The new values at t are

The solution technique is stepwise and repetitive with each step incorporating Equations 115 through 127 while utilizing the previous values of $\{\dot{q}\}$ and $\{q\}$.

Discussion of the Computer Program

The computer program for the passenger model was written in Fortran IV for the IBM 360/65 computer. The program listing of about 4,000 lines was considered too lengthy for presentation in this report. Documentation of the "input data" for the program is given in "APPENDIX IV. DESCRIPTION OF INPUT TO THE COMPUTER PROGRAM."

The Subroutines

"MAIN" is the control center for the program from which subroutines "INPUT", "CONST", "SEATIN", "PRNTIN", RNGKTA", "POSVEH", and "OUTPUT" are called.

Subroutine "INPUT" reads all data required by the program.

Subroutine "CONST" computes terms which remain constant throughout the program.

Subroutine "SEATIN" computes the passenger's initial position such that he is seated properly inside the vehicle.

Subroutine "PRNTIN" prints the information which is read in "INPUT" and computer in "SEATIN".

Subroutine "RNGKTA" integrates the equations of motion according to the Runge-Kutta method, Equations 115 through 127, and calls subroutine "SETUP" in the process.

Subroutine "SETUP" calculates the elements of [D] and $\{\overline{E}\}$ according to the functional changes indicated in Equations 115 through 127 then solves for $\{\overline{q}\}$ as shown in Equation 114. This is accomplished by calling subroutines "DMATX", "EMATX", "QFORCE", "JOINTS", "POTEGY", and "DGELS".

Subroutine "DMATX" computes the elements of [D] according to the equations given in "APPENDIX III. - EQUATIONS OF MOTION".

Subroutine "EMATX" computes the elements of {E} according to the equations given in "APPENDIX III. - EQUATIONS OF MOTION".

Subroutine "QFORCE" computes the elements of $\{Q_f\}$ according to the equations given in "APPENDIX III. - EQUATIONS OF MOTION". This is accomplished by first calling subroutine "CONTAC".

Subroutine "JOINTS" computes the elements of $\{F_s\}$ and $\{R\}$ according to the equations given in "APPENDIX III. - EQUATIONS OF MOTION".

Subroutine "POTEGY" computes the elements of $\{F_p\}$ according to the equations given in "APPENDIX III. - EQUATIONS OF MOTION".

Subroutine "DGELS" solves a set of simultaneous equations utilizing the "Gaussian elimination" technique on the upper triangular portion of a symmetric matrix of coefficients. Subroutine "CONTAC" (called by "QFORCE") computes contact forces due to passenger-vehicle interaction. "CONTAC" calls subroutine "CENTER".

Subroutine "CENTER" calculates the position of each "contact sphere" center with respect to the vehicle-fixed coordinate system.

Subroutine "POSVEH" updates the position of the vehicle in space by linearly interpolating between time stations of the tabular input for vehicle position versus time; it is called by "MAIN".

Subroutine "OUTPUT" prepares the solution for printing; it is called by "MAIN".

Subroutine "ANSWER" prints all pertinent information obtained from the solution and is called by "OUTPUT".

Output

The solution of the equations of motion consists of a time history of the following quantities:

- the coordinates of the end points of each body segment with respect to the vehicle-fixed coordinate system;
- 2. acceleration components of the center of mass of each body segment with respect to the segment-fixed coordinate system for that segment (this is total acceleration);
- angular acceleration components of each body segment with respect to its segment-fixed coordinate system;
- angular velocity components of each body segment with respect to its segment fixed coordinate system;

- 5. the force on each body "contact sphere" plus the identification of whatever vehicle interior surface is being hit;
- 6. the coordinates of the point of application of the contact force with respect to the center of the contact sphere in segment-fixed coordinates (only for the head, chest and pelvic area);
- the restraining force applied to the body by the lap and torso restraint belts.

VALIDATION STUDY

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Scope

An original objective of this project was to validate the passenger model's three-dimensional response capabilities by comparison with existing test data of this nature. To produce conclusive results, any such data should provide the following information:

- a time history of the vehicle's path in three-dimensional space (preferably numerical instead of photographic*);
- a corresponding time history of the occupant's dynamic behavior, e.g., accelerations, forces, or a photographic record of motion;
- a quantitative description of the occupant, i.e., dimensions, weight, etc.;
- force-deformation properties of the pertinent vehicle suffaces (could be measured).

Unfortunately, test results possessing all these qualities were not to be located and funds for full scale testing were not available thus precluding a validation of the general case at this time.

However, suitable test results were available (12) for a partial validation, i.e., the case of a frontal automobile collision.

*Photographic records can be used for application of the passenger model but only after validation is achieved.

Test Data

The test data used for comparison were generated at the Biomechanics Research Center of Wayne State University, Detroit, Michigan under the direction of Cornell Aeronautical Laboratory (CAL), Inc., Buffalo, New York for the United States Public Health Service, March, 1967. All experimental results shown in this report were copied directly from CAL's documentation of these tests (12).

The tests consisted of a dummy seated on a cart capable of controlled deceleration. Mounted on the cart were target assemblies for head, chest and knee impact. Accelerations of various parts of the dummy and impact forces were measured by instrumentation while the motion of the dummy was recorded on high speed film. Several cases were run consisting of lap restraint, lap and torso restraint, and no restraint for initial velocities of 10 and 20 miles per hour.

Also measured and documented (12) were the force-deformation characteristics of the targets, seat, and restraint belts plus the dummy's initial position and the amount of friction in each of its joints.

Discussion of Results

Response Comparison for No Restraint at 20 MPH Cart Velocity

Figures 21 through 28 show the comparison of dummy kinematics, head and chest forces, plus head and chest accelerations for the case of no restraint with 20 MPH cart velocity. Agreement between

simulated motion and the high speed film record is excellent. Quantitative comparisons (forces and accelerations) are only fair as a result of the idealized vehicle interior (simulation) being geometrically different from the target assemblies used in the test. The simulation utilized a full instrument panel and steering wheel as opposed to isolated targets of about six to eight inches in diameter for the test. This resulted in hand contact in the simulation which was absent during the test. Also the knee targets were inclined for the test producing a downward force component whereas the knees in the simulation contacted a vertical surface (instrument panel) with friction as the only downward force.

Response Comparison for Lap Restraint at 20 MPH Cart Velocity

Figures 29 through 36 show the comparison of dummy kinematics, head and chest forces, plus head and chest accelerations for the case of lap restraint with 20 MPH cart velocity. Agreement between simulated motion and the high speed film record is good, however it seems that the lap belt in the simulation is stiffer than that of the test causing a higher contribution of energy to the rotational mode. This observation is consistent with the higher predicted contact forces and accelerations as shown in Figures 31 through 36. This comparison was also subject to discrepancies resulting from the geometrical differences between idealized vehicle interior and target assemblies as discussed for the case of no restraint.



FIGURE 21.-DUMMY KINEMATIC COMPARISON NO RESTRAINT-CART VELOCITY 20 MPH, .000-.080 SEC.



FIGURE 22.-DUMMY KINEMATIC COMPARISON NO RESTRAINT-CART VELOCITY 20 MPH, .120-.200 SEC.



Time - Sec.

FIGURE 23.-HEAD FORCE-NO RESTRAINT-CART VELOCITY 20 MPH



FIGURE 24.-CHEST FORCE-NO RESTRAINT-CART VELOCITY 20 MPH



FIGURE 25.-HEAD ACCELERATION IN SEGMENT X DIRECTION-NO RESTRAINT-CART VELOCITY 20 MPH



FIGURE 26.-HEAD ACCELERATION IN SEGMENT Z DIRECTION-NO RESTRAINT-CART VELOCITY 20 MPH



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FIGURE 27.-CHEST ACCELERATION IN SEGMENT X DIRECTION-NO RESTRAINT-CART VELOCITY 20 MPH



Time - Sec.

FIGURE 28.-CHEST ACCELERATION IN SEGMENT Z DIRECTION-NO RESTRAINT-CART VELOCITY 20 MPH



FIGURE 29.-DUMMY KINEMATIC COMPARISON LAP RESTRAINT-CART VELOCITY 20 MPH, .000-.080 SEC.



FIGURE 30.-DUMMY KINEMATIC COMPARISON LAP RESTRAINT-CART VELOCITY 20 MPH, .120-.200 SEC.



FIGURE 31.-HEAD FORCE-LAP RESTRAINT-EART VELOCITY 20 MPH



Time - Sec.

FIGURE 32.-CHEST FORCE-LAP RESTRAINT-CART VELOCITY 20 MPH



FIGURE 33.-HEAD ACCELERATION IN SEGMENT X DIRECTION-LAP RESTRAINT-CART VELOCITY 20 MPH



FIGURE 34.-HEAD ACCELERATION IN SEGMENT Z DIRECTION-LAP RESTRAINT-CART VELOCITY 20 MPH



FIGURE 35.-CHEST ACCELERATION IN SEGMENT X DIRECTION-LAP RESTRAINT-CART VELOCITY 20 MPH



FIGURE 36.-CHEST ACCELERATION IN SEGMENT Z DIRECTION-LAP RESTRAINT-CART VELOCITY 20 MPH

Response Comparison for Lap and Torso Restraint at 20 MPH Cart Velocity

Figures 37 through 42 show the comparison of dummy kinematics plus head and chest accelerations for the case of lap and torso restraint with 20 MPH cart velocity. Agreement between simulated motion and the high speed film record is good including unsymmetrical body movements as a result of the unsymmetrical torso restraint belt. However the spring action of the torso belt seems to be excessive in the simulation since the arms are whipped back into the seat as shown in the last frame of Figure 38. This test was subject to the same sources of possible discrepancy, as the two cases previously discussed, plus an additional one. The anchor points for the ends of the belt were unknown and therefore estimated for the simulation. This could account for some of the difference in arm kinematics.

The comparison of accelerations, Figures 39 through 42 is also fair but better than the other two cases in that peak values are closer and the shapes of the curves are more compatible with test results.

Closure

The deceleration pattern used in the test (12) approached a 20 g square wave for a duration of about .08 second. It is interesting to note that the passenger experienced levels of acceleration on the order of 80 g's with durations of approximately .03 second for the cases of no restraint and lap restraint; and levels of approximately 40 g's with durations of about .03 second for the case of lap and torso restraint. This points to the fact that in some instances, the response of the vehicle is no indication of what the passenger actually feels, as was stated in the "INTRODUCTION."



FIGURE 37.-DUMMY KINEMATIC COMPARISON LAP AND TORSO RESTRAINT-CART VELOCITY 20 MPH, .000-.080 SEC.



FIGURE 38.-DUMMY KINEMATIC COMPARISON LAP AND TORSO RESTRAINT-CART VELOCITY 20 MPH, .120-.200 SEC.



FIGURE 39.-HEAD ACCELERATION IN SEGMENT X DIRECTION-LAP AND TORSO RESTRAINT-CART VELOCITY 20 MPH


FIGURE 40.-HEAD ACCELERATION IN SEGMENT Z DIRECTION-LAP AND TORSO RESTRAINT-CART VELOCITY 20 MPH



Time - Sec.

FIGURE 41.-CHEST ACCELERATION IN SEGMENT X DIRECTION-LAP AND TORSO RESTRAINT-CART VELOCITY 20 MPH



Time - Sec.

FIGURE 42.-CHEST ACCELERATION IN SEGMENT Z DIRECTION-LAP AND TORSO RESTRAINT-CART VELOCITY 20 MPH

CONCLUSIONS

The analytical model described herein provides the engineering profession with a useful tool with which to study vehicle and roadway problems culminating in saving lives and reducing occupant injuries. Admittedly, the model was validated for the planar case only; but this in no way precludes its application to three-dimensional motion, especially if qualitative results are being sought.

More specifically the passenger model is a solution to the problem of predicting the motion, acceleration, and forces experienced by a vehicle occupant during a collision or violent maneuver of the vehicle. From this standpoint, the application of the passenger model includes:

- 1. the evaluation of roadway geometry, i.e., sideslopes, ditches, terrain involving a variation of vertical and horizontal alignment, etc.; roadside safety features such as the breakaway sign, energy absorbing impact cushions, etc.; roadside protective barriers such as guardrails, bridge rails, median barriers, etc.;
- the design of the vehicle interior and restraint systems;
 the study of the dynamic behavior of a pedestrian when struck by an automobile; and
- 4. the study of collisions involving more than one vehicle.

RECOMMENDATIONS FOR EXTENDED RESEARCH

To implement the passenger model to its fullest capacity, consideration should be given to the following:

- validating the model for the general three-dimensional case utilizing controlled full-scale testing;
- conducting a parameter study to expose measures necessary to achieve consistent correlation with testing, i.e., damping constants, spring stiffnesses, etc.;
- compiling a source of force-deformation characteristics for various vehicle interior surfaces through testing;
- 4. determining the extent to which the vehicle model (9, 11, 15, 20) may be used in conjunction with the passenger model as an integral design tool;
- extending the model to include passive safety devices,
 e.g., the air bag, etc.;
- 6. writing a computer program to predict the dynamic response of a pedestrian struck by an automobile; this would involve only a slight modification of the present passenger program.

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APPENDIX II.-NOTATION

The following symbols are used in this report:

 F_{τ} = loop force in a restraint belt;

 \overline{F}_R = restraining force on the body due to a restraint belt; F_{xi}, F_{vi}, F_{zi} = components of the resultant force on "contact sphere"

No. i with respect to the space-fixed coordinate

system for the passenger;

g = acceleration due to gravity;

Ixn,Iyn,Izn = principal mass moments of inertia of body segment No. n
with respect to segment-fixed coordinates;

J_n = viscous damping coefficient for body joint No. n; K_{B1},K_{B2} = the two slopes of the idealized bilinear force-elongation relation for a restraint belt;

K_{c1},K_{c2} = the two slopes of the idealized bilinear force-deformation relation for a particular surface of the vehicle interior;

L_i = length of body segment No. 1;

L_{Bt} = length of a restraint belt at any time, t;

 L_{BO} = initial length of a restraint belt;

L = distance of the center of a "contact sphere" from vehicle interior surface No. m;

M_i = mass of body segment No. i;

- \overline{P}_{ij} = vector extending from point j to point i;
- q_j = the jth generalized coordinate or degree of freedom
 (j = 1,2...31);
- q_j = first time derivative of q_j, i.e., the jth generalized velocity;
- q_j = second time derivative of q_j, i.e., the jth generalized acceleration;
- r, = the radius of "contact sphere" No. i;
- - S = the position of "contact sphere" No. i on its respective segment;

U = kinetic energy;

V = potential energy;

X',Y',Z' = the space-fixed cartesian coordinate system for the passenger;

 X_n, Y_n, Z_n = the cartesian coordinate system fixed in segment No. n; X_v, Y_v, Z_v = the cartesian coordinate system fixed at the center of gravity of the vehicle;

 $X'_n, Y'_n, Z'_n =$ the coordinates of point n with respect to the spacefixed coordinate system for the passenger;

 X'_{v}, Y'_{v}, Z'_{v} = the space-fixed cartesian coordinate system for the vehicle;

- X_{vi},Y_{vi},Z_{vi} = the coordinates of point i with respect to the vehiclefixed coordinate system;
- $X'_{T1}, Y'_{T1}, Z'_{T1} =$ coordinates of the reference point on the articulated body (terminal end of segment No. 1) with respect to the space-fixed coordinate system for the passenger; these correspond to generalized coordinates q_1, q_2 , and q_3 , respectively;
- $X'_{vc}, Y'_{vc}, Z'_{vc} =$ coordinates of the vehicle center of gravity with respect to the space-fixed system for the vehicle; $\dot{X}'_{n}, \dot{Y}'_{n}, \dot{Z}'_{n} =$ components of velocity of the center of mass of body segment No. n with respect to the space-fixed system for the passenger;
 - α_{i} = the angle between an extreme body segment (forearm or lower leg) and its adjoining body segment; α_{7} , α_{8} , α_{11} , and α_{12} correspond to generalized coordinates q_{22} , q_{23} , q_{30} , and q_{31} , respectively;

 β_i = angular displacement of body joint No. i (as in Figure 7); $\dot{\beta}_i$ = angular velocity of body joint No. i $(\frac{d\beta}{dt})$;

 Δ = deformation during a collision between a "contact sphere" and a surface of the vehicle interior; ΔL_{R} = change in length (elongation) of a restraint belt;

- ϕ_1, θ_1, ψ_1 = the Eulerian angles of body segment No. i; these are also generalized coordinates;
- $\phi_{c}, \theta_{c}, \psi_{c}$ = the Eulerian angles of the vehicle;
- $\dot{\phi}_i, \dot{\theta}_i, \dot{\psi}_i$ = Eulerian angular velocities;
 - p = distance from the reference end of body joint No. i to
 its center of mass;

$$\bar{\rho}_i = L_i - \rho_i$$

 $\omega_{xn}, \omega_{yn}, \omega_{zn}$ = angular velocities of body segment No. n about principal axes fixed in the segment;

 $\bar{\omega}_{xm}, \bar{\omega}_{ym}, \bar{\omega}_{zm}$ = angular velocities of the body extremities (forearm and lower leg segments) with respect to principal axes fixed in segment No. m.

APPENDIX III.-EQUATIONS OF MOTION

$$[D]{\{\ddot{q}\}} = \{E\} + \{F_n\} + \{F_n\} + \{Q_e\} + \{R\}$$
(1)

The above matrix equation represents a set of 31 nonlinear, second-order, simultaneous differential equations describing the motion of the vehicle passenger.

Matrices D and E are strictly a result of kinetic energy considerations; D may be thought of as a pseudo-mass matrix and E as an inertial force matrix.

Column vector \ddot{q} is composed of the generalized accelerations. Column vector F_p contains the generalized forces due to potential energy of position (gravity loads).

Column vector F_s contains the generalized forces resulting from the potential energy of the rotational springs in the two back joints.

Column vector Q_f is comprised of generalized forces derived from externally applied loads, i.e., nonconservative contact forces from collisions with the vehicle interior.

Column vector R accommodates the nonconservative generalized forces due to frictional resistance in the joints (viscous damping).

In defining the elements of the matrices in Eq. 1, the following notation is used:

 $\{\dot{\mathbf{q}}\} = \frac{\mathbf{d}}{\mathbf{dt}}(\{\mathbf{q}\})$ $\{\mathbf{q}\} = \frac{\mathbf{d}}{\mathbf{dt}}(\{\dot{\mathbf{q}}\})$

$s\phi_i = sin\phi_i;$	$c\phi_i = cos\phi_i$
$s\theta_i = sin\theta_i;$	$c\theta_i = cos\theta_i$
$s\psi = sin\psi_i;$	$c\psi = \cos\psi$ i
sa _i = sina _i ;	ca _i = cosa _i

During the derivation process, it was discovered that the physical properties of the articulated body (length, mass, and mass-moment of inertia) combined in a repetitive manner thus creating these subsequent constant terms:

$$\begin{split} c_{1} &= \sum_{n=1}^{l_{2}} M_{n} \\ c_{2} &= -\overline{P}_{1}M_{1} - L_{1}(M_{4} + M_{9} + M_{10} + M_{11} + M_{12}) \\ c_{3} &= P_{2}M_{2} + L_{2}(M_{3} + M_{5} + M_{6} + M_{7} + M_{8}) \\ c_{4} &= P_{3}M_{3} \\ c_{5} &= -\overline{P}_{4}M_{4} - L_{4}(M_{9} + M_{10} + M_{11} + M_{12}) \end{split}$$

$$\begin{aligned} c_{6} &= -\overline{P_{5}} M_{5} - L_{5} M_{7} \\ C_{7} &= -\overline{P_{6}} M_{6} - L_{6} M_{8} \\ c_{8} &= -\overline{P_{7}} M_{7} \\ C_{9} &= -\overline{P_{8}} M_{8} \\ C_{10} &= -\overline{P_{9}} M_{9} - L_{9} M_{11} \\ C_{11} &= -\overline{P_{10}} M_{10} - L_{10} M_{12} \\ C_{12} &= -\overline{P_{11}} M_{11} \\ C_{13} &= -\overline{P_{12}} M_{12} \\ C_{14} &= L_{5} (M_{5} - M_{6} + M_{7} - M_{8}) \\ C_{15} &= L_{11} (M_{9} - M_{10} + M_{11} - M_{12}) \\ C_{16} &= M_{1} \overline{P_{1}}^{2} + L_{1}^{2} (M_{4} + M_{9} + M_{10} + M_{11} + M_{12}) \\ C_{17} &= M_{2} P_{2}^{2} + L_{2}^{2} (M_{3} + M_{5} + M_{6} + M_{7} + M_{8}) \\ C_{18} &= M_{3} P_{3}^{3} \\ C_{19} &= M_{4} \overline{P_{4}}^{2} + L_{4}^{2} (M_{9} + M_{10} + M_{11} + M_{12}) \\ C_{20} &= L_{3}^{2} (M_{5} + M_{6} + M_{7} + M_{8}) \\ c_{21} &= M_{5} \overline{P_{6}}^{2} + M_{7} L_{5}^{2} \\ C_{22} &= M_{6} \overline{P_{6}}^{2} + M_{8} L_{6}^{2} \\ c_{23} &= M_{7} \overline{P_{7}}^{2} \\ c_{24} &= M_{8} \overline{P_{8}}^{2} \\ c_{25} &= L_{4}^{2} (M_{9} + M_{10} + M_{11} + M_{12}) \\ c_{26} &= M_{9} \overline{P_{9}}^{2} + M_{11} L_{9}^{2} \\ c_{27} &= M_{10} \overline{P_{10}}^{2} + M_{12} L_{10}^{2} \end{aligned}$$

$$C_{28} = M_{11} \bar{P}_{11}^{2}$$
$$C_{29} = M_{12} \bar{P}_{12}^{2}$$

Following is the upper triangular portion of the symmetric D matrix (by rows), excluding zero terms:

$$\begin{split} D_{11} &= C_{1} \\ D_{14} &= -C_{2} \ s\phi_{1} s\phi_{1} \\ D_{15} &= C_{2} \ c\phi_{1} \ c\phi_{1} \\ D_{17} &= -C_{3} \ s\phi_{2} \ o\phi_{2} \ + C_{14} \ (s\phi_{2} \ c\phi_{2} \ s\phi_{2} \ - C\phi_{2} \ c\psi_{2}) \\ D_{18} &= C_{3} \ c\phi_{2} \ c\phi_{2} \ + C_{14} \ c\phi_{2} \ s\phi_{2} \ s\psi_{2} \\ D_{19} &= C_{4} \ (s\phi_{2} \ s\psi_{2} \ - \ c\phi_{2} \ c\phi_{2} \ c\psi_{2}) \\ D_{10} &= -C_{4} \ s\phi_{3} \ s\phi_{3} \\ D_{11} &= C_{4} \ c\phi_{3} \ c\phi_{3} \\ D_{18} &= -C_{5} \ s\phi_{4} \ s\phi_{4} \ + C_{15} \ (s\phi_{4} \ c\phi_{4} \ s\psi_{4} \ - C\phi_{4} \ c\psi_{4}) \\ D_{14} &= C_{5} \ c\phi_{4} \ c\phi_{4} \ + C_{15} \ c\phi_{4} \ s\phi_{4} \ s\psi_{4} \\ D_{16} &= -C_{6} \ s\phi_{5} \ s\phi_{5} \ - C_{8} \ (s\phi_{5} \ c\phi_{5} \ c\psi_{5} \ s\phi_{7} \ + \ c\phi_{5} \ s\phi_{5} \ s\phi_{7} \ - \ c\phi_{5} \ s\phi_{5} \ s\phi_{7} \\ D_{17} &= C_{6} \ c\phi_{5} \ c\phi_{5} \ s\psi_{5} \ s\phi_{7} \ + \ s\phi_{6} \ c\psi_{5} \ s\phi_{7} \ - \ c\phi_{5} \ s\phi_{5} \ c\psi_{5} \ s\phi_{7} \ - \ c\phi_{5} \ s\phi_{5} \ c\psi_{5} \ s\phi_{7} \ - \ c\phi_{5} \ s\phi_{7} \ - \ c\phi_{5} \ s\phi_{7} \ c\psi_{5} \ s\phi_{7} \ - \ c\phi_{5} \ s\phi_{7} \ - \ c\phi_{5} \ s\phi_{7} \ c\psi_{5} \ s\phi_{7} \ - \ c\phi_{5} \ s\phi_{7} \ c\psi_{7} \ - \ c\phi_{5} \ s\phi_{7} \ c\psi_{7} \ - \ c\phi_{5} \ s\phi_{7} \ c\psi_{7} \ s\phi_{7} \ - \ c\phi_{7} \ s\phi_{7} \ s\phi_{7} \ - \ c\phi_{7} \ s\phi_{7} \ s\phi_{7} \ s\phi_{7} \ - \ c\phi_{7} \ s\phi_{7} \ s\phi_{7} \ - \ c\phi_{7} \ s\phi_{7} \ s\phi_{7} \ s\phi_{7} \ - \ c\phi_{7} \ s\phi_{7} \ s\phi_{7} \ - \ c\phi_{7} \ s\phi_{7} \$$

