

A DISCRETE-ELEMENT METHOD FOR TRANSVERSE VIBRATIONS OF BEAM-COLUMNS  
RESTING ON LINEARLY ELASTIC OR INELASTIC SUPPORTS

by

Jack Hsiao-Chieh Chan  
Hudson Matlock

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of Beam-Columns and Grid-Beam and Slab Systems

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## PREFACE

This study describes a method developed to analyze beam-columns subjected to either static fixed loads or dynamic loads. The supports of the beam-column may be either linearly elastic or nonlinear and inelastic. The method incorporates previously developed discrete-element beam-column techniques.

This work was done under Research Project 3-5-63-56, "Development of Methods for Computer Simulation of Beam-Columns and Grid-Beam and Slab Systems," conducted at the Center for Highway Research, The University of Texas at Austin, as part of the Cooperative Highway Research Program sponsored by the Texas Highway Department and the Department of Transportation Federal Highway Administration.

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Jack Hsiao-Chieh Chan  
Hudson Matlock

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## LIST OF REPORTS

- Report No. 56-1, "A Finite-Element Method of Solution for Linearly Elastic Beam-Columns" by Hudson Matlock and T. Allan Haliburton, presents a solution for beam-columns that is a basic tool in subsequent reports. September 1966.
- Report No. 56-2, "A Computer Program to Analyze Bending of Bent Caps" by Hudson Matlock and Wayne B. Ingram, describes the application of the beam-column solution to the particular problem of bridge bent caps. October 1966.
- Report No. 56-3, "A Finite-Element Method of Solution for Structural Frames" by Hudson Matlock and Berry Ray Grubbs, describes a solution for frames with no sway. May 1967.
- Report No. 56-4, "A Computer Program to Analyze Beam-Columns under Movable Loads" by Hudson Matlock and Thomas P. Taylor, describes the application of the beam-column solution to problems with any configuration of movable non-dynamic loads. June 1968.
- Report No. 56-5, "A Finite-Element Method for Bending Analysis of Layered Structural Systems" by Wayne B. Ingram and Hudson Matlock, describes an alternating-direction iteration method for solving two-dimensional systems of layered grids-over-beams and plates-over-beams. June 1967.
- Report No. 56-6, "Discontinuous Orthotropic Plates and Pavement Slabs" by W. Ronald Hudson and Hudson Matlock, describes an alternating-direction iteration method for solving complex two-dimensional plate and slab problems with emphasis on pavement slabs. May 1966.
- Report No. 56-7, "A Finite-Element Analysis of Structural Frames" by T. Allan Haliburton and Hudson Matlock, describes a method of analysis for rectangular plane frames with three degrees of freedom at each joint. July 1967.
- Report No. 56-8, "A Finite-Element Method for Transverse Vibrations of Beams and Plates" by Harold Salani and Hudson Matlock, describes an implicit procedure for determining the transient and steady-state vibrations of beams and plates, including pavement slabs. June 1968.
- Report No. 56-9, "A Direct Computer Solution for Plates and Pavement Slabs" by C. Fred Stelzer, Jr., and W. Ronald Hudson, describes a direct method for solving complex two-dimensional plate and slab problems. October 1967.
- Report No. 56-10, "A Finite-Element Method of Analysis for Composite Beams" by Thomas P. Taylor and Hudson Matlock, describes a method of analysis for composite beams with any degree of horizontal shear interaction. January 1968.

Report No. 56-11, "A Discrete-Element Solution of Plates and Pavement Slabs Using a Variable-Increment-Length Model" by Charles M. Pearre, III, and W. Ronald Hudson, presents a method for solving freely discontinuous plates and pavement slabs subjected to a variety of loads. April 1969.

Report No. 56-12, "A Discrete-Element Method of Analysis for Combined Bending and Shear Deformations of a Beam" by David F. Tankersley and William P. Dawkins, presents a method of analysis for the combined effects of bending and shear deformations. December 1969.

Report No. 56-13, "A Discrete-Element Method of Multiple-Loading Analysis for Two-Way Bridge Floor Slabs" by John J. Panak and Hudson Matlock, includes a procedure for analysis of two-way bridge floor slabs continuous over many supports. January 1970.

Report No. 56-14, "A Direct Computer Solution for Plane Frames" by William P. Dawkins and John R. Ruser, Jr., presents a direct method of solution for the computer analysis of plane frame structures. May 1969.

Report No. 56-15, "Experimental Verification of Discrete-Element Solutions for Plates and Slabs" by Sohan L. Agarwal and W. Ronald Hudson, presents a comparison of discrete-element solutions with small-dimension test results for plates and slabs, including some cyclic data. April 1970.

Report No. 56-16, "Experimental Evaluation of Subgrade Modulus and Its Application in Model Slab Studies" by Qaiser S. Siddiqi and W. Ronald Hudson, describes a series of experiments to evaluate layered foundation coefficients of subgrade reaction for use in the discrete-element method. January 1970.

Report No. 56-17, "Dynamic Analysis of Discrete-Element Plates on Nonlinear Foundations" by Allen E. Kelly and Hudson Matlock, presents a numerical method for the dynamic analysis of plates on nonlinear foundations. July 1970.

Report No. 56-18, "A Discrete-Element Analysis for Anisotropic Skew Plates and Grids" by Mahendrakumar R. Vora and Hudson Matlock, describes a tridirectional model and a computer program for the analysis of anisotropic skew plates or slabs with grid-beams. August 1970.

Report No. 56-19, "An Algebraic Equation Solution Process Formulated in Anticipation of Banded Linear Equations" by Frank L. Endres and Hudson Matlock, describes a system of equation-solving routines that may be applied to a wide variety of problems by using them within appropriate programs. January 1971.

Report No. 56-20, "Finite-Element Method of Analysis for Plane Curved Girders" by William P. Dawkins, presents a method of analysis that may be applied to plane-curved highway bridge girders and other structural members composed of straight and curved sections. June 1971.

Report No. 56-21, "Linearly Elastic Analysis of Plane Frames Subjected to Complex Loading Conditions" by Clifford O. Hays and Hudson Matlock, presents a design-oriented computer solution for plane frames structures and trusses that can analyze with a large number of loading conditions. June 1971.

Report No. 56-22, "Analysis of Bending Stiffness Variation at Cracks in Continuous Pavements," by Adnan Abou-Ayyash and W. Ronald Hudson, describes an evaluation of the effect of transverse cracks on the longitudinal bending rigidity of continuously reinforced concrete pavements. April 1972.

Report No. 56-23, "A Nonlinear Analysis of Statically Loaded Plane Frames Using a Discrete Element Model" by Clifford O. Hays and Hudson Matlock, describes a method of analysis which considers support, material, and geometric nonlinearities for plane frames subjected to complex loads and restraints. May 1972.

Report No. 56-24, "A Discrete-Element Method for Transverse Vibrations of Beam-Columns Resting on Linearly Elastic or Inelastic Supports" by Jack Hsiao-Chieh Chan and Hudson Matlock, presents a new approach to predict the hysteretic behavior of inelastic supports in dynamic problems. June 1972.

(P) indicates Preliminary Report.

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## ABSTRACT

A discrete-element method of analysis for transverse vibration of beam columns resting on linearly or nonlinearly-elastic, or nonlinearly-inelastic, supports is presented. The applied forces include static fixed loads and time-dependent dynamic loads. The program is an extension of programs BMCOL 43 (Research Report No. 56-4) and DBCL (Research Report No. 56-8) to cover inelastic supports and time-dependent axial thrusts. Two multi-element models are used to simulate the inelastic characteristics of supports, one which allows the beam to lift off the support when it deflects, upward or downward, and the other which considers the resistance to either upward or downward deflection. An internal damping factor, which is related to the first derivative with respect to time of the curvature of the beam, has been included, in addition to the conventional external viscous damping factor.

The method is based on an implicit difference formulation of the Crank-Nicolson type. A computer program has been written to check the validities of the proposed multi-element models of tracing the loading paths of the nonlinearly-inelastic supports and of the implicit formulation of the Crank-Nicolson type. The results compare well with the theoretical results and with experimental data.

KEY WORDS: dynamic, static, nonlinearly-inelastic supports, multi-element model, beam column, implicit formulation, discrete-element method, computers, piles, bridges, earthquake, wave forces.

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## SUMMARY

A computer program, DBC5, is presented which can efficiently analyze a beam-column resting on linearly or nonlinearly-elastic, or nonlinearly-inelastic supports and subjected to either static fixed loads or dynamic loads.

The path-dependent history of loading of the nonlinearly-inelastic resistance-deflection curve of the support is considered in this study. Two multi-element models, which are used to simulate the nonlinear characteristics of the inelastic resistance-deflection curves, have been introduced. For problems with nonlinearly-elastic or nonlinearly-inelastic supports, the iteration process compares successively computed deflections until a specified tolerance is satisfied. An option available in the program allows a switch from a spring-load-iteration process (adjusting both the stiffness and the load from one iteration to the next, which is known as tangent modulus method) to a load-iteration technique (adjusting only the load) when the supports yield or disconnect from the beam-column. An internal damping factor, which is related to the first time derivative of the curvature of the beam, has been considered in addition to the external viscous damping factor which is normally encountered in a dynamic problem.

The results of an analysis can include

- (1) solutions for the member under static fixed loads,
- (2) solutions for the member under dynamic and static loads at each time station, and
- (3) plots of computed deflections or moments along either the time or the beam axis as required.

Seven example problems typical of those encountered by highway and foundation designers illustrate the uses of the program and the options available. Included are a three-span beam loaded with an AASHO standard 2-D truck moving at a uniform speed of 60 mph, a simply supported steel rod loaded by axial pulses, a partially embedded steel pipe pile loaded by idealized wave forces, and a partially embedded steel pipe pile excited by sinusoidal, earthquake-induced forces.

A guide for data input is presented which allows routine application of the method of analysis with little necessary reference to the body of the main report. Any number of analyses may be run at the same time.

## IMPLEMENTATION STATEMENT

The utilization of numerical methods to describe computer models of problems in structural dynamics has been an interesting subject to which many structural engineers have devoted their efforts in recent years.

In this study, an efficient computer program, DBC5, is developed for the analysis of beam-column problems, static or dynamic, which have either linearly-elastic or nonlinearly-inelastic supports. The path-dependent history of loading of the nonlinearly-inelastic resistance-deflection curve of the support is considered. Potential applications include study of the dynamic response of actual truck-loaded bridges, prediction of the response of offshore piles to wave forces, analysis of railroad loadings on continuous spans which are supported on soil foundations, analysis of transverse response of partially embedded piles to earthquake-induced forces, and prediction of the hysteresis effect of inelastic supports under pavement slabs.

Recommendations are made for further research in developing other better multi-element computer models for nonlinear supports so that buckling, fracture, softening, and relaxation (creep) of the support could also be considered in the program.

It is further recommended that this program be put into test use by designers of the Texas Highway Department to further evaluate its uses, and to investigate needed extensions or modifications to make it more usable for the practicing design engineer.

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## NOMENCLATURE

<u>Symbol</u>	<u>Typical Units</u>	<u>Definition</u>
$a_{k+1}, a_{k-1}$	----	Coefficients in stiffness matrix
$b_{k+1}, b_{k-1}$	----	Coefficients in stiffness matrix
C	in-lb	Concentrated applied couple
$c_{k+1}, c_k, c_{k-1}$	----	Coefficient in stiffness matrix
D	lb-in <sup>2</sup>	Lumped internal damping factor divided by time increment length
(D <sup>e</sup> )	lb-sec/in	Lumped external viscous damping factor
(D <sup>i</sup> )	lb-in <sup>2</sup> -sec	Lumped internal damping factor
$d_{k+1}, d_{k-1}$	----	Coefficient in stiffness matrix
e	----	Base of natural logarithms
E	lb/in <sup>2</sup>	Modulus of elasticity
$e_{k+1}, e_{k-1}$	----	Coefficient in stiffness matrix
$f_{j,k}$	----	Coefficient in stiffness matrix
F	lb-in <sup>2</sup>	Flexural stiffness, EI
h	in	Beam increment length
$h_t$	sec	Time increment length
I	in <sup>4</sup>	Moment of inertia of cross section
j	----	Beam station number
k	----	Time station number
$M_j$	in-lb	Static bending moment at beam station j
$M_{j,k}$	in-lb	Dynamic bending moment at beam station j and time station k

<u>Symbol</u>	<u>Typical Units</u>	<u>Definition</u>
$q^2$	----	$2\rho h^3/h_t^2$
$Q$	lb	Concentrated applied static transverse load
$Q^T$	lb	Concentrated applied dynamic transverse load
$Q_j^R, Q_j^S$	lb	Static support reaction at beam station $j$
$Q_{j,k}^R, Q_{j,k}^S$	lb	Dynamic support reaction at beam station $j$ and time station $k$
$\Delta Q_1^T, \Delta Q_2^T, \dots$	lb	Retained projected values of resistance of segments 1, 2, ... on support curve
$R$	in-lb/rad	Concentrated rotational restraint
$R_1, R_2, \dots$	lb	Resistances in the sub-springs 1, 2, ... of the nonlinear support
$R_1^S, R_2^S$	lb	Residual resistances in sub-springs 1, 2
$R_A, R_B$	lb	Resistances at points A and B on support curve
$S_j^S$	lb/in	Concentrated transverse linear spring restraint at beam station $j$
$S_{j,k}^N$	lb/in	Concentrated transverse nonlinear spring restraint at beam station $j$ and time station $k$
$S_1, S_2, \dots$	----	Retained values of slopes of segments 1, 2, ... on support curve
$T$	lb	Static axial thrust
$T^T$	lb	Dynamic axial thrust
$V_j$	lb	Static shear of bar $j$
$V_{j,k}$	lb	Dynamic shear of bar $j$ at time station $k$
$W_j$	in	Static transverse deflection at beam station $j$

## CHAPTER 1. INTRODUCTION

In recent years, a great deal of interest has been focused on the utilization of numerical methods to describe computer models of problems in structural dynamics. Many computer programs (for instance, Refs 3 and 10) are available for solving for the dynamic response of beams or slabs which are either linearly or nonlinearly supported; no program has been found, however, which considers the path-dependent history of loading of nonlinearly-inelastic supports. As a result of the necessity to consider the important effects of energy lost due to the hysteretic behavior of the supports, much of the present work has been devoted to the development of multi-element models to simulate the nonlinearly-inelastic supports. With these models, general rules in FORTRAN logic can be observed and used to predict the loading paths of resistance-deflection curves of the supports. The models for the nonlinearly-inelastic supports described in this work are not time-dependent; therefore, relaxation (creep) of resistance of the support is not included in the model. The retardation, or delayed elasticity, of the support can be considered by installing an external viscous dashpot in parallel with the support model. The softening of the nonlinearly-inelastic support is also not included but strain-hardening may be considered.

### Purpose and Scope of Program DBC5

The primary purpose of this investigation is to develop an efficient computer program for solving for the transverse dynamic response of a beam-column resting on linearly or nonlinearly-elastic or nonlinearly-inelastic supports which are simulated by the proposed multi-element models described in Chapter 3. When the hysteretic behavior of the supports is considered, the program is able to predict more accurately the dynamic response of beams, piles, slabs, or even bridges which are supported by soil foundations.

A marching method of solution is used which is based on an implicit formula introduced by Crank and Nicolson (Ref 2) to solve second-order heat flow problems. Salani (Ref 10) is credited with applying this implicit

formula in determining the transverse time-dependent linear deflections of a beam or plate. Essentially, the beam is replaced by an arbitrary number of rigid bars and deformable joints, and time is divided into discrete, equal intervals. The representation readily permits the flexural stiffness, the elastic restraints, the mass densities, and the applied external loadings to be discontinuous and lumped at the deformable joints which connect the rigid bars. The governing partial differential equation at each joint is approximated by a difference equation that includes several unknown deflections of the joint, which occur at specified time intervals. All difference equations are based on the assumptions of linear elasticity and the elementary beam theory. The effects of transverse shear and rotatory inertia are neglected.

The nonlinear characteristics of the supports are considered by using either a spring-load iteration process (adjusting both the stiffness and the load from one iteration to the next, which is known as tangent modulus method) or a load-iteration technique (adjusting only the load). The iteration process compares successively computed deflections until a specified tolerance is satisfied. Only three nonlinear characteristics of support curves are considered: the first is exhibiting the same resistance to either upward to downward deflection, hereafter referred to as the symmetric resistance-deflection curve; the second is allowing the beam to lift off the support when it deflects upward, hereafter referred to as the negative one-way resistance-deflection curve; the third is allowing the beam to lift off the support when it deflects downward, hereafter referred to as positive one-way resistance-deflection curve. Two multi-element models to simulate the three types of support are introduced in this work.

### Application

Computer Program DBC5 is versatile and efficient for

- (1) solving for the transverse response of beam-columns with linear and nonlinear supports under free or forced vibration;
- (2) computing slopes and shears of the bars, bending moments, and support reactions of the deformable joints, statically or dynamically; and
- (3) plotting computed deflections or moments along either the time or beam axis for the requested monitor stations.

Applied forces include static fixed loads, time variant axial thrusts, and time variant lateral loads. Static solutions of beam-columns under fixed loads may also be obtained.

The computer program is intended to provide an efficient tool for analyzing many problems encountered by highway and foundation designers which are complicated and unsolvable using classical methods. A variety of highway structures, such as bridge girders, guard rails, or even a whole bridge floor under moving loads or suddenly applied impact, can be simulated by the computer model of Program DBC5 and solved efficiently. The capability to treat supports that behave nonlinearly and inelastically provides for direct solutions of transverse deflections of railroad rails under moving loads and the prediction of the responses of offshore piles to wave forces, as well as the study of transverse responses of piles to earthquake-induced forces, since the parts of rails and piles supported by the soil can be reasonably represented by the proposed multi-element models for considering the path-dependent history of loading of the soil supports.

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## CHAPTER 2. DEVELOPMENT OF THE IMPLICIT OPERATOR

Program DBC5 is developed for the dynamic analysis of beam-columns resting on linearly-elastic or nonlinear supports under time variant axial thrusts and lateral loads. The program uses a discrete-element model for developing the equation of motion of beams. An implicit formula of the Crank-Nicolson (Ref 2) type is then utilized to form a marching operator. At each point in time, a set of simultaneous equations for the unknown deflections at the deformable joints can be obtained by systematically applying the marching operator at all joints. A recursive procedure is then used to solve the simultaneous equations. A brief discussion of the method of the recursive solution procedure is included in Appendix B.

### Discrete-Element Model for Dynamic Beam-Columns

Matlock and Taylor have developed a static model composed of rigid bars and springs (Fig 1) which can be used to simulate a beam-column. In Ref 5 the efficiency of this model has been proved by solving a variety of structures which can be simulated as beam-columns. By the addition of masses and damping factors lumped at the deformable joints, a dynamic model (Fig 2) is formed that can closely simulate the dynamic response of real beam-columns.

### Equation of Motion of Beam-Columns

The equation of motion for transverse vibration of beams can be obtained by summing all the forces, internal and external, at a particular joint  $j$  and a particular time station  $k$  (see Fig 3). The concept is based on D'Alembert's principle. Thus an equation can be written in terms of the unknown deflections  $w$ , moments  $M$ , internal damping factors  $D^i \dots$ ,

$$- M_{j-1} + 2M_j - M_{j+1} - (D^i)_{j-1} \frac{d}{dt} (\varphi_{j-1, k}) + 2(D^i)_j \frac{d}{dt} (\varphi_{j, k})$$

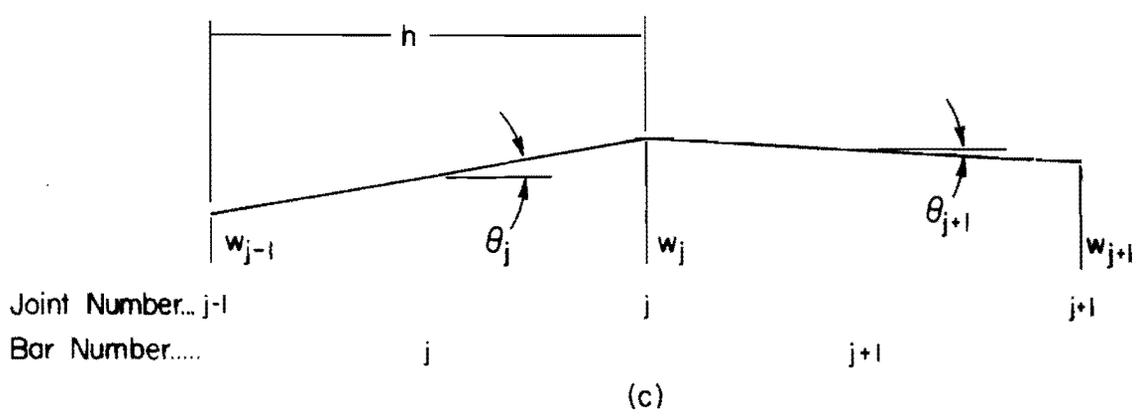
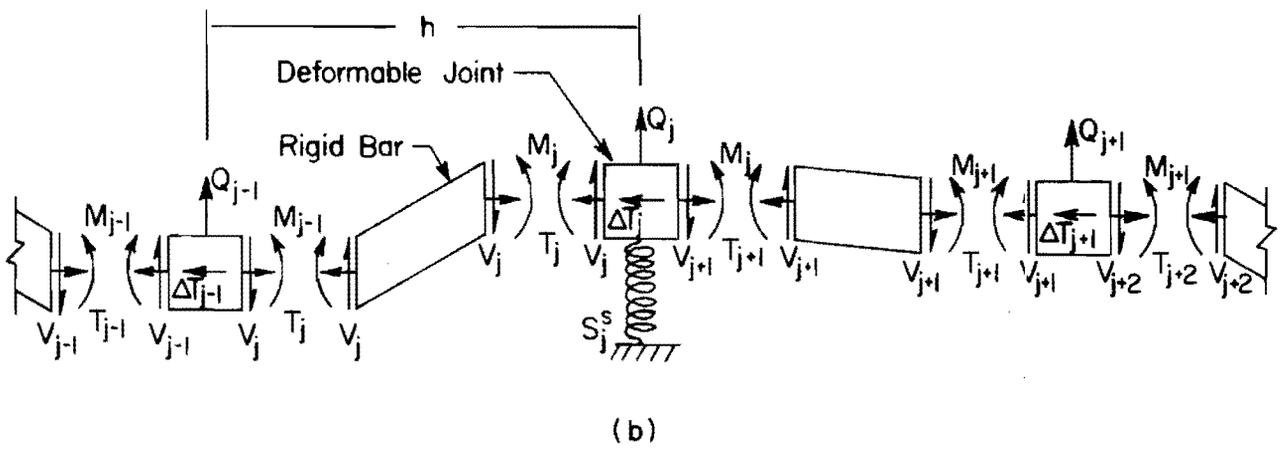
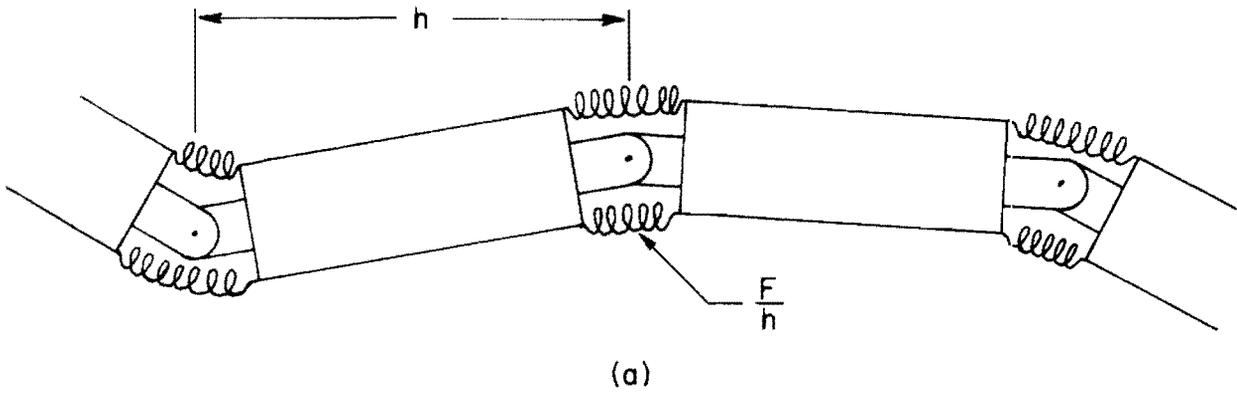


Fig 1. Mechanical model of beam-column.

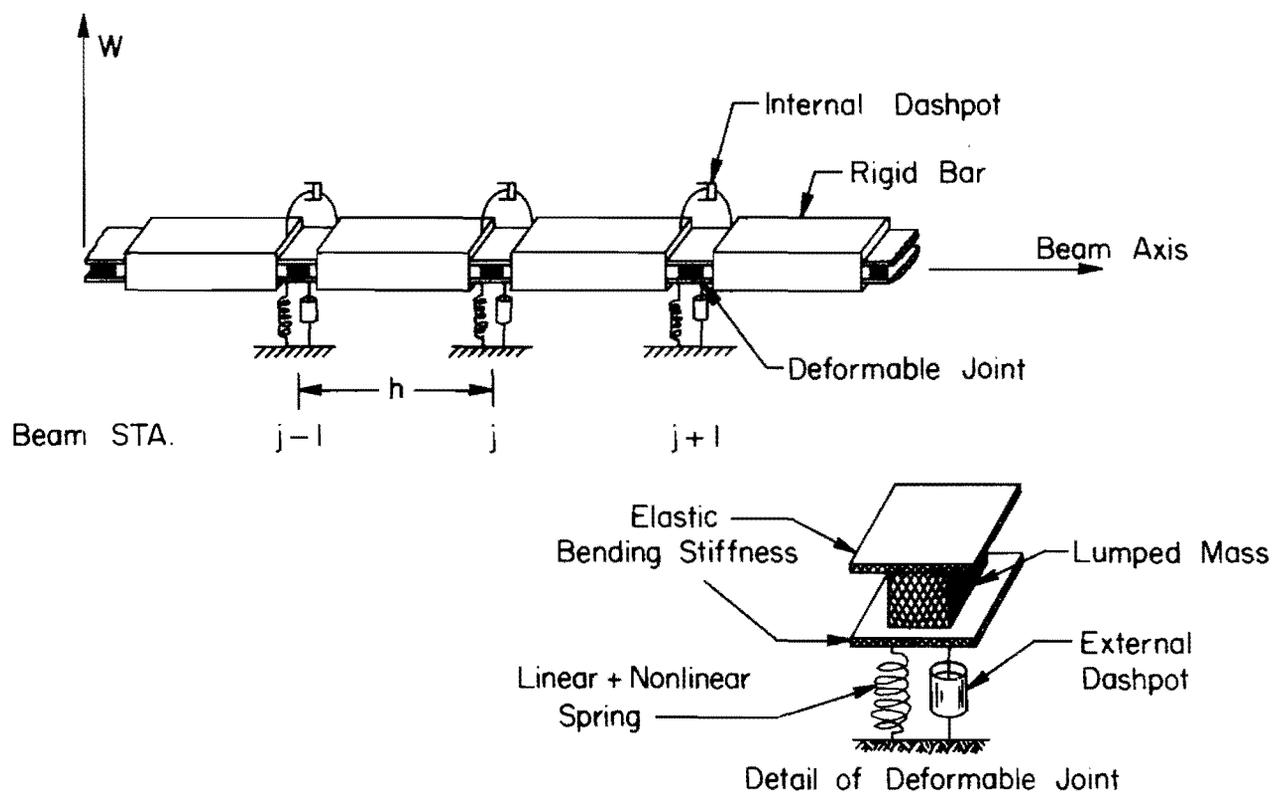


Fig 2. Dynamic model of beam-column.

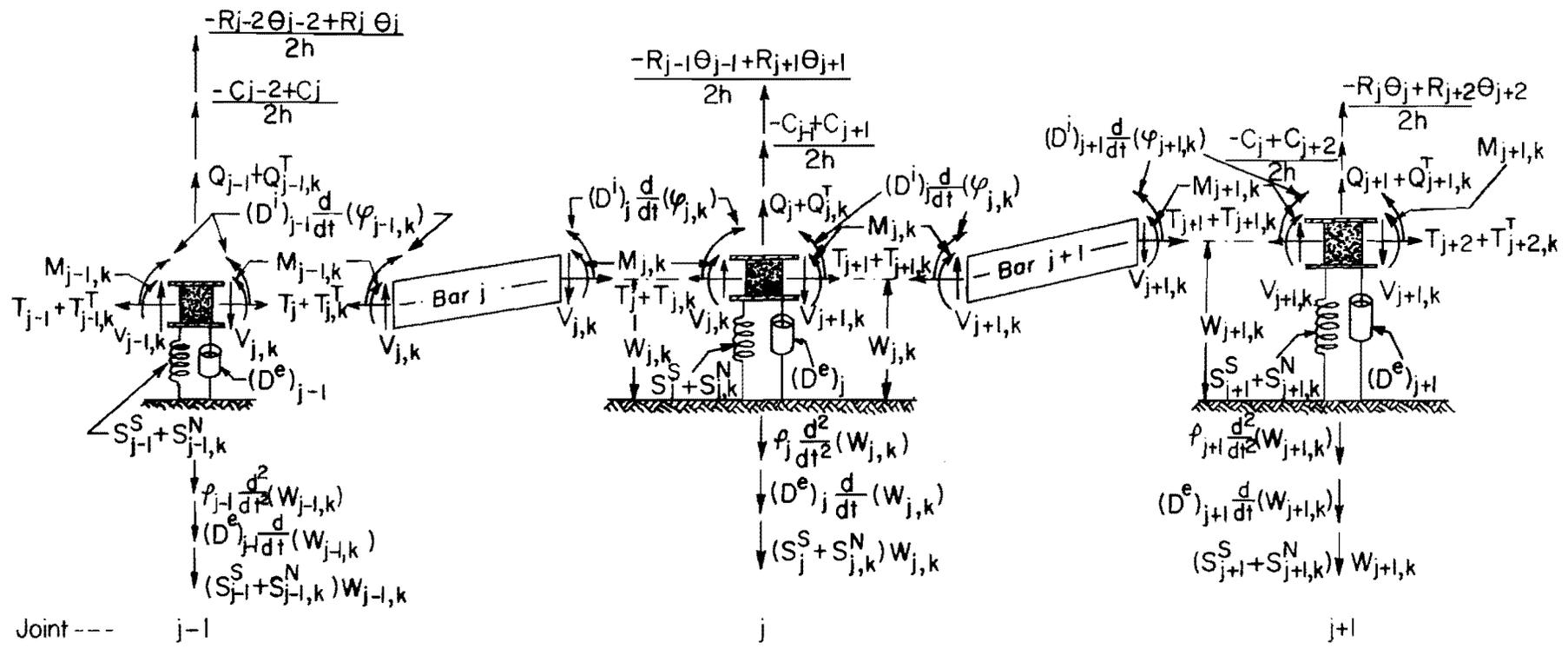


Fig 3. Free-body diagram of a portion of the dynamic beam-column model.

$$\begin{aligned}
& - (D^i)_{j+1} \frac{d}{dt} (\varphi_{j+1,k}) - (T_j + T_{j,k}^T) (-w_{j-1,k} + w_{j,k}) \\
& + (T_{j+1} + T_{j+1,k}^T) (-w_{j,k} + w_{j+1,k}) + hQ_{j,k}^T + hQ_j \\
& + 1/2(-R_{j-1}\theta_{j-1} + R_{j+1}\theta_{j+1}) + 1/2(-C_{j-1} + C_{j+1}) - hS_j^S w_{j,k} \\
& - hS_{j,k}^N w_{j,k} - h\rho_j \frac{d^2}{dt^2} (w_{j,k}) - h(D^e)_j \frac{d}{dt} (w_{j,k}) = 0 \quad (2.1)
\end{aligned}$$

By the use of Crank-Nicolson's implicit formula, Eq 2.1 can be further represented as

$$\begin{aligned}
& a_{k+1} w_{j-2,k+1} + b_{k+1} w_{j-1,k+1} + c_{k+1} w_{j,k+1} + d_{k+1} w_{j+1,k+1} + e_{k+1} w_{j+2,k+1} \\
& = c_k w_{j,k} + a_{k-1} w_{j-2,k-1} + b_{k-1} w_{j-1,k-1} + c_{k-1} w_{j,k-1} \\
& + d_{k-1} w_{j+1,k-1} + e_{k-1} w_{j+2,k-1} + f_{j,k} \quad (2.2)
\end{aligned}$$

where

$$\begin{aligned}
a_{k+1} &= F_{j-1} + \frac{1}{h_t} (D^i)_{j-1} - 0.25hR_{j-1} \\
b_{k+1} &= -2(F_{j-1} + F_j) - \frac{2}{h_t} \left[ (D^i)_{j-1} + (D^i)_j \right] - h^2 [T_j + 0.5(T_{j,k-1}^T + \\
& T_{j,k+1}^T)] \\
c_{k+1} &= (F_{j-1} + 4F_j + F_{j+1}) + \frac{1}{h_t} \left[ (D^i)_{j-1} + 4(D^i)_j + (D^i)_{j+1} \right] \\
& + h^2 [T_j + T_{j+1} + 0.5(T_{j,k-1}^T + T_{j,k+1}^T + T_{j+1,k-1}^T + T_{j+1,k+1}^T)]
\end{aligned}$$

$$\begin{aligned}
& + 0.25h(R_{j-1} + R_{j+1}) + 2h^3 \frac{\rho_i}{h_t^2} + h^3 (D^e)_j / h_t + 0.5h^3 [(S_j^s + S_{j,k-1}^N) \\
& + (S_j^s + S_{j,k+1}^N)]
\end{aligned}$$

$$\begin{aligned}
d_{k+1} &= -2(F_j + F_{j+1}) - \frac{2[(D^i)_j + (D^i)_{j+1}]}{h_t} \\
& - h^2 [T_{j+1} + 0.5(T_{j+1,k-1}^T + T_{j+1,k+1}^T)]
\end{aligned}$$

$$e_{k+1} = F_{j+1} + \frac{(D^i)_{j+1}}{h_t} - 0.25hR_{j+1}$$

$$c_k = \frac{4h^3 \rho_i}{h_t^2}$$

$$a_{k-1} = -a_{k+1} + \frac{2(D^i)_{j-1}}{h_t}$$

$$b_{k-1} = -b_{k+1} - \frac{4[(D^i)_{j-1} + (D^i)_j]}{h_t}$$

$$c_{k-1} = -c_{k+1} + \frac{2[(D^i)_{j-1} + 4(D^i)_j + (D^i)_{j+1}]}{h_t} + \frac{2h^3 (D^e)_j}{h_t}$$

$$d_{k-1} = -d_{k+1} - \frac{4[(D^i)_j + (D^i)_{j+1}]}{h_t}$$

$$e_{k-1} = -e_{k+1} + \frac{2(D^i)_{j+1}}{h_t}$$

$$f_{j,k} = h^3 \left[ (Q_j + Q_{j,k-1}^T) + (Q_j + Q_{j,k+1}^T) \right] + h^2 (-C_{j-1} + C_{j+1})$$

Detailed derivations of Eqs 2.1 and 2.2 are included in Appendix A. In Fig 4, an implicit operator of the Crank-Nicolson (Ref 2) type is shown for Eq 2.2.

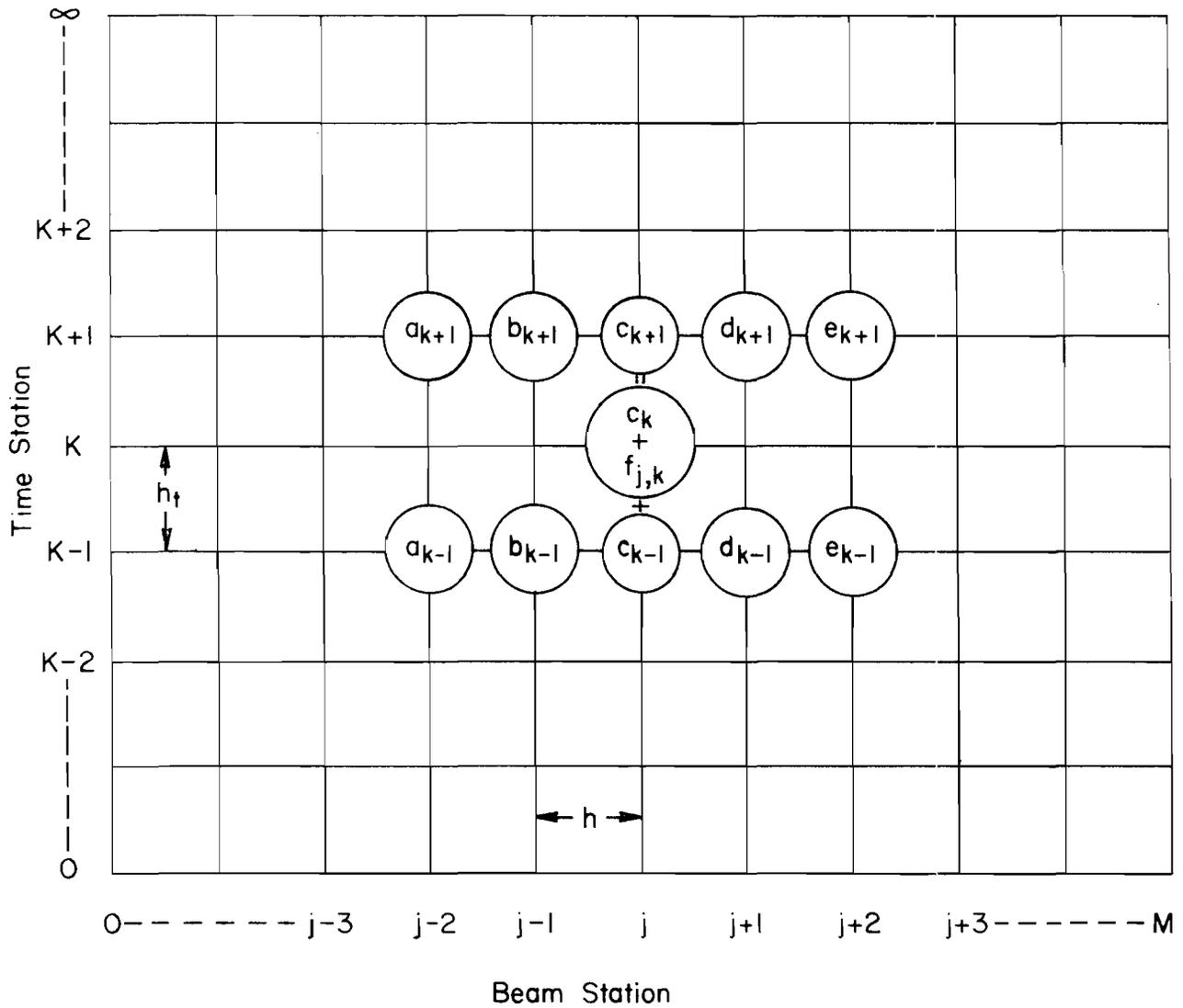
To start the dynamic solutions, a static model as described in Ref 2 is used twice, at time stations  $-2$  and  $-1$ , for solving the initial deflected shape of the beam-column due to static loads. Since the implicit operator requires the deflected shape at only the two previous time stations, there are no considerations of the initial velocities and initial accelerations of the deformable joints of the beam. This is consistent with normal practice, since deformable joints of a beam normally have no initial velocities or initial accelerations.

All deflections at time station  $k+1$  are unknown. To solve for the unknown deflections at time station  $k+1$ , the operator, which requires the deflected shapes of the beam at time stations  $k$  and  $k-1$  to be known, is applied systematically at beam joints  $j = -1, 1, 2 \dots M+1$ . This procedure establishes a set of simultaneous equations wherein each equation includes five unknown deflections. These equations are solved by a two-pass, recursive technique described in Ref 4 (see Appendix B) for the unknown deflection of every joint. Once the deflected shape of a member at time station  $k+1$  is known, the slopes, bending moments, shears, and support reactions for the member at the previous time station  $k$  can be determined by using the procedures described below.

Slope. Equation 2.3 is the simple-difference expression for slope of the individual bar  $j$  in terms of the deflections of joints  $j-1$  and  $j$  and the beam increment length  $h$  (see Fig 1(c)).

$$\theta_j = \frac{-w_{j-1} + w_j}{h} \quad (2.3)$$

The concept applied for both the static model and the dynamic model. It will be seen in results printed from the program that the slope is printed between the beam stations; this slope is that of the bar between the indicated beam stations.



$$\begin{aligned}
 & a_{k+1} w_{j-2,k+1} + b_{k+1} w_{j-1,k+1} + c_{k+1} w_{j,k+1} + d_{k+1} w_{j+1,k+1} + e_{k+1} w_{j+2,k+1} \\
 & - c_k w_{j,k} + a_{k-1} w_{j-2,k-1} + b_{k-1} w_{j-1,k-1} + c_{k-1} w_{j,k-1} + d_{k-1} w_{j+1,k-1} \\
 & + e_{k-1} w_{j+2,k-1} + f_{j,k}
 \end{aligned}$$

Fig 4. Implicit operator of Crank-Nicolson type used in Program DBC5.

Bending Moment. In conventional beams, the bending moment is equal to the product of the flexural stiffness and the curvature. In the finite-element model the flexibility of the beam and the curvature are lumped at the beam station points. The corresponding relation for bending moment  $M$  in the static model is

$$M_j = \frac{F_j}{h} \left( \frac{w_{j-1} - 2w_j + w_{j+1}}{h} \right) \quad (2.4)$$

In the dynamic model, the internal damping factor lumped at the joint contributes its effect in addition to the conventional bending moment. Thus,

$$\begin{aligned} M_{j,k} = & \frac{F_j}{h} \left( \frac{w_{j-1,k} - 2w_{j,k} + w_{j+1,k}}{h} \right) \\ & + \frac{(D^i)_j}{2h^2 h_t} \left( -w_{j-1,k-1} + 2w_{j,k-1} - w_{j+1,k-1} + w'_{j-1,k+1} \right. \\ & \left. - 2w_{j,k+1} + w_{j+1,k+1} \right) \end{aligned} \quad (2.5)$$

Shear. Shear  $V_j$  in the static model is found from the equation of moment equilibrium of bar  $j$  (Fig 1(b)). Thus,

$$V_j = \frac{-M_{j-1} + M_j}{h} - T_j \left( \frac{-w_{j-1} + w_j}{h} \right) \quad (2.6)$$

For the dynamic model, in addition to the effects of conventional bending moments and static axial thrusts, the moments contributed by the internal damping factors  $(D^i)_{j-1}$  and  $(D^i)_j$ , and the time dependent axial thrusts  $T_{j,k}^T$  are also found in the equation of moment equilibrium of bar  $j$  (Fig 3). Thus,

$$V_{j,k} = \frac{-M_{j-1,k} + M_{j,k}}{h} - [T_j + T_{j,k}^T] \left( \frac{-w_{j-1,k} + w_{j,k}}{h} \right)$$

$$\begin{aligned}
& - (D^i)_{j-1} \frac{d}{dt} \left( \frac{w_{j-2,k} - 2w_{j-1,k} + w_{j,k}}{h^3} \right) \\
& + (D^i)_j \frac{d}{dt} \left( \frac{w_{j-1,k} - 2w_{j,k} + w_{j+1,k}}{h^3} \right)
\end{aligned} \tag{2.7}$$

Substituting

$$\begin{aligned}
T_{j,k}^T &= \frac{1}{2} (T_{j,k-1}^T + T_{j,k+1}^T), \\
\frac{d}{dt} \left( \frac{w_{j-2,k} - 2w_{j-1,k} + w_{j,k}}{h^3} \right) &= \frac{1}{2h_t h^3} (-w_{j-2,k-1} + 2w_{j-1,k-1} \\
&\quad - w_{j,k-1} + w_{j-2,k+1} - 2w_{j-1,k+1} + w_{j,k+1})
\end{aligned}$$

and

$$\begin{aligned}
\frac{d}{dt} \left( \frac{w_{j-1,k} - 2w_{j,k} + w_{j+1,k}}{h^3} \right) &= \frac{1}{2h_t h^3} (-w_{j-1,k-1} + 2w_{j,k-1} \\
&\quad - w_{j+1,k-1} + w_{j-1,k+1} - 2w_{j,k+1} + w_{j+1,k+1})
\end{aligned}$$

into Eq 2.7, thus,

$$\begin{aligned}
V_{j,k} &= \frac{-M_{j-1,k} + M_{j,k}}{h} - \left[ T_j + 0.5 (T_{j,k-1}^T + T_{j,k+1}^T) \right] \left( \frac{-w_{j-1,k} + w_{j,k}}{h} \right) \\
&\quad - \frac{(D^i)_{j-1}}{2h_t h^3} (-w_{j-2,k-1} + 2w_{j-1,k-1} - w_{j,k-1} + w_{j-2,k+1}
\end{aligned}$$

$$\begin{aligned}
& - 2w_{j-1,k+1} + w_{j,k+1} \Big) + \frac{(D^i)_j}{2h_t h^3} \left( -w_{j-1,k-1} + 2w_{j,k-1} - w_{j+1,k-1} \right. \\
& \left. + w_{j-1,k+1} - 2w_{j,k+1} + w_{j+1,k+1} \right) \quad (2.8)
\end{aligned}$$

As seen in Fig 1(b) (or Fig 3),  $V_j$  (or  $V_{j,k}$ ) is the shear throughout the length of bar  $j$ . Therefore Program DBC5 is written such that the shear computed in each bar is printed between the adjacent beam stations. In the program, the rotational restraints and the applied couples, which conventionally are considered to be concentrated at a point, are acting on the beam as equal and opposite loads separated by two increments, as shown in Fig 5. Therefore, the shear for only the bars adjacent to beam stations with applied couples or rotational restraints is affected by these loads and is not the same as conventional shear.

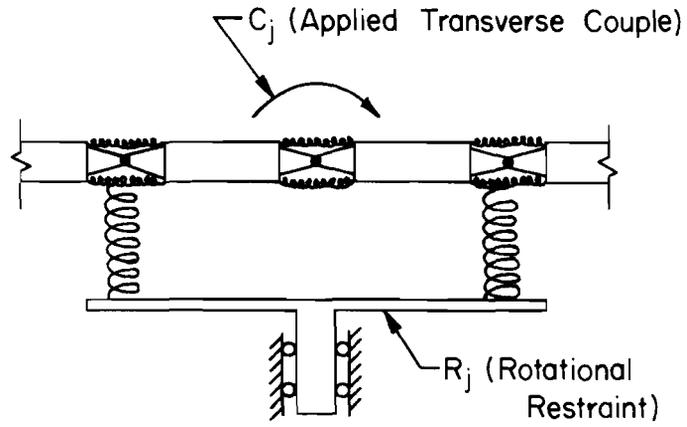
Support Reaction. As described in Ref 5, the support reaction for the static model can be obtained by Eq 2.9 or Eq 2.10.

If joint  $j$  is supported on a linear spring  $S_j^s$ , the reaction  $Q_j^s$  is

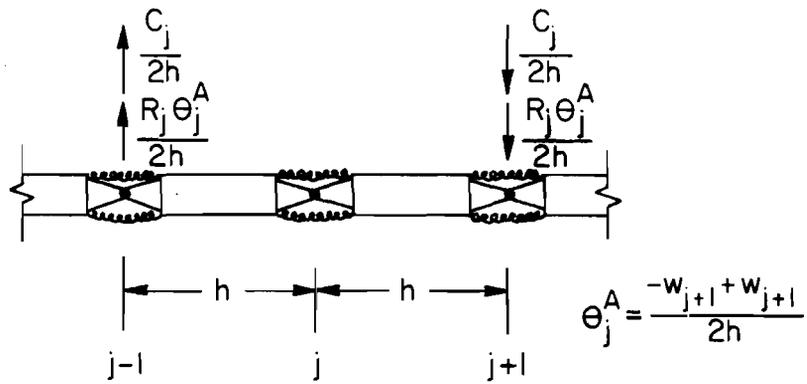
$$Q_j^s = - S_j^s w_j \quad (2.9)$$

If joint  $j$  is supported on a nonyielding support (deflection  $w_j$  equal to zero), the support reaction is

$$\begin{aligned}
Q_j^R = & \frac{M_{j-1} - 2M_j + M_{j+1}}{h} - Q_j + \frac{C_{j-1} - C_{j+1}}{2h} \\
& - \frac{R_{j-1}w_{j-2} - R_{j-1}w_j - R_{j+1}w_j + R_{j+1}w_{j+2}}{4h^2} \\
& - \frac{T_j w_{j-1} - T_j w_j - T_{j+1} w_j + T_{j+1} w_{j+1}}{h} \quad (2.10)
\end{aligned}$$



(a) Mechanical model.



(b) Equivalent forces.

Fig 5. Rotational resistance \$R\$ and applied couple \$C\$ acting on the mechanical model.

For the dynamic model, the support reaction can be obtained by Eq 2.11 or Eq 2.12.

If joint  $j$  is supported on a linear  $(S_j^S)$  or nonlinear  $(S_{j,k}^N)$  spring or both  $(S_j^S + S_{j,k}^N)$ , the support reaction  $Q_{j,k}^S$  is

$$Q_{j,k}^S = - S_{j,k}^S w_{j,k}$$

or

$$S_{j,k}^N w_{j,k} + Q_{i,k}^S \quad \text{or} \quad - (S_j^S + S_{j,k}^N) w_{j,k} + Q_{i,k}^S \quad (2.11)$$

where  $Q_{i,k}^S$  is the iterative correction reaction of the nonlinear support at time station  $k$ .

If joint  $j$  is supported on a nonyielding support, the support reaction is

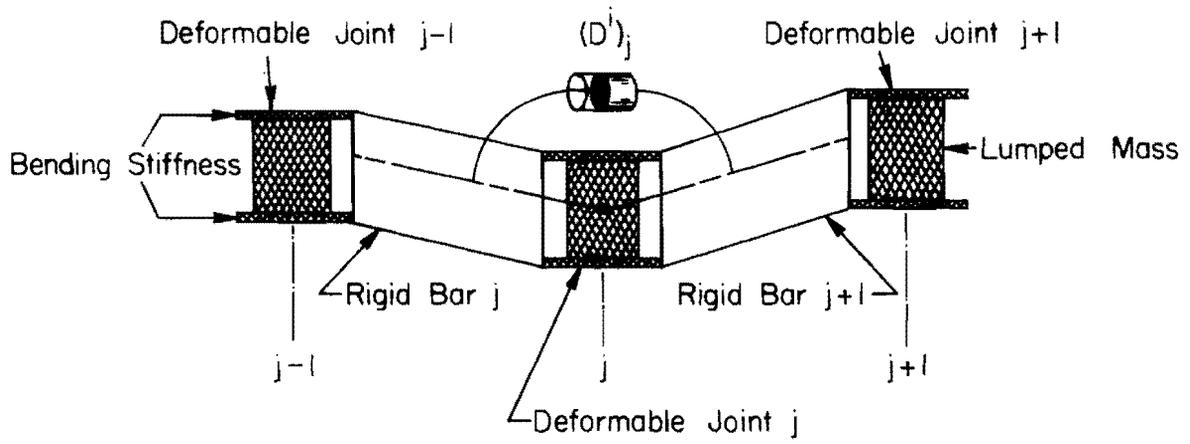
$$\begin{aligned} Q_{j,k}^R &= \frac{M_{j-1,k} - 2M_{j,k} + M_{j+1,k}}{h} - Q_j + \frac{C_{j-1} - C_{j+1}}{2h} \\ &- \left( \frac{Q_{j,k-1}^T + Q_{j,k+1}^T}{2} \right) - \frac{1}{4h^2} \left( R_{j-1} w_{j-2,k} - R_{j-1} w_{j,k} - R_{j+1} w_{j,k} \right. \\ &+ \left. R_{j+1} w_{j+2,k} \right) - \frac{1}{h} \left( T_j w_{j-1,k} - T_j w_{j,k} - T_{j+1} w_{j,k} + T_{j+1} w_{j+1,k} \right) \\ &+ \frac{(D^i)_{j-1}}{h^3 2h_t} \left( -w_{j-2,k-1} + 2w_{j-1,k-1} - w_{j,k-1} + w_{j-2,k+1} \right. \\ &- \left. 2w_{j-1,k+1} + w_{j,k+1} \right) - \frac{2(D^i)_j}{h^3 2h_t} \left( -w_{j-1,k-1} + 2w_{j,k-1} \right. \end{aligned}$$

$$\begin{aligned}
& - w_{j+1,k-1} + w_{j-1,k+1} - 2w_{j,k+1} + w_{j+1,k+1} \\
& + \frac{(D^i)_{j+1}}{h^3 2h_t} \left( - w_{j,k-1} + 2w_{j+1,k-1} - w_{j+2,k-1} + w_{j,k+1} \right. \\
& \left. - 2w_{j+1,k+1} + w_{j+2,k+1} \right) - \frac{1}{h} \left[ 0.5(T_{j,k-1}^T + T_{j,k+1}^T) \right. \\
& \left. (w_{j-1,k} - w_{j,k}) - 0.5(T_{j+1,k-1}^T + T_{j+1,k+1}^T) (w_{j,k} - w_{j+1,k}) \right] \\
& + \frac{\rho_j}{h^2} (w_{j,k-1} - 2w_{j,k} + w_{j,k+1}) + \frac{(D^e)_j}{2h_t} \left( - w_{j,k-1} \right. \\
& \left. + w_{j,k+1} \right) \tag{2.12}
\end{aligned}$$

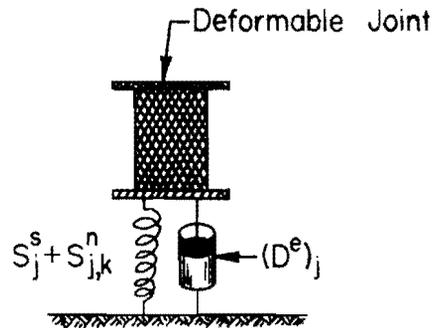
### Internal and External Damping

In Eq 2.1, two symbols  $(D^i)_j$  and  $(D^e)_j$  are introduced to represent the internal damping coefficients and the external viscous damping coefficients, which are both lumped at joint  $j$ . The rheological models of the two damping constants are shown in Fig 6. The typical units of  $(D^i)_j$  and  $(D^e)_j$  are  $\text{lb-in}^2\text{-sec/sta}$  and  $\text{lb-sec/in/sta}$ . The internal damping is due to the internal dynamic viscosity of materials. Boltzmann first proposed the hereditary theory which attributed the loss of energy, due to internal damping, to the elastic delay by which the deformation lagged behind the applied force.

Coulomb also proposed a viscous theory which assumed that the viscosity effects are proportional to the first time derivative of strain. The coefficient of proportionality (constant for each material at constant temperature) is called the coefficient of viscosity. In Ref 13, Volterra has shown a mathematical relationship between the viscous and hereditary damping theories. The relationship of stress  $\sigma$  to strain  $\epsilon$  for the material based on the hereditary theory can be assumed as



(a)



(b)

Fig 6. Rheological models of internal and external damping coefficients.

$$\sigma = E\epsilon + \sum_{n=1}^{\infty} P_n \frac{d^{n-1}\epsilon}{dt^{n-1}} \quad (2.13)$$

where

$$P_n = \frac{(-1)^{n-1}}{(n-1)!} \int_0^{T_0} \tau^{n-1} \phi(\tau) d\tau$$

$\tau$  is an instant of time between 0 and  $T_0$ ,  $T_0$  is the period of heredity ( $\phi(\tau) = 0$  for  $\tau > T_0$ ), and  $\phi(\tau)$  is the memory function which can be found from the experimental data.

If the coefficients  $P_n$  with  $n > 1$  can be neglected, Eq 2.13 reduces to

$$\sigma = E\epsilon + \nu \frac{d\epsilon}{dt} \quad (2.14)$$

where  $\nu = P_1$ . Equation 2.14 is the stress-strain relationship of the material based on viscous theory. Therefore, viscous theory is included in hereditary theory.

In the rheological model shown in Fig 6(a), the internal damping coefficients  $(D^i)_j$  are related to the first time derivative of curvature. Thus

$$M_j = F_j \varphi_j + (D^i)_j \frac{d\varphi_j}{dt} \quad (2.15)$$

gives the relationship between the moment  $M_j$  and the curvature  $\varphi_j$ .

Equation 2.15 is used in Program DBC5 for calculating the bending moments contributed by the flexural stiffness and the internal damping coefficient at the deformable joint  $j$ .

It will be shown in problem 4 described in Chapter 5 that the internal damping factors have little effect on the solutions of lower frequencies of a vibrating steel beam. The vibrations at higher frequencies, however, are damped out with time due to the effects of internal damping and rapid changes of curvatures.

The external viscous damping force is defined as  $f(w_{j,k}) = -(D^e)_j w_{j,k}$ . The constant  $D^e$  (in tons per unit velocity, lb-sec/inch, or kip-sec/ft) is called the coefficient of viscous damping. This type of damping occurs for small velocities in lubricated sliding surfaces, dashpots, and hydraulic shock-absorbers. The so-called viscous resistances are produced by the slow motion of immersed bodies in fluid, either liquid or gas. The value of the coefficient  $(D^e)_j$  depends essentially on the nature of the fluid, as well as on the form and the dimensions of the immersed body.

With these two types of damping factors, the Program DBC5 can solve the problems characterized by the so-called visco-elastic nonlinearity to a great extent.

#### Linearly-Elastic and Nonlinearly-Inelastic Supports

In the discrete-element model, lateral supports of various types can be represented by either an equivalent linearly-elastic spring or a nonlinearly-inelastic spring at each of the beam stations. For example, if the beam is supported by columns or piers, the axial stiffnesses of the columns or piers can be approximately estimated and included as equivalent linear spring constants. For more reality, the true behavior of many lateral supports can be better represented by the nonlinear characteristics of the axial resistance-deflection curves of columns, or piers, and soil supports.

If the beam is laterally supported by a soil foundation, better results are obtained by using the nonlinear characteristics of the resistance-deflection curves of the soil than by using the equivalent linear spring constants of the soil resistances, since these characteristics represent the true behavior of the soil supports. In Program DBC5, the nonlinear characteristics of each lateral support are described by a curve consisting of straight line segments. Only three types of the nonlinear characteristics of the lateral supports, symmetric, negative one-way, and positive one-way, are considered in this work.

The symmetrical nonlinear resistance-deflection curve is shown in Fig 7. The force developed against the beam-column model by the supports is plotted on the vertical axis and the model deflection is plotted on the horizontal axis. For both load and deflection, the positive sense is upward. The positive and negative one-way resistance-deflection curves are shown in Fig 8.

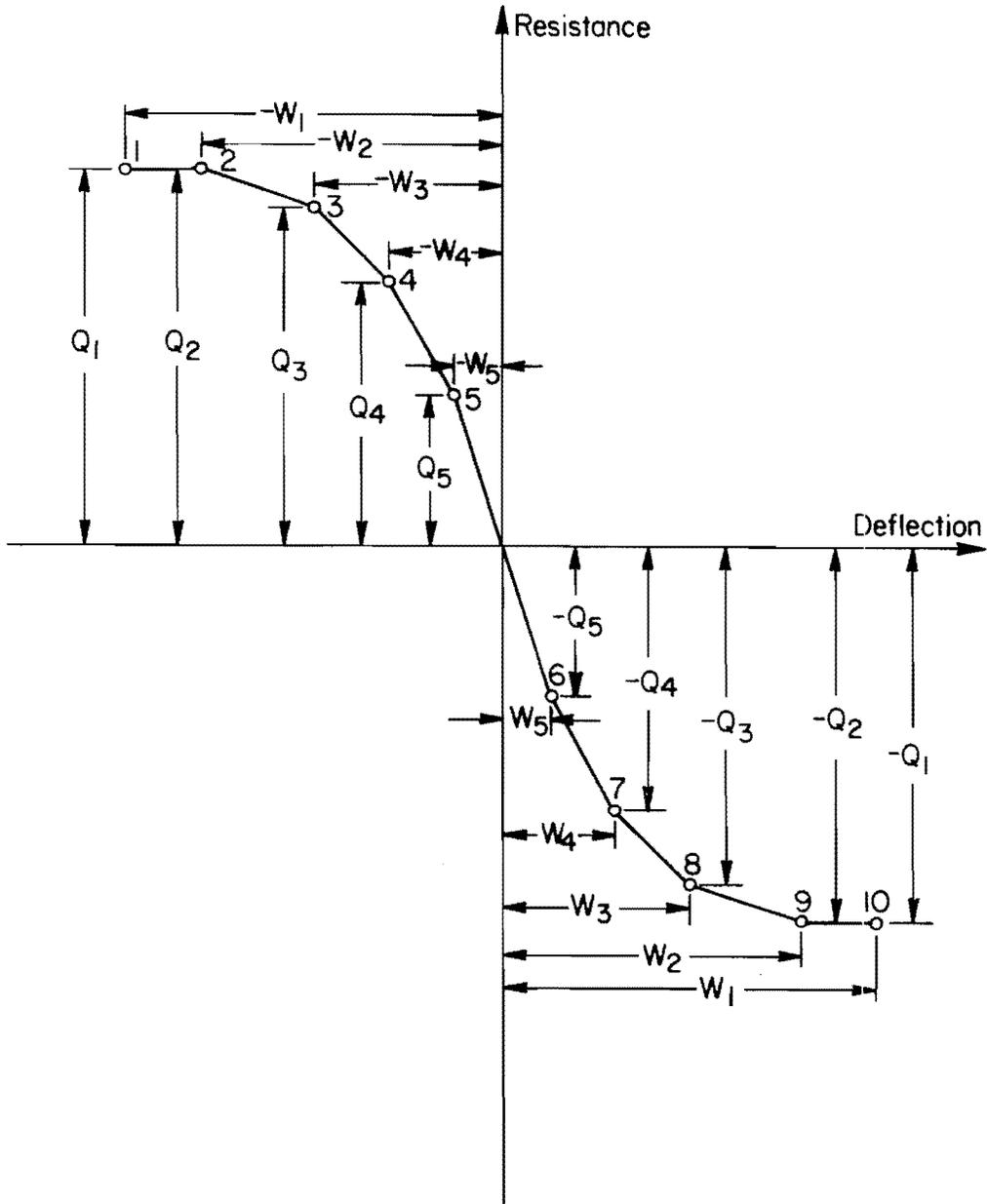
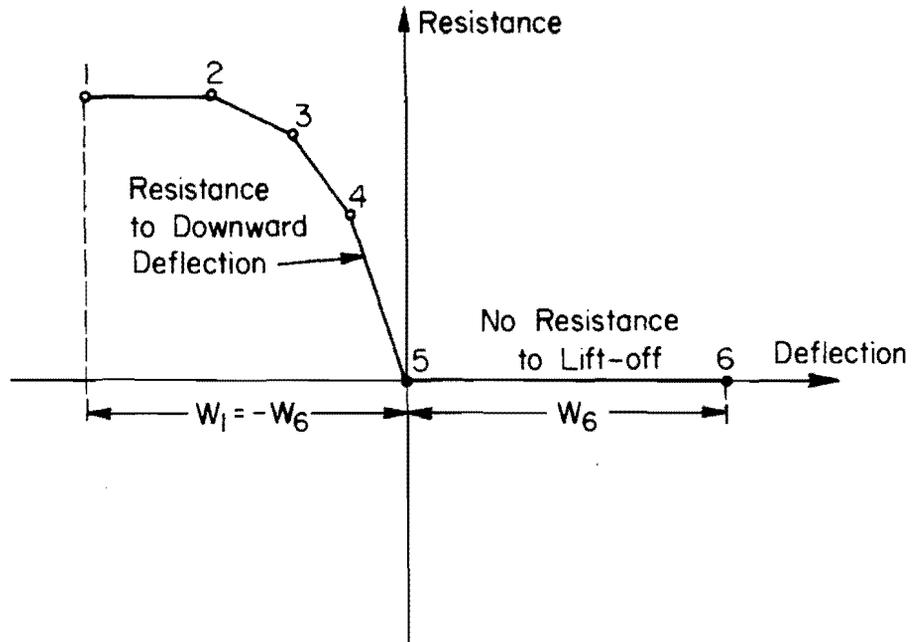
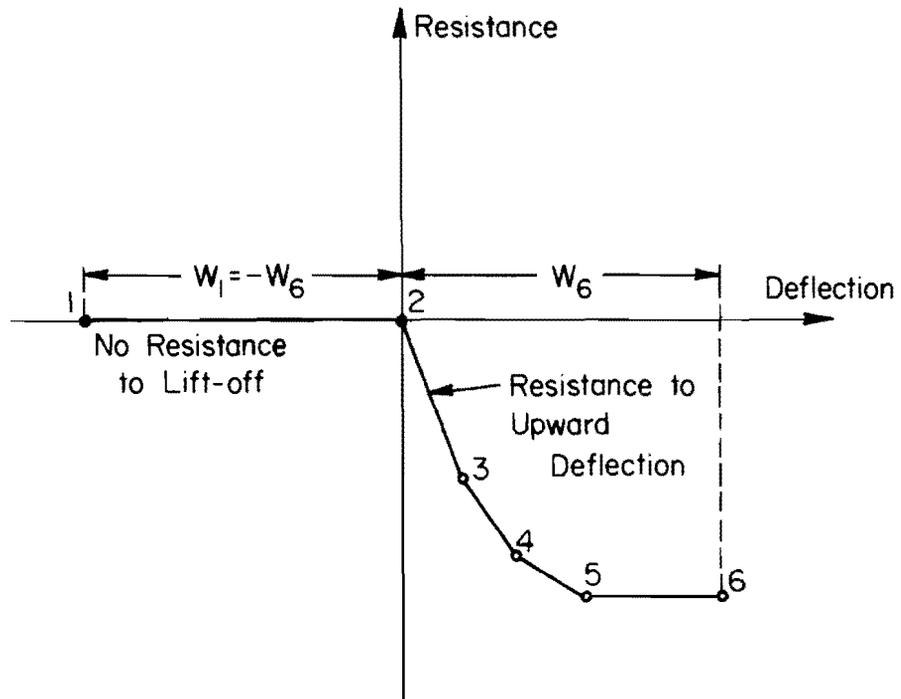


Fig 7. Symmetric resistance-deflection curve.



(a) Negative one-way resistance-deflection curve.



(b) Positive one-way resistance-deflection curve.

Fig 8. One-way resistance-deflection curves.

For the negative one-way support, resistance to deflection is developed only when deformable joints deflect in the negative or downward direction; for the positive one-way support, resistance to deflection is developed only when deformable joints deflect in the positive or upward direction. In order to be consistent with the multi-element model (see Chapter 3) used to simulate the nonlinear resistance-deflection curve of the supports, several limitations are placed on the nonlinear characterization:

- (1) The curve must pass through the origin.
- (2) The curve must be continuously concave as viewed from the horizontal axis, except that horizontal lines of zero stiffness can be input for representing ideally plastic behavior.
- (3) No softening of supports is permitted; that is, no reversal of slope sign of the segments on the support curve is permitted.

#### Stability of the Implicit Operator

Reference 10 has shown that when a uniform beam with well-defined boundary conditions under free vibration is analyzed using the implicit operator of the Crank-Nicolson form, the solution is stable for all positive values of  $EI$ ,  $\rho$ ,  $h$ , and  $ht$ .

If the same beam, with internal damping  $(D^i)_j$  lumped at joints, is under free vibration, the quadratic equation for evaluation of the stability criterion becomes

$$e^{2\phi} + e^{\phi} \left[ \frac{2q^2}{4(F+D)(\cos \beta_n - 1)^2 + q^2} \right] + \left[ \frac{4(F-D)(\cos \beta_n - 1)^2 + q^2}{4(F+D)(\cos \beta_n - 1)^2 + q^2} \right] = 0 \quad (2.16)$$

where

$$F = EI$$

$$D = (D^i)_j / ht$$

$$q^2 = 2\rho h^3 / h_t^2$$

The derivation of Eq 2.16 is included in Appendix C.

From Eq 2.16, the condition for bounded solutions as time approaches infinity becomes

$$16(F^2 - D^2)(\cos \beta_n - 1)^2 + 8Fq^2 \geq 0 \quad (2.17)$$

The above inequality is true, since  $D$  is usually less than  $F$  for a reasonable time increment length  $h_t$ , and the positive value of the term  $8Fq^2$  is always greater than the negative value of the term  $16(F^2 - D^2)(\cos \beta_n - 1)^2$  when  $h_t$  is extremely small.