$$\begin{array}{l} D_{1\,1q} = -C_{1} \, s\phi_{6} \, s\phi_{2} - C_{q} \left(s\phi_{6} \, c\phi_{2} \, c\psi_{6} \, s\sigma_{8} + c\phi_{6} \, s\psi_{6} \\ \\ \quad s\sigma_{8} + s\phi_{6} \, s\phi_{2} \, c\sigma_{8} \right) \\ D_{1\,20} = C_{7} \, c\phi_{6} \, C\phi_{2} \, + C_{q} \left(c\phi_{6} \, c\phi_{6} \, c\sigma_{8} - c\phi_{6} \, s\phi_{6} \, c\psi_{6} \\ \\ \quad s\sigma_{8} \right) \\ D_{1\,21} = -Cq \left(c\phi_{6} \, c\phi_{6} \, s\psi_{6} \, s\sigma_{8} + \, s\phi_{6} \, c\psi_{6} \, s\sigma_{8} \right) \\ D_{1\,22} = \, cg \left(c\phi_{5} \, c\phi_{5} \, c\psi_{5} \, c\sigma_{7} - s\phi_{8} \, s\psi_{5} \, c\sigma_{7} - c\phi_{5} \, s\phi_{5} \, s\sigma_{7} \right) \\ D_{1\,23} = \, Cq \left(c\phi_{6} \, c\phi_{6} \, c\psi_{6} \, c\sigma_{8} - \, s\phi_{6} \, s\psi_{6} \, c\sigma_{8} - c\phi_{6} \, s\phi_{8} \, s\phi_{7} \right) \\ D_{1\,23} = \, Cq \, (c\phi_{6} \, c\phi_{6} \, c\psi_{6} \, c\sigma_{8} - \, s\phi_{6} \, s\psi_{6} \, c\sigma_{8} - c\phi_{6} \, s\phi_{8} \, s\phi_{7} \right) \\ D_{1\,24} = \, -C_{10} \, s\phi_{9} \, s\sigma_{9} - C_{12} \, \left(s\phi_{9} \, c\phi_{9} \, c\psi_{9} \, s\sigma_{11} + c\phi_{9} \\ \\ s\psi_{9} \, s\sigma_{11} \, + \, s\phi_{9} \, s\phi_{9} \, c\sigma_{11} \right) \\ D_{1\,25} = \, C_{10} \, c\phi_{9} \, c\phi_{9} \, + \, C_{12} \, \left(c\phi_{9} \, c\phi_{7} \, c\sigma_{11} - c\phi_{9} \, s\phi_{7} \\ \\ c\psi_{9} \, s\sigma_{11} \right) \end{array}$$

$$D_{126} = -C_{12} (c \phi_{q} c \phi_{q} s \psi_{q} s \phi_{11} + s \phi_{q} c \psi_{q} s \phi_{11})$$

$$D_{127} = -C_{11} s \phi_{10} s \phi_{10} - C_{13} (s \phi_{10} c \phi_{10} c \psi_{10} s \phi_{12} + c \phi_{10} s \phi_{10} s \phi_{12} + s \phi_{10} s \phi_{10} c \phi_{12})$$

$$P_{128} = G_{11} C\phi_{10} C\phi_{10} + G_{13} (C\phi_{10} C\phi_{10} C\phi_{12} - C\phi_{10} \delta\phi_{10} C\phi_{10} + G_{12} - C\phi_{10} \delta\phi_{10} + G_{12} - G_{12$$

$$D_{12q} = -G_{13} \left(C\phi_{10} C\phi_{10} \leq \phi_{10} \leq \sigma_{12} + 5\phi_{10} C\phi_{10} \leq \sigma_{12} \right)$$
$$D_{130} = G_{12} \left(C\phi_{q} C\phi_{q} C\phi_{q} C\phi_{11} - 5\phi_{q} \leq \phi_{q} C\phi_{11} - C\phi_{q} \\ \leq \phi_{q} \leq \sigma_{11} \right)$$

 $D_{131} = C_{13} (c\phi_{10} C \Theta_{10} C \phi_{10} C \sigma_{12} - s\phi_{10} s \phi_{10} C \sigma_{12} - c\phi_{10}$ $s \Theta_{10} s \sigma_{12})$

$$\begin{array}{l} D_{2,2} = G_{1} \\ D_{2,4} = C_{2,5}\phi_{1,5}\phi_{1} \\ D_{2,5} = C_{2,5}\phi_{1,5}\phi_{1} \\ D_{2,7} = C_{3,5}\phi_{2,5}\phi_{2} - G_{4}\left(c\phi_{2,5}\phi_{2,5}\phi_{2,5}+5\phi_{2,5}\phi_{2,5}\right) \\ D_{2,8} = C_{3,5}\phi_{2,5}\phi_{2,5} + C_{4,5}\phi_{2,5}\phi_{2,5}\phi_{2,5} \\ D_{2,8} = C_{3,5}\phi_{2,5}\phi_{2,5}\phi_{2,5}\phi_{2,5}\phi_{2,5}\phi_{2,5} \\ D_{2,9} = C_{4,5}\left(s\phi_{2,5}\phi_{2,5}\phi_{2,5}\phi_{2,5}\phi_{2,5}\phi_{2,5}\right) \\ D_{2,10} = C_{4,5}\phi_{3,5}\phi_{3,5} \\ D_{2,13} = C_{5,5}\phi_{4,5}\phi_{3,5} - C_{15}\left(c\phi_{4,5}\phi_{4,5}\phi_{4,5}\phi_{4,5}\phi_{4,5}\right) \\ D_{2,14} = C_{5,5}\phi_{4,5}\phi_{3,5} - C_{15}\left(c\phi_{4,5}\phi_{4,5$$

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$$D_{223} = Cq (sp_{c}Cp_{c}Cp_{c}Cq_{B} + Cp_{c}sp_{c}Cq_{B} - sp_{c}sp$$

 $D_{2 24} = C_{10} c_{pq} s_{eq} + C_{12} (c_{pq} c_{eq} c_{pq} s_{11} - s_{pq} s_{pq} s_{pq} s_{11} + c_{pq} s_{eq} c_{n11})$

 $D_{2\,25} = C_{10} \, s\phi_{q} \, c_{\varphi} \, + \, C_{12} \, (s\phi_{q} \, c_{\varphi} \, c_{\chi_{11}} - \, s\phi_{q} \, s_{\varphi} \, c_{\varphi} \, c_{\chi_{11}} - \, s\phi_{q} \, s_{\varphi} \, c_{\varphi} \, c_{\chi_{11}} \,)$

 $D_{2 26} = C_{12} (c \phi_{1} c \phi_{2} s \kappa_{11} - s \phi_{2} c \phi_{2} s \phi_{2} s \kappa_{11})$ $D_{2 27} = C_{11} c \phi_{10} s \phi_{10} + C_{13} (c \phi_{10} c \phi_{10} c \phi_{10} s \kappa_{12} - s \phi_{10} s \phi_{10})$ $s \kappa_{12} + c \phi_{10} s \phi_{10} c \kappa_{12})$

 $D_{2 28} = C_{11} \epsilon \phi_{10} C \phi_{10} + C_{13} (\delta \phi_{10} C \phi_{10} C \alpha_{12} - \delta \phi_{10} \delta \phi_{10}$ $C \psi_{10} \delta \alpha_{12})$

 $D_{2 2q} = C_{13} (C \phi_{0} C \psi_{0} 5 \alpha_{12} - 5 \phi_{0} C \phi_{0} 5 \psi_{0} 5 \alpha_{12})$ $D_{2 30} = C_{12} (S \phi_{0} C \phi_{0} C \psi_{0} C \alpha_{11} + C \phi_{0} 5 \psi_{0} C \alpha_{11} - 5 \phi_{0}$ $5 \phi_{0} 5 \alpha_{11})$

 $D_{231} = C_{13} \left(s_{10}^{4} C_{10}^{4} C_{10}^{4} C_{12}^{4} + C_{10}^{4} s_{10}^{4} C_{12}^{4} - s_{10}^{4} \right)$ $s_{70}^{4} s_{12}^{4} \right)$

$$D_{3 3} = G_{1}$$

$$D_{3 5} = -C_{2 5 6_{1}}$$

$$D_{3 5} = -C_{3 5 6_{2}} + C_{14} - C_{62} 5 4_{2}$$

$$D_{3 9} = C_{14} 5 6_{2} - C_{42}$$

$$D_{3 11} = -C_{4} 5 6_{3}$$

$$D_{3 14} = -C_{5} 5 6_{4} + C_{15} - C_{6} 5 4_{4}$$

$$\begin{array}{l} D_{3 \ 16} = G_{15} \ 5 \oplus_{q} C \mu_{q} \\ D_{3 \ 17} = -C_{4} \ 5 \oplus_{5} - C_{8} \left(C \oplus_{5} C \mu_{5} \ 5 \infty_{7} + 5 \oplus_{5} C \infty_{7} \right) \\ D_{3 \ 18} = C_{8} \ 5 \oplus_{5} \ 5 \mu_{5} \ 5 \infty_{7} \\ D_{3 \ 20} = -G_{7} \ 5 \oplus_{6} - C_{9} \left(C \oplus_{6} C \mu_{6} \ 5 \sigma_{8} + \sigma \oplus_{6} C \sigma_{8} \right) \\ D_{3 \ 21} = C_{9} \ 5 \oplus_{6} \ 5 \mu_{6} \ 5 \infty_{8} \\ D_{3 \ 22} = -C_{8} \left(5 \oplus_{5} C \mu_{5} \ C \sigma_{7} + C \oplus_{5} \ 5 \sigma_{7} \right) \\ D_{3 \ 23} = -C_{9} \left(5 \oplus_{6} C \mu_{6} \ C \sigma_{8} + C \oplus_{7} \ 5 \sigma_{8} \right) \\ D_{3 \ 25} = -C_{10} \ 5 \oplus_{7} - C_{12} \left(c \oplus_{7} C \mu_{9} \ 5 \sigma_{11} + 5 \oplus_{7} C \sigma_{11} \right) \\ D_{3 \ 26} = C_{12} \ 5 \oplus_{7} \ 5 \mu_{9} \ 5 \sigma_{12} \\ D_{3 \ 29} = -C_{12} \ (5 \oplus_{7} C \mu_{9} \ 5 \sigma_{12} + 5 \oplus_{70} \ C \sigma_{12}) \\ D_{3 \ 29} = -C_{12} \ (5 \oplus_{7} C \mu_{9} \ C \sigma_{11} + C \oplus_{9} \ 5 \sigma_{11}) \\ D_{3 \ 30} = -C_{12} \ (5 \oplus_{70} C \mu_{70} \ C \sigma_{12} + C \oplus_{70} \ 5 \sigma_{12}) \\ D_{3 \ 31} = -C_{13} \ (5 \oplus_{70} C \mu_{10} \ C \sigma_{12} + C \oplus_{70} \ 5 \sigma_{12}) \end{array}$$

$$D_{4} q = 5^{2} \Theta_{1} \left[C_{16} + I_{X_{1}} c^{2} \psi_{1} + I_{X_{1}} 5^{2} \psi_{1} \right] + I_{Z_{1}} c^{2} \Theta_{1}$$

$$D_{4} 5 = (I_{Y_{1}} - I_{X_{1}}) 3\psi_{1} c\psi_{1} 3\Theta_{1}$$

$$D_{4} 6 = I_{Z_{1}} c\Theta_{1}$$

$$D_{4} 13 = L_{1} 3\Theta_{1} \left\{ C_{15} \left[(3\phi_{4} c\Theta_{4} 3\psi_{4} - c\phi_{4} c\psi_{4}) 3\phi_{1} + (c\phi_{4} c\Theta_{4} 3\psi_{4} + s\phi_{4} c\psi_{4}) c\phi_{1} \right] - C_{5} S\Theta_{4}$$

$$(3\phi_{4} 3\phi_{1} + c\phi_{4} c\phi_{1}) \right\}$$

$$\begin{array}{l} D_{q \ 14} = L_{1} s_{q \ 1} \left(c_{q \ 1} s_{q \ 1} - s_{q \ 2} c_{q \ 2} \right) \left[c_{s \ 2} c_{q \ 1} + C_{1s \ 3} s_{q \ 5} s_{q \ 5} \right] \\ D_{q \ 15} = L_{1} C_{15} s_{q \ 5} \left[\left(s_{q \ 4} s_{q \ 4} - c_{q \ 2} c_{q \ 2} c_{q \ 5} \right) s_{q \ 1} \\ + \left(s_{q \ 2} c_{q \ 2} + c_{q \ 5} s_{q \ 5} \right) c_{q \ 7} \right] \\ D_{q \ 24} = -L_{1} s_{q \ 5} \left\{ C_{10} s_{q \ 6} \left(s_{q \ 5} s_{q \ 4} + c_{q \ 6} c_{q \ 5} \right) + C_{12} \\ & \left[\left(s_{q \ 4} s_{q \ 6} c_{-q \ 1} + s_{q \ 6} c_{0 \ 6} c_{q \ 5} s_{q \ 5} + c_{q \ 7} \right] \\ S_{q \ 9} s_{q \ 5} s_{q \ 1} \right] s_{p, \ 1} + \left(c_{q \ 6} c_{q \ 6} c_{q \ 5} s_{q \ 5} + c_{q \ 5} s_{q \ 5} \right) c_{q \ 7} \right] \\ D_{q \ 25} = L_{1} s_{q \ 7} \left[\left(c_{q \ 5} s_{q \ -} -c_{q \ 5} s_{q \ 7} \right) \left[c_{10} c_{q \ 7} + C_{12} \\ & \left(c_{q \ 6} c_{q \ 1} + c_{q \ 5} s_{q \ 5} c_{-1} \right) c_{q \ 7} \right] \right] \\ D_{q \ 25} = -L_{1} s_{q \ 7} \left[\left(c_{q \ 5} s_{q \ -} -c_{q \ 5} s_{q \ 7} \right) \left[c_{10} c_{q \ 7} + C_{12} \\ & \left(c_{q \ 6} c_{q \ 1} - s_{q \ 6} c_{q \ 5} s_{q \ 7} + c_{q \ 5} s_{q \ 6} c_{q \ 7} \right) \right] \\ D_{q \ 25} = -L_{1} s_{q \ 7} \left[\left(c_{q \ 5} s_{q \ 5} - c_{q \ 5} s_{q \ 7} \right) \left[c_{10} c_{q \ 7} \\ & \left(c_{q \ 6} c_{q \ 5} s_{q \ 5} + c_{q \ 6} c_{q \ 7} \right) \right] \\ D_{q \ 26} = -L_{1} s_{q \ 7} \left[\left(c_{10} c_{0 \ 5} c_{12} + c_{12} \\ & \left(c_{10} c_{0 \ 5} c_{12} + c_{12} \\ & \left(c_{10} c_{12} c_{12} + c_{12} \\ & \left(c_{10} c_{12} c_{12} c_{12} \right) c_{q \ 7} \right] \right] \\ D_{q \ 27} = -L_{1} s_{q \ 7} \left[\left(c_{10} c_{0 \ 5} c_{12} + c_{12} c_{12} c_{12} \\ & \left(c_{10} c_{12} c_{12} c_{12} c_{12} \right) c_{12} \right] \right] \\ D_{q \ 28} = L_{1} s_{q \ 7} \left[\left(c_{10} c_{0 \ 5} c_{12} + c_{12} c_{12} c_{12} c_{12} \\ & \left(c_{10} c_{12} c_{12} c_{12} c_{12} c_{12} c_{13} \\ & \left(c_{10} c_{12} c_{1$$

De 31 ° L, C, 3 3 0, [(cp, co, cy, cr, 2 - 5 0, 5 4) cr, 2 - c\$ 50, 50, 50, 2) 5\$ - (5\$ 00 CO, CY, Cor, 2 + C\$, 5 4% Caiz - 5 \$ 50, 50, 50, 2 (\$)]

D55 = G6 + Ix, 52 4, + Iy, 624, D513 = L, co, { C5 300 (30 ch, - ch, 50) - C, 5 [(304 Coq 54 - cop C4) cop - (cop coq 54 + 504 cy) 56,7} D5 14 = - L, { (L5 Cog + (15 504 54) (cp co co co + 50) Co, 50, 1+ 50, (C5 50, -(45 CO, 54)} D3 15 = - 61 G15 }- C 44 [304 50, + C0, C04 (CA4 CA) + 5\$\$_4 5\$,)]+ 3\$\$ c+, (5\$, cp, -c\$, 5\$,)] D5 24 = L, co, Elio 300 [30, cd, - cd, 30,] + C12 [(30, Co, Cyg 5~1, + Cpg 544 50, + 54g 50g Cal,) Cp, - (chg cog cyg sa,1-5¢g syg sa,1+C¢g sog ca,1)5¢.]} D5 25 = - 61 { Con [Go Cog + Giz (Cog Con - 50g Cup 5~,,)][cqcdq+sq,sqg]+30,[Goseq + C12 (cog cy say + 30g can)]} D5 26 = 41 412 { 50,1 [con ((con sy + song cy) C\$, - (C\$q C\$4, -5\$q 000 54) 3\$ + 500 544 50,] {

$$D_{5,27} = L_{1} cq [C_{11} 3q_{10} (sd_{10} cd_{10} - cd_{10} 3d_{1}) + C_{13}$$

$$\left\{ (3d_{10} cd_{10} cd_{10} cd_{10} sd_{12} + cd_{10} sd_{10} sd_{12} + sd_{10} sd_{10} cd_{12}) cd_{1} - 3d_{1} (cd_{10} cd_{10} cd_{10} sd_{12} + sd_{10} sd_{12} - sd_{10} sd_{12} - sd_{10} sd_{12} - sd_{10} cd_{12} - sd_{10} cd_{11} - sd_{10} cd_{10} cd_{10} cd_{10} cd_{11} - sd_{10} sd_{11} - cd_{10} sd_{11} - sd_{10} sd_{11} - cd_{10} sd_{11} - sd_{10} sd_{10} cd_{11} - sd_{10} sd_{11} - sd_{10} sd_{11} - sd_{10} sd_{11} - sd_{10} sd_{11} - sd_{10} sd_{10} cd_{11} - sd_{10} sd_{10} cd_{10} sd_{10} cd_{10} - sd_{10} sd_{10} cd_{10} cd_{10} cd_{10} cd_{10} cd_{10} cd_{10} cd_{10} c$$

$$D_{6} = I_{\overline{z}_{1}}$$

D7 7 = 3202 (C17 + Ix2 C242 + Iy2 5242) - 2 L2 C14 302 Coz 542 + Iz, c202 + C20 [c202 5242 + (24) 7 D78 = cy2[so2 sy2(Iy2-Ix2-C20)-L2(402] D7 9 = Co2 (C20+I22) - L2 C14 302 342 $D_{7/0} = L_2 C_4 \frac{50}{50} \frac{50}{3} (50 + C_3 + C_3 + C_2)$ $D_{7 | 1} = L_2 (q c \theta_3 s \theta_2 (s \phi_3 c \phi_2 - c \phi_3 s \phi_2))$ $D_{7/1} = \int L_2 s \phi_2 s \sigma_2 - L_s (s \phi_2 c \sigma_2 s \psi_2 - c \phi_2 c \psi_2) T$ Γ C6 3\$ 55 + C8 (3\$ 505 C45 5x7+C\$5 545 507 + 5\$ 505 505 CM7)]+[Lz C\$ 502 -Ls (cp2 c+ sp2 + sp2 cy2) [C6 cp5 505 + 68 (6 6 6 6 6 4 507 - 30 54 507 + 605 505 Car,)7 $D_{7 17} = \left[C_{6} C_{95} + C_{8} (C_{95} C_{7} - 50_{5} C_{45} S_{7}) \right]$ {C\$=[L3(3\$= C= 3\$= C\$= C\$= C\$= C\$= L2 5\$==] $+ 5\phi_5 [L_2 c\phi_2 5\phi_2 - L_5 (c\phi_2 c\phi_2 5\psi_2 + 5\phi_2$ Cyz)7? D718 = C850, {(C\$ C0, 54, + 3\$ C\$)[L23\$, 502 -L3 (3\$ C = 3\$ = C\$ C\$ C\$)7 + (C\$ C\$ - 5\$ 60, 5\$ 5\$ [L2 C\$ 50, - L3 (C\$ 00, 5\$ + 5 \$ C \$]] }

$$\begin{array}{l} D_{7} & 1 q = \left[L_{2} 3 \phi_{2} 5 \phi_{2} + L_{3} (3 \phi_{2} C \phi_{2} 5 \psi_{2} - C \phi_{2} C \psi_{2}) \right] \\ & \left[C_{7} 5 \phi_{c} 5 \phi_{2} + C_{9} (3 \phi_{c} C \phi_{c} C \psi_{c} 5 \pi_{3} + C \phi_{c} 5 \psi_{c} 5 \phi_{c} + 5 \phi_{c} 5 \phi_{c} 2 \phi_{c} 5 \phi_{c} + C \phi_{c} 5 \phi_{c} 5 \phi_{c} + C \phi_{c} (c \phi_{c} C \phi_{c} 5 \pi_{3} - 3 \phi_{c} 5 \psi_{c} 5 \phi_{c} + C \phi_{c} (c \phi_{c} C \phi_{c} 5 \pi_{3} - 3 \phi_{c} 5 \psi_{c} 5 \phi_{c} + C \phi_{c} (c \phi_{c} C \phi_{c} 5 \pi_{3} - 3 \phi_{c} 5 \psi_{c} 5 \phi_{c} + C \phi_{c} - 5 \phi_{c} C \phi_{c} 5 \phi_{c} + 5 \phi_{c} C \phi_{c} 5 \phi_{c} + 5 \phi_{c} C \phi_{c} 5 \phi_{c} - 5 \phi_{c} C \phi_{c} 5 \phi_{c} + 5 \phi_{c} C \phi_{c} 5 \phi_{c} + 5 \phi_{c} C \phi_{c} 5 \phi_{c} + 5 \phi_{c} C \phi_{c} - 5 \phi_{c} C \phi_{c} 5 \psi_{c} + 5 \phi_{c} C \phi_{c} - 5 \phi_{c} C \phi_{c} 5 \psi_{c} + 5 \phi_{c} C \phi_{c} - 5 \phi_{c} C \phi_{c} 5 \psi_{c} + 5 \phi_{c} C \phi_{c} - 5 \phi_{c} C \phi_{c} 5 \psi_{c} - C \phi_{c} C \psi_{c} - 5 \phi_{c} C \phi_{c} 5 \psi_{c} - C \phi_{c} C \phi_{c} - 5 \phi_{c} C \phi_{c} 5 \psi_{c} - 5 \phi_{c} C \phi_{c} - 6 \phi_{c} C \phi_{c} - 5 \phi_{c} C \phi_{c} - 5 \phi_{c} C \phi_{c} - 6 \phi_{c} C \phi_{c} - 6 \phi_{c} C \phi_{c} - 6 \phi_{c} C \phi_{c} - 5 \phi_{c} C \phi_{c} - 5 \phi_{c} C \phi_{c} - 6 \phi_{c} -$$