For more complicated cases, such as a beam with internal damping, nonlinear spring supports, and rotational restraints under forced vibration, the analytical proofs for the stability of the implicit operator are not feasible. However, stable numerical solutions have been obtained for most complex practical problems that are physically stable. For beams with nonlinear supports under forced vibration, cautious selection of a reasonably small time increment length (for instance, less than 1/10 of the fundamental period) must be made, since the basic assumption of the Crank-Nicolson implicit formula is that the time-dependent forcing function and the time variant nonlinear springs are smoothly varied with time. Extremely small time increment length is not possible for most practical problems due to the present limitation of a maximum of 1000 time stations provided in Program DBC5.

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## CHAPTER 3. RESISTANCE-DEFLECTION CURVES FOR THE LATERAL SUPPORTS

Three types of lateral supports are considered in this work: linearly-elastic, nonlinearly-elastic, and nonlinearly-inelastic. Chapter 2 discusses how the nonlinear support characteristics can be reasonably described by nonlinear resistance-deflection curves which are approximated by a series of straight-line segments. This chapter introduces two multi-element models each of which is made up of several sub-elements that can be used to simulate the nonlinearly-inelastic characteristics of either a symmetric or a one-way resistance-deflection curve. A short discussion of the Baushinger effect (Refs 9 and 11) as it relates to the formation of the loading paths of nonlinearly-inelastic resistance-deflection curves is included. Finally, a systematic approach to tracing the loading paths of nonlinearly-inelastic resistance-deflection curves in dynamic problems is explained.

### Multi-Element Models

A multi-element model consisting of several parallel sub-elements for simulating the symmetric resistance-deflection curve is shown in Fig 9(a). Each sub-element has a linear spring connected in series with a Coulomb friction block. The sub-elements are assumed to behave as perfectly elastic-plastic and their resistance-deflection curves are shown in Fig 9(c). We see that it is possible to represent a symmetric resistance-deflection curve with a number of perfectly elastic-plastic simple elements in parallel. Buckling and fracture phenomena are not considered in this work. Now let us study the behavior of the sub-elements. If the multi-element model is starting to deform downward, all of the four sub-springs will deform elastically and the resistance will be built up to point A, as shown in Fig 9(b).

At this point the resistance force in the sub-spring  $S_1$  is just equal to the friction force created at the sliding surface of the Coulomb friction block. Beyond the point A the entire sub-element 1 will slide and no resistance, therefore, will be taken by this sub-element. If the model continues to deform downward the resistance will be taken by sub-springs  $S_2$ ,  $S_3$ , and  $S_4$

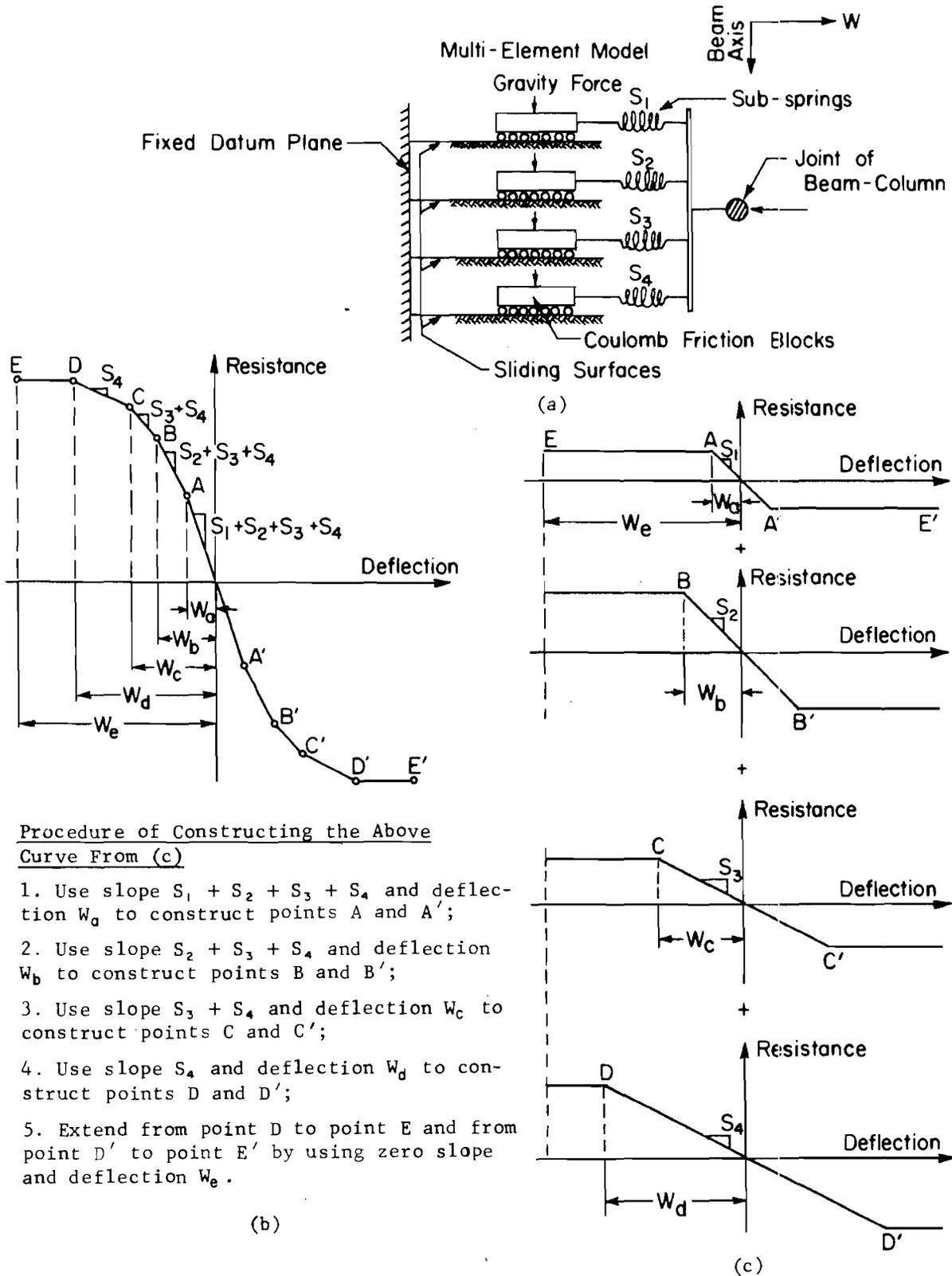


Fig 9. Multi-element model of symmetric nonlinear resistance-deflection curve of the support.

up to point B. At point B the sub-element 2 will start to slide. Beyond the point B the resistance will be taken by sub-springs  $S_3$  and  $S_4$  up to point C. At point C the sub-element 3 will also start to slide. Beyond point C the resistance will be taken only by sub-spring  $S_4$  up to point D. Finally at point D the entire support will start to yield plastically and offer no more resistance to the deflection. Based on the above assumption, a continuously-straight-line-segment curve which represents the nonlinear characteristics of the support can be easily obtained by the summation of all ideally elastic-plastic resistance-deflection curves of the sub-elements. The procedure of constructing such a curve is shown in Fig 9(b). Theoretically, the more segments there are in a curve, the smoother the representation of the nonlinear characteristics will be. Practically, however, a symmetric curve that consists of a maximum of 19 straight-line segments (9 sub-elements) is suitable to represent the nonlinear characteristics.

A multi-element model, as shown in Fig 10(a), is used to simulate the negative one-way resistance-deflection curve. The model is similar to the one used in the symmetric case except that there is no connection between the springs and the friction blocks. The spring-blocks work the same way as the ones used in the symmetric case when the deflection is downward (negative). While the deflection is upward (positive), the springs lift off the friction blocks and therefore no resistances can be built up in the sub-springs. Once the sub-springs are contacting the friction blocks again, the resistances in the sub-springs will again begin to build up and the spring-blocks act exactly as if the model is deflecting downward. The procedure of constructing a negative one-way curve from the individual sub-elements is shown in Fig 10(b).

The same multi-element model as shown in Fig 10(a) can be used to simulate the positive one-way resistance-deflection curve. In this case, the support model is to the right of (or above) the joint of beam-column; therefore, the sub-springs disconnect from the friction blocks and exhibit no resistances while the deflection is downward (negative).

#### Baushinger Effect of Nonlinear Resistance-Deflection Curve

If a specimen of mild steel is subjected to compression after a previous loading in tension, the applied compression stress, combined with the residual stress induced by the preceding tensile test, will produce yielding in the most unfavorably oriented crystals before the average compressive stress

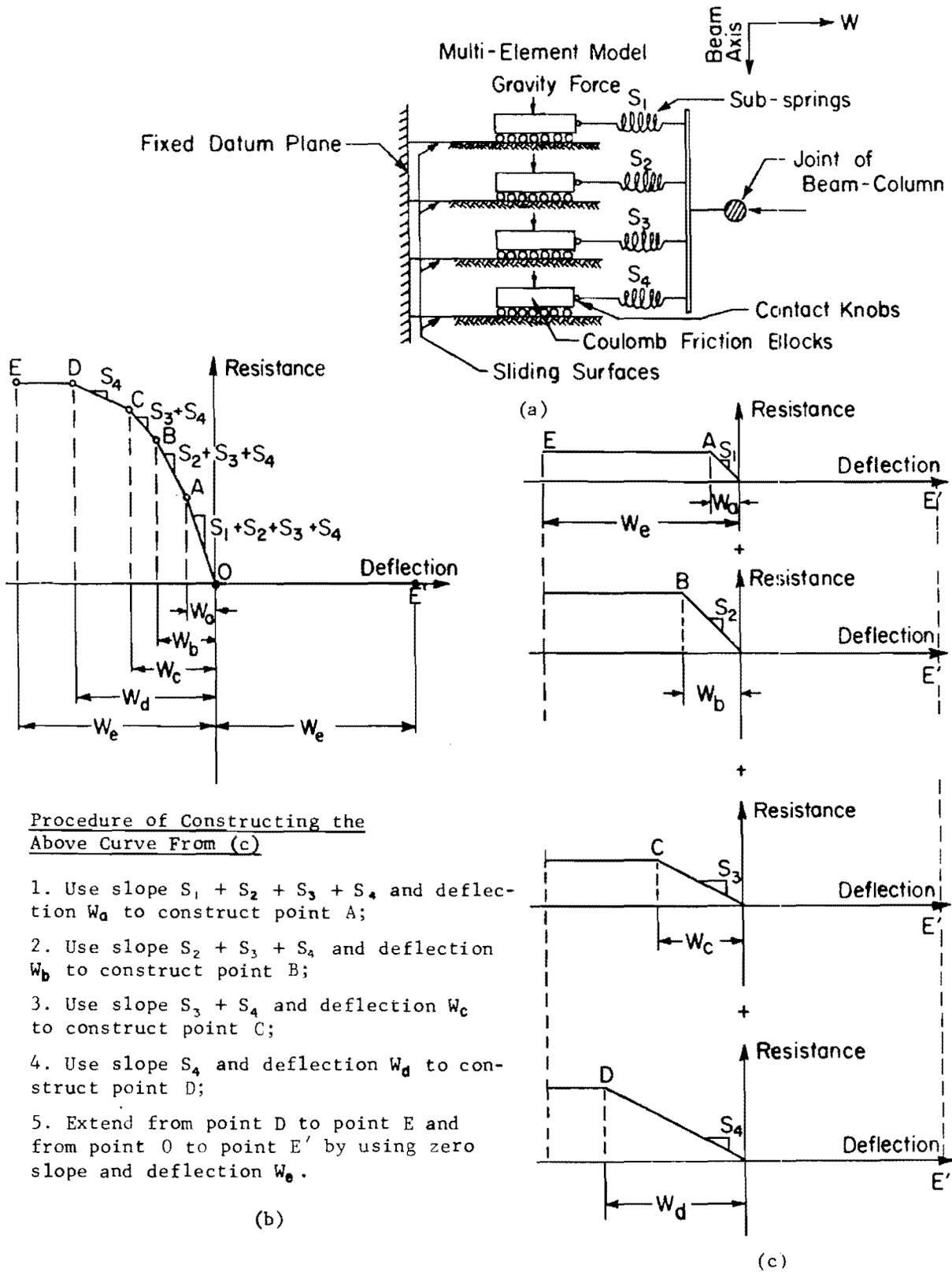


Fig 10. Multi-element model of one-way resistance-deflection curve of the support.

reaches the value at which slip bands would be produced in a specimen in its original state. Thus, the tensile test cycle raises the elastic limit in tension, but lowers the elastic limit in compression. This phenomenon is called the Baushinger effect (Refs 9 and 11).

From the multi-element models shown in Fig 9(a) and Fig 10(a), we can see that each individual sub-element is allowed to slide during the loading-unloading process. As a result of sliding or plastic deformation produced on loading, a system of residual resistances is introduced on unloading. Obviously, these residual resistances will influence the deformation produced by subsequent loads. Now let us study the so-called Baushinger effect based on the multi-element models shown in Fig 9(a) and Fig 10(a). For simplicity, consider a single spring support which has a nonlinear symmetric resistance-deflection curve with only three segments of resistances as shown in Fig 11(b). The nonlinear spring can be constructed by a multi-element model consisting of only two sub-elements each with an ideally elastic-plastic resistance-deflection curve as shown in Fig 11(c) and Fig 11(d). If a vertical load  $Q_1$  is applied downward to the spring, it will produce resistances  $R_1$  and  $R_2$  in the sub-springs  $S_1$  and  $S_2$ , respectively. Thus,

$$R_1 = \frac{Q_1 S_1}{(S_1 + S_2)} \quad (3.1)$$

$$R_2 = \frac{Q_1 S_2}{(S_1 + S_2)} \quad (3.2)$$

By gradually increasing the load  $Q_1$  we reach the condition at which the friction block in the sub-element 1 starts to slide while the sub-spring  $S_2$  continues to deform elastically. This condition corresponds to point A in Fig 11b. If at point A the deflection is equal to a value  $w_1$ , then the corresponding resistance  $R_A$  is found from the equation

$$R_A = (S_1 + S_2)w_1 \quad (3.3)$$

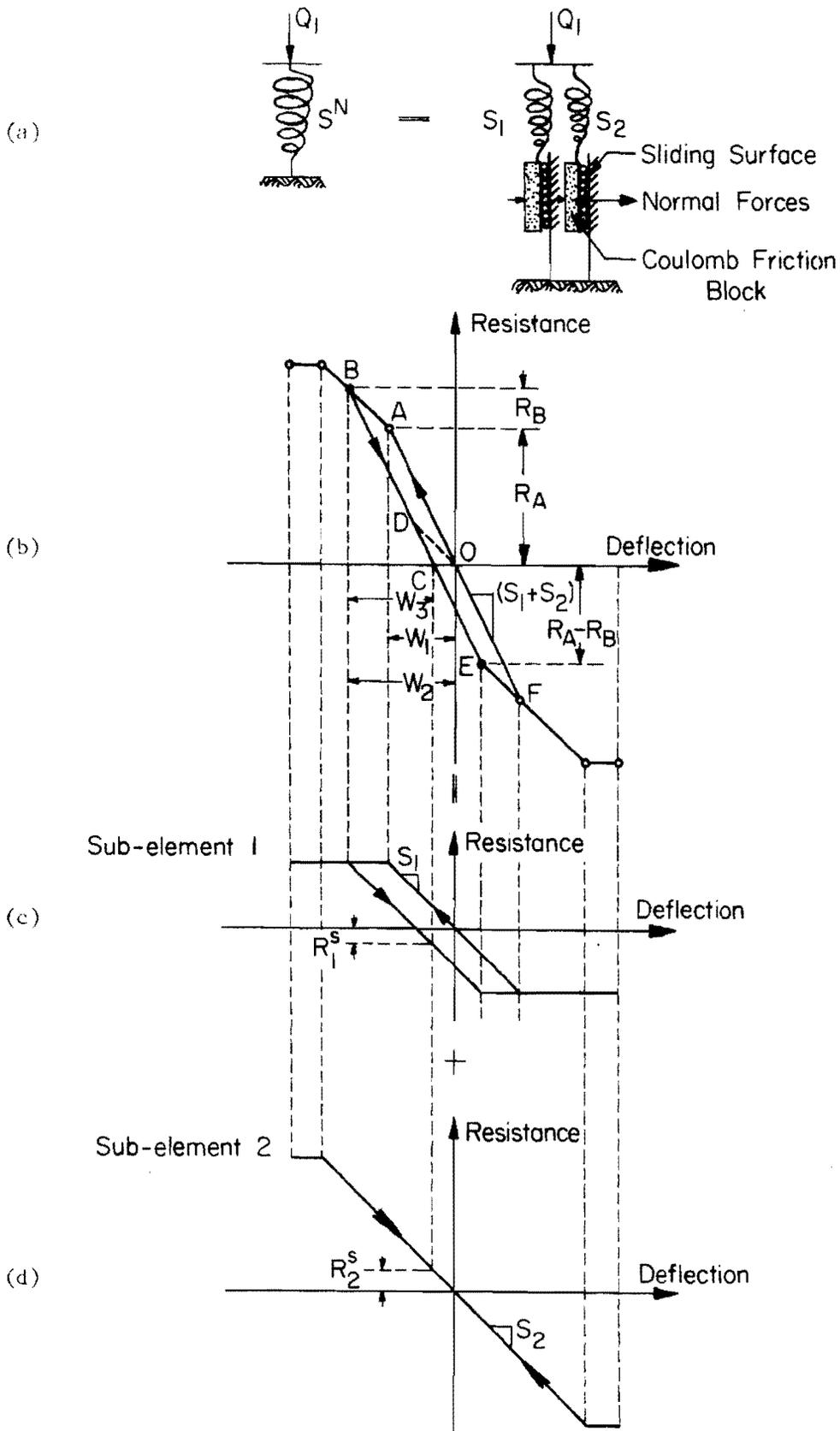


Fig 11. Baushinger effect of symmetric nonlinear resistance-deflection curve of the support.

If we continue to increase the load, the sub-element 1 will slide and exhibit no resistance, and the additional resistance  $R_B$  will be taken by the sub-spring  $S_2$ . The additional resistance  $R_B$  corresponding to the deflection  $w_2$  will be

$$R_B = S_2(w_2 - w_1) \quad (3.4)$$

The relation between  $R_B$  and  $w_2$  is shown in Fig 11(b) by the inclined line AB. If we begin unloading the model after reaching point B on the diagram, both sub-springs  $S_1$  and  $S_2$  will behave elastically and the relation between the removed resistance and the upward displacement will be given by Eq 3.3. In the diagram we therefore obtain the line BC parallel to OA, and the total vertical displacement upward during unloading is

$$w_3 = (R_A + R_B)/(S_1 + S_2) \quad (3.5)$$

Substituting Eq 3.3 into Eq 3.5 yields

$$w_3 = w_1 + \frac{R_B}{S_1 + S_2} \quad (3.6)$$

From Eq 3.4 we obtain

$$w_2 = w_1 + \frac{R_B}{S_2} \quad (3.7)$$

Comparing Eqs 3.6 and 3.7, we find that  $w_2$  is greater than  $w_3$ . We see that because of the sliding of sub-element 1, the model does not return to its initial state and the permanent set  $\overline{OC}$  is produced. The magnitude of the permanent set is found from the equation

$$\overline{OC} = w_2 - w_3 = R_B \left[ \frac{S_1}{S_2(S_1 + S_2)} \right] \quad (3.8)$$

Because of this permanent set there will be positive residual resistance  $R_2^S$  in the sub-spring  $S_2$ , and this resistance will be balanced by the negative residual resistance  $R_1^S$  in the sub-spring  $S_1$ . The magnitudes of  $R_1^S$  and  $R_2^S$  can be shown graphically in Fig 11(c) and Fig 11(d), respectively.

To show the Baushinger effect let us consider a second cycle of loading. At small values of load both sub-springs deform elastically, and in Fig 11(b) the second loading process will begin at point C and proceed along the straight line CB. When we reach point D the sub-spring  $S_1$  will be relieved of the residual negative resistance  $R_1^S$  and during further loading will have positive resistance. At point B the positive resistance in the sub-spring  $S_1$  will equal the friction force created by the friction block, and sliding of the sub-element 1 will begin. It is seen that the sliding resistance of the sub-element 1 is raised because of the residual resistances. The sliding resistance was at point A with resistance  $R_A$  in the first cycle and at point B with resistance  $R_A + R_B$  in the second cycle. The process of unloading in the second cycle is perfectly elastic and follows the line BC.

Let us now reverse the direction of the force and apply an upward load  $Q_1$ . The deformation will then proceed along the straight line CE, which is a continuation of line BC. At point E the total negative resistance is  $R_A - R_B$ , and the negative resistance in the sub-spring  $S_1$  is equal to the friction force created by the friction block. As the load is increased, the deformation proceeds along the line EF. If the model is unloaded after reaching point F, it will return to the initial state represented by point O. It is seen that by loading the model downward to point B, the positive sliding resistance in the sub-spring  $S_1$  was increased from  $R_A$  to  $R_A + R_B$ . At the same time the negative sliding resistance in the sub-spring  $S_1$  was diminished from  $R_A$  to  $R_A - R_B$ . This illustrates the Baushinger effect. The area of the parallelogram OABCEFO gives the amount of mechanical energy lost per cycle. The same concept discussed above can be expanded to a model consisting of any number of sub-elements.

For a negative one-way resistance-deflection curve as shown in Fig 12b, the Baushinger effect for this model can be illustrated by the loading paths indicated by the parallelogram OABCO and the area defined by OAB'DEFO. Notice that unless the model reaches the condition at which both sub-elements are sliding during the loading process (point D) there will be no permanent set after unloading from point B in the diagram. This is because the sub-spring

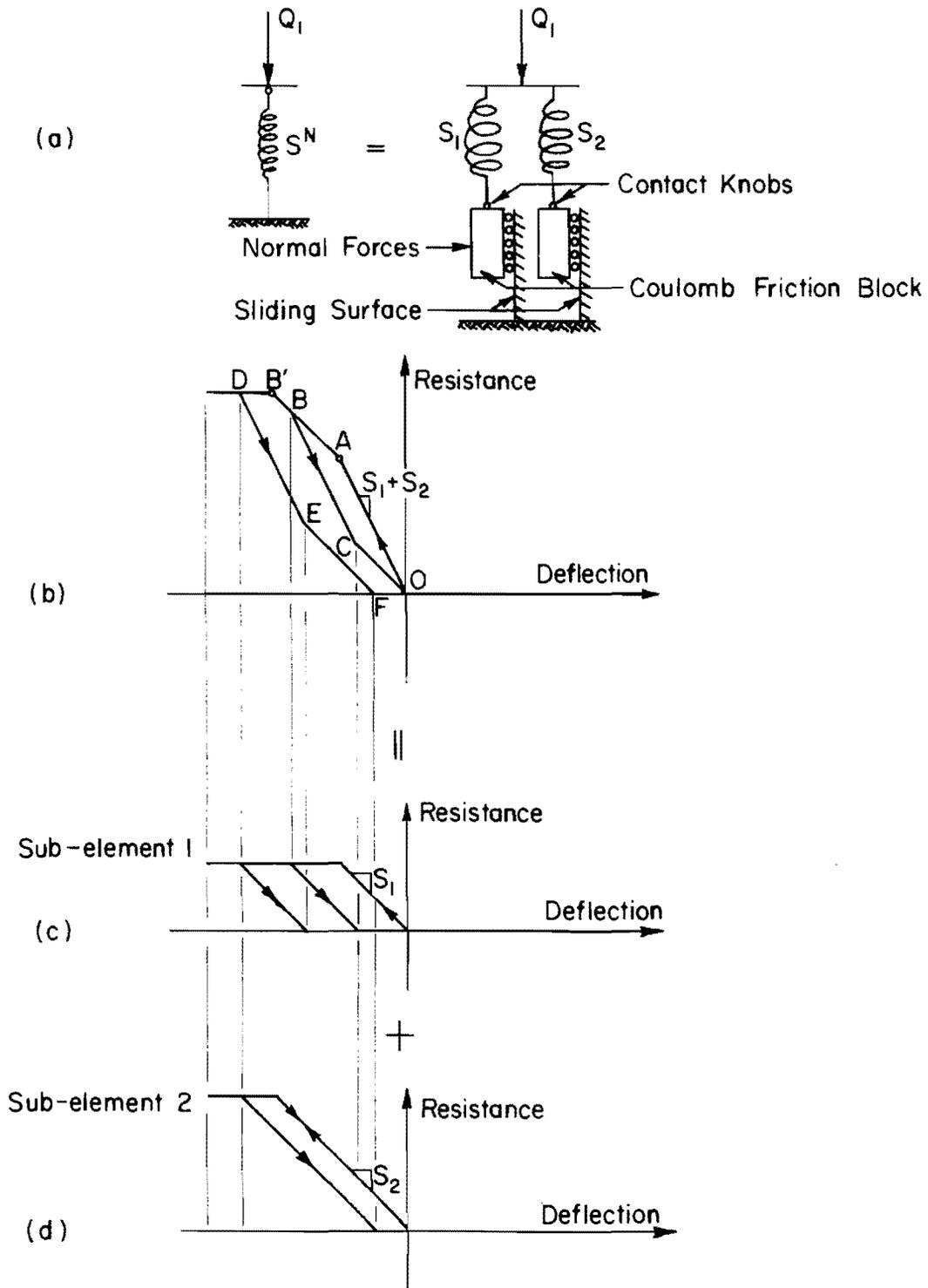


Fig 12. Baushinger effect of negative one-way resistance-deflection curve of the support.

$S_1$  and the friction block are disconnected after reaching the point C in the diagram, and hence no residual resistance can be built up in the sub-spring  $S_1$ . With further unloading the deformation proceeds along the line CO and the model will return to the initial state represented by point O. At this point, if we reverse the direction of load, the deformation will proceed along the new curve line OCBB'D. This is because the sub-spring  $S_1$  lifts off the friction block before reaching the point C.

It is seen that because the sub-springs are able to lift off the friction blocks, no negative resistances can be built up in the sub-springs and therefore the loading paths can generate only on the negative side of deflection. The above concept also can be expanded to the model which has more than two sub-elements. The Baushinger effect of a positive one-way resistance-deflection curve is similar to a negative one-way resistance-deflection curve, except that the loading paths can generate only on the positive side of the deflection.

A general procedure for tracing the loading paths of either a symmetric nonlinear or a one-way resistance-deflection curve of any support which has more than two sub-elements will be discussed below.

#### Systematic Approach for Following Loading Paths

In Figs 9(c) and 10(c), we see that the multi-element models for simulating the nonlinearly-inelastic supports can be represented by a number of ideally elastic-plastic sub-elements. Although it is possible to retain the ideally elastic-plastic resistance-deflection curves of each individual sub-element in the program, a more efficient and easier way is to retain the overall curve which is obtained by the summation of all ideally elastic-plastic resistance-deflection curves of the sub-elements as shown in Figs 9(b) and 10(b).

In Program DBC5, nonlinear supports can be described for any beam station. The nonlinear resistance-deflection curve of the support is described by a simple tabular input which generates a curve of straight line segments with a series of point numbers. After defining the resistance-deflection curve by a series of points, which are numbered in order, the resistance and deflection of each point must be retained sequentially in the variables of Q and W, respectively. The retained values of the deflection w must be increasing positively, while that of the resistance Q must be decreasing or equal for the consecutive points.

A typical symmetric nonlinear resistance-deflection curve is shown in Fig 13a.

In order to be able to systematically generate the loading paths of the curve during the loading-unloading process, the slope of each segment, the projected horizontal (deflection) value and the projected vertical (resistance) value of each segment must be retained sequentially in the variables of  $S$ ,  $\Delta w$ , and  $\Delta Q$ , correspondingly. Notice that the retained values of the variable  $S$  must be in descending order.

Now let us study the procedure of constructing the new path of the original resistance-deflection curve shown in Fig 13(a), when the deformation of the support starts to reverse its direction at point A as shown in Fig 13(b). The systematic procedure consists of (1) drawing line AB parallel to segment 4-5 so that the projected value of line AB on the horizontal axis is equal to  $\Delta w_1^T$  and the projected value on the vertical axis is equal to  $\Delta Q_1^T$ , (2) drawing line BC parallel to segment 3-4 so that the projected value of line BC on the horizontal axis is equal to  $\Delta w_2^T$  and the projected value on the vertical axis is equal to  $\Delta Q_2^T$ , (3) connecting point C to point 7 on the original curve, and (4) renumbering the new path of the resistance-deflection curve as 1, 2, ... 7 as shown in Fig 13(b).

We see that the number of points on the new curve has been reduced to 7 from 8, which is the previous number of points on the original curve. If the deformation of the support again starts to reverse its direction at point A', shown in Fig 13(b), another new path of the resistance-deflection curve, numbered 1, 2, ... 6, will be generated by a procedure similar to that described above, except that three lines, A'B', B'C', and C'D', are drawn before the new curve connects to the previous curve.

For a typical negative one-way resistance-deflection curve, as shown in Fig 14(a), the logic of tracing two new curves can be shown graphically in Fig 14(b). The procedure is similar to the one used in the symmetric case. It is seen that loading paths are generated only by downward (negative) deflections and positive resistances. We also see that the second adjusted curve is generated when the deformation of the support, after going back to the initial state at the end of the first loop, continues to proceed along the line 5CBA2A' and starts to reverse its direction at point A'. The deformation at point A' is greater than the deformation at point A, where the first

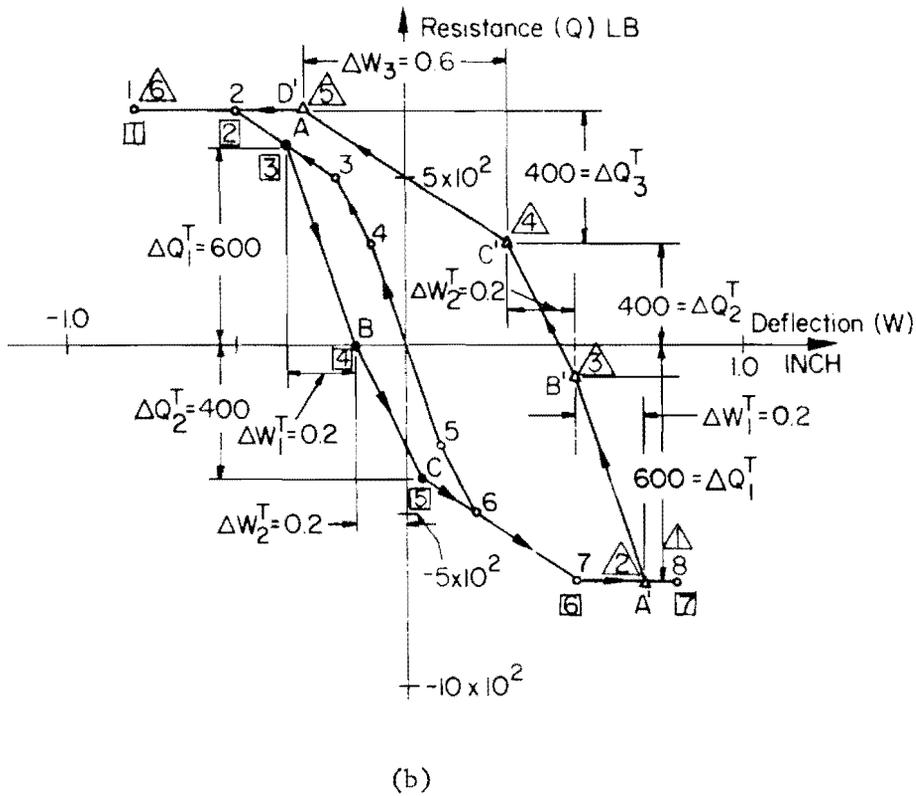
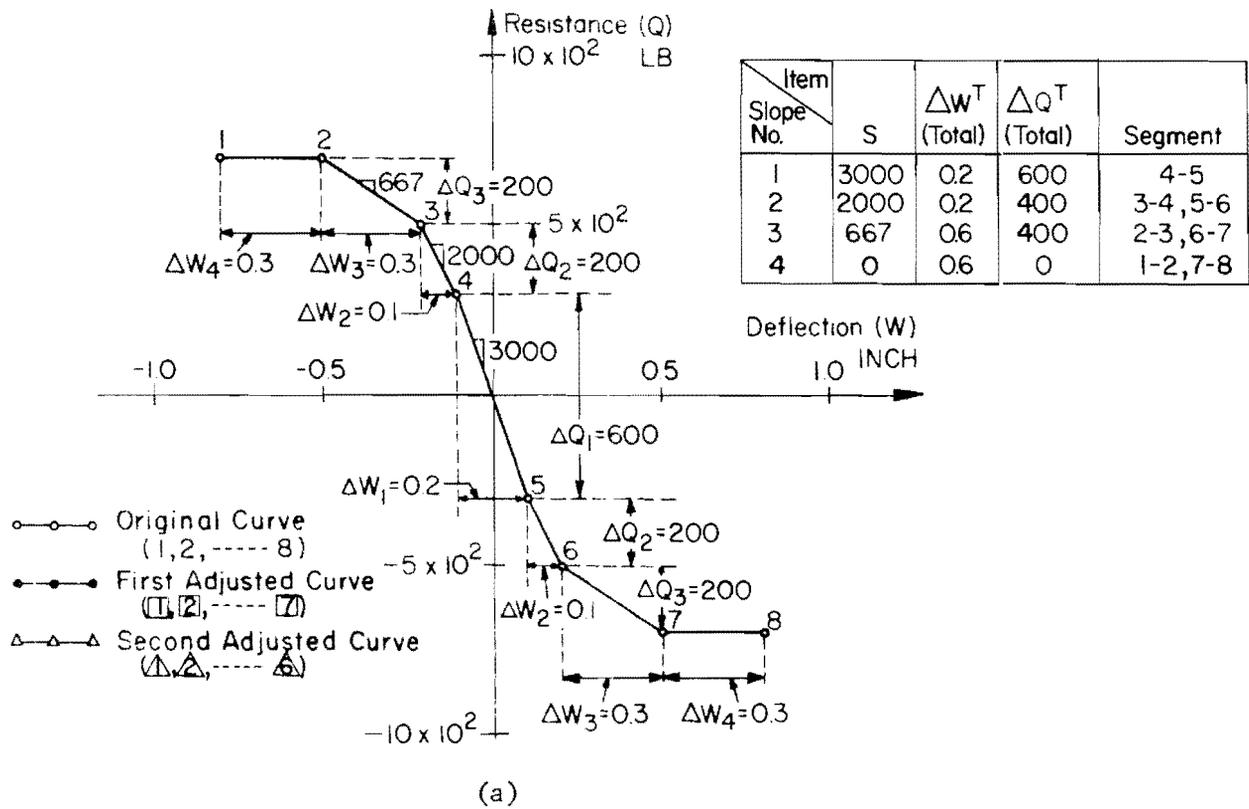


Fig 13. Logic of tracing hysteresis loops for symmetric nonlinear resistance-deflection curve.

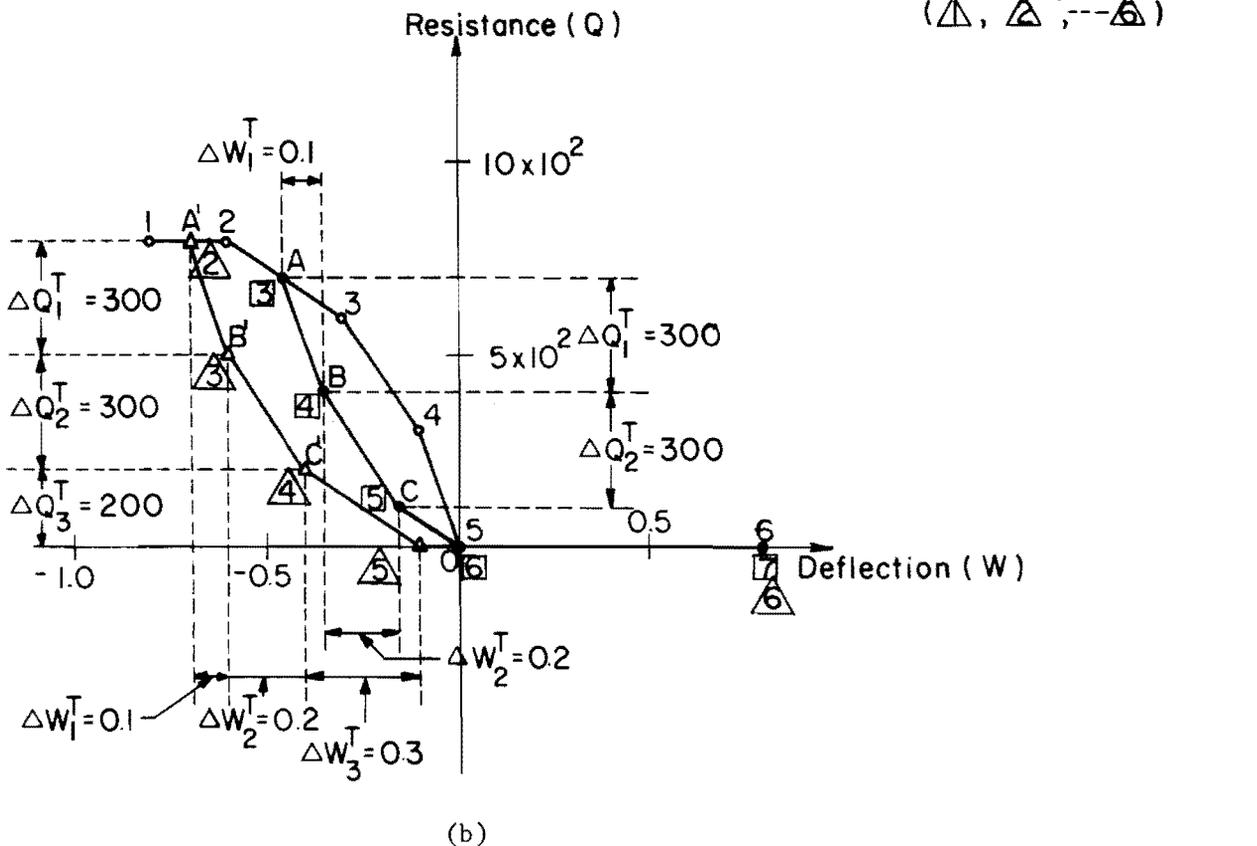
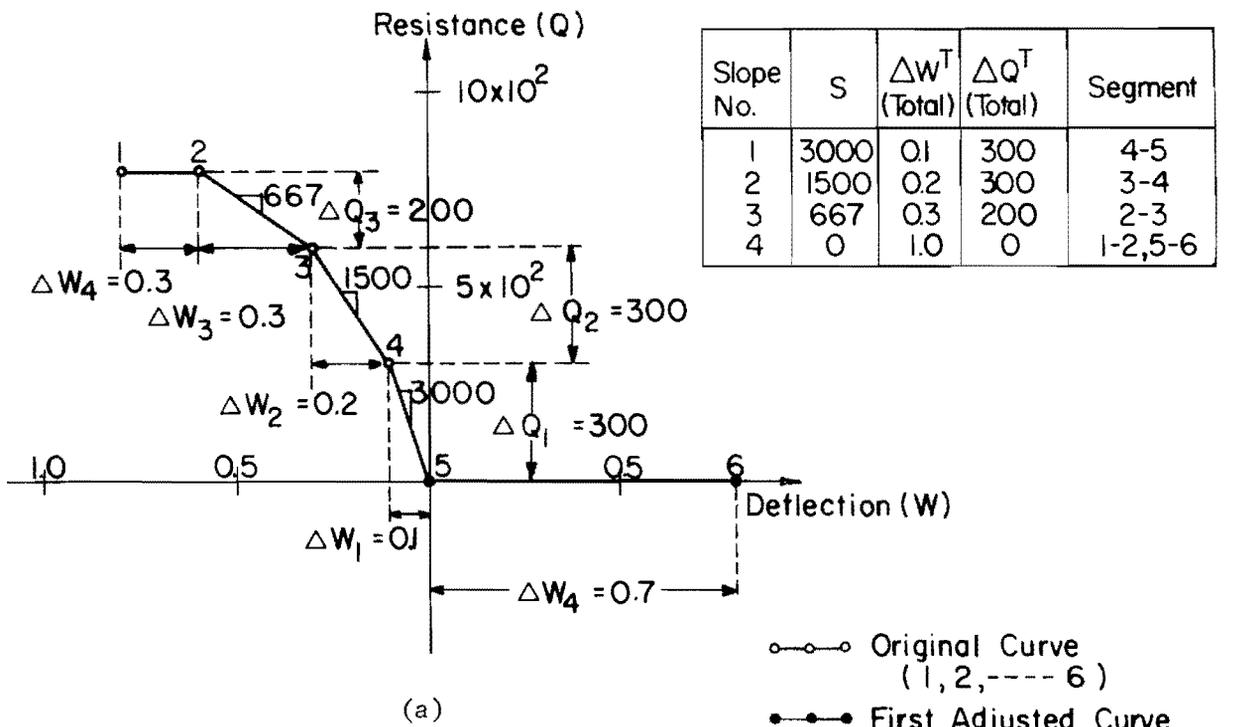


Fig 14. Logic of tracing hysteresis loops for a negative one-way resistance-deflection curve.

adjusted curve started. The logic of tracing the loading paths of a positive one-way resistance-deflection curve is similar to the one used in a negative one-way resistance-deflection curve, except that the loading paths are on the positive side of the deflection.

A detailed flow chart relating to the FORTRAN logic, which is used in the program, of tracing the loading paths of the nonlinearly-inelastic resistance-deflection curves of the support will be found in Appendix F.

## CHAPTER 4. DESCRIPTION OF PROGRAM DBC5

### General

Program DBC5 is written in FORTRAN language for Control Data Corporation (CDC) 6600 computers. With minor changes, the program would be compatible with IBM 7090 computers and with other systems. Four available FORTRAN sub-routines are included in the program:

- (1) INTERP3 (Ref 5) for interpolating the input data,
- (2) TICTOC (Ref 5) for evaluating the computation time,
- (3) SPLOT9 for plotting the results by using the printer plotting method, and
- (4) ZOT1 for plotting the results by using the microfilm or ball-point paper plotting method.

The program uses in-core storage which requires 58,500 words. Although the use of auxiliary tapes will reduce the storage, it raises the computation time to a great extent due to tape operations; therefore, the in-core solution method has been chosen rather than using many auxiliary tapes. A summary flow chart of the program is presented in Fig 15. The definitions of symbols used in the program and a listing of the program with several general flow charts are included in Appendix E and Appendix F. Subroutines SPLOT9 and ZOT1 were developed particularly for the highway research projects at The University of Texas at Austin.

### Procedure for Data Input

A guide for data input is included in Appendix D. The guide is designed so that additional copies may be furnished as separately bound extracts for routine use. A parallel study of the guide will help the reader to understand the following discussion.

Any number of problems may be stacked and run together. The sequence of problems is preceded by two cards which describe the run. The first card of each problem contains the problem number and a brief description of the problem

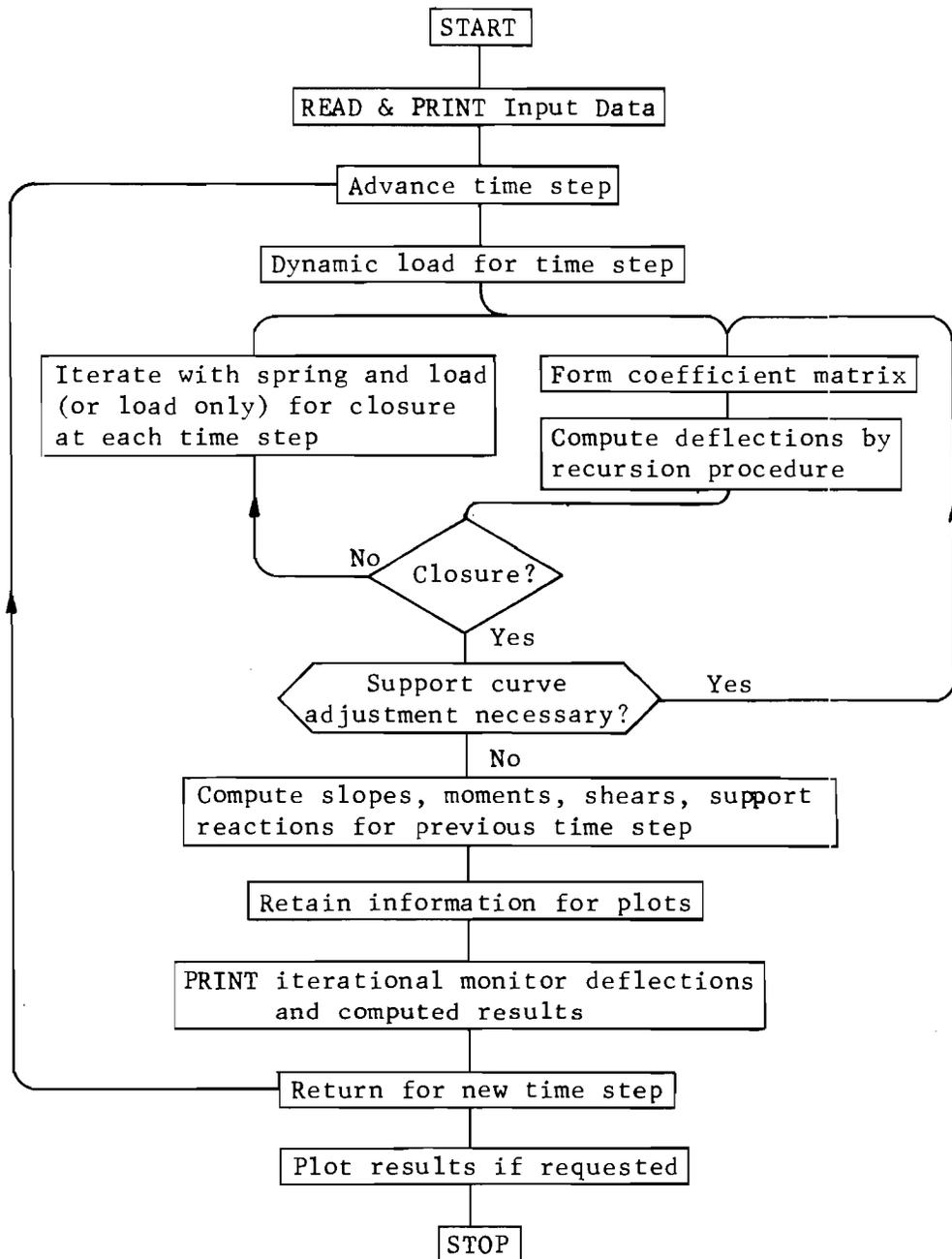


Fig 15. Summary flow chart of DBC5 program.

The program continues working problems until a blank problem number or an error is encountered; then the run is terminated.

### Tables of Input Data

Table 1 is the data-control table. It consists of a single card which must be input for each problem. The number of cards in the remaining tables and the hold options of these tables are specified in this table. The hold options for the various tables are independent of each other, but care must be exercised to insure that the data in the various tables are compatible. For example, if a previous 40-increment problem had a distributed load from station 0 to station 40, then the user should not change the total number of increments in the new problem to some number less than 40 unless he erases the loads past the end of the new beam.

Table 2 contains

- (1) number of increments into which the beam-column is divided,
- (2) beam increment length,
- (3) number of increments into which the time axis is divided,
- (4) time increment length,
- (5) printing option for the computed results,
- (6) number of monitor beam stations which are requested for printing the computed results,
- (7) iteration switch to show whether or not the problem has nonlinear supports,
- (8) number of monitor beam stations which are requested for printing the iteration data,
- (9) option to choose the method of plotting the results, and
- (10) option of using lines or points for the plots.

If the number of time increments is 0, the problem is considered to be a static case and the complete results of the static solution will be printed at every beam station. In this case, two kinds of plots, which have either deflection or moments for the vertical axis and beam stations for the horizontal axis, may be requested by inputting "0" for the time station in Table 9.

For any station in the beam, a deflection, slope, or both, in either the initial or the permanent condition, may be specified in Table 3. If an initial condition of deflection, slope, or both is specified, it will be disregarded

after the dynamic solutions are started. A specified deflection is equivalent to a single lateral support that is very stiff and is unable to deform freely.

A fixed-end support can be simulated in the computer by the specification of a permanent zero deflection and a permanent zero slope at the same beam station. The ability to specify a permanent deflection other than zero provides the user with a simple method of studying problems in which the supports settle. The ability to specify an initial deflection other than zero, on the other hand, provides the user with capability to study dynamic problems with initial displacements. Due to the method of simulation (see Fig 5), the specified slopes and deflections must conform to the following minimum spacing requirements:

- (1) A slope may not be specified closer than three increments from another specified slope.
- (2) A deflection may not be specified closer than two increments from a specified slope, except that both a slope and deflection may be specified at the same beam station.

Slope and deflection conditions may be specified at no more than 20 beam stations. Each specification requires a separate card. The cards may be stacked in any order within the table.

Fixed loads and restraints are described in Table 4. The input values are beam stiffness, fixed static loads, linear support springs, static axial thrusts, rotational restraints, and applied couples. Couples and rotational restraints appear only as concentrated effects, while axial thrusts are usually distributed. The remaining values may be either concentrated or distributed. The method used for the description of distributed data is illustrated in Appendix D. All of the input values of Table 4 are accumulated algebraically in storage. Therefore, there are no restrictions on the order of the cards, except that within a distribution sequence, the beam stations must be in ascending order. Axial thrusts must be described in the same manner as other values in this table because there is no provision in the program for automatically distributing the internal effects of an externally applied axial thrust. For example, an axial thrust applied to the ends of the beam-column must be specified as either an axial tension or an axial compression at each interior beam station. The number of cards accumulated in Table 4 cannot exceed 100.

Table 5 is used to describe the lumped mass densities, the lumped internal damping factors, and the lumped external damping factors at beam stations. Data in this table are input the same as in Table 4. The maximum allowable number of cards accumulated in this table is 100.

Time-dependent axial thrusts are input in Table 6 as values applying to designated bars. The values of time-dependent axial thrusts specified in this table may vary linearly both in the beam axis direction and in the time axis direction. There are no restrictions on the order of cards, except that beam stations which are stated as the starting station and the ending station in the same card must be in ascending order. A concentrated effect of time-dependent axial thrust at a beam station is not permitted to be stated in this table since it would have no physical meaning.

The method used for the description of distributed data is also illustrated in Appendix D. The number of cards accumulated in this table cannot exceed 100.

Table 7 is used to describe time-dependent lateral loadings applied at the station points. The variation and the distribution of the input data are the same as in Table 6, except that the concentrated effect of the lateral load at a single station is allowed in this table. The number of cards accumulated in this table cannot exceed 100.

Nonlinear resistance-deflection curves of the supporting beam stations are input in Table 8. Every particular nonlinear support curve is described by three consecutive cards. The values input on the first card are resistance-multiplier, deflection-multiplier, number of points, symmetry option, deflection-tolerance, and resistance-tolerance. The values input on the second card are resistance values which are multiplied by the resistance-multiplier to obtain the final resistances of each point on the curve. The values input on the third card are deflection values which are multiplied by the deflection-multiplier to obtain the final deflections of each point on the curve.

There are no restrictions on the order of support curves, except that within any distribution sequence, the beam stations must be in regular order. More than one curve may be placed at a beam station. The maximum number of support curves cannot exceed 20 (60 cards).

Four kinds of plots may be requested by inputting the name of the horizontal axis, the name of the vertical axis, the particular beam or time station, and the multiple plot switch in Table 9. As many as five of the same

kinds of plots, each of which is described by one separate card, may be superimposed by using the multiple plot switch. There are no restrictions on the order of cards, except that beam or time stations must be in regular order within a sequence of the same kind of plots. Superimposed plots must all be of the same kind. The maximum number of cards input in this table cannot exceed 10.

Three kinds of plotting methods, namely printer plots, microfilm plots, and ball-point paper plots, are built into the program. The switch which tells the program to choose one of the above three methods for plots is input in Table 2. Various plot options are illustrated in example problems described in Chapter 5.

### Error Messages

Program DBC5 provides many checks for common types of data errors as well as for particular errors which occur due to either violations of FORTRAN logic or solution errors considered by the program.

Conditions which are considered to be of the type of data errors are

- (1) the allowable number of cards for an input table is exceeded;
- (2) a slope or deflection is specified too close to another slope or deflection;
- (3) data are input at stations beyond the end of the beam-column;
- (4) the station numbers in a distribution sequence are out of order;
- (5) the symmetry option is not equal to 1 when only one point is input on a support curve;
- (6) the final values of deflection and resistance, which are input for defining the points on a support curve, are improperly specified;
- (7) the allowable number of nonlinearly supported beam stations is exceeded;
- (8) the number of cards input in Table 8 is not zero when the problem is a linear case; and
- (9) repeated data are input in Table 9.

The conditions which are considered solution errors are

- (1) calculated displacements exceed maximum allowable deflection specified by the user,
- (2) calculated displacements exceed limits of support curves during the iteration process, and
- (3) solution does not close within the specified number of iterations.

An error message which defines the error will be printed if any of the above conditions is encountered.

In addition to the specific error messages, a general purpose error message is provided for a number of unlikely errors. If the message "UNDESIGNATED ERROR STOP" is printed, the user must investigate the program to determine what caused the error. Any error detected by the program will cause the entire run to be abandoned.

### Description of Problem Results

The output of results is arranged so that the input quantities of Tables 1 through 9 are available for all problems and are printed with explanatory headings.

Table 10 presents the calculated results which are tabulated according to the following order:

- (1) monitor deflections during the iteration process of the static solutions (none if the problem is a linear case or if the number of monitor beam stations for printing the iteration data is zero),
- (2) complete results of the static solutions,
- (3) monitor deflections during the iteration process of the dynamic solutions at time station 0 (see explanation in the parentheses of procedure 1),
- (4) complete results of the dynamic solutions at time station 0,
- (5) monitor deflections during the iteration process of the dynamic solutions at this time station (see explanation in the parentheses of procedure 1),
- (6) monitor or complete results of the dynamic solutions at this time station, and
- (7) repeat outputs of procedures 5 and 6 until the last time station specified in this problem is encountered.

The proper headings are printed at the top of the outputs of each procedure described above.

If printout of the calculated results are not requested, the message "PRINTING RESULTS IS NOT REQUESTED" will be printed under the headings of Table 10.

### Plots of Results

Options are available which allow the user to obtain plots of the deflections or moments along either beam or time axis. Three kinds of plotting methods, namely printer plots, microfilm plots, and ball-point plots, are available in the program. Both microfilm and ball-point plots fit in an 8-inch by 9-inch space. The optimum scales for the plots, both horizontal and vertical, are automatically selected by the program. Two fictitious points, one with the minimum horizontal and vertical values of all the points to be plotted on a single frame and the other with the maximum horizontal and vertical values, are plotted on the microfilm or ball-point plots for the purpose of choosing the proper vertical scale. These two fictitious points are not in the stored values of the points to be plotted on this frame and, therefore, should be disregarded. An example plot of the deflections along the time axis for inelastically supported mass under free vibration is shown in Fig 16. The plot in Fig 16 is made using the ball-point plotting method; the problem number and the identification of the plot variables are printed beneath the plot. The plot variables which identify either a microfilm plot or a ball-point plot include three sets of characters. The first set to the right of the problem number is one of the following four symbols: DVT or MVT (deflection or moment along time axis for a particular beam station), DVB or MVB (deflection or moment along beam axis for a particular time station). In the center is either BEAM STA = or TIME STA =. On the right are the station numbers for which the deflections or moments are plotted. The plot in Fig 16 is from Example Problem 1, described in Chapter 5. Another example plot, which is plotted using the microfilm plotting method, is shown in Fig 17. In this example, the moments along the time axis for the mid-point of a three-span beam on which an AASHO standard 2-D truck is moving with a speed of 60 mph is plotted on the microfilm, which then can be developed on an enlarged magnetic print. The plot is from Example Problem 3, described in Chapter 5. Many more ball-point plots which have more than one curve plotted on one frame can be found in Example Problem 5, described in Chapter 5.

A typical example plot which is plotted by using the printer plotting method is shown in Fig 18. The printer plotting method plots the horizontal axis vertically on the sheet. The plot is from Example Problem 6, described in Chapter 5. Different symbols (as many as five) are used on each frame of

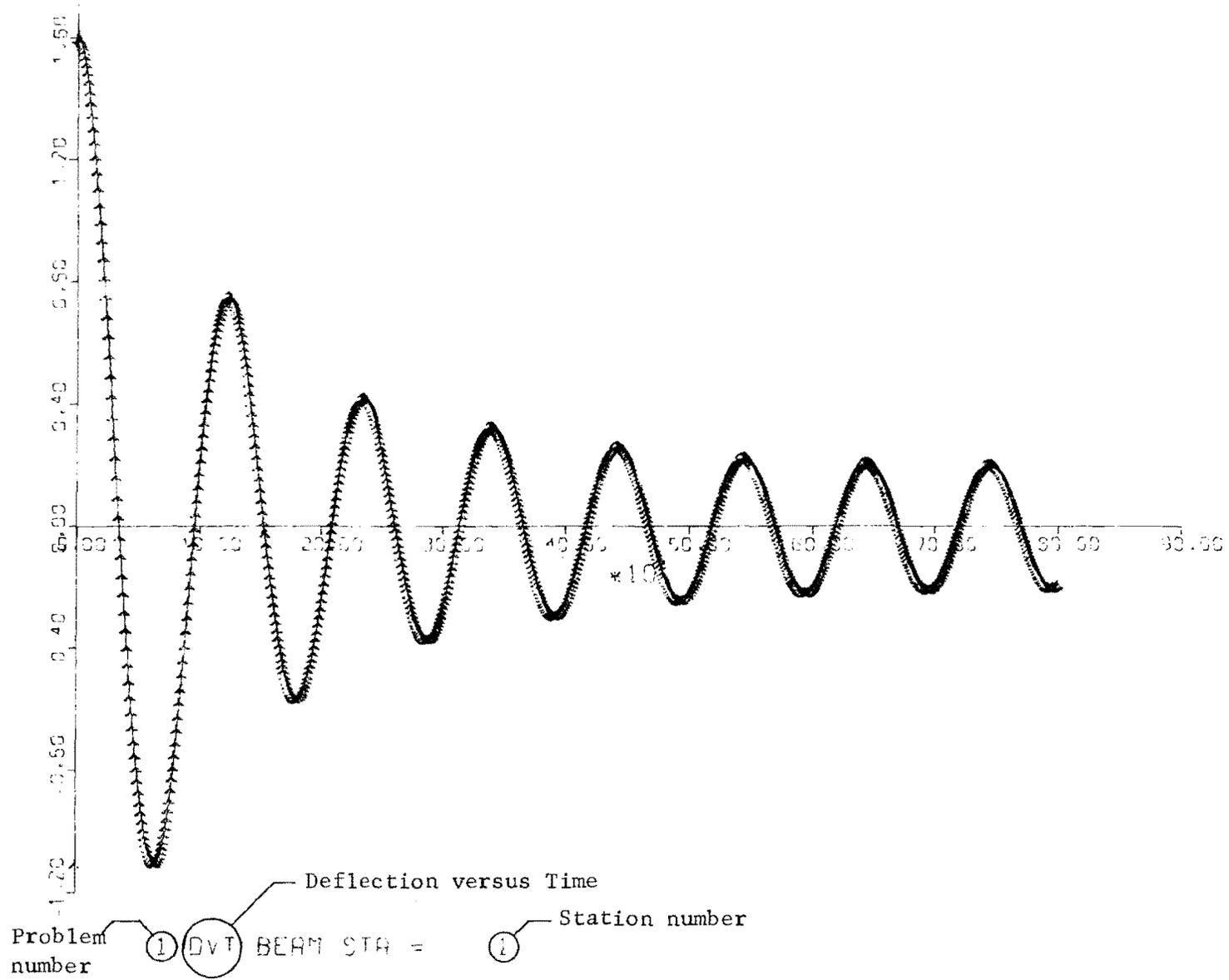


Fig 16. A typical ball-point paper plot.

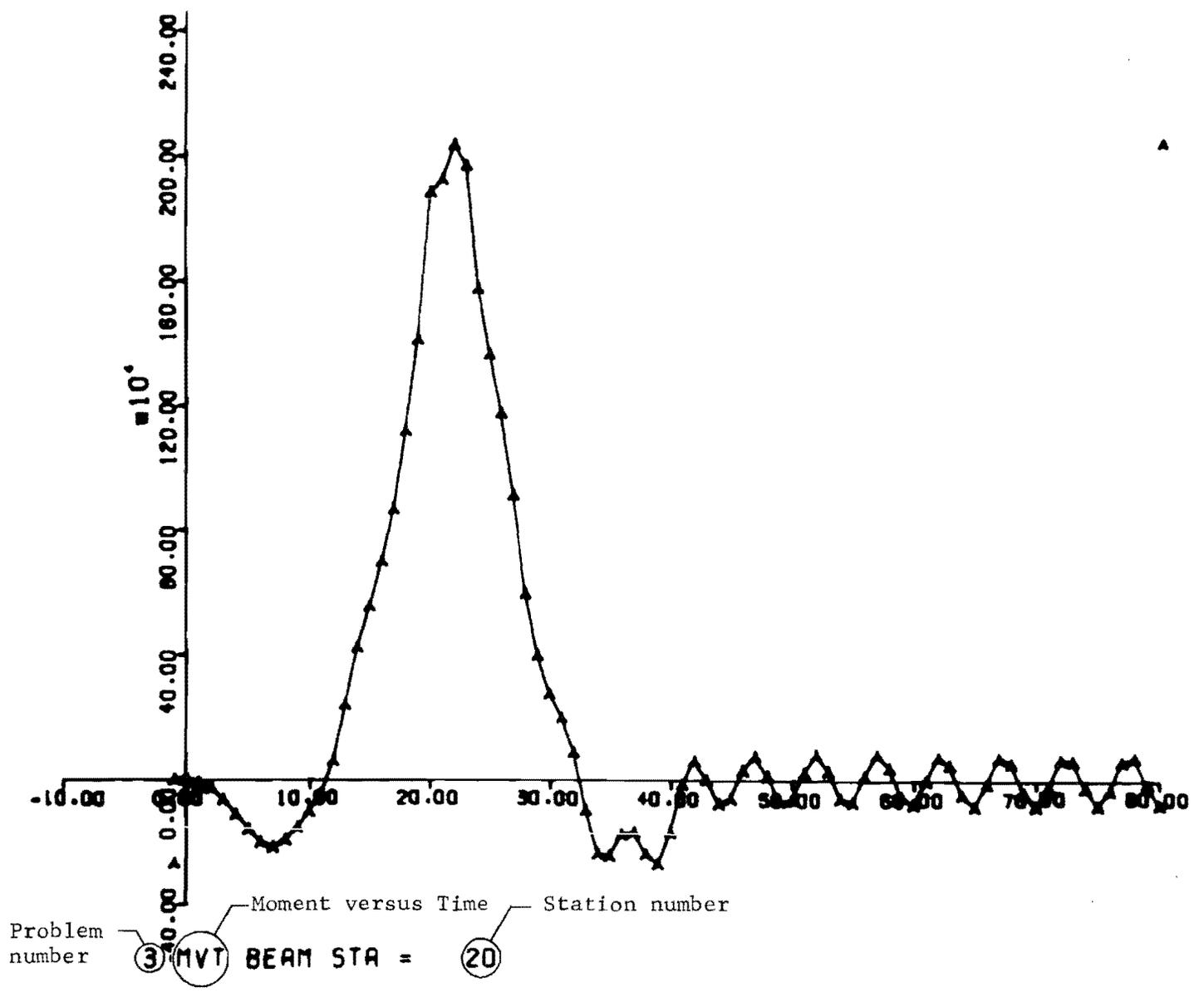


Fig 17. A typical microfilm plot.

PROGRAM DRCS - MASTER - JACK CHAN - MATLOCK - DECK1-REVISION DATE = 26 JUN 71  
 EXAMPLE PROBLEMS FOR PROGRAM DRCS BY JACK CHAN JUNE 1971  
 DYNAMIC BEAM-COLUMN PROGRAM ( 5-1-5 IMPLICIT OPERATOR )

PROB (CONTD)

6 THREE-SPAN BEAM WITH ONE-WAY AND SYMMETRIC NONLINEAR SUP LOADED BY T.P  
 \*\*\*\*\* PLOT OF DEFLECTION VS TIME FOR BEAM STATIONS OF: \*\*\*\*\*

1 CURVE (\*) = 8  
 2 CURVE (+) = 17

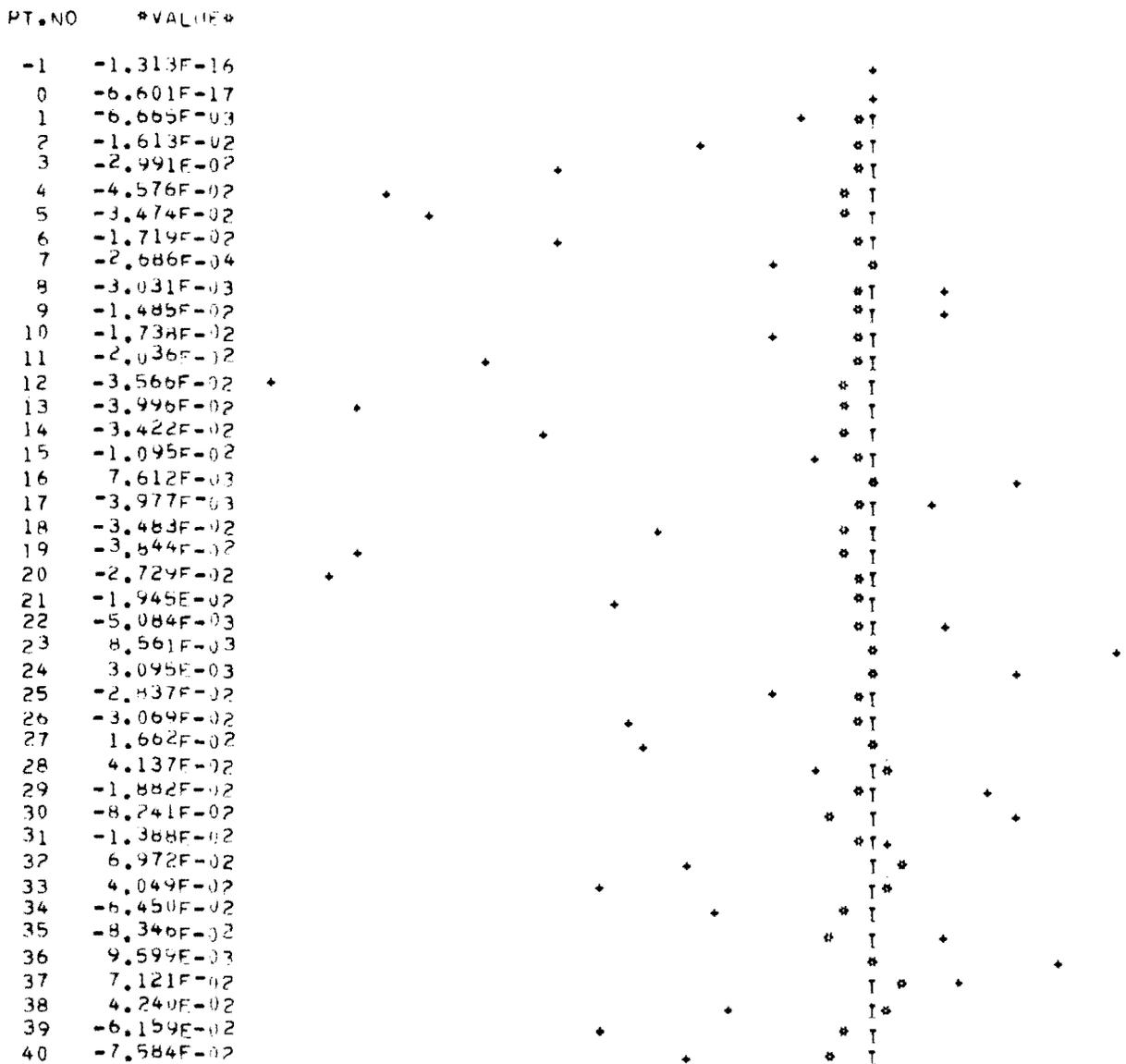


Fig 18. A typical printer plot.

the printer plots for identifying different curves superimposed on this frame. The identifications of the plot variables (such as description of program and run, problem number and problem description, description of the type of the plots, and beam stations or time stations) are printed on the top of each frame of plots. The numerical values printed vertically to the left of the plots are those of the first plot on this frame. The scale of the vertical (deflection or moment) axis is not plotted on the plots, and the horizontal (beam or time station) axis is scaled equally for each point on the plots and plotted by the symbols I. A series of numbers corresponding to the beam or time stations is printed vertically on the left end of the plots for the purpose of indexing. The printer plots fit on 8-1/2-inch by 11-inch standard letter-size paper.

## CHAPTER 5. EXAMPLE PROBLEMS

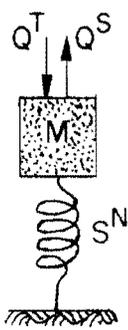
In this chapter seven example problems are solved by Program DBC5. Problems 1 and 2 each have two different cases and Problem 4 has three different cases. Problems 1 and 1A demonstrate the validity of the program in predicting the formation of hysteresis loops on the nonlinear resistance-deflection curve of the spring which supports a freely vibrating mass.

Problems 2-1 and 2-2 present a comparison between the numerical solutions solved by DBC5 and the theoretical solutions of vibrating beams illustrated in Ref 12, pages 375 and 378. Problem 3 demonstrates the capability to use the computed dynamic tire forces of a moving truck from another available computer program as input data in Program DBC5 to predict the dynamic responses of a highway structure. Problem 4 compares the experimental data with the computed results of a simply supported steel rod under axial pulses. Problem 5 demonstrates the capability to predict the response of off-shore piles to wave forces. The solution of a three-span beam (30 beam increments) which is simply supported at the ends and loaded by a transient pulse at station 17 is illustrated in Problem 6. Finally, Problem 7 demonstrates the capability to predict the response of a partially embedded pile to earthquake-induced forces. A listing of input data is included in Appendix G, and the sample computer output listing is included in Appendix H.

### Example Problem 1. Inelastically Supported Mass Under Free Vibration

Figure 19 shows a mass of density  $1.0 \text{ lb-sec}^2/\text{in.}$  supported by a nonlinearly inelastic spring; the mass is loaded by a static load of 100 pounds for the initial deflection and then released by applying a constant dynamic load of 100 pounds. The weight of the mass is neglected. The nonlinear characteristics of the resistance-deflection curve of the spring are also shown in Fig 19. Time increment length is equal to  $6.283 \times 10^{-3}$  second, which is approximately 1/100 of the natural period. The vibration of the mass is solved at 800 time stations. This first problem is intended to provide confidence in the proposed multi-element models to predict the loading paths of the nonlinear-inelastic

Problem 1



Mass density :  $1.0 \text{ lb-sec}^2/\text{in}$

$Q^S$  100 lb

$Q^T$  -100 lb/each time sta

Number of beam increments = 1

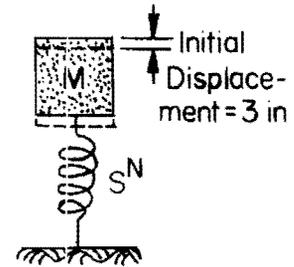
Beam increment length = 1 inch

Number of time increments = 800

Time increment length =

$6.283 \times 10^{-3}$  seconds

Problem 1A



Nonlinearly-Inelastic Resistance-Deflection Curve of  $S^N$

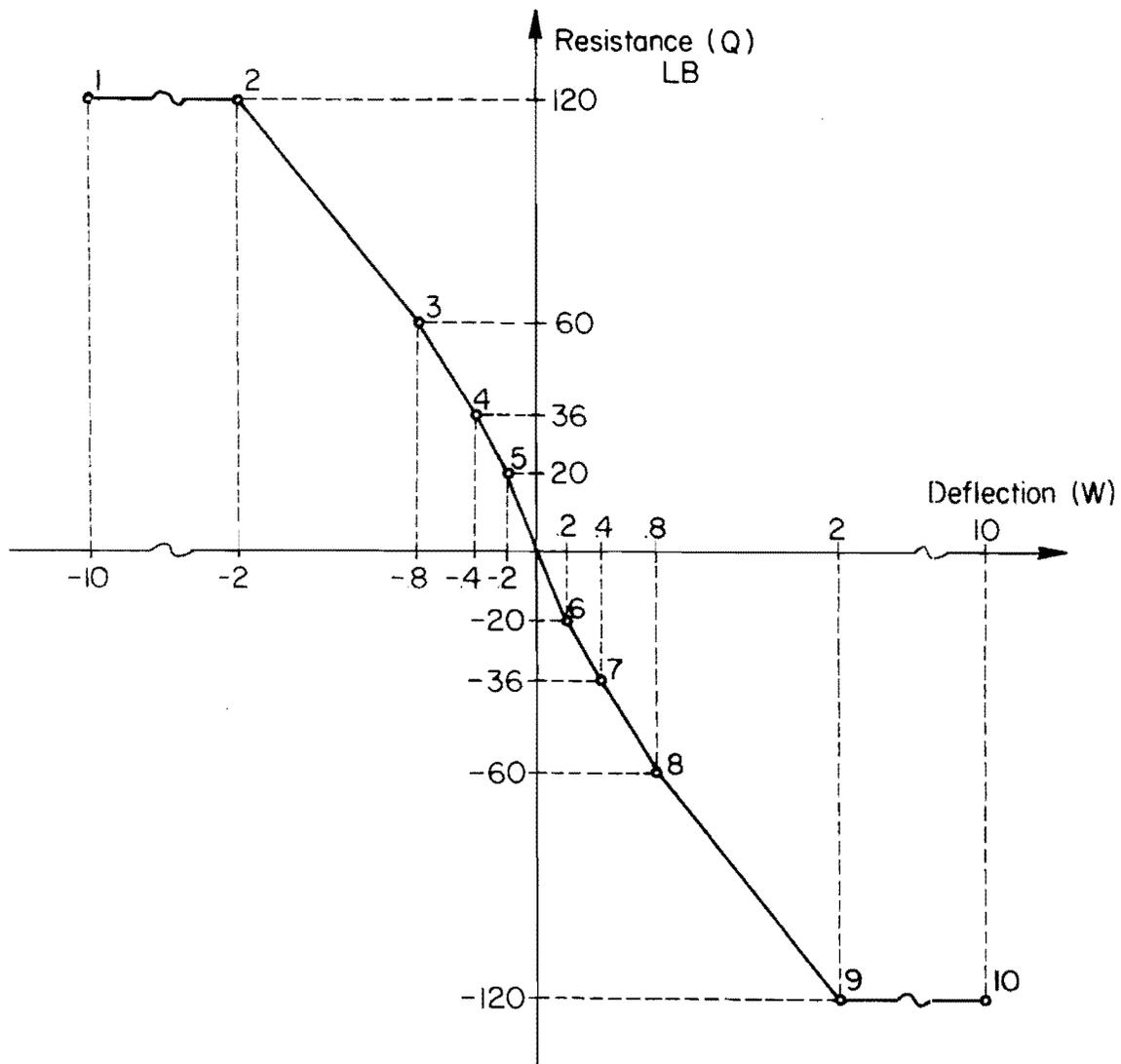


Fig 19. Example Problems 1 and 1A. Inelastically supported mass under free vibration.

support. From the computer output, a plot of deflections versus support reactions can be obtained, as shown in Fig 20.

It is seen that due to partial yielding of the support at the beginning, a hysteresis loop is formed in each cycle of vibration and mechanical energy is lost in each loop. The mechanical energy lost from point A to point J in Fig 20 is equal to the area defined by JKEFGHIJ, which is equivalent to the difference between the summation of the areas defined by ALDCBA and DEFPD and the summation of the areas defined by FPNHGF and JMNIJ. Therefore, the deflections are damping out with time until the response finally becomes a free elastic vibration within the range of greatest stiffness. This can be proved from the tabular output of the program when large time increment lengths ( $6.283 \times 10^{-2}$  second) are used. The computer plot of deflections versus time is shown in Fig 21.

#### Example Problem 1A. Free Vibration of Mass by Specifying Initial Displacements

An additional run on the preceding problem was made by specifying an initial displacement of 3 inches instead of applying a static load for the initial deflection. From the plot of support reactions versus deflections (Fig 22), it is seen that a permanent set is obtained because of initial yielding of the support. The mass is finally freely vibrated at a position about a mean permanent upward deflection of approximately 1 inch. This can be shown in the computer plot of deflections versus time in Fig 23.

#### Example Problem 2-1. Lateral Vibration of a Simply Supported Beam with Axial Compression Force

Figure 24 shows a simply supported beam which is compressed with an axial force of  $3.70 \times 10^5$  pounds and placed under free vibration by specifying an initially deflected sine curve for which the maximum deflection at the center is 3.952 inches. The beam is 120 inches long and has a uniform flexural stiffness of  $1.08 \times 10^9$  lb-in<sup>2</sup>. The beam is divided into 10 elements, each is 12 inches long. The mass density of the beam is  $1.085 \times 10^{-1}$  lb-sec<sup>2</sup>/in./sta. The number of time increments is 200, and the time increment length is  $3.752 \times 10^{-4}$  seconds. The theoretical natural period of the vibration is  $3.752 \times 10^{-2}$  seconds (Ref 9, p 375).

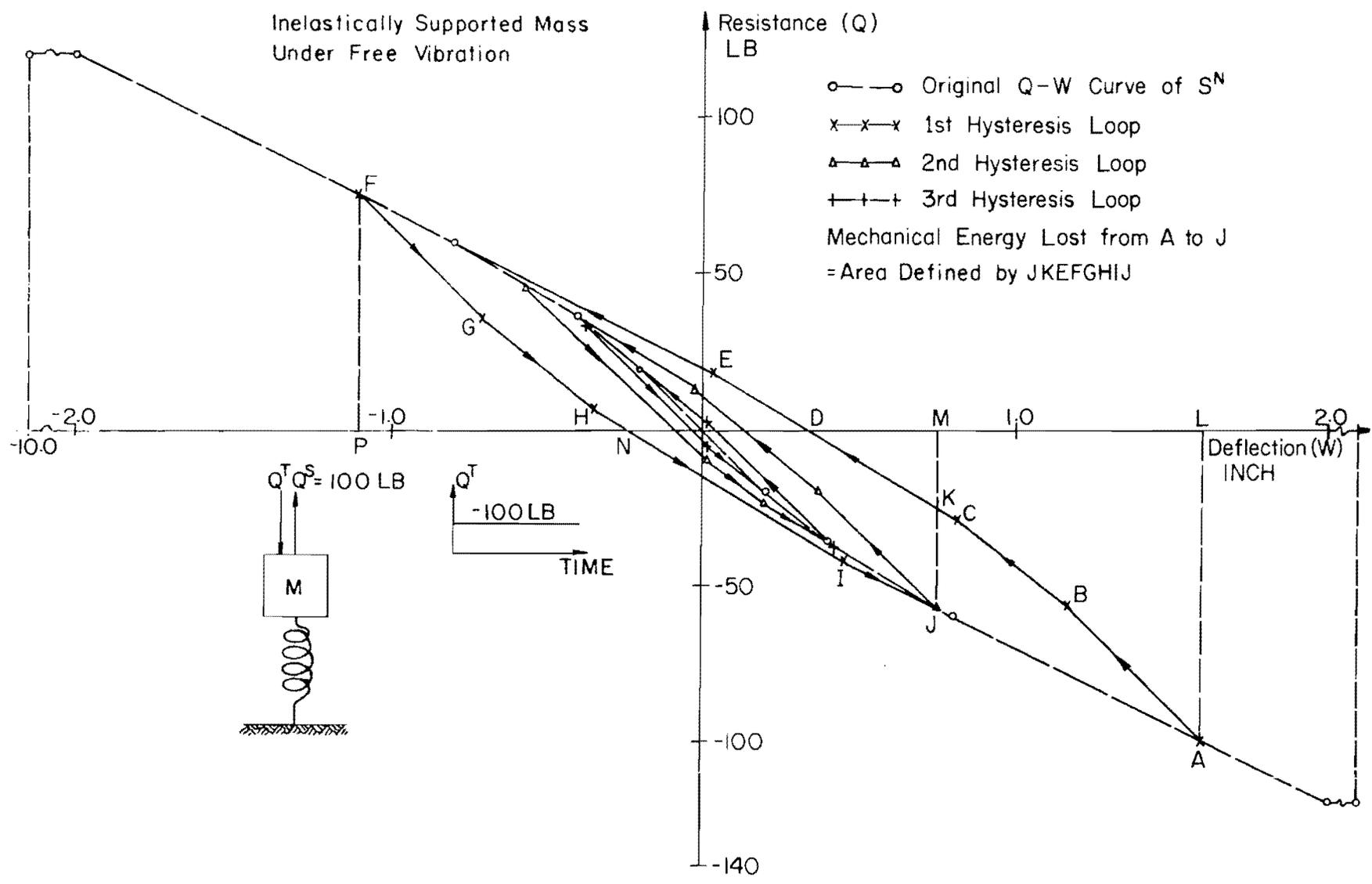
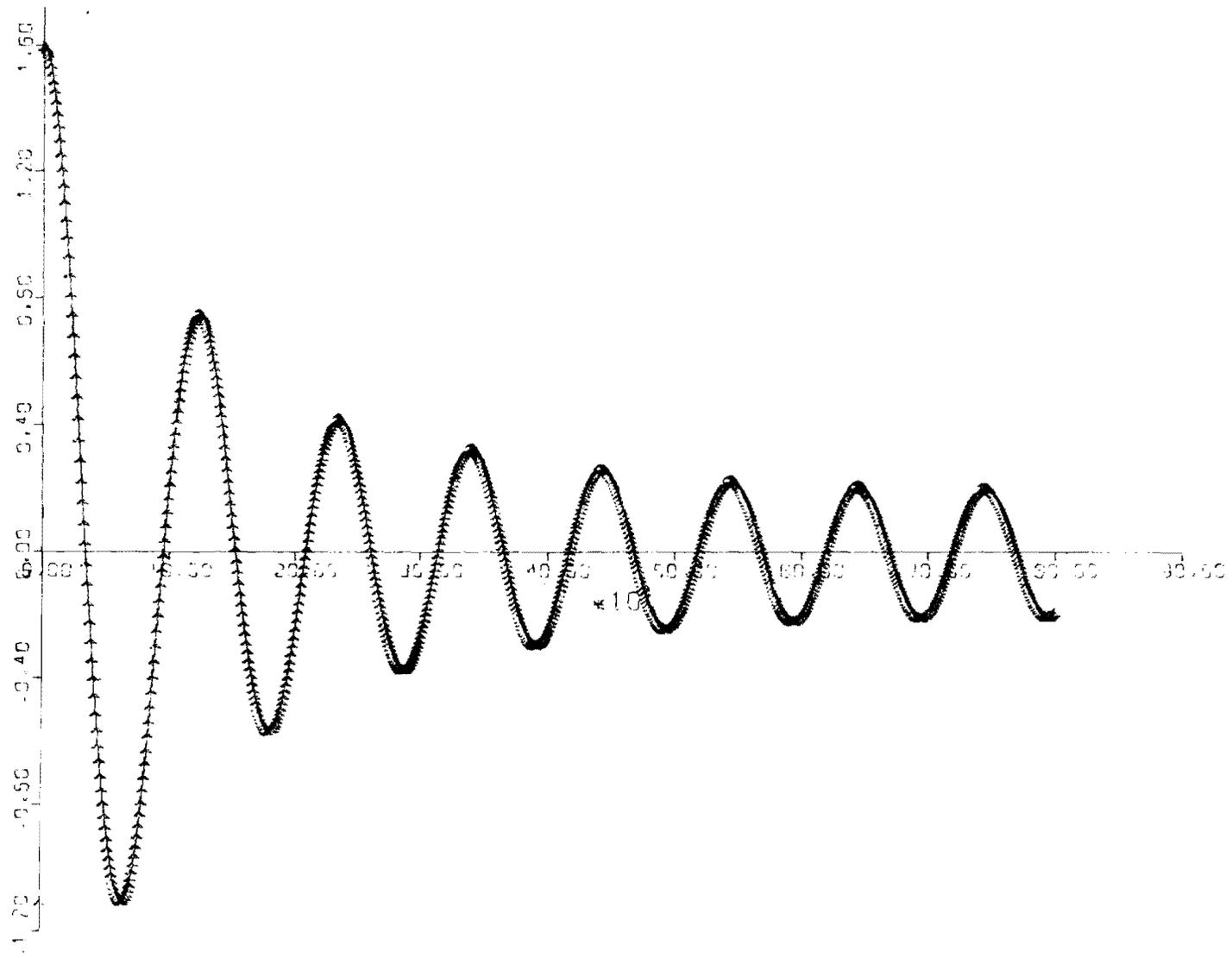


Fig 20. Hysteresis loops of computed resistance-deflection curve of Example Problem 1.



1 DVT BEAM STA = 1

Fig 21. Example Problem 1. Response of the nonlinearly supported mass.

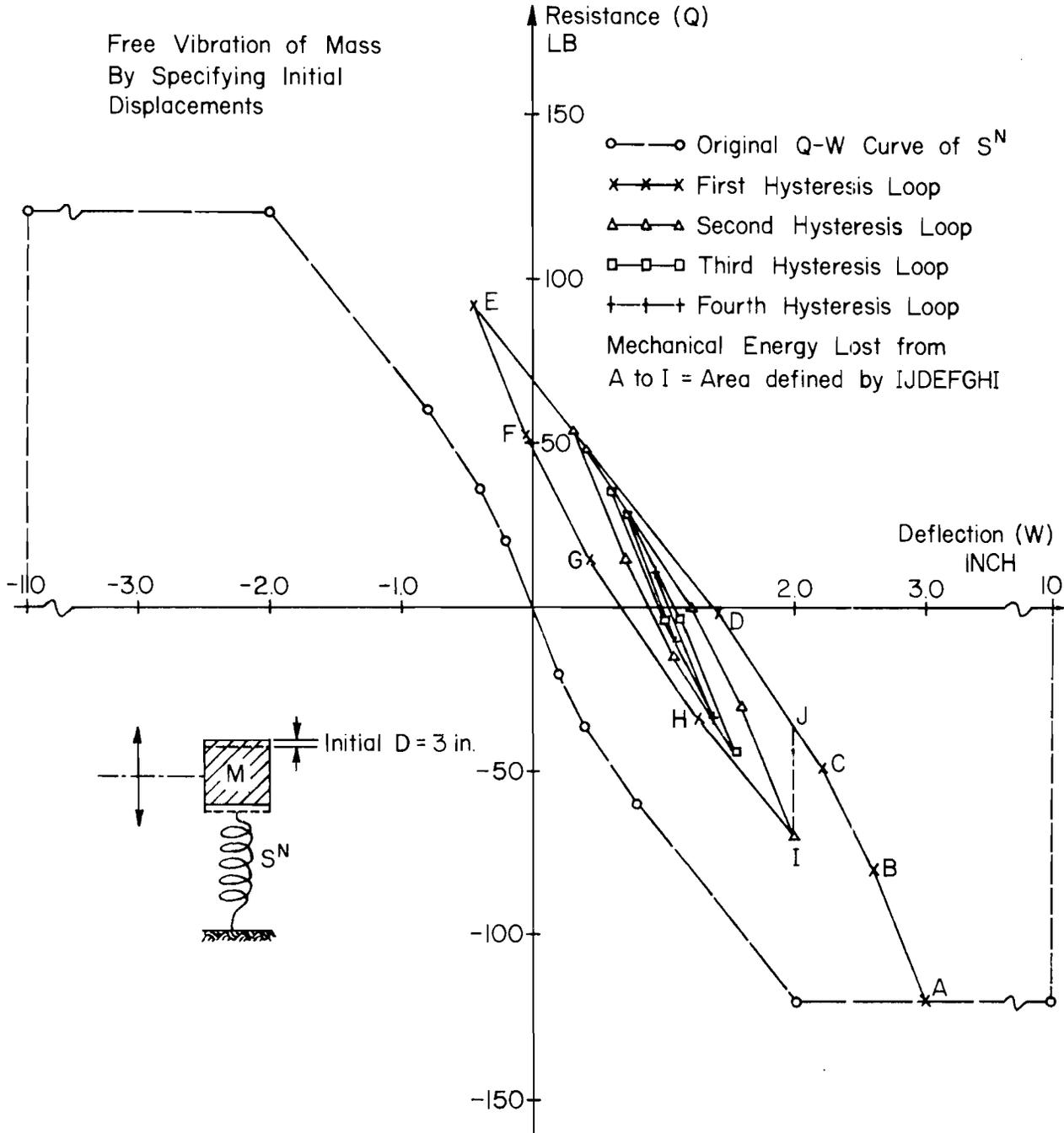
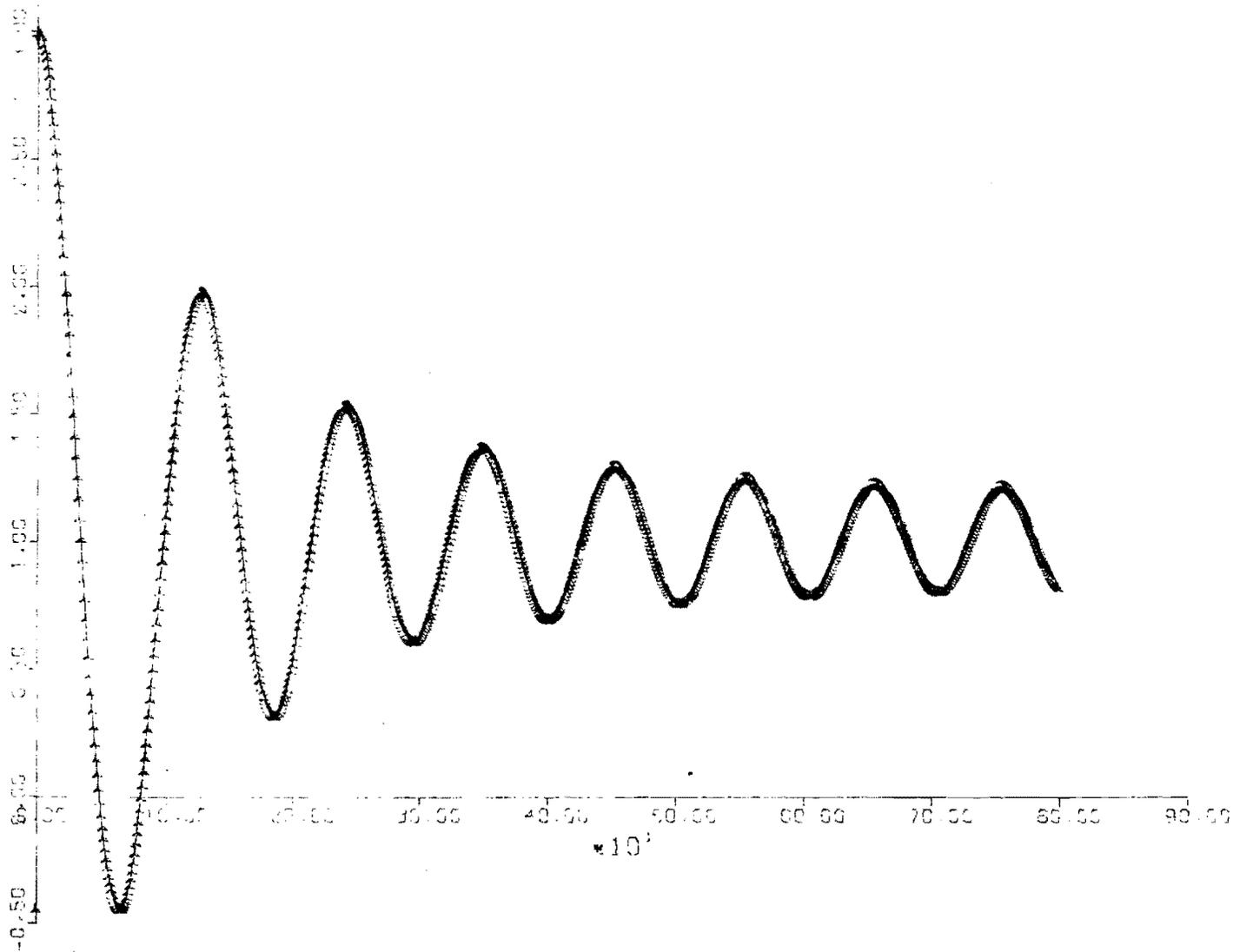
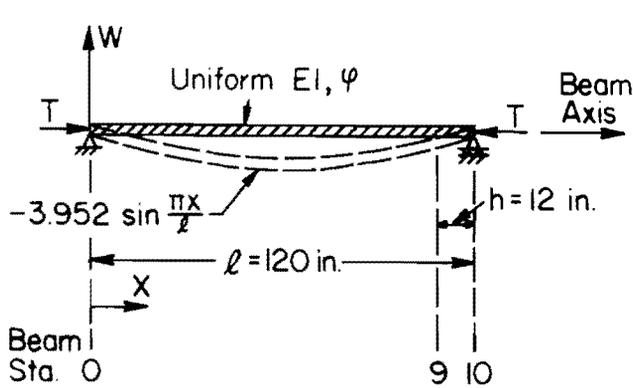


Fig 22. Hysteresis loops of computed resistance-deflection curves of Example Problem 1A.



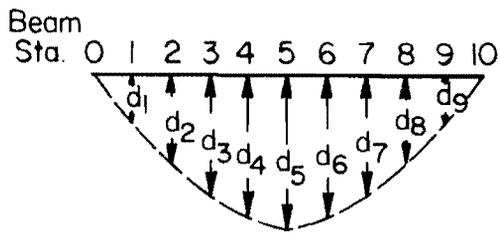
1A DVT BEAM STA = 1

Fig 23. Example Problem 1A. Response of the nonlinearly supported mass.



$EI = 1.08 \times 10^9 \text{ lb-in}^2$   
 Mass density  $1.085 \times 10^{-1} \text{ lb-sec}^2/\text{in/sta}$   
 $T = -3.70 \times 10^5 \text{ lb}$   
 Number of beam increments = 10  
 Beam increment length = 12 inches  
 Number of time increments = 200  
 Time increment length =  $3.752 \times 10^{-4} \text{ seconds}$

Initially Deflection Sine Curve



$$\begin{aligned}
 d_1 = d_9 &= -3.952 \times \sin 18^\circ = -1.221 \text{ inches} \\
 d_2 = d_8 &= -3.952 \times \sin 36^\circ = -2.323 \text{ inches} \\
 d_3 = d_7 &= -3.952 \times \sin 54^\circ = -3.197 \text{ inches} \\
 d_4 = d_6 &= -3.952 \times \sin 72^\circ = -3.759 \text{ inches} \\
 d_5 &= -3.952 \times \sin 90^\circ = -3.952 \text{ inches}
 \end{aligned}$$

Fig 24. Example Problem 2-1. Lateral vibration of a simple supported beam with axial compression force.

The theoretical responses (Eq d, p 375, Ref 12) and the computed responses based on a time increment length of  $3.752 \times 10^{-3}$  seconds (1/10 of the natural period) are plotted for the center of the beam on the computer plot of deflections versus time for beam station 5, as shown in Fig 25. The responses of the small time increment length (1/100 of the natural period) are almost perfectly matched with the theoretical responses. The responses of the large time increment length (1/10 of the natural period) show more variation when compared to the theoretical responses in the first period, and after the first period are shifted with a phase angle. The more increments, however, the beam is divided into, the better the results. The computed moments, based on both small and large time increment lengths along the time axis for the center of the beam, are shown in Fig 26.

#### Example Problem 2-2. Lateral Vibration of a Beam on Elastic Foundation

Figure 27 shows the beam in Problem 2-1 resting on an elastic foundation which has a spring constant of  $1.2 \times 10^4$  lb/in./sta. The beam is under free vibration with an initially deflected shape of sine curve, in which the maximum deflection at the center is 6.672 inches. The number of time increments is 200, and the time increment length is  $1.540 \times 10^{-4}$  seconds. The theoretical natural period is  $1.540 \times 10^{-2}$  seconds (Ref 12, p 378).

The comparisons between the computed responses, based on both small and large time increment lengths, and the theoretical responses (Eq c, p 378, Ref 12) for the center of the beam are shown in Fig 28. The difference between the computed responses of the small time increment length (1/100 of the natural period) and the theoretical responses is slight. The computed moments, based on both small and large time increment lengths, along the time axis are shown in Fig 29.

#### Example Problem 3. Three-Span Beam Loaded with an AASHO Standard 2-D Truck Moving at a Speed of 60 MPH

This problem demonstrates the capability to input the dynamic tire forces interpreted from the computer output of the program described in Ref 1 into Table 7 of Program DBC5 to predict the responses of a three-span beam; the beam is loaded with an AASHO standard 2-D truck which is moving at a uniform speed of 60 mph, as shown in Fig 30.

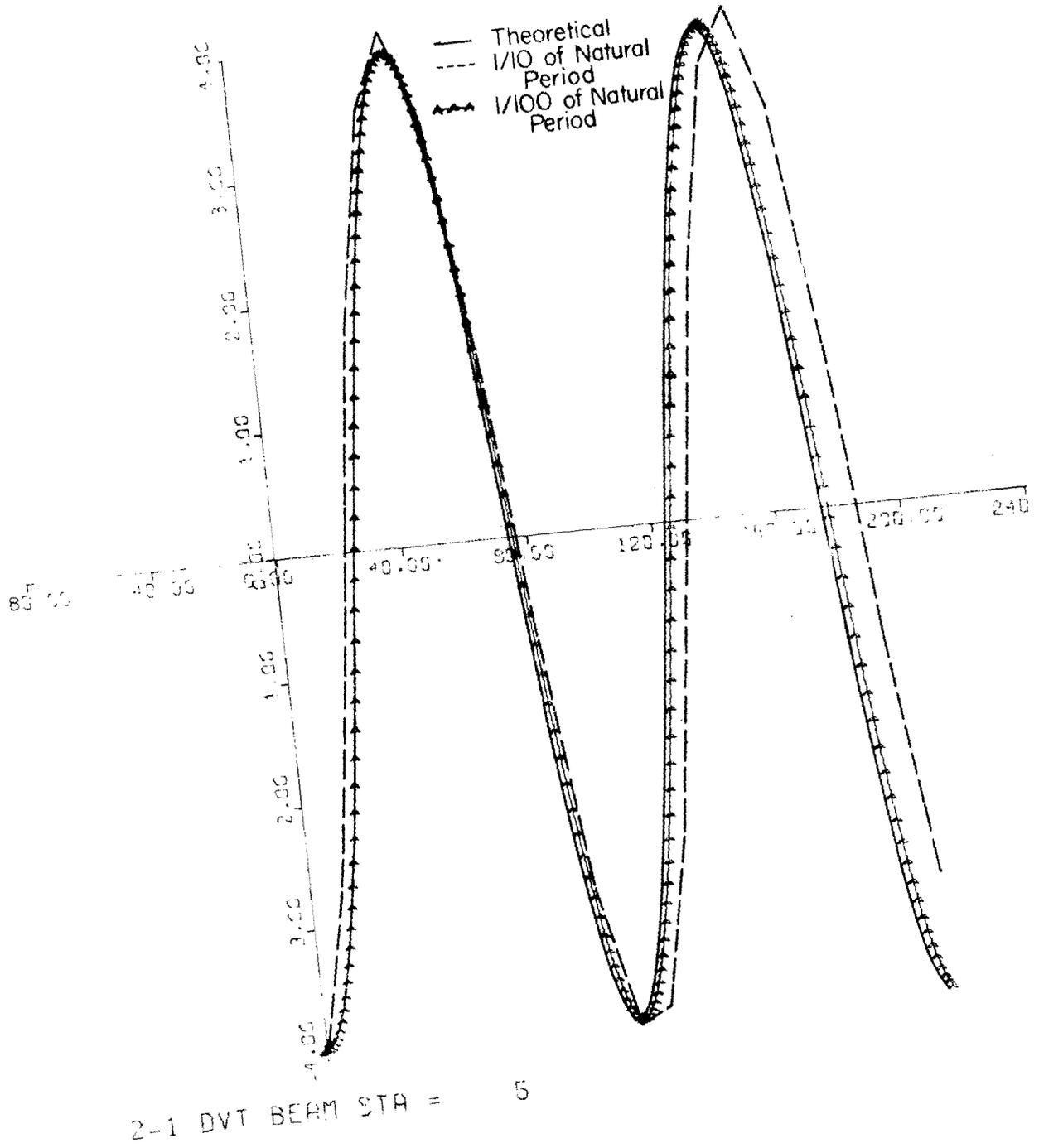
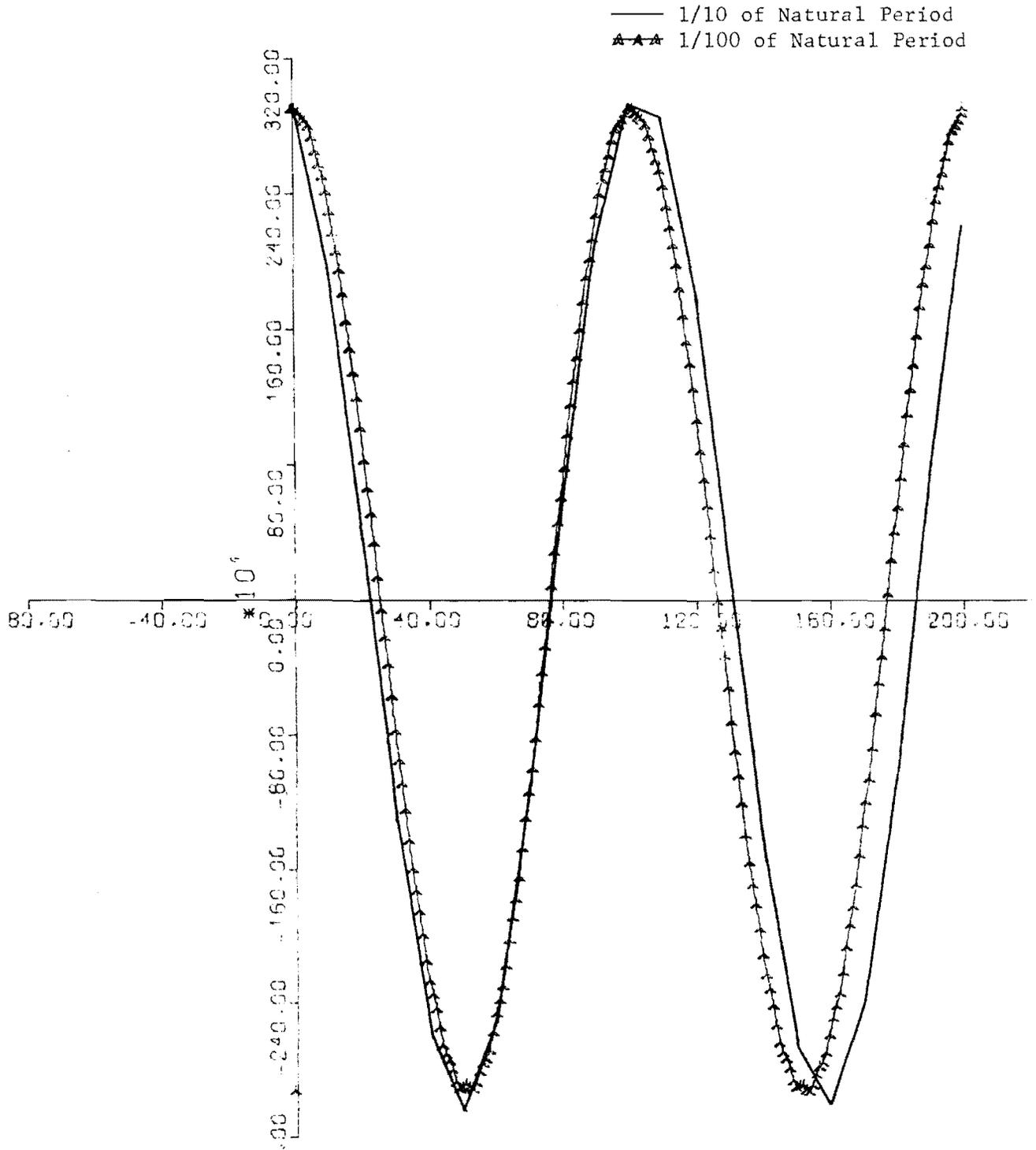
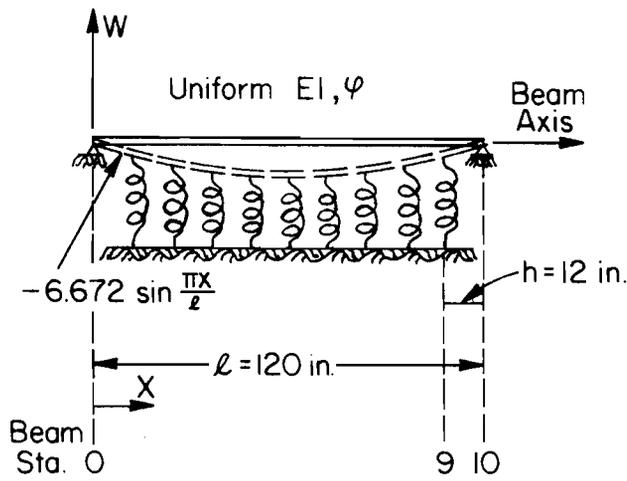


Fig 25. Example Problem 2-1. Deflection vs time for the center of beam.



2-1 MVT BEAM STA = 5

Fig 26. Example Problem 2-1. Moment vs time for the center of beam.



$$EI = 1.08 \times 10^9 \text{ lb-in}^2$$

$$\text{Mass density} = 1.085 \times 10^{-1} \text{ lb-sec}^2/\text{in/sta}$$

$$\text{Elastic spring constant} = 1.2 \times 10^4 \text{ lb/in/sta}$$

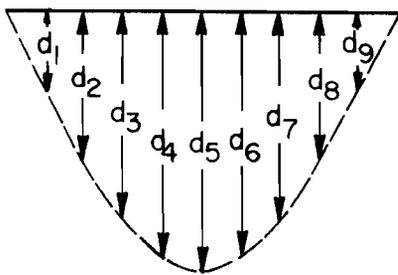
$$\text{Number of beam increments} = 10$$

$$\text{Beam increment length} = 12 \text{ inches}$$

$$\text{Number of time increments} = 200$$

$$\text{Time increment length} = 1.540 \times 10^{-4} \text{ seconds}$$

### Initially Deflected Sine Curve



$$d_1 = d_9 = -6.672 \times \sin 18^\circ = -2.061 \text{ inches}$$

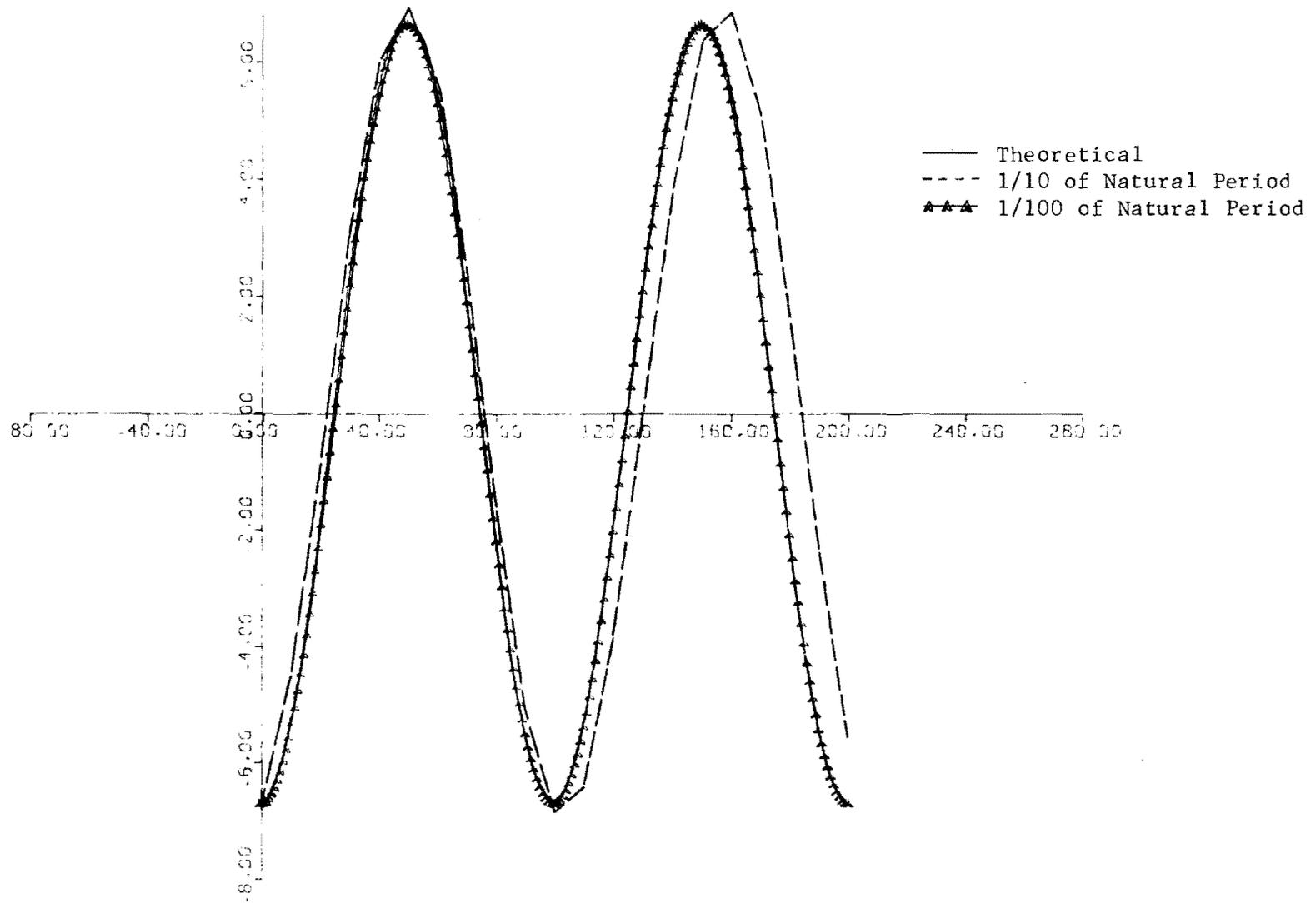
$$d_2 = d_8 = -6.672 \times \sin 36^\circ = -3.922 \text{ inches}$$

$$d_3 = d_7 = -6.672 \times \sin 54^\circ = -5.398 \text{ inches}$$

$$d_4 = d_6 = -6.672 \times \sin 72^\circ = -6.345 \text{ inches}$$

$$d_5 = -6.672 \times \sin 90^\circ = -6.672 \text{ inches}$$

Fig 27. Example Problem 2-2. Lateral vibration of a beam on elastic foundation.



2-2 DVT BEAM STA = 5

Fig 28. Example Problem 2-2. Deflection vs time for the center of beam.

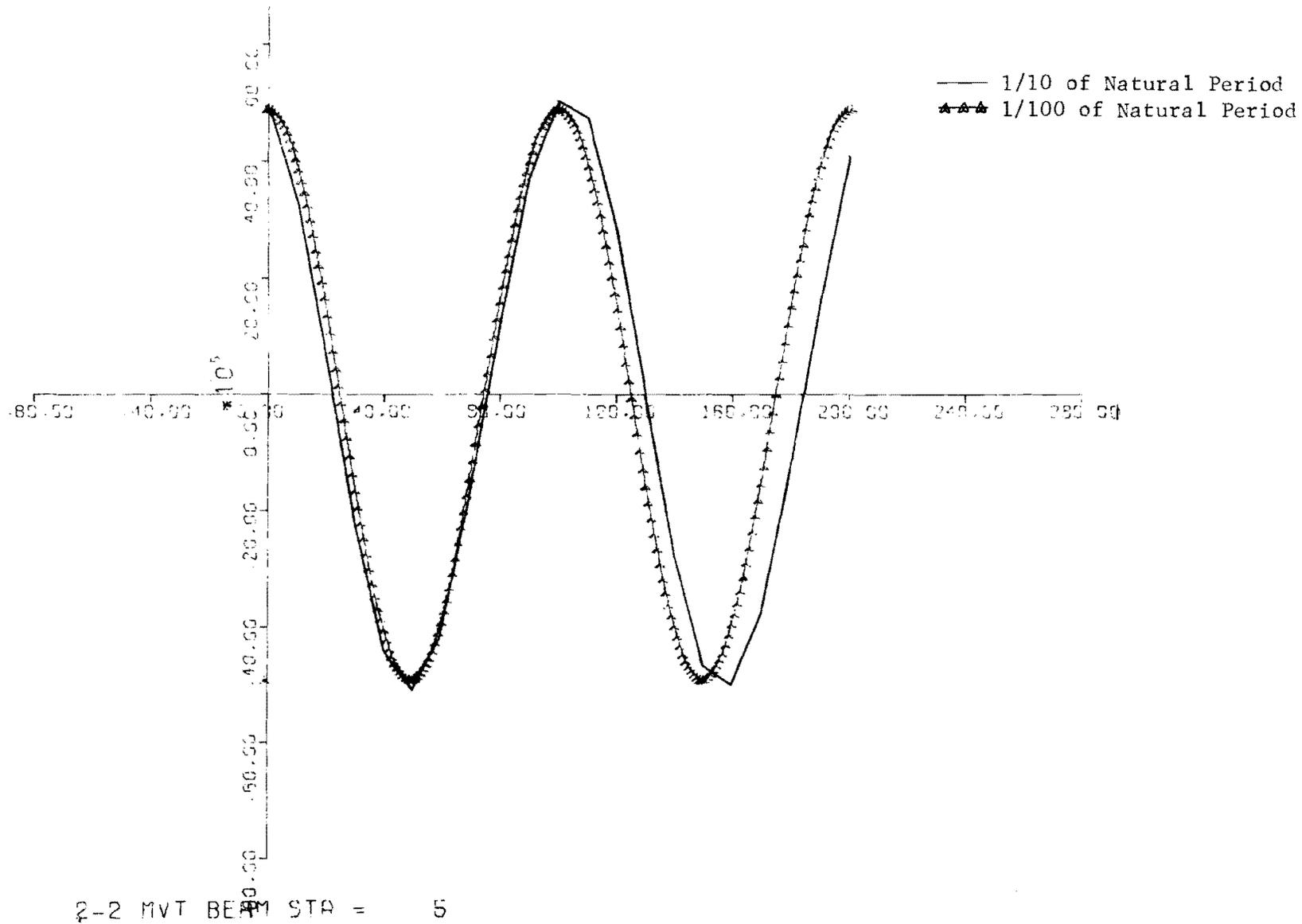
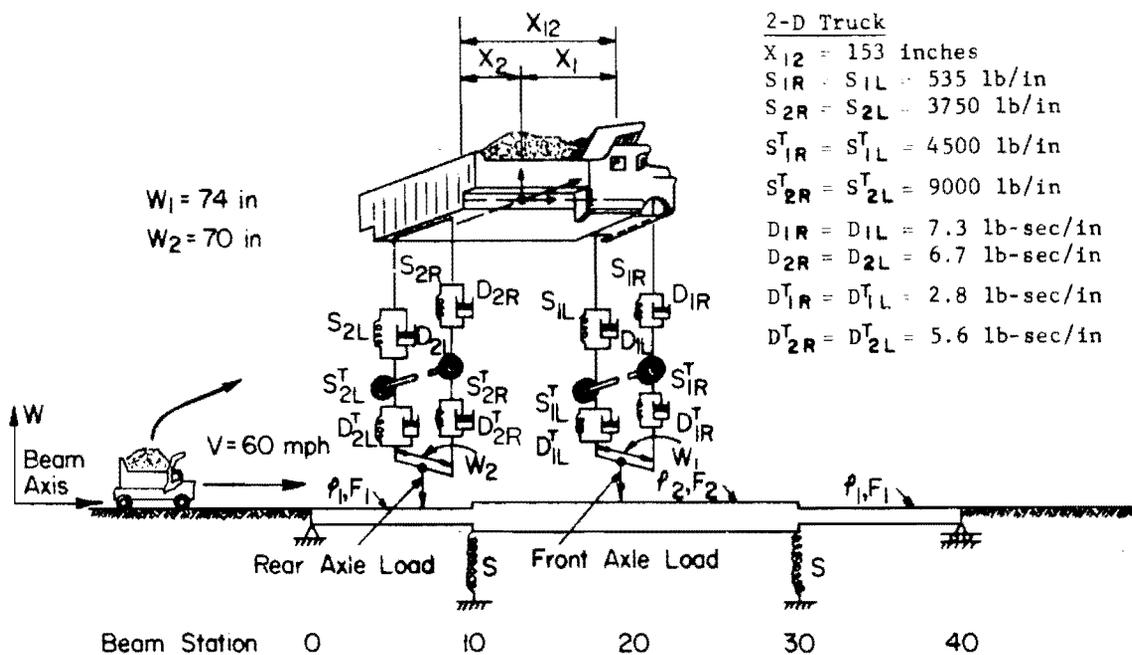


Fig 29. Example Problem 2-2. Moment vs time for the center of beam.



2-D Truck  
 $X_{12} = 153$  inches  
 $S_{1R} = S_{1L} = 535$  lb/in  
 $S_{2R} = S_{2L} = 3750$  lb/in  
 $S_{1R}^T = S_{1L}^T = 4500$  lb/in  
 $S_{2R}^T = S_{2L}^T = 9000$  lb/in  
 $D_{1R} = D_{1L} = 7.3$  lb-sec/in  
 $D_{2R} = D_{2L} = 6.7$  lb-sec/in  
 $D_{1R}^T = D_{1L}^T = 2.8$  lb-sec/in  
 $D_{2R}^T = D_{2L}^T = 5.6$  lb-sec/in

Static Weights of Wheels

Axle 1 right	3139 lb
Axle 1 left	3012 lb
Axle 2 right	7780 lb
Axle 2 left	7103 lb

Static Loads of Axles

Front	3075 lb
Rear	7441 lb

Dynamic Forces of the Front and Rear Axles

Beam Properties

$F_1 = 4.5 \times 10$  in. lb-in  
 $F_2 = 9.0 \times 10$  in lb-in  
 $S = 2.0 \times 10$  lb/in  
 $\rho_1 = 0.5964$  lb-sec /in/sta  
 $\rho_2 = 0.2982$  lb-sec /in/sta  
 Beam increment length = 60 inches  
 Time increment length =  $5.666 \times 10^{-2}$  sec

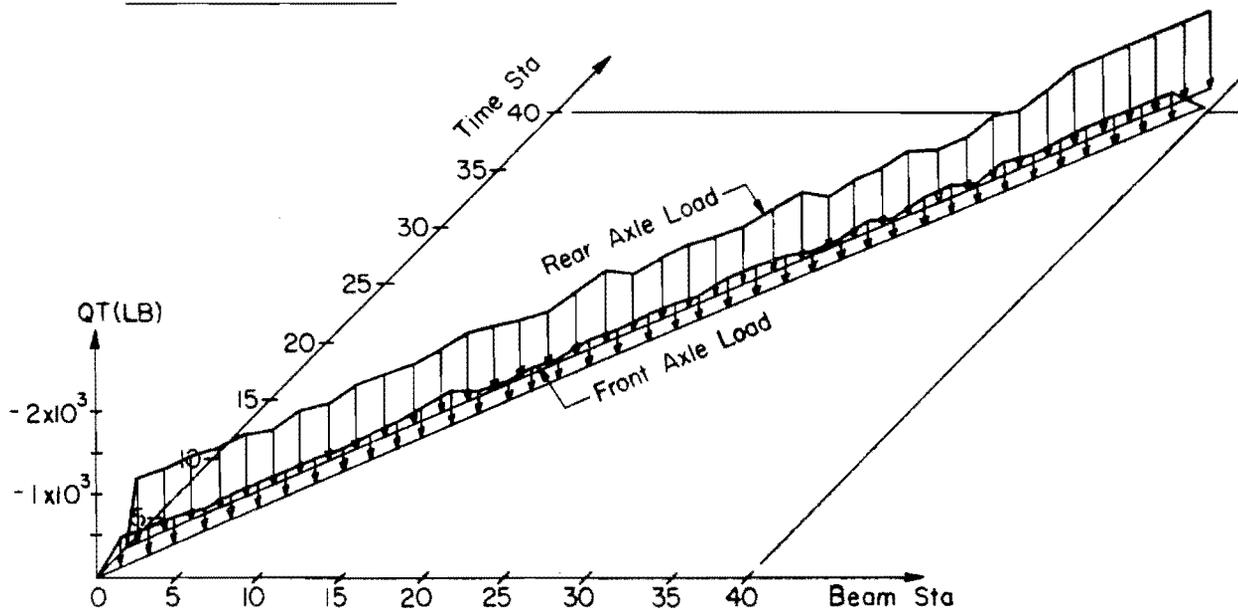
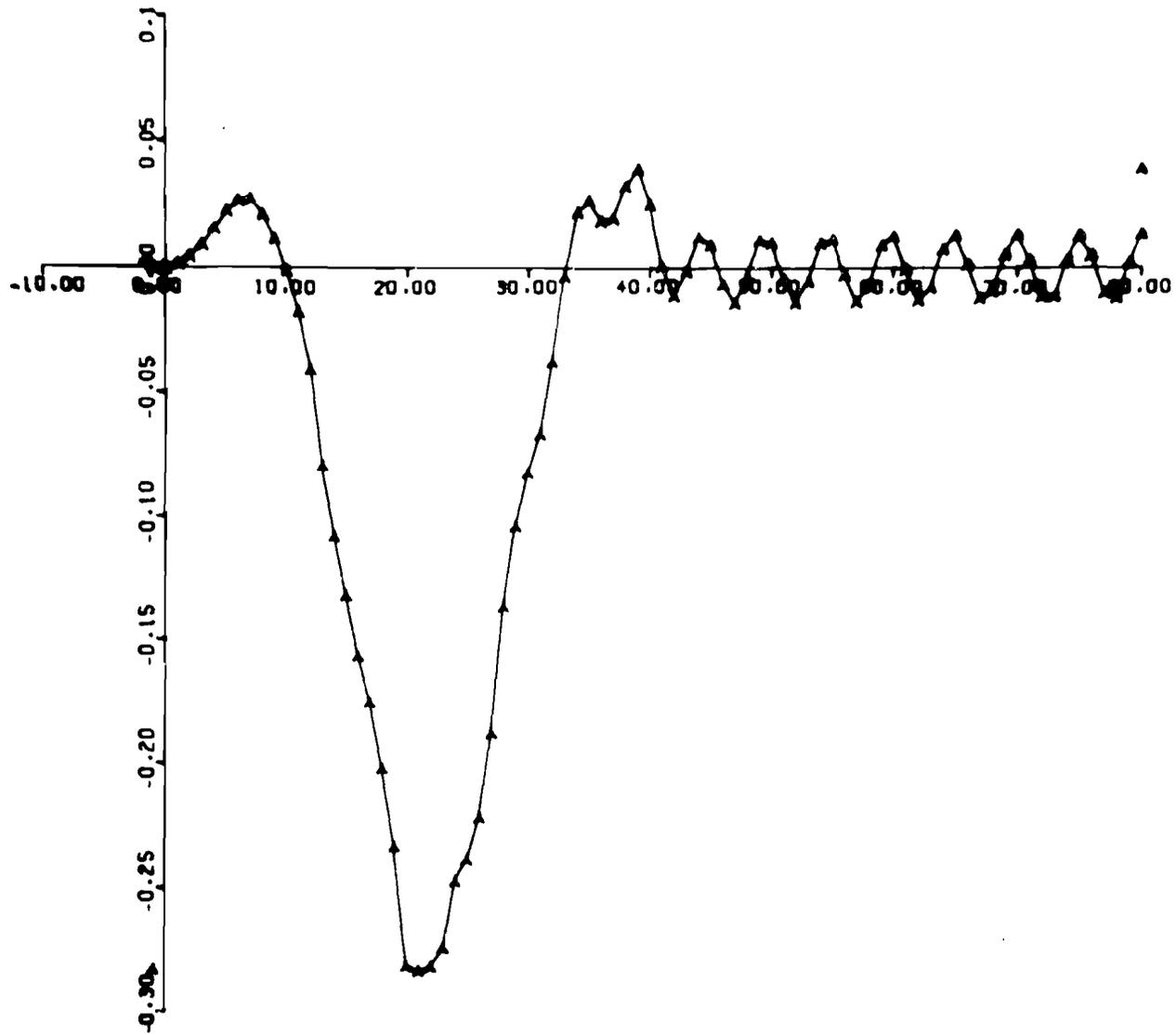


Fig 30. Example Problem 3. Three-span beam loaded by an AASHTO standard 2-D truck moving at a speed of 60 mph.

The beam is simply supported at the ends and has a length of 200 feet. The beam is divided into 40 increments, each of which is 60 inches long. The two linear springs are located at beam stations 10 and 30 and have a value of  $2 \times 10^5$  lb/in. The first and third spans of the beam have a uniform flexural stiffness of  $4.5 \times 10^{11}$  lb-in.<sup>2</sup> and a uniform mass density of 0.5964 lb-sec<sup>2</sup>/in./sta. The second span of the beam has a uniform flexural stiffness of  $9.0 \times 10^{11}$  lb-in.<sup>2</sup> and a uniform mass density of 0.2982 lb-sec<sup>2</sup>/in./sta. The computer model used in Ref 1 for simulating the AASHO standard 2-D truck is shown in Fig 30. The input data for the springs and dashpots which simulate the tires of the truck and the static weights of the wheels are also shown in Fig 30. The computed dynamic forces of the four tires are averaged to give the two axle forces which then are input into Table 7 of Program DBC5. The distribution of these two axle forces on both the beam and time axes is shown in Fig 30. The time axis is divided into 80 increments, each of which has a length of  $5.666 \times 10^{-2}$  seconds so that the front axle will move one beam increment length per time station.

The computed deflections and moments along the time axis for the center of the beam are plotted by the program as shown in Figs 31 and 32, respectively. The plots are produced on microfilm which then can be enlarged.

A comparison of the dynamic response to the static response of the deflections along the beam axis is shown in Fig 33. This static response of the plot in Fig 33 is computed by specifying the static load of the front axle at beam station 21 and the static load of the rear axle at the center of the bar, between beam stations 18 and 10; the dynamic response of the plot, therefore, is the computed deflection along the beam axis for time station 21. The dynamic effect on the responses of the beam produced by a moving 2-D truck is fairly small compared to the static responses produced by the same truck. This is because the dynamic forces generated by the 2-D truck, which is moving on a smooth pavement, are smoothly varied with time, and therefore there are small dynamic effects. Any other AASHO standard truck or semi-trailer can be used in the program described in Ref 1 to generate the corresponding dynamic tire forces, which then can be input with a limited amount of interpretation into Table 7 of Program DBC5 to predict the dynamic responses of highway structures that can be simulated by beam-columns.



3 DVT BEAM STA = 20

Fig 31. Example Problem 3. Deflection vs time for the center of beam.

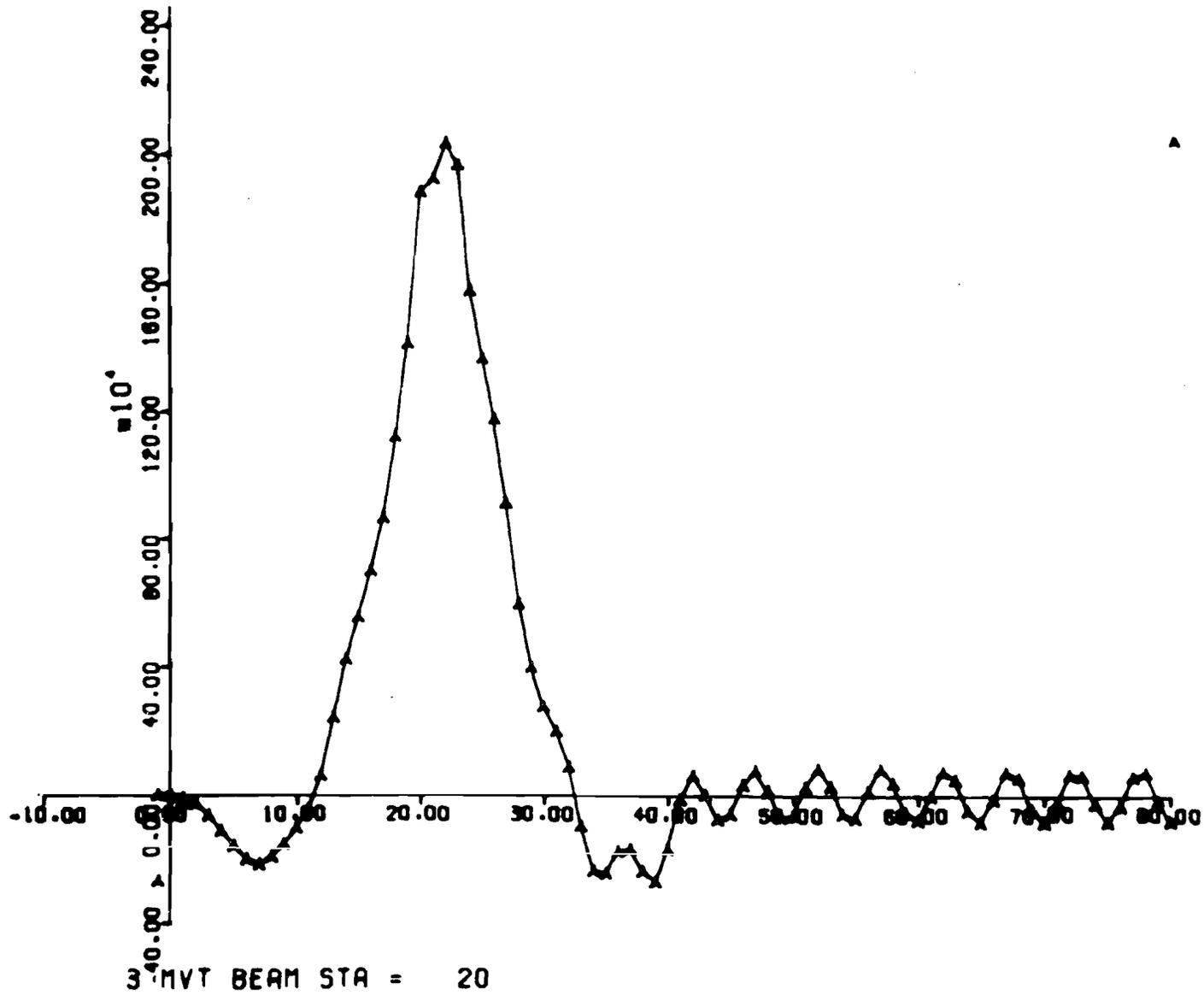
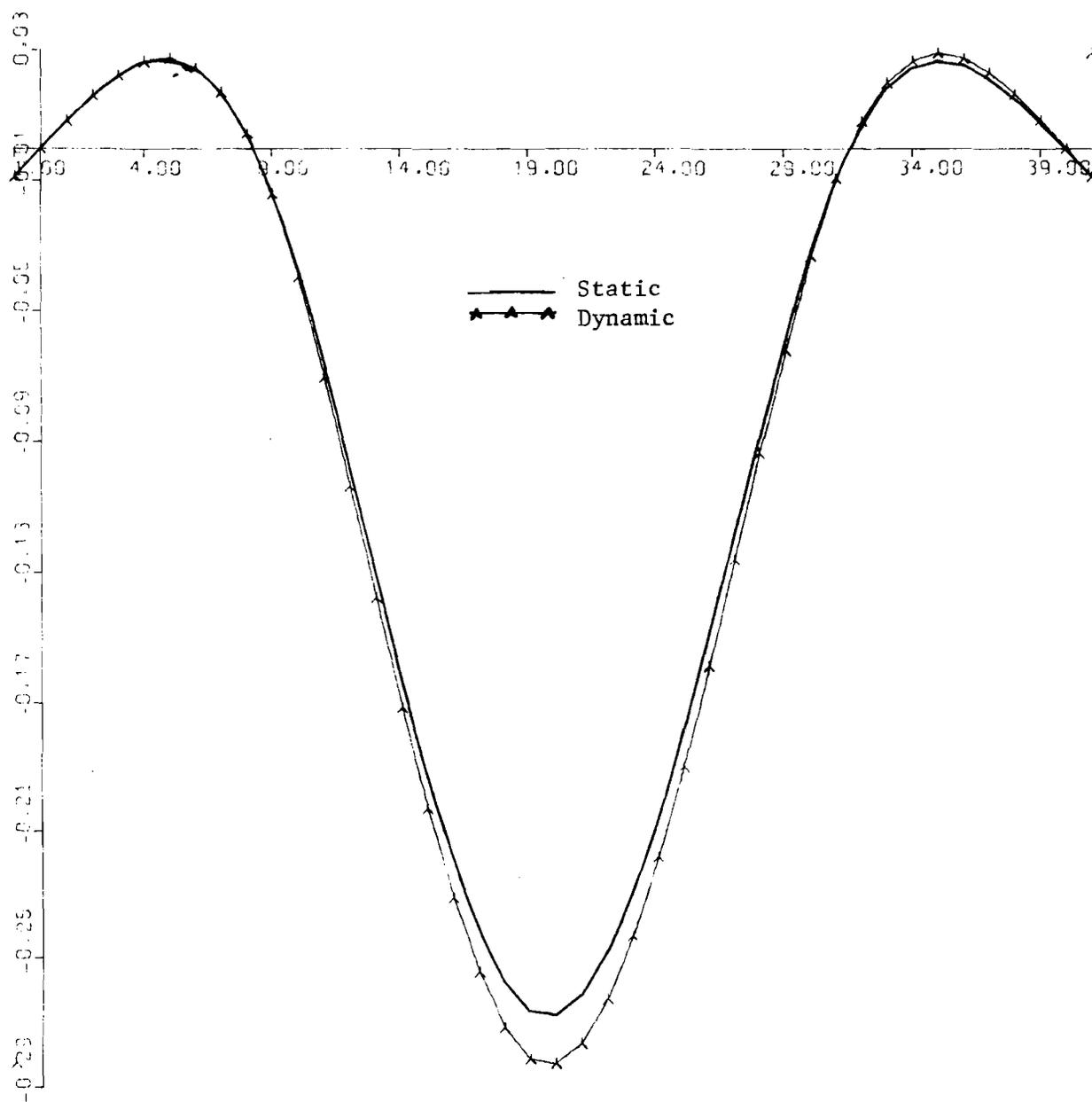


Fig 32. Example Problem 3. Moment vs time for the center of beam.



3 DVB TIME STA = 21

Fig 33. Example Problem 3. Comparison of dynamic to static response (deflection along the beam axis).

Example Problem 4-1. Simply Supported Steel Rod - Axial Pulse Base Time = 0.10 Second

In the following three problems, a simply supported steel rod is loaded by three axial pulses with different peak loads and base time, as shown in Fig 34. The problem is a real experiment which was done by Matlock and Vora (Ref 7). The steel rod is 153.75 inches long and 11/16 of an inch in diameter. The steel rod is divided into 41 increments, each of which has a length of 3.75 inches. The mass density of the rod is  $1.023 \times 10^{-3}$  lb-sec<sup>2</sup>/in./sta. The static weight of the rod is  $3.95 \times 10^{-1}$  lb/sta. An internal damping factor of 20 lb-in.<sup>2</sup>-sec is input at each station.

The first axial pulse input for this problem has a peak value of 175 pounds and a base time of 0.10 seconds. The axial pulse was recorded on the oscilloscope in the experiment by Matlock and Vora described in Ref 7. The true axial pulse is described at 20 points by scaling from the recorded output of strain gages, which then can be input in Table 6 of Program DBC5. The time increment length is set equal to  $5 \times 10^{-3}$  seconds and 40 time stations are solved for this problem.

Figure 35 shows the comparison of the computed bending moments along the time axis to the measured bending moments in the experiment for the center station of the steel rod. It is seen that good agreement exists between the predicted results and the measured results before 0.9 second, while the measured bending moments are smaller than the predicted bending moments after 0.9 second. This deviation is mainly due to the assumption that the beam properties are fully elastic in the dynamic model of Program DBC5. Other factors, such as the errors obtained by scaling from the recorded output, the difficulty in establishing the ideal hinged supports, and failure to consider the axial deformation of the rod, also affect the accuracy of the predicted results.

An additional curve of the bending moment along the time axis for the center of the rod, which is computed without inputting the internal damping factor, is also plotted on Fig 35. Little is changed by ignoring the effect of the internal damping factor, since the change of the curvature at this point is very smooth along the time axis. The computed deflections along the time axis for the center station of the steel rod are plotted by the program, as shown in Fig 36.