Dy 23 = Conf (Cof con cy cag - 30 54 cag - cof 50 5a8)[-L25\$2582-L3(5\$2C82542-C\$2 (42)]+ (50, c= c4 cas+ co 54, cas-50) 50, 5x, 8) [L2 c \$ 502 + L5 (c\$ 2 542 + 5\$ cyz)]}

$$D_{8} = C_{17} + 3^{2} \psi_{2} (C_{20} + I_{x_{2}}) + I_{x_{2}} C^{2} \psi_{2}$$

$$P_{8} q = -L_{2} C_{4} C_{4_{2}}$$

$$D_{8 10} = L_{2} C_{4} C_{2} = 3 \cdot 3 (c_{4}^{2} + 5 \cdot 4_{3}^{2} - 5 \cdot 4_{3}^{$$

$$\begin{array}{l} \mathcal{O}_{8} & 1 q = (L_{2} \, co_{2} - L_{3} \, so_{2} \, sy_{2}) \left\{ \begin{array}{l} - cy_{2} \left[\begin{array}{l} C_{7} \, sy_{2} \, so_{2} \\ + \ Cq \left(sy_{2} \, co_{2} \, cy_{2} \, so_{8} + cy_{2} \, sy_{2} \, so_{8} + cy_{4} \, sy_{2} \\ so_{8} \, co_{8} \, sy_{1}^{2} + sy_{2} \left[\begin{array}{l} C_{7} \, cy_{6}^{2} \, so_{8}^{2} + Cq \left(cy_{6}^{2} \, cy_{2}^{2} \\ - cy_{4}^{2} \, so_{8}^{2} - sy_{2}^{2} \, sy_{2}^$$

$$Dq q = C_{20} + I_{z_{2}}$$

$$Dq IL = L_{3} \left\{ (cp_{2} cp_{2} cp_{2} - sp_{2} sp_{2}) \right[C_{4} sp_{4} sp_{5} + C_{8} - (sp_{2} cp_{2} cp_{2} + cp_{3} sp_{3} sp_{3} + sp_{5} op_{5} cp_{3}) \right] - (sp_{2} cp_{2} cp_{2} + cp_{3} sp_{3} sp_{3} + sp_{5} op_{5} cp_{3}) \right] - (sp_{2} cp_{2} cp_{2} + cp_{3} sp_{3} sp_{3} + sp_{5} op_{5} cp_{3}) \right] - (sp_{2} cp_{2} cp_{2} + cp_{3} sp_{3} sp_{3} + sp_{5} op_{5} cp_{3}) \right] - (sp_{2} cp_{2} cp_{2} + cp_{3} sp_{3} sp_{3} + sp_{5} op_{5} cp_{3}) \right]$$

$$Dq IT = L_{3} \left\{ \left[C_{4} cp_{5} + C_{8} (cp_{5} cp_{1} - sp_{5} cp_{5} sp_{5} sp_{3} + sp_{5} (sp_{5} cp_{2} cp_{2} + cp_{2} sp_{3}) - sp_{5} (sp_{5} cp_{5} sp_{5} + cp_{5} (cp_{5} cp_{2} pp_{2} + cp_{2} sp_{3}) - sp_{5} (sp_{5} cp_{5} sp_{5} + cp_{5} (cp_{5} sp_{5}) - sp_{5} (sp_{5} sp_{5}) - sp_{5} sp_{5} sp_{5} cp_{5} sp_{5}) \right]$$

$$Dq IB = -L_{5} CB sp_{1} \left[(cp_{5} cp_{5} sp_{5} + sp_{5} cp_{5} sp_{5}) - sp_{5} sp_{5} sp_{5} cp_{5} sp_{5}) \right]$$

$$Cp_{2} (sp_{2} cp_{2} cp_{2} + cp_{2} sp_{2}) - sp_{5} sp_{5} sp_{5} cp_{5} sp_{5}) \left[(sp_{5} cp_{5} sp_{5} - sp_{5} cp_{5} sp_{5}) \right] \right]$$

$$Dq IIP = -L_{5} \left\{ (cp_{2} cp_{2} cp_{2} cp_{2} - sp_{5} sp_{5} sp_{5} cp_{5} sp_{5}) \right] \left[(sp_{5} sp_{5} cp_{5} sp_{5}) \right] \right]$$

$$Cp_{1} (sp_{2} cp_{2} cp_{2} cp_{2} + cp_{2} sp_{2}) \left[(cp_{5} cp_{5} sp_{5} sp_{5} cp_{5} sp_{5}) \right] \left[(cp_{5} sp_{5} cp_{5} sp_{5}) \right] \left[(cp_{5} sp_{5} cp_{5} sp_{5}) \right] \right] \left[(cp_{5} sp_{5} cp_{5} sp_{5}) \right] \left[(cp_{5} sp_{5} cp_{5} sp_{5}) \right] \left[(cp_{5} sp_{5} cp_{5} sp_{5}) \right] \left[(cp_{5} cp_{5} cp_{5} sp_{5} cp_{5} sp_{5}) \right] \left[(cp_{5} sp_{5} cp_{5} sp_{5} cp_{5} sp_{5}) \right] \left[(cp_{5} sp_{5} cp_{5} sp_{5} cp_{5} sp_{5}) \right] \left[(cp_{5} sp_{5} sp_{5} cp_{5} cp_{5} sp_{5} cp_{5} sp_{5} cp_{5} sp_{5} cp_{5} cp_{5} sp_{5} cp_{5} sp_{5} cp_{5} sp_{5} cp_{5} sp_{5} cp_{5} sp_{5} cp_{5} sp_{5} sp_{5} cp_$$

$$D_{9} = L_{5} C_{9} \text{ som}_{8} \left[(c_{1}^{0} c_{2}^{0} s_{1}^{0} + s_{1}^{0} c_{1}^{0})(s_{2}^{0} s_{2}^{0} + c_{2}^{0} c_{2}^{0}) + (c_{1}^{0} c_{1}^{0} - s_{2}^{0} c_{2}^{0} s_{2}^{0})(s_{2}^{0} - s_{2}^{0} c_{2}^{0} s_{2}^{0})(s_{2}^{0} - s_{2}^{0} c_{2}^{0} s_{2}^{0})(s_{2}^{0} - s_{2}^{0} c_{2}^{0} s_{2}^{0})(s_{2}^{0} - s_{2}^{0} s_{2}^{0} - s_{2}^{0})(c_{2}^{0} c_{3}^{0} - s_{3}^{0} s_{2}^{0} - s_{4}^{0} s_{4}^{0} s_{4}^{0} - s_{4}^{0} s_{4}^{0} - s_{4}^{0} s_{4}^{0} -$$

$$D_{10\ 10} = 5^{2} \Theta_{3} \left[\left(I_{X_{3}} c^{2} \psi_{3}^{*} + I_{Y_{3}} 5^{2} \psi_{3}^{*} \right) + C_{18} \right] + I_{\overline{e}_{3}} c^{2} \Theta_{3}$$

$$C_{\Theta_{3}}^{e}$$

$$D_{10\ 11} = \left(I_{Y_{3}} - I_{X_{3}} \right) + S_{\Theta_{3}} + S_{\Theta_{3}} c_{\Theta_{3}} c_{\Theta$$

 $D_{11 11} = C_{18} + I_{X_3} = 2\psi_3 + I_{Y_3} C^2 \psi_3$

$$\mathcal{O}_{12} \, I_{2} = \mathcal{I}_{23}$$

$$\begin{array}{l} D_{13\ 27} = -\left[s\phi_{10} s\phi_{10} C_{11} + C_{13} \left(s\phi_{10} c\phi_{10} c\psi_{10} s\alpha_{12} \right. \right. \right. \\ & + c\phi_{10} s\psi_{10} s\alpha_{12} + s\phi_{10} s\phi_{10} c\alpha_{12} \right] \right] \\ & \left[\left[L_{4} s\phi_{4} s\phi_{4} - L_{41} \left(s\phi_{4} c\phi_{4} s\psi_{4} - c\phi_{4} c\psi_{4} \right) \right] \right. \\ & - \left[c\phi_{10} s\phi_{10} C_{11} + C_{13} \left(c\phi_{10} c\phi_{0} c\psi_{10} s\alpha_{12} \right. \right. \\ & - s\phi_{10} s\psi_{10} s\alpha_{12} + c\phi_{10} s\phi_{10} c\alpha_{12} \right] \right] \left[L_{4} c\phi_{4} \\ & s\phi_{4} - L_{41} \left(c\phi_{4} c\phi_{4} s\psi_{4} + s\phi_{4} c\psi_{4} \right) \right] \right] \\ D_{13\ 28\ 26\ (c\phi_{10} c\phi_{10} - c\phi_{13} s\phi_{10}) + L_{41} \left[s\phi_{10} c\phi_{4} \right. \\ & c\phi_{4} s\psi_{4} + s\phi_{4} c\psi_{4} \right] - c\phi_{10} \left(s\phi_{4} c\phi_{4} s\phi_{4} - c\phi_{4} \right. \\ & c\phi_{4} s\psi_{4} + s\phi_{4} c\psi_{4} \right) - c\phi_{10} \left(s\phi_{4} c\phi_{4} s\phi_{4} - c\phi_{4} c\psi_{4} \right) \right] \\ D_{13\ 29\ 27\ - C_{13\ 5} s\alpha_{12\ 2}\ \left\{ \left(c\phi_{10} c\phi_{10} - s\phi_{10} s\psi_{10} + s\phi_{10} c\psi_{10} \right) \right. \\ & \left[L_{4} s\phi_{4} s\phi_{4} - L_{44} \left(s\phi_{4} c\phi_{4} s\psi_{4} - c\phi_{4} c\psi_{4} \right) \right] \right] \\ & + \left(c\phi_{10\ } c\psi_{10} - s\phi_{10\ } c\phi_{10\ } s\psi_{10\ } s\psi_{10\ } c\phi_{11\ } - s\phi_{9\ } s\psi_{4} c\phi_{11} \right] \\ D_{13\ 30\ 27\ - C\phi_{4\ } c\phi_{4\ } s\psi_{4\ } + s\phi_{4\ } c\psi_{4\ } \right) \left[L_{4\ } c\phi_{4\ } s\phi_{4\ } - L_{44} \left(c\phi_{4\ } c\phi_{4\ } s\phi_{4\ } - c\phi_{4\ } c\psi_{4\ } \right) \right] \\ & \left[L_{4\ } s\phi_{4\ } s\phi_{4\ } - L_{44} \left(s\phi_{4\ } c\phi_{4\ } s\phi_{4\ } - c\phi_{4\ } c\psi_{4\ } \right) \right] \right]^{2} \\ D_{13\ 30\ =\ C_{12\ } \left\{ \left(c\phi_{9\ } c\phi_{9\ } c\psi_{4\ } c\phi_{11} - s\phi_{9\ } s\psi_{4\ } c\phi_{11} \right) \right] \\ & \left[c\phi_{9\ } c\phi_{9\ } s\psi_{4\ } c\phi_{11} - s\phi_{9\ } s\psi_{4\ } c\phi_{11} \right]^{2} \\ D_{13\ 30\ =\ C_{12\ } \left\{ \left(c\phi_{9\ } c\phi_{9\ } s\phi_{4\ } s\phi_{4\ } c\phi_{4\ } \right) \right]^{2} \\ D_{13\ 30\ =\ C_{12\ } \left\{ \left(c\phi_{9\ } c\phi_{9\ } s\phi_{9\ } s\phi_{11} - s\phi_{9\ } s\psi_{9\ } c\phi_{11} \right) \right] \\ & \left[c\phi_{9\ } c\phi_{9\ } s\phi_{9\ } s\phi_{11} - (s\phi_{9\ } c\phi_{9\ } s\phi_{11} - c\phi_{9\ } s\phi_{9\ } s\phi_{11} - c\phi_{9\ } s\phi_{11} - c\phi_{11} - s\phi_{9\ } s\phi_{11} - c\phi_{11} - s\phi_{11} - s\phi_{11} - s\phi_{11} - s\phi_{11} - s\phi_{11} + c\phi_{11} - s\phi_{11} - s\phi_{11} + c\phi_{11} - s\phi_{11} - s\phi_{11} - s\phi_{11} + c\phi_{11} + c\phi_{11} - s\phi_{$$

$$D_{13 \ 31} = C_{13} \begin{cases} (c\phi_{10}c\phi_{10}c\phi_{10}c\phi_{12} - s\phi_{10}s\phi_{10}c\phi_{12} \\ -c\phi_{10}s\phi_{10}s\phi_{10}s\phi_{12}) [L_{4}s\phi_{4}s\phi_{4} - L_{4}(s\phi_{4}c\phi_{4} \\ s\phi_{4} - c\phi_{4}c\phi_{4})] - (s\phi_{10}c\phi_{10}c\phi_{10}c\phi_{12} + c\phi_{4}c\phi_{4})] - (s\phi_{10}s\phi_{10}c\phi_{10}c\phi_{12} + c\phi_{4}c\phi_{4}) \\ c\phi_{10}s\phi_{10}c\phi_{12} - s\phi_{10}s\phi_{10}s\phi_{12}) [L_{4}c\phi_{4} \\ s\phi_{4} - L_{4}(c\phi_{4}c\phi_{4}s\phi_{4} + s\phi_{4}c\phi_{4})] \end{cases}$$

$$D_{14 14} = C_{19} + (C_{25} + T_{X_4}) = {}^{2} H_{4} + T_{X_4} c^{2} H_{4}$$

$$D_{14 15} = L_4 C_{15} c_{44}$$

$$D_{4 24} = [L_4 c_{94} - L_4 s_{94} s_{4}] [C_{10} s_{94} (s_{4} - s_{4} c_{4} q) + C_{12} [c_{44}(s_{4} c_{9} - c_{4} s_{9} - s_{4} c_{4} q) + C_{12} [c_{44}(s_{4} c_{9} - c_{4} s_{9} - s_{4} c_{4} q) + c_{12} [c_{44}(s_{4} c_{9} - c_{9} c_{4} s_{9} - s_{9} c_{11} - s_{9} s_{9} (c_{4} - s_{9} c_{11} - s_{9} s_{9} s_{9} c_{11} + c_{9} s_{9} c_{11} + s_{9} c_{9} c_{9} s_{11} + s_{9} s_{9} c_{11} + s_{9} c_{9} s_{11} + s_{9} c_{9} s_{11} + s_{9} c_{9} s_{11} + s_{9} s_{9} c_{11} + s_{9} s_{9} c_{11} + s_{9} c_{9} s_{11} + s_{9} s_{9} c_{11} + s_{9} c_{9} s_{11} + s_{9} c_{9} c_{9} s_{11} + s_{9} s_{9} c_{9} s_{11} + s_{9} s_{9} c_{11} + s_{9} s_{9} s_{11} + s_{11}$$

$$\begin{array}{l} D_{14\ 27} = \left\{ \begin{array}{l} C_{11} \, se_{10} \left[\, s\phi_{10} \, c\phi_{1} \, - \, c\phi_{10} \, s\phi_{1} \, \right]^{+} \, C_{13} \left[\, c\phi_{1} \\ \left(\, s\phi_{10} \, c\phi_{10} \, c\phi_{10} \, s\sigma_{12} + \, c\phi_{10} \, s\phi_{10} \, s\sigma_{12} + \, s\phi_{10} \\ s\phi_{10} \, c\sigma_{12} \, \right)^{-} \, s\phi_{1} \left(c\phi_{10} \, c\phi_{10} \, c\phi_{12} - \, s\phi_{10} \\ s\phi_{10} \, s\sigma_{12} \, + \, c\phi_{10} \, s\phi_{10} \, c\sigma_{12} \, \right) \right] \right\} \left\{ \begin{array}{l} L_{q} \, ce_{q} \, + \, L_{H} \\ s\phi_{q} \, s\psi_{q} \, \right\} \\ D_{14\ 28\ = \, - \, (L_{q} \, c\phi_{q} + \, L_{H} \, s\phi_{q} \, s\phi_{q} \,) (c\phi_{10} \, c\phi_{1} \, + \, s\phi_{10} \\ s\phi_{q} \,) \left[C_{11} \, c\phi_{10} \, + \, C_{13} \, (c\phi_{10} \, c\phi_{12} - \, s\phi_{10} \, c\psi_{10} \\ s\sigma_{12} \,) \right]^{-} \left(L_{q} \, s\phi_{q} \, - \, L_{H} \, c\phi_{q} \, s\psi_{q} \,) \left[C_{11} \, s\phi_{10} \\ + \, C_{13} \, (c\phi_{10} \, c\psi_{10} \, s\sigma_{12} + \, s\phi_{0} \, c\phi_{12} \,) \right] \\ D_{14\ 29\ = \, C_{13} \, s\sigma_{12} \, \sum \left[L_{q} \, c\phi_{q} \, + \, L_{H} \, s\phi_{q} \, s\psi_{q} \,] \left[c\phi_{q} \\ \left(c\phi_{10} \, c\phi_{10} \, s\psi_{10} \, \right) - \, s\phi_{4} \, (c\phi_{10} \, c\psi_{10} \\ - \, s\phi_{10} \, c\phi_{10} \, s\psi_{10} \,) \right] + \, s\phi_{10} \, c\psi_{10} \, c\phi_{12} \, s\psi_{10} \\ - \, L_{H} \, c\phi_{q} \, s\psi_{10} \,) \right] \\ D_{14\ 30\ = \, C_{12} \, \left[\left(c\phi_{q} \, (c\phi_{q} \, c\phi_{q} \, c\psi_{1} \, c\phi_{1} \, - \, s\phi_{1} \, s\psi_{1} \\ c\phi_{11} \, - \, c\phi_{q} \, s\phi_{11} \,) + \, s\phi_{4} \, (s\phi_{1} \, c\phi_{2} \, c\psi_{1} \\ c\phi_{11} \, - \, c\phi_{q} \, s\psi_{1} \,) \right] \\ \left[- L_{q} \, c\phi_{1} \, + \, L_{H} \, s\phi_{2} \, s\psi_{1} \, \right] - \left[\left(s\phi_{2} \, c\psi_{2} \\ c\phi_{11} \, + \, c\phi_{2} \, s\psi_{1} \,) \right] \right] \\ \left[- L_{q} \, c\phi_{1} \, + \, L_{H} \, s\phi_{2} \, s\psi_{1} \,] - \left[\left(s\phi_{2} \, c\psi_{2} \\ c\phi_{11} \, + \, c\phi_{2} \, s\psi_{1} \,) \right] \right] \\ \left[- L_{q} \, c\phi_{2} \, + \, L_{H} \, s\phi_{2} \, s\psi_{1} \, s\psi_{1}$$

Dia 31 = C13 { [Cda (Cd, Co, CH, Ca12 - 50, 54, Ca12 - cp, 50, 50,)+ 5\$4 (5\$, C+, CH, CA,2 + cop 5 4/0 co1, 2 - 3 \$ 50,0 50,0 50,2)][- L4C04 - CH 30 544]- [(50, CH, Care + Co, 50, 2) $(L_{4}50_{4} - L_{H}C0_{4}5\psi_{4})]$

$$\begin{split} D_{15 \ 15} &= C_{25} + I_{24} \\ D_{15 \ 24} &= -L_{H} \left\{ (s \# s \psi_{4} - c \#_{4} c \#_{6} C \psi_{4}) \right[G_{10} s \#_{9} s \#_{9} \\ &+ G_{12} (s \#_{9} c \#_{9} c \#_{9} s \#_{1} + c \#_{9} s \#_{9}) [G_{10} c \#_{9} \\ &- s \#_{9} c \#_{11})] + (s \#_{4} c \#_{4} c \#_{4} s \#_{4}) [G_{10} c \#_{9} \\ &- s \#_{9} c \#_{11})] + (s \#_{4} c \#_{4} c \#_{4} s \#_{4}) [G_{10} c \#_{9} \\ &- s \#_{9} c \#_{11})]] \right\} \\ D_{15 \ 25} &= L_{H} \left\{ [G_{10} c \#_{9} C \#_{9} c \#_{9} s \#_{11} - s \#_{9} s \#_{9} s \#_{11} \\ &+ c \#_{9} s \#_{9} c \#_{11})] \right\} \\ D_{15 \ 25} &= L_{H} \left\{ [G_{10} c \#_{9} + G_{12} (c \#_{9} c \#_{11} - s \#_{9} s \#_{11})] \\ &- [c \#_{9} (s \#_{4} s \#_{4} - c \#_{4} c \#_{4} c \#_{4}) - s \#_{9} (s \#_{4} c \#_{4} \\ &- c \#_{4} s \#_{4})] - s \#_{4} c \#_{4} (c \#_{5} s \#_{11})] \right\} \\ D_{15 \ 2L} &= -L_{H} \left\{ S_{12} s - S_{11} \left[(c \#_{9} c \#_{9} - s \#_{9} + s \#_{9} c \#_{9}) \\ &- (s \#_{4} s \#_{4} - c \#_{4} c \#_{4} c \#_{4} + s \#_{9} c \#_{9}) \\ &- (s \#_{4} s \#_{4} - c \#_{4} c \#_{4} c \#_{4} s \#_{4}) - s \#_{9} c \#_{9} \right\} \\ C_{15 \ 2L} &= -L_{H} \left\{ S_{12} s - c \#_{4} c \#_{4} c \#_{4} s \#_{9} - s \#_{9} \\ &- (s \#_{4} s \#_{4} - c \#_{4} c \#_{4} c \#_{4} s \#_{9}) - s \#_{9} c \#_{9} \right\} \\ S_{14} \ S_{14} \ S_{14} \ C \#_{14} \ S_{14} \$$

$$\begin{split} & \mathcal{P}_{15} \ \ 27 = L_{H} \left\{ C_{II} \left[(3\phi_{IO} s\phi_{D}) (- c\phi_{q} c\phi_{q} c\mu_{q} + s\phi_{q} s\mu_{q}) + (c\phi_{IO} s\phi_{O}) (s\phi_{I} c\phi_{q} c\mu_{q} + c\phi_{q} s\mu_{q}) \right] \\ & + C_{I3} \left[(s\phi_{IO} c\phi_{IO} c\mu_{IO} s\phi_{IZ} + c\phi_{IO} s\mu_{IO} s\phi_{IZ} + s\phi_{IO} s\phi_{IO} c\phi_{IZ}) (-c\phi_{IO} c\phi_{IO} s\phi_{IZ} + c\phi_{IO} s\mu_{IO} s\phi_{IZ} + c\phi_{IO} s\phi_{IO} c\phi_{IZ}) (s\phi_{I} c\phi_{Q} c\mu_{Q} + c\phi_{Q} s\mu_{I}) \right] \right\} \\ & \mathcal{P}_{15} \ \ 28 = L_{41} \left\{ C_{II} \left[c\phi_{IO} c\phi_{IO} c\phi_{IO} (c\phi_{I} c\phi_{Q} c\mu_{I} + c\phi_{I} s\mu_{I}) \right] \right\} \\ & + 3\phi_{IO} c\phi_{IO} (s\phi_{IC} c\phi_{IC} c\phi_{IO} c\phi_{IZ} - s\phi_{IO} s\phi_{IO} s\phi_{IZ} + c\phi_{IO} s\phi_{IO} c\phi_{IO} c\phi_{IO} c\phi_{IC} c\phi_{IC} c\phi_{IO} c\phi_{IZ} - c\phi_{IO} s\phi_{IO} s\phi_{IO} c\phi_{IZ} - c\phi_{IO} s\phi_{IO} c\phi_{IZ} - c\phi_{IO} s\phi_{IO} c\phi_{IZ} - c\phi_{IO} s\phi_{IO} c\phi_{IZ} - c\phi_{IO} s\phi_{IO} c\phi_{IZ} - s\phi_{IO} s\phi_{IO} c\phi_{IZ} - s\phi_{IO} s\phi_{IO} c\phi_{IZ} - s\phi_{IO} s\phi_{IO} c\phi_{IZ} - c\phi_{IO} s\phi_{IO} c\phi_{IZ} - s\phi_{IO} s\phi_{IO} c\phi_{IZ} - c\phi_{IO} s\phi_{IZ} - c\phi_{IO} s\phi_{IO} c\phi_{IZ} - s\phi_{IO} s\phi_{IO} c\phi_{IZ} - s\phi_{IO} s\phi_{IZ} + s\phi_{IO} c\phi_{IZ} - s\phi_{IO} s\phi_{IZ} + c\phi_{IZ} s\phi_{IZ} + c\phi_{IZ} s\phi_{IZ} + c\phi_{IZ} s\phi_{IZ} + c\phi_{IZ} s\phi_{IZ} + s\phi_{IZ} s\phi_{IZ} + s\phi_{IZ} c\phi_{IU} s\phi_{IZ} + s\phi_{IZ} s\phi_{IU} + c\phi_{IZ} s\phi_{IZ} + s\phi_{IZ} s\phi_{IU} s\phi_{IZ} + s\phi_{IZ} s\phi_{IU} s\phi_{IZ} + s\phi_{IZ} s\phi_{IU} s\phi_{IZ} + s\phi_{IZ} s\phi_{IU} s\phi_{IZ} s\phi_{IZ$$

D15 31 = LH C13 [(C\$10 C#10 C412 - S\$10 Sto Ca12 - C\$10 5010 5×12) (C\$4 C\$4 C\$4 - S\$0 5\$4) + (5\$ 0 C+ C C 10 C 912 + C \$ 0 S\$ C 912 - S\$ 50 Saiz) (S\$4 CO4 CH4 + C\$4 SH4) + (SO10 CH10 Ca12 + Co, Sa12) Sou CH4]

016 16 = Stor [Cz1 - 2L5 Co cay + Ix5 C245 + (IY5 + Iy7) 5245] + Iz5 CO5 + C25 [545 507 + $(S \theta_5 (\alpha_7 + (\theta_5 (\psi_5 S \alpha_7)^2) - 2 L_5 C_8 S \theta_6)$ C+5 C+5 Say + Ixy (3+5 C+5 Cay + C+5 Say) + Iz7 (Cos Cay - Sos CH 5a7)2 D16 17 = Sty [C+5 Say [L5 C8 - Cap (C23 + Ixy - IZ7)] + Sto Ct Say (C23 + Ix7 - I27) + Iy5 + Iy7 - Ix5 -Ix7 7} 016 18 = 50 CH3 Say [Cay (C23 + Ix7 - I27) - 668] + Cos [Say (C23 + Ix7 - I27) + I27 + I25] D16 22 = Sto 5 54 [C23 - L5 C8 Cay + I47] Din 17 = (21 + Can[C23 Can - 265 (8] + CH5 [C23 5 an

 $+ I_{Y_5} + I_{Y_7}] + s^2 t_5 [I_{X_7} C_{A_7} + I_{Z_7} s^2 A_7 + I_{X_5}]$ $D_{17} \quad 18 = s t_5 s a_7 [L_5 C_8 - C a_7 (C_{Z_3} + I_{X_7} - I_{Z_7})]$ $D_{17} \quad ZZ = C t_5 [C_{Z_3} - L_5 C_8 c a_7 + I_{Y_7}]$

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D18 18 = ((23 + Ix7 - Iz7) Say + Iz5 + Iz7

 $D_{19} = S^{2} \Theta_{6} \left[C_{22} - Z L_{6} C_{9} C_{8} + C^{2} H_{6} I_{x6} + (I_{4}) + I_{4} S_{8} \right] S^{2} H_{6} - Z L_{6} C_{9} S_{6} C_{6} C_{4} S_{8} S_{8} + C_{24} \left[S^{2} H_{6} S^{2} \alpha_{8} + (C \Theta_{6} C_{4} S^{2} \alpha_{8} + S \Theta_{2}) + C^{2} \Theta_{6} I_{26} + I_{x8} (S \Theta_{6} C_{4} S^{2} \alpha_{8} + S \Theta_{2}) + C^{2} \Theta_{6} I_{26} + I_{x8} (S \Theta_{6} C_{4} S^{2} \alpha_{8} + C \Theta_{6} S^{2} \alpha_{8})^{2} + I_{28} (C \Theta_{6} C_{4} S^{2} - S \Theta_{6} C_{4} S^{2} \alpha_{8})^{2} + I_{28} (C \Theta_{6} C_{4} S^{2} - S \Theta_{6} C_{4} S^{2} \alpha_{8})^{2} + I_{28} (C \Theta_{6} C_{4} S^{2} - S \Theta_{6} C_{4} S^{2} \delta_{8})^{2} + I_{28} (C \Theta_{6} C_{4} S^{2} - S \Theta_{6} C_{4} S^{2} \delta_{8})^{2} + I_{28} (C \Theta_{6} C_{4} S^{2} - S \Theta_{6} C_{4} S^{2} \delta_{8})^{2} + I_{x8} - I_{x8} + S^{2} \alpha_{8} + (C \Theta_{6} S^{2} + I_{x8} - I_{x8} + S^{2} \alpha_{8} + S \Theta_{6} C_{4} S^{2} + I_{x8} - I_{x8} + S^{2} \alpha_{8} + (C \Theta_{6} S^{2} + I_{x8} - I_{x8} - I_{x8} + I_{x8} - I_{x8}) \right] + C \Theta_{6} S^{2} S^{2} \left[L_{6} C_{9} - C^{2} + C \Omega_{8} + (C \Theta_{6} S^{2} + I_{x8} - I_{28}) \right] \right] + C \Theta_{6} S^{2} S \Theta_{8} \left[L_{6} C_{9} - C^{2} + I_{x8} - I_{x8} - I_{x8} - I_{x8} \right] \right] + C \Theta_{6} S^{2} S \Theta_{6} \left[C_{24} + I_{x8} - I_{28} \right] \right] + C \Theta_{6} S^{2} S \Theta_{8} \left[L_{6} C_{9} - C^{2} + G \Theta_{8} + C \Theta_{8} S \Theta_{8} \left[C (C_{24} + I_{x8} - I_{28}) - L_{6} C_{9} \right] \right] + C \Theta_{6} S \Theta_{8} \left[L_{6} C_{9} - C^{2} + C \Theta_{8} + C \Theta_{8} S \Theta_{8} \left[C (C \Theta_{8} (C_{24} + I_{x8} - I_{28}) - L_{6} C_{9} \right] \right] + C \Theta_{6} S \Theta_{8} \left[L_{6} C_{9} - C^{2} + C \Theta_{8} S \Theta_{8} \left[C (C \Theta_{8} (C_{24} + I_{x8} - I_{28}) - L_{6} C_{9} \right] \right] + C \Theta_{6} S \Theta_{8} \left[L_{6} C_{9} - C^{2} + C \Theta_{8} S \Theta_{8} \left[C (C \Theta_{8} (C_{24} + I_{x8} - I_{28}) - L_{6} C_{9} \right] \right] + C \Theta_{8} S \Theta_{8} \left[L_{6} C_{9} + L_{6} C_{9} \right] + C \Theta_{8} S \Theta_{8} \left[L_{6} C_{9} + L_{6} C_{9} \right] + C \Theta_{8} S \Theta_{8} \left[L_{6} C_{9} + L_{6} C_{9} \right] + C \Theta_{8} S \Theta_{8} \left[L_{6} C_{9} + L_{6} C_{9} \right] + C \Theta_{8} S \Theta_{8} \left[L_{6} C_{9} + L_{6} C_{9} \right] + C \Theta_{8} S \Theta_{8} \left[L_{6} C_{9} + L_{6} C_{9} \right] + C \Theta_{8} S \Theta_{8} \left[L_{6} C_{9} + L_{6} C_{9} \right] + C \Theta_{8} S \Theta_{8} \left[L_{6} C_{9} + L_{6} C_{9} \right]$

+C + C + C = S + C =

 $D_{ZO ZO} = C_{ZZ} + C_{AB} \left[C_{Z4} C_{AB} - Z L_{6} C_{9} \right] + C_{6}^{z} \frac{1}{4} \left[I_{Y6} + I_{Y8} + C_{Z4} S_{AB}^{z} \right] + S_{6}^{z} \frac{1}{4} \left[I_{X6} + I_{X8} + S_{AB}^{z} \left(-I_{X8} + I_{Z8} \right) \right] \right]$ $D_{ZO ZI} = S_{6}^{4} S_{AB} \left[C_{AB} \left(-C_{Z4} - I_{XB} + I_{Z8} \right) + L_{6} C_{9} \right]$ $D_{ZO Z3} = C_{6}^{4} \left[C_{Z4} - L_{6} C_{9} C_{AB} + I_{YB} \right]$

DZI ZI = 5 x8 (CZA + Ix8 - IZ8) + IZ6 + IZ8
Dzz zz= Cz3 + Iy7

023 23 = Cz4 + Iy8

 $\begin{array}{l} D_{Z4} \ z_{4} &= s^{2} \vartheta_{4} \left[C_{c6} - z L_{9} C_{l2} \ C \alpha_{ll} + C^{2} \psi_{4} \ I_{x9} + (I_{y9} \\ &+ I_{y11} \right) s^{2} \psi_{9} \right] + C \vartheta_{9} \left[C \vartheta_{9} \ I_{Z9} - z L_{9} \ C_{l2} \ S \vartheta_{9} \\ C \psi_{9} \ s \alpha_{ll} \end{array} \right] + C_{28} \left[s^{2} \psi_{9} \ s \alpha_{ll} + (C \vartheta_{9} \ C \psi_{9} \ s \alpha_{ll} + s \vartheta_{9} \\ C \alpha_{ll} \right)^{2} \right] + I_{x11} \ (s \vartheta_{9} \ C \psi_{9} \ c \alpha_{ll} + C \vartheta_{9} \ s \alpha_{ll} \right)^{2} + \\ I_{Z11} \ \left(C \vartheta_{9} \ c \alpha_{l1} - S \vartheta_{9} \ C \psi_{9} \ s \alpha_{ll} \right)^{2} \\ D_{Z4} \ z_{5} \ = s \psi_{9} \left\{ C \vartheta_{9} \ s \alpha_{ll} \left[L_{9} \ C_{l2} + C \alpha_{ll} \ \left(-C_{28} \ -I_{x11} \ + I_{z11} \right) \\ &+ s \vartheta_{9} \ C \psi_{9} \left[s^{2} \alpha_{l1} \ \left(C z_{8} \ + I_{x11} \ -I_{z11} \right) + I_{y9} \ + I_{y11} \ -I_{x9} \ -I_{x11} \right] \right\} \\ D_{Z4} \ z_{6} \ = s \vartheta_{9} \ C \psi_{9} \ s \alpha_{l1} \left[C \alpha_{l1} \ \left(C z_{8} \ + I_{x11} \ -I_{z11} \right) - L_{9} \ C_{l2} \right] \\ &+ C \vartheta_{9} \left[I_{Z9} \ + I_{Z11} \ + s^{2} \alpha_{l1} \ \left(C z_{8} \ + I_{x11} \ -I_{z11} \right) \right] \\ D_{Z4} \ z_{0} \ = \ s \vartheta_{9} \ S \psi_{9} \left[C z_{8} \ - L_{9} \ C_{l2} \ C \alpha_{l1} \ + I_{y11} \right] \end{array}$

 $D_{25} z_{5} = C_{26} + C\alpha_{II} \left[C_{28} c\alpha_{II} - 2 L_{q} C_{12} \right] + C_{q}^{2} \left(I_{q} q + I_{qII} + C_{28} s_{qII} \right) + s_{q}^{2} \left[I_{xq} + I_{xII} + s_{qII}^{2} (-I_{xII} + I_{zII}) \right]$ $D_{25} z_{6} = s_{q}^{4} s_{qII} \left[L_{q} C_{12} + C\alpha_{II} (-C_{28} - I_{xII} + I_{zII}) \right]$ $D_{25} z_{6} = C_{q}^{4} \left(C_{28} - L_{q} C_{12} c_{qII} + I_{qII} \right)$

Dz6 z6 = 5 an (Cz8 + Ix11 - Iz11) + Iz9 + Iz11

$$\begin{array}{l} D_{27} \ z_{7} = \ S^{2} \varPhi_{10} \left[C_{27} + c^{2} \#_{10} I_{X10} + (I_{Y10} + I_{Y12}) S^{2} \#_{10} \right. \\ \left. - 2 L_{10} C_{13} C_{X12} \right] + (z_{9} \left[S^{2} \#_{10} S^{2} \#_{12} + (c \#_{10} C_{10}) \right] \\ \left. S \alpha_{12} + S \varPhi_{10} C_{12} \right]^{2} \right] + C \varPhi_{10} \left[C \varTheta_{10} I_{Z10} - 2 \right. \\ \left. L_{10} C_{13} S \varPhi_{10} C_{10} S \alpha_{12} \right] + I_{X12} \left(S \varTheta_{10} C_{10} C_{12} \right. \\ \left. + C \varTheta_{10} S \alpha_{12} \right)^{2} + I_{Z12} \left(C \varTheta_{10} C \alpha_{12} - S \varTheta_{10} C_{10} S \alpha_{12} \right)^{2} \\ D_{27} \ z_{8} = S \#_{10} \left[C \varTheta_{10} S \alpha_{12} \left[L_{10} C_{13} + C \alpha_{12} \left(-C_{29} - I_{X12} + I_{Z12} \right) \right] \\ \left. + S \varTheta_{10} C \#_{10} \left[S^{2} \alpha_{12} \left(-I_{Z12} + I_{X12} + C \vartheta \right) - I_{X12} + I_{Y10} + I_{Y10} - I_{Y10} \right] \right] \\ D_{27} \ z_{9} = S \varTheta_{10} C \#_{10} S \alpha_{12} \left[C \alpha_{12} \left(C_{29} + I_{X12} - I_{Z12} \right) - L_{10} C_{13} \right] \\ \left. + C \varTheta_{10} \left[I_{210} + I_{Z12} + S^{2} \alpha_{12} \left(C_{29} - I_{Z12} + I_{X12} \right) \right] \right] \\ D_{27} \ z_{1} = S \varTheta_{10} S \#_{10} \left[C_{29} + I_{Y12} - L_{10} C_{13} C \alpha_{12} \right] \end{array}$$

$$D_{28} \ z_8 = C_{27} + C_{12} \left[C_{29} C_{12} - ZL_{10} C_{13} \right] + C_{10}^2 I_{10} \\ + I_{Y12} + C_{29} S_{12}^2 \right] + S_{10}^2 \left[I_{x,0} + I_{x12} + S_{12} \\ (-I_{x12} + I_{z12}) \right] \\D_{28} \ z_9 = S_{10}^{\mu} S_{12} \left[L_{10} C_{13} + C_{12} (-C_{29} - I_{x12} + I_{z12}) \right] \\D_{28} \ z_1 = C_{10}^{\mu} \left[C_{29} + I_{Y12} - L_{10} C_{13} c_{12} \right]$$

To define the elements of column vector E, these parameters are necessary:

 $A_{\mu}^{2} = - \left(\phi_{\mu} \quad s \in \left(\phi_{\mu}^{2} + \phi_{\mu}^{2} \right) - 2 \phi_{\mu} \phi_{\mu} \quad s \phi_{\mu} \quad c \in \mathcal{A}_{\mu}$ $A_{22}^{\eta} = -s\phi_n s\phi_n \left(\phi_n + \phi_n\right) + 2\phi_n \phi_n c\phi_n c\phi_n$ $A_{22}^n = -\dot{\Theta}_n^2$ (Θ_n

 $\begin{aligned} A_{12}^{n} &= c\phi_n \left[\left(\phi_n^{z} + \phi_n^{z} + \phi_n^{z} \right) c\phi_n s\psi_n + 2\phi_n \psi_n s\phi_n \right. \\ &\quad c\psi_n \left[+ s\phi_n \left(\left(\phi_n^{z} + \psi_n^{z} \right) c\psi_n - 2\phi_n \phi_n s\phi_n s\phi_n^{z} \right) \right. \\ &\quad + 2\phi_n \psi_n \left(s\phi_n c\phi_n c\psi_n + c\phi_n s\psi_n \right) \\ A_{22}^{n} &= \left(\phi_n^{z} + \phi_n^{z} + \psi_n^{z} \right) s\phi_n c\phi_n s\psi_n - \left(\phi_n^{z} + \psi_n^{z} \right) \left(c\phi_n c\psi_n \right) \\ &\quad + 2 \left[\phi_n s\phi_n \left(\psi_n s\phi_n c\psi_n + \phi_n c\phi_n s\psi_n \right) + \left(\phi_n \psi_n \right) \right. \\ &\quad \left(s\phi_n s\psi_n - c\phi_n c\phi_n c\psi_n \right) \right] \\ A_{32}^{n} &= - \left(\phi_n^{z} + \psi_n^{z} \right) s\phi_n s\psi_n + 2 \left(\phi_n \psi_n c\phi_n c\psi_n \right) \end{aligned}$

$$\begin{split} \vec{A}_{13}^{nm} &= -(\vec{\phi}_n^2 + \vec{\theta}_n^2 + \vec{\phi}_n^2 + \vec{a}_m^2)(c\phi_n c\phi_n c\psi_n s\sigma_m) \\ &+ (\vec{\phi}_n^2 + \vec{\psi}_n^2 + \vec{a}_m^2)(s\phi_n s\psi_n s\sigma_m) - (\vec{\phi}_n^2 + \vec{e}_n^2 \\ &+ \vec{a}_m^2)(c\phi_n s\phi_n c\sigma_m) + 2 \frac{2}{2} \phi_n \left[\frac{e}{ch} s\phi_n (s\phi_n c\phi_n s\phi_n \\ c\psi_n s\sigma_m - c\phi_n c\sigma_m) + \psi_n s\sigma_m (s\phi_n c\phi_n s\phi_n \\ &- c\phi_n c\psi_n) - \vec{a}_m (s\phi_n c\phi_n c\psi_n c\sigma_m + c\phi_n s\psi_n \\ c\sigma_m - s\phi_n s\phi_n s\sigma_m) \int - \vec{a}_m \left[\frac{e}{ch} (c\phi_n s\phi_n c\phi_n \\ c\sigma_m + c\phi_n c\phi_n s\sigma_m) \right] (cont. on next page) \end{split}$$

(A, g cont'd.) + Un (cp, con sy, cam + sp, cyn cam)] + on $\psi_n c\phi_n s = s\psi_n s = m_n^2$ $\overline{A}_{23}^{-nm} = -\left[(\phi_n^2 + \phi_n^2 + \psi_n^2 + \sigma_m^2) (s\phi_n c = c\psi_n s = m) \right]$ $+(\dot{\sigma}_{+}^{2}+\dot{\omega}_{+}^{2}+\dot{\sigma}_{m}^{2})(c\phi_{n}s\psi_{n}s\phi_{m})+(\dot{\sigma}_{n}^{2})$ + + + + + + m) (= p = 50 (cm)] + 2 { p [+ Cp] (Con Cam - 50, CU, Sam) - Un Sam (Con con sy + sp cy) + am (con (con cyn (am - son som) - sonsy com)7 - ~m [+ sp (st cy cam + con sam) + y cam (sp, con syn - cp, cyn) 7+ on 4 (3\$ 30 50 54 54m)? $\vec{A}_{32} = (\vec{\Theta}_n^2 + \vec{\psi}_n^2 + \vec{\alpha}_m^2)(\vec{S}_n C \vec{\psi}_n \vec{S}_m) - (\vec{\Theta}_n^2 + \vec{\alpha}_m^2)$ (concorm)+2 } in am (-con cy cam + 30 sam) + 4 54 (& con sam + am son ()?