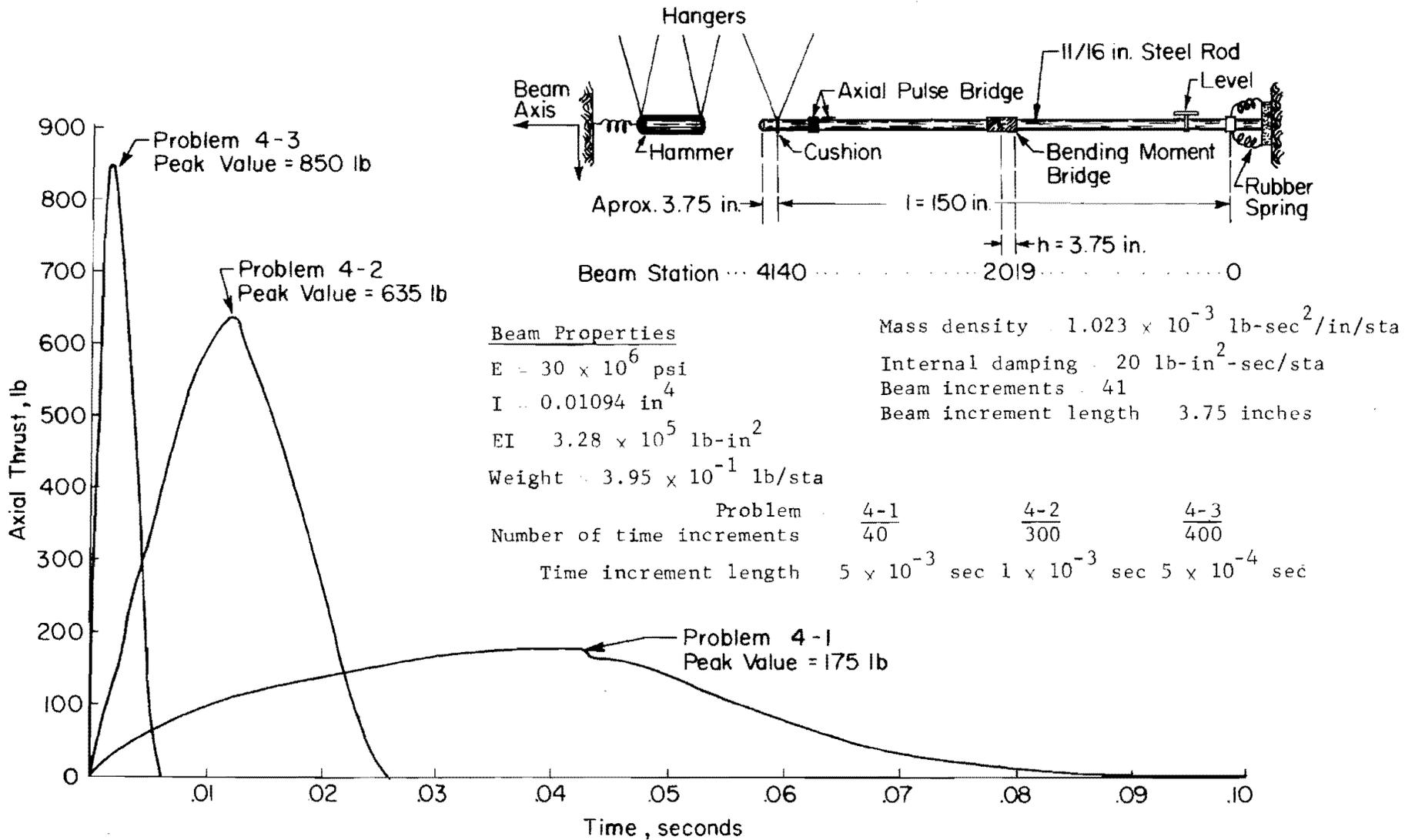


Fig 34. Example Problems 4-1, 4-2, and 4-3. Simply supported steel rod loaded by axial pulses.

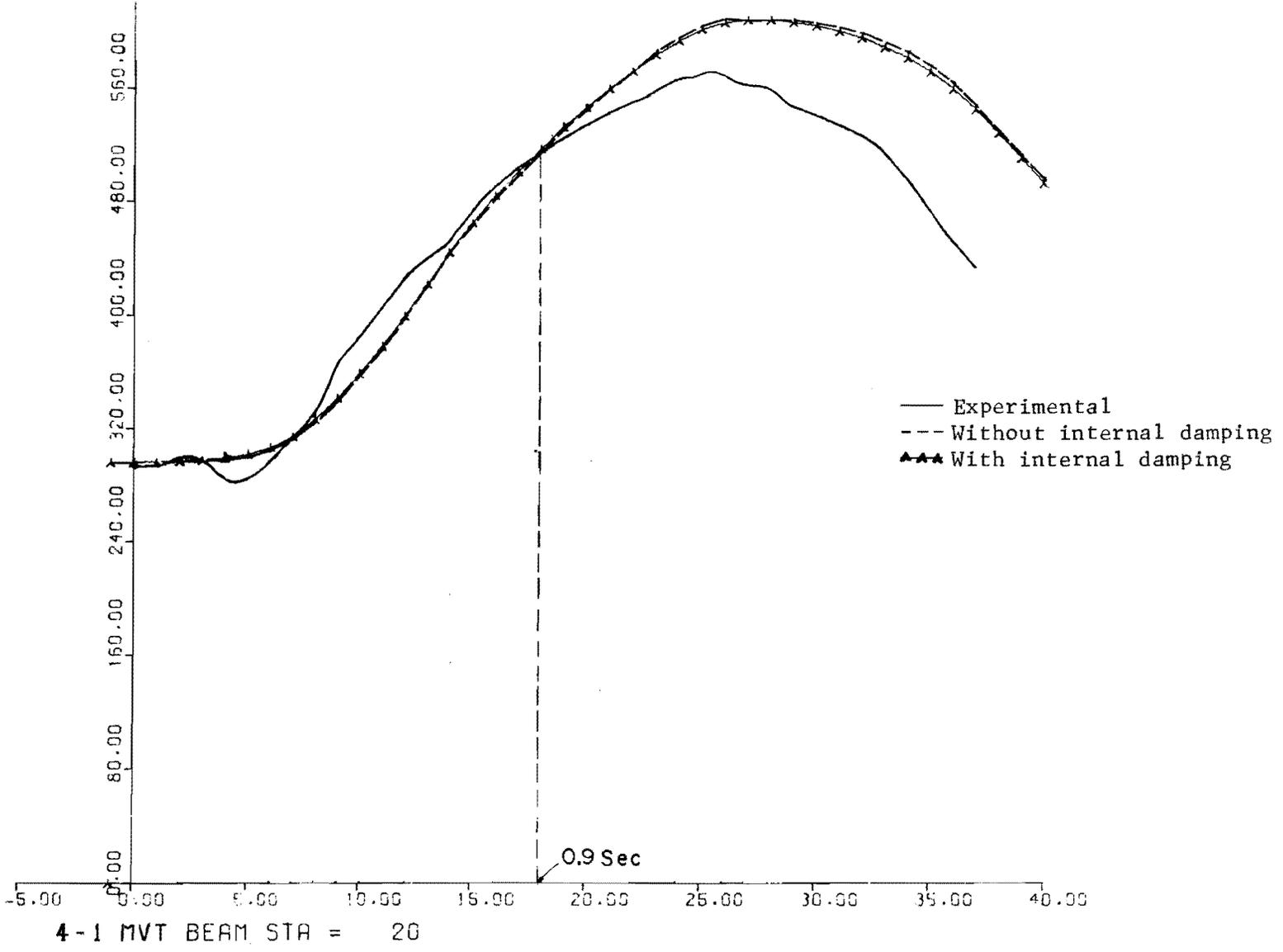
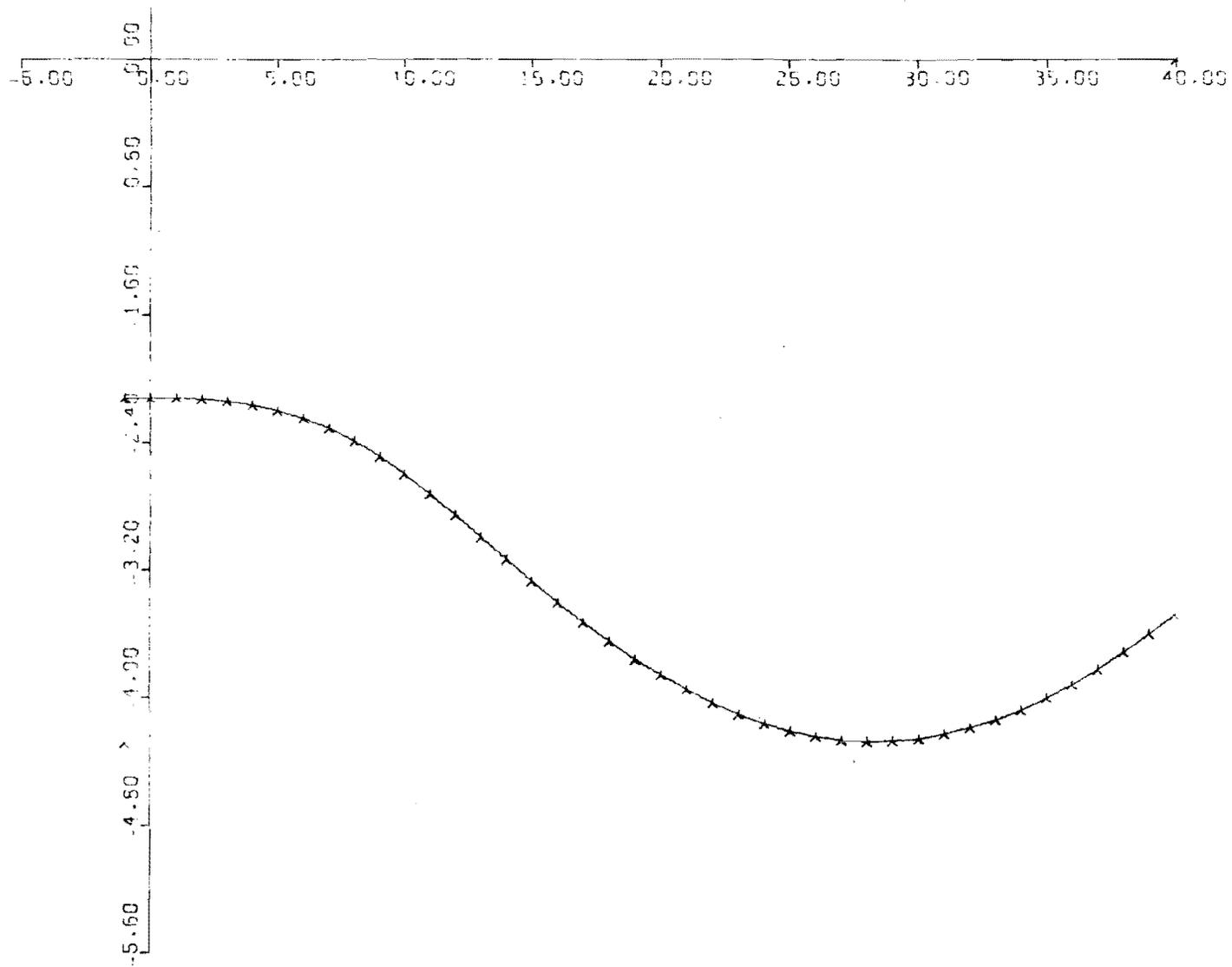


Fig 35. Example Problem 4-1. Comparison of moments vs time for the center of beam.



4-1 DVT BEAM STA = 20

Fig 36. Example Problem 4-1. Deflection vs time for the center of beam.

Example Problem 4-2. Simply Supported Steel Rod - Axial Pulse Base Time = 0.026 Second

In this problem, the steel rod in the preceding problem is loaded by another pulse with a shorter base time but larger peak value. The base time of the pulse is 0.026 second and the peak value of the pulse is increased to approximately 635 pounds. The axial pulse is described at 26 points by scaling from the recorded output of the experiment.

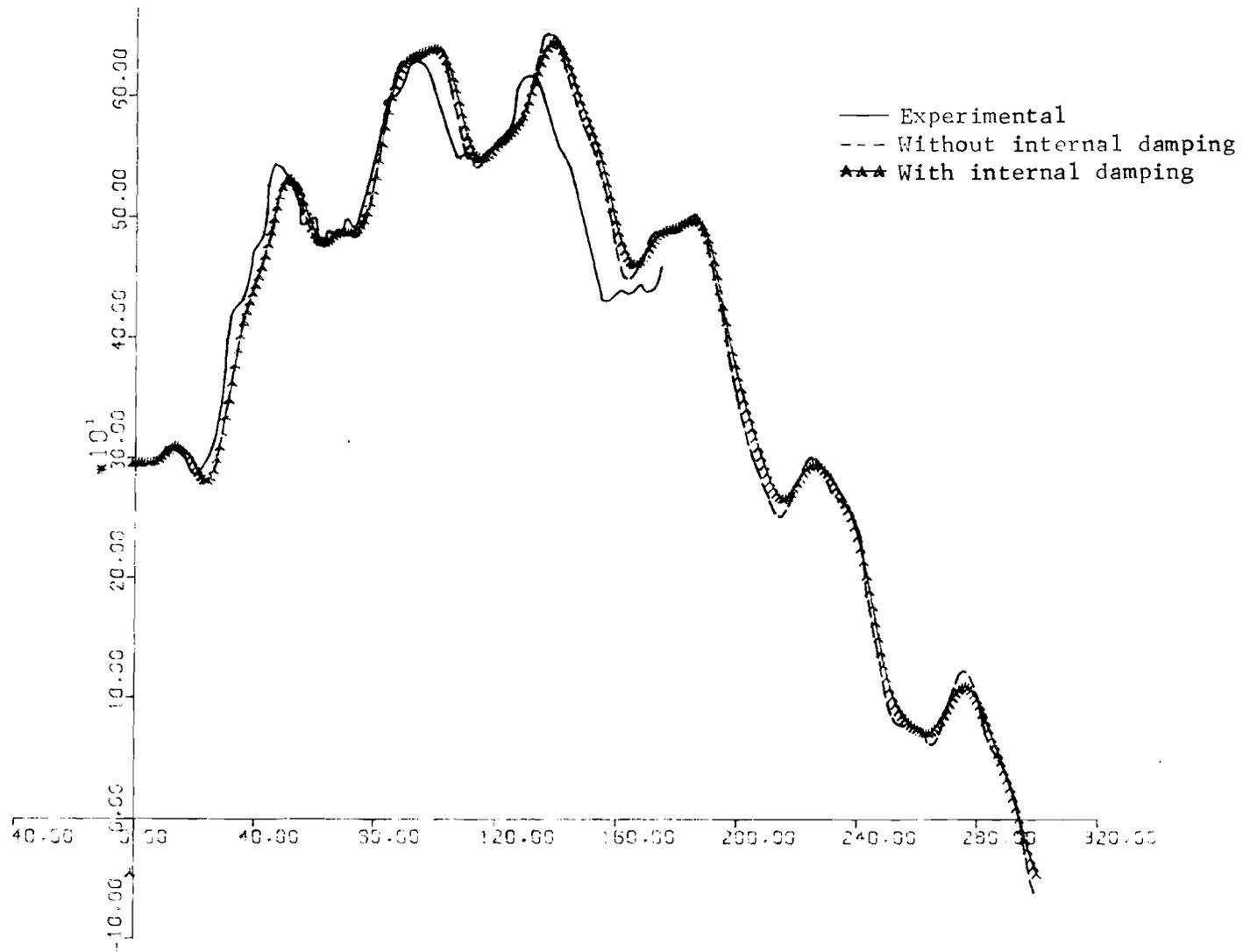
The computed bending moments along the time axis, with and without internal damping effect, and the measured bending moments are plotted for comparison, as shown in Fig 37. Results similar to those for the preceding problem are observed, but the effect of the internal damping factor is a little greater than in the preceding problem. Figure 38 shows the computed deflections along the time axis for the center station of the steel rod.

Example Problem 4-3. Simply Supported Steel Rod - Axial Pulse Base Time = 0.006 Second

In this problem the loaded axial pulse has a base time of 0.006 seconds and a peak value of 850 pounds. Again the computed bending moments along the time axis, with and without internal damping factor, and the measured bending moments are plotted as shown in Fig 39. Because of the higher frequency of vibrations, the mode shapes of the beam vary rapidly with time and cause large variation of curvature; therefore, the effect of the internal damping factor is much greater than in the preceding two cases. The estimated value of 20 lb-in.<sup>2</sup> sec/sta input for the internal damping factor is higher than the real value. Figure 40 shows the computed deflections along the time axis for the center station of the steel rod.

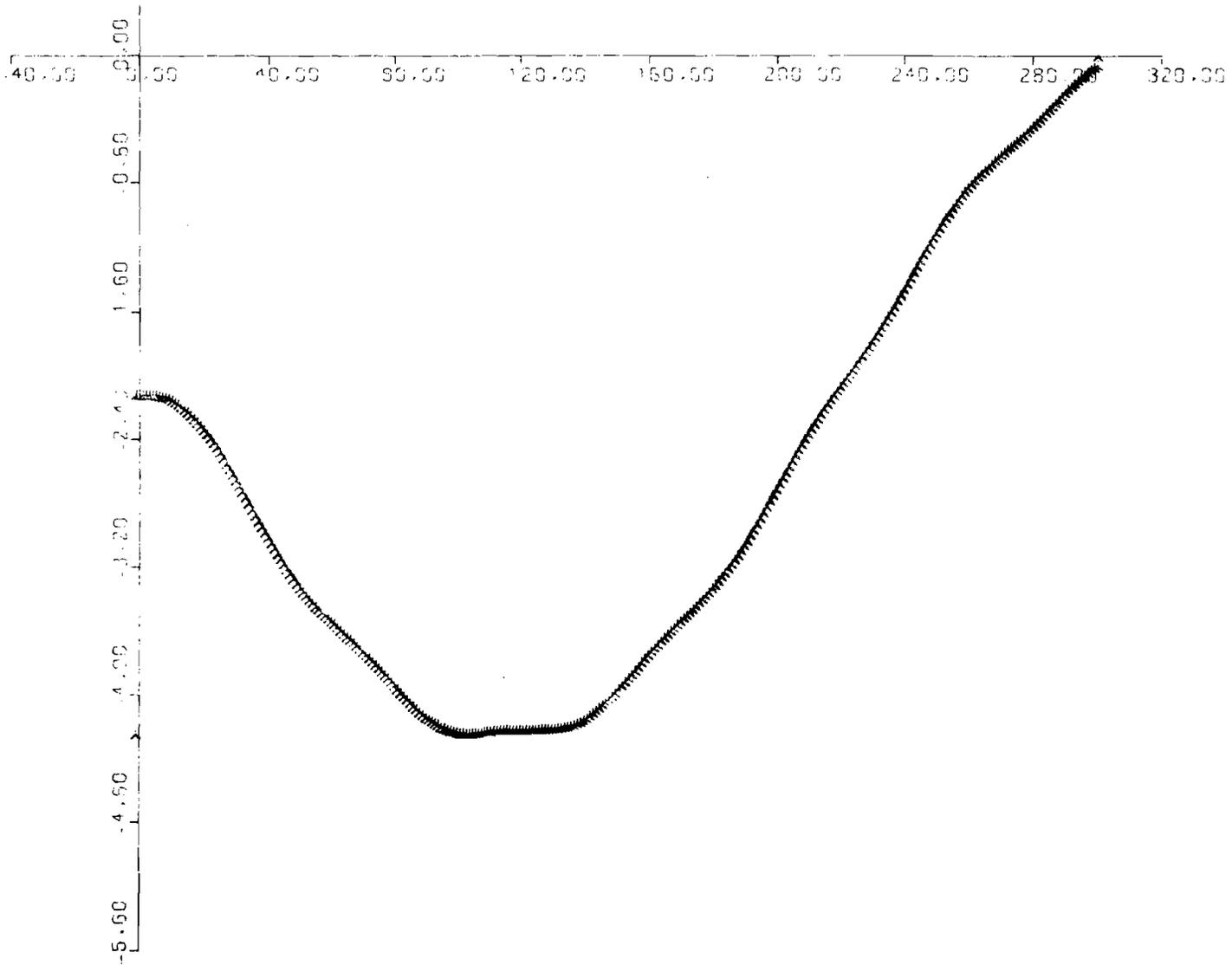
Example Problem 5. Partially Embedded Steel Pipe Pile Loaded by Wave Forces

This problem shows how DBC5 can be used to solve the dynamic response of laterally loaded piles to wave forces. Figure 41 presents a steel pipe of 30-inch outside diameter and 28-inch inside diameter which is partially embedded in the soil to a depth of 80 feet and extends slightly above the surface of the water which is 40 feet deep. The steel pipe is divided into 31 increments, each of which has a length of 48 inches. The flexural stiffness of the pipe is  $2.877 \times 10$  inch lb-in.<sup>2</sup>. The mass density of the pipe is 3.21



4-2 MVT BEAM STA = 20

Fig 37. Example Problem 4-2. Comparison of moments vs time for the center of beam.



4-2 DVT BEAM STR = 20

Fig 38. Example Problem 4-2. Deflection vs time for the center of beam.

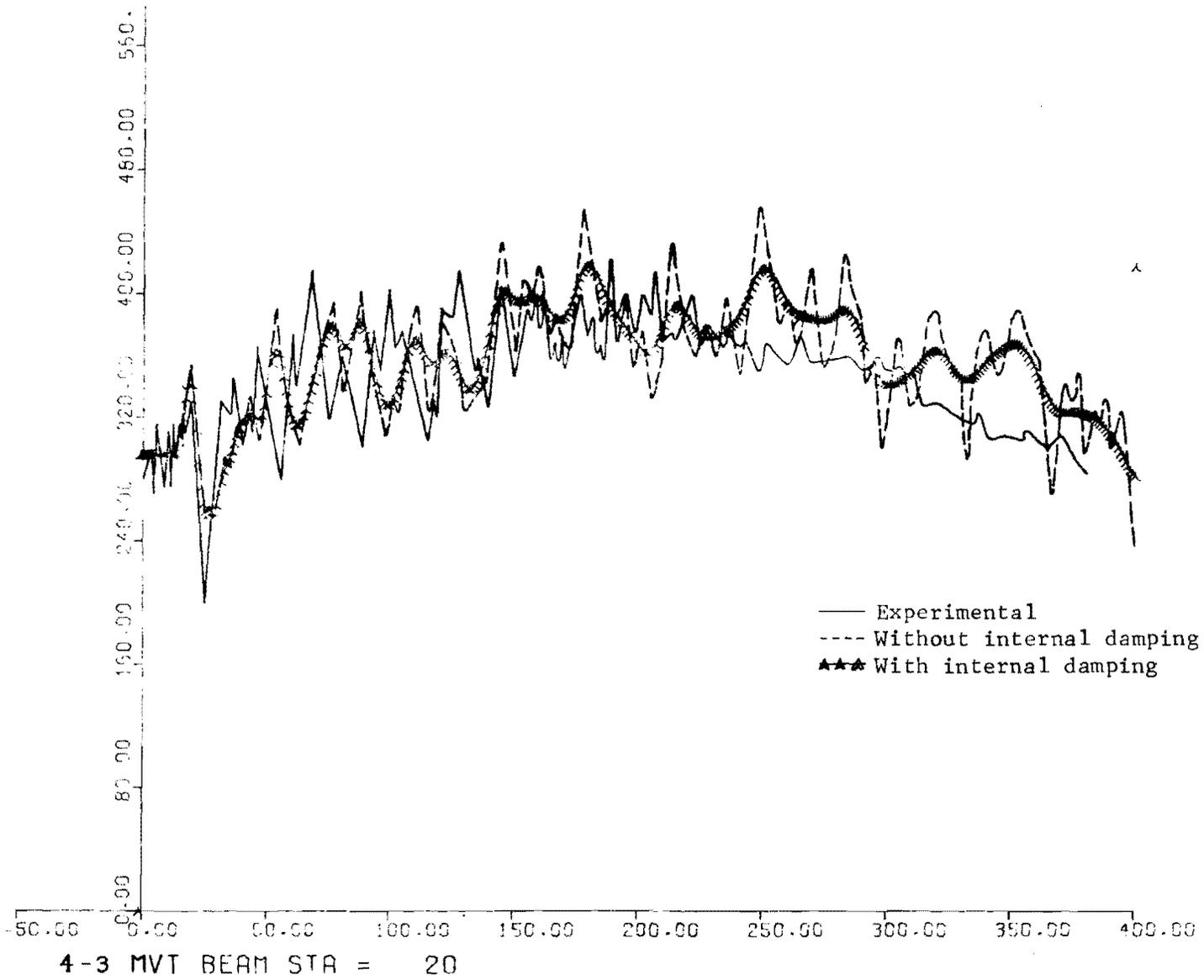
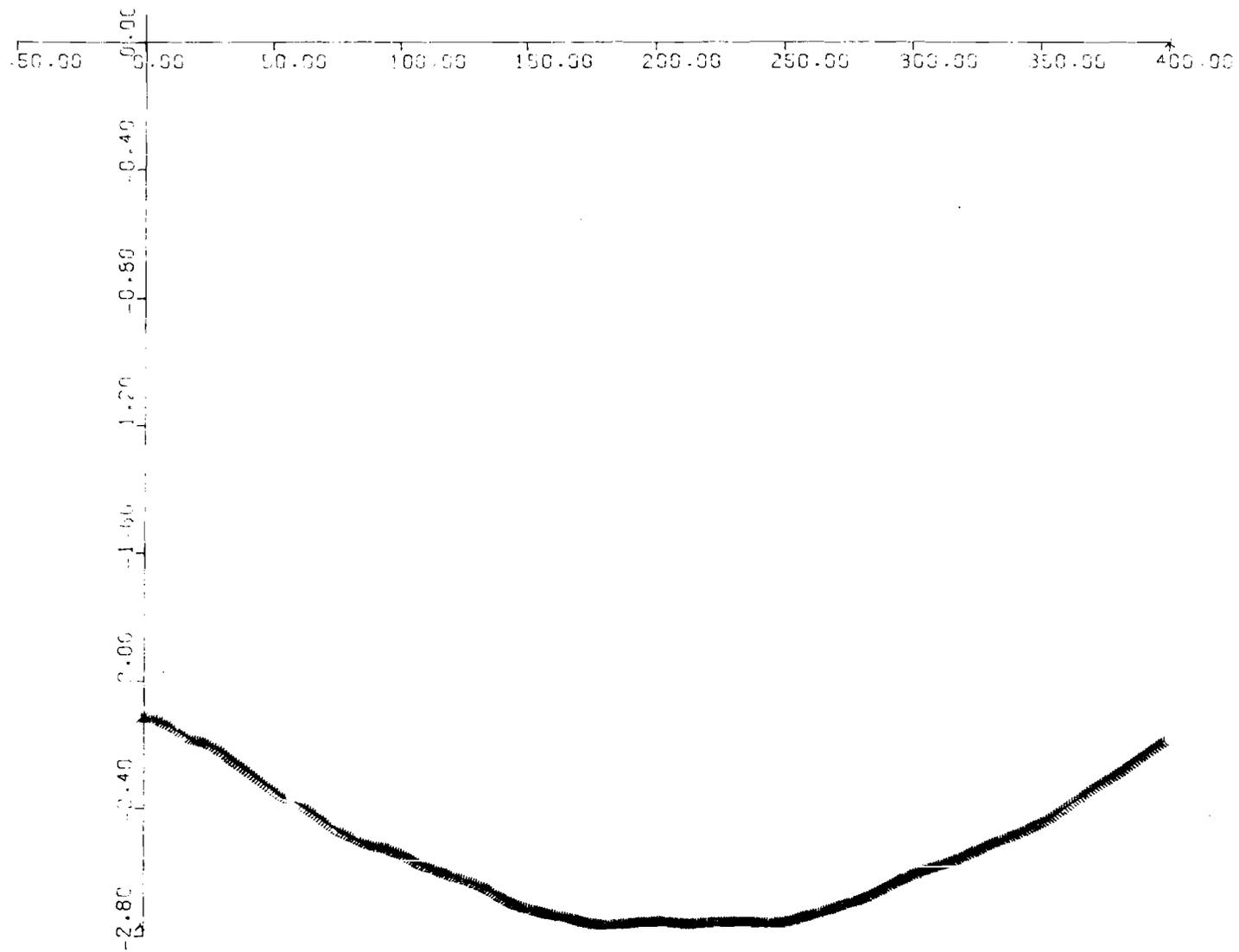
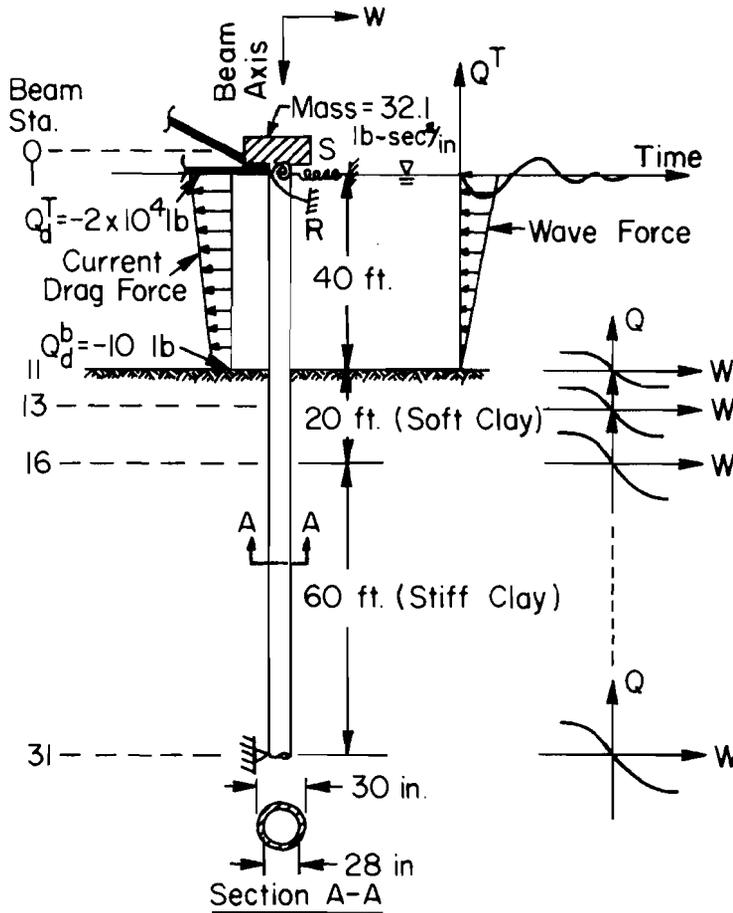


Fig 39. Example Problem 4-3. Comparison of moments vs time for the center of beam.



4-3 DVT BEAM STA = 20

Fig 40. Example Problem 4-3. Deflection vs time for the center of beam.



$$EI = 2.877 \times 10^{11} \text{ lb-in}^2$$

Mass density =  
3.21 lb-sec<sup>2</sup>/in/sta

Internal damping factor =  
1.918 × 10<sup>6</sup> lb-in<sup>2</sup>-sec/sta

$$S = 3 \times 10^7 \text{ lb/in}$$

$$R = 2.1 \times 10^{10} \text{ lb-in/rad}$$

$$Q_d^T = -2 \times 10^4 \text{ lb}$$

$$Q_d^b = -1 \times 10^4 \text{ lb}$$

Beam increments = 31

Beam increment length = 48 in.

Time increments = 400

Time increment length =  
1.0 × 10<sup>-2</sup> seconds

Idealized Wave Force at Beam Station 1

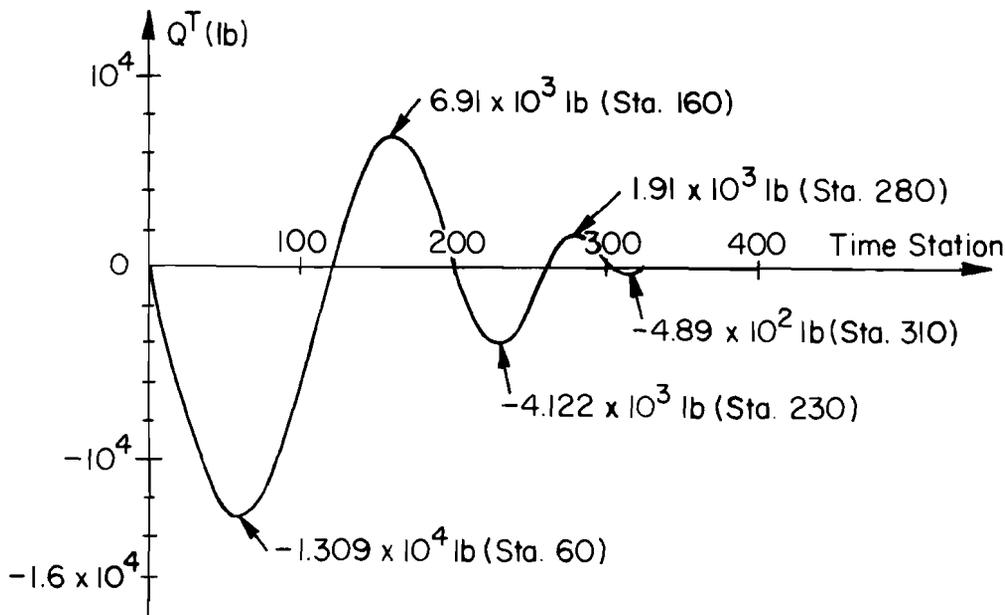


Fig 41. Example Problem 5. Partially embedded steel pipe loaded by idealized wave forces.

lb-sec<sup>2</sup>/in. from station 1 to station 31. A large mass, which is arbitrarily set at 32.1 lb-sec<sup>2</sup>/in., is lumped at station 0 to represent the weights of the structure, equipment, etc. The static weights of the pipe and the large mass lumped at station 0 are considered to be the axial compression forces, which are distributed from  $1.220 \times 10^4$  pound at station 1 to  $4.924 \times 10^4$  pound at station 31. An estimated value of  $1.918 \times 10^6$  lb-in.<sup>2</sup>-sec/sta is input for the internal damping factor.

The pipe is assumed to be simply supported at station 31 and assumed to have a linear spring support of  $3 \times 10^7$  lb/in. as well as a rotational restraint of  $2.1 \times 10^{10}$  lb-in./rad at station 1. From station 11 to station 31, the pipe is laterally supported by soil. The soil from station 11 to station 16 is assumed to be a soft clay which has a shear strength that varies from 200 lb/ft<sup>2</sup> at the top to 333.4 lb/ft<sup>2</sup> at the bottom. The soil below station 16 is assumed to be a stiff clay which has a shear strength of approximately 475 lb/ft<sup>2</sup>. Generation of the resistance-deflection curves for the soil supports at stations 11 and 13 is based on a technique presented by Matlock (Ref 6). The resistances and deflections for both support curves are described at 20 points as shown in Fig 42. The remaining support curves between stations 11 and 16 are linearly distributed.

The nonlinear characteristics of the resistance-deflection curves for the soil supports from station 16 to station 31 are obtained from the data for the experiment done by Matlock and Vora in Ref 8. The nonlinear resistance-deflection curves are also described at 20 points, as shown in Fig 43.

Current drag forces are assumed to vary from  $-2 \times 10^4$  pound at station 1 to  $-1 \times 10^4$  pound at station 11. Idealized wave forces are shown at the bottom of Fig 41. The wave forces are assumed to vary linearly from station 1 to station 11. The time increment length is equal to  $1.0 \times 10^{-2}$  seconds. Four hundred time stations are solved for this problem.

The computed deflections and moments versus time for stations 10 and 21 are plotted by the program, as shown in Figs 44 and 45, respectively. Figure 46 shows the computed deflections along the beam axis for time stations 60 and 160. Figure 47 shows the computed deflection along the beam axis for time stations 230 and 280. Figure 48 shows the computed moments along the beam axis for time stations 60 and 320. All plots in this problem are automatically plotted using the ball-point plotting method.

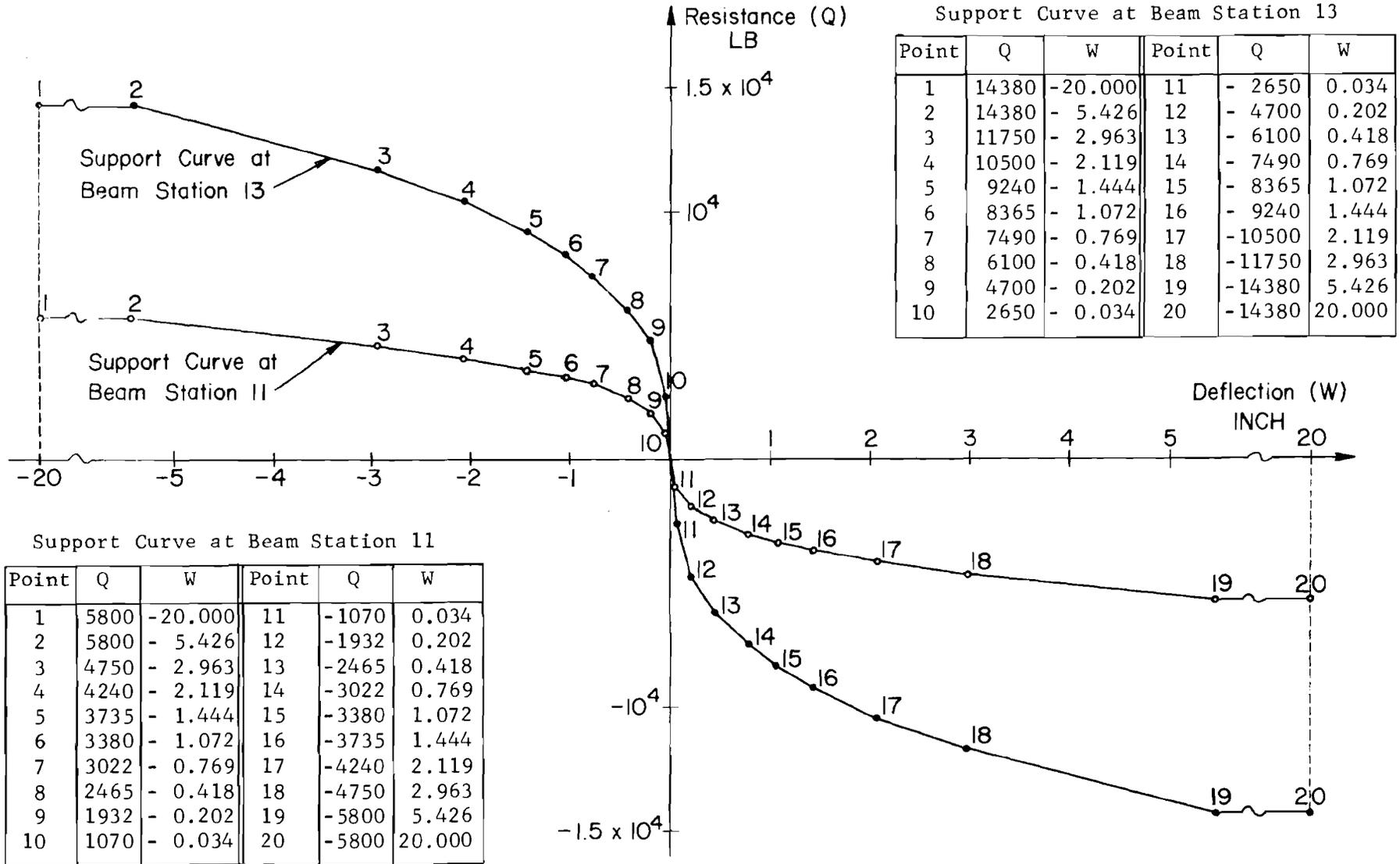


Fig 42. Symmetric resistance-deflection curves of soft clay at beam stations 11 and 13.

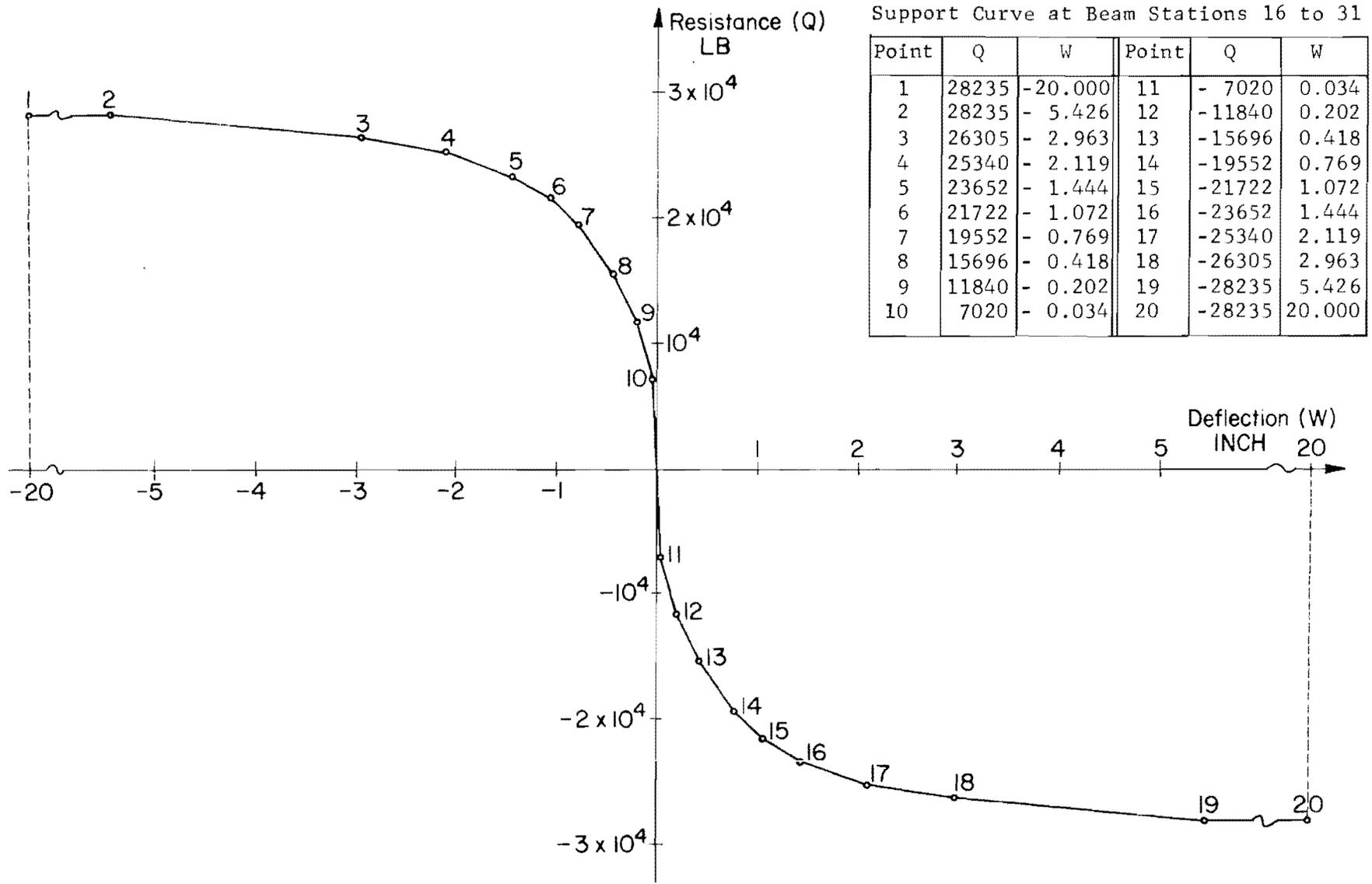
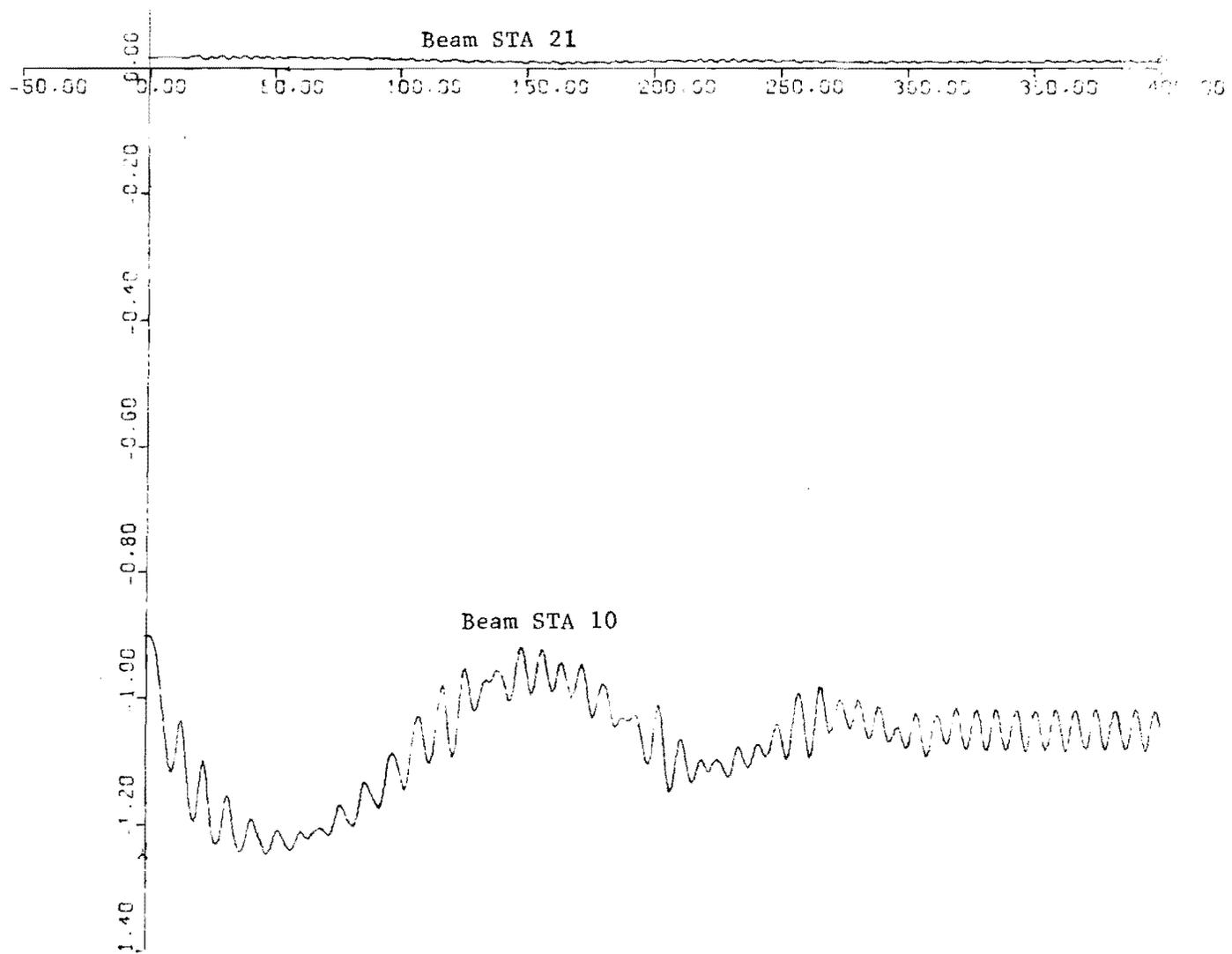
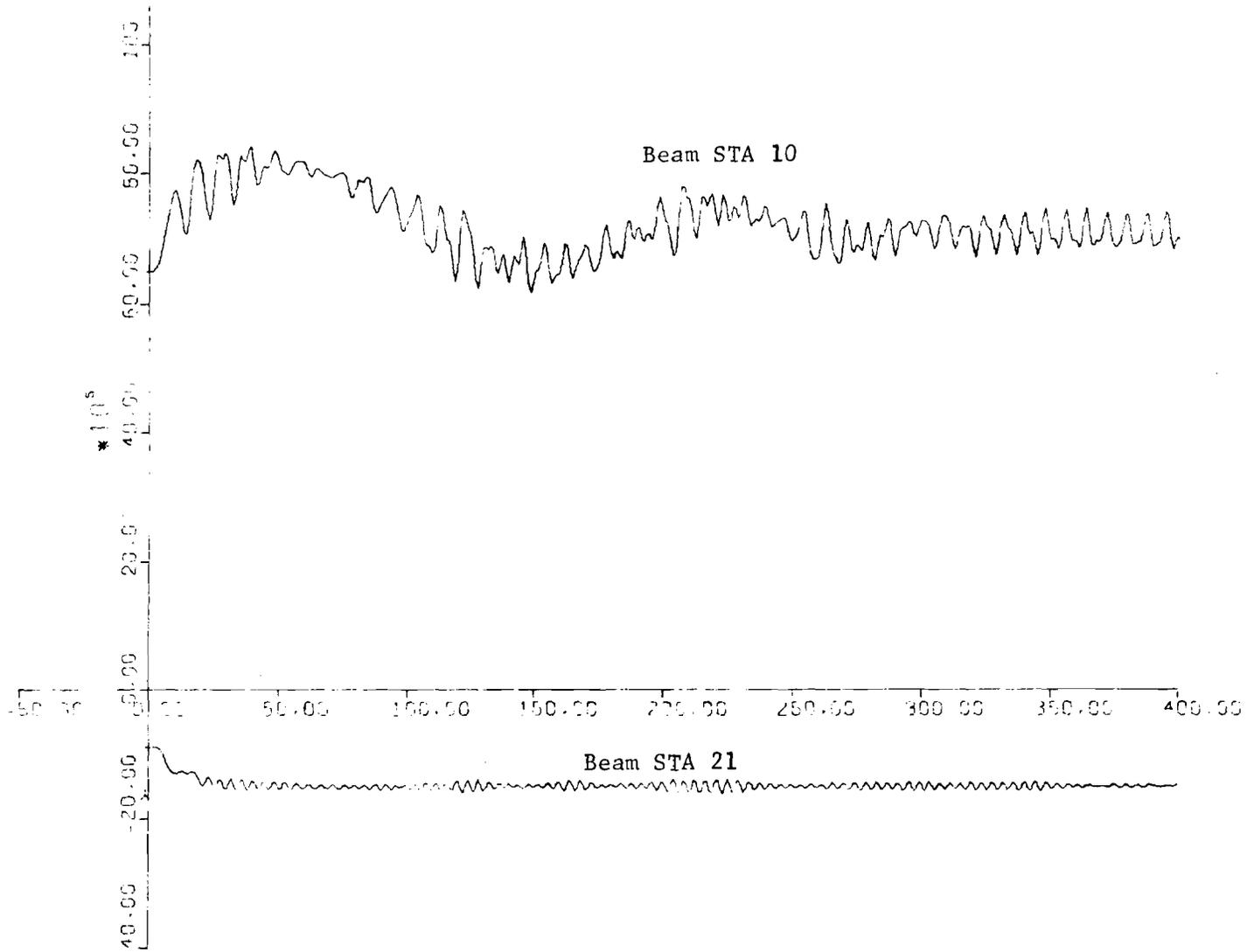


Fig 43. Symmetric resistance-deflection curves of stiff clay at beam stations 16 to 31.



5 DVT BEAM STA = 10 21

Fig 44. Example Problem 5. Deflection vs time for beam stations 10 and 21.



5 MVT BEAM STA = 10 21

Fig 45. Example Problem 5. Moment vs time for beam stations 10 and 21.

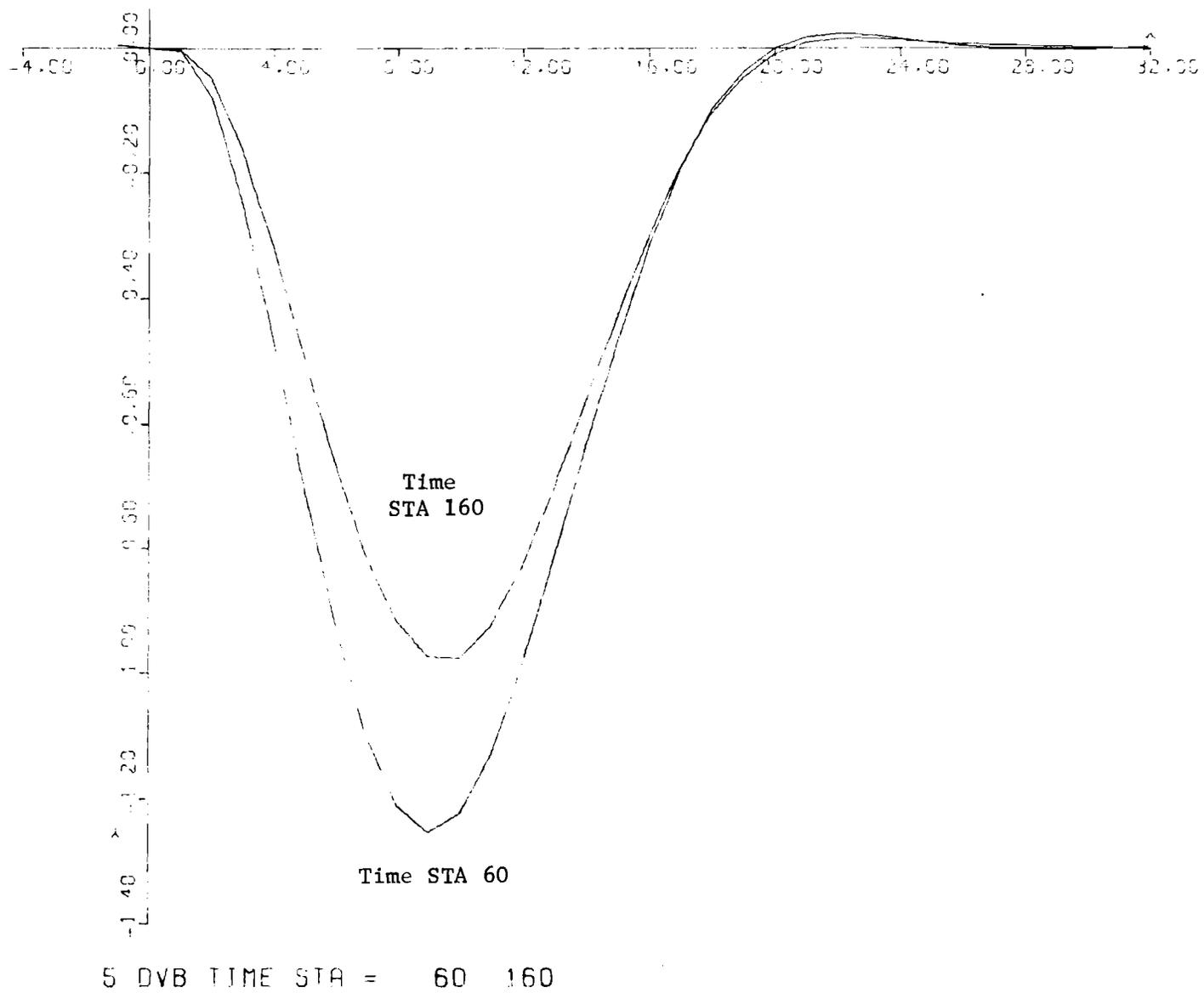


Fig 46. Example Problem 5. Deflection along the beam axis for time stations 60 and 160.

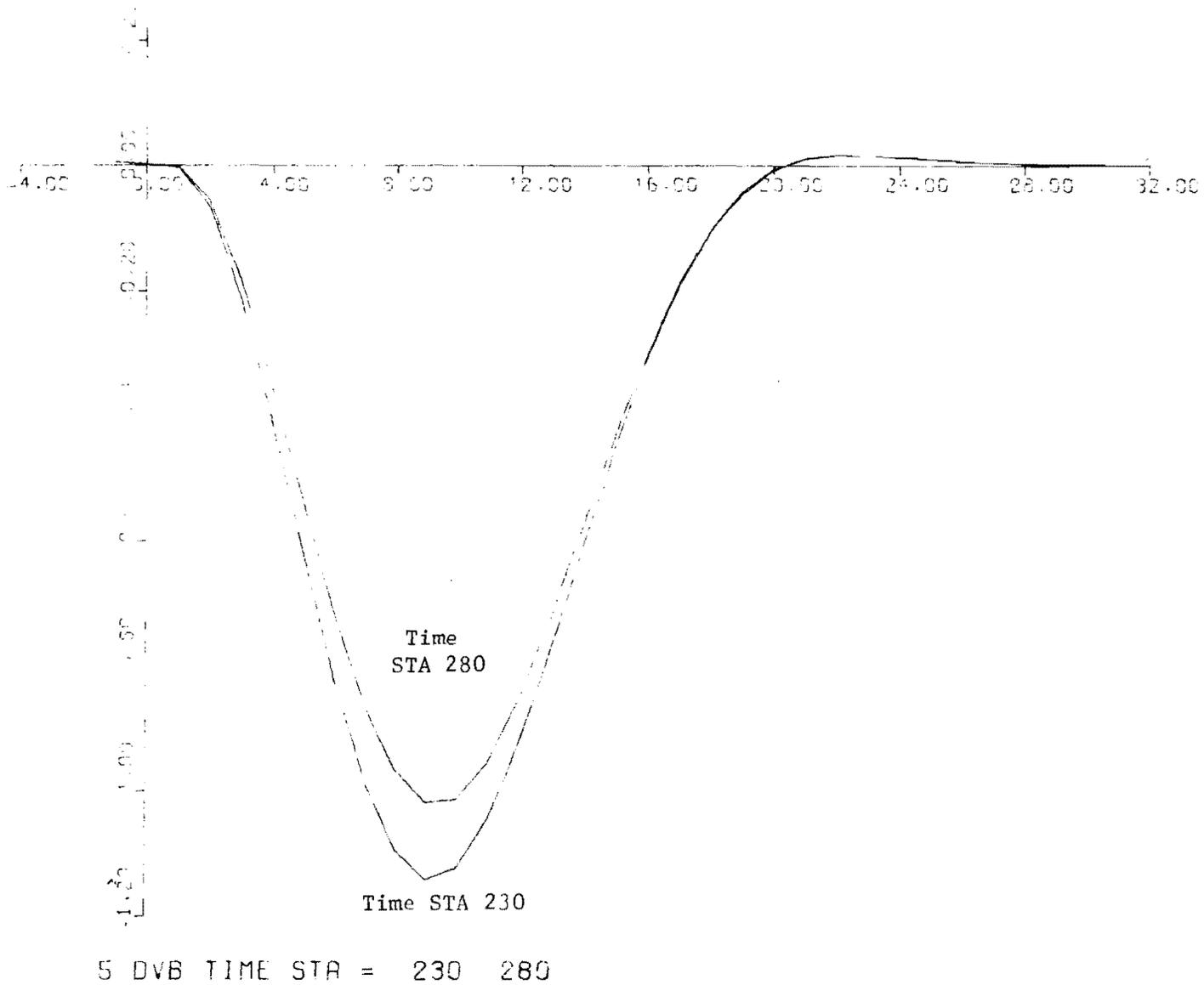
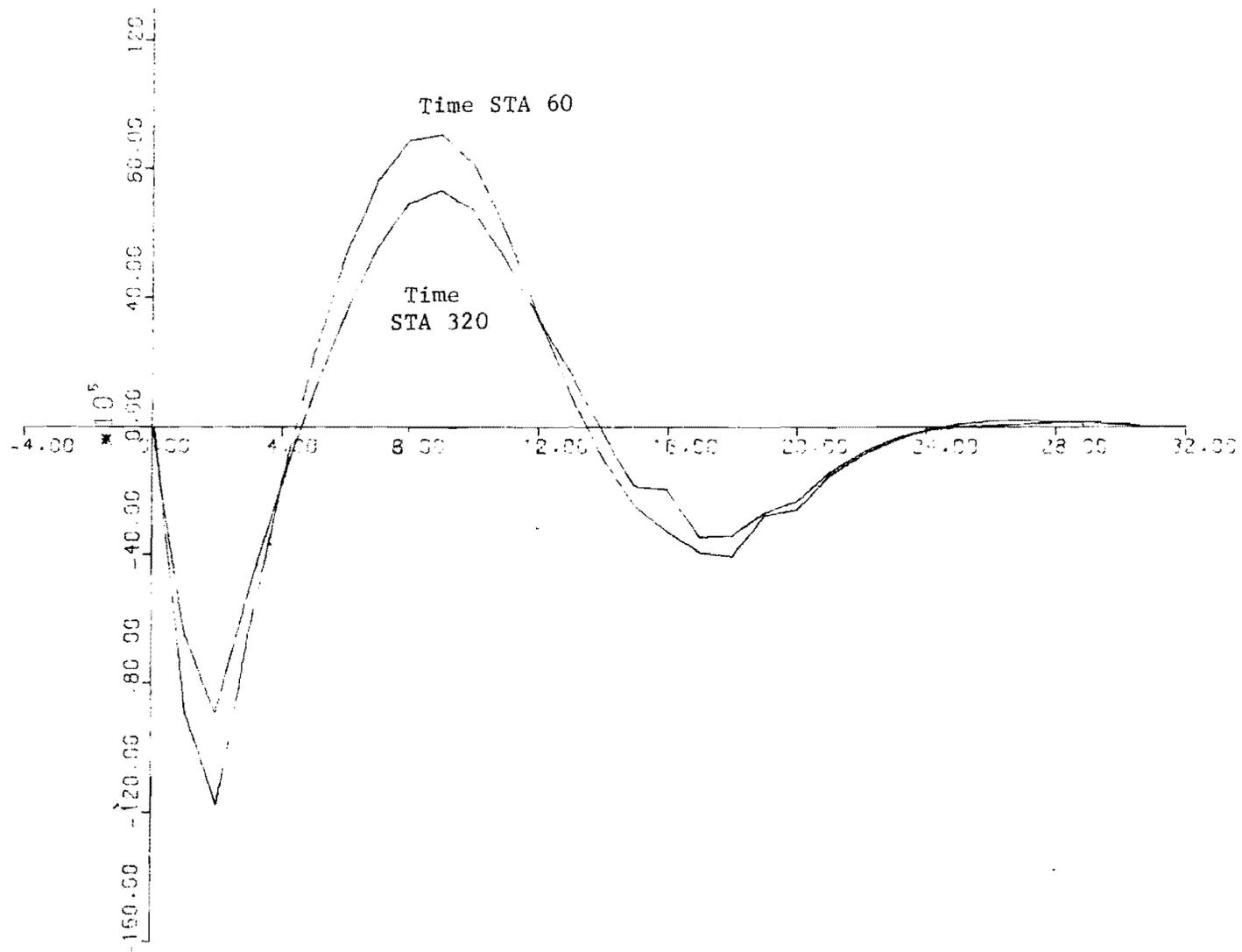


Fig 47. Example Problem 5. Deflection along the beam axis for time stations 230 and 280.



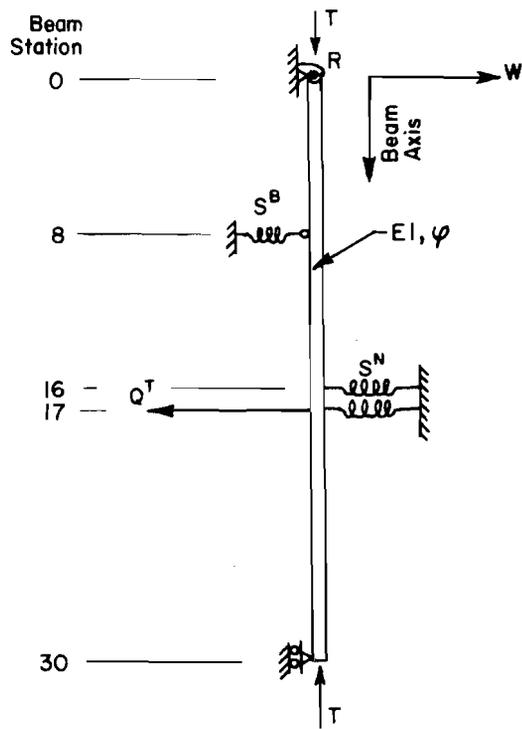
S MVB TIME STA = 60 320

Fig 48. Example Problem 5. Moment along the beam axis for time stations 60 and 320.

Example Problem 6. Three-Span Beam with One-Way and Symmetric Nonlinear Supports Loaded with a Transient Pulse

This problem is intended to demonstrate that the option of switching from a spring-load (tangent modulus method) iteration to a load iteration technique which is built into Program DBC5 is useful when the tangent modulus method fails because of lift-off of the one-way support or yielding of the nonlinear supports. Figure 49 shows a three-span beam (30 beam stations) which is simply supported at the ends, one-way supported at station 8 and nonlinearly supported at stations 16 and 17 and is loaded with a transient pulse at station 17. The beam has a uniform flexural stiffness of  $2.4 \times 10^{10}$  lb-in.<sup>2</sup> and a uniform mass density of  $5.184 \times 10^{-1}$  lb-sec<sup>2</sup>/in./sta. A rotational restraint of  $3.0 \times 10^{10}$  is applied at beam station 0 and a constant axial compression force of  $10^5$  pounds is assumed. The negative one-way resistance-deflection curve of the support at beam station 8 is shown in Fig 50. The symmetric resistance-deflection curve of the support at beam station 17 is also shown in Fig 50. The nonlinear characteristic of the support at beam station 16 is linearly distributed between beam station 15, which has no resistance to deflection, and beam station 17. The applied transient pulse at beam station 17 is shown at the bottom of Fig 49. The time increment length is equal to  $4 \times 10^{-2}$  seconds and 40 time stations are solved for this problem.

The computed results and the monitor deflections at beam stations 8, 15, 16, and 17 during the iteration process are printed on the computer output. It is seen that the tangent modulus method failed at time stations 5, 9, 13, 16, and 24 due to abrupt changes of time-dependent loads or lift-off of the support at beam station 8 at these time stations; therefore the program automatically switches to the load iteration method and succeeds in obtaining the closed solutions. If the time increment length is reduced to  $4 \times 10^{-3}$  seconds and the same problem is rerun for 400 time stations, the tangent modulus method works nicely at each time station except three at which the beam starts to lift off the support at beam station 8. This is because smoother changes of time-dependent loads are obtained due to the small time increment length and, therefore, the tangent modulus method only fails at some of the critical time stations at which lift-off of the support occurred. The computed deflections and moments versus time for beam stations 8 and 17 are plotted using the printer plotting method, as shown in Figs 51 and 52,



$EI = 2.4 \times 10^{10} \text{ lb-in}^2$   
 $\phi = 0.5184 \text{ lb-sec}^2/\text{in/sta}$   
 $T = 10^5 \text{ lb}$   
 $R = 3 \times 10^{10} \text{ lb-in/rad}$   
 $S^B$  and  $S^N$  see Fig 50  
 Beam increments = 30  
 Beam increment length = 48 inches  
 Time increments = 40  
 Time increment length =  $4 \times 10^{-2}$  seconds

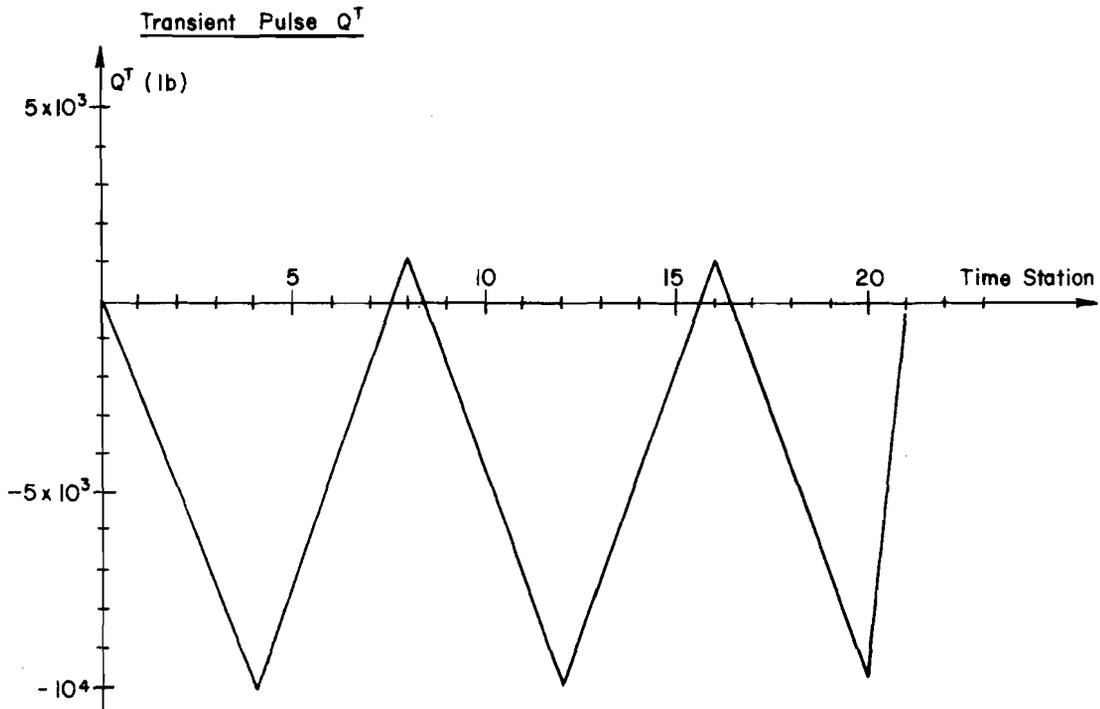


Fig 49. Example Problem 6. Three-span beam loaded by transient pulse.

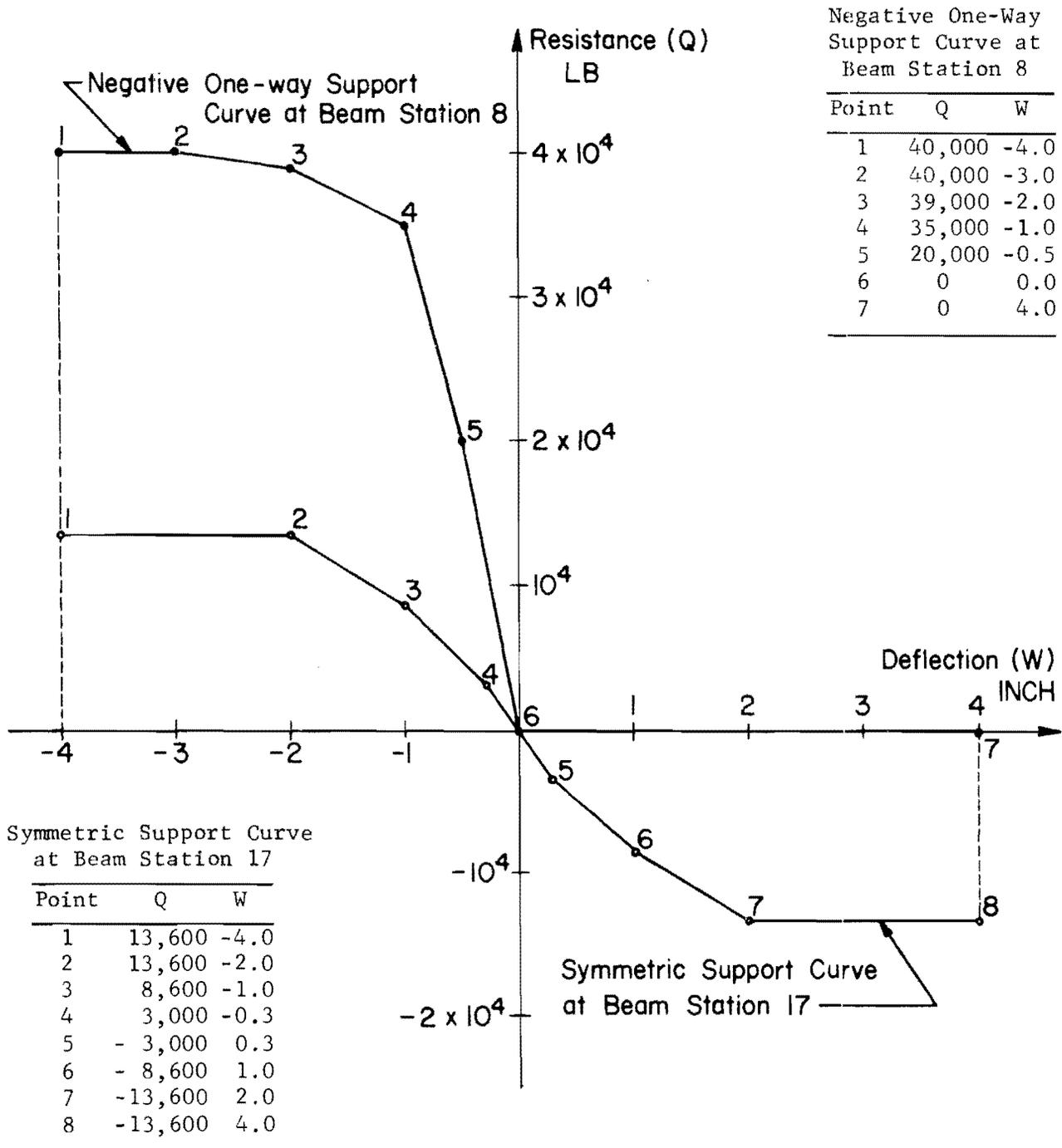


Fig 50. Resistance-deflection curves of the supports at beam stations 8 and 17.

PROGRAM DRCS - MASTER - JACK CHAN - MATLOCK - DECK)-REVISION DATE = 26 JUN 71  
 EXAMPLE PROBLEMS FOR PROGRAM DRCS BY JACK CHAN (JUNE 1971)  
 DYNAMIC BEAM-COLUMN PROGRAM ( 5-1-5 IMPLICIT OPERATOR )

PROB (CONTD)

6 THREE-SPAN BEAM WITH ONE-WAY AND SYMMETRIC NONLINEAR SUP LOADED BY T.P  
 \*\*\*\*\* PLOT OF DEFLECTION VS TIME FOR BEAM STATIONS OF: \*\*\*\*\*

1 CURVE (\*) = 8  
 2 CURVE (+) = 17

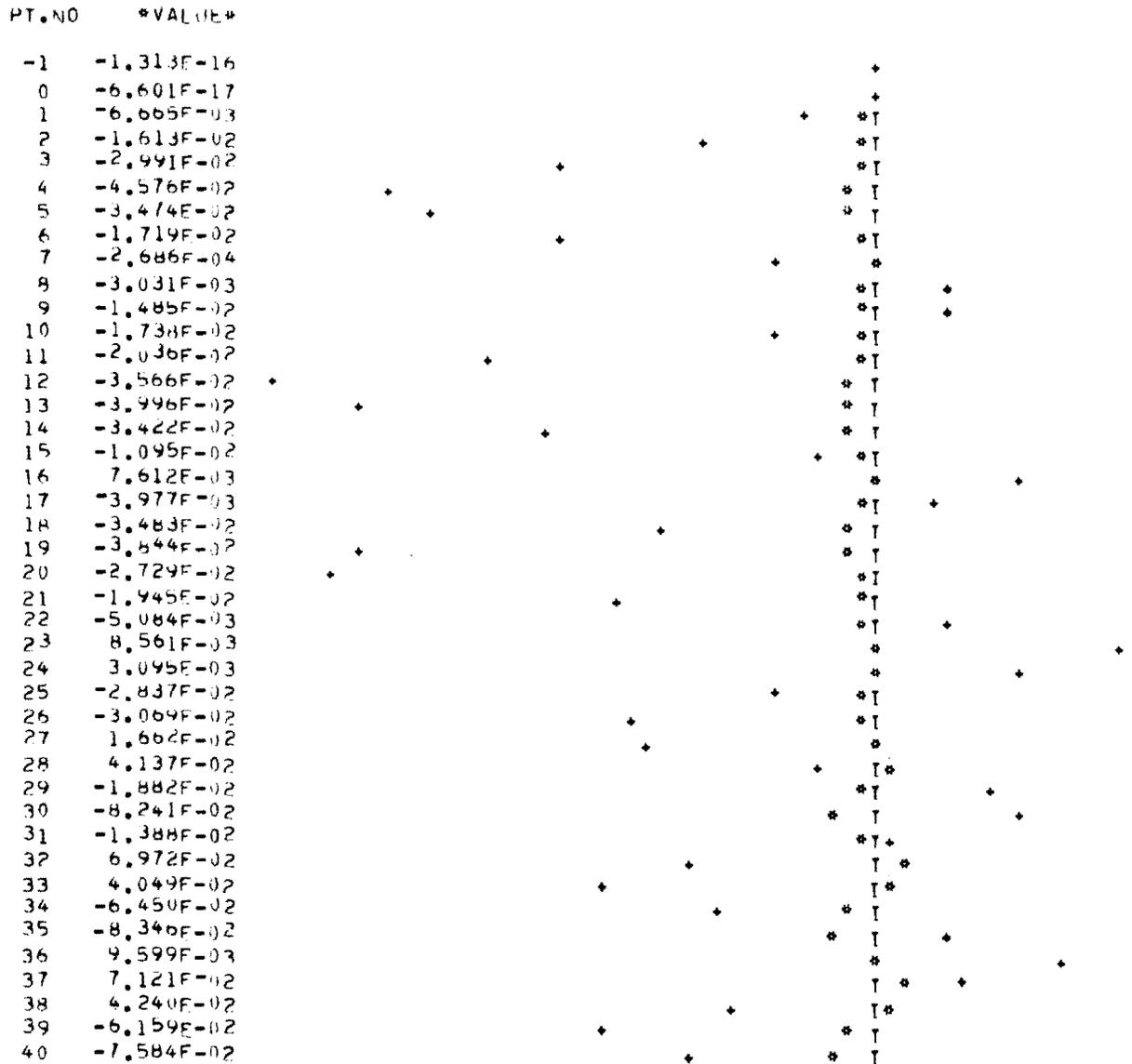


Fig 51. Example Problem 6. Deflection vs time for beam stations 8 and 17.

PROGRAM DBC5 - MASTER - JACK CHAN - MATLOCK - DECK1-REVISION DATE = 26 JUN 71  
 EXAMPLE PROBLEMS FOR PROGRAM DBC5 BY JACK CHAN JUNE 1971  
 DYNAMIC BEAM-COLUMN PROGRAM ( 5-1-5 IMPLICIT OPERATOR )

PROB (CONTD)

6 THREE-SPAN BEAM WITH ONE-WAY AND SYMMETRIC NONLINEAR SUP LOADED BY T.P  
 \*\*\*\*\* PLOT OF BEND. MOM. VS TIME FOR BEAM STATIONS OF: \*\*\*\*\*

1 CURVE (\*) = 8  
 2 CURVE (+) = 17

PT.NO \*VALUE\*

-1 -3.358E-10  
 0 -3.891E-10  
 1 -3.578E+04  
 2 -9.538E+04  
 3 -1.812E+05  
 4 -2.722E+05  
 5 -2.115E+05  
 6 -1.409E+05  
 7 -4.657E+04  
 8 1.139E+04  
 9 -5.331E+03  
 10 -6.771E+04  
 11 -1.828E+05  
 12 -2.875E+05  
 13 -2.433E+05  
 14 -1.828E+05  
 15 -5.922E+04  
 16 3.178E+04  
 17 1.222E+04  
 18 -9.157E+04  
 19 -2.589E+05  
 20 -2.965E+05  
 21 -1.227E+05  
 22 7.166E+04  
 23 1.018E+05  
 24 -1.266E+04  
 25 -8.175E+04  
 26 -3.985E+04  
 27 -1.949E+04  
 28 -4.340E+04  
 29 -3.052E+04  
 30 2.709E+04  
 31 1.262E+04  
 32 -5.033E+04  
 33 -7.626E+04  
 34 -5.714E+04  
 35 -1.049E+04  
 36 2.866E+04  
 37 4.163E+04  
 38 -4.764E+04  
 39 -9.793E+04  
 40 -7.125E+04

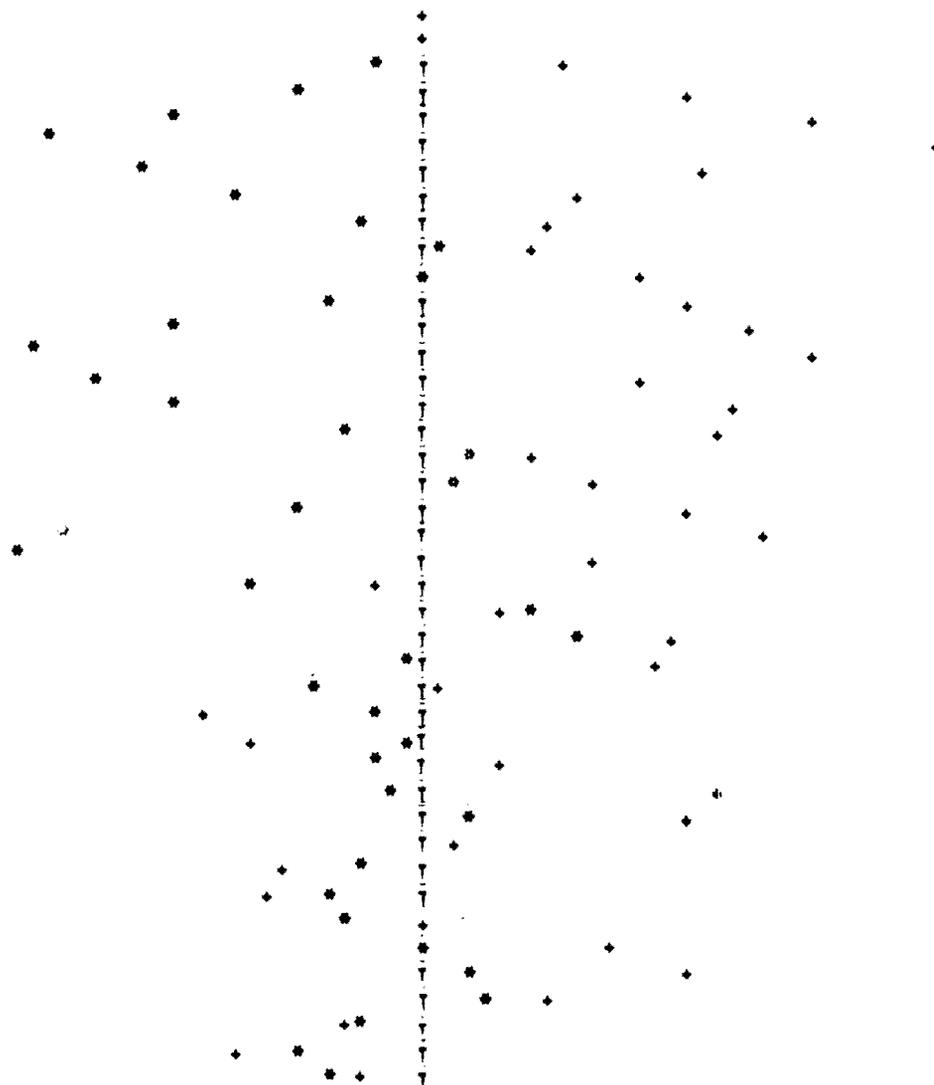


Fig 52. Example Problem 6. Moment vs time for beam stations 8 and 17.

respectively. Figure 53 shows the computed deflections along the beam axis for time stations 4, 8, and 20. Figure 54 shows the computed moments along the beam axis for time stations 12, 16, and 30.

Example Problem 7. Partially Embedded Steel Pipe Pile Excited by Sinusoidal Earthquake-Induced Forces

The capability of Program DBC5 to analyze the response of a partially embedded steel pipe pile which is excited by sinusoidal earthquake-induced forces is illustrated with problem 7. Figure 55(a) shows a 12.75-inch outside diameter  $\times$  0.5-inch wall steel pipe pile embedded in the soil to a depth of 40 feet and extending 10 feet above the ground surface. The pile has a flexure stiffness of  $10.9 \times 10^9$  lb-in.<sup>2</sup> and is divided into 50 beam increments. Each beam increment has a length of 12 inches. An axial compression force of 10,000 pounds is assumed in the pile. From beam station 0 to beam station 40, the lumped mass density of soil and pile is assumed to be 5.184 lb-sec<sup>2</sup>/in./sta. From beam station 40 to beam station 50, the lumped mass density of soil and lateral load is assumed to be 2.592 lb-sec<sup>2</sup>/in./sta. The negative and positive one-way resistance-deflection curves, which describe the lateral soil supports from beam station 0 to beam station 40, are shown in Fig 55(b) and (c), respectively. The time increment length is  $5 \times 10^{-3}$  seconds and 160 time stations are solved for this problem.

The pile is excited by a sinusoidal earthquake-induced force, as shown in Fig 55(d). The sinusoidal earthquake induced force has a base time of 0.1 second (equivalent to 20 time stations) and is induced from an assumed sinusoidal acceleration with a maximum value of 1/10 G ( $G = 386$  in./sec<sup>2</sup>) and a base time of 0.1 second. Based on the discrete-element beam-column model, the applied transverse load  $Q^T$  is equal to the product of deflection  $W$  and spring force  $S^N$ . By double integration of the equation of sinusoidal acceleration and evaluation of the constants of integration, the relation between the maximum deflection ( $W_{\max}$ ) and the maximum acceleration ( $\ddot{W}_{\max}$ ) is found to be

$$W_{\max} = \ddot{W}_{\max} t_c^2 / \pi^2$$

where  $t_c$  is the base time and  $\pi$  is equal to 3.1416. If the nonlinear spring

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 EXAMPLE PROBLEMS FOR PROGRAM DBC5 BY JACK CHAN JUNE 1971  
 DYNAMIC BEAM-COLUMN PROGRAM ( 5-1-5 IMPLICIT OPERATOR )

PROG (CONTD)

6 THREE-SPAN BEAM WITH ONE-WAY AND SYMMETRIC NONLINEAR SUP LOADED BY T.P  
 \*\*\*\*\* PLOT OF DEFLECTIONS ALONG BEAM AXIS AT TIME STATIONS OF. \*\*\*\*\*

1 CURVE (\*) = 4  
 2 CURVE (+) = 8  
 3 CURVE (x) = 20

PT.NO \*VALUE\*

-1 5.388E-03  
 0 0.  
 1 5.754E-03  
 2 1.787E-02  
 3 3.148E-02  
 4 4.173E-02  
 5 4.370E-02  
 6 3.293E-02  
 7 4.549E-03  
 8 -4.576E-02  
 9 -1.222E-01  
 10 -2.203E-01  
 11 -3.353E-01  
 12 -4.625E-01  
 13 -5.970E-01  
 14 -7.339E-01  
 15 -8.654E-01  
 16 -9.957E-01  
 17 -1.111E+00  
 18 -1.192E+00  
 19 -1.238E+00  
 20 -1.250E+00  
 21 -1.229E+00  
 22 -1.178E+00  
 23 -1.098E+00  
 24 -9.927E-01  
 25 -8.640E-01  
 26 -7.155E-01  
 27 -5.510E-01  
 28 -3.743E-01  
 29 -1.892E-01  
 30 0.  
 31 1.892E-01

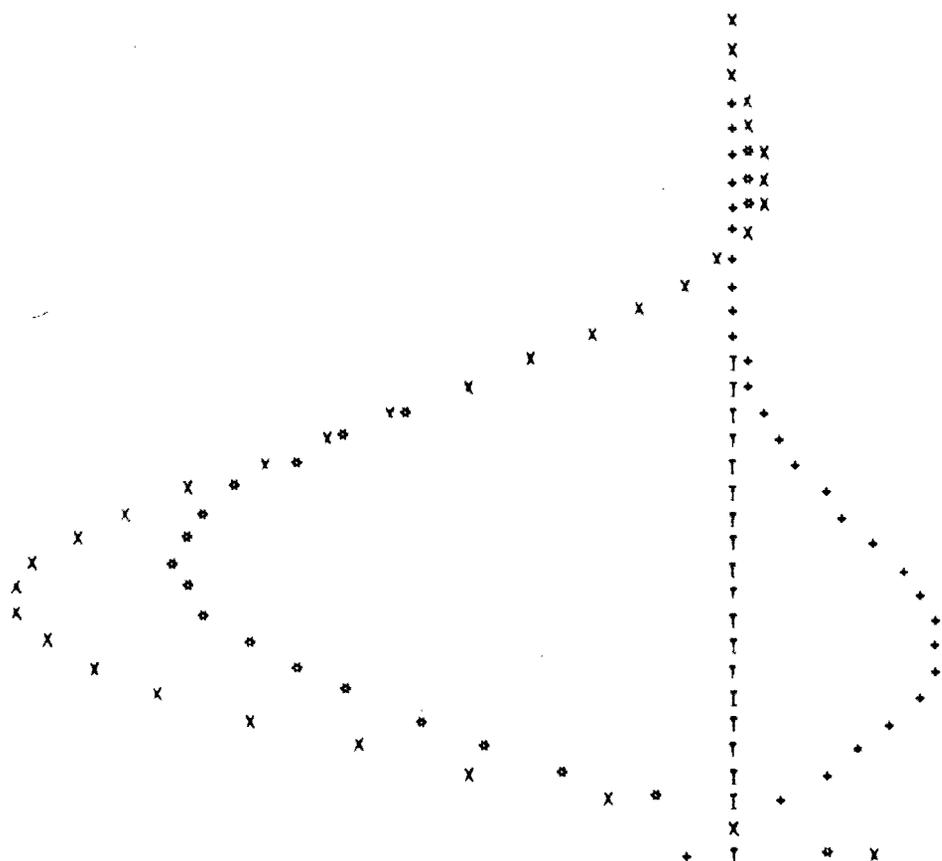


Fig 53. Example Problem 6. Deflection along the beam axis for time stations 4, 8, and 20.

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 EXAMPLE PROBLEMS FOR PROGRAM DBCS BY JACK CHAN JUNE 1971  
 DYNAMIC BEAM-COLUMN PROGRAM ( 5-1-5 IMPLICIT OPERATOR )

PROR (CONTD)

6 THREE-SPAN BEAM WITH ONE-WAY AND SYMMETRIC NONLINEAR SUP LOADED BY T.P  
 \*\*\*\*\* PLOT OF BEND. MOM. ALONG BEAM AXIS AT TIME STATIONS OF, \*\*\*\*\*

1 CURVE (#) = 12  
 2 CURVE (+) = 16  
 3 CURVE (X) = 30

PT.NO \*VALUE\*

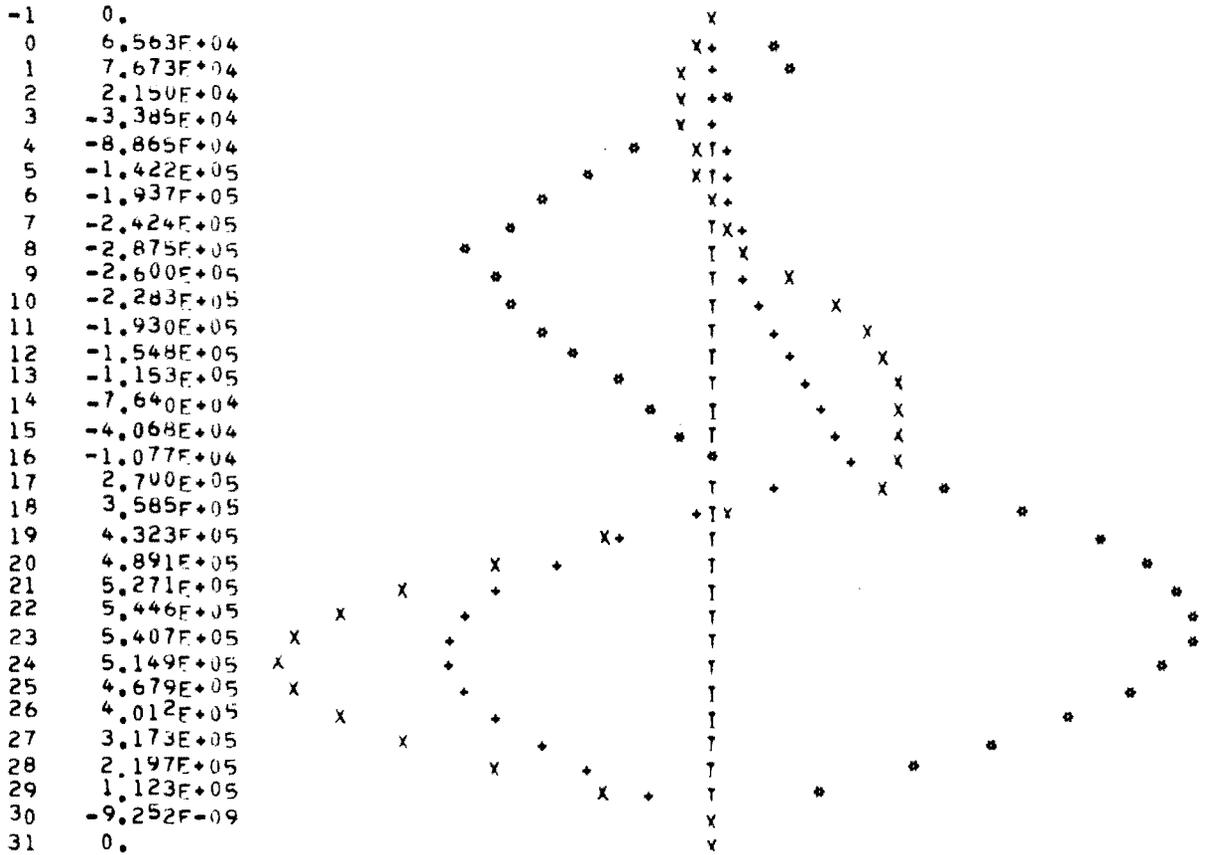
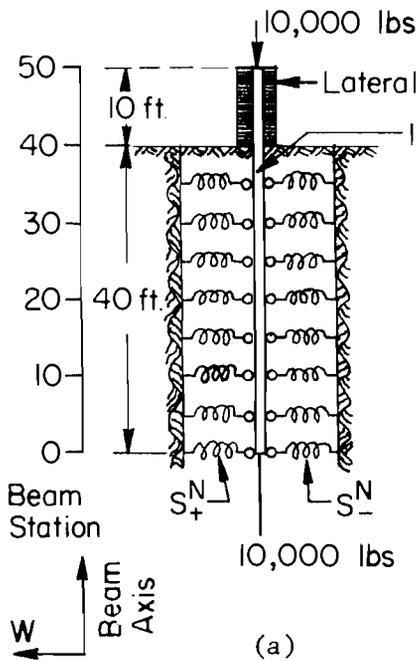


Fig 54. Example Problem 6. Moment along the beam axis for time stations 12, 16, and 30.



$$F = EI = 30 \times 10^6 \times \frac{\pi}{64} [(12.75)^4 - (11.75)^4]$$

$$= 10.9 \times 10^9 \text{ lb-in}^2$$

Mass Density:

Sta 0 to 40: Weight of soil + pile  $\approx 200 \text{ lbs/ft}$

$$\therefore \rho = 200/386.4 = 5.184 \text{ lb-sec}^2/\text{in/sta}$$

Sta 40 to 50: Weight of pile + lateral load

$\approx 10,000 \text{ lbs.}$  Distributing on 10' length.

$$\therefore \rho = 10,000/(10 \times 386.4) = 2.592 \text{ lb-sec}^2/\text{in/sta}$$

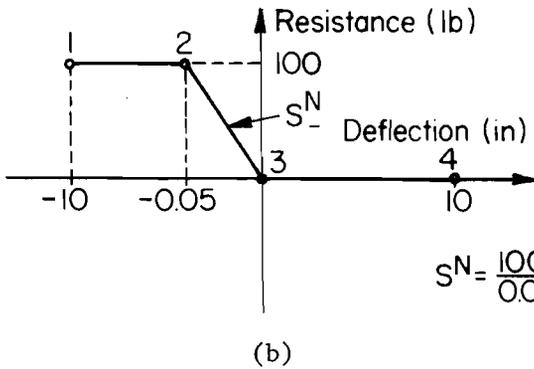
Number of beam increments = 50

Beam increment length = 12 inches

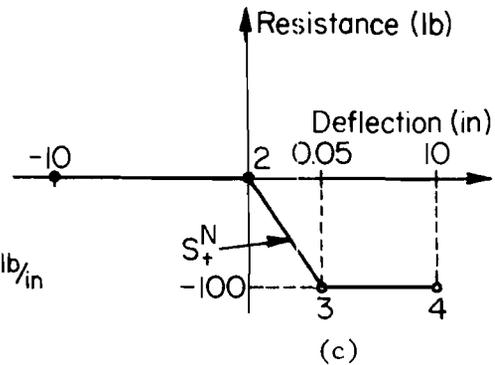
Number of time increments = 160

Time increment length =  $5 \times 10^{-3}$  seconds

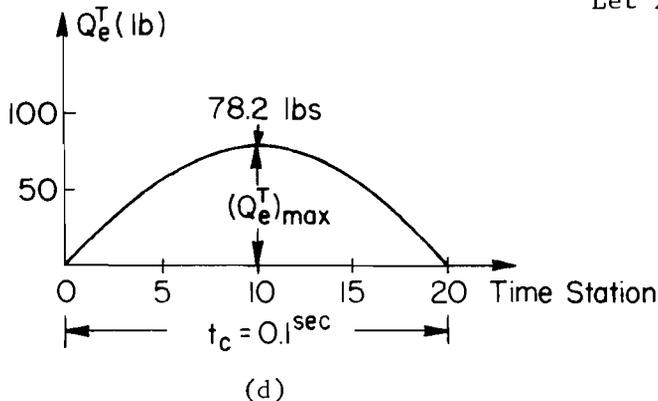
One-Way Resistance-Deflection Curves  $S_-^N$  and  $S_+^N$



$$S_-^N = \frac{100}{0.05} = 2000 \text{ lb/in}$$



Earthquake-Induced Forces



Let  $\ddot{X}_{\max} = 1/10 G = 38.6 \text{ in/sec}^2$

$$(Q_e^T)_{\max} \approx \frac{S_t^N t_c^2}{\pi^2} \ddot{X}_{\max}$$

$$= \frac{2000 \times 0.1^2}{\pi^2} \times 38.6$$

$$= 78.2 \text{ lbs}$$

Fig 55. Example Problem 7. Partially embedded pile excited by sinusoidal earthquake-induced forces.

force  $S^N$  can be assumed to be constant, then the maximum sinusoidal earthquake-induced force can be obtained by the equation

$$Q_{\max}^T = S^N t_c^2 \ddot{W}_{\max} / \pi^2$$

The earthquake-induced forces are applied at beam stations 0 to 40 to approximately describe the ground movements.

The computed resistance-deflection curves of the negative and positive one-way supports at beam station 20 are shown in Fig 56. The first loading path is seen to form a loop which starts from point A and follows the path of ABCDAEFGA. The mechanical energy lost in the first loop is equal to the area defined by the loading path. The second loading path follows the path of ADHDAGIGA; no mechanical energy is lost in this path because the energy is absorbed by or transformed to adjacent supports. Between points D and G in the second loading path, the supports at beam station 20 exhibit no resistance and hence a hole is produced around the pile at this station. The third loading path is similar to the second loading path except that in the third the maximum positive deflection increases from point H to point J, and the maximum negative deflection decrease from point I to point K. The succeeding paths are similar but have progressively smaller maximum positive and maximum negative deflections until finally no more energy is absorbed by or transformed to adjacent supports and the entire pile exists in a state of free vibration since no damping factor has been included. Figure 57 shows the plot of deflection versus time for beam station 20. Here the response is tapering down to the two limits at which the entire pile is in a state of free vibration.

A similar computed resistance-deflection curve can be constructed for any other support, with a similar result expected unless the support (for instance, at beam station 40) has not yet reached the yielding condition; then the computed resistance-deflection curve will be a straight line. Figure 58 shows the computer plot of deflection versus time for beam stations 10 and 30. Figure 59 shows the computer plot of deflection versus time for beam stations 40 and 50. Figure 60 shows the computer plot of deflection along the beam axis at time stations 10 and 20. Figure 61 shows the computer plot of deflection along the beam axis at time stations 50 and 150. Figure 62 shows the computer plot of moment along the beam axis at time stations 30 and 100.

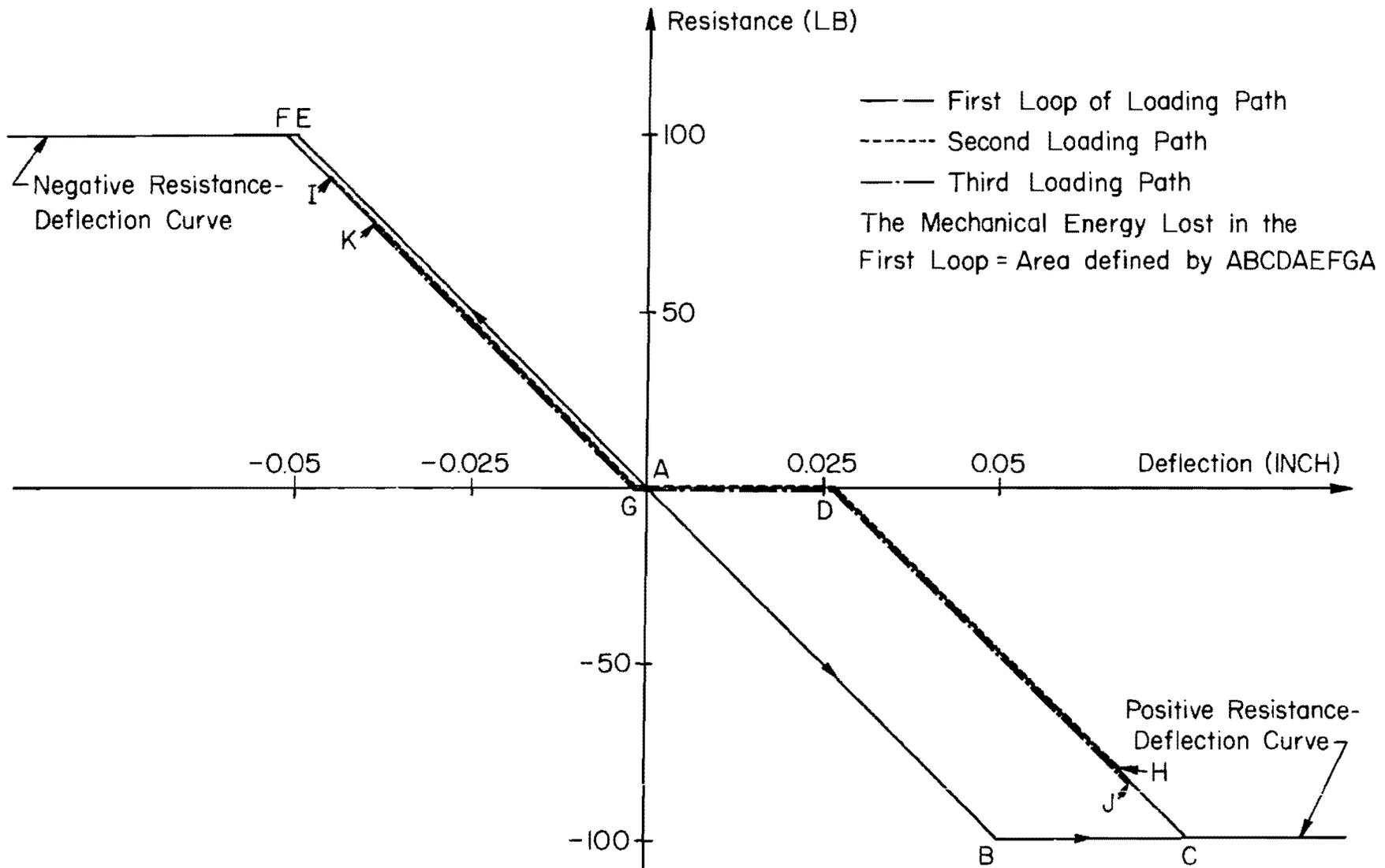


Fig 56. Computed resistance-deflection curves of the one-way supports at beam station 20 of Example Problem 7.

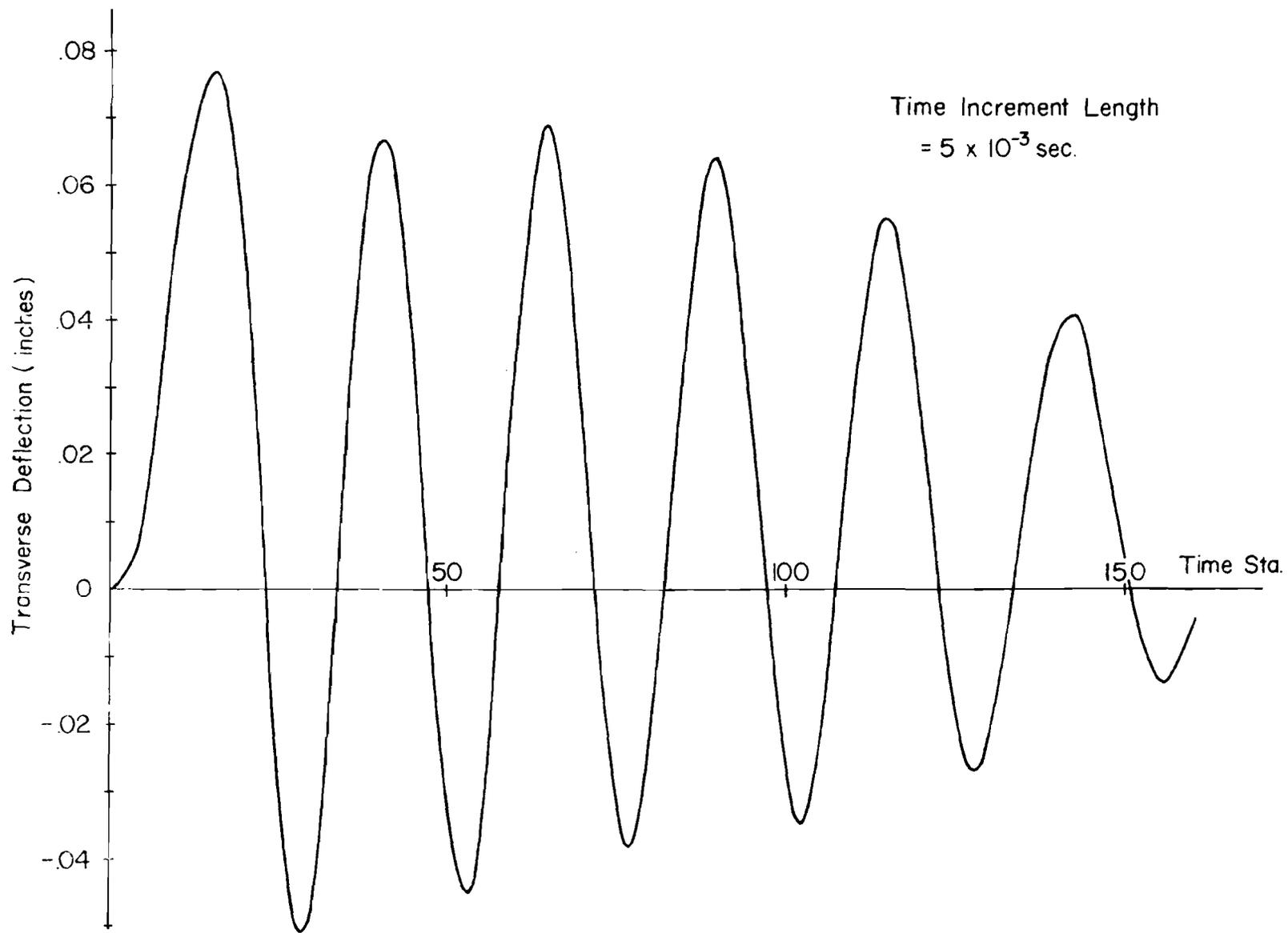
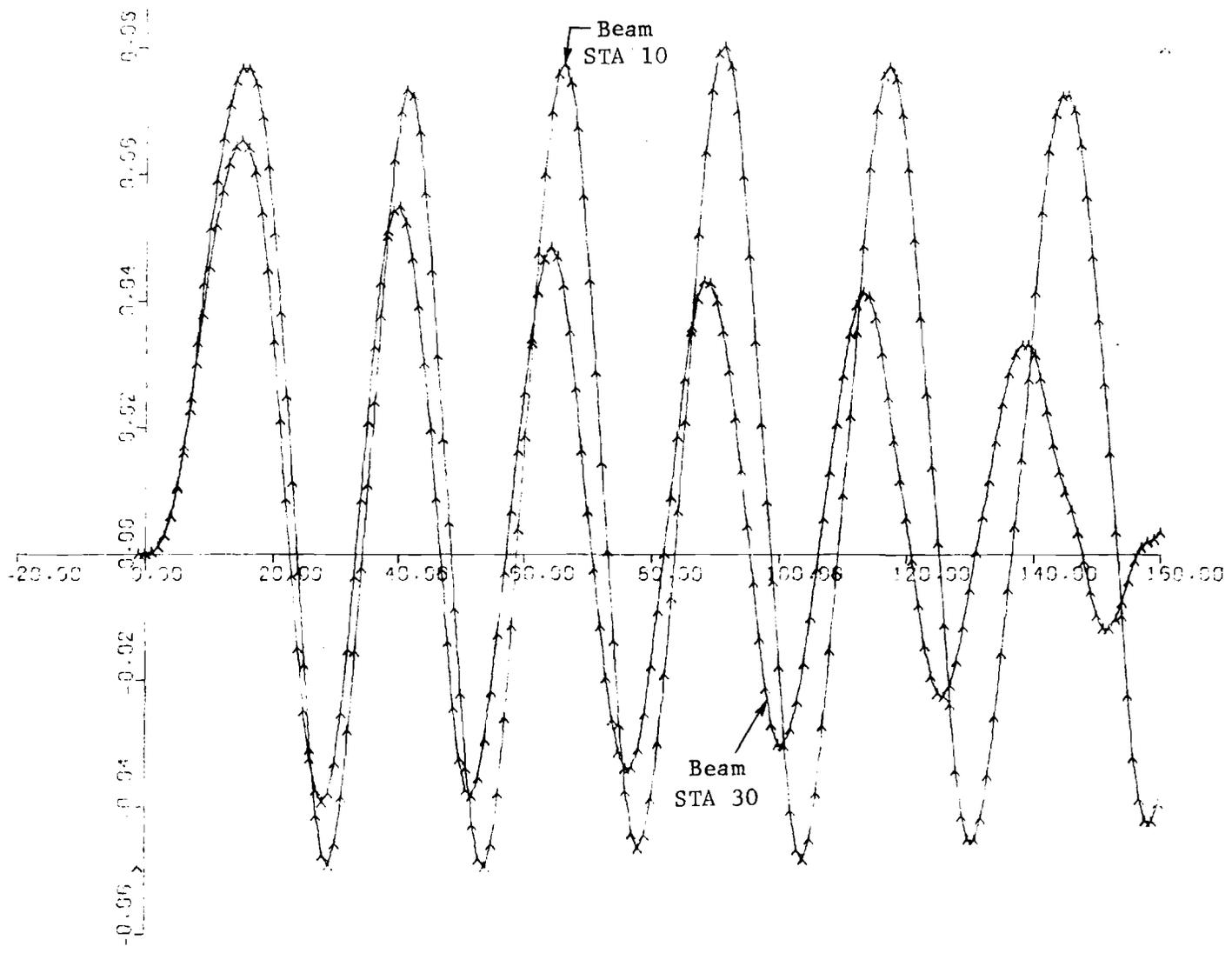
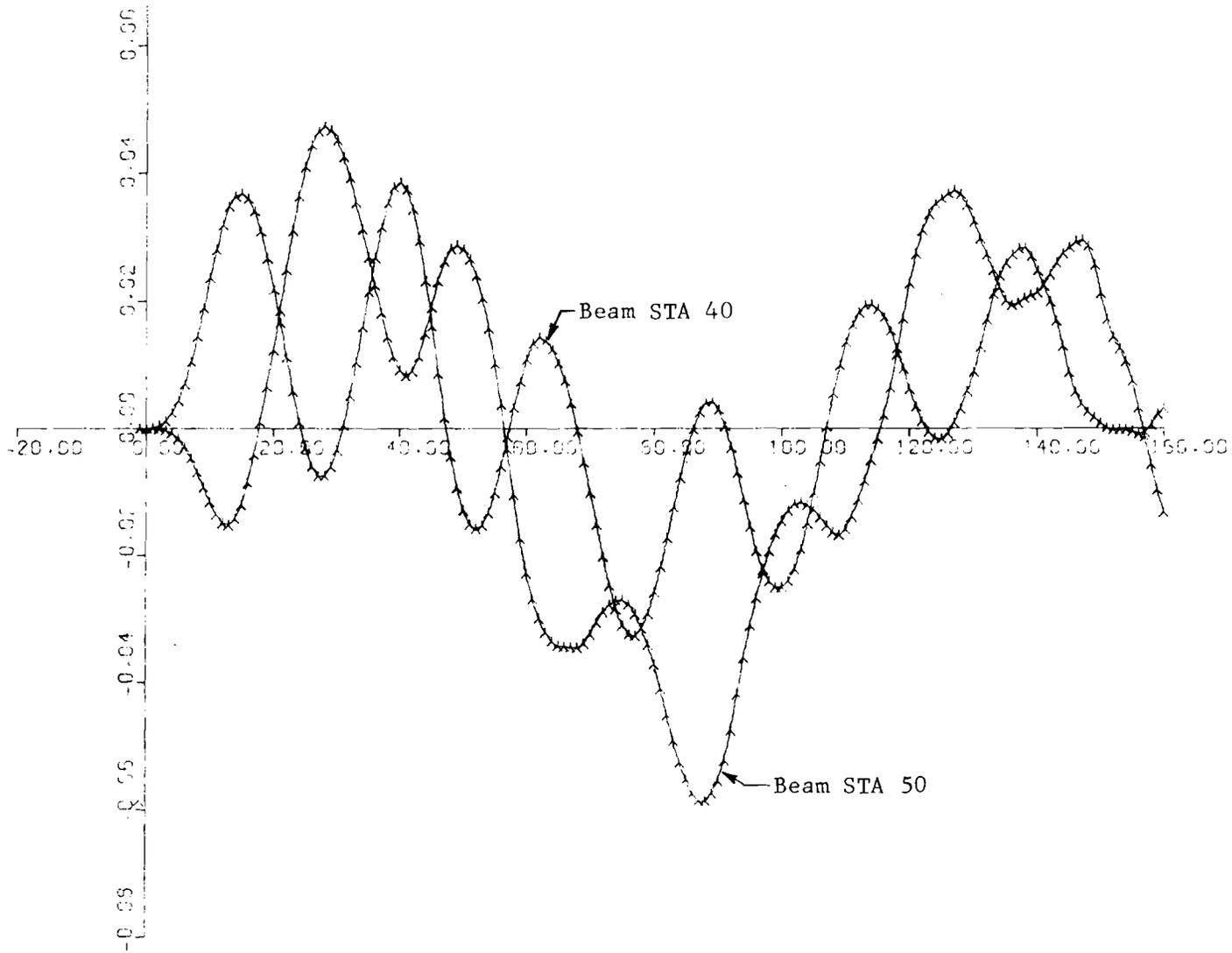


Fig 57. Example Problem 7. Deflection vs time for beam station 20.



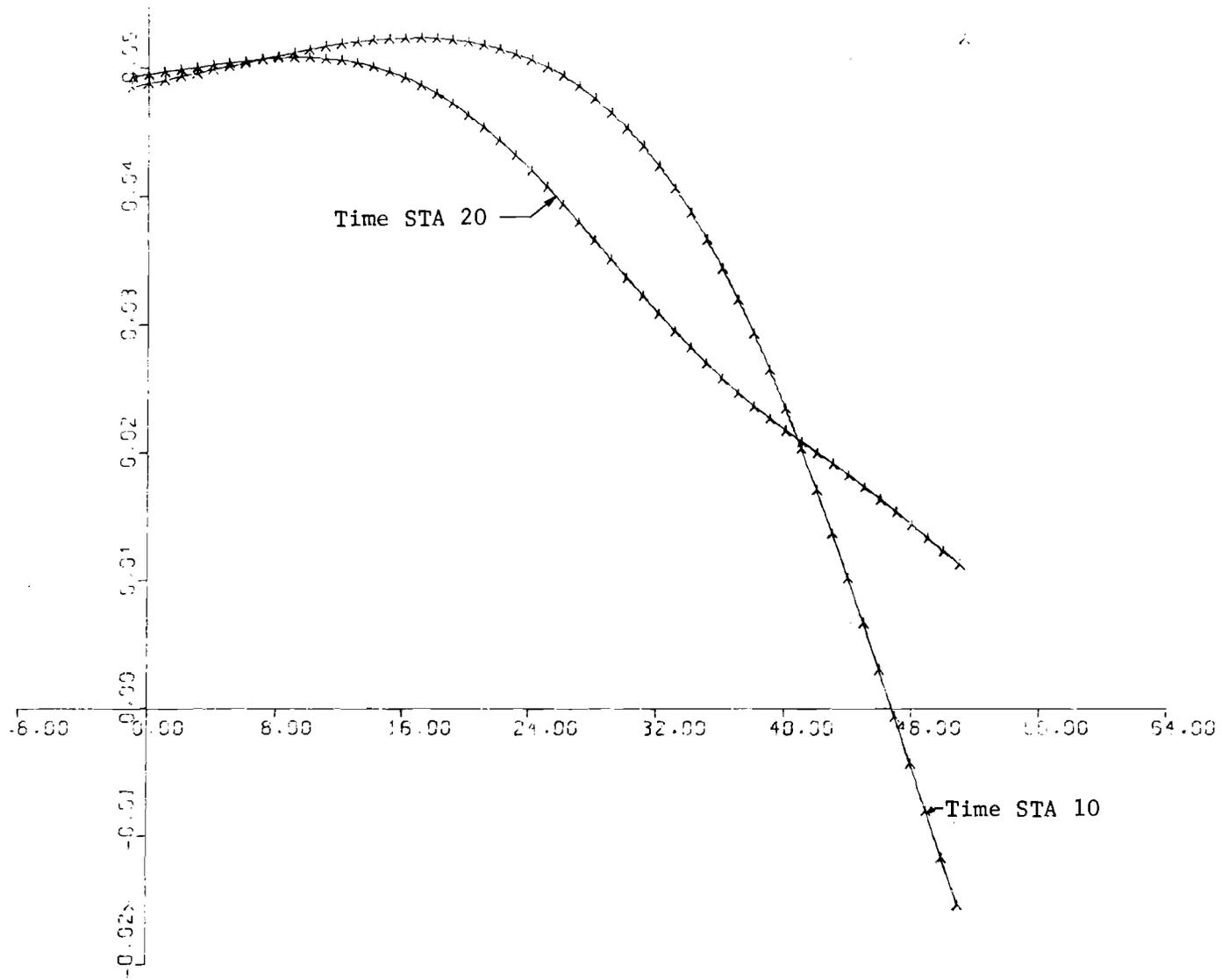
7 DVT BEAM STA = 10 30

Fig 58. Example Problem 7. Deflection vs time for beam stations 10 and 30.



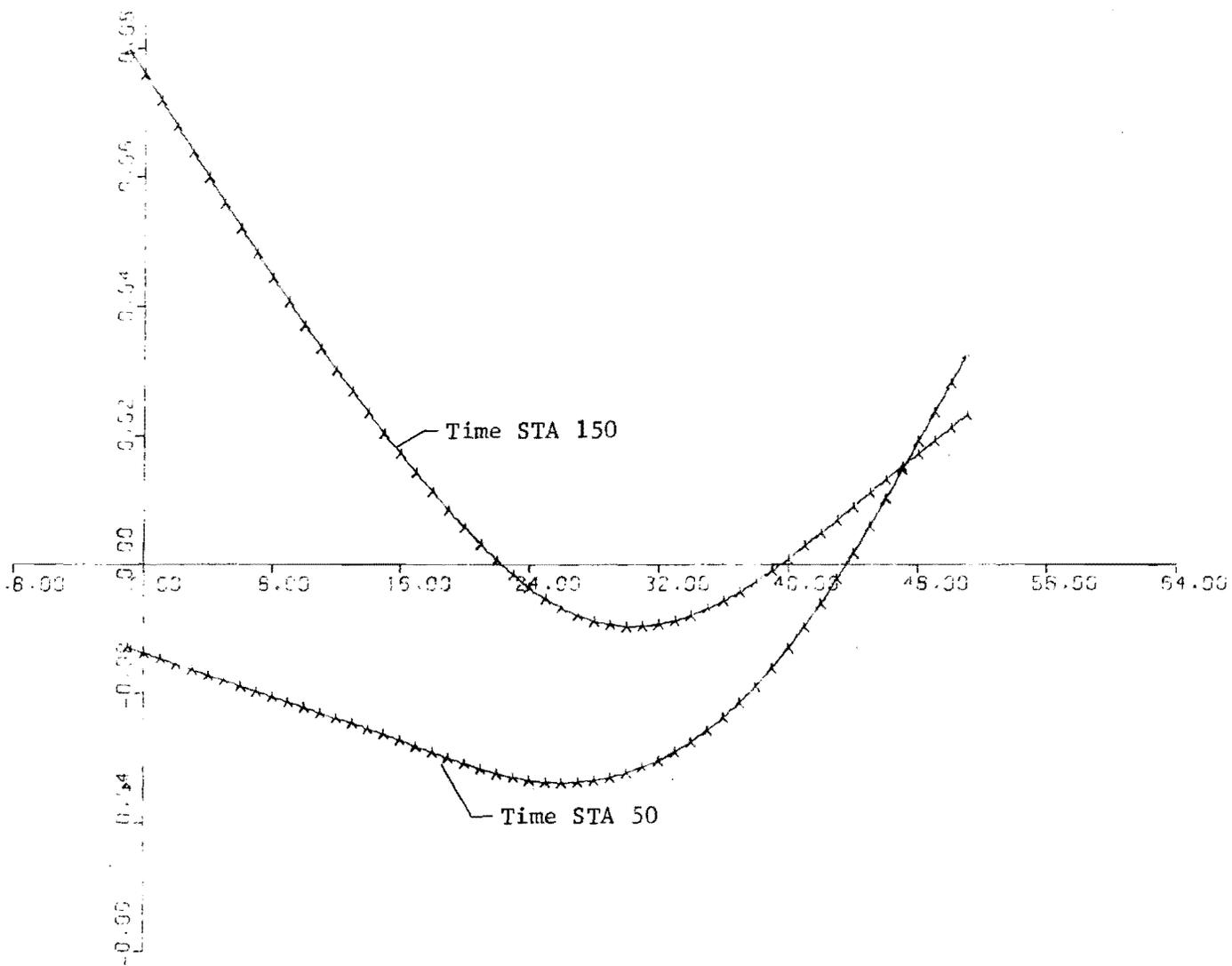
7 DVT BEAM STA = 40 50

Fig 59. Example Problem 7. Deflection vs time for beam stations 40 and 50.



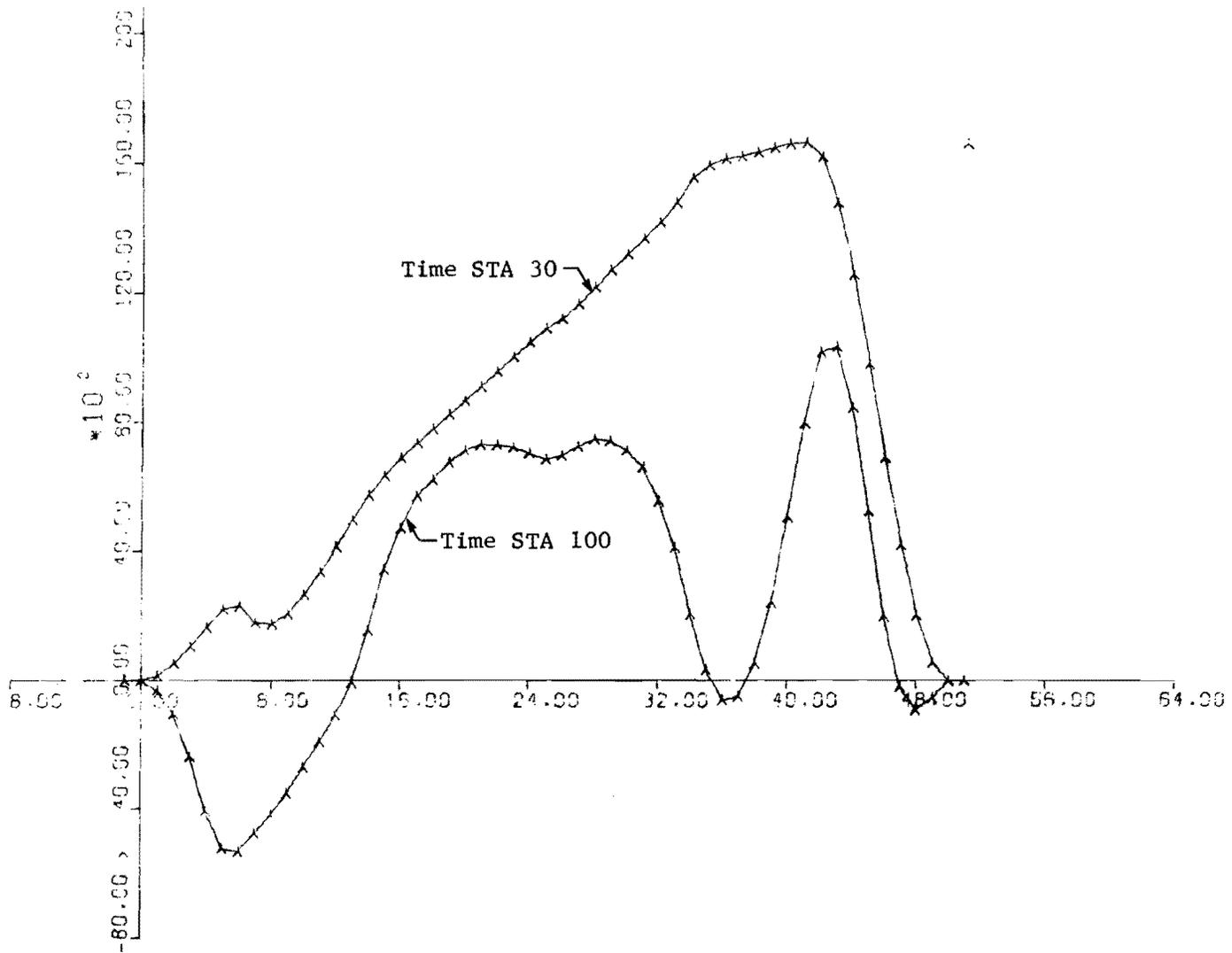
7 DVB TIME STA = 10 20

Fig 60. Example Problem 7. Deflection along the beam axis for time stations 10 and 20.



7 DVB TIME STA = 50 150

Fig 61. Example Problem 7. Deflection along the beam axis for time stations 50 and 150.



7 MVB TIME STA = 30 100

Fig 62. Example Problem 7. Moment along the beam axis for time stations 30 and 100.

## CHAPTER 6. SUMMARY AND CONCLUSIONS

In this study, the dynamic solution of the discrete-element beam-columns of Ref 10 (DBC1) and the static solution of the discrete-element beam-columns of Ref 5 (BMCOL 43) have been modified and extended to cover nonlinear supports and loads with time-dependent axial thrusts; the program has the added capability of producing plots of the deflections or moments along either beam or time axis of any requested monitor stations.

The path-dependent history of loading of the nonlinearly-inelastic resistance-deflection curve of the support is considered in the study. Two multi-element models, which are used to simulate the nonlinear characteristics of symmetric and one-way resistance-deflection curves, have been introduced. An internal damping factor, which is related to the first time derivative of the curvature of the beam, has been considered in addition to the external viscous damping factor which is normally encountered in a dynamic problem. A computer program which has been presented, DBC5, allows the user to obtain a variety of results and plots. All of the uses and capabilities of DBC1 and BMCOL 43, except construction of the envelopes of maximums provided in BMCOL 43, have been incorporated in the DBC5 program.

This method is based on an implicit difference formula of the Crank-Nicolson type. Problems in which only linear spring supports are specified are not subject to the instability of dynamic solutions. Problems in which nonlinearly-inelastic supports are specified, however, require cautious selection of a time increment length, to pick one reasonably but not extremely small (for instance, less than 1/10 of the fundamental period), since the basic assumption of the Crank-Nicolson implicit formula is that the time dependent forcing function and the time variant nonlinear springs are smoothly varied with time. For problems with nonlinear supports, the iteration process compares the successively computed deflections until a specified tolerance is satisfied. The option of switching from the spring-load iteration technique (tangent modulus method) to the load iteration technique for problems with nonlinear

supports is useful when the support yields or disconnects from the beam-column since the spring-load iteration method fails in these cases.

This report is intended to encourage use of DBC5 as a tool in computer aided design. Highway structural and foundation designers can use it to solve many problems, static or dynamic, which they encounter and which presently are solved by rough approximations or by tedious hand calculations. A very important use would be for a study of the dynamic response of actual truck loaded beam-columns, as illustrated in Example Problem 3 described in Chapter 5. A highly sophisticated computer program (Ref 1) is available to compute the time variant dynamic forces of the tires of a truck moving at a uniform speed on highway pavements. With limited interpretation, the computed dynamic tire forces could be input in the table provided for the time dependent lateral loads in Program DBC5.

Further application to design problems that are difficult to solve by conventional means is possible. The possible variations include analyses of a bridge structure as foundation supports settle, railroad loadings on continuous spans supported on soil foundations, the effect of axial loads induced from tractive forces or temperature changes, approach slabs connected integrally with the bridge structure and supported on soil foundations, offshore piles loaded with wave forces and partially embedded in the soil, and transverse response to earthquake-induced forces.

Future extensions to the model and the program might include, for highway pavement analysis, the coupling of a vehicle model and pavement roughness characteristics to the beam-column model for generation of dynamic loads (similar to the model described in Ref 1) and the inelastic analysis of beam properties.

The nonlinear solution capabilities should be extended to the beam bending stiffness variable. Nonlinear moment-curvature relations could be incorporated in the iterative procedure, thereby permitting analysis of the beam material for stresses in the nonlinear range. The hysteresis effect of the nonlinear moment-curvature relations should also be considered.

Studies of the nonlinear closure procedure should be continued. Methods for accelerating closure in the load iteration technique should be developed. In Program DBC5, the iteration process is performed at each time step. It is possible, however, to iterate only at the critical time stations; at the rest of the time stations, the support stiffness can be assumed to be unchanged

since a small time increment length must be used in these problems. The existing discrete-element model requires the user to know and specify the distribution of axial thrust throughout the beam. A valuable extension of this work would be modification of the model to include axial deformations and development of the force-deformation equations for axial thrust.

Finally, experimental data are required for further evaluation of the method, especially in predicting the path-dependent behavior of the supports. Research of this nature will furnish additional data on the nonlinear behavior of the supports, which could be applied to modification of the existing multi-element models for nonlinear supports so that buckling, fracture, softening, and relaxation (creep) of the support could also be considered in the program.

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## APPENDIX A

DERIVATION OF THE DYNAMIC IMPLICIT OPERATOR BASED ON THE  
CRANK-NICOLSON IMPLICIT FORMULA

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APPENDIX A. DERIVATION OF THE DYNAMIC IMPLICIT OPERATOR BASED ON THE CRANK-NICOLSON IMPLICIT FORMULA

The dynamic model used in Program DBC5 is discussed in Chapter 2 and the free-body diagram of the model is shown in Fig 3.

Since the rigid bars connected between the deformable joints are only responsible for transferring the bending moments, the shears, and the axial thrusts from one joint to another, the equation of motion of the beam-column can be obtained by summing up all the forces, internal or external, at any deformable joint where the beam properties are lumped. For the convenience of the reader, the free-body diagram of a portion of the dynamic beam-column model is shown in Fig A1.