 $A_{x}^{n} = \hat{\sigma}_{n} C \psi_{n} (\psi_{n} - \phi_{n} C \hat{\sigma}_{n}) + \phi_{n} \psi_{n} S \psi_{n} S \hat{\sigma}_{n}$ $A_{y}^{n} = \hat{\sigma}_{n} S \psi_{n} (\phi_{n} C \hat{\sigma}_{n} - \psi_{n}) + \phi_{n} \psi_{n} C \psi_{n} S \hat{\sigma}_{n}$ $A_{z}^{n} = -\phi_{n} \hat{\sigma}_{n} S \hat{\sigma}_{n}$

Following are the elements of column vector E:

 $E_{1} = -(C_{2}A_{13}^{2} + C_{3}A_{13}^{2} + C_{4}A_{13}^{3} + C_{5}A_{13}^{4} + C_{2}A_{13}^{5}$ + C7 A13 + C8 A13 + C9 A13 + C10 A13 + C11 A13 + C12 A13 + C13 A13 + C14 A12 + C15 A12) $E_{2} = -\left(C_{2}A_{23}' + C_{3}A_{23}^{2} + C_{4}A_{23}^{3} + C_{5}A_{23}^{4} + C_{6}A_{23}^{5}\right)$ + C7 A23 + C8 A23 + C9 A23 + C10 A23 + G1 A23 + C12 A 23 + C13 A 23 + C14 A22 + C15 A22) $E_{3} = -(C_{2}A_{33}^{\prime} + C_{3}A_{33}^{\prime} + C_{4}A_{33}^{\prime} + C_{5}A_{33}^{\prime} + C_{6}A_{33}^{\prime} +$ + C7 A33 + C8 A33 + C9 A33 + C10 A33 + C10 A33 + C1 A33 + C17 A 33 + C12 A 33 + C14 A32 + C15 A32) Eq = 30, { CIL (A13 50, - A23 CO,) + L, [C5 (-A13 $= \phi_1 + A_{23} - (\phi_1) + C_{15} (-A_{12} = \phi_1 + A_{22} - \phi_1)$ $+C_{10}(-A_{13}=\phi, +A_{23}=\phi)+C_{11}(-A_{13}=\phi, +A_{23}=\phi)$ cd,)+ C12 (-A 13 5¢, + A 23 C¢,) + C12 (-A 13 50, + A 25 CO,)]-Ax Ix, CY, + Ay Iy, 54 + Iz, wz, +, - Ix, wx, +, 54, - IX, wx, +, c4, } + co, [A= Iz, + 0; (Ix, Wx, cy, - Iy, Wy, sy)] $E_{5} = C_{\theta_{1}} \sum c_{\theta_{1}$ + GIO AI3 + GII AI3 + GIZ AI3 + GIZ AI3 + CIZ AI3)] + 50, [-C16 A23 + L1 (C5 A23 + C15 A22 + C10 A23 + C11 A23 + C12 A 23 + C13 A 23 +\$ (Ix, wy, 54, -Ix, wx, cy)] - 50,[-C16 A33 + L, (C5 A33 (Cont. on next paye)

(EB cont'd.) $-C_{R}(\bar{A}_{13}^{57}C\phi_{2}+\bar{A}_{23}^{57}S\phi_{2})-C_{9}(\bar{A}_{13}^{68}C\phi_{2}+\bar{A}_{23}^{68})$ 5\$,)7+5\$2[L3(-C6A33+C7A33-C8A33 + (q A33) - CZO A32]+ \$\$ (IY2 WY2 = \$2 - Ix2 Wx2 C42)] + 502 { G17 A33 + L2 [C4 A33 + C14 A32 + C6 A33 + C7 A33 + C8 A33 + C9 A337 + 54 (-L2 [C12 (A13 C\$ + A22 5A)] - C20 [A12 cd, + A22 5 \$]+ Ls[-C2 (A13 cd, + A23 $5\phi_2 + C_7(A_{13} - c\phi_2 + A_{23} - c\phi_2) - C_8(A_{13} - c\phi_2) - C_8(A$ $+ A_{23}^{57} = \phi_2 + C_q (A_{13}^{68} c \phi_2 + A_{23}^{68} = \phi_2)]$ $-\phi_{z}I_{z}\omega_{z} + cy_{z}[A_{y}^{2}I_{y_{z}} - \psi_{z}I_{x_{z}}]$ Wx2] + 542 [Ax Ix2 + 42 Iy2 Wy2] $E_{q} = -(5\phi_{2}5\psi_{2}-c\phi_{2}c\phi_{2}c\psi_{2})[L_{2}C_{14}A_{13}^{2}]$ + Czo Aiz + Ls (Cc Ai3 - C7 Ai3 + C8 Ai3 $-C_{q} \overline{A}_{13}^{-68} + (s\phi, C_{2}C\psi, +C\phi, s\psi)$ [Lz C14 A23 + C20 A22 + Ls (C6 A23 - G7 A23 + C8 A 23 - C9 A 23)] - 502 { C42 [L2 C14 A33 + C20 A32 + L3 (C6 A33 - C7 A33 + C8 A33 - Cy A33)] - \$2(Ix2 Wx2 542 + IY2 $\omega_{Y_2} \subset \psi_2 \big) \big\} + \frac{1}{\sigma_2} \big(\overline{I}_{X_2} \, \omega_{X_2} \, C \, \psi_2 - \overline{I}_{Y_2} \, \omega_{Y_2} \, s \psi_2 \big)$ $E_{10} = 3\Theta_3 \left\{ L_2 \left(4 \left(A_{13}^2 5 \phi_3 - A_{23}^2 C \phi_3 \right) + C_{18} \left(A_{13}^3 5 \phi_3 \right) \right\}$ -A23 Cp3) (Cont. on next page)

(E, contd.) - cy3 (Ax Ix3 + Iy3 Wy3 43) + 543 (Ay Iy3 - 43 Ix3 Wy3) + og Izg Wzg } + cog {AZIZg + og (Ixg Wxg cyg - Iyg Wy 54) } $E_{ij} = -co_3 \left[L_2 C_4 \left(A_{i3}^2 c_{g} + A_{23}^2 5_{g}^4 \right) + C_{i8} \left(A_{i3}^3 c_{g}^4 + A_{23}^3 5_{g}^4 \right) \right]$ + \$\$ (Ix3 Wx3 cy - Iy3 Wy3 543)]+ 303 (C18 A33 - \$3 I23 Wz3 + L2 (4 A33) + sy3 (Ax Ix3 + 43 Iy3 Wy3) + C43 (Ay3 IY3- 4, INg WY3) $E_{12} = A_2^3 I_{23} - \epsilon_3 (I_{Y_3} \omega_{Y_3} = \psi_3 - I_{X_3} \omega_{X_3} - \psi_3) + \varphi_3$ 303 (Ix, Wx, 54+Iy, Wy, c/) E13 = 30 Spa [-4 C5 A13 + C19 A13 - 64 (C15 A12 + C10 A13 + C1, A13 + C12 A13 + C13 A13)]+C0a[L1 C5 A23 - G19 A23 + L4 (C15 A22 + C10 A23 + C11 AZ3 + C12 A 23 + C13 A 23)]-Ax Ixa C4 {+ (5g C= 34 - C & C 4) [C15 (L1 A13 + L 4 A13 - 625 $A_{12}^{4} - L_{H}(C_{10}A_{13}^{9} - C_{11}A_{13}^{10} + C_{12}A_{13}^{-9''} - C_{13}A_{13})]$ -(C\$ a co + 5\$ + 5\$ cy)[C15 (L1 A23 + L4 A23) - C25 A22 - LH (C10 A23 - C11 A23 + C12 A 23 + C13 A 23)] - 4 30 (Ixq Wxq 54 + Iyq Wyq c 4/4) + 0/2 [c 0/2 (Ix w x c y - I x w x s 4/2) + Iz4 $\mathcal{W}_{z_{q}} = \mathcal{S}_{q}^{q} + A_{z}^{q} I_{z_{q}} \subset \mathcal{S}_{q} + A_{y}^{q} I_{y_{q}} = \mathcal{Y}_{q} = \mathcal{S}_{q}^{q}$

Eng = Cog { Cog [L, C5 A13 - C19 A13 + L4 (C15 A12 + C10 A13 + C11 A13 + C12 A 13 + C13 A 13)] + 5\$ [L, C5 A23 - G19 A23 + L4 (G5 A22 + C10 A23 + C11 A23 + C12 A 23 + C13 A 23)7 + 3 44 [C15 (C1 A33 + L4 A33) - C25 A32 - CH (C10 A33 - C11 A33 + C12 A33 - C13 A 33) + Iy wy \$ \$] - \$ \$ Ix Wx 64 } + 504 {-\$4 Izq Wzq + 542 (CA [C15 (L1 A13 + L4 A13) - C25 A12 - CH (C10 A13 - C1 A13 + C12 A13 - C13 A 13)]+ 5 \$ [C15 (C1 A23 + C4 A23) - C25 A22 - 64 (G10 A23 - 611 A29 + 612 A 23 - G12 A 28)]) - [L4 G15 A32 + 616 A33 - C19 A33 + L4 (C10 A33 + C11 A33 + C12 A33 + C13 A 33)] + Ax Ixa 34 + Ay IXa CH + 4 (Ix WX 54 - Ix Wx 64) $E_{15} = (-C\phi_{q}C\phi_{q}C\phi_{q} + 5\phi_{q}S\phi_{q})[C_{15}(L_{1}A_{13} + L_{4}$ A13) - C25 A12 - 64 (C10 A13 - 611 A13 + C12 A 13 - C13 A 13)] - (sp con cy + c\$4 544)[C15 (4, AZ3 + 64 AZ3) - C25 AZZ - Ly (C10 A23 - C11 A23 + C12 A23 - C13 A 23)] [C15 (L1 A'33 (cont. on next page)

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(E15 contd) + La A33) - C25 A32 - LN (C10 A33 - C11 A32 + C12 A33 - C13 A33)]} + Og (Ix4 Wx4 (44 - Iva Wva 544) - AZIZA E16 = (AisLz + AizLs)[C6 5\$5 505 + CB (5\$5 $C \Theta_5 C \psi_5 S \alpha_7 + C \phi_5 S \psi_5 S \alpha_7 + 5 \phi_5$ 505 (a,)] - (A23 Lz + A22 Ls) [(6 COS 505+CB(COS COS CUS 507-505 545 50, + CØS 505 (01) HABL5CR - A13 (23) (- 3\$ 5 C+5 C+5 5 - - C\$ 545 Jan - Jøs Jos Can) + (Az3 L5 Co - Az3 (23) (COS COS C 45 507 - 505 545 507 + Cos 305 Can) + 305[(A13 Cz1 - A13 Ls (A) 505 - (A23 CZI - A23 L5 CB) COS 7 + SOS [+Ax Ix5 C45 - Ay IY5 545] - Az Iz5 COS + Ixs (05 545 - 05 C45 505) (-45 545 505 + 05 CUS COS + IY5 (05545 505 + 05 C45) (- 45 C45 505 - 05 545 COS) + Izs (\$\$ CO5 + \$\$) \$5 305 + Ixy (305 CHS Can + Cos Jan) { Ax Can + Az Jan - ~ 1 (Wx5 Say + W25 Cay) - [47 (305 (Cont. on next page)

(EIL Contd) $(5\psi_s)A_v^s + I_{z_1}(-C\Theta_s C\alpha_1 + S\Theta_s C\psi_s 5\alpha_1)$ $\left[\dot{\alpha}_{1}(\omega_{x_{5}} c \alpha_{1} - \omega_{z_{5}} s \alpha_{1}) + A_{x}^{s} s \alpha_{1}\right]$ + A= Can] - Ix, Wx, [+5 (50, 50, - COS CUS Cay) + 4's SOS SUS Cay + in (505 C45 50, - CO5 Can) 7 - IY, WY, (+5 C+5 5 45 + 45 5+5 C+5) + Iz7 Wz7 $\left[\dot{\theta}_{5}\left(S\theta_{5}\,C\alpha_{7}+C\theta_{5}\,C\phi_{5}\,S\alpha_{7}\right)-\dot{\psi}_{5}\,S\theta_{5}\,S\phi_{5}\right]$ 3x7 + x7 (CO5 5x7 + 505 C45 (x7)) $E_{17} = \left[-L_z \left(A_{13}^z \, c\phi_s + A_{23}^z \, s\phi_s \right) - L_s \left(A_{12}^z \, c\phi_s + A_{22}^z \right) \right]$ 505)][C6 C85 + C81-505 C45 547 + C85 Cay)]+[A33 Lz+A32 Ls][C6 505+C8 $(C\theta_{5}C\Psi_{5}S\alpha_{1}+S\theta_{5}C\alpha_{1})] - [A_{13}S\phi_{5}+A_{2$ 5\$5][Cz, Cos - L5 C8 (-505 C45 50, + Cos Can)]-[A13 C\$5 + A23 5\$5][C23(-505 C45 San + Cos Can) - L5 CB Cos] +[L5C8 A33 - A33 C23][-CO5C45 501-505 Ca,]+[43 Cz, - A33 L5 C8] 505 + Ixs [-4x 54s + (+ 54s - + 645 545 505)(-45 C45 CO-5 - 45 C45)] + IY5 [-A, C45 + (\$5 545 505 + 05 C 45) (05 545 CO5 + 45 545) (Cont. on next page)

(En Cont'd) - Izs \$ 505 (\$ 505 (\$ 5 + 45) - Ix+ (345 Can) {Ax Can + Az Jan - an (Wxs San + Wes (a)} - Iv, Cus Av - Iz, (345 50,) [~, (wxs Can - wzs San) + Ax San + Az Can] + Ix, wx1[-\$\$ (Cos Cus Can - J85 50,) - 45 C45 Ca, + 2, 345 50,] + IV7 WV7 (\$5 COS 5\$5 + 45 545) + IZ7 WZ7 [-Øs (CO3 (4, 5x, + 50, 2x,) - 4, C4, 5a, - à, 545 Ca, J $E_{18} = [C_8 (A_{13}^2 L_2 + A_{12}^2 L_3) - L_5 C_8 A_{12}^5 + A_{12}^{-57}]$ (23] (COS COS 545 50, + 5\$5 C45 50,) +[CB(A)2 L2+ A22 L3)-L5 CBA22 1 An CZ3 7 (5\$ COS 545 50, - C\$ C45 50,)-[Co(A33L2+A32L3)-L5CRA3 + A33 (237 (505 545 507) + (Ix5 - Ivs) (+ 345 - \$5 C45 300) (+ 5 C4+ + 5 545 SOS) - AZ IZS + IX7 507 [Ax Ca, +Az 5~, - ~, (wx5 3~, + w=5 C~,)] -Tz, Ca, [a, (Wxs Ca, -Wzs Ja,) + Ax Say + Az Cay I + (Ix, Wx, Cay (Cont. on next page)

(Ele cont'd) + Iz, Wz, San) (0, 50, 54, +0, C4, + ~,) + Iy, wy 10, 50, C4, - 0, 54; En= 500 {- 500[-C7(12Ais - (sAir) + 6 Cg A13 - Czz A13] + CO6 [-C7(Lz A23 - Ls Azz) + L6 Cg Azz - Czz Azz 7- C46 (-Ax Ix6 + 4' IY6 WY6) + + + IZ6 W26 + 546 (-A, IVE - 46 ING WXG)} - (5\$ COCC46 508 + COG 54 501 + 5 \$ 50 500 Colo) [- (g(L2 A)) -Ls Aiz - L6 Ais) - Czq Ais] + (CO6 CO6 C46 506 - 5\$6 546 508 + C\$6 506 (008) [-Cq (L2 A23 - LS A22 - L6 A23) - C24 A23] + COL DE (IXE WXG CHE - IYG WYG 542) - AZ IZG 7 + IXB (SOG CAG COR + COL Sara)[A" Car + Az Sara - de (Wx6 500 + WZG COR)] - IYB (506 546) AV - IZB (CAG Codg - 5 QG C46 500)[ing (WKG Codg -WZG Sarb) + Ax Sab + Az Cab] - IxB WXB [06 (500 500 - COL CHE COL)+ 42 502 542 Castas (586 C46 508 - C86 Cas)] - IY8 WY8 (06 CO6 546 + 4 506 C46) (Cont. on next page)

(sout. on next page) - the cap take 5 th same] + I yo way + IX8 mx8 [- \$ ((6 + 6 + 6 + 6 - 2 + 5 - 2 + 2 + 5 - 2 + 5 - 2 + 2 - 2 + 2 + 2 - 2 + 2 + 2 -[8x] + 7x 2x8 + 7x 2x8 + 7 5 5x8 - m -- INB CHE LY - IZB SHE SOR LON COR + 4 5 2018 - 018 (m)×6 2018 + (2018)] - 7 1 INC CHE - IX8 SHE COURT OUR Wye 5 42 - Ixe Wxe C42) - Ax Ixe 542 25 5 7 5 7 6 7 7 (8 00 2 0 5 + 8 0 5 2 0 0 7 - (5 7 2 5 - (5 7 2 5 -) 7 - (5 7 2 5 -) 7 -) 7 -) 7 -) S[27 77- (27 77+ 27 27- 27 57) 67] 205 + [EIN +27 - (SV 77+ SV 27-- (20° 62° 2008 - 50° 5008) { 50° 7 50° 7 50° 7 50° 7 50° 7 50° 7 - 622 233 + 66 433 + 6° 727 m2° 3 (== 12 m x (ch?) } - 20 { { (2/ 2 - 2 = (27 27 - 22 TS7) 2) 7 4S + [20 77 + Ene Con [Called and - Called -[(ap)] 30295+8~592) 30+ (3~5365) 201-+ [58 m 58 [Br (28° Gas + 50° Gas 2018) (Fld Contd.)

(Ezo Contd.) [\$6 COL 542 + 46 546 7 + Izz Wie - \$6 (CA6 C4, 5x8+50, Cx8)-4, C4, 5x8 - de S46 Carl Ez, = - 500 { (COG COG 546 + 506 C42)[Co (Ls A12 - Lz A13 + L6 A13) - Cz A13] + (S\$ Co. 54- - Co. C4.)[Cg(LsAzz-LzAzz + L6 Az3) - Cz4 Az3] - 500 346 [Cg (Ls Azz - Lz A33 + LG A33) - Cza A32] & - Az Izl + \$6 500 (Ix6 Wx6 542 + I 46 Wy6 C42) + OG (IX6 WX6 C46 - IY6 WY6 542) + IX8 Sag[Ax Cas + Az Sag - as (Wx6 Sag IWEG COLB)] - IZB COLB [OB (WXG COLB -WZG Sab) + Ax Sab + AZ Cab] + [Ixe Wixe Care + Ize Wize Sar][\$ 502 542 + & C46 + 28] + IY8 WY8 (& SOL C46 - OL 546) Ezz= (COS COS CUS Ca, - 50, 543 Ca, - COS 503 50m) {-Co[L2 Ais +Ls Ai2 -Ls Ai3]-Czz A,3 } + (5\$ Cos Cos Ca, + Cos 543 Ca, - 5\$5 585 50,)[-[8 (12 Az3 + 45 Azz (Cont. on next page)

(Ezz Contd) - L5 Ara) - C23 Ara] - (SOSCWSCart COS 50,7)[-Ce(LzA32+LsA32-LSA33)-C23 A33]- IY, Ay + IX, WX, (-WX5 507 - WES Can) + IE, WE, (WXS Can - WES 54) E23=1COGCOGCUCCUB-506546COB-COG 50 500 B)[Cg(LsAiz - LzAis +LGAis) -Cz4 [3] + 150, CO, C4, CX8 + C4, 54 [dg - 5\$ 50 50 50 8)[[g(Ls Azz - Lz Azz + Lo A23) - Cz4 Az3] - (50% C42 CXB + COG JOUR) [Cq(L3A32 - L2A32 + L6A33) - Cza A33 7 - Ay IYB + IXB WX8 (- WX6 5018 - WZG CORD + IZE WZE (WXG COR - WZG Jars) Ezq=[AisL, + AisLq-AizLy][-Cio Spg SBg - CIZ (5\$ 0 CHg CHg 3 x, + C 0 5 4 3 5 x, 1 + 50g 50g (a,1)]+[Az3L,+Az3L4 - AzzLy] [Cio Chy Sty + Ciz (Chy Cty Cty SX1, - 5 dy 349 5x, + Cog 509 Cx,)] + A13 [Cz6 509 589 + Lq C12 1- 5\$ q CBq C49 5 x11 - CA9 549 5x ... - 5 \$ 9 5 8 9 Ca ...)] - Azz [Czc Ch 589 (Cont. on next page)

(Era Contd.) -La Ciz (Cog Cog Cug Sx 1 - 50g 54g Sx 11 + Cog Stog Can)] + A13 [CzB (Sog Cog Cog) 5~,1+ CQ 549 5~,1+ 5 \$ 509 Ca,1)- Lg Ciz 509 509]-Az3 [Cz8 (C\$ COg C 4 5x,) -509 549 50, + CDg 50g Ca,) - Lg Ciz COg 5897+589 (+A, Ix9 C49 - A, Ix9 549) - Az Izg Cog + Ixg (0, 54g - \$9 (49 50g) (Đg L 4g L 8g - 4g 54g 58g) + Iyg (Đg 54g 50g + 0g CAg) (- 4g CUg 50g - 0g 54g COg) + Izq (Qg Cog + 4q X og 30g) + Ix11 (30g $C \Psi_{q} C \alpha_{\mu} + C \Theta_{q} S \alpha_{\mu}) [A_{\chi}^{q} C \alpha_{\mu} + A_{z}^{q} S \alpha_{\mu}]$ - ~ (Wxg San + Wzg Can)] - Ivn (Stg $5\psi_{q})A_{v}^{q}+I_{z,i}(-C\theta_{q}C\alpha_{i,i}+5\theta_{q}C\psi_{q}S\alpha_{i,i})$ $\left[\mathcal{A}_{II}(\omega_{xg} C \alpha_{II} - \omega_{zg} S \alpha_{II}) + A_{x}^{g} S \alpha_{II} + A_{z}^{g} \right]$ CX,1] + IX,1 Wx,1 [- 04 (509 5x,1 - CO4 C49 (x11) - 4g 50g 54g (x1, - x1) (50g (4g 5x1) - (to (Can)] + Iy , Wy (- to Cto 54 - 49 Sty CHq) + Iz II WEII [tg (Sty Can + Ctg (49 50,1) - 49 509 549 50,1 + 0,1 (CDg $5\alpha_{11} + 5\theta_{q} C \psi_{q} C \alpha_{11})]$

E25=(CQq CQq)[Cio(L, A13+Lq A13-LH A12) -C26 A13 + A13 Lg C12] + C \$9 (- 589 C49 5011 + (& D (Q12) [C12 (L, A13 + L4 A13 - LH A12 + Lg A13) - A13 C28] + Sbg Ctg [(L) AZ3 + LAAZ3 - LAAZZ) CIO - CZ6 AZ3 + Lg C12 A23] + 5\$9 (-589 C49 5x1, + C89 Can)[Ciz (L, Az3 + L& Az3 - L+ Az3 + L9 Az3) - C28 Az3] - 50g [Cio (L, A33 + L& A33 -LN A32)-CZ6 A33 + Lg C12 A33]+ (-CBg C4q 501, - 58q Ca,) [C12 (L, A33 + L4 A33 - LH A32 + Lg A33 - CZB A33] - Ax Ixa 549 - Ay Ing CHg - Ing Wing CHg 109 CBg + 4g) + Iyg Wyg 54g (0g Cog + 4g) - Izg WE9 \$9 509 - IX11 549 CX11 [Ax CX11 + Az 5x11 - x11 (Wxg 5x11 + Wzg Ca11)] - Iy11 C4g 4" - Iz 549 50, [~, [wxg Ca, - wzg 50,) + Ax 5 x11 + Az Cx11] + Ix11 Wx11 [- \$q(Coq C4g Can - 589 3an / - 49 C4g Can tan 549 5 x n] + Ixn Wyn (Dg Cog 549 + 49 54g) + Iz1, Wz1, [- 0g (COg C4g 5x1, + (Cont. on next page)

(Ez+ Contd) 58g (a,)-4g (4g 50, - i), 54g (2,] EZG= 50, 1 & [CIZ (A'13 L, + A'3 L4 - A12 LH + A13 Lg) - A13 CZB] [- C\$ Q C & 5 4 - 5 \$ C42] +[C12 (A72 L1 + A22 L4 - A22 LN + A23 Lg) - Azz [- 5\$ Q C & 549 + C \$ Q C & 3] + 509 549 [Ciz (A33 L, + A32 L4 - A32 LH + A33 Lg) - A33 C28]} + (Ixg - Iyg) (Đg 54g - Ø C4g 50g X Đg C4g + Øg 54g 50g) - Az Izg + Ixn Sxn [Ax Can + Az San - ~ (Wxg 5~,+ Wzg Ca,)] - Iz, Ca, [~, [wxg (~, - wzg 5~,) + 1x 5~, + Az Can] + (Ix, Wx, Can + Izn Wzn 5~,,)(\$\$ 50, 54, + \$\$ C\$ 4 + \$\$ 1) + Iy 11 Wyn (\$9 50g C4 - 0g 54g) Ez= (-5\$10 500) [(A13L, + A13 La + A1, LA) CII - A13 CZ7 + A13 LIO CI3 7+ (-5010 C&10 C410 5012 - CO10 5410 5012 - 5010 50,0 Carz) [C13 (A13 L, + A13 La + A12 LA + A13 LID) - A13 Czg] + (CO10 500) [(AZ3 L, + AZ3 Lq + AZ2 LH) CII - AZ3 CZ7 (Cont. on next page)