If we take equilibriums for the moments in Bar j and Bar j+1 and for the vertical forces at joint J, then

for Bar j

$$M_{j-1} - M_j + (D^i)_{j-1} \frac{d}{dt} (\varphi_{j-1,k}) - (D^i)_j \frac{d}{dt} (\varphi_{j,k}) + V_j h + (T_j + T_{j,k}^T)(-w_{j-1,k} + w_{j,k}) = 0 \quad (A.1)$$

for Bar j+1

$$M_j - M_{j+1} + (D^i)_j \frac{d}{dt} (\varphi_{j,k}) - (D^i)_{j+1} \frac{d}{dt} (\varphi_{j+1,k}) + V_{j+1} h + (T_{j+1} + T_{j+1,k}^T)(-w_{j,k} + w_{j+1,k}) = 0 \quad (A.2)$$

for Joint j

$$V_j - V_{j+1} + Q_{j,k}^T + Q_j + \frac{-R_{j-1}\theta_{j-1} + R_{j+1}\theta_{j+1}}{2h}$$

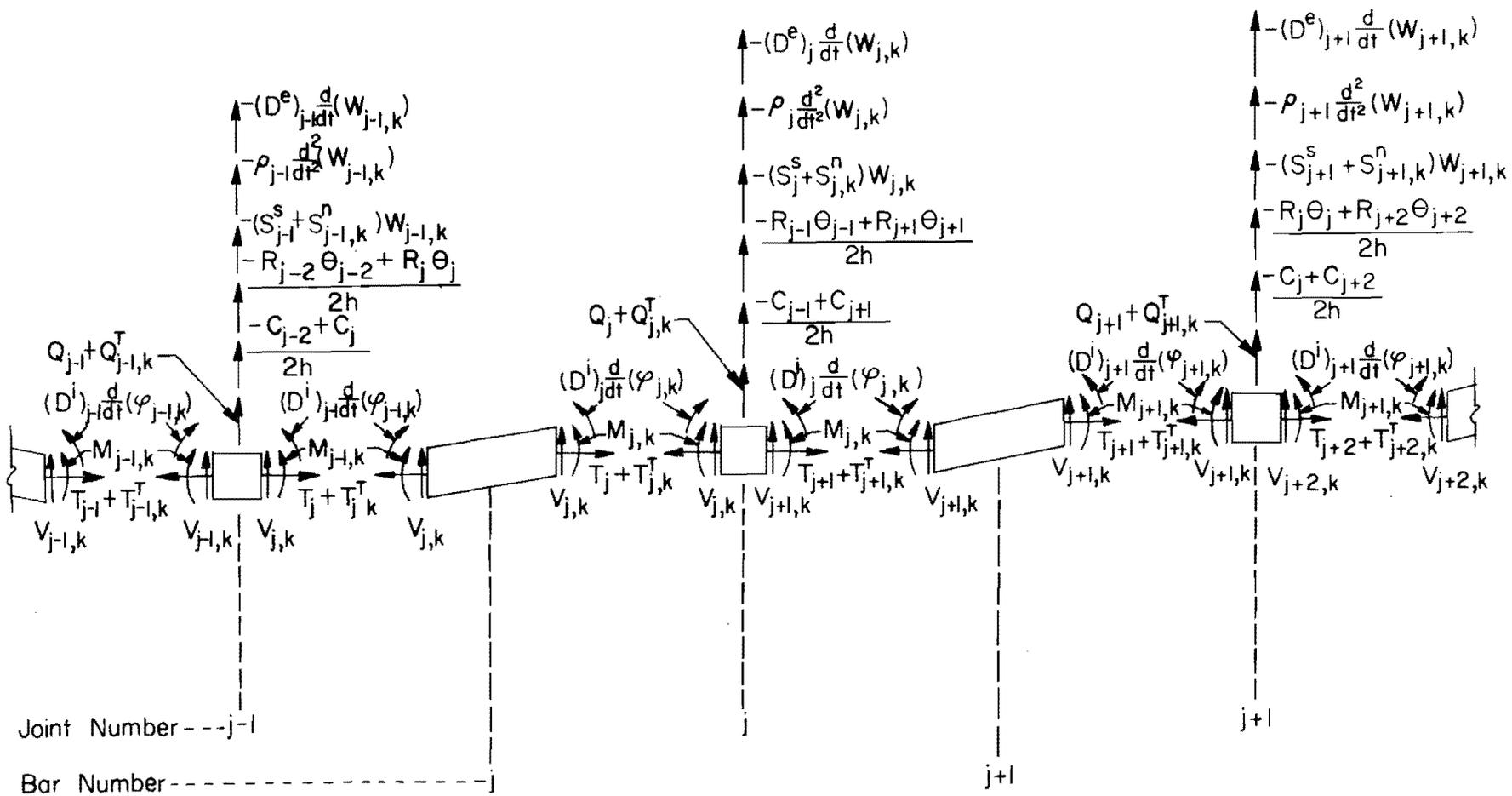


Fig A1. Free-body diagram of a portion of the dynamic beam-column model.

$$\begin{aligned}
& + \frac{-C_{j-1} + C_{j+1}}{2h} - S_{j,k}^N w_{j,k} - S_j^S w_{j,k} - \rho_j \frac{d^2}{dt^2} (w_{j,k}) \\
& - (D^e)_j \frac{d}{dt} (w_{j,k}) = 0 \tag{A.3}
\end{aligned}$$

Solving for the shears  $V_j$  and  $V_{j+1}$  in Eqs A.1 and A.2 and substituting into Eq A.3 and multiplying by  $h$  yields

$$\begin{aligned}
& - M_{j-1} + 2M_j - M_{j+1} - (D^i)_{j-1} \frac{d}{dt} (\varphi_{j-1,k}) + 2(D^i)_j \frac{d}{dt} (\varphi_{j,k}) \\
& - (D^i)_{j+1} \frac{d}{dt} (\varphi_{j+1,k}) - (T_j + T_{j,k}^T)(-w_{j-1,k} + w_{j,k}) \\
& + (T_{j+1} + T_{j+1,k}^T)(-w_{j,k} + w_{j+1,k}) + hQ_{j,k}^T + hQ_j \\
& + \frac{(-R_{j-1}\theta_{j-1} + R_{j+1}\theta_{j+1})}{2} + \frac{(-C_{j-1} + C_{j+1})}{2} - hS_j^S w_{j,k} \\
& - hS_{j,k}^N w_{j,k} - h\rho_j \frac{d^2}{dt^2} (w_{j,k}) - h(D^e)_j \frac{d}{dt} (w_{j,k}) = 0 \tag{A.4}
\end{aligned}$$

Based on the Crank-Nicolson implicit formula, we can assume that

$$\begin{aligned}
M_{j-1} = & F_{j-1} \frac{1}{2h^2} \left[ w_{j-2,k-1} - 2w_{j-1,k-1} + w_{j,k-1} + w_{j-2,k+1} \right. \\
& \left. - 2w_{j-1,k+1} + w_{j,k+1} \right],
\end{aligned}$$

$$\begin{aligned}
M_j = & F_j \frac{1}{2h^2} \left[ w_{j-1,k-1} - 2w_{j,k-1} + w_{j+1,k-1} + w_{j-1,k+1} - 2w_{j,k+1} \right. \\
& \left. + w_{j+1,k+1} \right],
\end{aligned}$$

$$M_{j+1} = F_{j+1} \frac{1}{2h^2} \left[ w_{j,k-1} - 2w_{j+1,k-1} + w_{j+2,k-1} + w_{j,k+1} \right. \\ \left. - 2w_{j+1,k+1} + w_{j+2,k+1} \right],$$

$$\frac{d}{dt}(\varphi_{j-1,k}) = \frac{1}{h^2 2h_t} \left[ -w_{j-2,k-1} + 2w_{j-1,k-1} - w_{j,k-1} + w_{j-2,k+1} \right. \\ \left. - 2w_{j-1,k+1} + w_{j,k+1} \right],$$

$$\frac{d}{dt}(\varphi_{j,k}) = \frac{1}{h^2 2h_t} \left[ -w_{j-1,k-1} + 2w_{j,k-1} - w_{j+1,k-1} + w_{j-1,k+1} \right. \\ \left. - 2w_{j,k+1} + w_{j+1,k+1} \right],$$

$$\frac{d}{dt}(\varphi_{j+1,k}) = \frac{1}{h^2 2h_t} \left[ -w_{j,k-1} + 2w_{j+1,k-1} - w_{j+2,k-1} + w_{j,k+1} \right. \\ \left. - 2w_{j+1,k+1} + w_{j+2,k+1} \right],$$

$$\theta_{j-1} = \frac{1}{4h} \left[ -w_{j-2,k-1} - w_{j-2,k+1} + w_{j,k-1} + w_{j,k+1} \right],$$

$$\theta_{j+1} = \frac{1}{4h} \left[ -w_{j,k-1} - w_{j,k+1} + w_{j+2,k-1} + w_{j+2,k+1} \right],$$

$$S_{j,k}^N = \frac{1}{2} \left[ S_{j,k-1}^N + S_{j,k+1}^N \right],$$

$$\frac{d^2}{dt^2} (w_{j,k}) = \frac{1}{h_t^2} \left[ w_{j,k-1} - 2w_{j,k} + w_{j,k+1} \right],$$

$$\frac{d}{dt} (w_{j,k}) = \frac{1}{2h_t} [-w_{j,k-1} + w_{j,k+1}],$$

$$w_{j-1,k} = \frac{1}{2} [w_{j-1,k-1} + w_{j-1,k+1}],$$

$$w_{j,k} = \frac{1}{2} [w_{j,k-1} + w_{j,k+1}],$$

$$w_{j+1,k} = \frac{1}{2} [w_{j+1,k-1} + w_{j+1,k+1}],$$

$$Q_{j,k}^T = \frac{1}{2} [Q_{j,k-1}^T + Q_{j,k+1}^T],$$

$$T_{j,k}^T = \frac{1}{2} [T_{j,k-1}^T + T_{j,k+1}^T],$$

and

$$T_{j+1,k}^T = \frac{1}{2} [T_{j+1,k-1}^T + T_{j+1,k+1}^T] \quad (\text{A.5})$$

Substituting Eq A.5 into Eq A.4 and rearranging the order, we obtain

$$\begin{aligned} & \left[ F_{j-1} + \frac{(D^i)_{j-1}}{h_t} - 0.25hR_{j-1} \right] w_{j-2,k+1} + \left\{ -2(F_{j-1} + F_j) \right. \\ & \quad \left. - \frac{2}{h_t} \left[ (D^i)_{j-1} + (D^i)_j \right] - h^2 \left[ T_j + 0.5(T_{j,k-1}^T + T_{j,k+1}^T) \right] \right\} \\ & w_{j-1,k+1} + \left\{ (F_{j-1} + 4F_j + F_{j+1}) + \frac{1}{h_t} \left[ (D^i)_{j-1} + 4(D^i)_j \right] \right\} \end{aligned}$$

$$\begin{aligned}
& + (D^i)_{j+1}] + h^2 [T_j + T_{j+1} + 0.5(T_{j,k-1}^T + T_{j,k+1}^T + T_{j+1,k-1}^T \\
& + T_{j+1,k+1}^T)] + 0.25h(R_{j-1} + R_{j+1}) + \frac{h^3}{2} [(S_j^S + S_{j,k-1}^N) \\
& + (S_j^S + S_{j,k+1}^N)] + \frac{2h^3 \rho_j}{h_t^2} + \frac{h^3 (D^e)_j}{h_t} \} w_{j,k+1} + \{ -2(F_j + F_{j+1}) \\
& - \frac{2}{h_t} [(D^i)_j + (D^i)_{j+1}] - h^2 [T_{j+1} + 0.5(T_{j+1,k-1}^T + T_{j+1,k+1}^T)] \} \\
& w_{j+1,k+1} + [F_{j+1} + \frac{(D^i)_{j+1}}{h_t} - 0.25hR_{j+1}] w_{j+2,k+1} \\
& = [ \frac{4h^3 \rho_j}{h_t^2} ] w_{j,k} + [-F_{j-1} + \frac{(D^i)_{j-1}}{h_t} + 0.25hR_{j-1}] w_{j-2,k-1} \\
& + \{ 2(F_{j-1} + F_j) - \frac{2}{h_t} [(D^i)_{j-1} + (D^i)_j] + h^2 [T_j + 0.5(T_{j,k-1}^T \\
& + T_{j,k+1}^T)] \} w_{j-1,k-1} + \{ - (F_{j-1} + 4F_j + F_{j+1}) \\
& + \frac{1}{h_t} [(D^i)_{j-1} + 4(D^i)_j + (D^i)_{j+1}] - h^2 [T_j + T_{j+1} + 0.5(T_{j,k-1}^T \\
& + T_{j,k+1}^T + T_{j+1,k-1}^T + T_{j+1,k+1}^T)] - 0.25h(R_{j-1} + R_{j+1}) \\
& - \frac{h^3}{2} [(S_j^S + S_{j,k-1}^N) + (S_j^S + S_{j,k+1}^N)] - \frac{2h^3 \rho_j}{h_t^2} + \frac{h^3 (D^3)_j}{h_t} \} w_{j,k-1} \\
& + \{ 2(F_j + F_{j+1}) - \frac{2}{h_t} [(D^i)_j + (D^i)_{j+1}] + h^2 [T_{j+1} + 0.5(T_{j+1,k-1}^T
\end{aligned}$$

$$\begin{aligned}
& + T_{j+1,k+1}^T \Big] \Big\} w_{j+1,k-1} + \left[ -F_{j+1} + \frac{(D^i)_{j+1}}{h_t} + 0.25hR_{j+1} \right] \\
& w_{j+2,k-1} + h^3 \left[ (Q_{j,k-1}^T + Q_{j,k+1}^T) + 2Q_j \right] + h^2 (-C_{j-1} \\
& + C_{j+1}) \tag{A.6}
\end{aligned}$$

Collecting the terms and rearranging the order in Eq A.6, we obtain a dynamic implicit operator:

$$\begin{aligned}
& a_{k+1} w_{j-2,k+1} + b_{k+1} w_{j-1,k+1} + c_{k+1} w_{j,k+1} + d_{k+1} w_{j+1,k+1} \\
& + e_{k+1} w_{j+2,k+1} \\
& = c_k w_{j,k} + a_{k-1} w_{j-2,k-1} + b_{k-1} w_{j-1,k-1} + c_{k-1} w_{j,k-1} \\
& + d_{k-1} w_{j+1,k-1} + e_{k-1} w_{j+2,k-1} + f_{j,k}
\end{aligned}$$

where

$$\begin{aligned}
a_{k+1} &= F_{j-1} + \frac{(D^i)_{j-1}}{h_t} - 0.25hR_{j-1}, \\
b_{k+1} &= -2(F_{j-1} + F_j) - \frac{2}{h_t} \left[ (D^i)_{j-1} + (D^i)_j \right] - h^2 \left[ T_j + \right. \\
& \left. 0.5(T_{j,k-1}^T + T_{j,k+1}^T) \right], \\
c_{k+1} &= (F_{j-1} + 4F_j + F_{j+1}) + \frac{1}{h_t} \left[ (D^i)_{j-1} + 4(D^i)_j + (D^i)_{j+1} \right] \\
& + h^2 \left[ T_j + T_{j+1} + 0.5(T_{j,k-1}^T + T_{j,k+1}^T + T_{j+1,k-1}^T + T_{j+1,k+1}^T) \right]
\end{aligned}$$

$$\begin{aligned}
& + 0.25h(R_{j-1} + R_{j+1}) + \frac{2h^3 \rho_j}{h_t^2} + \frac{h^3 (D^e)_j}{h_t} + \frac{h^3}{2} [(S_j^S + S_{j,k-1}^N) \\
& + (S_j^S + S_{j,k+1}^N)] ,
\end{aligned}$$

$$\begin{aligned}
d_{k+1} & = -2(F_j + F_{j+1}) - \frac{2}{h_t} [(D^i)_j + (D^i)_{j+1}] - h^2 [T_{j+1} \\
& + 0.5(T_{j+1,k-1}^T + T_{j+1,k+1}^T)] ,
\end{aligned}$$

$$e_{k+1} = F_{j+1} + \frac{(D^i)_{j+1}}{h_t} - 0.25hR_{j+1} ,$$

$$c_k = \frac{4h^3 \rho_j}{h_t^2} ,$$

$$a_{k-1} = -a_{k+1} + \frac{2(D^i)_{j-1}}{h_t} ,$$

$$b_{k-1} = -b_{k+1} - \frac{4}{h_t} [(D^i)_{j-1} + (D^i)_j] ,$$

$$c_{k-1} = -c_{k+1} + \frac{2}{h_t} [(D^i)_{j-1} + 4(D^i)_j + (D^i)_{j+1}] + \frac{2h^3 (D^e)_j}{h_t} ,$$

$$d_{k-1} = -d_{k+1} - \frac{4}{h_t} [(D^i)_j + (D^i)_{j+1}] ,$$

$$e_{k-1} = -e_{k+1} + \frac{2(D^i)_{j+1}}{h_t} ,$$

$$f_{j,k} = h^3 \left[ (Q_{j,k-1}^T + Q_j) + (Q_{j,k+1}^T + Q_j) \right] + h^2 (-c_{j-1} + c_{j+1})$$

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APPENDIX B

DERIVATION FOR RECURSIVE SOLUTION OF EQUATIONS  
(Extracted from Appendix 2 of Ref 4)

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APPENDIX B. DERIVATION FOR RECURSIVE SOLUTION OF EQUATIONS  
(Extracted from Appendix 2 of Ref 4)

Any of the fourth-order difference equations for beams or beam-columns can be written in the following form:

$$a_j w_{j-2} + b_j w_{j-1} + c_j w_j + d_j w_{j+1} + e_j w_{j+2} = f_j \quad (B.1)$$

Definitions of the coefficients  $a_j$  through  $f_j$  will vary according to the particular beam formulation considered. The general process of simultaneous solution of a complete system of such equations will be the same in any case. The matrix formed by writing coefficients  $a_j$  through  $e_j$  at all stations has non-zero terms along only the five main diagonals, and the most efficient method of solutions, therefore, is a direct process that amounts to simple Gaussian elimination. In a forward pass, two unknowns are eliminated from each equation, resulting in a triangularized coefficient matrix of only three diagonals. On the reverse pass, the solution is completed by back substitution. The necessary recursion equations for the process are derived below.

Assume, temporarily, that  $w_{j-2}$  and  $w_{j-1}$  can be eliminated so that  $w_j$  can be written in terms of deflections at two stations to the right. In general

$$w_j = A_j + B_j w_{j+1} + C_j w_{j+2} \quad (B.2)$$

Writing equations of this form for  $w_{j-2}$  and  $w_{j-1}$ ,

$$w_{j-2} = A_{j-2} + B_{j-2} w_{j-1} + C_{j-2} w_j \quad (B.3)$$

$$w_{j-1} = A_{j-1} + B_{j-1} w_j + C_{j-1} w_{j+1} \quad (B.4)$$

Substituting Eqs B.3 and B.4 into Eq B.1,

$$a_j [A_{j-2} + B_{j-2} (A_{j-1} + B_{j-1} w_j + C_{j-1} w_{j+1}) + C_{j-2} w_j]$$

$$\begin{aligned}
& + b_j(A_{j-1} + B_{j-1}w_j + C_{j-1}w_{j+1}) + c_jw_j + d_jw_{j+1} + e_jw_{j+2} \\
& = f_j
\end{aligned} \tag{B.5}$$

Multiplying and collecting terms,

$$\begin{aligned}
& (a_j B_{j-2} B_{j-1} + a_j C_{j-2} + b_j B_{j-1} + c_j)w_j \\
& + (a_j B_{j-2} C_{j-1} + b_j C_{j-1} + d_j)w_{j+1} \\
& + (e_j)w_{j+2} = f_j - (a_j A_{j-2} + a_j B_{j-2} A_{j-1} + b_j A_{j-1})
\end{aligned} \tag{B.6}$$

Equation B.6 can be rewritten in the form assumed in Eq B.2:

$$w_j = A_j + B_j w_{j+1} + C_j w_{j+2} \tag{B.7}$$

where

$$A_j = D_j(E_j A_{j-1} + a_j A_{j-2} - f_j) \tag{B.7a}$$

$$B_j = D_j(E_j C_{j-1} + d_j) \tag{B.7b}$$

$$C_j = D_j(e_j) \tag{B.7c}$$

and where

$$D_j = -1/(E_j B_{j-1} + a_j C_{j-2} + c_j) \tag{B.7d}$$

$$E_j = a_j B_{j-2} + b_j \tag{B.7e}$$

For beam and beam-column problems, the coefficients  $A_j$ ,  $B_j$ , and  $C_j$  can be thought of as expressing the physical continuity of the system. In

these coefficients all of the known input data are digested and stored. The coefficients at any one station depend not only on the load and stiffness data at that station but also on effects from all previous stations. These coefficients have therefore been termed "Continuity Coefficients."

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APPENDIX C

STABILITY ANALYSIS FOR THE DYNAMIC IMPLICIT OPERATOR  
WITH INTERNAL DAMPING COEFFICIENT

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APPENDIX C. STABILITY ANALYSIS FOR THE DYNAMIC IMPLICIT OPERATOR  
WITH INTERNAL DAMPING COEFFICIENT

In studying the stability criterion for the dynamic solutions of a beam with hinged ends and  $M$  increments, a solution to the equation of motion described in Chapter 2 (Eq 2.1 or 2.2) can be assumed to be

$$w_{j,k} = A \sin(j\beta_n) e^{k\phi} \quad (C.1)$$

in which  $A$  is a constant,  $j = 0, 1, 2, \dots, M$ , and  $k = 1, 2, 3, \dots, \infty$ .

If we consider only the flexibility of the beam  $F_j$ , the mass density of the beam  $\rho_j$ , and the internal damping coefficient of the beam  $(D^i)_j$ , then Eq 2.2, becomes

$$\begin{aligned} & \left[ F_{j-1} + \frac{(D^i)_{j-1}}{h_t} \right] w_{j-2,k+1} - 2 \left\{ (F_{j-1} + F_j) + \frac{1}{h_t} \left[ (D^i)_{j-1} \right. \right. \\ & \left. \left. + (D^i)_j \right] \right\} w_{j-1,k+1} + \left\{ (F_{j-1} + 4F_j + F_{j+1}) + \frac{1}{h_t} \left[ (D^i)_{j-1} \right. \right. \\ & \left. \left. + 4(D^i)_j + (D^i)_{j+1} \right] + \frac{2h^3 \rho_j}{ht^2} \right\} w_{j,k+1} - 2 \left\{ (F_j + F_{j+1}) \right. \\ & \left. + \frac{1}{h_t} \left[ (D^i)_j + (D^i)_{j+1} \right] \right\} w_{j+1,k+1} + \left[ F_{j+1} + \frac{(D^i)_{j+1}}{h_t} \right] w_{j+2,k+1} \end{aligned}$$

$$\begin{aligned}
&= \left( \frac{4h^3 \rho_j}{h_t^2} \right) w_{j,k} - \left[ F_{j-1} - \frac{(D^i)_{j-1}}{h_t} \right] w_{j-2,k-1} + 2 \left\{ (F_{j-1} + F_j) - \right. \\
&\frac{1}{h_t} \left[ (D^i)_{j-1} + (D^i)_j \right] \left. \right\} w_{j-1,k-1} - \left\{ (F_{j-1} + 4F_j + F_{j+1}) \right. \\
&- \frac{1}{h_t} \left[ (D^i)_{j-1} + 4(D^i)_j + (D^i)_{j+1} \right] + \frac{2h^3 \rho_j}{h^2} \left. \right\} w_{j,k-1} \\
&+ 2 \left\{ (F_j + F_{j+1}) - \frac{1}{h_t} \left[ (D^i)_j + (D^i)_{j+1} \right] \right\} w_{j+1,k-1} \\
&- \left[ F_{j+1} - \frac{(D^i)_{j+1}}{h_t} \right] w_{j+2,k-1} \tag{C.2}
\end{aligned}$$

Assuming that the beam has a uniform cross section, uniform elastic properties, uniform mass density, and a uniform internal damping coefficient, Eq C.2 can be simplified as

$$\begin{aligned}
&(F + D)w_{j-2,k+1} - 4(F + D)w_{j-1,k+1} + 6(F + D)w_{j,k+1} - 4(F + D) \\
&w_{j+1,k+1} + (F + D)w_{j+2,k+1} + (F - D)w_{j-2,k-1} - 4(F - D)w_{j-1,k-1} \\
&+ 6(F - D)w_{j,k-1} - 4(F - D)w_{j+1,k-1} + 6(F - D)w_{j,k-1} \\
&- 4(F - D)w_{j+1,k-1} + (F - D)w_{j+2,k-1} \\
&= - \frac{2\rho h^3}{h_t^2} (w_{j,k+1} + 2w_{j,k} + w_{j,k-1}) \tag{C.3}
\end{aligned}$$

where

F = all F terms (EI)

D = all  $(D^i)$  terms divided by  $h_t$

$\rho$  = all  $\rho$  terms

Substituting Eq C.1 into Eq C.3, we obtain

$$\begin{aligned}
 & (F + D) f(\beta_n, j) e^{(k+1)\phi} + (F - D) f(\beta_n, j) e^{(k-1)\phi} \\
 & = - \frac{2\rho h^3}{h_t^2} \left[ \sin \beta_n j e^{(k+1)\phi} + 2 \sin \beta_n j e^{k\phi} \right. \\
 & \quad \left. + \sin \beta_n j e^{(k-1)\phi} \right] \tag{C.4}
 \end{aligned}$$

where

$$\begin{aligned}
 f(\beta_n, j) & = \sin \beta_n (j-2) - 4 \sin \beta_n (j-1) + 6 \sin \beta_n j \\
 & \quad - 4 \sin \beta_n (j+1) + \sin \beta_n (j+2)
 \end{aligned}$$

Let

$$q^2 = \frac{2\rho h^3}{h_t^2}$$

Then, dividing Eq C.4 by  $e^{(k-1)\phi}$ , we obtain

$$\begin{aligned} & \left\{ (F + D) \left[ f(\beta_n, j) + q^2 \sin \beta_n j \right] \right\} e^{2\phi} + (2q^2 \sin \beta_n j) e^\phi \\ & + \left\{ (F - D) \left[ f(\beta_n, j) + q^2 \sin \beta_n j \right] \right\} = 0 \end{aligned} \quad (C.5)$$

Substituting the following trigonometric identities,

$$\sin \beta_n (j-2) = \sin \beta_n j \cos 2\beta_n - \cos \beta_n j \sin 2\beta_n$$

$$4 \sin \beta_n (j-1) = 4(\sin \beta_n j \cos \beta_n - \cos \beta_n j \sin \beta_n)$$

$$4 \sin \beta_n (j+1) = 4(\sin \beta_n j \cos \beta_n + \cos \beta_n j \sin \beta_n)$$

$$\sin \beta_n (j+2) = \sin \beta_n j \cos 2\beta_n + \cos \beta_n j \sin 2\beta_n$$

into Eq C.5 and simplifying the equation, we obtain

$$\begin{aligned} & \left[ 2(F + D)(\cos 2\beta_n - 4 \cos \beta_n + 3) + q^2 \right] e^{2\phi} + 2q^2 e^\phi \\ & + \left[ 2(F - D)(\cos 2\beta_n - 4 \cos \beta_n + 3) + q^2 \right] = 0 \end{aligned} \quad (C.6)$$

We can simplify Eq C.6 by using the trigonometric identities

$$\cos 2\beta_n = 2 \cos^2 \beta_n - 1, \quad (\cos \beta_n - 1)^2 = \cos^2 \beta_n - 2 \cos \beta_n + 1,$$

and dividing by the coefficient of  $e^{2\phi}$ ; thus

$$e^{2\phi} + \left[ \frac{2q^2}{4(F + D)(\cos \beta_n - 1)^2 + q^2} \right] e^\phi$$

$$+ \left[ \frac{4(F - D)(\cos \beta_n - 1)^2 + q^2}{4(F + D)(\cos \beta_n - 1)^2 + q^2} \right] = 0 \quad (C.7)$$

Let

$$B = \frac{q^2}{4(F + D)(\cos \beta_n - 1)^2 + q^2} \quad \text{and}$$

$$C = \frac{4(F - D)(\cos \beta_n - 1)^2 + q^2}{4(F + D)(\cos \beta_n - 1)^2 + q^2}$$

Hence Eq C.7 becomes

$$e^{2\phi} + 2Be^\phi + C = 0 \quad (C.8)$$

Equation C.8 is the quadratic equation which can be used to evaluate the stability criterion of the dynamic solutions of the beam. The criterion for the solution of  $w_{j,k}$  in Eq C.1, which is to be bounded as time approaches infinity, is that the roots of the quadratic,  $e^{\phi_1}$  and  $e^{\phi_2}$ , must satisfy the condition that

$$|e^{\phi_1}|, |e^{\phi_2}| \leq 1 \quad (C.9)$$

The two roots of Eq C.8 are

$$e^{\phi_1} = -B + \sqrt{B^2 - C}$$

and

$$e^{\phi/2} = -B - \sqrt{B^2 - C} \quad (\text{C.10})$$

The conditions of Eq C.9 are satisfied if the following inequality is

true

$$B^2 - C \leq 0 \quad (\text{C.11})$$

Substituting the equalities of B and C into Eq C.11 yields

$$\frac{q^2}{[4(F+D)(\cos \beta_n - 1)^2 + q^2]^2} \leq \frac{4(F-D)(\cos \beta_n - 1)^2 + q^2}{4(F+D)(\cos \beta_n - 1)^2 + q^2} \quad (\text{C.12})$$

The above inequality can be reduced to

$$16(F^2 - D^2)(\cos \beta_n - 1)^4 + 8F(\cos \beta_n - 1)^2 q^2 \geq 0 \quad (\text{C.13})$$

By omitting the positive term  $(\cos \beta_n - 1)^2$ , the criterion for the stability of the dynamic solution of a uniform beam with hinged ends and internal damping coefficient becomes

$$16(F^2 - D^2)(\cos \beta_n - 1)^2 + 8Fq^2 \geq 0 \quad (\text{C.14})$$

Since  $\frac{(D^i)_j}{h_t}$  is usually less than  $F_j$  for a reasonable time increment length  $h_t$  and the positive value of the term  $8Fq^2$  is always greater than the negative value of the term  $16(F^2 - D^2)(\cos \beta_n - 1)^2$  when the time increment length is extremely small, the above inequality is true.

APPENDIX D

GUIDE FOR DATA INPUT FOR DBC5

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IDENTIFICATION OF PROGRAM AND RUN (Two alphanumeric cards per run)

	80
	80

IDENTIFICATION OF PROBLEM (One card for each problem)

Prob. No.	Description of problem
1 5	11 80

TABLE 1. PROGRAM CONTROL DATA (One card for each problem)

Enter "1" to keep data from prior tables									Number of cards input for tables								
2	3	4	5	6	7	8	9	2	3	4	5	6	7	8	9		
1	5	10	15	20	25	30	35	40	45	50	55	60	65	70	75	80	

TABLE 2. CONSTANTS (One, two, or three cards; none if preceding Table 2 is held)

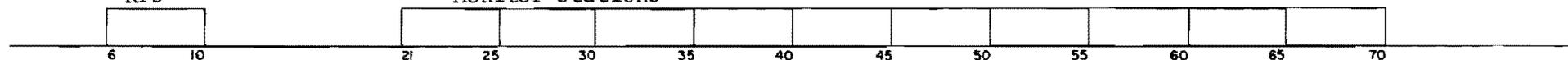
NUM of beam incre- ments M	Beam increment length H	NUM of time incre- ments MT	Time increment length HT	NUM of monitor Print- ing Switch- ing IPS*	STA (print- ing) MONS	Itera- tion switch ITSW**	NUM of monitor STA (itera- tion) MONI	Plot- ting switch MOP	Option for line or point plots LOP			
6	10	20	26	30	40	46	50	55	60	65	70	75

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\* If printing switch is -1, use the following card, otherwise ignore:

Complete print  
 out at  
 time sta  
 interval  
 KPS

Monitor stations



\*\* If iteration switch is not zero, use the following card, otherwise ignore:

Max num of iterations for any time step  
 MAXIT  
 Maximum allowable deflection  
 WMAX  
 Deflection closure tolerance  
 WTOL

Monitor stations

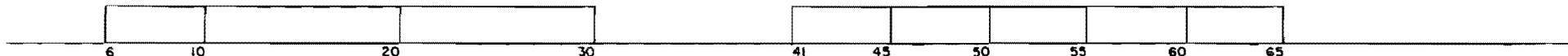
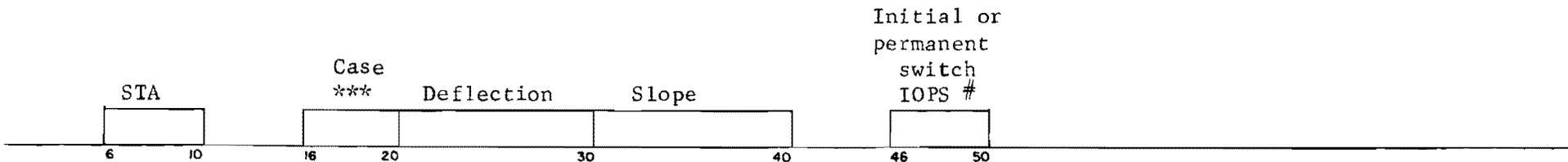


TABLE 3. SPECIFIED DEFLECTIONS AND SLOPES (The number of cards as shown in Table 1, none if preceding Table 3 is held)



\*\*\* If case = 1, specified deflection  
 = 2, specified slope  
 = 3, specified both

# If IOPS = 1, initial deflection or slope (or both) is specified.  
 = 2, permanent deflection or slope (or both) is specified.

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TABLE 4. STIFFNESS AND FIXED-LOAD DATA (The number of cards as shown in Table 1)

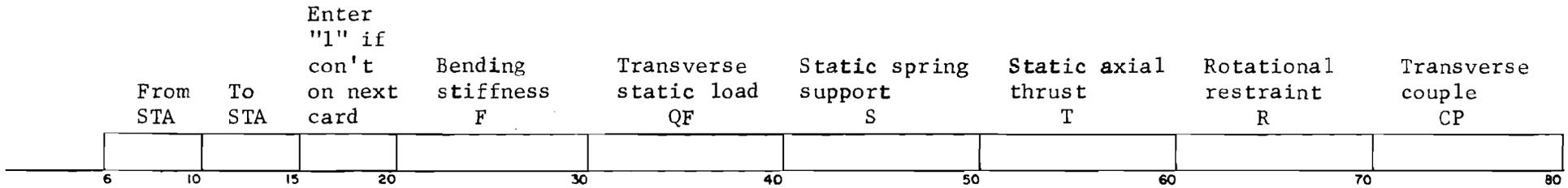


TABLE 5. MASS AND DAMPINGS (The number of cards as shown in Table 1)

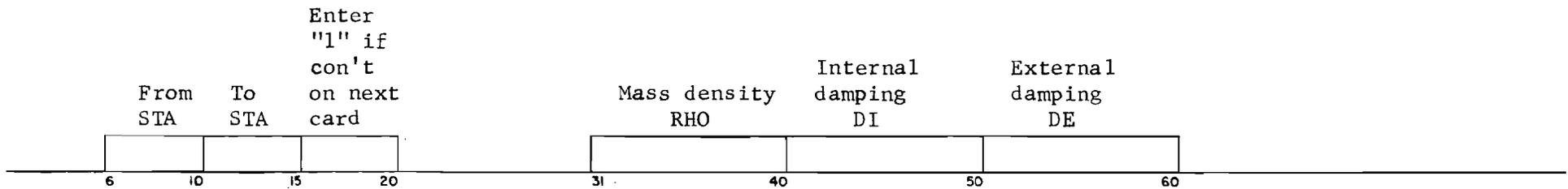
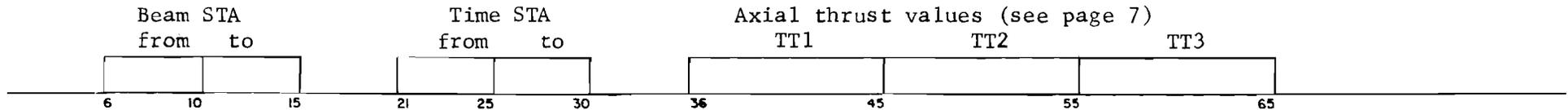


TABLE 6. TIME DEPENDENT AXIAL THRUST (The number of cards as shown in Table 1)



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TABLE 7. TIME DEPENDENT LATERAL LOADING (The number of cards as shown in Table 1)

Beam STA		Time STA		Lateral Loads (see page 9)		
From	To	From	To	QT1	QT2	QT3
6	15	21	30	36	55	65

TABLE 8. NONLINEAR SUPPORT CURVES (Three cards per curve, the number of cards as shown in Table 1)

From Beam STA	To Beam STA	Enter "1" if cont'd on next card	Resistance Multiplier QMP	Deflection Multiplier WMP	Num Points NPOC	Symmetry Option KSYM	Deflection Tolerance WNTOL	Resistance Tolerance QNTOL
6	15							

Resistance Values	Q
	31
	35
	40
	45
	50
	55
	60
	65
	70
	75
	80

Deflection Values	W
	31
	35
	40
	45
	50
	55
	60
	65
	70
	75
	80

TABLE 9. PLOTTING SWITCHES (One card for each plot, the number of cards as shown in Table 1)

Horizontal axis	Vertical axis	Beam or time STA	Multiple plot switch
7	17	26	36
10	20	30	40

 Enter either "TIME" or "BEAM"

 Enter either "DEFL" or "MOMT"

 Enter either a beam station number if TIME horizontal axis was specified or a time station number if BEAM horizontal axis was specified.

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# The consecutive plots, which are entered 1 for Multiple Plot Switch, will superimpose on common axes. The last plot of the group of plots which are superimposed on common axes must be entered 0 for Multiple Plot Switch.

For consecutive plots which are entered 0 for Multiple Plot Switch, each plot is plotted separately on different axes.

STOP CARD (One blank card of the end of each run)

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The data cards must be stacked in proper order for the program to run.  
 A consistent system of units must be used for all input data, for example, lbs and inches.  
 All words of 5-spaces or less are understood to be right justified integers, whole decimal numbers, or alphanumeric words. 34 or TIME  
 All words of 10-spaces are right justified floating-point decimal numbers. -4.231E+01

Table 1. Program-Control Data

For each of Tables 2 and 3, a choice must be made between holding all the data from the preceding problem or entering entirely new data. If the hold option for any of these Tables is set equal to 1, the number of cards input for that Table must be zero.  
 For Tables 4 through 9, the data is accumulated in storage by adding to previously stored data. The number of cards input is independent of the hold option, except that the cumulative total of cards cannot exceed 100 in Tables 4 through 7; 60 in Table 8; and 10 in Table 9. Card counts entered in Table 1 should be rechecked carefully after the coding of each problem is completed.

Table 2. Constants

Variables:	H	HT	WMAX	WTOL
Typical Input Units:	in	sec	in	in

The maximum number of increments into which the beam-column may be divided is 200. Typical units for the value of beam increment length are inches. The maximum number of increments into which the time axis may be divided is 1000. Typical units for the value of time increment length are seconds. If the printing switch (IPS) is set equal to -1, only results at monitor beam stations will be printed, except at every time station interval specified (KPS) where results at all beam stations will be printed. If the printing switch (IPS) is set equal to +1, the complete results of all beam stations will be printed at every time station and the second card therefore is not necessary. If the printing switch is equal to zero or left blank, no intermediate results will be printed. MONS is the number of specified monitor beam stations which are entered on the second card from column 21 to column 70, for printing the monitor stations results at every time station. The maximum number of monitor beam stations which may be used is 10.

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The second input card should be omitted when the printing switch (IPS) is set equal to zero, blank, or +1.

For nonlinear support curves, which are stated in Table 8, the switch (ITSW) is set equal to either a negative or positive integer, for example, -1 or +1. If ITSW is set equal to zero or left blank, no nonlinear support curves may be entered in Table 8 and the third input card must be omitted.

MONI is the number of monitor beam stations, which are entered on the third input card from column 41 to column 65, for printing monitor deflections during the iteration process. The maximum number of monitor beam stations which may be requested is 5.

MAXIT is the number of iterations to be allowed for this problem. The maximum number is 50.

WMAX is the maximum allowable deflection for this problem.

WTOL is the closure tolerance for the iteration process.

MOP is a switch for selecting the method of plotting the results of beam or time stations which are stated in Table 9.

If MOP is set equal to -1, microfilm plots are made.

If MOP is set equal to zero or left blank, printer plots are made.

If MOP is set equal to 1, 12-inch standard ball-point paper plots are made.

LOP is an optional switch for choosing point or line plots for the microfilm or paper plots.

If LOP = -j, point plots are made at every jth point.

If LOP = 0 or blank, line plots are made. If LOP = j, line plots with a point plot at every jth point are made.

If no cards are entered for Table 9, MOP and LOP may be left blank.

If MOP is set equal to zero or left blank, LOP may be left blank.

### Table 3. Specified Deflections and Slopes

The maximum number of beam stations at which deflections and slopes may be specified is 20.

A slope may not be specified closer than 3 increments from another specified slope.

A deflection may not be specified closer than 2 increments from a specified slope, except that both a deflection and a slope may be specified at the same beam station.

IOPS is a switch for specifying either an initial deflection (or slope) or a permanent deflection (or slope) at the beam station stated on the same card.

If IOPS is set equal to 1, initial deflection or slope (or both) are specified for the static solutions only.

If IOPS is set equal to 2, permanent deflection or slope (or both) are specified, both for the static solutions and the dynamic solutions.

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Table 4. Stiffness and Fixed-Load Data

Variables:	F	QF	S	T	R	CP
Typical Input Units:	$lb \times in^2$	lb	lb/in	lb	in $\times$ lb/rad	in $\times$ lb

Axial tension or compression values T must be stated at each beam station in the same manner as any other distributed data; there is no mechanism in the program to automatically distribute the internal effects of an externally applied axial force.

Data should not be entered in this table (nor held from the preceding problem) which would express effects at fictitious stations beyond the ends of the real beam-column.

The left end of the beam-column must be located at station 0.

For the interpolation and distribution process, there are four variations in the beam station numbering and referencing for continuation to succeeding cards. These variations are explained and illustrated on page 11. There are no restrictions on the order of cards in Tables 4 and 5, except that within a distribution sequence the beam stations must be in regular order.

Table 5. Mass and Damping Data

Variables:	RHO	DI	DE
Typical Input Units:	$lb \times sec^2/in$	$lb \times in^2 \times sec$	$lb \times sec/in$

See description in Table 4.

Table 6. Time Dependent Axial Thrust

Variables:	TT1	TT2	TT3
Typical Input Units:	lb	lb	lb

Time dependent axial thrust values "TT1" and "TT2" are the distributed values entered at beam stations "FROM" and "TO", respectively, of time station "FROM". The values "TT3" are the distributed values entered at beam stations "FROM" of time stations "TO".

All the values of time dependent axial thrusts at the beam stations within the two limits of "FROM" and "TO" of the time stations between the two limits of "FROM" and "TO" will be linearly

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interpolated by the program. There are no restrictions on the order of cards, except that beam stations which are entered as "FROM" and "TO" in the same card must be in ascending order, or in other words, the same beam station is not allowed to be entered as "FROM" and "TO" in the same card.

See page 12 for illustrated examples of interpolation and distribution.

Table 7. Time Dependent Lateral Loading

Variables:	QT1	QT2	QT3
Typical Input Units:	lb	lb	lb

All restrictions stated in Table 6 are also true in this Table, except that same beam station is allowed to be entered as "FROM" and "TO" in the same input card.

See page 13 for illustrated examples of interpolation and distribution.

Table 8. Nonlinear Support Curves

QMP is a constant which is multiplied by Q-VALUES to obtain the vertical resistance values of the corresponding points of the resistance-deflection curve.

WMP is a constant which is multiplied by W-VALUES to obtain the horizontal deflection values of the corresponding points of the resistance-deflection curve.

NPOC is the number of points input on the resistance-deflection curves.

KSYM is the symmetry option of the input resistance-deflection curve.

If KSYM is set equal to 1, then the total number of points on the curve will be twice the value of NPOC. In this case, the point of origin of the curve should not be specified.

If KSYM is set equal to 0 (or left blank) or -1, the point of origin of the curve must be specified. In this case, the deflections which are stated as the first point and the last point on the curve must be equal but opposite in sign.

If KSYM is set equal to 0, the support is a negative one-way support; that is the beam will lift off the support when it deflects upward.

If KSYM is set equal to -1, the support is a positive one-way support; that is the beam will lift off the support when it deflects downward.

There are no restrictions on the order of support curves, except that within any distribution sequence the beam stations must be in regular order. More than one curve may be placed at a beam station.

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The maximum number of points which may be input on any given curve is 10, although additional points up to a total of 20 may be created internally if the curve symmetry option is exercised.

WNTOL is the closure tolerance of deflections for matching up the unadjusted part of the resistance-deflection curve when the original curve needs to be adjusted.

QNTOL is the closure tolerance of resistances for matching up the unadjusted part of the resistance-deflection curve when the original curve needs to be adjusted.

For any particular resistance-deflection curve, the final deflections of the points in storage (WMP times W-values) must be increasing positively, while the final resistances (QMP times Q-values) must be in descending or equaling order. In other words, the resistance-deflection curves must be continuously concave as viewed from the horizontal axis, except that horizontal lines of zero stiffness can be input for representing the plastic regions of the curve. Softening of the support is not permitted; that is, no reversal of slope sign of the segment on the support curve is permitted.

For both one-way (negative and positive) supports, the input order of the points on the curve is the same, e.g., the final deflections of the points in storage must be increasing positively; internally, the program reverses the order of the points on the positive one-way support curve after the necessary information for tracing the loading paths has been retained.

If continuing on next curves, the deflections must be equal for the corresponding points of each curve in the sequence of continuation. In this case, WNTOL, NPOC, and KSYM also must be equal.

The resistance-deflection curve at every nonlinearly supported beam station within the two limits of "FROM" and "TO" will be linearly interpolated by the program.

The maximum number of beam stations which can be nonlinearly supported is 100.

#### Table 9. Plotting Switches

Four kinds of plots, deflection or moment along time axis, and deflection or moment along beam axis may be requested.

If the horizontal axis is TIME, a beam station must be entered from column 26 to column 30.

If the horizontal axis is BEAM, a time station must be entered from column 26 to column 30.

As many as five plots may be superimposed by using the multiple plot switch.

Superimposed plots must all be of the same kind.

There are no restrictions on the order of cards except that beam or time stations must be in regular order within a sequence of the same kind of plots.

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Individual-card Input

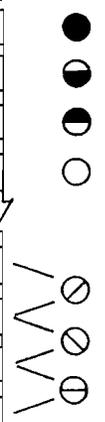
- Case a.1 Data concentrated at one sta .....
- Case a.2. Data uniformly distributed .....

FROM STA	TO STA	CONT'D TO NEXT CARD P	F	Q	etc...
7	7	0=NO		3.0	
5	15	0=NO	2.0		
15	20	0=NO	4.0	1.0	
10	20	0=NO		2.0	

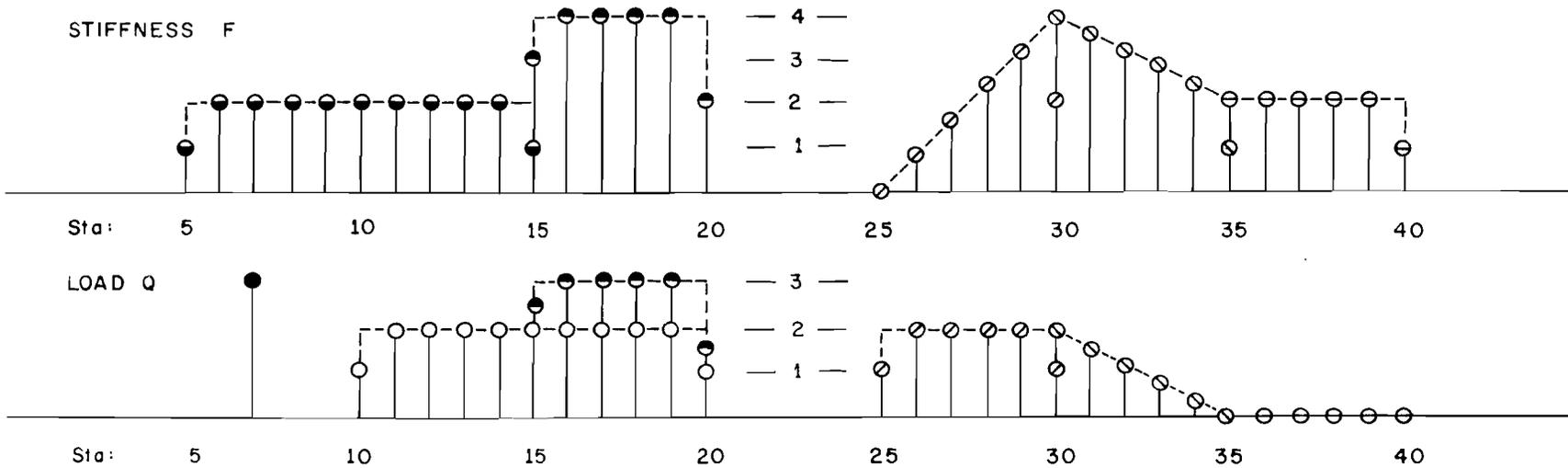
Multiple-card Sequence

- Case b. First-of-sequence .....
- Case c. Interior-of-sequence .....
- Case d. End-of-sequence .....

25		1=YES	0.0	2.0	
	30	1=YES	4.0	2.0	
	35	1=YES	2.0	0.0	
	40	0=NO	2.0		



Resulting Distribution of Data

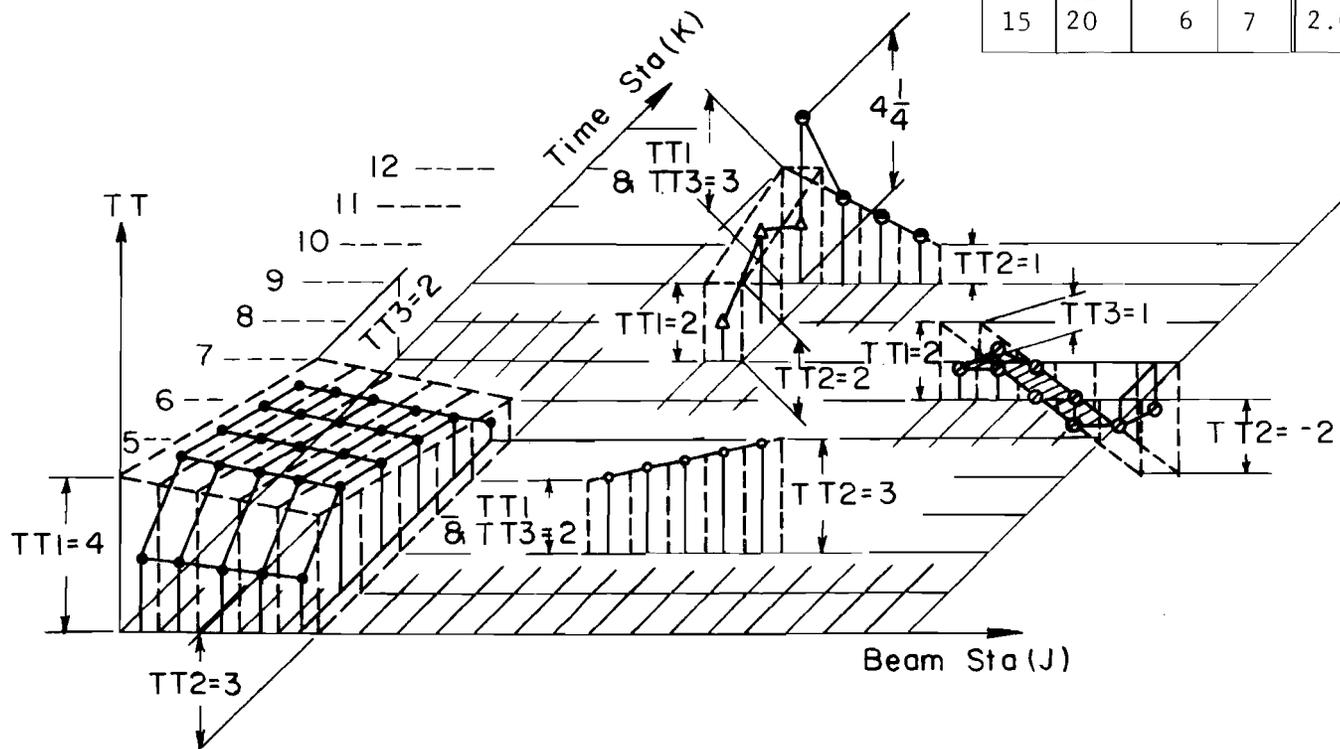


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TABLE 6. TIME DEPENDENT AXIAL THRUST (CONTINUED)

The variable  $TT(J,K)$  is input at any bar-number and time station by specifying the value of axial thrust distributed at beam station "FROM" and time station "FROM" in the columns of inputting  $TT1$ , the value of axial thrust distributed at beam station "TO" and time station "FROM" in the columns of inputting  $TT2$ , and the value of axial thrust distributed at beam station "FROM" and time station "TO" in the columns of inputting  $TT3$ .

Beam From	Sta To	Time From	Sta To	TT1	TT2	TT3	
0	5	0	5	4.0E+00	3.0E+00	2.0E+00	●
10	15	2	2	2.0E+00	3.0E+00	2.0E+00	○
8	9	7	9	2.0E+00	2.0E+00	3.0E+00	△
8	12	9	9	3.0E+00	1.0E+00	3.0E+00	●
15	20	6	7	2.0E+00	-2.0E+00	1.0E+00	●

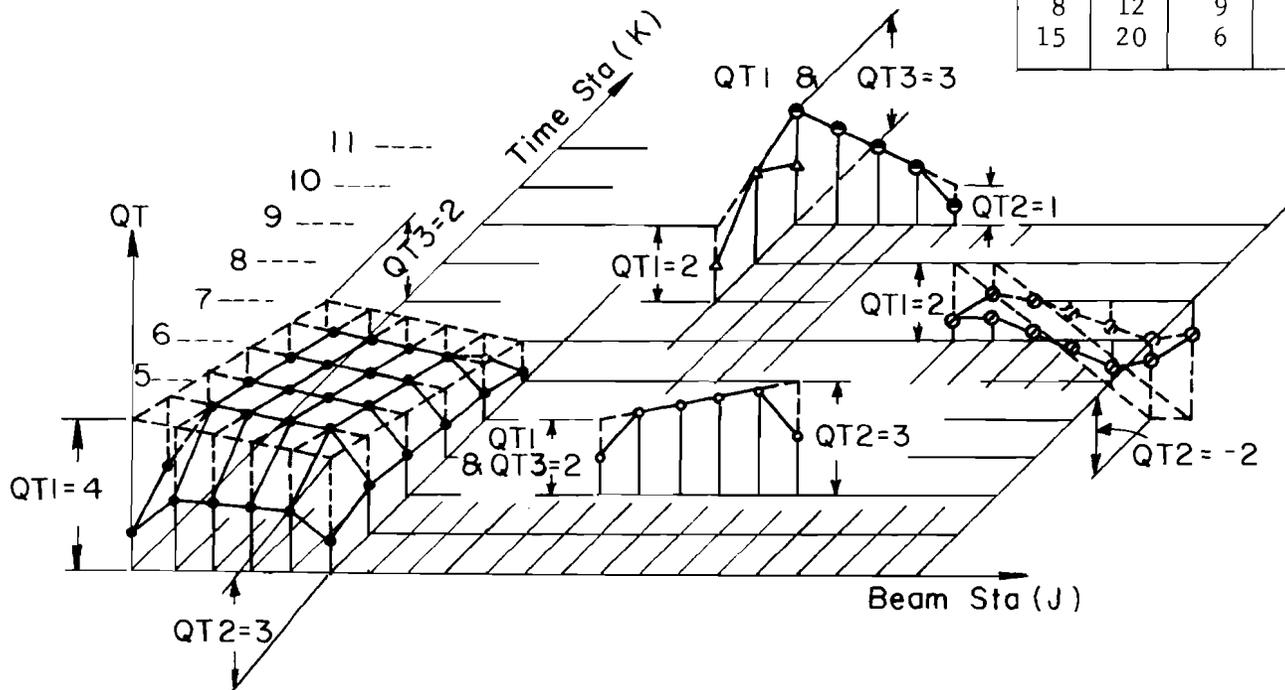


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TABLE 7. TIME DEPENDENT LATERAL LOADING (CONTINUED)

The variable  $QT(J,K)$  is input at any beam station and time station by specifying the value of lateral load distributed at beam station "FROM" and time station "FROM" in the columns of inputting  $QT1$ , the value of lateral load distributed at beam station "TO" and time station "FROM" in the columns of inputting  $QT2$ , and the value of lateral load distributed at beam station "FROM" and time station "TO" in the columns of inputting  $QT3$ .

Beam From	Sta To	Time From	Sta To	QT1	QT2	QT3	
0	5	0	5	4.0E+00	3.0E+00	2.0E+00	●
10	15	2	2	2.0E+00	3.0E+00	2.0E+00	○
8	8	7	9	2.0E+00	2.0E+00	3.0E+00	△
8	12	9	9	3.0E+00	1.0E+00	3.0E+00	●
15	20	6	7	2.0E+00	-2.0E+00	1.0E+00	●



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APPENDIX E

GLOSSARY OF NOTATION FOR DBC5

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C	-----NOTATION FOR CASE		07J01			
C	A( ), ATEMP, AHEV	CONTINUITY COEFFICIENT	07J01	C	ICPS( )	SPECIFIED SWITCH INPUT IN TABLE 7 TO INDICATE THAT THE SPECIFIED CONDITIONS OF DISPLACEMENTS ARE PERMANENT OR INITIALLY TEMPORARY VALUE OF ICPS( )
C	AN1( ), AN2( ),	IDENTIFICATION AND REMARKS TO BE OBSERVED	07J01	C	IOPSAV	ROUTING SWITCH (LOAD ITERATION METHOD)
C	A1( ), B1( ), C1( )	RECURSION OR CONTINUITY COEFFICIENTS AT BEAM STA J	07J01	C	IOSW( )	OPTIONAL SWITCH FOR PRINTING THE RESULTS OF MONITOR STAS (-1, BLANK, OR +1)
C	B( ), BTEMP, BHEV	CONTINUITY COEFFICIENT	07J01	C	IPS	INDICATOR THAT IA <sub>2</sub> IS EVEN OR ODD
C	BDENOM	BEAM INCREMENTS DENOMINATOR	07J01	C	IQS	INDICATOR THAT IA <sub>2</sub> IS EVEN OR ODD
C	B1AC	BEAM INCREMENTS FOR INTERPOLATION IN TABLES 6 AND 7	07J01	C	IQ2( ), IQ3( )	INITIAL AND FINAL EXTERNAL STA (TABLE 7)
C	BMI( )	BENDING MOMENT	07J01	C	IKLL	OPTION TO MOVE TO A NEW PLOT FRAME
C	C( ), CTEMP, CHEV	CONTINUITY COEFFICIENT	07J01	C	ISTAL( )	INTERNAL BEAM STA WHERE A NONLINEAR SUPPORT EXISTS
C	CN1( ), CP1( )	STATIC TRANSVERSE TORQUE (INPUT AND TOTAL)	07J01	C	ISW( )	ROUTING SWITCH FOR HALF VALUES
C	CTMP, CREV	MULTIPLIER IN CONTINUITY COEFFS	07J01	C	ITCS	ROUTING SWITCH (1ST CYCLE OF ITERATIONS)
C	DEW( )	SHEAR BETWEEN ADJACENT STATIONS	07J01	C	ITCSR	ROUTING SWITCH (2ND CYCLE OF ITERATIONS)
C	DEWOM	DENOMINATOR	07J01	C	IT1( ), IT2( )	INITIAL AND FINAL EXTERNAL STA (TABLE 6)
C	DEK1( ), DEK2( )	LUMPED EXTERNAL DAMPING COEFFICIENT (INPUT AND TOTAL)	07J01	C	ITSW	ROUTING SWITCH TO INDICATE THAT THE PROBLEM HAS LINEAR (O OR BLANK), NONLINEAR OR BOTH (OTHER THAN 0) SUPPORTS
C	DIFF	DIFFERENCE	07J01	C	IUS( )	ROUTING SWITCH TO INDICATE THAT THE Q-W CURVE AT STA NEEDS TO BE ADJUSTED
C	DIK1( ), DIK2( )	LUMPED INTERNAL DAMPING COEFFICIENT (INPUT AND TOTAL)	07J01	C	IX( )	INDEPENDENT VARIABLE (INDEXING)
C	DW( )	FIRST DERIVATIVE OF BMD/ DEF (SLOPE)	07J01	C	IYSW( )	EXTERNAL TIME OR BEAM STA INPUT IN TABLE 2
C	DWS( )	SPECIFIED VALUE OF SLOPE AT STA IS	07J01	C	J	INTERNAL BEAM STA
C	DWSAV( )	TEMPORARY VALUE OF DWS( )	07J01	C	JIMS( )	ARRAY CONTAINING THE MONITOR STA FOR PRINTING THE ITERATION DATA
C	E	TERM IN CONTINUITY COEFFS	07J01	C	JM	INTERNAL MONITOR STA NUM
C	ESH, ESM, EST	FACTORS FOR DISTRIBUTING HALF VALUES AT END STAS	07J01	C	JN	INDEX OF NUM OF POINTS OF THE PREVIOUS PLOTS TO BE ACCUMULATED ON NEXT PLOT
C	ESM	MULTIPLIER FOR HALF VALUES AT END STAS	07J01	C	JPS( )	ARRAY CONTAINING THE MONITOR STA FOR PRINTING THE COMPUTED RESULTS
C	E12( ) * F( )	FLEXURAL STIFFNESS (FI) (INPUT AND TOTAL)	07J01	C	JS	INTERNAL BEAM STA FOR SPECIFIED CONDITIONS IN TABLE 3
C	F	BEAM INCREMENT LENGTH	07J01	C	JSTA	INTERNAL BEAM STA FOR PRINTING THE RESULTS
C	F12	H SQUARED	07J01	C	JT1	INITIAL INTERNAL STA (TABLES 6 AND 7)
C	F12	H CUBED	07J01	C	JT2	FINAL INTERNAL STA (TABLES 6 AND 7)
C	HT	TIME INCREMENT LENGTH	07J01	C	J1	INITIAL INTERNAL BEAM STA USED IN TABLE 4
C	HT2	H TIMES 2	07J01	C	J2	FINAL INTERNAL BEAM STA USED IN TABLE 4
C	HT2	H SQUARED	07J01	C	K	INTERNAL TIME STA
C	H31	H CUBED DIVIDED BY HT	07J01	C	KASE( )	CASE NUM OF SPECIFIED CONDITIONS 1 = DEFL, 2 = SLOPE, 3 = ROT-
C	H32	H CUBED DIVIDED BY HT SQUARED	07J01	C	KCLOSE	SWITCH TO INDICATE THAT DEFLECTIONS ARE NOT CLOSED (IF = 0)
C	IA	NUM OF PLOTS TO BE PLOTTED ON ONE FRAME	07J01	C	KCNT( )	COUNTER FOR CLOSURE OF DEFLECTIONS IF=1, KEEP PRIOR DATA, TABLES 2 = 4
C	IA2	IA DIVIDED BY 2 (INTEGER)	07J01	C	KEY( )	ROUTING SWITCH FOR IDENTIFYING CASE NUM OF SPECIFIED CONDITIONS OF DISPLACEMENTS
C	IC( ), IS( ), ISS( )	VARIABLES FOR STOPPING THE PLOT TITLES	07J01	C	KEYP( )	SWITCH TO INDICATE THAT PLOT AXIS IS TIME (IF=1) OR BEAM (IF=2)
C	ICL( )	NUM OF POINTS IN JTH PLOT	07J01	C	KEYPT( ), KYST( )	TEMPORARY VALUE OF KEYP( ) \$ KYST( ) FOR PLOT OF MOMENT ALONG THE BEAM AXIS
C	IFLS1( )	SWITCH FOR FIRST UNLOADING CASE	07J01	C	KK	COUNTER FOR THREE CONSECUTIVE TIME STA
C	IN13( )	BEAM STA WHICH IS SPECIFIED FOR DISPLACEMENT IN TABLE 3	07J01	C	KMAX	COUNTER FOR NUM OF STAS EXCEEDING MAX ALLOWABLE DEFLECTION
C	IN14( ), IN15( )	INITIAL EXTERNAL BEAM STA (TABLES 4 AND 5)	07J01			
C	IN18( ), IN21( )	INITIAL AND FINAL EXTERNAL STA (TABLE 6)	07J01			
C	IN24( ), IN25( )	FINAL EXTERNAL BEAM STA (TABLES 4 AND 5)	07J01			
C	IOPAV	TEMPORARY VALUE OF IOPSAV	07J01			
C	IOPAVT	NUM OF NONLINEAR SUPPORTS WHERE COMPUTED DEF ARE OSCILLATING BETWEEN PRIOR AND ADJUSTED Q-W CURVES WHEN UNLOADING CASE OCCURS	07J01			
C	IOP( )	ROUTING SWITCH FOR INITIALLY AND PERMANENTLY SPECIFIED CONDITIONS	07J01			

C KNCNT( ) VALUE OF KNCNT( ) (FIRST ITERATION) 07J01  
C KN1( ) INITIAL INTEGRAL TIME STA (TABLES 6 AND 7) 07J01  
C KN2( ) FINAL INTEGRAL TIME STA (TABLES 6 AND 7) 07J01  
C KOFFC( ) SWITCH TO INDICATE THAT DEFLECTIONS EXCEED 07J01  
C LIMITS OF SUPPORT CURVES  
C KPC ROUTING COUNTER FOR COMPLETE PRINT-OUT 07J01  
C OF RESULTS  
C KPI COMPLETE PRINTOUT OF THE RESULTS OF EVERY 07J01  
C STA AT EVERY KPI TIME STA  
C KPS4 INTERNAL TIME STA OF KPI ( KPI\*4 ) 07J01  
C KQ1( ) INITIAL EXTERNAL TIME STA USED IN TABLE 7 07J01  
C KQ2( ) FINAL EXTERNAL TIME STA USED IN TABLE 7 07J01  
C KR1 PRIOR VALUE OF KR24( ), KR25( ), KR28( ) 07J01  
C KR24( ), KR25( ), CONTINUE SWITCH FOR INPUT CONTINUING ON 07J01  
C KR28( ) NEXT CARD ( TABLES 4, 5, AND 8 ) 07J01  
C KSI( ) SAVED VALUE OF KSAV( ) FOR EACH PLOT FRAME 07J01  
C KSAV( ) TEMPORARY VALUE OF KASE( ) 07J01  
C KSAV( ) ARRAY CONTAINING EXTERNAL BEAM OR TIME STA 07J01  
C FOR THE TITLES OF PLOTS  
C KSA4( ), KSW5( ), ROUTING SWITCH IN TABLES 4, 5 AND 8 07J01  
C KSW4( )  
C KSYM( ) SYMMETRY OPTION USED IN TABLE 8 07J01  
C KSYM(J) SPECIFIED VALUE OF KSYM( ) AT EACH STA J 07J01  
C KTI( ) INITIAL EXTERNAL TIME STA USED IN TABLE 6 07J01  
C KT2( ) FINAL EXTERNAL TIME STA USED IN TABLE 6 07J01  
C KYS( ) SWITCH TO INDICATE THAT VERT PLOT AXIS IS 07J01  
C DEFLECTION (IF=1) OR MOMENT (IF=2)  
C LOP OPTIONAL LINE OR POINT PLOT SWITCH WHEN 07J01  
C USING MICROFILM OR BALL-POINT PLOTS  
C LSM SWITCH WHICH CAUSES THE AXIAL LOADS TO BE 07J01  
C DISTRIBUTED TO HALF-STAS  
C LSTA INDEX FOR EXTERNAL BEAM STA 07J01  
C M NUM OF BEAM INCREMENTS 07J01  
C MAXI NUM OF POINTS FOR EACH PLOT FRAME 07J01  
C MAXIT NUM OF ITERATIONS SPECIFIED (MAX = 50) 07J01  
C MC NUM OF CHARACTERS IN PLOT TITLES 07J01  
C MONI NUM OF MONITOR STATIONS FOR PRINTING THE 07J01  
C ITERATION DATA (MAX = 5)  
C MONS NUM OF MONITOR STATIONS FOR PRINTING RE- 07J01  
C SULTS (MAX = 10)  
C MOP OPTIONAL PLOTTING METHOD SWITCH 07J01  
C (-1=MICROFILM, 0 OR BLANK=PRINTER,  
C +1=BALL-POINT)  
C MPC( ) MULTIPLE PLOT SWITCH SAVED FOR EACH PLOT 07J01  
C MPCT( ) TEMPORARY VALUE OF MPC( ) FOR PLOTS OF 07J01  
C MOMENTS ALONG THE BEAM AXIS  
C MPS( ) MULTIPLE PLOT SWITCH INPUT IN TABLE 9 (IF 07J01  
C +1 PLOT IS SUPERIMPOSED WITH NEXT)  
C MP1 THRU MP7 M PLUS ONE THRU M PLUS SEVEN 07J01  
C MT NUM OF TIME INCREMENTS 07J01  
C MTF2 MT PLUS TWO 07J01  
C MTF4 MT PLUS FOUR 07J01  
C NA NUM OF PLOTS SUPERIMPOSED ON ONE FRAME 07J01  
C NAC INDEX OF NUM OF PLOTS 07J01

C NCC2 THRU NCC9 NUM OF CARDS INPUT IN TABLES 2 THRU 9 FOR 07J01  
C THIS PROBLEM  
C NCT2 THRU NCT7+NCT9 TOTAL NUM OF CARDS IN THE PARTICULAR TABLE 07J01  
C NCVR TOTAL NUM OF CURVES IN TABLE 8 07J01  
C NC14 THRU NC19 INITIAL INDEX VALUE FOR THE INPUT TO THE 07J01  
C PARTICULAR TABLE  
C NEFR NUM OF CARDS INPUT REPEATEDLY IN TABLE 9 07J01  
C NIT INDEX USED IN THE ITERATION DO LOOP 07J01  
C NITS ITERATION NUMBER AT WHICH THE LOAD ITERA- 07J01  
C TION METHOD STARTS  
C NITT SAVED VALUE OF NIT FOR THE FIRST CYCLE OF 07J01  
C ITERATIONS  
C NNC NUM OF SUPPORTING STAS WHICH HAVE NON- 07J01  
C LINEAR RESISTANCE-DEFLECTION CURVES  
C NDCI ROUTING SWITCH WHICH TELLS THE PROGRAM DO 07J01  
C ONE MORE CYCLE OF LOAD ITERATIONS  
C NOS WHEN THE SOLUTION DOES NOT CLOSE 07J01  
C INDEX FOR COUNTING THE NUM OF SLOPES OF 07J01  
C NONLINEAR CV  
C NOSJ(J) NUM OF SLOPES ON THE NONLINEAR RESISTANCE- 07J01  
C DEFLECTION CV AT A PARTICULAR STA J  
C NPB INDEX FOR TOTAL NUM OF PLOTS 07J01  
C NPBT INDEX FOR NUM OF PLOTS OF BENDING MOMENTS 07J01  
C ALONG THE BEAM AXIS  
C NPCNT ROUTING COUNTER FOR THE NUM OF POINTS ON 07J01  
C THE NONLINEAR RESISTANCE-DEFLECTION  
C CV WHICH IS INCREASING ITS SPECIFIED 07J01  
C VALUE OF DEFLECTION FROM NEG TO ZERO  
C OR FROM NEG TO POSITIVE 07J01  
C NPCT TEMP VALUE OF NUM OF POINTS ON SUP CV 07J01  
C NPCTS(J) TOTAL NUM OF POINTS ACTUALLY ON THE NON- 07J01  
C LINEAR RESISTANCE-DEFLECTION CV AT A  
C PARTICULAR STA J 07J01  
C NPCC( ) NUM OF POINTS INPUT IN TABLE 8 TO DEFINE A 07J01  
C PARTICULAR NONLINEAR RESISTANCE-  
C DEFLECTION CURVE 07J01  
C NPROB PROBLEM NUMBER (PROG STOPS IF = BLANK) 07J01  
C NPS INDEX WHICH INDICATES THAT THE 07J01  
C DEFLECTION AT A PARTICULAR STA IS  
C WITHIN THE SEGMENT CONNECTED BETWEEN 07J01  
C POINT NPS AND POINT (NPS-1) ON THE  
C NONLINEAR SUP CV 07J01  
C NPT INDEX FOR NUM OF PLOTS ALONG TIME AXIS 07J01  
C NPPTS(J) NUM OF POINTS ON THE NONLINEAR RESISTANCE- 07J01  
C DEFLECTION CV AT A PARTICULAR STA J  
C NPZERC ROUTING COUNTER WHICH DEFINES THE NUM OF 07J01  
C SLOPES SPECIFIED ON THE PARTICULAR  
C NONLINEAR RESISTANCE-DEFI CV 07J01  
C NDIS ROUTING SWITCH TO INDICATE THAT THE 07J01  
C DEFLECTION AT A PARTICULAR SUP STA IS  
C EQUAL TO THE VALUE OF THE FIRST 07J01  
C POINT ON THE NONLINEAR SUP CV (IF =1)  
C AS INDEX NUM FOR SPECIFIED CONDITIONS 07J01  
C NUMOC(J) NUM OF OSCILLATING CYCLES DURING THE 07J01

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C          ITERATION PROCESS OF THE DEFLECTION 07J01
C          AT A PARTICULAR STA J              07J01
C          INTERPOLATION FRACTION              07J01
PART      TRANSVERSE STATIC LOAD (INPUT AND TOTAL) 07J01
C          GDR2( ) , R(J)
C          GDRDP      PROJECTED VERTICAL VALUE OF EACH SEGMENT 07J01
C                   ON THE SUP CV              07J01
C          GI(J)      ITERATIVE LOAD DISTRIBUTED AT BEAM STA J 07J01
C                   FOR THE PRESENT TIME STA   07J01
C          GIM1(J)    ITERATIVE LOAD DISTRIBUTED AT BEAM STA J 07J01
C                   FOR THE PREVIOUS TIME STA  07J01
C          GIM2(J)    ITERATIVE LOAD DISTRIBUTED AT BEAM STA J 07J01
C                   TWO TIME STAS AGO         07J01
C          GMP( )     INPUT VALUE OF RESISTANCE-MULTIPLIER 07J01
C          GNTOL( )   INPUT VALUE OF RESISTANCE-TOLERANCE 07J01
C          GNTOLW( )  DISTRIBUTED VALUE OF RESISTANCE-TOLERANCE 07J01
C          GP         INPUT SUP CV RESISTANCE VALUES 07J01
C          GPT        FINAL DISTRIBUTED RESISTANCE VALUES 07J01
C          GPV        FINAL SUP CV RESISTANCE VALUES 07J01
C          GPVT       TEMPORARY VALUES OF RESISTANCE OF THE 07J01
C                   POINTS ON SUP CV          07J01
C          GS(J)      SLOPE CORRECTION LOAD AT BEAM STA J FOR 07J01
C                   THE PREVIOUS TIME STA     07J01
C          GT(J,1)    TRIAL VALUE OF TIME DEPENDENT LOAD 07J01
C                   DISTRIBUTED AT BEAM STA J  07J01
C                   FOR THE PRESENT TIME STA   07J01
C          GT(J,2)    TOTAL VALUE OF THE TIME DEPENDENT LOAD 07J01
C                   TWO TIME STA AGO         07J01
C          GT1( ) , GT2( ) , GT3( ) INPUT VALUES OF TIME DEPENDENT LOADS 07J01
C                   (TABLE 7)                07J01
C          REACT( )   NET REACTION ON THE RUCOL AT EACH BEAM STA 07J01
C          RHON2( ) , RHO(J) LUMPED MASS DENSITY (INPUT AND TOTAL) 07J01
C          RN2( ) , RI(J) ROTATIONAL RESTRAINT (INPUT AND TOTAL) 07J01
C          SLOPE      SLOPE OF EACH SEGMENT ON THE SUP CV 07J01
C          SM2( ) , S(J) LINEAR SPRING SUP STIFF (INPUT AND TOTAL) 07J01
C          SS(J)      FINAL SPRING CONSTANT DISTRIBUTED AT BEAM 07J01
C                   STA J FOR THE PRESENT TIME STA 07J01
C          SSKM1(J)   FINAL SPRING CONSTANT DISTRIBUTED AT BEAM 07J01
C                   STA J FOR THE PREVIOUS TIME STA 07J01
C          SSKM2(J)   FINAL SPRING CONSTANT DISTRIBUTED AT BEAM 07J01
C                   STA J TWO TIME STAS AGO     07J01
C          SYMB( )    SYMBOLS USED ON THE PRINTER PLOTS 07J01
C          TLENDM     TIME INCREMENTS DENOMINATOR 07J01
C          TESTB     * 4 LETTERS #BEAM# USED TO INDICATE THAT 07J01
C                   PLOT IS ALONG BEAM AXIS    07J01
C          TESTD     * 4 LETTERS #DEFL# USED TO INDICATE THAT 07J01
C                   THE VERT AXIS IS DEFLECTION 07J01
C          TESTM     * 4 LETTERS #MOMT# USED TO INDICATE THAT 07J01
C                   THE VERT AXIS IS MOMENT     07J01
C          TESTT     * 4 LETTERS #TIMX# USED TO INDICATE THAT 07J01
C                   PLOT IS ALONG TIME AXIS    07J01
C          TINC       TIME INCREMENTS FOR INTERPOLATION IN 07J01
C                   TABLES 6 AND 7           07J01
C          TN2( ) , T(J) STATIC AXIAL TENSION OR COMPRESSION (INPUT 07J01
C                   AND TOTAL)                07J01

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C          TT1(J,1)  TOTAL VALUE OF TIME DEPENDENT AXIAL 07J01
C                   THRUST FOR THE PRESENT TIME STA 07J01
C          TT1(J,2)  TOTAL VALUE OF TIME DEPENDENT AXIAL 07J01
C                   THRUST AT TWO TIME STA AGO 07J01
C          TT1( ) , TT2( ) , TT3( ) INPUT VALUES OF TIME DEPENDENT AXIAL 07J01
C                   THRUSTS (TABLE 6)         07J01
C          W(L,KK)   COMPUTED DEFLECTION AT BEAM STA J FOR THE 07J01
C                   PRESENT TIME STA         07J01
C          W(L,KK-1) COMPUTED DEFLECTION AT BEAM STA J FOR THE 07J01
C                   PREVIOUS TIME STA        07J01
C          W(L,KK-2) COMPUTED DEFLECTION AT BEAM STA J TWO 07J01
C                   TIME STAS AGO           07J01
C          WDRDP     PROJECTED HORIZONTAL VALUE OF EACH SEGMENT 07J01
C                   ON THE SUP CV           07J01
C          WM        ITERATION DATA STORED FOR THE FIRST CYCLE 07J01
C                   OF ITERATION           07J01
C          WMAX      SPECIFIED MAXIMUM ALLOWABLE DEFLECTION 07J01
C          WMP( )    INPUT VALUE OF DEFLECTION-MULTIPLIER 07J01
C          WNTOL( )  INPUT VALUE OF DEFLECTION-TOLERANCE 07J01
C          WNTOLW( ) DISTRIBUTED VALUE OF DEFLECTION-TOLERANCE 07J01
C          WP        INPUT SUP CV DEFLECTION VALUES 07J01
C          WPT       FINAL DISTRIBUTED REFLECTION VALUES 07J01
C          WPV       FINAL SUP CV DEFLECTION VALUES 07J01
C          WPVT      TEMPORARY VALUES OF DEFLECTION OF THE 07J01
C                   POINTS ON SUP CV        07J01
C          WS1(J,5)  SPECIFIED VALUE OF DEFLECTION AT STA JS 07J01
C          WU        ITERATION DATA STORED FOR THE SECOND CYCLE 07J01
C                   OF ITERATION           07J01
C          WW( )     DEFLECTION AT PREVIOUS ITERATION 07J01
C          WWW( )    DEFLECTION TWO ITERATIONS AGO 07J01
C          XAXIS( )  NAME OF HORIZONTAL AXIS INPUT IN TABLE 9 07J01
C          XFC( )    STORED VALUES OF BEAM OR TIME STA NUM 07J01
C                   FOR THE PLOTS          07J01
C          XM( )     SCALE FOR THE HORIZ PLOT AXIS 07J01
C          Y( ) , Yt( ) , YP( ) STORED VALUES OF DEFLECTIONS OR MOMENTS 07J01
C                   FOR THE PLOTS          07J01
C          YA, AA   COEFF IN STIFFNESS MATRIX 07J01
C          YAXIS( ) NAME OF VERTICAL AXIS INPUT IN TABLE 9 07J01
C          YB, HB   COEFF IN STIFFNESS MATRIX 07J01
C          YC, CC   COEFF IN STIFFNESS MATRIX 07J01
C          YD, DC   COEFF IN STIFFNESS MATRIX 07J01
C          YE, EE   COEFF IN STIFFNESS MATRIX 07J01
C          YF, FF   COEFF IN STIFFNESS MATRIX 07J01
C          YM( )    SCALE FOR THE VERT PLOT AXIS 07J01

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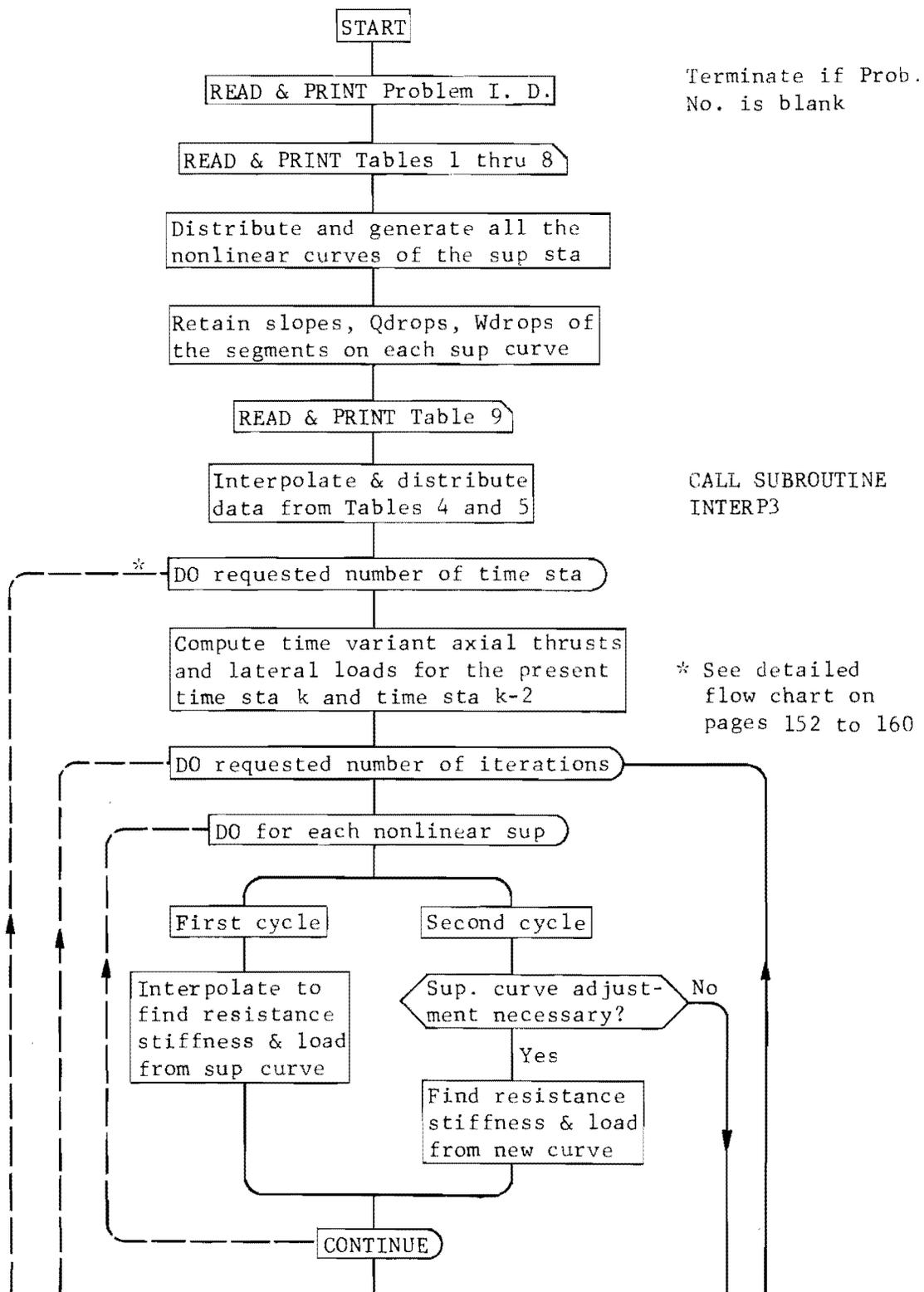
APPENDIX F

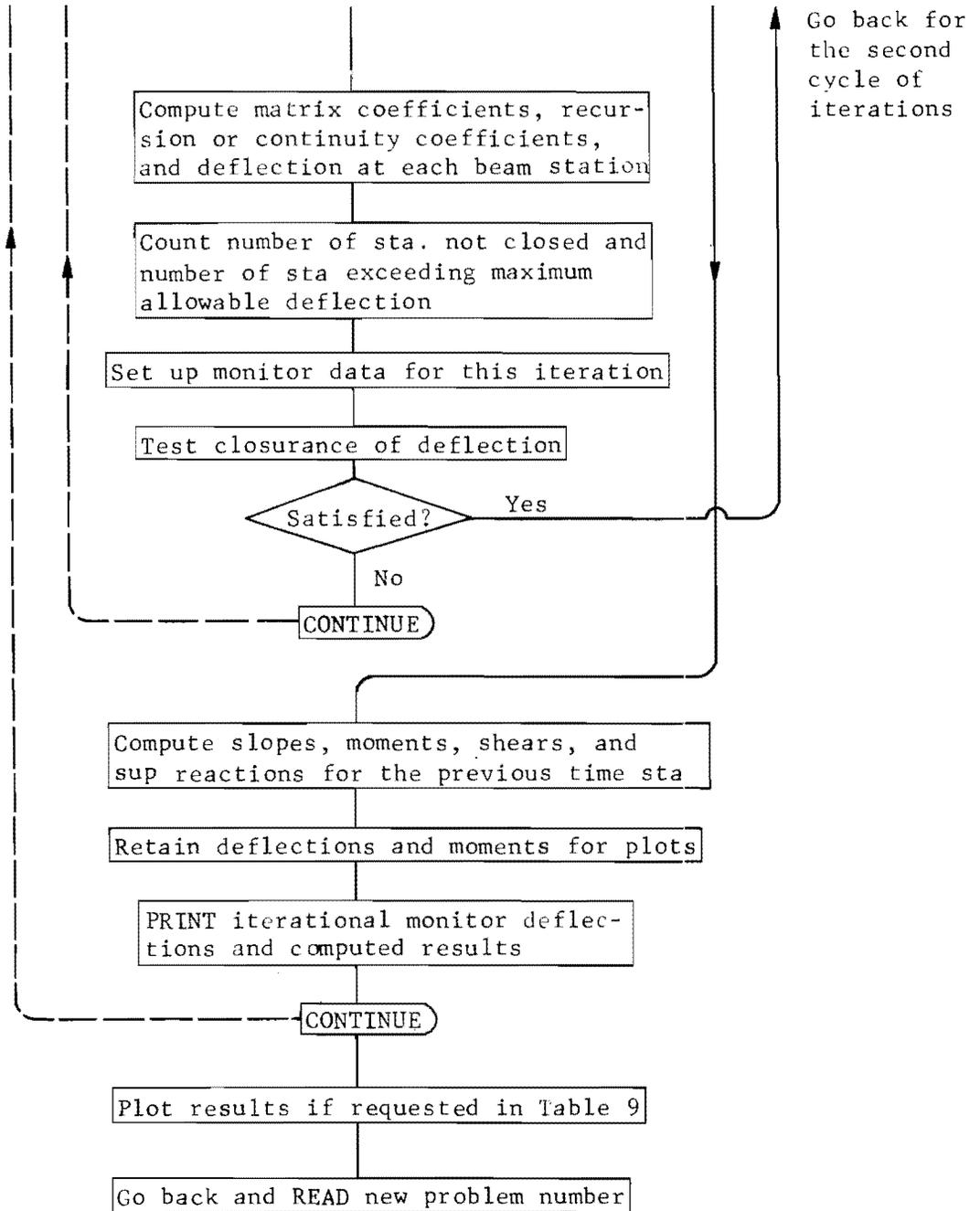
FLOW CHARTS AND LISTING OF DECK OF PROGRAM DBC5

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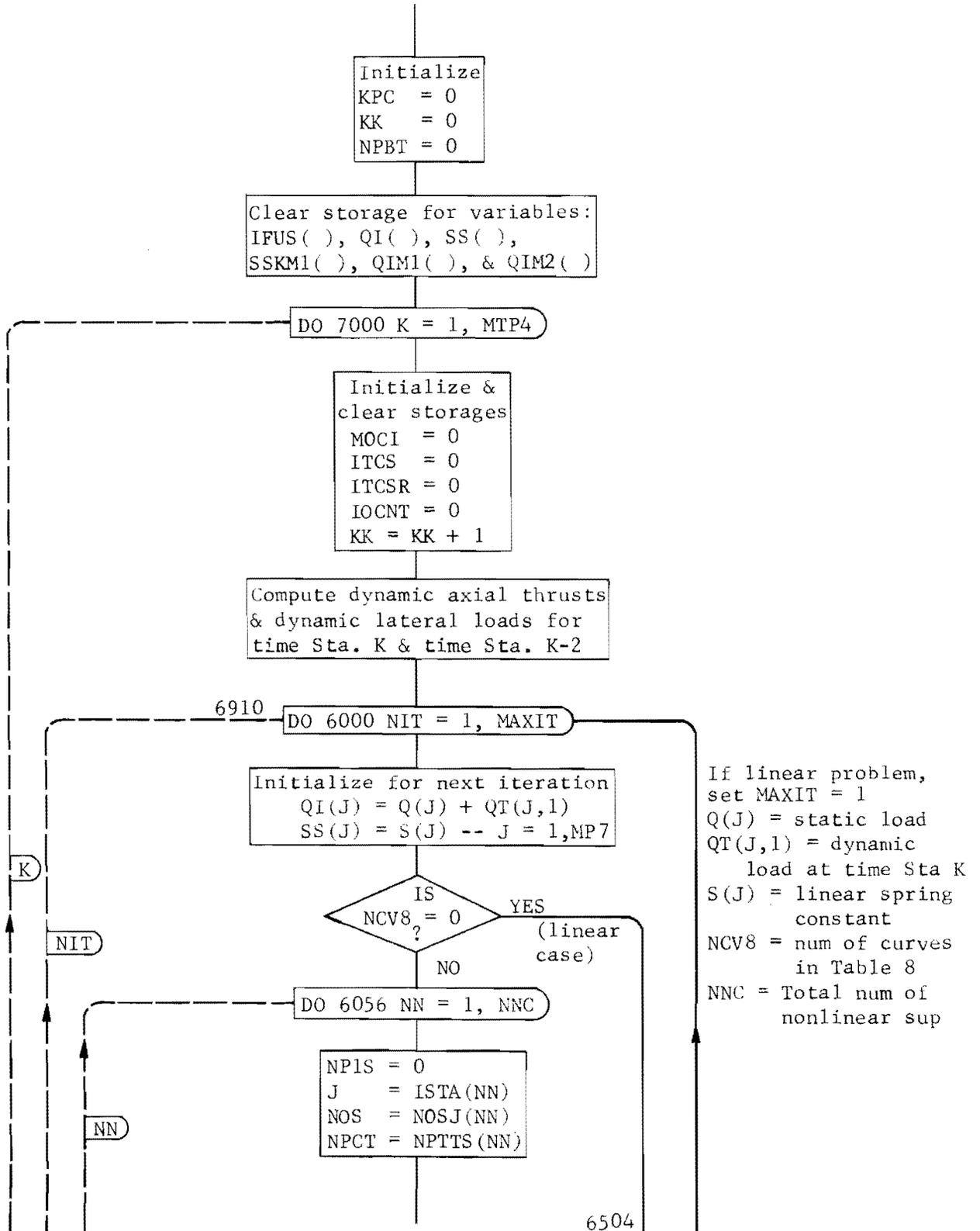
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GENERAL FLOW CHART OF DBC5



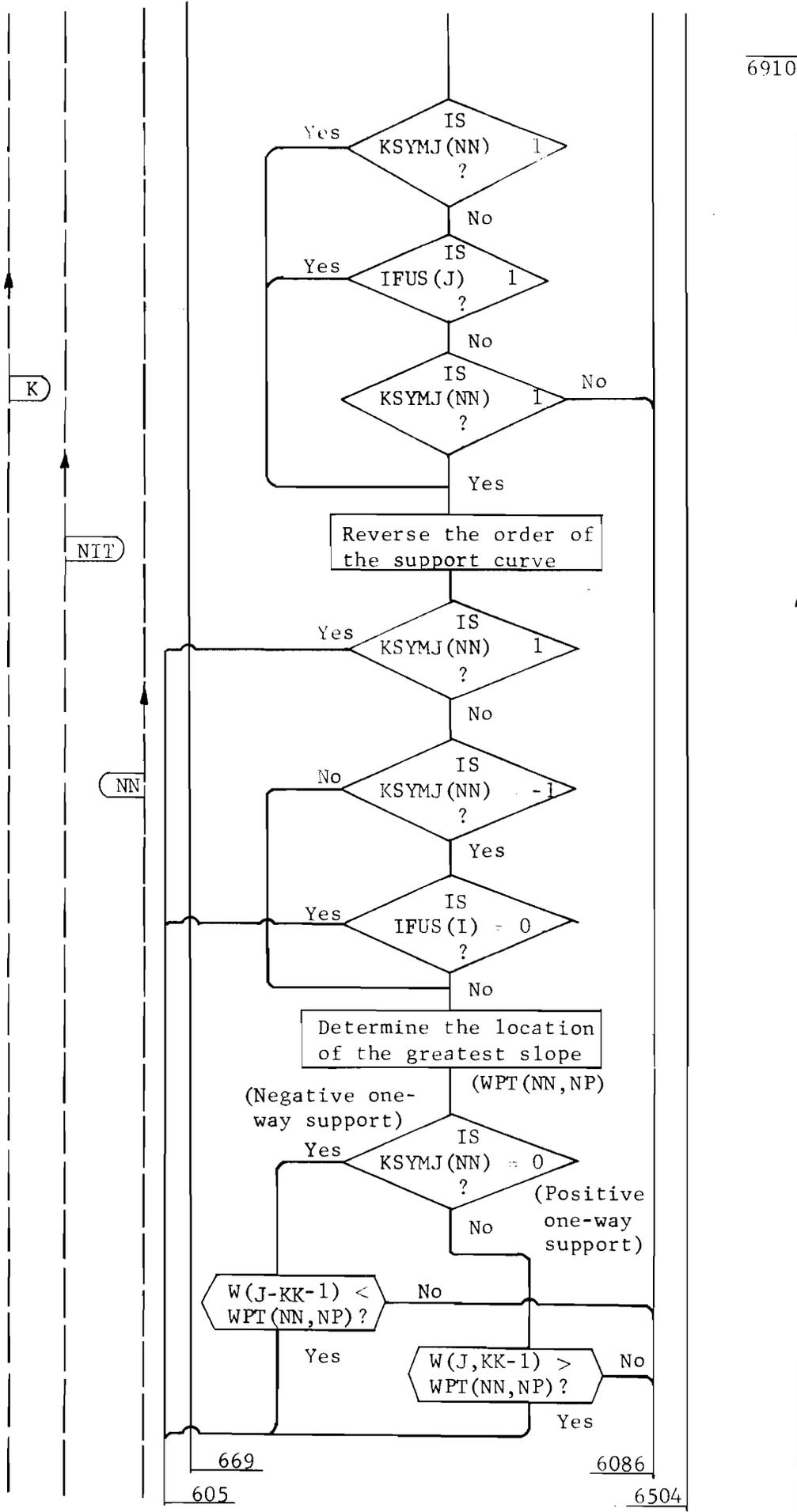


GENERAL FLOW CHART OF TIME DO LOOP





6910



K

NIT

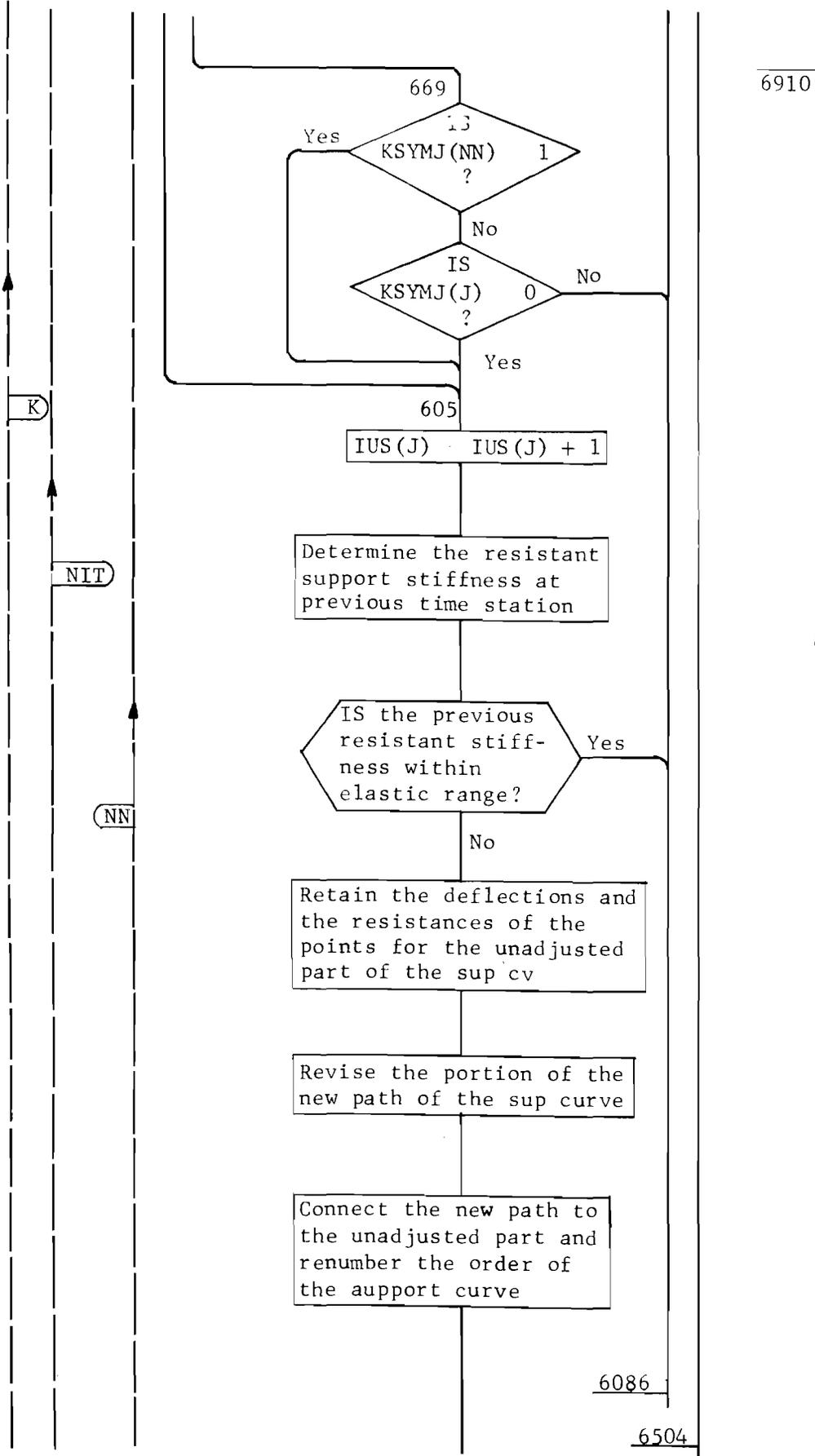
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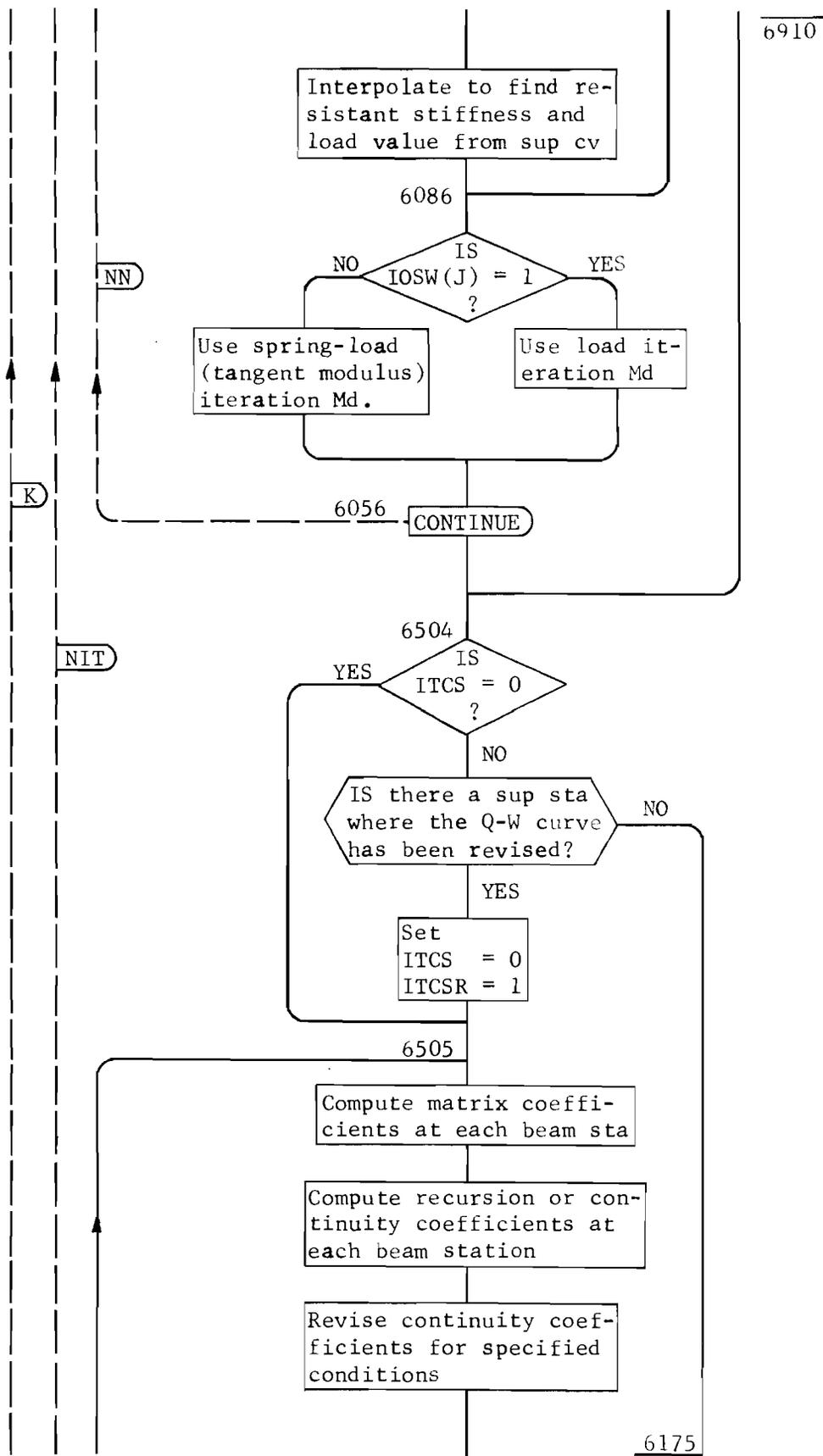
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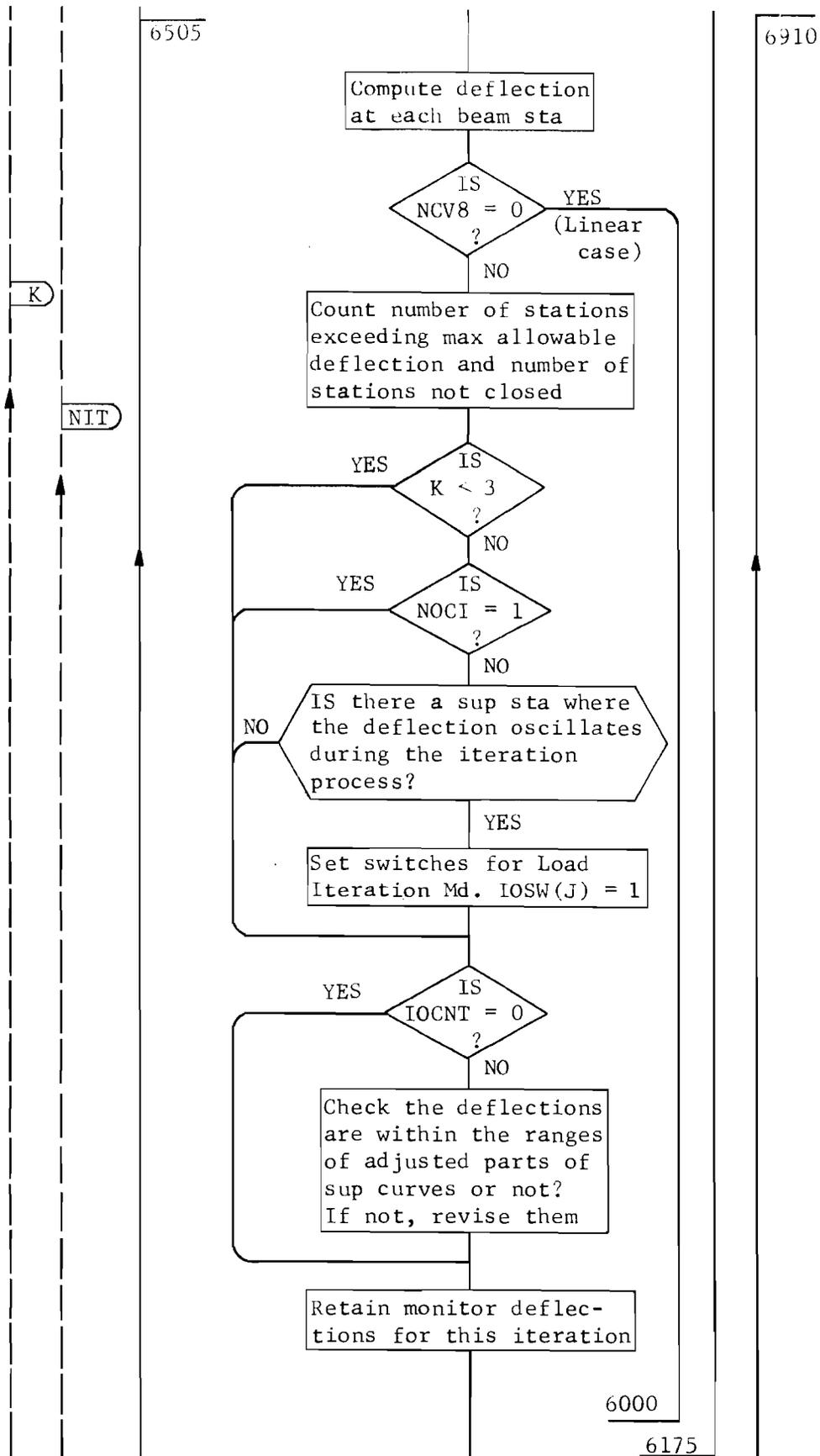
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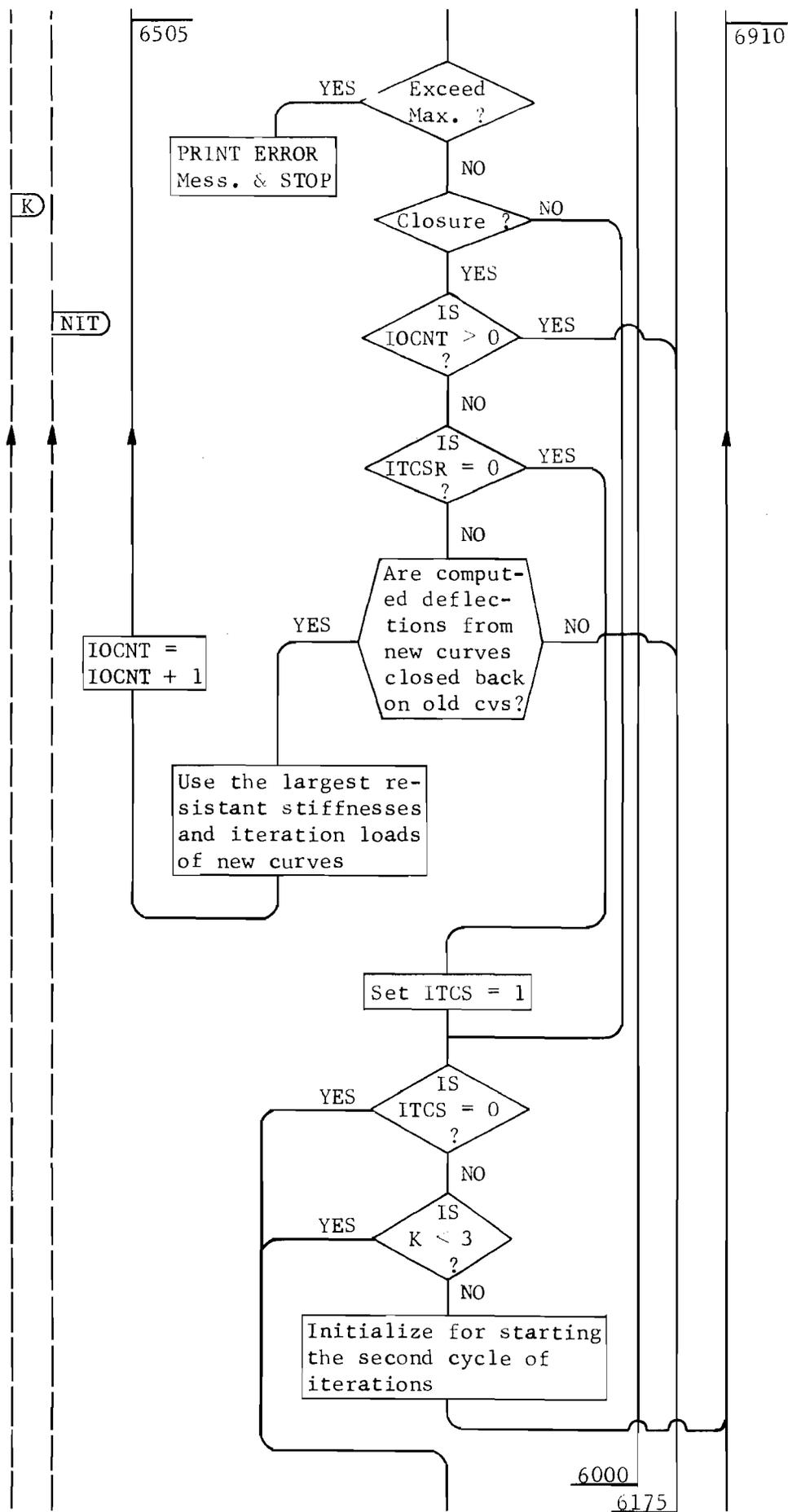
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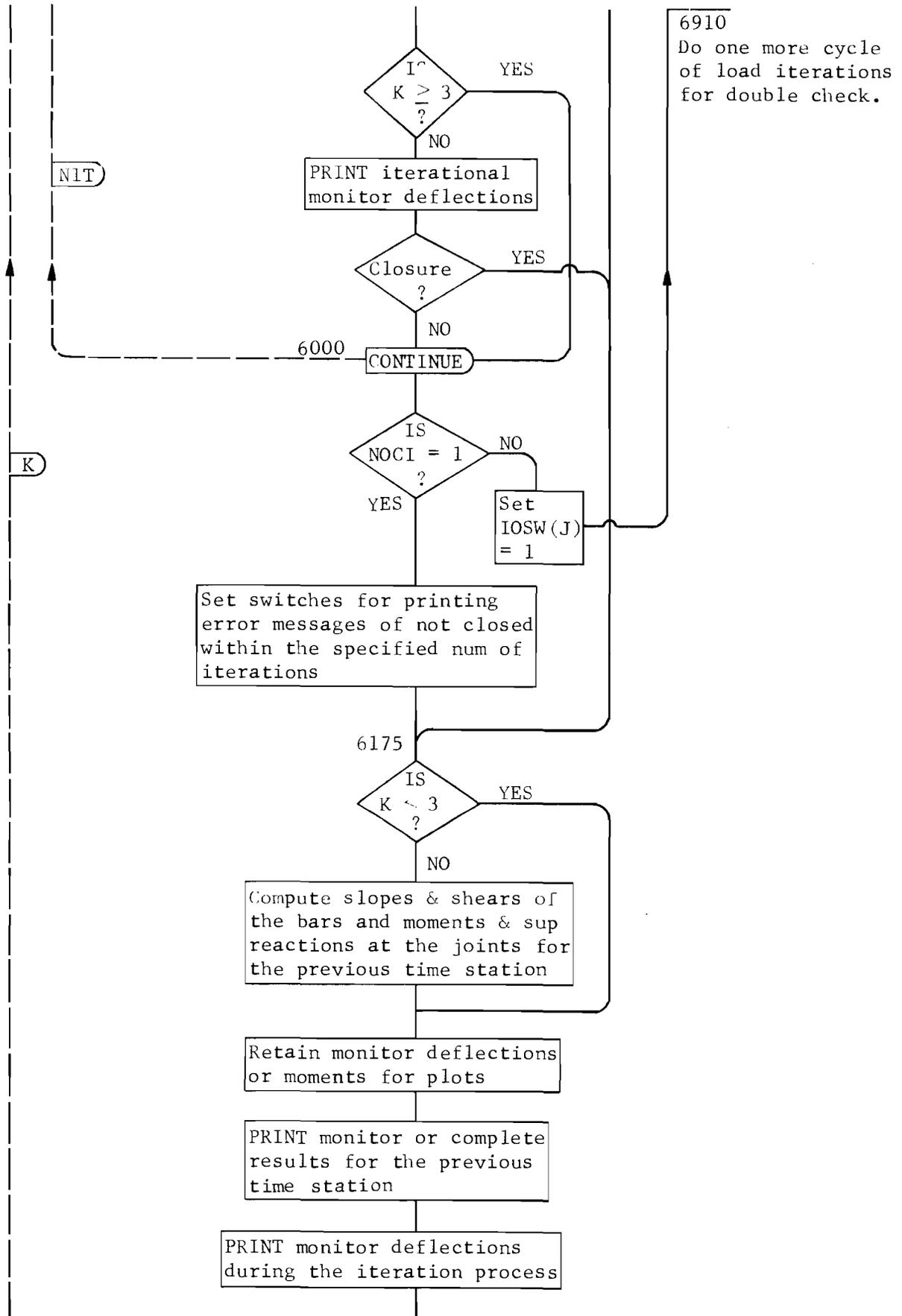


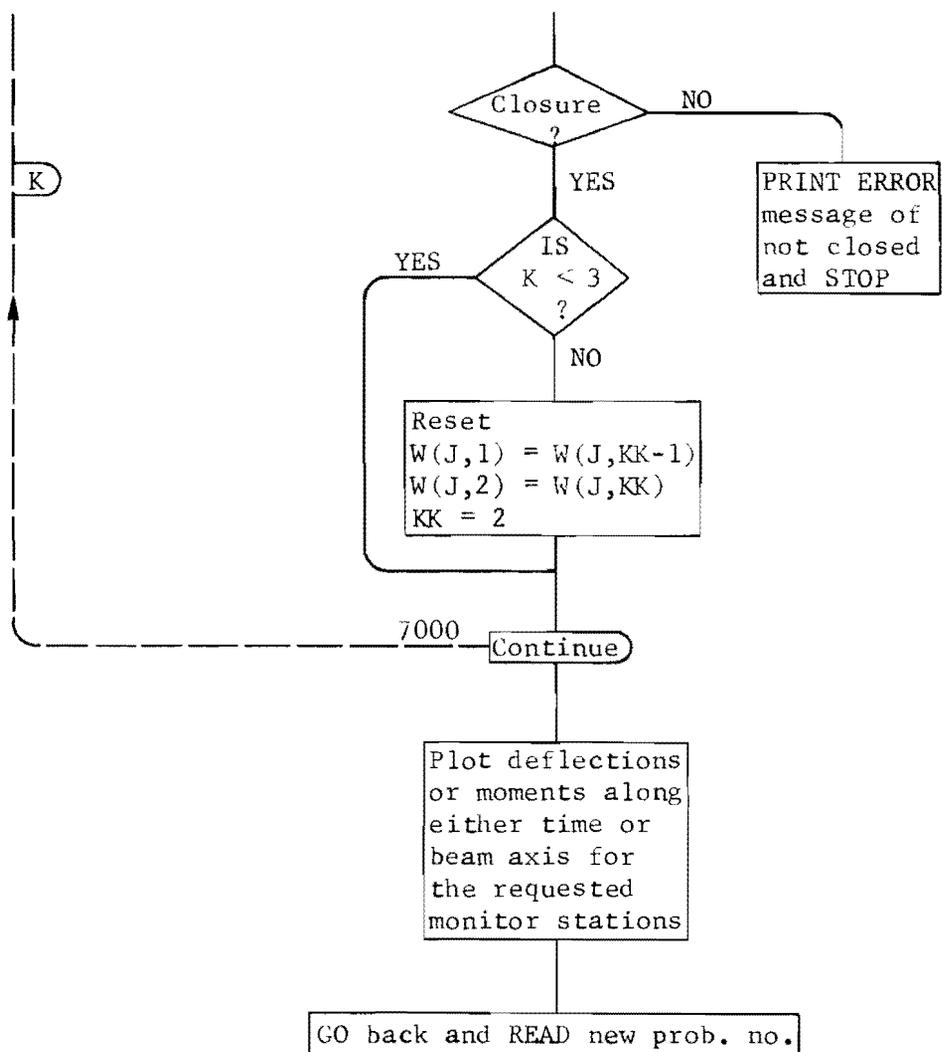
IUS(J) = Switch to indicate that the sup cv is revised











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PROGRAM DRCE ( INPUT, OUTPUT )
1 FORMAT ( ' F2H PROGRAM DRCE - MASTER - JACK CHAN - MATLOCK - 04JAI
1 32HLECK1-REVISION DATE = 26 JUN 71 ) 26JUI
1 DIMENSION AM1(2), AM2(14), A(207), R(207), C(207), 100C0
1 F(207), Q(207), S(207), T(207), R(207), CP(207), 100C0
2 KEY(207), IN13(20), KASF(20), WS(20), CWS(40), 100C0
3 IN14(100), I124(100), KR24(100), FN2(100) 055F0
4 QK2(100), S12(100), IN2(100), RK2(100), CN2(100), 055F0
5 KSW4(100), I115(100), IN25(100), KR25(100), 055F0
6 KSW5(100), QW(207), RHO(207), QI(207), DF(207), 100C0
7 RH(207), W(207,3), DR4(207), REACT(207), *S*(207), 100C0
8 RHO2(100), BIN2(100), DEN2(100), LPS(100), SYM(10) 055F0
1 DIMENSION TT(207,2), QT(207,2), IT1(100), IT2(100), IY1(100), 100C0
1 IY2(100), KT1(100), KT2(100), Q1(100), KQ2(100), 055F0
2 IT1(100), IT2(100), IT3(100), QT1(100), QI2(100), 055F0
3 QT1(100), XAXIS(10), IYSW(10), MPS(10), KSAV(10), 055F0
4 KEVP(10), YI(10,100), MPO(10), IXX(10), X5(10), 055F0
5 YK( 50,10), Ix(1002), YK(1004), IY(10), IS(10), *SS(2) 175E0
1 DIMENSION I1(120), INR(20), KR2(20), GPR(20), WSP(20), 04JAI
2 KSYM(20), KSWR(20), QP(20,10), WPI(20,10), QPV(20,20) 04JAI
3 , WPI(20,20), NPCTS(20), JIMS(5), QI(207), IUS(20) 04JAI
4 IOP(207), SS(207), WWW(207), KOFF(20), XAXIS(10), 04JAI
5 KCT(50), W(50,5), NUMOC(207), *C*(207), IOR(120), 04JAI
6 IOR(207), K(50,5), *STA(100), *C*(207,2), 15AF1
7 WPI(100,20), GPUT(100,20), KALC(20), *WRP(100,10) 15FE1
A SLOPE(100,10), WNROR(100,10), KTR(10), NOSJ(100), 15FE1
9 WPI(100,20), QPT(100,20), NPPTS(100), WNTOLJ(100), 15FE1
A WNTOLJ(100), WNTOL(20), QNTOL(20), KSYMJ(100), 15FE1
C KXNT(50), IJUS(207), KSAV(5), KEYPT(5), MPO(15), 15FE1
R KYCT(5), *T(5,207), SSXW1(207), YK(4), XW(4) 15FE1
C DIMENSION SSXW2(207), GIM(207), QIN2(207), QS(207) 2)AP1
1 DATA SYMB / 1H, 1H, 14X, 1H, 14X, 1H, 14X, 1H, 14X, 1H, 14X /
COMMON/SPLIT/WIDTH/SYMB/LL/IR/SYMB
COMMON //OT/ COP, PG, TRILL, WOP
INTEGER WIDTH
4 FORMAT ( A5, A5 )
5 FORMAT ( I5, I5 )
6 FORMAT ( I5, A5 )
10 FORMAT ( ' 5H * P0X, 10HT----TRIM )
11 FORMAT ( ' 5H1 * P0X, 10HT----TRIM )
12 FORMAT ( ' 14A5 )
13 FORMAT ( ' 5X, 14A5 )
14 FORMAT ( A5, 5X, 14A5 )
15 FORMAT ( ///10H PROR / 5X, A5, 5X, 14A5 )
16 FORMAT ( ///17H PROR (CONTR), / 5X, A5, 5X, 14A5 )
20 FORMAT ( ' 14T5 )
21 FORMAT ( 2( 5X, I5, F10.3 ), 5X, 6I5 )
22 FORMAT ( 5X, I5, 10X, 10T5 )
23 FORMAT ( 5X, I5, 2(10.3), 10X, F10.3 )
31 FORMAT ( 2(5X, I5), 2(F10.3, 5X, I5 )
41 FORMAT ( 5X, 3I5, F10.3 )
51 FORMAT ( 5X, 3I5, 10X, 3(F10.3 )
61 FORMAT ( 5X, 2( 2I5, 5X ), 3(F10.3 )
81 FORMAT ( 5X, 3I5, 2(10.3, 2I5, 2(10.3 )

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82 FORMAT ( 30X, 10F5.0 )
91 FORMAT ( 2( 6X, A4), 2(5X, I5 )
100 FORMAT ( ///3EH TABLE 1 - PROGRAM-CONTROL DATA
1 / 43X, 20H TABLE NUMBER
2 / 43X, 40H 2 3 4 5 6 7 8 9
3 // 41H HOLD FROM PRECEDING PROBLEM (1=HOLD); 2X
4 RTS
5 / 32H NUM CARDS INPUT THIS PROBLEM, 10X, 8I5 )
200 FORMAT ( ///24H TABLE 2 - CONSTANTS / )
201 FORMAT ( 32H NUM OF BEAM INCREMENTS, 43X, I5,
1 / 32H BEAM INCREMENT LENGTH, 39X, F10.3,
2 / 32H NUM OF TIME INCREMENTS, 43X, I5,
3 / 32H TIME INCREMENT LENGTH, 38X, F10.3,
4 / 35H OPTIONAL PRINTING SWITCH, 40X, I5,
5 / 41H NUM OF MONITOR STA ( PRINTING ), 34X, I5,
6 / 40H ITERATION SWITCH (0=LINEAR), 35X, I5,
7 / 42H NUM OF MONITOR STA ( ITERATION ), 33X, I5,
A / 54H PLOTTING METHOD (1=MIC, 0=PAPER) 25JAI
R *2)X, I5, LINE OR POINTS PLOT OPTION, 39X, I5 ) 30DE0
P / 34H -----COMPLETE PRINTING TIME INTERVAL AND MONITC 25JAI
202 FORMAT ( / 2PH STATIONS FOR PRINTING----- 25JAI
1 / 45H TIME STA INTERVAL OF COMPLETE PRINT, 30X, I5, 055E0
4 / 52H MONITOR STATIONS (PRINTING) 04JAI
5 / 54H 1 2 3 4 5 6 7 8 04JAI
6 6H9 10, / 5X, 5HSTA J, 10I5 ) 04JAI
203 FORMAT ( / 2PH -----ITERATION DATA FOR NONLINEAR SUP. CV.----- 25JAI
1 / 35H MAXIMUM ITERATION NUMBER, 40X, I5, 04JAI
2 / 40H MAXIMUM ALLOWABLE REFLECTION, 30X, F10.3, 04JAI
3 / 38H DEFLECTION CLOSURE TOLERANCE, 34X, F10.3, 04JAI
4 / 10X, 27HMONITOR STATIONS (ITERATION), 04JAI
5 / 10X, 25H 1 2 3 4 5, 04JAI
6 / 5X, 5HSTA J, 5I5 ) 04JAI
300 FORMAT ( ///47H TABLE 3 - SPECIFIED DEFLECTIONS AND SLOPES 20JAI
1 // 5X, 4PH STA CASE DEFLECTION SLOPE 175E0
2 7X, 22HTPS (IF=1, INITIAL, 26JUI
3 / 6X, 22H (IF=2, PERMANENT ) 26JUI
311 FORMAT ( ' 10X, 13, 7X, 12, 4X, E10.3, 9X, 4HNONE, 5X, I5 ) 175E0
312 FORMAT ( ' 10X, 13, 7X, 12, 11X, 4HNONE, 8X, E10.3, 5X, I5 ) 175E0
313 FORMAT ( ' 10X, 13, 7X, 12, 3X, 2(5X, E10.3), 5X, I5 ) 175E0
400 FORMAT ( ///48H TABLE 4 - STIFFNESS AND FIXED-LOAD DATA 230C4
1 // 51H FROM TO CONTD F CP / QF 055E0
2 2PH I F CP / ) 055E0
411 FORMAT ( ' 5X, 2I4, 13, 1X, 6E11.3 ) 27W04
412 FORMAT ( ' 5X, 14, 4X, 13, 1X, 6E11.3 ) 23W04
413 FORMAT ( ' 9X, 14, 13, 1X, 6E11.3 ) 23W04
500 FORMAT ( ///48H TABLE 5 - MASS AND DAMPING DATA 055E0
1 // 45H FROM TO CONTD RHO DI DE //) 055E0
AND FORMAT ( ///42H TABLE 6 - TIME DEPENDENT AXIAL THRUST 055E0
1 // 5X, 4PH BEAM STA TIME STA 055E0
2 / 4X, 4PH FROM TO FROM TO IT) 055E0
3 2PH IT2 IT3 / ) 055E0
610 FORMAT ( 10X, 2I4, 7X, 2I4, 10X, 3E11.3 ) 055E0
700 FORMAT ( /// 40H TABLE 7 - TIME DEPENDENT LOADING 055E0

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1 //5X, 30H BEAM STA TIME STA 05CE0
2 //5X, 40H FROM TO FROM TO 04CE0
3 20H QT2 QT3 / 05CE0
800 FORMAT (///40H TABLE 8 = NONLINEAR SUPPORT CURVES // 24DF0
1 51H FROM TO CONTD Q-MULTIPLIER =MULTIPLIER PCINP24DF0
2 34HTS SYM OPT W-TOLERANCE C-TOLERANCE / ) 15FE1
811 FORMAT ( 5X, 31F, 2E13,3, 217, 2E12,3 ) 15FE1
812 FORMAT ( 5X, 15, 5X, 15, 2F13,3, 217, 2E12,3 ) 15FE1
813 FORMAT ( 10X, 21F, 2E13,3, 217, 2E12,3 ) 15FE1
814 FORMAT ( 10H 0 10F7,0 ) 24DF0
815 FORMAT ( 10H W 10F7,0 ) 24DF0
820 FORMAT ( // 3FH TABLE 10= CALCULATED RESULTS / 04JA1
1 3FH BEAM AXIS, K+TIME AXIS / ) 05SE0
900 FORMAT ( /// 32H TABLE 9 = PLOTTING SWITCHES 30DE0
1 /// 5X,40H HORIZONTAL VERTICAL BEAM OR TIME MULTIPLE PLOT * 10FE1
2 // 5X,40H AXIS AXIS STATION SWITCH * 10FE1
3 27H(IF=1,SUPERIMPOSE WITH NEXT // ) 24JUI
4 53X,27H IF=0,PLOT ALL SAVED PLOTS ) 24JUI
901 FORMAT ( // 40H STA J DIST DEFL SLOPE 05SE0
1 36H MOM SHEAR SUP REACT // ) 05SE0
902 FORMAT ( // 31H ***** STATIC RESULTS ***** // ) 30DF0
903 FORMAT ( // 25H *NONE* ) 05FE0
904 FORMAT ( /// 40H TOO MUCH DATA FOR AVAILAHLE STORAGE // ) 05SE0
905 FORMAT ( 46H USING DATA FROM THE PREVIOUS PROBLEM ) 05FE4*
907 FORMAT ( // 52H ERROR STOP -- SLOPES AND DEFLECTIONS IMPROPERLY 31DE4
1 10H SPECIFIED ) 31DE4
908 FORMAT ( 9X, 44, 8X, A4, 5X, 15, 8X, 15 ) 10FE1
910 FORMAT ( 43H ADDITIONAL DATA FOR THIS PROBLEM ) 31DE4
911 FORMAT ( 55H ***** PLOT OF DEFLECTION VS TIME FOR BEAM STATIONS 05SE0
1 14H OF,, 6H ***** // ) 05SE0
912 FORMAT ( 10X, 12, 8H CURVE ( A1, 4H ) = , 13 ) 05SE0
913 FORMAT ( /// 5X,10HPT,NC *VALUE*/ ) 05FE0
915 FORMAT ( // 32H ***** DYNAMIC RESULTS ***** // ) 05SE0
916 FORMAT ( // 30X, 13HTIME STA K = , 19, / ) 05FE0
917 FORMAT ( // 40H ERROR STOP -- STATIONS NOT IN ORDER ) 05SE0
919 FORMAT ( 55H ***** PLOT OF BEND, MOM, VS TIME FOR BEAM STATIONS 10FE1
1 14H OF,, 4H ***** // ) 05CE0
921 FORMAT ( 5X, 14, 2X, 2F12,3, 10X, F12,3, 10X, F12,3 ) 05SE0
922 FORMAT ( 34X, F12,3, 10X, F12,3 ) 05SE0
923 FORMAT ( // 35X, 25HMCATOR STATIONS, /, 20X, 3MJ ** 28DE0
1 51 TS, 7X ) ) 28DE0
924 FORMAT ( // 10X, 20F00, 12, 21H ITERATION, THERE ARE, 14* 28DF0
1 21H STATIONS NOT CLOSED ) 28DF0
925 FORMAT ( 20X, 5F12,3 ) 28DF0
929 FORMAT ( 54H ***** PLOT OF DEFLECTIONS ALONG BEAM AXIS AT TIME 05SE0
1 13H STATIONS OF,40H ***** // ) 05FE0
939 FORMAT ( 54H ***** PLOT OF BEND, MOM, ALONG BEAM AXIS AT TIME 10FE1
1 13H STATIONS OF,40H ***** // ) 05FE0
960 FORMAT ( /49X, 31H*****ERROR STOP IN TABLE ***** // 05SE0
1 50H *EITHER SPECIFYING ONE BEAM STA IN THE SAME 05SE0
2 38HAD OF BEAM STA NOT IN ASCENDING ORDER, / ) 05SE0
970 FORMAT ( 5X, 35HPRINTING RESULTS IS NOT REQUESTED, / ! 05SE0
971 FORMAT ( 5X, 29HMICROFILM PLOT IS REQUESTED, / ) 17SE0
972 FORMAT ( 5X, 50HSTANDARD 12 INCH HALLPOINT PAPER PLOT IS REQUFSTE17SE0

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1 240, / ) 17FE0
973 FORMAT ( 5X, 27HPRINTER PLOT IS REQUESTED, / ) 17FE0
980 FORMAT ( ///50H UNDESIGNATED ERROR STOP 105SE0
990 FORMAT ( //5X, 44H***** TIME INCREMENTS SPECIFIED EXCEEDS LIMIT , 05SE0
1 54***** // ) 05SE0
9976 FORMAT ( 5X, 50HERROR INPUT IN TABLE 9, MORE THAN 5 CURVES TO BE 05FE1
1 25H(LOTTED IN ONE FRAME BY PRINTER PLOT ) 15FE1
9977 FORMAT ( 14X, 49H* USED ADJUSTED LOAD-DEFL CURVE TO CALCULATE DEFL 15FE1
1 9DEFLECTIONS * ) 15FE1
9978 FORMAT ( // 44H NCVB IS NOT ZERO WHEN ITSM SWITCH IS ZERO ) 29DE0
9979 FORMAT ( // 46H ITSM SWITCH IS NOT ZERO WHEN NCVA IS ZERO ) 29DF0
9981 FORMAT ( 54H *****SUPPORT CURVE DATA IMPROPERLY SPECIFIED***** 24DE0
1 ) 24DE0
9982 FORMAT ( // 5X, 65HMAXIMUM ITERATION NUMBER IMPROPERLY SPECIFIED 12DF0
9983 FORMAT ( // 44H CALCULATED DISPLACEMENTS EXCEED ALLOWABLE 29DE0
1 2H AT, 13, 24H STATIONS AT TIME STA = , 14 ) 15FE1
9984 FORMAT ( // 44H CALCULATED DISPLACEMENTS EXCEED LIMITS OF 28DE0
1 36H SUPPORT CURVES DURING 1ST CYCLE OF , / 15FE1
2 18H ITERATION NO., 13, 15H AT TIME STA = , 14 ) 15FE1
9985 FORMAT ( // 46H ***** ERROR OR DIAGNOSTIC ***** ) 29DE0
9986 FORMAT ( // 47H SOLUTION DID NOT CLOSE IN SPECIFIED NUMBER 28DE0
1 34H OF ITERATIONS FOR THE FIRST CYCLE ) 15FE1
9987 FORMAT ( // 47H SOLUTION DID NOT CLOSE IN SPECIFIED NUMBER 15FE1
1 35H OF ITERATIONS FOR THE SECOND CYCLE ) 15FE1
9988 FORMAT ( 5X, 50H*****ERROR STOP DUE TO THE SLOPES OF THE NONLINEAR 21AD1
1 10H CV AT STA, 14, 28H IS NOT IN PROPER ORDER***** ) 21AD1
9993 FORMAT ( 5X, 50H*****ERROR INPUT IN TABLE 8, STATION NO. IN ASCEND 15FE1
1 14HTING ORDER***** ) 15FE1
9994 FORMAT ( // 44H CALCULATED DISPLACEMENTS EXCEED LIMITS OF 28DE0
1 35H SUPPORT CURVES DURING 2ND CYCLE OF , / 15FE1
2 18H ITERATION NO., 13, 15H AT TIME STA = , 14 ) 15FE1
9995 FORMAT ( // 30H ***** ERROR IN TABLE 8 INPUT *****// 05SE0
1 5X, 12, 39H CARDS ARE SPECIFYING SAME BEAM STATION, 16, 05SE0
2 21H FOR ONE TIME STATION ) 05SE0
9996 FORMAT ( // 39H ***** ERROR IN TABLE 8 INPUT *****// 05SE0
1 5X, 12, 30H CARDS ARE SPECIFYING SAME TIME STATION, 15, 05DE0
2 28H FOR PLOT ALONG BEAM AXIS ) 05DE0
9998 FORMAT ( 5X, 27H***** ERROR IN NPS OPTION ***** // ) 05FE0
C-----START EXECUTION OF PROGRAM SEE GENERAL FLOW DIAGRAM 05FE0
ITEST = 5H 05FE0
TEST1 = 4HTIME 05SE0
TEST2 = 4HBFAM 05DE0
TEST3 = 4HDEFL 10FE1
TEST4 = 4HMOMT 10FE1
WIDTH = 60 05SE0
SMALL = 0.0 05FE0
RIG = 0.0 05SE0
NCT2 = 0 05FE0
NCT4 = 0 05SE0
NCT5 = 0 05SE0
NCT6 = 0 05SE0
NCT7 = 0 05FE0
NCVB = 0 05JAI
NCT9 = 0 04JAI

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1897 CONTINUE 15FF1
      NDC(N) = NOS 15FF1
GO TO 1884 21AP1
1879 NPCT = 0 15FF1
      NPCT1 = NPCT - 1 15FF1
DO 1898 NP = 1, NPCT1 15FF1
      IF ( ABS(WPT(N,NP+1) - WPT(N,NP) ) .NE. ( ABS(WPT(N,NP+1)) * 15FF1
      ABS(WPT(N,NP)) ) ) GO TO 1898 15FF1
      NPZFC = NP 15FF1
      NPCT = NPCT1 + 1 15FF1
1898 CONTINUE 15FF1
      IF ( NPCT .LE. 2 ) GO TO 1885 15FF1
      NOS = NPZFC 15FF1
      NOS1 = NOS - 1 15FF1
DO 1899 NP = 1, NOS1 15FF1
      QDRCP(N,NP) = - ( QPT(N,NOS-NP+1) - QPT(N,NOS-NP) ) 15FF1
      WDRCP(N,NP) = WPT(N,NOS-NP+1) - WPT(N,NOS-NP) 15FF1
      SLOPE(N,NP) = QDRCP(N,NP) / WDRCP(N,NP) 15FF1
1899 CONTINUE 15FF1
      NOSJ(N) = NOS1 15FF1
      IF ( SLOPE(N,NOS1) .LE. 1.0E-06 ) GO TO 1864 15FF1
      SLOPE(N,NOS) = 0.0 15FF1
      WDRCP(N,NOS) = WPT(N,NOS+1) - WPT(N,NOS) 15FF1
      QDRCP(N,NOS) = 0.0 15FF1
      NOSJ(N) = NOS 21AP1
GO TO 1884 15FF1
      WDRCP(N,NOS) = WPT(N,NOS+1) - WPT(N,NOS) + WDRCP(N,NOS1) 15FF1
C-----CHECK THE PROPER ORDER OF SLOPES FOR THIS NONLINEAR SUP CURVE 21AP1
1886 NOSN = NOSJ(N) - 1 21AP1
DO 1887 NN = 1, NOSN 21AP1
      IF ( SLOPE(N,NN) .LE. 1.0E-05 ) GO TO 1887 21AP1
      IF ( SLOPE(N,NN) .GT. SLOPE(N,NN+1) ) GO TO 1887 21AP1
      JJ = ISTAN(N) - 4 21AP1
      PRINT 999R, JJ 21AP1
      GO TO 9999 21AP1
1887 CONTINUE 21AP1
C-----REVERSE THE ORDER OF POINTS ON POSITIVE ONE-WAY SUPPORTS 2AJU1
      IF ( KSYMJ(N) .NE. -1 ) GO TO 1876 2AJU1
DO 1888 NP = 1, NPCT 2AJU1
      NPR = NPCT - NP + 1 2AJU1
      QPVT(N,NP) = - QPT(N,NP) 2AJU1
      WPVT(N,NP) = - WPT(N,NP) 2AJU1
1888 CONTINUE 2AJU1
DO 1889 NP = 1, NPCT 2AJU1
      QPT(N,NP) = QPVT(N,NP) 2AJU1
      WPT(N,NP) = WPVT(N,NP) 2AJU1
1889 CONTINUE 2AJU1
1876 CONTINUE 2AJU1
C-----INPUT TABLE 9 3ND00
1900 PRINT 900 3ND00
      IF ( AFFF9 ) 9980, 1901, 1910 3ND00
1901 NC19 = 1 3ND00
      NCT5 = ACPV 3ND00
      GO TO 1930 3ND00