(En Contd.) + Azz Lio C13]+(CØ10 CO10 C410 5012 - 50,0 54,0 50,12 + CO,0 50,0 Carz) [(Az3 L, + Az3 L& + Az2 LH + Az3 Lio) C13 - Azz Cza] - Ax Ix10 (- C410 5010)-AV IV10 (5410 500) - AZ IZ10 (000 + IX10 WX10 (-4,0 54,0 50,0 + 0,0 C4,0 C0,0) + Iy,0 Wy10 (-4,0 CHIO SO10- O10 SHID CO10)+ TZID WEID (DIO SOID) + IXIZ (SOID CHIOCAIZ + (0,0 50,17) [Ax Ca12 + Az 50,12 - \$12 (WX10 5012 + WEID CO12)]-IYIZ 50,054,0A,"-Iz,2 (CO10 Care-50,0 C410 5aiz)[aiz (WX10 Caiz - WE10 Sanz) + Ax" Saiz + Az Canz] + Ixiz WX12 [- \$10 (50,0 5x12 - CA10 C410 Carz) - 410 5010 5410 Car, - 217 (50,0 CHIO Saiz - COIO Caiz) - IVIZ WYIZ (to CO10 5410 + 410 SOIOCHID) + IZIZ WZIZ [010 (SOIO CAIZ + COIO CUIO SAIZ) - 410 50,0 54,0 50,12 + 012 (CO10 5012 + 50,0 C4,0 Cara)]

E28 = (c\$10 (0,0) (1, (A13 Li + A15 La + A12 LH) - A13 (27 + A13 LIO (13] + (- C\$10 Stroct 0 SA12 + C\$10 CO10 Caiz) [C13 (A13L1 + A13L4 + A12L4 + A13L10) - A13 (29]+(5\$,0 C=0)[(" (A23L1 + A23L4 + A22LH) - Az3 (27+ Az3 Lio (13 + (- 50,0 50,0 CH,0 59,12 + 5\$,0 C+10 CA12) [C13 (A23 6, + A23 64 + A22 6 H + 423 L10) - A23 (29] + (-50,0) [C11 (A33 L1 + A33 L4 + A32 LH) - A33 C27 + A33 LIO C13]+ $\left(- c \Theta_{10} C H_0 S \alpha_{12} - S \Theta_{10} C \alpha_{12} \right) \left[C_{13} \left(A_{33} L_1 + A_{33} L_4 + A_{33} L_4 + A_{33} L_{10} \right) - A_{33} C_{29} \right] - A_x^{10} I_{x10} S H_0$ - Ay IY10 CHIO + IX10 WX10 (-\$10 CHIO COTO -\$10 CHIO) + IY10 Wy10 (\$,054,0 CO10 + \$10540) + IZ10 W210 (-\$10 50,0) - IXIZ (54,0 CAIZ) [Ax" CAIZ + Az" SAIZ - ~ 12 (WX10 Sale + WZ10 (an)] - IVIZ (40 Ay" - IZIZ Sto SAIZ [~ 12 (WX10 CA12 - WZ10 SA12) + Ax "Saiz + Az "Caiz] + Ixiz Wxiz [-\$10 (CO10 CHIO CAIZ - 50,0 50/2) - 4,0 CHIO CAIZ + x12 54,0 5 912] + IY12 WY12 (\$10 CO10 54,0 + 4,050) + IZIZ WZIZ [- \$10 (CO,0 CH0 50/2 + 500 (0/2) - 40 CHO 50/2 - 212 54,0 Ca12 /

 $E_{29} = (-c\phi_{10} C + c\phi_{10} S + c\phi_{10} S + c\phi_{10} C + c\phi_{10} S + c\phi_{1$ [C13 (A13 L, + A13 La + A12 LH + A13 L10) - A13 (29]+ (-5\$10 C+10 5410 5912 + C\$10 C410 5912) [C13 (A23 L, + A23 L4 + A22 LH + A23 L10) -Az3 C29 + (50,054,25x,2) [C,3 (A33 L) + A33 L4 + A32 LH + A33 LIO) - A33 (29] - AZ Iz10 + Ix10 Wx10 (\$10 C410 + \$10 54,0 500) + Iy10 Wy10 (\$10 CH,0 50,0 - 0,0 54,0) + IX12 5012 [Ax" caz + Az" Saz - 2,2 (WX10 Saiz + WZIOCAIZ)] - IZIZ CAIZ [aiz (WXIO (a12 - WZ10 Sa12) + 4x Sa12 + 42 Ca12] + [IXIZ WXIZ CO12 + IZIZ WZIZ SO12] [\$10 500 54,0+ =10 CH0+ \$ 12]+ IYIE WYIE (\$10 50,0 CH0-0,05Kg) $E_{30} = \left[C_{12} \left(A_{13} L_{1} + A_{13}^{4} L_{4} - A_{12}^{4} L_{1} + A_{13}^{9} L_{q} \right) - \overline{A_{13}}^{-9} \right]$ (28] [C\$ q C to C 4 C 4 - 5 \$ 5 \$ 5 \$ C 4 - C\$ 9 5 to 504,1]+[Cie (AzsL, + AzsL4 - AzzLH + AzsL9) - Az3 Czo][Spo Cog Ctg Cd, + Cpg 5tg Cd, - 5 \$ 5 \$ 5 \$ 5 \$ 5 \$ 1,1] + [Cie (A33 Li + A33 Li - A32 Li) +A33 (q) - A33 (28][-500 CHg Can - Cog San] - Ay Jy11 + Ix11 Wx11 (- Wx9 Sol11 - Wz9 Col11) + IZ11 WZ11 (WX9 (A11 - WZ9 5A11)

E31 = [C13 (A13 L1 + A13 L4 + A12 LH + A13 L10) - A13 C29] [(cφ, c+, c+, c+, -5φ, 54, c+, -cφ, 5+, 59,2)]+[C13(A23L,+A23L4+A22LH+A23L10) - Azs (29 / 5\$10 C + 10 C 4,2 + C \$10 5\$10 C 4,2 - 5 \$10 50,0 50,2]+[C13(A33L1+A33L4+A32 LH + A33 (10) - A33 (29 / - Soro C410 Ca12 - C010 59,27 - Ay IVIZ + IXIZ WXIZ (-WXIO SAIZ - WZIO CONZ)+ IZIZ WZIZ (WXID CONZ - WZID SONZ)

The elements of column vector ${\tt F}_{\rm p}$ are:

$$F_{P1} = 0$$

$$F_{P2} = 0$$

$$F_{P3} = -gC,$$

$$F_{P4} = 0$$

$$F_{P5} = gC_{2}S + \frac{1}{2}$$

$$F_{P6} = 0$$

$$F_{P7} = 0$$

$$F_{P8} = g(C_{3}S + \frac{1}{2} - C_{14}C + \frac{1}{2}S + \frac{1}{2})$$

$$F_{P9} = -gC_{14}S + \frac{1}{2}C + \frac{1}{2}$$

$$F_{P10} = 0$$

$$F_{P10} = 0$$

$$F_{P11} = gC_{4}S + \frac{1}{3}$$

$$F_{P12} = 0$$

$$\begin{split} F_{P13} &= 0 \\ F_{P14} &= g\left(C_{5} = 3e_{4} - C_{15} - Ce_{4} = 54\right) \\ F_{P15} &= -g - C_{15} - 3e_{4} - C4_{4} \\ F_{P16} &= 0 \\ F_{P17} &= g\left[C_{6} = 3e_{5} + C_{8} \left(Ce_{5} - C4_{5} + 3e_{5} - ce_{7}\right)\right] \\ F_{P88} &= -g - G - 3e_{5} = 34_{5} - 3e_{7} \\ F_{P19} &= 0 \\ F_{P20} &= g\left[C_{7} = 3e_{6} + C_{9} \left(Ce_{2} - C4_{6} + 3e_{5} - ce_{8}\right)\right] \\ F_{P21} &= -g - G - 3e_{6} + C_{9} \left(Ce_{2} - C4_{6} + 3e_{7} - ce_{8}\right) \\ F_{P22} &= g - G - 2e_{7} + Ce_{7} - 2e_{7} + 2e_{7} \\ F_{P23} &= g - 2e_{7} - 2e_{7} - 2e_{7} + 2e_{7} - 2e_{7} \\ F_{P23} &= g - 2e_{7} - 2e_{7} - 2e_{7} - 2e_{7} \\ F_{P24} &= 0 \\ F_{P25} &= g\left[C_{10} - 3e_{9} + C_{12} \left(Ce_{9} - 2e_{9} - 3e_{11}\right) \\ F_{P26} &= -2e_{7} - 2e_{7} - 2e_{7} - 2e_{7} \\ F_{P28} &= g\left[C_{11} - 3e_{10} + C_{13} - 2e_{7} - 2e_{7}$$

The constant g represents the acceleration due to gravity.

To define column vector F_s , the following terms must also be defined:

 β_{i} = angle between the upper and middle torso segments β_{i0} = initial value of β_{i} β_{z} = angle between the middle and lower torso segment β_{z0} = initial value of β_{z}

 K_{l} = stiffness of radial spring (in.-1b./rad.) located between upper and middle torso segments

K₂ = stiffness of radial spring (in.-lb./rad.) located between middle and lower torso segments

 $u_{1} = 30, 30_{2}(30, 30_{2} + C0, C0_{2}) + C0, C0_{2}$ B= cos uy -u, = 30, 30, (c, c, c, + 5, 5, 5, + co, co, B, = cos u,

Column vector \mathbf{F}_{s} is composed of the following nonzero elements:

VI-12; 5\$230,7

+ $\frac{K_2(\beta_2 - \beta_{20})}{[-5\phi, 5\phi, c\phi_4 + c\phi]}$ VI-12 50, 5\$\$ 50, 5\$

 $F_{55} = \frac{K_{1}(B_{1}-B_{10}) \left[c\phi_{1} c\phi_{2} c\phi_{2} s\phi_{2} + s\phi_{1} c\phi_{1} \right]}{\sqrt{1-m_{1}^{2}} s\phi_{2} s\phi_{2} - s\phi_{1} c\phi_{2}}$ + K2(B2-B2) [cpco, cd, 50, +50, co, $\overline{V_{1-\mu_{i}}}$ $5\phi_{d} 5\phi_{q} - 5\phi_{i} c\phi_{q}$ $F_{57} = \frac{K_1(B_1 - B_{10}) \left[-c\phi_1 3\phi_1 3\phi_2 3\phi_2 + 5\phi_1 3\phi_1 \right]}{\sqrt{1 - M_1^2} c\phi_2 3\phi_2 7}$ $F_{58} = \frac{K_{1}(B_{1}-B_{10})}{\sqrt{1-M_{1}^{2}}} \left[c\phi_{1} \ s\phi_{2} \ c\phi_{2} \ c\phi_{2} \ c\phi_{2} \ + \ s\phi_{1} \ s\phi_{2} \ c\phi_{2} \ - \ c\phi_{1} \ s\phi_{2} \]$ $F_{3/3} = \frac{K_1(B_2 - B_{20})}{\sqrt{1 - M_2^2}} \left[-c\phi_1 + s\phi_1 + s\phi_4 +$ $F_{314} = \frac{K_2(B_2 - B_{20}) \left[c\phi_1 s\phi_1 c\phi_4 c\phi_4 + s\phi_1 s\phi_1 \\ -\sqrt{1 - 4c_1^2} s\phi_4 c\phi_4 - c\phi_1 s\phi_4 \right]$

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The elements of column vector Q_f are as follows:

 $\varphi_{f_1} = \sum_{\substack{n=1\\n=1\\n=1}}^{1/7} F_{X_n}$ $\varphi_{f_2} = \sum_{\substack{n=1\\n=1\\n=1}}^{1/7} F_{Y_n}$ $\varphi_{f_3} = \Sigma$

 $Q_{4} = L_{1} = \phi_{1} = \phi_{1} = \phi_{1} = F_{X_{2}} + \sum_{n=10}^{17} F_{X_{n}} - L_{1} = C \phi_{1} = C \phi_{1} = F_{Y_{n}}$ + E FYAT 9fx = - L, co, co, [Fx3 + E10 Fx1,]-L, 30, co, [Fy3 + E Fyn7+L, so,[Fes+E Fen] 9F6 = 0 $q_{f_1} = - \left[s \phi_2 C \phi_2 C \psi_2 + C \phi_2 S \psi_2 \right] \left[F_{x_2} F_{x_2} + F_{x_4} F_{x_4} \right]$ + Fx5 1x5]+ [s\$2 c+2 s42 - c\$2 c42][Fx2 ry2 + Fx4 + y4 + Fx5 44 + L5 (- Fx4 + Fx5 -Fx6 +Fx7-Fx8 + Fx9)]-5\$2 [Fx2 (52++=,)+ Fx4+=4+ Fx5+=5+L2(Fx1 + ~ Fxn]] + [co, co, cy, - so, sy] [Fyz +x, + Fyz +x4 + Fys +x5]-[c\$2002 542 + 5\$ CH2 7[FY2 + Y2 + FY4 + F15 H15 + L3 (-Fya + Fys - Fy2 + Fy7 - Fy8 + Fy9)] $+ c \phi_2 5 \phi_2 \left[F_{Y_2} \left(5_2 + F_{z_2} \right) + F_{Y_4} F_{z_4} + F_{Y_5} \right]$ +25 + L2 (Fy + 5 Fy)] $Q_{f_{\theta}} = -C\phi_{z} s \Theta_{z} c \psi_{z} \left[F_{x_{z}} r_{x_{z}} + F_{x_{q}} r_{x_{q}} + F_{x_{5}} r_{x_{5}} \right]$ + cp2 302 542 [Fx2 + y2 + Fx4 + y4 + Fx5 + 45 $(-F_{x_4} + F_{x_5} - F_{x_6} + F_{x_7} - F_{x_8} + F_{x_9})] + c\phi_2 c\phi_2$ [Fx2 (52+ 122)+ Fx4 124 + Fx5 125 + L2 (Fx1 + 2 Fx,)]- 5\$2 502 C42 [Fy2 Kx2 + Fx4 Kx4 + Fy5 (cont. on next page) 1×5]

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(Qse cont'd.) + 50, 50, 54 [Fyz 14, + Fy4 14 + Fy5 1/5 + Ls (- Fy4 + Fy5 - Fy6 + Fy7 - Fy8 + Fy9)] + 5\$ coz [Fyz (5z++=z) + Fyz +=z + Fys +=s + Lz (Fy + E Fyn)]-coz cy [Fz Kz + Fz Yxq + Fzs 1x5]+ Cozsy [Fzz 1/2 + Fzq 1/4 + FZ5 Hy5 + L3 (- FZ4 + FZ5 - FZ6 + FZ7 - FZ8 + Fzq)] - 502 [Fz2 (52 + Fz2) + Fz4 Fz4 + Fas 1/25 + L2 (Fat 2 Fan)7 9fg = - (cp2 co2 sy2 + sp2 cy2)[Fx2 + x2 + Fxq + x4 + Fx5 rx]+ (- c & c + s & s & s &) [Fx + y, + Fx + y_4 + Fx5 + 45 (- Fx4 + Fx5 - Fx6 + Fx7 - Fx8 + Fx9)] + (- 3 & coz sy + cp cy)[Fy, tx, + Fy, txa + Fys tx]- (so, co, cy, + co, su) [Fyz ty, + Fy4 1/4 + Fy5 1/5 + Ls (-Fy4 + Fy5 - Fy6 + Fy7 - Fyg + Fyg)7+ 502 54 [Fzz +x2 + Fzg +xq + F25 4x5]+ 302 C42 [F22 + 42 + F24 444 + F25 rys + Ls (-Fz+ + Fz5 - Fz6 + Fz7 - Fz8 + Fzq)] Qf10 = Fx, [- 1x, (sog cog cy + cog sy) + ry, (5 \$ c = 5 \$ - c \$ c \$] - 5 \$ 3 = 3 (5, + 12,)] +Fy [tx, (cd cog cy - 50 54) - +y, (cdg $C_{\Theta_3} = \psi_3 + = \phi_3 C \psi_3 + c \phi_3 = (S_1 + F_{Z_1})$

qf11 = Fx1 [-+x, cq 303 cy3 + Hy, cq 503 543 + cq3 Co3 (5, + 1=,)] + Fy, [-+x, 50, 503 cy3 + +y, 5\$ 503 5\$ + 5\$ co3 (5, ++=,)] + F=, [-+x, co3 cy ++y, co3 sy2 - 503 (5,++2,)] Qf12 = Fx, [-+x, (cg, cg, sy, +sg, cy)+ry, (-cg, Co3 c 43 + 5 \$ 5 \$ 5 \$] + Fy, [+x, (- 5 \$ € Co3 543 + cb3 c43)-+y, (563 c63 c43+c63543)] + Fz, [1x, 50, 54] + Ky, 50, C4] $= -F_{x_3} F_{x_3} (s\phi_q c\phi_q c\psi_q + c\phi_q s\psi_q) + (s\phi_q$ $\varphi_{f_{13}}$ Coq 544 - Coq C44)[Fx3 Vy3 + LH (-Fx10 + Fx11 - Fx12 + Fx13 - Fx14 + Fx15 - Fx16 + Fx17)] - 5\$\$ 504 [Fx3 (+23 - 53) - LA En Fxm] $+F_{y_3}F_{x_3}(C\phi_4C\phi_4C\psi_4-5\phi_45\psi_4)-(C\phi_4C\phi_4)$ 5 4 + 5 6 4 C/4) [Fy3 + Y3 + LH (- Fy10 + Fy11 -Fy12 + Fy13 - Fy14 + Fy15 - Fy16 + Fy17)]+ Cp 504 [Fy3 (+23-53)-64 = Fym] \$414 = - Fx3 1x3 c\$4 300 c\$4 + c\$4 504 544 [Fx3 143 + LH (-Fx10 + FII - FX12 + Fx13 - Fx14 + Fx15 - FX16 + Fx17] + coq Coq [Fx3 (F=3 - 53) - Lq = 10 Fxn] - Fy 1x3 3 \$ \$ 504 CH4 + 3 \$ 504 304 3 4 [Fy3 My3 + LH (-Fy0 + Fy1 - Fy2 + Fy13 - Fy14 + FYIS - FYIL + FYIT)] (cont. on next paye)

(Pf14 contd.) - Fz +x3 co4 ey + co4 sy4 [Fz +y3 + LH (-F210 + F211 - F212 + F213 - F214 + F215 - F216 + FZID)]- 50 [FZ3 (KZ3-53)- L4 En [FZn] 9915 = - Fx3 +x3 [cp4 co4 344 + 5\$ c4] + (-c\$ Coy cy + 5 \$ 54 54) [Fx3 + 4 (-Fx10 + Fx11 -Fx12 + Fx13 - Fx14 + Fx15 - Fx16 + Fx17)]+Fy3 1×3 (-5\$ con 5\$ + con cy) - (5\$ con -Fy12 + Fy13 - Fy14 + Fy15 - Fy16 + Fy17)]+Fz3 1×3 504 544 + 504 C/4 [F=3 + 43 + L4 (-F=10 + FEII - FEIZ + FEI3 - FEIG + FEIS - FEIL + FEIT)] Qf16 = - Fx7 +x7 (5\$ cos cy + c\$ sy)+ (5\$ cos 545 - C\$ C45) [Fx, My, + Fxq Myg] - 5\$ 505 [Fx, (12, -L,) - Fxq L,]+Fy, 1x, (cd, (05) C45-5\$554)- (C\$ C€5 545+5\$5 C45) [Fy, ty, + Fy, ty,]+ co, 50, [Fy, (tz, - L5)-Fyg L5] + Fxg txg (-sp cos cy Cary - Cog 545 Cary + 305 505 507) - Fxg (1= - 5q) 5 \$ 505 CO5 CH5 507 + C\$ 545 507 + 3\$ 385 Car] (Cont. on next page)

(Qfile cont'd.) + Fyq txq [cds cos cys ca, - 5\$ 5\$ ca, - cds 50, 50, 50,] + Fyg (1=g - 5g)[cds cos cy sa, - 3\$ sy sa, + cg so, ca,] 9717 = - Fxy +xy cds 505 cys + cds 505 sys [Fxy ry, + Fxq ryg]+ cds cos [Fx7 (+z7-Ls)-Fxq L5] - Fy, 1x, 5\$ 30, c/3 + 5\$ 50, 345 [Fy, 1/17 + Fyg 1/9]+ 5\$ cos [Fy7 127 - L5 (FY7 + Fyg)] - Fzy Kxy coscys + cos 545 [Fzy Ky + Fzq hyg]- 305 [Fz7 rz7 - L5 (Fz7 + Fzq)] - Fxy txy (cd 50, cy can + cd co 50) + (1= - 5q)[+ Fxq (-cd 50-cy 507+cb cos can) + Fyg (-5\$ 505 cy san + 5\$ cos can) - Fzq (cos cys san + sos can)]-Fyq txq (5\$ 505 CH CA7 + 3\$ CO5 507) + F29 1x9 (- cos cy can + 50, 50) Qf18 = - (c\$ cos 545 + 5\$ cy) [Fx, rx, + Fxq[rxqch] + (1=q - 5q) say] + (- c\$ c+ c+ c+ s + 5\$ 5\$) [Fx7 ry, + Fxg hyg]+ (-5\$ CG 54 + C\$ C43) {Fy, rx, + Fyg [+xg chy + 5+7 (+zg - 5g)]] - (5\$ (85 cy + chy 345)[Fy+ ty+ + Fy+ ty]+ 303 545 Fz+ tx+ Fz+ [tx+ cay+ (1=q-5q)=47] +505 cy5[F=, Ny+ F=g Ny]

$$\begin{aligned} & \varphi_{f_{1q}} = -F_{x_{c}} F_{x_{c}} (\varphi_{c}^{c} (\varphi_{c}^{c} (\varphi_{c}^{c} + (\varphi_{c}^{$$

9==-(cp co sys + sp cy) / Fx + x + Fx [+xg [+xg cr8 + (128-58) 508] + (- cp co cy + sp sy) [Fx6 1/6+ Fx8 1/8]+ (-5\$ c= 3\$ + c\$ c\$) {Fy6 x6 + Fyg [+xg Cxg + (1=3-58) 508] {-(54 C+ cy + c\$ 54)[Fy +y + Fy +y]+ 3€ 54 {Fzc +x6 + Fzg [+x8 ca8 + (+z8 - 58) 548]} + 30, c.4 [Fz, + y, + Fz, +y8] Qfzz = Fxy +xy [50 - cd cos cy + 50 54) - cd 505 cay] - Fyg Fxg [507 (3\$ 5 cos cy + cos 545)+5\$ 305 Cay]+Fzg +xg [505 C45 5ay - cos cay]+ (tzoy - 39) { Fxg [cay (cos cos cy5-5¢5 545)-c¢5 505 507]+ Fyg [Corg (5¢5 cos cy + cos sys)- sos sos son]-F=q [sos c \$ ca7 - co5 3a7]{ 9523 = Fx8 1x8 [508 (- c\$ c+ c+ 5\$ 5\$)-c\$ 502 Cors] - Fys Txs [Sors (50 cor cy, + co 54)+ 30 50 cong]+ Fz + x8 [50 c4 508 - (06 Cas]+ (128 - 58) & Fx8 [Ca8 (C\$6 Co cy - 30 34) - co 50 308]+ Fyz -FZ8[306 C46 Cag- C06 508]{

Pfeq = - (5φq ceq cψq + cφq 5ψq) [Fx11 + Fx13 1×13]+ (5\$q cog 5\$g - c\$q c\$g)[Fx11 + y11 + Fx13 + Y13 + Fx15 + Y15 + Fx17 + 1717]-5\$\$ 509 [Fx11 (ra11 - 511) + Fx13 (ra13 - Lay) - Lay(Fx15 + Fx17)]+ (cdg cog chg - 50g 54g)[Fy11 rx11 + Fy13 +x13] - (cog cog 54, + 30, cyg)[Fy1, +y1 + Fy13 My3 + Fy15 Ky15 + Fy17 Ky17]+ C\$q 309[Fy11 (ra11-311)+ Fy13 (ra13-6q)-6q (Fy15+Fy17)] + [can (- 5 \$ con cy - c \$ y 5 \$) + 3 \$ y 509 5a1,][Fx15 + x15 + Fx17 +x17]-[5a1, (5\$, 69, 69] Cyg+cpg 34g)+3\$g 50g cor, [[Fx15 (+215-515) + Fx17 (+217-517)]+[Car,1 (cdg cog cyg-sog 54g)- Cog 50g 501 [[Fys tris + Fyi trin] + $[5q_1, (cb_q ce_q cy_q - 5\phi_q 5\psi_q) + c\phi_q 5e_q$ Call][Fyis (Fais - Jis) + Fyin (Fain - Jin)] A= = c \$ = 5 = [- c \$ (Fx11 + x11 + Fx13 + x13) + 5 \$ (Fx11 + Fx13 + Fx15 + Fx15 + Fx17 + F + cog[Fx11 (FE11 - 511) + Fx13 FE13 - Lay(Fx13 +Fx15 + Fx17)]-(509 cyg ca11 + Cog 5011) [Fx15 +x15 + Fx17 +x17]+ (-509 cyg say, +cog Cali) [Fx15 (+215-515) + Fx17 (+217-517)]{ (cont. on next page)

(Qf25 Contd.) + 5 \$ \$ 500 [- C/4 (Fy, Fx, + Fy, Tx, 3) + 5 \$ (Fy1, ry1, + Fy13 ry13 + Fy15 ry15 + Fy17 ry17)] + cog [Fy11 (1211 - 311) + Fy13 + 213 - Lg (Fy13 $+F_{Y_{15}}+F_{Y_{17}})]-(3\Theta_{g}c_{y_{g}}c_{y_{1}}+c_{\Theta_{g}}s_{y_{1}})$ [Fyis txis + Fyin txin]+ (- 300 cyly 501, + 000 Call) [Fyis (FZIS - 515) + Fyin (FZIN - 517)]} $+ c \theta_q [- c \psi_q (F_{z_{11}} + F_{z_{13}} + F_{z_{13}}) + s \psi_q (F_{z_{11}})$ + Y11 + FZ13 + Y13 + FZ15 + Y15 + FZ17 + Y17)]-50 [FZ11 (FZ11 - 511) + FZ13 +Z13 - Lq (FZ13 + FZ15 + FEIT)]+ (-cog cfg call + 50g 5011)[FEIS 1×15 + FZIT +×17] - (Cog cyg 30,1+50g CX11) [FZ15 (FZ15 - 515) + FZ17 (FZ17 - 517)] Pf26 = - (cog cog sig + sog cyg) & Fx11 + x11 + Fx13 + x13 + car, [Fx15 +x15 + Fx17 +x17]+ Sa11 [Fx15 (1215 - S15)+ Fx17 (1=17-517)] + (-C\$q Coq c\$q + 5 \$ 9 5 \$ 9 [Fx, + Y11 + Fx13 + Y13 + Fx15 + Y15 + Fx17 147]+ (-5\$ cog 5\$ +c\$ q cyg) { Fy1, Kx1, + Fy3 Kx13 + Ca1, [Fy15 Kx15 + Fy17 Kx17] + Sor, [Fy, 5 (F215 - 515) + Fy, (F217 - 517)]} - (5\$ q cog cy + cog 3 yg) [Fy, + y, + Fy3 Ky13 + Fy15 Ky15 + Fy17 Ky17 [(cont. on next suge)

(PF26 contd.) + 509 549 SFZII + XII + FZIZ + CAII [FZIS + XIS + FZIT FXIT]+ SOII [FZIS (FZIS - SIS)+ FZIT (FZIT - 517)]] + 30g cyg [FZ11 My1 + FZ13 My3 + FZ15 My15 + FZIT MYTT] 9427= - (5\$, CO10 CH10 + C\$10 540) [Fx10 + X10 + Fx12 Tx12]+ (30 co, 340-co, c40)[Fx, ty0 + Fx12 + Y12 + Fx14 + Y14 + Fx16 + Y16]- (3\$)0 5010)[Fx10 (FZ10-510)+Fx12 FZ12- L10(FX12 + Fx14 + Fx16)]+ (cd, co, c4, - 30, 54) [Fyis 1x10 + Fyiz 1x12] - (C\$10 CO10 5410 + 5\$10 Cybo)[Fy10 ty10 + Fy12 ty12 + Fy14 ty14 + Fy16 1416] + cop, 5010 [Fy10 (1210 - 510) + Fy12 4212 - LIO (FYIZ + FYIX + FYIG)]+[COIN (- 5\$10 co10 c/10 - c \$10 5 410) + 5\$10 5010 30, 2] FX14 1×14 + FX16 1×16]-[3012 (5\$, C+10 c 410 + c \$10 5 €10) + \$ \$10 5010 cd12][Fx14 (+24-514)+ FX16 (+216-516)]+[cx12(c\$10) Coro cho - 50 540) - cho 50, 50, 50, 2 [Fx, 4 1×14 + Fyis +x16]+[5x12 (c\$10 c\$10 c\$10 - 5\$) 5410)+cpo 5010 cd12][Fy14(+214-514) + Fyil (1216-516)]

168
9528 = Cop { 500 [-Cy10 (Fx10 + Fx12 + x12) + 5 Jo (FX10 Ky10 + FX17 Ky17 + FX14 Ky14 + FX16 + Y16)]+ (010 [FX10 (1210 - 510)+FX12 +=12 - LIO (FX12 + FX14 + FX16)]-(50,0C4) Caiz + COIO SAIZ)[FX1A FX1A + FX16 FX14] + (-5 0,0 CU 50,2 + CO10 CO12)[FX14 (+210 - 514) + Fx16 (+216-516)] \$ + 5\$10 \$50,0 [-Cy (Fy, Fx, +Fy, +x,)+ 540 (Fy, tyio + Fyiz tyiz + Fyia tyia + Fyic tyic)] + C=10 [Fy10 (1=10-510) + Fy12 1=12-410 (Fy12 + Fy1+ + Fy16)]- (5010 Cy10 Co12 + CO10 So12) [Fy14 +x14 + Fy16 +x16]+(-50,0 CH0 50,2 + coio coiz)[Fy14 (1214 - 514) + Fy16 (1216 -516)] + c = [- cy (Fzi + xio + Fziz + xiz) + 3Ho (FZIO TYIO + FEIZ MYIZ + FEIA MYIA + FEIG Hy16)7- 30, [FZ10 (+210-510)+ FZ12 +Z12 - LIO (FEIZ + FEIA + FEIL)]+ (- (010 CHIO CAIZ + 50, 50,2)[FZ14 + FZ16 + X16]-(C0,0 CHIO SOLIZ + SEIO COXIZ) [FZIA (+ZIA - 514) + FZ/((FZ/6-5/6)] 9+29 = - (c\$ co co = 4 = 4 co cy) { [Fx10 + x10 + FXIZ FXIZ] (wht. on next page)

(Qfzq cont'd.) + Ca12 [Fx14 + x14 + Fx16 +x16] + Sa12 [Fx14 (1=14-514)+ Fx16 (1=16-516)] + (-c\$10 C=10 C 410 + 5 \$10 5 €10) [Fx10 + y10 + Fx12 + y12 + Fx4 + Y14 + Fx16 + Y16] + (- 5\$,0 C = ,0 54,0 + (\$10 (4)) [Fy10 + x10 + Fy12 + x12] + (a12 [Fy14 +x14 + Fy16 +x16]+ 5x12 [Fy14 (FE14 - 514)+ Fy16 (+216-516)]}-(5\$, co, cy, + c \$ 10 5 410) [F Y10 + Y10 + F Y12 + F Y14 + Y14 + FY16 416]+ 3010 540 2 [FZ10 4x10 + FZ12 +x12]+ CA12 [F214 + X14 + F216 + X16]+ Sa12 [FZ14(FZ14-514)+FZ16(FZ16-516)]]+50, CYO [FZID MINT FZIZ MIZ + FZIA MIA + FZIG tyis] 9530 = (- C\$q C\$q C\$q 5x11 + 5\$q 5\$9 5x11 - C\$q 30g Crill Fx15 + x15 + Fx17 +x17]+ (cbg cog c 4g ca11 - 5 \$g 5 4g ca11 - c \$g 30g 5 a11) [FX15 (+215 - 515) + FX17 (+217 - 517)] - (5\$\$ cog cyg san + c\$ \$ 549 5011 + 5\$\$ 50 g Car,) [Fy 5 1x15 + Fy 1 +x17]+ (50g Cog cyg cr, + chg sig c-1, - shg sog sr,) (cont. on next page)

1.70

(Qfan cont'd.) [Fy1= (F=15 - 515) + Fy17 (F=17 - 517)]+(509 c 4 g Sa, - Cog Ca,)[FZIE V×15 + FZIT V×17] -(30g C/g Ca, + C0g Sa,)[Fzir (+zir - 515) + Fz17 (+217-517)] QF31 = (- c\$ c\$ c\$ sa12 + 50, 54, 54, 54, 54, 54, 2 Cd 30, 30, Carz) [Fxid rxid + Fxile rxile T + (C\$ 10 C\$ 10 C\$ 17 - 5\$ \$ 54 C\$ 12 - C\$ 10 5010 301,2)[FX1a(+211-514)+FX11 (+211-511)] - (50, CO, CU, SA, + CO, SU, SA, +50, 50,0 CA12)[FY14 HX14 + FY14 HX14 7+ (50,0 CO10 CH, CA12 + CØ10 5410 CA12 - 5\$,0 30,0 5x,,)[Fy1a(+21a-51a)+Fy16(+211-516)] + (Sero CHIO SAIZ-COIO CAIZ) FFZIA HX14 + FZII FXII 7- (SOID CHID CAID + COD SAID) [FZIA (FZIA-514)+FZIL (FZIL-516)]

In defining the column vector R the following terms are necessary:

 $H_{1} = T_{12}^{2} T_{10}^{3} + T_{22}^{2} T_{22}^{3} + T_{22}^{2} T_{22}^{3}$ $G_{1} = T_{13}^{2} T_{13}^{3} + T_{13}^{2} T_{13}^{3} + T_{23}^{2} T_{23}^{3} + T_{23}^{2} T_{23}^{3}$ + T32 Tez + T82 T32

H= T2 T6+ T2 T6 + T2 T6 9= Tiz Tiz + Tiz Tia + Tzz Tez + Tzz Tez + T22 T22 + T22 T22 Ha = Tie Tis + Tos Tos + Tas Tas $G_3 = T_{13}^2 + T_{13}^2 + T_{13}^5 + T_{23}^2 + T_{$ + Tax Tax + Taz Taz $H_{q} = T_{12} T_{13}^{2} + T_{22} T_{23}^{2} + T_{32} T_{33}^{2}$ G4 = T13 T13 + T12 T13 + T22 T23 + T22 T22 + T33 T32 + T32 T32 Hy = T' T' + T' T' T' + T' T' T' T' GT = TI3 TIA + TI3 TIA + T23 T23 + T73 TE3 + Taz Taz + Taz Taz $H_8 = T_{13}^4 T_{13}^{10} + T_{23}^4 T_{23}^{10} + T_{33}^4 T_{33}^{10}$ GB = TI2 TI2 + TI2 TI2 + T23 T23 + T23 T23 + T23 T23 + T33 T33 + T33 T33 $G_{9} = T_{13}^{4} T_{13}^{9} + T_{13}^{4} T_{13}^{9} + T_{23}^{4} T_{23}^{9} + T_{23}^{4} T_{23}^{9}$ + T 33 T 33 + T 33 T 33 $T_{12} = C\phi_{1} S \Theta_{1}$ $T_{13}^{n} = -\phi_{n} = \phi_{n} = \phi_{n}$ T12= 30 50 T23 = \$ c\$ 30, to, so, con

1.1.2

T3. = Com T' = - 6, 50,

J_n = viscous damping coefficient of body joint number n
 (in.-1b.-sec.)

Column vector R contains the ensuing elements:

 $R_{i} = 0$ $R_2 = 0$ R3=0 $R_{q} = - \left[\frac{J_{q} G_{q}}{I - (H_{r})^{2}} \right] \left\{ T_{13}^{2} \left(-3\phi, 5\phi_{j} \right) + T_{23}^{2} \left(c\phi, 5\phi_{j} \right) \right\}$ $-\left[\frac{J_{7}}{(H_{7})^{2}}\right]\left\{T_{13}^{4}(-5\phi,5\sigma,)+T_{23}^{4}(c\phi,5\sigma,)\right\}$ $R_{5} = -\left[\frac{J_{4}}{(+++)^{2}}\right] \left\{ T_{13}^{2} (c\phi_{1}c\phi_{1}) + T_{23}^{2} (s\phi_{1}c\phi_{1}) + T_{33}^{2} - \frac{J_{4}}{(-+++)^{2}}\right] \left\{ (-+++)^{2} - \frac{J_{4}}{(-+++)^{2}}\right\}$ $-\left[\frac{J_{7} \, q_{7}}{I-(H_{7})^{2}}\right] \left\{ T_{13}^{4}(c\phi, c\phi,) + T_{23}^{4}(s\phi, c\phi,) + T_{33}^{4}(s\phi,) + T_{33}^{4}(s\phi,) + T_{33}^{4}($

 $R_{i} = 0$ $R_{7} = -\left[\frac{J_{1}}{I(H_{1})^{2}}\right] \left\{ T_{13}^{3} \left(-5\phi_{2} 5\phi_{2}\right) + T_{23}^{3} \left(C\phi_{2} 5\phi_{2}\right) \right\}$

(cont. on next page)

(Ry contd.) $-\left[\frac{J_{2}}{I-(4_{1})}\right] \left\{ T_{13}^{6}(-3\phi_{2},5\phi_{2}) + T_{23}^{6}(c\phi_{2},5\phi_{2}) \right\}$ $-\left[\frac{J_{3}}{(H_{2})^{2}}\right] \left\{ T_{I_{3}}^{5} \left(-s\phi_{2} s\phi_{2}\right) + T_{2_{3}}^{5} \left(c\phi_{2} s\phi_{2}\right) \right\}$ $-\left[\frac{J_{4}}{(H_{4})^{2}}\right]\left\{T_{13}\left(-5\phi_{2}5\phi_{2}\right)+T_{23}\left(c\phi_{2}5\phi_{2}\right)\right\}$ $R_{B} = - \begin{bmatrix} J_{1} G_{1} \\ I - (H_{1})^{2} \end{bmatrix} \begin{cases} T_{13}^{3} (c\phi_{2} c\phi_{2}) + T_{23}^{3} (s\phi_{2} c\phi_{2}) \\ + T_{33}^{3} (-s\phi_{2}) \end{cases}$ $-\left[\frac{J_{2}G_{2}}{I-(H_{2})^{2}}\right] \begin{cases} T_{13}^{6} (c\phi_{2}(\Theta_{2}) + T_{23}^{6}(s\phi_{2}(\Theta_{2})) \\ T_{32}^{6}(-s\phi_{3}) \end{cases}$ $-\left[\frac{J_{3}}{I-(H_{3})^{2}}\right] \begin{cases} T_{13}^{5} (c\phi_{2} c\phi_{2}) + T_{23}^{5} (s\phi_{2} c\phi_{2}) \\ + T_{33}^{5} (-s\phi_{2}) \end{cases}$ $-\left[\frac{J_{4}G_{4}}{I-(H_{4})^{2}}\right] \begin{cases} T_{13}'(c\phi_{z}c\phi_{z}) + T_{23}'(s\phi_{z}c\phi_{z}) \\ + T_{33}'(-s\phi_{z}) \end{cases}$ Rg=0 $R_{10} = -\left[\frac{J_{1}}{(H_{1})^{2}}\right] \left\{ T_{13}^{2} \left(-5\phi_{3} 5\phi_{3}\right) + T_{23}^{2} \left(-\phi_{3} 5\phi_{3}\right) \right\}$
$$\begin{split} \mathcal{R}_{II} &= - \left[\frac{J_{i} \, G_{i}}{I - (\mathcal{H}_{i})^{2}} \right] \left\{ T_{I3}^{2} \left(c \phi_{3} \, c \phi_{3} \right) + T_{23}^{2} \left(5 \phi_{3} \, c \phi_{3} \right) \\ &+ T_{33}^{2} \left(- 5 \phi_{3} \right) \right\} \end{split}$$
 $R_{12} = 0$

 $R_{13} = -\left[\frac{J_{7}}{(1-(H_{*})^{2})}\right] \left\{ T_{13}\left(-5\phi_{4}5\phi_{4}\right) + T_{23}\left(c\phi_{4}5\phi_{4}\right)\right\}$ $-\left[\frac{J_{B}}{I-(H_{B})^{2}}\right] \left\{ T_{13}^{\prime 0}\left(-5\phi_{4}5\phi_{4}\right) + T_{23}^{\prime 0}\left(c\phi_{4}5\phi_{4}\right) \right\}$ $-\left[\frac{J_{9}G_{9}}{I-(H_{0})^{2}}\right]\left\{T_{13}^{9}\left(-s\phi_{4}s\phi_{4}\right)+T_{23}^{9}\left(c\phi_{4}s\phi_{4}\right)\right\}$ $R_{14} = -\left[\frac{J_{7}}{(H_{7})^{2}}\right] \begin{cases} T_{13} (c\phi_{4} c\phi_{4}) + T_{23} (s\phi_{4} c\phi_{4}) \\ + T_{33} (-s\phi_{4}) \end{cases}$ $-\left[\frac{J_{8}}{(H_{8})^{2}}\right] \begin{cases} T_{13}^{\prime 0} (c\phi_{q} c\phi_{q}) + T_{23}^{\prime 0} (s\phi_{q} c\phi_{q}) \\ + T_{33}^{\prime 0} (-s\phi_{q}) \end{cases}$ $-\left[\frac{J_{q}}{(H_{g})^{2}}\right]\left\{T_{J_{3}}^{q}\left(c\phi_{4}c\phi_{4}\right)+T_{z_{3}}^{q}\left(s\phi_{4}c\phi_{4}\right)\right.\\\left.+T_{33}^{q}\left(-5\phi_{4}\right)\right\}$ RIS = C $R_{16} = -\left[\frac{J_3}{J_3}\frac{4_3}{4_3}\right] \left\{ T_{13}^{2} \left(-5\phi_{5} 5\phi_{5}\right) + T_{23}^{2} \left(c\phi_{5} 5\phi_{5}\right) \right\}$ $R_{17} = -\left[\frac{J_3 \, 4_3}{1 - (H_3)^2}\right] \begin{cases} T_{13}^2 (c\phi_5 \, c\phi_5) + T_{23}^2 (s\phi_5 \, c\phi_5) \\ + T_{33}^2 (-s\phi_5) \end{cases}$ $R_{ig} = 0$ $R_{19} = -\left[\frac{J_{2}}{J_{-11}}\right]^{2} \left\{ T_{13}^{2} \left(-5\phi_{1} s_{\Theta_{1}}\right) + T_{23}^{2} \left(c\phi_{1} s_{\Theta_{2}}\right) \right\}$ $R_{20} = - \left[\frac{J_2 \, 4_2}{I - (H_2)^2} \right] \left\{ T_{13} \left(c \phi_c \, c \phi_c \right) + T_{23} \left(s \phi_c \, c \phi_c \right) \right\} \\ + T_{23}^2 \left(- s \phi_c \right) \left\{ T_{23} \left(s \phi_c \, c \phi_c \right) \right\}$

 $R_{z_1} = 0$ Rzz = - Js az R23 = - J6 48 $R_{24} = -\left[\frac{J_{9}}{I-(H_{0})^{2}}\right] \left\{ T_{13}^{4} \left(-5\phi_{q} + 5\phi_{q}\right) + T_{23}^{4} \left(-\phi_{q} + 5\phi_{q}\right) \right\}$ $R_{25} = -\left[\frac{J_{9}}{(-)} \frac{G_{9}}{G_{9}}\right] \left\{ T_{13} \left(c\phi_{9} c\phi_{9} \right) + T_{23} \left(s\phi_{9} c\phi_{9} \right) + T_{33} \left(s\phi_{9} c\phi_{9} \right) + T_{33} \left(-3\phi_{9} \right) \right\}$ $R_{26} = 0$ $R_{27} = -\left[\frac{J_{\theta}}{I-1}\frac{\xi_{\theta}}{\xi_{0}}\right] \left\{ T_{13}^{4} \left(-3\phi_{10} 5\phi_{10}\right) + T_{23}^{4} \left(c\phi_{10} 5\phi_{10}\right) \right\}$ $R_{2g} = -\left[\frac{J_{e}}{I-(H_{g})^{2}}\right] \left\{ T_{13}^{4}(c\phi_{10} c\phi_{10}) + T_{e3}^{4}(s\phi_{10} c\phi_{10}) + T_{33}^{4}(-s\phi_{10}) + T_{33}^{4}(-s\phi_{10}) \right\}$ $R_{zq} = 0$ $R_{30} = -J_{11} \alpha_{11}$ R31 = - J10 912

APPENDIX IV.-DESCRIPTION OF INPUT TO THE COMPUTER PROGRAM

First Card, Format(20A4)

Program

HED

Variable

<u>Col. No.</u>

1-80

alphameric information for the purpose of identification, to be printed at top of each output page

Description

Second Card, Format(20A4)

<u>Col. No.</u>	Program Variable	Description
1-80	HED	a continuation of the first card

Third Card, Format(7F10.0,110)

Program Col. No. Variable Description 1 - 10T1initial time (sec.) 11-20 \mathbf{TF} final time (sec.) 21-30 DT increment of time for integration (sec.) 31-40 DTPRNT print interval (sec.) 41-50 MREAD = 0.0, signifies that vehicle position data will be supplied on cards rather than read from disk, hence twelfth series of

0.0, represents the total
number of consecutive positions
or time stations for which
vehicle position data will be
read from the disk, hence omit

cards must be included

twelfth series of cards (this
number is punched with a
decimal)

= 0.0, program computes passenger's initial conditions based on data from fourth card and tenth series of cards (sixth series may be left out)

= 1.0, initial conditions are to be supplied in sixth series of cards (fourth card and tenth series may be left out)

= 0.0, signifies that data necessary for plotting will be put on disk

= 1.0, signifies that no data will be put on disk, hence no plotting

leave blank

= 3 (for third card)

Fourth Card, Format(7F10.0, I10)

Initial Translational Velocity of Passenger (Not needed if NPOS=1.0 on third card)

<u>Col. No.</u>	Program Variable	Description
1-10	VXVC	velocity of passenger in vehicle- coordinate X direction (in./sec.)
11-20	VYVC	velocity of passenger in vehicle- coordinate Y direction (in./sec.)
21-30	VZVC	velocity of passenger in vehicle- coordinate Z direction (in./sec.)
31-79		leave blank
80		= 4

51-60 NPOS

NDISK

NCARD

•

71-79 80

61-70

Fifth Series of Cards (13 Cards)

Body-Segment Properties

a. First card of fifth series, Format(7F10.0,I10)

<u>Col. No.</u>	Program <u>Variable</u>	Description		
1-10	XLS	length of shoulder or half of shoulder width (inches)		
11-20	XLH	length of hip or half of hip width (inches)		
21-79		leave blank		
80	NCARD	= 5		

b. Next twelve cards of fifth series, Format(12,8X,6F10.0)

<u>Col. No.</u>	Program Variable	Description
1-2	NSEG	body-segment number
3-10		leave blank
11-20	XLE(NSEG)	length of segment (inches)
21-30	RHO (NSEG)	distance from reference end of segment to center of gravity of segment (inches)
31-40	XM(NSEG)	mass of segment (1bsec. ² / in.)
41-50	XIN(NSEG)	I _X (mass moment of inertia about X axis of segment (inlbsec. ²)
51-60	YIN (NSEG)	Ι _Υ
61-70	ZIN(NSEG)	IZ

Note: Part b is repeated for twelve segments.

Sixth Series of Cards (9 Cards)

Initial Conditions on Generalized Coordinates, Generalized Velocities, and Other Pertinent Data (Needed only when NPOS=1.0 on third card)

a. First card of sixth series, Format(7F10.0,I10)

	Program			
<u>Col. No.</u>	Variable	Description		
1–10	BETA4	initial value of the angle between middle and upper back segments (radians)		
11-20	BETA7	initial value of the angle between middle and lower back segments (radians)		
21-30	BLTLEO(1)*	initial length of lap belt if in use (inches)		
31-40	BLTLEO(2)*	initial length of shoulder belt if in use (inches)		
41-79		leave blank		
80	NCARD	= 6		

b. Second card of sixth series, Format(8F10.0)

<u>Col. No.</u>	Program Variable	Description
1-10	Q0(1)	X'1, Y'1, Z'1, respectively, representing the coordinates
11-20	Q0(2)	of the reference point on the articulated body (termi-
21-30	Q0(3)	<pre>nal end of segment No. 1) in space-fixed coordinate sys- tem (inches)</pre>

*Leave blank when belt is not to be used.

31-40 41-50 51-60	QO (4) QO (5) QO (6)		ϕ_1 , θ_1 , ψ_1 , respectively representing the Euler angles of body segment No. 1 (radians)*		
61-70	QO(7)	•	•		¢2
71-80	QO(8)				^θ 2

c. Third card of sixth series, Format(8F10.0)

<u>Col. No.</u>	Program Variable		Description
1-10	QO(9)		Ψ_2
11-20	QO(10)		[¢] 3
21-30	QO(11)		θ ₃
31-40	QO(12)		ψ_3
41-50	QO(13)		^ф 4
51-60	QO(14)		θ4
61-70	QO(15)		Ψ_4
71-80	QO(16)	· · · · · · · · · · · ·	[¢] 5

*For the remainder of this sixth series, $\phi_n,~\theta_n,$ and ψ_n will represent the Euler angles of body segment n in radians.

•	Fourth	card	of	sixth	series,	Format	(8F10.0))
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<u>Col. No.</u>	Program <u>Variable</u>	Description
1-10	QO(17)	θ5
11-20	QO(18)	Ψ5
21-30	QO(19)	^ф б
31-40	QO(20)	θ ₆
41-50	QO(21)	Ψ_{6}
51-60	QO(22)	α ₇ , position angle of seg- ment No. 7 relative to seg ment No. 5 (radians)*
61-70	QO(23)	α8
71-80	QO(24)	¢ġ

e.

Fifth card of sixth series, Format(8F10.0)

<u>Col. No.</u>	Program Variable		Description
1-10	QO(25)		θ ₉
11-20	QO(26)		^ψ 9
21-30	QO(27)		[¢] 10
31-40	QO(28)		^θ 10
41-50	QO(29)		ψ_{10}
51-60	QO(30)		α ₁₁
61-70	QO(31)		^α 12
71-80	blank	•	

*For the remainder of this sixth series, α_{i} will represent the position angle of segment No. i relative to segment No. i-2 in radians.

. Sixth card of sixth series, Format()		Sixth c	card of	sixth	series.	Format(8F10.0)
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<u>Col. No.</u>	Program Variable	Description
1-10	QDO(1)	\dot{x}_{T1}^{\prime} , \dot{y}_{T1}^{\prime} , \dot{z}_{T1}^{\prime} , respectively, representing the components
11-20	QDO(2)	of velocity of the reference point on the articulated
21-30	QDO(3)	body in space-fixed coordi- nates (in./sec.)
31-40	QDO(4)	$\dot{\phi}_1, \dot{\theta}_1, \dot{\psi}_1,$ respectively,
41-50	QDO(5)	representing the Euler angular velocities of body
51-60	QDO(6)	segment No. 1 (rad./sec.)*
61-70	QDO(7)	$\dot{\phi}_2$
71-80	QDO(8)	θ ₂

g. Seventh card of sixth series, Format(8F10.0)

<u>Col. No.</u>	Program Variable	Descriptio	on
1-10	QDO(9)	ψ ₂	• • . •
11-20	QDO(10)	•	
21-30	QDO(11)	θ ₃	
31-40	QDO(12)	• i	
41-50	QDO(13)	¢4	
51-60	QDO(14)	ė ₄	
61-70	QDO(15)	•	
71-80	QDO(16)	¢5	

*For the remainder of this sixth series, $\dot{\phi}_n$, $\dot{\theta}_n$, and $\dot{\psi}_n$ will represent the Euler angular velocities of body segment No. n in radians per second.

h. Eighth card of sixth series, Format(8F10.0)

<u>Col. No.</u>	Program <u>Variable</u>	Description
1-10	QDO(17)	θ ₅
11-20	QDO(18)	ψ ₅
21-30	QDO(19)	•¢6
31-40	QDO(20)	θ ₆
41-50	QDO(21)	¹ ⁄ ₄ 6
51-60	QDO(22)	ά7, representing the angula velocity of segment No. 7 relative to segment No. 5 (rad./sec.)*
61-70	QDO(23)	å
71-80	QDO(24)	•

i. Ninth card of sixth series, Format(8F10.0)

<u>Col. No.</u>	Program Variable	Description
1-10	QDO(25)	ė ₉
11-20	QDO(26)	¥9
21-30	QDO(27)	\$10
31-40	QDO(28)	
41-50	QDO(29)	\$\vec{\psi_{10}}{\vec{1}}
51-60	QDO(30)	ά ₁₁
61-70	QDO(31)	ά ₁₂
71-80		leave blank

*For the remainder of this sixth series, $\dot{\alpha}_i$ will represent the angular velocity of segment No. i relative to segment No. i-2 in radians per second.

Seventh Series of Cards (4 Cards) Radii of Contact Spheres (inches)

a. First card of seventh series, Format(7F10.0,18)

<u>Col. No.</u>	Program Variable	Description
1-79		leave blank
80	NCARD	7

b. Second card of seventh series, Format(8F10.0)

<u>Col. No.</u>	Program <u>Variable</u>	Description
1-10	RS(1)	radius of contact sphere No. 1 (inches)*
11-20	RS (2)	
21-30	RS(3)	
31-40	RS(4)	
41-50	RS(5)	
51-60	RS(6)	
61-70	RS(7)	
71-80	RS (8)	

*For the remainder of the seventh series, RS(N) will represent the radius of contact sphere No. N in inches.

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۰.	Infilu Calu	ŲΤ	Seventin	serres,	rormat	(0110.0)

<u>Col. No.</u>	Program Variable	
1-10	RS(9)	
11-20	RS (10)	
21-30	RS(11)	
31-40	RS(12)	
41-50	RS(13)	
51-60	RS(14)	
61-70	RS(15)	
71-80	RS(16)	

c

d. Fourth card of seventh series, Format(8F10.0)

Col. No.	Program <u>Variable</u>	Description
1-10	RS(17)	
11-80		leave blank

Eighth Series of Cards (7 Cards)

Dimensions and Coordinates for Idealized Passenger Compartment

a. First card of eighth series, Format(7F10.0,18)

	Program	•	
<u>Col. No.</u>	Variable	11	Description
	· · · · ·		half of vehicle compartment
1-10	VCW	:	width (positive number)
			(inches)

Description

11 2 0	VRW	half of vehicle roof width (positive number) (inches)
21-30	SWP	steering wheel position (distance from center line of vehicle to center of wheel) (inches)
31-40	DSW	diameter of steering wheel (inches)
41-79		leave blank
80	NCARD	= 8

b. Second card of eighth series, Format(8F10.0)

<u>Col. No.</u>	Program Variable	Description
1-10	XV(1)	X and Z coordinates, res- pectively, of point No. 1
11-20	ZV(1)	in vehicle-fixed coordinate system (inches)*
21-30	XV(2)	
31-40	ZV(2)	
41-50	XV(3)	
51-60	ZV(3)	
61-70	XV(4)	
71-80	ZV(4)	

*For the remainder of this eighth series, XV(N) and ZV(N) will represent the X and Z coordinates of passenger-compartment point No. 1 in vehicle-fixed coordinates (at c.g. of vehicle) in inches.

c. Third card of eighth series, Format(8F10.0)

<u>Col. No.</u>	Program Variable
1-10	XV(5)
11-20	ZV(5)
21-30	XV(6)
31-40	ZV(6)
41-50	XV(7)
51-60	ZV(7)
61-70	XV(8)
71-80	ZV(8)

d.

Fourth card of eighth series, Format(8F10.0)

Col. No.		Program Variable	Description
1-10		XV(9)	
11-20		ZV(9)	
21-30		XV(10)	
31-40		ZV(10)	
41-50	•	XV(11)	
51-60		ZV(11)	
61-70		XV(12)	
71-80		ZV(12)	

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Description

· · · · · ·					
T.J. C		- -			$\mathbf{E}_{\mathbf{n}} = \mathbf{E}_{\mathbf{n}} + \mathbf{E}_{\mathbf{n}} = \mathbf{E}_{\mathbf{n}} = \mathbf{E}_{\mathbf{n}} + \mathbf{E}_{\mathbf{n}} = $
FIITN.	cara	OI	eignin	series.	Format(SF10.0)
				,	

<u>Col. No.</u>	Program Variable
1-10	XV(13)
11-20	ZV(13)
21-30	XV(14)
31-40	ZV(14)
41-50	XV(15)
51-60	ZV(15)
61-70	XV(16)
71-80	ZV(16)

e.

f. Sixth card of eighth series, Format(8F10.0)

<u>Col. No.</u>	Program Variable	Description
1-10	XV(17)	
11-20	ZV(17)	
21-30	XV(18)	
31-40	ZV(18)	
41-50	XV(19)	
51-60	ZV(19)	
61-70	XV(20)	
71-80	ZV(20)	

Description

g. Seventh card of eighth series, Format(8F10.0)

<u>Col. No.</u>	Program <u>Variable</u>	Description
1-10	XV(21)	
11-20	ZV(21)	
21-30	XV(22)	
31-40	ZV(22)	
41-80		leave blank

Ninth Series of Cards (26 Cards)

Deformation Properties and Friction Coefficients of the Twenty-five Contact Surfaces of the Idealized Passenger Compartment

a. First card of ninth series, Format(7F10.0, I10)

<u>Col. No.</u>	Program Variable	Description
1-79		leave blank
80	NCARD	= 9

b. Second through twenty-sixth cards of ninth series, Format(12,8X,6F10.0)

Col. No. Variable	Description
1–2 J	contact surface number (1-25), punched right justified
3-10	leave blank

		first and smallest of spring coefficients for the dis-
11-20	STIF1(J)	sipative bi-linear spring
		(lbs /in)
		(103.711.7)
		second and largest of spring coefficients for the dis-
21-30	STIF2(J)	sipative bi-linear spring
		of contace surface J (1bs./in.)
		the value of total deforma-
31_40	DETR(T)	tion (that of contact sphere
JT-40	0)	which STIF2(J) comes into effect
		fraction of strain energy to
		be conserved during a col-
41-50	AMU1(J)	contact surface J, when the total deformation is less
		than or equal to DELB(J)
		fraction of strain energy to
e de la composición d		be conserved during a col-
51-60	AMU2(J)	lision between passenger and contact surface J, when the
		total deformation is greater than DELB(J)
	•	
		coefficient of friction for
61-70	COFR(J)	contact surface J, i.e.,
		friction force = $(normal$
		IOTCE) X UUFR(J)
71-80		leave blank
/1.00		TOULD FRUIT

Include one card as described above for each of the 25 contact antion of the dealted vehicle interior. These saids do not follow any particular numbering sequence, but J must be specified on each card.

Tenth Series of Cards (2 Cards)

Passenger Seating Information (see figure) (Not needed if NPOS=1.0 on third card)

a. First card of tenth series, Format(7F10.0,18)

<u>Col. No.</u>	Program Variable	Description
1-10	NOPTSP	<pre>= 1.0, driver position = 2.0, right front = 3.0, left rear = 4.0, right rear</pre>
11-20	GAMA1	γ ₁ - position of middle back segment (degrees)
21-30	GAMA 2	γ ₂ - position of upper back segment (degrees)
31-40	GAMA 3	γ ₃ - position of head seg- ment (degrees)
41-50	GAMA4	γ ₄ - position of lower back segment (degrees)
51-60	GAMA 5	γ5 - position of upper arm segment (degrees)
61-70	ALPHA1	α ₁ - position of lower arm segment (degrees)
71-78		leave blank
79-80	NCARD	= 10

b. Second card of tenth series, Format(8F10.0)

<u>Col. No.</u>	Program Variable	Description
1–10	XHPV	X _{VH} - X coordinate of pas- senger's heels in vehicle- coordinate system (inches)

Z_{VH} - Z coordinate of passenger's heels in vehiclecoordinate system (inches)

leave blank

Eleventh Series of Cards (5 Cards)

Joint Information (Viscous Damping and Elasticity) and Acceleration due to Gravity

a. First card of eleventh series, Format(7F10.0,I10)

ZHPV

<u>Col. No.</u>	Program Variable	Description
1-10	XJNN	viscous damping coefficient of neck joint (lb.sec.)
11-20	AGSN	limiting value of angular travel of neck with refer- ence to upper torso (radians)
21-30	XJSN	amount of viscous damping at AGSN (1bsec.)
31-40	AGNN	value of angular neck travel at which viscous damping begins increasing linearly toward XJSN (radians)
41–50	XJNS	viscous damping coefficient of a shoulder joint (lb sec.)
51-60	AGSS	limiting value of angular travel of either upper arm with reference to upper torso (radians)
61-70	XJSS	amount of viscous damping at AGSS (1bsec.)
71-78		leave blank
79-80	NCARD	= 11

11-20

21-80

•	Second	card	of	eleventh	series,	Format((8F10.0)	

b

<u>Col. No.</u>	Program Variable	Description
1-10	AGNS	value of angular travel of an upper arm at which vis- cous damping begins in- creasing linearly toward XJSS (radians)
11-20	XJNUB	viscous damping coefficient in upper back joint (1b sec.)
21-30	AGSUB	limiting value of relative angular travel between upper and middle torso segments (radians)
31-40	XJSUB	amount of viscous damping at AGSUB (1bsec.)
41-50	AGNUB	value of angular travel in upper back joint at which viscous damping begins to increase linearly toward XJSUB (radians)
51-60	XJNLB	viscous damping coefficient in lower back joint (lb sec.)
61-70	AGSLB	limiting value of relative angular travel between middle and lower torso segments (radians)
71-80	XJSLB	amount of viscous damping at AGSLB (1bsec.)

c. Third card of eleventh series, Format(8F10.0)

	Program	
<u>Col. No.</u>	Variable	Description
1-10	AGNLB	value of angular travel in lower back joint at which viscous damping begins to increase linearly toward XJSLB (radians)
11-20	XJNH	viscous damping coefficient of a hip joint (lbsec.)
21-30	AGSH	limiting value of angular travel of either upper leg with reference to lower torso (radians)
31-40	XJSH	amount of viscous damping at AGSH (1bsec.)
41-50	AGNH	value of angular travel of an upper leg at which vis- cous damping begins in- creasing linearly toward XJSH (radians)
51-60	XJNE	viscous damping coefficient of elbow joint (lbsec.)
61–70	AGSE(1)	limiting value of angular travel of forearm relative to upper arm approaching the closed-elbow position (approx. minus 130 deg.) (radians)
71-80	AGSE(2)	limiting value of angular travel of forearm relative to upper arm approaching the straight-arm position (approx. minus 10 deg.) (radians)

<u>Col. No.</u>	Program Variable	Description
1-10	XJSE	amount of viscous damping at AGSE(1) and/or AGSE(2) (1bsec.)
11-20	AGNE(1)	value of angular elbow travel (approaching closed- elbow position) at which viscous damping begins increasing linearly toward XJSE (radians)
21-30	AGNE (2)	value of angular elbow travel (approaching straight-arm position) at which viscous damping begins increasing linearly toward XJSE (radians)
31-40	XJNK	viscous damping coefficient of knee joint (lbsec.)
41-50	AGSK (1)	limiting value of angular travel of foreleg relative to upper leg approaching the straight-leg position (approx. zero deg. from the positive side) (radians)
51-60	AGSK(2)	limiting value of angular travel of foreleg relative to upper leg approaching the closed-knee position (approx. plus 140 deg.) (radians)
61-70	XJSK	amount of viscous damping at AGSK(1) and/or AG S K(2) (1bsec.)
71-80	AGNK(1)	value of angular knee travel (approaching straight-leg position) at which viscous damping begins increasing linearly toward XJSK (radians)

d. Fourth card of eleventh series, Format(8F10.0)

e. Fifth card of eleventh series, Format(8F10.0)

Col. No.	Program <u>Variable</u>	Description
1-10	AGNK (2)	value of angular knee travel (approaching closed- knee position) at which viscous damping begins increasing linearly toward XJSK (radians)
11-20	SPJNT4	stiffness of radial spring in upper back joint (in1b./rad.)
21-30	SPJNT7	stiffness of radial spring in lower back joint (in1b./rad.)
31-40	GR	acceleration due to gravity if potential energy of position is desired (386.04 in./sec./sec.)
41-80		leave blank

Twelfth Series of Cards (No. of Cards=NVPOS+1)

Vehicle-Position versus Time Data (Included when MREAD=0.0 on third card)

a. First card of twelfth series, Format(7F10.0,I10)

<u>Col. No.</u>	Program Variable	Description
1-10	NVPOS	exactly the number of cards to be included in part b of this series (punch decimal)
11-78		leave blank
79-80	NCARD	= 12

b. Second through (NVPOS+1)th card of twelfth series, Format(8F10.0)

<u>Col. No.</u>	Program Variable	Description
1-10	CALT(J)	time at which the vehicle position is being specified
11-20	CALX(J)	X coordinate of vehicle center of gravity in the space-fixed coordinate system defined by CAL* for vehicle model (inches)
21-30	CALY(J)	Y coordinate of vehicle center of gravity in CAL space-fixed system (inches)
31-40	CALZ(J)	Z coordinate of vehicle center of gravity in CAL space-fixed system (inches)
41-50	CALPH(J)	the angle ϕ of the Euler angle system used by CAL (roll angle) (degrees)
51–60	CALTH (J)	the angle θ of the Euler angle system used by CAL (pitch angle) (degrees)
61-70	CALPS (J)	the angle ψ of the Euler angle system used by CAL (yaw angle) (degrees)
71-80		leave blank

Note: J = 1, NVPOS, hence NVPOS number of cards must be included in

part b.

*Cornell Aeronautical Laboratory (CAL) uses a different spacefixed coordinate system as well as a different Euler system to derive their mathematical vehicle model. This twelfth series corresponds to the CAL system.

Thirteenth Series of Cards (3 Cards)

Lap and Shoulder Belt Information (This series if left out for unrestrained case)

a. First card of thirteenth series, Format(7F10.0, I10)

Col. No.	Program Variable	Description
1-10	NOPBLT	<pre>= 1.0, lap belt only = 2.0, shoulder belt only = 3.0, lap and shoulder belts</pre>
11-20**	P1(1,1)	X coordinate* of lap-belt anchor point No. 1 (inches)
21-30**	P1(1,2)	Y coordinate* of lap-belt anchor point No. 1 (inches)
31-40**	P1(1,3)	Z coordinate* of lap-belt anchor point No. 1 (inches)
41-50**	P2(1,1)	X coordinate* of lap-belt anchor point No. 2 (inches)
51-60**	P2(1,2)	Y coordinate* of lap-belt anchor point No. 2 (inches)
61-70**	P2(1,3)	Z coordinate* of lap-belt anchor point No. 2 (inches)
71-78		leave blank
79-80	NCARD	= 13

*Vehicle-fixed coordinate system **Leave blank if NOPBLT=2.0 b. Second card of thirteenth series, Format(8F10.0)

<u>Col. No.</u>	Program Variable	Description
1–10*+	P1(2,1)	X coordinate* of shoulder- belt anchor point No. 1 (inches)
11-20*+	P1(2,2)	Y coordinate* of shoulder- belt anchor point No. 1 (inches)
21-30*+	P1(2,3)	Z coordinate* of shoulder- belt anchor point No. l (inches)
31-40*+	P2(2,1)	X coordinate* of shoulder- belt anchor point No. 2 (inches)
41-50*+	P2(2,2)	Y coordinate* of shoulder- belt anchor point No. 2 (inches)
51-60*+	P2(2,3)	Z coordinate* of shoulder- belt anchor point No. 2 (inches)
61-70**	SLACK(1)	amount of initial slack (looseness of fit) in lap- belt (inches)
71-80*+	SLACK(2)	amount of initial slack in shoulder-belt (inches)

*Vehicle-fixed coordinate system **Leave blank if NOPBLT=2.0 *+Leave blank if NOPBLT=1.0 c. Third card of thirteenth series, Format(8F10.0)

<u>Col. No.</u>	Program Variable	Description
1-10**	SK1(1)	the first slope of the idealized* force-elongation curve for the lap-belt (lb./in.)
11-20**	SK2(1)	the second slope of the idealized* force-elongation curve for the lap-belt (lb./in.)
21–30**	DELC(1)	the value of lap-belt elon- gation at which the slope of its force-elongation curve* changes from SK1(1) to SK1(2) (inches)
31–40*+	SK1(1)	the first slope of the idealized* force-elongation curve for the shoulder-belt (lb./in.)
41–50*+	SK2(2)	the second slope of the idealized* force-elongation curve for the shoulder-belt (lb./in.)
51-60*+	DELC(2)	the value of shoulder-belt elongation at which the slope of its force-elon- gation curve* changes from SK1(2) to SK2(2) (inches)
61-80		leave blank

*A bi-linear representation **Leave blank if NOPBLT=2.0 *+Leave blank if NOPBLT=1.0 Fourteenth Series of Cards (NOCRD+1 Cards)

Information* Necessary to Extend an Existing Computer Solution without Duplication of Effort (Needed only for extending an existing solution)

a. First card of fourteenth series, Format(7F10.0,110)

<u>Col. No.</u>	Program Variable	Description
1–10	NOCRD	exactly the number of cards to be included in part b of this series (punch decimal)
11-78		leave blank
79-80	NCARD	= 14

b. Second through (NOCRD+1)th card of fourteenth series, Format(212,F6.0)

<u>Col. No.</u>	Program Variable	Description
1–2 J	r o J q (o	efer to last printout sheet f computer solution in uestion for numerical value punch right justified with- ut a decimal)
3-4 K	r o q (o	efer to last printout sheet f computer solution in uestion for numerical value punch right justified with- ut a decimal)

*"Contact Sphere No. J" and "Contact Surface No. K" for which NARRY(J,K)=0, initially. This information is given on last printout sheet of existing computer solution in question.

= 0.0 or leave blank

11-80

5-10

leave blank

Note: A separate card as shown above is required for each NARRY(J,K) to be initialized.

Fifteenth Series of Cards (1 card)

Terminates Reading of Numerical Input

Col. No.	Program Variable	Description	
1-76		leave blank	
77-80	NCARD	= 1000	

Sixteenth Series of Cards (1 card)

X-Y Plot Option

<u>Col. No.</u>	Program Variable	Description
1-4	HED(1)	= STAN, yields X-Y plots of 14 selected variables vs. time
		= NONE, no plots desired
5-80		leave blank

Final Card

<u>Col. No.</u>	Program Variable	Description
1-4	HED(1)	= FINI
5-80		leave blank