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1010 PRINT 905 30DE0
      IF ( ACT9 = 1 ) 1926, 1911, 1911 30DE0
1911 DO 1925 N = 1, ACP9 30DE0
      PRINT 908, YAXIS(N), YAXIS(N), IYSW(N), MPS(N) 10FE1
1925 CONTINUE 30DE0
1926 CONTINUE 30DE0
      PRINT 910 30DE0
      NC19 = ACP9 * 1 30DE0
      NCT9 = ACP9 * NCD9 30DE0
1930 IF ( ACP9 = 10 ) 1935, 1935, 1933 05JA1
1933 PRINT 904 20DE0
      GO TO 9999 30FE0
1935 IF ( ACP9 ) 1990, 1937, 1940 30DE0
1937 PRINT 903 30DE0
      GO TO 2000 30DE0
1940 IF ( ACP9 = ACP9 ) 2000, 1945, 1945 30DE0
1945 DO 1970 N = NC19, ACP9 30DE0
      READ 91, YAXIS(N), YAXIS(N), IYSW(N), MPS(N) 10FE1
      PRINT 909, YAXIS(N), YAXIS(N), IYSW(N), MPS(N) 10FE1
1970 CONTINUE 30DE0
C-----INTERPOLATE AND DISTRIBUTE VALUES FROM TABLE 4 AND TABLE 5 05SE0
2000 LSW = 0 30DE0
      CALL INTERP3 ( NP7, ACP4, IN14, IN24, KR24, F02, F, LSW, KSW4 ) 100C0
      CALL INTERP3 ( NP7, ACP4, IN14, IN24, KR24, QCN2, Q, LSW, KSW4 ) 100C0
      CALL INTERP3 ( NP7, ACP4, IN14, IN24, KR24, SN2, S, LSW, KSW4 ) 100C0
      CALL INTERP3 ( NP7, ACP4, IN14, IN24, KR24, RN2, R, LSW, KSW4 ) 100C0
      CALL INTERP3 ( NP7, ACP4, IN14, IN24, KR24, CN2, CP, LSW, KSW4 ) 100C0
      CALL INTERP3 ( NP7, ACP5, IN15, IN25, KR25, RHON2, RHO, LSW, KSW5 ) 100C0
      CALL INTERP3 ( NP7, ACP5, IN15, IN25, KR25, DIN2, DI, LSW, KSW5 ) 100C0
      CALL INTERP3 ( NP7, ACP5, IN15, IN25, KR25, DEN2, DE, LSW, KSW5 ) 100C0
      LSW = 1 05SE0
      CALL INTERP3 ( NP7, ACP4, IN14, IN24, KR24, IN2, I, LSW, KSW4 ) 100C0
C-----START BRANCHLINE SOLUTION FOR EACH TIME STATION 05SE0
      PRINT 11 05SE0
      PRINT 1 05SE0
      PRINT 13, ( ANJ(N), N = 1, 32 ) 05SE0
      PRINT 14, WDRCP, ( AN2(N), N = 1, 14 ) 05SE0
      PRINT *21 05SE0
      KPC = 0 05SE0
      KK = 0 05SE0
      NPRT = 0 05SE0
C-----INITIALIZING 05SE0
DO 6997 I = 1, NP7 30DE0
      IFUR(I) = 0 15FE1
      OI(I) = 0.0 25JA1
      SS(I) = 0.0 15FE1
      SSM1(I) = 0.0 21AP1
      DIM1(I) = 0.0 21AP1
      DIM2(I) = 0.0 21AP1
6997 CONTINUE 30DE0
      DO 7000 N = 1, NPA4 3ND00
      NDC1 = 0 10FE1
      NITS = 1 15AP1
      IOCAT = 0 15FE1

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IF ( NP .LT. NPCT ) GO TO 6198
SLOPE21 = 0.0
QI21 = 0.0
6198 GO TO 6112
SLOPE21 = ABS ( ( QPT(INN,NP+1) - QPT(INN,NP) ) /
1 ( WPT(INN,NP+1) - WPT(INN,NP) ) )
IF ( SLOPE21 .LE. 1.0E-06 ) SLOPE21 = 0.0
QI21 = QPT(INN,NP) + SLOPE21 * WPT(INN,NP)
IF ( NP .GT. 1 ) GO TO 6112
SLOPE21 = SLOPE21 * 2.0
QI21 = QI21 * 2.0
SLOPE22 = 0.0
QI22 = 0.0
6112 GO TO 6113
SLOPE22 = ABS ( ( QPT(INN,NP) - QPT(INN,NP-1) ) /
1 ( WPT(INN,NP) - WPT(INN,NP-1) ) )
IF ( SLOPE22 .LE. 1.0E-06 ) SLOPE22 = 0.0
QI22 = QPT(INN,NP-1) + SLOPE22 * WPT(INN,NP-1)
IF ( NP .LT. NPCT ) GO TO 6113
SLOPE22 = SLOPE22 * 2.0
QI22 = QI22 * 2.0
6113 SS(J) = SS(J) + 0.5 * ( SLOPE21 + SLOPE22 ) * ESM
QI(J) = QI(J) + 0.5 * ( QI21 + QI22 ) * ESM
GO TO 6056
6110 IF ( IOSW(J) .EQ. 0 ) GO TO 6197
GO TO 6196
C-----TANGENT MODULUS METHOD ( LOAD-SPRING ITERATION METHOD )
6197 SLOPE2 = ABS ( ( QPT(INN,NP) - QPT(INN,NP-1) ) /
1 ( WPT(INN,NP) - WPT(INN,NP-1) ) )
IF ( SLOPE2 .LE. 1.0E-06 ) SLOPE2 = 0.0
SS(J) = SS(J) + SLOPE2 * ESM
QI(J) = QI(J) + ( QPT(INN,NP) + SLOPE2 * WPT(INN,NP) ) *
1 FSM
GO TO 6056
C-----LOAD ITERATION METHOD
6196 DO 6296 NL = 2, NPCT
26JUI STEMP = ABS ( ( QPT(INN,NL) - QPT(INN,NL-1) ) /
1 ( WPT(INN,NL) - WPT(INN,NL-1) ) )
IF ( ABS ( STEMP - SLOPE(INN,NL) ) .LE. 1.0E-06 ) GO TO 6297
6296 CONTINUE
PRINT 990
GO TO 9999
6297 IF ( KSYM(JNK) .EQ. 1 ) GO TO 6298
C-----SFT APPROXIMATE LINEAR SPRING CONSTANT FOR ONE-WAY SUP CV
IF ( ABS(QPT(INN,NL)) .LE. 1.0E-12 ) GO TO 6310
S1 = ABS ( QPT(INN,NL-1) / WPT(INN,NL-1) )
GO TO 6299
6310 S1 = ABS ( QPT(INN,NL) / WPT(INN,NL) )
GO TO 6299
C-----SFT APPROXIMATE LINEAR SPRING CONSTANT FOR SYMMETRIC SUP CV
6298 SI = SLOPE(INN,NL)
IF ( IFUS(J) .EQ. 0 ) GO TO 6299
S1 = ABS ( ( QPT(INN,NL-1) - QPT(INN,NL+1) ) /
1 ( WPT(INN,NL-1) - WPT(INN,NL+1) ) )
21AP1 6299 SS(J) = SS(J) + S1 * FSM
26API WC = - W(J,KK) * WPT(INN,NP)
21AP1 S1 = ABS ( ( QPT(INN,NP) - QPT(INN,NP-1) ) /
21AP1 ( WPT(INN,NP) - WPT(INN,NP-1) ) )
21AP1 1 IF ( S1 .LE. 1.0E-06 ) S1 = 0.0
21AP1 QC = QPT(INN,NP) + WC * S1
21AP1 QS = - W(J,KK) * S1
10FF1 QI(J) = QI(J) + ( QC - WS ) * ESM
10FE1
6056 CONTINUE
10FE1 6504 IF ( ITCS .EQ. 0 ) GO TO 6505
C-----TFST TO SEE IF THERE IS A STA OF SUPPORT UNLOADED
DO 6506 J = 4, NPA
10FE1 IF ( IUS(J) .EQ. 1 ) GO TO 6503
10FE1 6506 CONTINUE
21AP1 GO TO 6175
21AP1 6503 ITCS = 0
21AP1 ITCSP = 1
10FE1 6505 NS = 1
21AP1 A(1) = 0
21AP1 A(2) = 0
21AP1 R(1) = 0
21AP1 R(2) = 0
15FE1 C(1) = 0
15FE1 C(2) = 0
DO 6060 J = 3, MPS
06AP1 C-----COMPUTE MATRIX COEFFS AT EACH STA J FOR FIRST AND SECOND TIME STA
06AP1 YA = F(J-1) * .25 * W * R(J-1)
21AP1 YB = -2.0 * ( F(J-1) * F(J) ) - HE2 * T(J)
21AP1 YC = F(J-1) * 4.0 * F(J) + F(J+1) * HE3 * SS(J) +
1 .25 * H * ( R(J-1) + R(J+1) ) * HE2 * ( T(J) +
2 T(J+1) )
YD = -2.0 * ( F(J) + F(J+1) ) - HE2 * T(J+1)
YE = F(J+1) * .25 * W * R(J+1)
YF = HE3 * C1(J) + 0.5 * HE2 * ( CP(J-1) - CP(J+1) )
IF ( K = 3 ) 7004, 7005, 7005
06AP1 AA = YA
06AP1 RR = YR
26JUI CC = YC
26JUI DD = YD
26JUI EE = YE
26JUI FF = YF
GO TO 700A
7005 AA = YA + DI(J-1) / HT
26JUI RR = YR + 2.0 * ( DI(J-1) + DI(J) ) / HT + 0.5 * HE2 *
1 ( TT(J,2) + TT(J,1) )
26JUI CC = YC + ( DI(J+1) + 4.0 * DI(J) + DI(J+1) ) / HT + 0.5
1 HE2 * ( TT(J,2) + TT(J,1) + TT(J+1,2) + TT(J+1,1) )
26JUI DD = YD + 2.0 * H3T2 * RHO(J) * H3T1 * DE(J)
26JUI EE = HE3 * ( - SS(J) + SSM2(J) )
26JUI DD = YD + 2.0 * ( DI(J) + DI(J+1) ) / HT + 0.5 * HE2 *
1 ( TT(J+1,2) + TT(J+1,1) )
26JUI FF = YF + DI(J+1) / HT
26JUI FF = 2.0 * YF + 4.0 * H3T2 * RHO(J) * W(J,KK-1) * ( -AA
1 2.0 * DI(J-1) / HT ) * W(J-2,KK-2) + ( -RR + 4.0 * (100C

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2      DI(J-1) * DI(J) / HT ) * W(J-1, KK-2) * ( -CC + 2.0100C0
3      * ( DI(J-1) * 4.0 * DI(J) * DI(J+1) / HT * 2.0 * 100C0
4      H3T1 * CE(J) ) * W(J, KK-2) * ( -DD - 4.0 * ( DI(J) * 100C0
5      DI(J+1) / HT ) * W(J+1, KK-2) * ( -EE * 2.0 * DI(J) * 100C0
6      1) / HT ) * W(J+2, KK-2) * HE3 * ( -GI(J) * DIM2(J) ) 21AP1
C-----COMPUTE REACTION OR CONTINUITY COEFFS AT EACH STA 05SE0
7006      E = AA * R(J-2) * RR 05SE0
      DENOM = E * B(J-1) * AA * C(J-2) + CC 05SE0
      IF ( DENOM = 0 ) 6010, 6005, 6010 05SE0
C-----NOTE IF DENOM IS ZERO, BEAM DOES NOT EXIST, D = 0 SETS UDFL = 0. 05SE0
6005      D = 0.0 05SE0
      GO TO 6015 05SE0
6010      D = - 1.0 / DENOM 05SE0
6015      C(J) = D * EE 05SE0
      R(J) = D * ( E * C(J-1) + DD ) 05SE0
      A(J) = D * ( F * A(J-1) + AA * A(J-2) + FF ) 05SE0
C-----CONTROL RESET ROUTINES FOR SPECIFIED CONDITIONS 05SE0
      KEYJ = KEY(J) 05SE0
6018      IF ( K = 3 ) 6019, 6019, 6018 100C0
      IF ( ( JDP(J) , NE, 1 ) ) GO TO 6019 100C0
      NS = NS + 1 100C0
      GO TO 6040 100C0
6019      GO TO ( 6040, 6020, 6030, 6020, 6050 ) , KEYJ 05SE0
C-----RESET FOR SPECIFIED DEFLECTION 05SE0
6020      C(J) = 0.0 05SE0
      B(J) = 0.0 05SE0
      A(J) = WS(NS) 05SE0
      IF ( KEYJ = 7 ) 6059, 6030, 6060 05SE0
C-----RESET FOR SPECIFIED SLOPE AT NEXT STA 05SE0
6030      DTMP = 0 05SE0
      CTMP = C(J) 05SE0
      RTMP = B(J) 05SE0
      ATMP = A(J) 05SE0
      C(J) = 1.0 05SE0
      R(J) = 0.0 05SE0
      A(J) = - HT2 * DWS(NS) 05SE0
      GO TO 6040 05SE0
C-----RESET FOR SPECIFIED SLOPE AT PRECEDING STATION 05SE0
6050      DRFV = 1.0 / ( 1.0 - ( BTEMP * B(J-1) + CTEMP * 1.0 ) * 05SE0
      1      D / DTEMP ) 05SE0
      CREV = DREV * C(J) 05SE0
      RREV = DREV * ( B(J) + ( BTEMP * C(J-1) ) * D / DTEMP ) 05SE0
      AREV = DREV * ( A(J) + ( HT2 * DWS(NS) + ATMP * RTMP 05SE0
      1      * A(J-1) ) * D / DTEMP ) 05SE0
      C(J) = CREV 05SE0
      R(J) = RREV 05SE0
      A(J) = AREV 05SE0
      NS = NS + 1 05SE0
6059      CONTINUE 05SE0
6060      CONTINUE 05SE0
C-----COMPUTE DEFLECTIONS 05SE0
DO 6100 J = 3, MP5 05SE0
      L = M * R - J 05SE0
      WWW(L) = WW(L) 13JA1
      WW(L) = W(L, KK) 30DE0

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      W(L, KK) = A(L) * R(L) * W(L+1, KK) + C(L) * W(L+2, KK) 05SE0
6100      CONTINUE 05SE0
      W(2, KK) = 2.0 * W(3, KK) - W(4, KK) 05SE0
      W(MP4, KK) = 2.0 * W(MP5, KK) - W(MP4, KK) 05SE0
C-----COUNT NUMBER OF STATIONS EXCEEDING MAXIMUM ALLOWABLE DEFLECTIONS 2APDE0
C----- AND NUMBER OF STATIONS NOT CLOSED 2APDE0
      KCNT(INIT) = 0 15FE1
      KMAX = 0 2PDE0
      IF ( NCVP , FG, 0 ) GO TO 6000 2PDE0
      DO 6150 J = 4, MP4 2APDE0
      IF ( WMAX = ABS(W(J, KK)) ) 6155, 6160, 6160 2PDE0
      KMAX = KMAX + 1 2APDE0
6155      IF ( ( INCNT , RT, 0 ) ) GO TO 6150 15FF1
6160      IF ( ABS(W(J, KK) - WW(J) ) - WTOL ) 6150, 6150, 6165 15FE1
6165      KCNT(INIT) = KCNT(INIT) + 1 05JA1
6150      CONTINUE 2PDE0
      IF ( K , LT, 3 ) GO TO 6541 21AP1
      IF ( ( NCCT , EQ, 1 ) ) GO TO 6541 21AP1
C-----TEST TO SEE IF THERE IS A SUP STA WHERE THE DEFLECTION IS OSCILLATING 21AP1
C-----DURING THE ITERATION PROCESS 21AP1
      DO 6540 NN = 1, NNC 06AP1
      J = ISTAT(NN) 06AP1
      IF ( ABS(W(J, KK) - WWW(J) ) , GT, 1.0E-08 ) GO TO 6540 21AP1
      IF ( ABS(W(J, KK)) , LE, WTOL ) GO TO 6540 21AP1
      IF ( ABS(W(J, KK) - WW(J) ) , LE, WTOL ) GO TO 6540 21AP1
      NUMOC(J) = NUMOC(J) + 1 21AP1
      IF ( NUMOC(J) = 2 ) 6540, 6164, 6164 06AP1
6164      NITS = NIT + 1 21AP1
6540      CONTINUE 06AP1
      IF ( NITS , LT, 2 ) GO TO 6541 21AP1
C-----SETTING SWITCH IOSW(J) FOR LOAD ITERATION METHOD NEXT ITERATION 21AP1
      DO 6542 NN = 1, NNC 21AP1
      J = ISTAT(NN) 21AP1
      IOSW(J) = 1 21AP1
      NUMOC(J) = 0 21AP1
6542      CONTINUE 21AP1
6541      IF ( ( INCNT , FG, 0 ) ) GO TO 6151 15FE1
C-----TEST TO SEE IF THE DEFLECTIONS COMPUTED FROM THE INITIAL SPRINGS 21AP1
C-----ITERATION LOADS ARE WITHIN THE ADJUSTED RANGE OF THE Q-W CURVE 21AP1
      DO 6152 N = 1, NNC 15FE1
      J = ISTAT(N) 15FE1
      IF ( ( IFUS(J) , EQ, 0 ) ) GO TO 6152 15FE1
      IF ( ( KSYM(J) , EQ, 1 ) ) GO TO 6149 15FE1
      IF ( ( KSYM(J) , EQ, 0 ) ) GO TO 6249 26JU1
      IF ( ( W(J, KK-1) , LE, 0.0 ) ) GO TO 6152 26JU1
      GO TO 6149 26JU1
6249      IF ( ( W(J, KK-1) , GE, 0.0 ) ) GO TO 6152 26JU1
6149      IF ( ( WPT(N+1) ) 6153, 9991, 6154 15FF1
6153      IF ( ( W(J, KK) , GT, W(J, KK-1) ) ) GO TO 6152 15FF1
      W(J, KK) = W(J, KK-1) + 1.0E-06 15FE1
      GO TO 6152 15FE1
6154      IF ( ( W(J, KK) , LT, W(J, KK-1) ) ) GO TO 6152 15FE1
      W(J, KK) = W(J, KK-1) - 1.0E-06 15FF1
6152      CONTINUE 15FF1

```



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      RW(J) = ( -W(J-1, KK-1) * W(J, KK-1) ) / H      05SE0
      WM(J) = F(J) * ( W(J-1, KK-1) - 2.0 * W(J, KK-1) *
      W(J+1, KK-1) ) / HE2      05SE0
1      IF ( K = 3 ) R000, R000, R000      21AP1
8006      AM(J) = AM(J) + D1(J) * ( -W(J-1, KK-2) * 2.0 * W(J, KK-2) ) +
      - W(J-1, KK-2) * W(J-1, KK) - 2.0 * W(J, KK) *
      W(J+1, KK) ) / ( HE2 * 2.0 * MT )      21AP1
H000      CONTINUE      05SE0
      RM(2) = 0.0      05SE0
      RM(MPA) = 0.0      05SE0
      DO R100 J = 3, MPA      05SE0
8105      IF ( K = 3 ) R105, R105, R106      05SE0
      DRM(J) = ( -BM(J-1) * BM(J) ) / H - T(J) * ( -W(J-1, KK-1)
      + W(J, KK-1) ) / H      05SE0
8106      GO TO R107      05SE0
      DRM(J) = ( -RM(J-1) * BM(J) ) / H - ( 0.5 * ( TT(J+1) +
      TT(J+2) ) * T(J) ) * ( -W(J-1, KK-1)
      + W(J, KK-1) ) / H - D1(J+1) * ( -W(J-2, KK-2) +
      2.0 * W(J-1, KK-2) - W(J, KK-2) + W(J+2, KK) - 2.0 *
      + W(J-1, KK) + W(J, KK) ) / ( HE3 * MT * 2.0 )      100C0
      + D1(J) * ( -W(J-1, KK-2) + 2.0 * W(J, KK-2) - W(
      J+1, KK-2) + W(J-1, KK) - 2.0 * W(J, KK) + W(J+1, KK
      ) ) / ( HE3 * MT * 2.0 )      100C0
R107      KEYJ = KEY(J)      05SE0
      IF ( K .EQ. 3 ) GO TO R10A      21AP1
      IF ( IOP(J) .NE. 1 ) GO TO R108      21AP1
      GO TO R140      21AP1
810A      GO TO ( R140, R120, R140, R120, R140 ), KEYJ      21AP1
8120      REAC = ( RM(J-1) - 2.0 * BM(J) + RM(J+1) ) / H      05SE0
      - Q(J) + ( CP(J-1) - CP(J+1) ) / ( 2.0 * H )      04JA1
      - ( R(J-1) * W(J-2, KK-1) - R(J+1) * W(J, KK-1)
      - R(J+1) * W(J, KK-1) + R(J-1) * W(J+2, KK-1) )
      / ( 4.0 * HE2 ) - ( T(J) * W(J-1, KK-1) - T(J)
      * W(J, KK-1) - T(J+1) * W(J, KK-1) + T(J+1) *
      W(J+1, KK-1) ) / H      05SE0
      IF ( K = 3 ) R121, R121, R122      05SE0
M121      GO TO R100      05SE0
8122      REACT(J) = REAC - ( QT(J+1) + QT(J+2) ) * 0.5 * RHO(J) *
      ( W(J, KK-2) - 2.0 * W(J, KK-1) + W(J, KK) ) /
      HTE2 * DE(J) * ( -W(J, KK-2) * W(J, KK) ) / ( 2.0 *
      * MT ) + ( Q1(J+1) * ( -W(J-2, KK-2) + 2.0 *
      W(J-1, KK-2) - W(J, KK-2) + W(J+2, KK) - 2.0 *
      W(J-1, KK) + W(J, KK) ) - 2.0 * D1(J) * (
      -W(J-1, KK-2) + 2.0 * W(J, KK-2) - W(J+1, KK-2) +
      W(J-1, KK) - 2.0 * W(J, KK) + W(J+1, KK) ) +
      D1(J+1) * ( -W(J, KK-2) + 2.0 * W(J+1, KK-2) -
      W(J+2, KK-2) * W(J, KK) - 2.0 * W(J+1, KK) +
      W(J+2, KK) ) / ( HE3 * MT * 2.0 )      100C0
      REACT(J) = REACT(J) - ( 0.5 * ( TT(J+1) + TT(J+2) ) *
      W(J-1, KK-1) - 0.5 * ( TT(J+1) + TT(J+2) ) *
      W(J, KK-1) - 0.5 * ( TT(J+1, 1) + TT(J+1, 2) ) *
      W(J, KK-1) + 0.5 * ( TT(J+1, 1) + TT(J+1, 2) ) *
      W(J+1, KK-1) ) / H      100C0

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      GO TO R100      05SE0
R140      REACT(J) = -SSM(J) * W(J, KK-1) * QS(J)      21AP1
R100      CONTINUE      05SE0
C-----SAVE DEFLECTIONS FOR PLOTS ALONG THE TIME AXIS      15FE1
R101      NPT = 0      29DE0
      DO R1A0 J = 4, MPA      29DE0
      NPPR = 0      15FE1
      LSTA = J - 4      29DE0
      IF ( NCT9 ) 99A0, 6180, 61A5      29DE0
61A5      IF ( K = 2 ) R317, 6190, 6190      06AP1
6190      DO 6195 NCP = 1, NCT9      29DF0
      IF ( XAXIS(NCP) .NE. TEST1 ) GO TO 6195      29DE0
      IF ( YAXIS(NCP) .NE. TEST1 ) GO TO 6195      15FE1
      IF ( TYS(NCP) .NE. LSTA ) GO TO 6195      29DE0
      NPT = NPT + 1      29DE0
      KSAV(NPT) = LSTA      29DE0
      KEYP(NPT) = 1      29DE0
      MPR(NPT) = MFS(NCP)      29DE0
      KYS(NPT) = 1      10FE1
      IF ( K .EQ. MPA ) GO TO 6195      15FE1
      Y(NPT, KK) = W(J, KK)      10FE1
      NEPR = KPRR + 1      10FE1
6195      CONTINUE      29DE0
      IF ( NEPR .LE. 1 ) GO TO R1B0      15FE1
      PRINT 999A, KPRR, LSTA      29DE0
      GO TO 9999      29DE0
61B0      CONTINUE      29DE0
C-----SAVE MOMENTS FOR PLOTS ALONG THE TIME AXIS      15FE1
      DO 62A0 J = 4, MPA      15FE1
      NEPR = 0      15FE1
      LSTA = J - 4      15FE1
      IF ( NCT9 ) 99A0, 62B0, 62A5      15FE1
62B5      DO 62A0 NCP = 1, NCT9      15FE1
      IF ( XAXIS(NCP) .NE. TEST1 ) GO TO 62A0      15FE1
      IF ( YAXIS(NCP) .NE. TEST1 ) GO TO 62A0      15FE1
      IF ( TYS(NCP) .NE. LSTA ) GO TO 62A0      15FE1
      NPT = NPT + 1      15FE1
      KSAV(NPT) = LSTA      15FE1
      KEYP(NPT) = 1      15FE1
      MPR(NPT) = MFS(NCP)      15FE1
      KYS(NPT) = 2      10FE1
      IF ( K .LE. 3 ) GO TO 62A0      15FE1
      Y(NPT, KK) = RM(J)      10FE1
      NEPR = KPRR + 1      15FE1
62A0      CONTINUE      15FE1
      IF ( NEPR .LE. 1 ) GO TO 62A0      15FE1
      PRINT 999A, KPRR, LSTA      15FE1
      GO TO 9999      15FE1
62P0      CONTINUE      15FE1
C-----SAVE DEFLECTIONS FOR PLOTS ALONG THE REAM AXIS      15FE1
      IF ( NCT9 ) 99A0, 62P0, 6205      29DE0
6205      IF ( K = 3 ) 6200, 6210, 6215      29DE0
6210      NPT = NPT      29DE0
6215      NEPR = 0      29DE0

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SUBROUTINE INTERP3 ( MP7, NCT, JN1, JN2, KR2, ZN, Z, LSM, KSW ) 055F0
DIMENSION JN1(100), JN2(100), KR2(100), ZN(100), Z(207), KSW(100) 10FE6
905 FORMAT ( //40H ERROR STOP -- STATIONS NOT IN ORDER ) 14MY5
908 FORMAT ( //43H UNDESIGNATED ERROR STOP IN SUBROUTINE ) 14MY5
909 FORMAT ( //46H ERROR -- NON-ZERO DATA BEYOND END OF BEAM ) 14MY5
DO 1603 J = 1, MP7 055F0
Z(J) = 0.0 264D5
1603 CONTINUE 264D5
ASM = LSM 12A25
ASM = ASM / 2, 09A85
M = MP7 - 7 055F0
KH1 = 0 12JA5
IF ( NCT = 1 ) 1674+1604+1604 285E7
DO 1675 NC = 1, NCT 265E7
IF ( KR1 ) 1609, 1605, 1607 10FF6
1605 NC1 = NC 12JA5
JV = JN1(NC1) + LSM 12A05
IF ( KR2(NC) ) 1698, 1607, 1670 10FF6
IF ( JN2(NC) = M ) 1609, 1609, 1611 24FE6
JS2 = JN2(NC) 10FF6
ZNS = ZN(NC) 21FE6
GO TO 1619 10FF6
1611 JS2 = M 10FE6
ZNS = ZN(NC1) + ( ZN(NC) - ZN(NC1) ) * ( JS2 - JV + LSM ) 10MR6
1 / ( JN2(NC) - JV + LSM ) 10MR6
KASS = KSW(NC) 215E66
GO TO ( 1613, 1615, 1617, 1617 ), KASS 215E66
1613 IF ( JN1(NC) = M ) 1619, 1619, 1670 10FF6
1675 IF ( JN1(NC1) = M ) 1619, 1619, 1670 10FE6
1617 IF ( KSW(NC1) = 2 ) 1618, 1615, 1618 09MR6
1618 IF ( JN2(NC1) = M ) 1619, 1619, 1670 09MR6
1619 J1 = JV + 4 10FE6
J2 = JS2 + 4 10FE6
DENOM = J2 - J1 + LSM 12A05
JINCR = 1 12JA5
ESM = 1.0 12JA5
ISW = 1 12JA5
IF ( DENOM ) 1695, 1620, 1630 12JA5
1620 DENOM = 1.0 12JA5
ISW = 0 12JA5
IF ( J2 - J1 ) 1651, 1630, 1630 08MR6
DO 1650 J = J1, J2, JINCR 12JA5
F = J 09A85
F1 = J1 09A85
DIFF = F - F1 + LSM 09A85
PART = DIFF / DENOM 12JA5
Z(J) = Z(J) + ( ZN(NC1) + PART * ( ZN(NC) - ZN(NC1) ) ) * ESM 10FF6
CONTINUE 12JA5
1651 IF ( LSM ) 1698, 1652, 1660 08MR6
1652 IF ( ISW ) 1698, 1660, 1655 09A85
1655 JINCR = J2 - J1 12JA5
ESM = -0.5 12JA5
ISW = 0 12JA5
GO TO 1630 12JA5

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1660 IF ( KR2(NC) ) 1698, 1670, 1665 12JA5
1665 JV = JN2(NC) + LSM 12A05
1670 KR1 = KR2(NC) 12JA5
NC1 = NC 12JA5
1675 CONTINUE 215FAA
1674 CONTINUE 055F0
MP5 = MP7 - 2 055F0
MP6 = MP7 - 1 055F0
C---TEST FOR DATA ERRONEOUSLY STORED BEYOND ENDS OF REAL BEAM 0A0D8
IF ( ZCA ) 1699, 1676, 1699 055F0
1676 RETURN 14MY5
1675 PRINT 905 14MY5
GO TO 1709 05JA67
1638 PRINT 908 14MY5
GO TO 1709 05JA67
1609 PRINT 909 14MY5
1709 CONTINUE 05JA67
END 12JA5

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SUBROUTINE SPLOT 9 ( NPLT, X, IX, IY, IX )
C * * * THE LATEST REVISION DATA FOR THIS ROUTINE IS - - 11 JUN 70 REVISED
COMMON / SPLOT / WIDTH, SMALL, RIG, SYMR
DIMENSION IX(1), X(1), IY(1), SPACE(42), SYMR(10)
INTEGER WIDTH
DATA SPACE / 62*1H / , SYMR / 1HT / , RLANK / 1H /
C***** THIS ROUTINE SUPERIMPOSES UP TO 10 PLOTS PER FRAME
C THE FIRST PLOT SHOULD BE THE LONGEST (MOST POINTS)
C THE PAPER SHOULD BE POSITIONED PROPERLY AND ALL
C HEADINGS PRINTED BEFORE CALLING.
C **** INPUT -NPLT, THE NUMBER OF PLOTS FOR THIS FRAME
C IX, ARRAY CONTAINING LENGTH OF THE RESPECTIVE PLOTS
C IX(1) = NUMBER OF POINTS IN FIRST PLOT, ETC.
C X, STARTING ADDRESS OF THE FIRST PLOT
C - NOTE - THE PLOTS MUST BE STORED CONTIGUOUSLY
C IX, INDEPENDENT VARIABLE ( INDEXING )
C WIDTH, WIDTH OF PLOT ( LESS THAN 63 )
C SMALL, LOWER LIMIT OF VALUES TO BE CONSIDERED
C RIG, UPPER LIMIT OF VALUES TO BE CONSIDERED
C **** OUTPUT- NO ACTUAL VALUES ARE RETURNED TO THE
C CALLING ROUTINE - THE VALUES ARE PRINTED
C AND PLOTTED VERTICALLY.
C THE VALUES PRINTED ARE THOSE OF THE FIRST PLOT.
C IF WIDTH IS GREATER THAN 0 THE X-AXIS (....) IS PLOTTED
C WITH THE APPROPRIATE ADJUSTMENTS IN SCALE MADE, IF NECESSARY
C IF WIDTH IS LESS THAN 0 THE X-AXIS IS NOT PLOTTED AND THE
C ROUTINE FUNCTIONS JUST LIKE SPLOT 7
C **** FRANK L ENDRES - PAX 1A92 ****
12 FORMAT ( 4X, I4, 3X E10.3, 2X 62A1 )
      NEND = 0
      MMAX = 0
      IMSK = 1
      DO 30 I = 1, NPLT
      IF ( MMAX.LT.IX(I) ) MMAX = IX(I)
      NEND = NEND + IX(I)
30 CONTINUE
      IF ( NEND.LE.0 ) GO TO 999
      IWIDE = WIDTH
      IAX = 0
      IF ( IWIDE.LT.0 ) IAX = 1
      IF ( IWIDE.LT.0 ) IWIDE = -IWIDE
      IOTA = IWIDE / 2
      OMEGA = X(1)
      THETA = X(1)
      IF ( NEND.EQ.1 ) GO TO 45
      DO 50 I = 2, NEND
      IF ( OMEGA.LT.X(I) ) OMEGA = X(I)
      IF ( THETA.GT.X(I) ) THETA = X(I)
50 CONTINUE
      IF ( SMALL.GE.RIG ) GO TO 43
      IF ( OMEGA.GT.RIG ) OMEGA = RIG
      IF ( THETA.LT.SMALL ) THETA = SMALL
      IF ( IAX.EQ.1 ) GO TO 45
      IF ( THETA.GT.0. ) THETA = 0.

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      OMEGA = 0.
      GO TO 40
      SIGMA = ( IWIDE - 1 ) / ( OMEGA - THETA )
      IF ( IAX.EQ.1 .OR. NEND.EQ.1 ) GO TO 60
      BETA = 1. - SIGMA * THETA
      IOTA = BETA
      IF ( (BETA - IOTA).GE. .5 ) IOTA = IOTA + 1
      ISKX = IOTA
      IMSK = ISKX
60 DO 100 I = 1, MMAX
      IF ( OMEGA.NE.THETA ) GO TO 45
      IMSK = IOTA
      SPACE(IOTA) = SYMR(I)
      GO TO 88
      IIXX = 0
      IF ( IAX.EQ.1 ) GO TO 70
      SPACE(ISKX) = SYMR
70 DO 85 NP = 1, NPLT
      IF ( NP.GT.1 ) IIXX = IX(NP-1) + IIXX
      IF ( I.GE.IX(NP) ) GO TO 85
      IF ( X(I+IIXX).LT.OMEGA ) GO TO 85
      IF ( X(I+IIXX).GT.OMEGA ) GO TO 85
      BETA = SIGMA * ( X(I+IIXX) - THETA ) + 1.
      IOTA = BETA
      IF ( (BETA - IOTA).GE.0.5 ) IOTA = IOTA + 1
      IF ( IOTA.GT.IMSK ) IMSK = IOTA
      SPACE(IOTA) = SYMR(NP)
85 CONTINUE
88 PRINT 12, IX(I), X(I), ( SPACE(L), L=1, IMSK )
      DO 90 J = 1, 62
90 SPACE(J) = RLANK
100 CONTINUE
990 RETURN
C END SPLOT 9
      END

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SUBROUTINE ZOT 1 ( XF, YF, NP, TD )
C * * * THE LATEST REVISION DATA FOR THIS ROUTINE IS - - 11 JUN 70
COMMON / ZOT / LOP, MC, IROLL, MOP
DIMENSION XF(1), YF(1), ID(1)
DATA INC, IT1, X, Y, XL, YL, X0, Y0 / 1.12, 0., 0., 9., 8., 0., 90. /
DATA IT1, IT2 / -1., 0. /
C   XF - ARRAY CONTAINING THE X - COORDINATES
C   YF - ARRAY CONTAINING THE Y - COORDINATES
C   NP - NUMBER OF POINTS TO BE PLOTTED
C   LOP - LINE OR POINT PLOT OPTION
C         = 0 , LINE PLOT
C         = -J , POINT PLOT AT EVERY J-TH POINT
C         = +J , LINE PLOT WITH A POINT PLOT AT EVERY J-TH PT.
C   ID - VARIABLE OR ARRAY CONTAINING TITLE OF PLOT
C   MC - NUMBER OF CHARACTERS IN TITLE ( 0 IF NO TITLE )
C   IROLL - OPTION TO MOVE TO A NEW FRAME - AFTER THIS PLOT
C          = 1 , SAME FRAME
C          EQUAL TO 0 , NEW FRAME
C          LESS THAN 0 , TERMINATE
C   MOP - MICROFILM OR PAPER PLOT OPTION
C        = 1 , PAPER PLOTS
C        = -1 , MICROFILM
C * * * * FRANK L ENDRES - PAX 1992 * * * *
      NC = MC
      IT1 = IT1 + 1
      IF ( IT1.NE.0 ) GO TO 20
      IF ( MOP.EQ.1 ) CALL RGNPLT
      IF ( MOP.EQ. -1 ) CALL RGNPLT ( ALFILMPL )
20  IF ( NC.NE.0 ) GO TO 50
      NC = 10
      ID = 10M
50  CONTINUE
C - - - POSITION ORIGIN
      IF ( IT2.EQ.0 ) CALL PLT ( 1., 1.5, -3 )
      IF ( IT2.EQ.1 ) GO TO 100
C - - - SCALE X - AXIS
      CALL SCALE ( XF, XL, NP, INC )
C - - - SCALE Y - AXIS
      CALL SCALE ( YF, YL, NP, INC )
C - - - SET UP X-AXIS
      YM = - YF(NP+1) / YF(NP+2)
      CALL AXIS ( X, YM+1H, -1, XL, X0, XF(NP+1), XF(NP+2) )
C - - - SET UP Y-AXIS
      XM = - XF(NP+1) / XF(NP+2)
      CALL AXIS ( XM, Y, 1H, 1, YL, Y0, YF(NP+1), YF(NP+2) )
C - - - PRINT TITLE
      CALL SYMROL ( *1, -x5, .14, TD, X0, MC )
C - - - PLOT THE FUNCTION
100 CALL LINE ( XF, YF, NP, 1, LOP, IT1 )
      IF ( IT2.EQ.1 ) IT2 = 0
      IF ( IROLL.EQ.0 ) CALL PLT ( .0, .0, 999 )
      IF ( IROLL.LT.0 ) CALL ENDPLOT
      IF ( IROLL.GT.0 ) IT2 = 1
      IF ( IROLL.LT.0 ) IT1 = -1

```

RETURN  
END

9JF0  
9JF0

APPENDIX G

LISTING OF DATA FOR EXAMPLE PROBLEMS

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EXAMPLE PROBLEMS FOR DICE PROGRAM BY JACK CHAN APRIL 1971  
DYNAMIC BEAM-COLUMN PROGRAM USING 5-1-5 IMPLICIT OPERATOR  
INELASTICALLY SUPPORTED BEAMS UNDER FREE VIBRATION

1 0 0 0 0 0 0 0 2 0 1 1 0 3 1 1  
1 1.000E+00 800 6.283E-03 0 -1 0 1 1 1  
50 1.000E+01 1.000E-06 1.000E+02 1.000E+00  
1 1 0 1.000E+00  
1 1 0 0 -5.000E+01-5.000E+01-5.000E+01  
1 1 0 800 -1.000E+02-1.000E+02-1.000E+02  
1 1 800 800 -4.000E+01-5.000E+01-5.000E+01  
1 1 0-1.000E+00 1.000E+01 5 1 1.000E+02 1.000E+00  
20 36 60 120 120  
2 4 8 20 100

TIME OFFL 1  
1A FREE VIBRATION OF BEAMS BY SPECIFYING INITIAL DISPLACEMENTS  
0 0 0 0 0 0 0 2 1 0 1 0 0 1 1  
1 1.000E+00 800 6.283E-03 0 -1 0 1 1 1  
50 1.000E+01 1.000E-06 1.000E+00  
1 1 0 1.000E+00  
1 1 0-1.000E+00 1.000E+01 5 1 1.000E+02 1.000E+00  
20 36 60 120 120  
2 4 8 20 100

TIME OFFL 1  
2-1 LATERAL VIBRATION OF A SIMPLY SUPPORTED BEAM WITH AXIAL COMPRESSION F.  
0 0 0 0 0 0 0 0 1 11 1 1 0 0 0 2  
10 1.200E+01 200 3.752E-04 0 0 0 1 1 1  
0 1 0.000E+00 2  
1 1-1.221E+00 1  
2 1-2.324E+00 1  
3 1-3.197E+00 1  
4 1-3.758E+00 1  
5 1-3.952E+00 1  
6 1-3.758E+00 1  
7 1-3.197E+00 1  
8 1-2.324E+00 1  
9 1-1.221E+00 1  
10 1 0.000E+00 2  
0 10 0 1.000E+09 -3.700E+05  
1.000E+01  
0 10 0 5 0  
0 10 0 5 0

TIME OFFL 5  
TIME MOMT 5 0  
2-2 LATERAL VIBRATION OF A BEAM ON ELASTIC FOUNDATION  
0 0 0 1 0 0 0 1 1 11 1 0 0 0 0  
10 1.200E+01 200 1.540E-04 0 0 0 1 1 1  
0 1 0 2  
1 1-2.062E+00 1  
2 1-3.923E+00 1  
3 1-5.398E+00 1  
4 1-6.345E+00 1  
5 1-6.672E+00 1  
6 1-6.345E+00 1  
7 1-5.398E+00 1

8 1-3.923E+00 1  
9 1-2.062E+00 1  
10 1 0 2  
0 10 0 1.000E+09 1.200E+04  
THREE-SPAN BEAM, AN IASHO STANDARD 2-D TRUCK MOVING WITH 60 MPH  
0 0 0 0 0 0 1 2 4 2 0 0 74 0 2  
40 6.000E+01 80 5.444E+02 0 0 0 -1 1 1  
0 1 0.000E+00 2  
40 1 0.000E+00 2  
0 40 0 4.500E+11  
10 30 0 4.500E+11  
10 10 0 2.000E+05  
30 30 0 2.000E+05  
0 40 0 5.944E+00  
10 30 1 -2.902E+00  
1 1 1 1 -7.146E+03-3.145E+03-7.146E+03  
2 2 2 2 -3.133E+03-3.133E+03-3.133E+03  
3 3 3 3 -7.019E+03-3.017E+03-3.019E+03  
0 1 3 3 -7.441E+03-7.441E+03-7.441E+03  
4 4 4 4 -2.608E+03-2.608E+03-2.608E+03  
1 2 4 4 -7.445E+03-7.445E+03-7.445E+03  
5 5 5 5 -7.022E+03-3.022E+03-3.022E+03  
2 3 5 5 -7.487E+03-7.487E+03-7.487E+03  
6 6 6 6 -7.030E+03-3.030E+03-3.030E+03  
3 4 6 6 -7.391E+03-7.391E+03-7.391E+03  
7 7 7 7 -2.417E+03-2.417E+03-2.417E+03  
4 5 7 7 -7.491E+03-7.491E+03-7.491E+03  
8 8 8 8 -3.234E+03-3.234E+03-3.234E+03  
5 4 8 8 -6.407E+03-6.407E+03-6.407E+03  
9 9 9 9 -3.047E+03-3.047E+03-3.047E+03  
6 7 9 9 -7.009E+03-7.009E+03-7.009E+03  
10 10 10 10 -3.056E+03-3.056E+03-3.056E+03  
7 8 10 10 -6.860E+03-6.860E+03-6.860E+03  
11 11 11 11 -3.171E+03-3.171E+03-3.171E+03  
8 9 11 11 -7.686E+03-7.686E+03-7.686E+03  
12 12 12 12 -3.160E+03-3.160E+03-3.160E+03  
9 10 12 12 -7.828E+03-7.828E+03-7.828E+03  
13 13 13 13 -4.144E+03-4.144E+03-4.144E+03  
10 11 13 13 -7.537E+03-7.537E+03-7.537E+03  
14 14 14 14 -2.844E+03-2.844E+03-2.844E+03  
11 12 14 14 -7.244E+03-7.244E+03-7.244E+03  
15 15 15 15 -2.324E+03-2.324E+03-2.324E+03  
12 13 15 15 -7.894E+03-7.894E+03-7.894E+03  
16 16 16 16 -2.499E+03-2.499E+03-2.499E+03  
13 14 16 16 -7.728E+03-7.728E+03-7.728E+03  
17 17 17 17 -2.672E+03-2.672E+03-2.672E+03  
14 15 17 17 -7.100E+03-7.100E+03-7.100E+03  
18 18 18 18 -3.054E+03-3.054E+03-3.054E+03  
15 16 18 18 -6.214E+03-6.214E+03-6.214E+03  
19 19 19 19 -2.445E+03-2.445E+03-2.445E+03  
16 17 19 19 -7.148E+03-7.148E+03-7.148E+03  
20 20 20 20 -3.273E+03-3.273E+03-3.273E+03  
17 18 20 20 -8.766E+03-8.766E+03-8.766E+03  
21 21 21 21 -3.386E+03-3.386E+03-3.386E+03



```

41 3.750E+00      400 5.000E-04      0      0      1 -1
0 41      0 1      0.0      0.0      -4.750E+02
0 41      1 2      -4.750E+02-4.750E+02+6.000E+02
0 41      2 3      -6.000E+02-6.000E+02+8.500E+02
0 41      3 4      -8.500E+02-8.500E+02+8.250E+02
0 41      4 5      -8.250E+02-8.250E+02+7.500E+02
0 41      5 6      -7.500E+02-7.500E+02+6.250E+02
0 41      6 7      -6.250E+02-6.250E+02+5.150E+02
0 41      7 8      -5.150E+02-5.150E+02+3.850E+02
0 41      8 9      -3.850E+02-3.850E+02+2.750E+02
0 41      9 10     -2.750E+02-2.750E+02+1.600E+02
0 41      10 11    -1.600E+02-1.600E+02+7.000E+01
0 41      11 12    -7.000E+01-7.000E+01 0.0
5 PARTIALLY EMBEDDED STEEL PIPE LOADED BY WAVE FORCES
0 0 0 0 0 0 0 0 2 0 7 2 0 32 12 10
31 4.800E+01      400 1.000E-02      0      -1 0 1 1
50 2.000E+01 1.000E-06      3.000E+07      2.100E+10
1 1 0      1.000E+02      1.000E+20      2.100E+10
31 31 0      2.877E+11
0 31 0      -2.000E+04
1 1 1      -1.000E+04
1 31 0      -1.220E+04
1 31 0      -4.924E+04
0 0 0      3.210E+00 1.918E+06
0 0 0      3.210E+01
1 11      1 10     -3.090E+02 0.      -3.090E+03
1 11      10 20    -3.090E+03 0.      -6.180E+03
1 11      20 30    -6.180E+03 0.      -8.988E+03
1 11      30 40    -8.988E+03 0.      -1.118E+04
1 11      40 50    -1.118E+04 0.      -1.260E+04
1 11      50 60    -1.260E+04 0.      -1.309E+04
1 11      60 70    -1.309E+04 0.      -1.260E+04
1 11      70 80    -1.260E+04 0.      -1.118E+04
1 11      80 90    -1.118E+04 0.      -8.988E+03
1 11      90 100   -8.988E+03 0.      -6.180E+03
1 11      100 110  -6.180E+03 0.      -3.090E+03
1 11      110 119  -3.090E+03 0.      -3.090E+02
1 11      121 130  2.788E+02 0.      2.788E+03
1 11      130 140  2.788E+03 0.      5.000E+03
1 11      140 150  5.000E+03 0.      6.421E+03
1 11      150 160  6.421E+03 0.      6.910E+03
1 11      160 170  6.910E+03 0.      6.421E+03
1 11      170 180  6.421E+03 0.      5.000E+03
1 11      180 190  5.000E+03 0.      2.788E+03
1 11      190 199  2.788E+03 0.      2.788E+02
1 11      201 210  -2.212E+02 0.      -2.212E+03
1 11      210 220  -2.212E+03 0.      -3.633E+03
1 11      220 230  -3.633E+03 0.      -4.122E+03
1 11      230 240  -4.122E+03 0.      -3.633E+03
1 11      240 250  -3.633E+03 0.      -2.212E+03
1 11      250 259  -2.212E+03 0.      -2.212E+02
1 11      261 270  1.421E+02 0.      1.421E+03
1 11      270 280  1.421E+03 0.      1.910E+03

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1 -11      280 290      1.910E+03 0.      1.421E+03
1 11      290 299      1.421E+03 0.      1.421E+02
1 11      301 310      -4.890E+01 0.      -4.890E+02
1 11      310 319      -4.890E+02 0.      -4.890E+01
11 1-1.000E+00 1.000E-03 10      1 1.000E-04 1.000E+00
1070 1932 2465 3022 3380 3735 4240 4750 5800 5800
34 202 418 769 1072 1444 2119 2963 542620000
13 1-1.000E+00 1.000E-03 10      1 1.000E-04 1.000E+00
2650 4700 6100 7490 8365 924010500117501438014380
34 202 418 769 1072 1444 2119 2963 542620000
16 1-1.000E+00 1.000E-03 10      1 1.000E-04 1.000E+00
7020118401569619522217222365225340263052823528235
34 202 418 769 1072 1444 2119 2963 542620000
31 0-1.000E+00 1.000E-03 10      1 1.000E-04 1.000E+00
7020118401569619522217222365225340263052823528235
34 202 418 769 1072 1444 2119 2963 542620000

TIME DEFL 10 1
TIME DEFL 21 0
TIME MOMT 10 1
TIME MOMT 21 0
BEAM DEFL 60 1
BEAM DEFL 160 0
BEAM DEFL 230 1
BEAM DEFL 280 0
BEAM MOMT 60 1
BEAM MOMT 320 0
6 THREE-SPAN BEAM WITH ONE-WAY AND SYMMETRIC NONLINEAR SUP LOADED BY T,P
0 0 0 0 0 0 0 0 3 2 2 1 0 5 9 10
30 4.800E+01 40 4.000E-02 -1 4 -1 4 0
4 8 15 16 17
50 4.000E+00 1.000E-06 8 15 16 17
0 1 0. 2
30 1 0. 2
0 0 0 3.000E+10 -1.000E+05
0 30 0 2.400E+10
0 30 0 5.184E-01
17 17 0 4 0.0 0.0 -1.000E+04
17 17 4 8 -1.000E-04-1.000E+04 1.000E+03
17 17 8 12 1.000E+03 1.000E+03=1.000E+04
17 17 12 16 -1.000E+04-1.000E+04 1.000E+03
17 17 16 20 1.000E+03 1.000E+03=1.000E+04
8 8 0-1.000E+03 1.000E-01 7 1.000E-02 1.000E+00
-40 -40 -39 -35 -20 0 0
-40 -30 -20 -10 -5 0 40
15 1-1.000E+02 1.000E-01 4 1 1.000E-04 1.000E+00
0 0 0 0
3 10 20 40
17 0-1.000E+02 1.000E-01 4 1 1.000E-04 1.000E+00
30 86 136 136
3 10 20 40
TIME DEFL 8 1
TIME DEFL 17 0
TIME MOMT 8 1
TIME MOMT 17 0

```

BEAM	DEFL	4	1
BEAM	DEFL	8	1
BEAM	DEFL	20	0
BEAM	MOMT	12	1
BEAM	MOMT	16	1
BEAM	MOMT	30	0

7 PARTIALLY EMBEDDED PILE EXCITED BY EARTHQUAKE INDUCED FORCES (1/10 G)

0	0	0	0	0	0	3	0	1	2	0	19	6	10
50	1.200E+01	10	20	30	40	50	-1	5	-1	0	1	1	
50	1.000E+01	1.000E-06											
0	50	0	1.090E+10										
0	40	0		5.184E-01									
41	50	0		2.592E+00									
0	40	1	1	1.223E+01	1.223E+01	1.223E+01							
0	40	2	2	2.417E+01	2.417E+01	2.417E+01							
0	40	3	3	3.550E+01	3.550E+01	3.550E+01							
0	40	4	4	4.597E+01	4.597E+01	4.597E+01							
0	40	5	5	5.530E+01	5.530E+01	5.530E+01							
0	40	6	6	6.327E+01	6.327E+01	6.327E+01							
0	40	7	7	6.968E+01	6.968E+01	6.968E+01							
0	40	8	8	7.437E+01	7.437E+01	7.437E+01							
0	40	9	9	7.724E+01	7.724E+01	7.724E+01							
0	40	10	10	7.820E+01	7.820E+01	7.820E+01							
0	40	11	11	7.724E+01	7.724E+01	7.724E+01							
0	40	12	12	7.437E+01	7.437E+01	7.437E+01							
0	40	13	13	6.968E+01	6.968E+01	6.968E+01							
0	40	14	14	6.327E+01	6.327E+01	6.327E+01							
0	40	15	15	5.530E+01	5.530E+01	5.530E+01							
0	40	16	16	4.597E+01	4.597E+01	4.597E+01							
0	40	17	17	3.550E+01	3.550E+01	3.550E+01							
0	40	18	18	2.417E+01	2.417E+01	2.417E+01							
0	40	19	19	1.223E+01	1.223E+01	1.223E+01							
0	40	0	1.000E+00	-1.000E-02	4	0	1.000E-03	1.000E+00					
			100	100	0	0							
			1000	5	0-1000								
0	40	0	1.000E+00	-1.000E-02	4	-1	1.000E-03	1.000E+00					
			100	100	0	0							
			1000	5	0-1000								

TIME	DEFL	10	1
TIME	DEFL	30	0
TIME	DEFL	40	1
TIME	DEFL	50	0
BEAM	DEFL	10	1
BEAM	DEFL	20	0
BEAM	DEFL	50	1
BEAM	DEFL	150	0
BEAM	MOMT	30	1
BEAM	MOMT	100	0

APPENDIX H

PARTIAL SAMPLE COMPUTER OUTPUT FOR EXAMPLE PROBLEMS

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-- CTR Library Digitization Team

PROGRAM NBCS - MASTER - JACK CHAN - MATLOCK - DECK1-REVISION DATE # 26 JUN 71  
 EXAMPLE PROBLEMS FOR NBCS PROGRAM BY JACK CHAN APRIL 1971  
 DYNAMIC BEAM-COLUMN PROGRAM USING 4-1-5 IMPLICIT OPERATOR

PROB 1 ONE MASS SUPPORTED BY A NONLINEAR SPRING UNDER FREE VIBRATION

TABLE 1 - PROGRAM-CONTROL DATA

	TABLE NUMBER								
	2	3	4	5	6	7	8	9	
HOLD FROM PRECEDING PROBLEM (1=HOLD)	0	0	0	0	0	0	0	0	0
NUM CARDS INPUT THIS PROBLEM	2	0	1	1	0	3	3	1	

TABLE 2 - CONSTANTS

NUM OF BEAM INCREMENTS	1
BEAM INCREMENT LENGTH	1.000E+00
NUM OF TIME INCREMENTS	800
TIME INCREMENT LENGTH	6.283E-03
OPTIONAL PRINTING SWITCH	0
NUM OF MONITOR STA ( PRINTING )	-1
ITERATION SWITCH (0=LINEAR)	0
NUM OF MONITOR STA ( ITERATION )	1
PLOTTING METHOD (1=MIC, 0=PRINTER, 1=PAPER)	1
LINE OR POINTS PLOT OPTION	1

-----ITERATION DATA FOR NONLINEAR SUP. CV.-----

MAXIMUM ITERATION NUMBER	50
MAXIMUM ALLOWABLE DEFLECTION	1.000E+01
DEFLECTION CLOSURE TOLERANCE	1.000E-06
MONITOR STATIONS (ITERATION)	1 2 3 4 5

STA J -0

TABLE 3 - SPECIFIED DEFLECTIONS AND SLOPES

STA	CASE	DEFLECTION	SLOPE	IOPS (IF=1, INITIAL , IF=2, PERMANENT)
*NONE*				

TABLE 4 - STIFFNESS AND FIXED-LOAD DATA

FROM TO CONTD	F	QF	S	T	R	CP
1 1 0	-0.	1.000E+02	-0.	-0.	-0.	-0.

TABLE 5 - MASS AND DAMPINGS DATA

FROM TO CONTD	MMO	DT	DE
1 1 0	1.000E+00	-0.	-0.

TABLE 6 - TIME DEPENDENT AXIAL THRUST

BEAM STA FROM TO	TIME STA FROM TO	TT1	TT2	TT3
*NONE*				

TABLE 7 - TIME DEPENDENT LOADING

BEAM STA FROM TO	TIME STA FROM TO	QT1	QT2	QT3
1 1	0 0	-5.000E+01	-5.000E+01	-5.000E+01
1 1	0 800	-1.000E+02	-1.000E+02	-1.000E+02
1 1	800 800	-5.000E+01	-5.000E+01	-5.000E+01

TABLE 8 - NONLINEAR SUPPORT CURVES

FROM TO CONTD	Q-MULTIPLIER	W-MULTIPLIER	POINTS	SYM	OPT	W-TOLERANCE	Q-TOLERANCE
1 1 0	-1.000E+00	1.000E-01	5	1	1.000E-02	1.000E+00	
Q	20	36	60	120	120		
W	2	4	8	20	100		

TABLE 9 - PLOTTING SWITCHES

HORIZONTAL AXIS	VERTICAL AXIS	BEAM OR TIME STATION	MULTIPLE PLOT SWITCH	(IF=1, SUPERIMPOSE WITH NEXT IF=0, PLOT ALL SAVED PLOTS)
TIME	DEFL	1	0	

PROGRAM DBCS - MASTER - JACK CHAN - MATLOCK - DECK1-REVISION DATE = 26 JUN 71  
EXAMPLE PROBLEMS FOR DBCS PROGRAM BY JACK CHAN APRIL 1971  
DYNAMIC BEAM-COLUMN PROGRAM USING 5-1-5 IMPLICIT OPERATOR

PROB (CONTD)  
1 ONE MASS SUPPORTED BY A NONLINEAR SPRING UNDER FREE VIBRATION

PROB (CONTD)  
1 ONE MASS SUPPORTED BY A NONLINEAR SPRING UNDER FREE VIBRATION

TIME FOR THIS PROBLEM = 0 MINUTES 6.841 SECONDS

TABLE 10- CALCULATED RESULTS  
J=BEAM AXIS, K=TIME AXIS

ELAPSED CPU TIME = 0 MINUTES 31.086 SECONDS

\*PRINTING RESULTS IS NOT REQUESTED\*

\*STANDARD 12 INCH BALLPOINT PAPER PLOT IS REQUESTED\*

PROGRAM NBCS - MASTER - JACK CHAN - MATLOCK - DECK1-REVISION DATE = 26 JUN 71  
 EXAMPLE PROBLEMS FOR PROGRAM DHC5 BY JACK CHAN JUNE 1971  
 DYNAMIC BEAM-COLUMN PROGRAM ( 5-1-5 IMPLICIT OPERATOR )

0 0 0 -0. -0. -0. -0. 3.000E+10 -0.  
 1 30 0 2.400E+10 -0. -0. -1.000E+05 -0. -0.

PROB 6 THREE-SPAN BEAM WITH ONE-WAY AND SYMMETRIC NONLINEAR SUP LOADED BY T.P

TABLE 5 - MASS AND DAMPINGS DATA  
 FROM TO CONTD RHO DT DE  
 0 30 0 5.184E-01 -0. -0.

TABLE 1 - PROGRAM-CONTROL DATA  

	2	3	4	5	6	7	8	9
HOLD FROM PRECEDING PROBLEM (1=HOLD)	0	0	0	0	0	0	0	0
NUM CARDS INPUT THIS PROBLEM	3	2	2	1	0	5	9	10

TABLE 6 - TIME DEPENDENT AXIAL THRUST  
 BEAM STA TIME STA  
 FROM TO FROM TO TT1 TT2 TT3

\*NONE\*

TABLE 2 - CONSTANTS  
 NUM OF BEAM INCREMENTS 30  
 BEAM INCREMENT LENGTH 4.000E+01  
 NUM OF TIME INCREMENTS 40  
 TIME INCREMENT LENGTH 4.000E-02  
 OPTIONAL PRINTING SWITCH -1  
 NUM OF MONITOR STA ( PRINTING ) 4  
 ITERATION SWITCH (0=LINEAR) -1  
 NUM OF MONITOR STA ( ITERATION ) 4  
 PLOTTING METHOD (-1=MIC,0=PRINTED,1=PAPER) 0  
 LINE OR POINTS PLOT OPTION -0

TABLE 7 - TIME DEPENDENT LOADING  
 BEAM STA TIME STA  
 FROM TO FROM TO QT1 QT2 QT3  
 17 17 0 4 0. 0. -1.000E+04  
 17 17 4 8 -1.000E+04 -1.000E+04 1.000E+03  
 17 17 8 12 1.000E+03 1.000E+03 -1.000E+04  
 17 17 12 16 -1.000E+04 -1.000E+04 1.000E+03  
 17 17 16 20 1.000E+03 1.000E+03 -1.000E+04

-----COMPLETE PRINTING TIME INTERVAL AND MONITOR STATIONS FOR PRINTING-----  
 TIME STA INTERVAL OF COMPLETE PRINT 4  
 MONITOR STATIONS(PRINTING)  
 1 2 3 4 5 6 7 8 9 10  
 STA J B 15 16 17

TABLE 8 - NONLINEAR SUPPORT CURVES  
 FROM TO CONTD Q-MULTIPLIER W-MULTIPLIER POINTS SYM OPT W-TOLERANCE Q-TOLERANCE  
 8 8 0 -1.000E+03 1.000E-01 7 -0 1.000E-02 1.000E+00  
 Q -40 -40 -39 -35 -20 0 0  
 W -40 -30 -20 -10 -5 0 40  
 15 1 1 -1.000E+02 1.000E-01 4 1 1.000E-04 1.000E+00  
 Q 0 0 0 0  
 W 3 10 20 40  
 17 0 -1.000E+02 1.000E-01 4 1 1.000E-04 1.000E+00  
 Q 30 80 136 176  
 W 3 10 20 40

-----ITERATION DATA FOR NONLINEAR SUP. CV.-----  
 MAXIMUM ITERATION NUMBER 50  
 MAXIMUM ALLOWABLE DEFLECTION 4.000E+00  
 DEFLECTION CLOSURE TOLERANCE 1.000E-06  
 MONITOR STATIONS(ITERATION)  
 1 2 3 4 5  
 STA J B 15 16 17

TABLE 9 - PLOTTING SWITCHES  
 HORIZONTAL VERTICAL BEAM OR TIME MULTIPLE PLOT  
 AXIS AXIS STATION SWITCH (IF=1, SUPERIMPOSE WITH NEXT IF=0, PLOT ALL SAVED PLOTS)  
 TIME DEFL 8 1  
 TIME DEFL 17 0  
 TIME MOMT 8 1  
 TIME MOMT 17 0

TABLE 3 - SPECIFIED DEFLECTIONS AND SLOPES  

STA	CASE	DEFLECTION	SLOPE	IOPS (TF=1, INITIAL, TF=2, PERMANENT)
0	1	0.	NONE	2
30	1	0.	NONE	2

TABLE 4 - STIFFNESS AND FIXED-LOAD DATA  
 FROM TO CONTD F QF S T R CP

```

BEAM   DEFL      4      1
BEAM   DEFL      8      1
BEAM   DEFL     20      0
BEAM   MOMT     12      1
BEAM   MDMT     16      1
BEAM   MOMT     30      0

```

PROGRAM DBCS = MASTER - JACK CHAN - MATLOCK - DECK1-REVISION DATE = 26 JUN 71  
 EXAMPLE PROBLEMS FOR PROGRAM DBCS BY JACK CHAN JUNE 1971  
 DYNAMIC BEAM-COLUMN PROGRAM ( 5-1-6 IMPLICIT OPERATOR )

PROB (CONTD)  
 6 THREE-SPAN BEAM WITH ONE-WAY AND SYMMETRIC NONLINEAR SUP LOADED BY T,P

TABLE 10- CALCULATED RESULTS  
 J\*BFAM AXIS, K\*TIME AXIS

TIME STA K = -2

J	MONITOR	STATIONS
8	14	16 17

FOR 1 ITERATION, THERE ARE 0 STATIONS NOT CLOSED  
 -1.313E-16 -1.126E-15 -1.236E-15 -1.317E-15

TIME STA K = -1

J	MONITOR	STATIONS
8	14	16 17

FOR 1 ITERATION, THERE ARE 0 STATIONS NOT CLOSED  
 -1.313E-16 -1.126E-15 -1.236E-15 -1.317E-15

\*\*\*\*\* STATIC RESULTS \*\*\*\*\*

STA J	DIST	DEFL	SLOPE	MOM	SHEAR	SUP REACT
-1	-4.800E+01	5.001E-18	-1.642E-19	0.	0.	0.
0	0.	0.	1.114E-19	5.390E-11	1.123E-12	-1.189E-12
1	4.800E+01	5.346E-18	2.117E-19	5.017E-11	-6.651E-14	0.
2	9.600E+01	1.551E-17	2.117E-19	-7.936E-12	-1.189E-12	0.
3	1.440E+02	2.491E-17	1.054E-19	-6.596E-11	-1.189E-12	0.
4	1.920E+02	2.798E-17	6.792E-20	-1.234E-10	-1.189E-12	0.
5	2.400E+02	1.920E-17	-1.628E-19	-1.796E-10	-1.189E-12	0.
6	2.880E+02	-6.810E-18	-5.419E-19	-2.341E-10	-1.189E-12	0.
7	3.360E+02	-5.529E-17	-1.010E-18	-2.863E-10	-1.189E-12	0.
8	3.840E+02	-1.313E-16	-1.583E-18	-3.358E-10	-1.189E-12	2.625E-12
9	4.320E+02	-2.395E-16	-2.254E-18	-2.560E-10	1.436E-12	0.
			-2.766E-18		1.436E-12	

10	4.800E+02	-3.722E-16	-1.739E-10	0.	0.	0.
11	5.280E+02	-5.217E-16	-3.110E-18	-8.999E-11	1.436E-12	0.
12	5.760E+02	-6.798E-16	-3.294E-18	-5.247E-12	1.436E-12	0.
13	6.240E+02	-8.384E-16	-3.478E-18	7.953E-11	1.436E-12	0.
14	6.720E+02	-9.974E-16	-3.662E-18	1.676E-10	1.436E-12	0.
15	7.200E+02	-1.125E-15	-2.818E-18	2.460E-10	1.436E-12	0.
16	7.680E+02	-1.236E-15	-2.376E-18	3.261E-10	1.436E-12	-1.094E-12
17	8.160E+02	-1.317E-15	-1.474E-18	3.505E-10	3.417E-13	-3.924E-13
18	8.640E+02	-1.363E-15	-9.730E-19	3.394E-10	-3.507E-13	0.
19	9.120E+02	-1.379E-15	-2.963E-19	3.279E-10	-3.507E-13	0.
20	9.600E+02	-1.361E-15	3.496E-19	3.044E-10	-3.507E-13	0.
21	1.008E+03	-1.315E-15	9.584E-19	2.870E-10	-3.507E-13	0.
22	1.056E+03	-1.242E-15	1.524E-18	2.598E-10	-3.507E-13	0.
23	1.104E+03	-1.144E-15	2.042E-18	2.322E-10	-3.507E-13	0.
24	1.152E+03	-1.023E-15	2.507E-18	2.033E-10	-3.507E-13	0.
25	1.200E+03	-8.835E-16	2.913E-18	1.729E-10	-3.507E-13	0.
26	1.248E+03	-7.271E-16	3.258E-18	1.401E-10	-3.507E-13	0.
27	1.296E+03	-5.573E-16	3.538E-18	1.062E-10	-3.507E-13	0.
28	1.344E+03	-3.772E-16	3.751E-18	7.139E-11	-3.507E-13	0.
29	1.392E+03	-1.903E-16	3.894E-18	3.587E-11	-3.507E-13	0.
30	1.440E+03	0.	3.965E-18	0.	0.	3.507E-13
31	1.488E+03	1.903E-16	0.	0.	0.	0.

TIME STATION = 0

J = 8 MONITOR STATIONS  
1E 1A 17

FOR 1 ITERATION, THERE ARE 0 STATIONS NOT CLOSED  
-6.601E-17 -1.104E-15 -1.223E-15 -1.310E-15

\*\*\*\*\* DYNAMIC RESULTS \*\*\*\*\*

TIME STATION = 0

STA J DIST DEF L SLOPE MOM SHEAR SUP REACT

-1	-4.800E+01	7.329E-18	0.	0.	0.	0.
0	0.	0.	-1.427E-19	7.999E-11	1.645E-12	-1.397E-12
1	4.870E+01	7.835E-18	1.432E-19	9.014E-11	2.488E-13	0.
2	9.600E+01	2.432E-17	3.475E-19	2.137E-11	-1.394E-12	0.
3	1.440E+02	4.286E-17	3.862E-19	-4.787E-11	-1.404E-12	0.
4	1.920E+02	5.891E-17	2.005E-19	-1.172E-10	-1.416E-12	0.
5	2.400E+02	5.950E-17	5.806E-20	-1.863E-10	-1.434E-12	0.
6	2.880E+02	4.430E-17	-3.146E-19	-2.549E-10	-1.460E-12	0.
7	3.360E+02	4.831E-18	-8.264E-19	-3.226E-10	-1.494E-12	0.
8	3.840E+02	-6.601E-17	-1.472E-18	-3.891E-10	-1.532E-12	2.640E-12
9	4.320E+02	-1.740E-16	-2.250E-18	-2.958E-10	1.718E-12	0.
10	4.800E+02	-3.104E-16	-2.842E-18	-2.018E-10	1.676E-12	0.
11	5.280E+02	-4.462E-16	-3.245E-18	-1.077E-10	1.636E-12	0.
12	5.760E+02	-6.373E-16	-3.460E-18	-1.426E-11	1.600E-12	0.
13	6.240E+02	-7.997E-16	-3.489E-18	-1.426E-11	1.569E-12	0.
14	6.720E+02	-9.597E-16	-3.333E-18	7.779E-11	1.544E-12	0.
15	7.200E+02	-1.104E-15	-2.998E-18	1.679E-10	1.525E-12	0.
16	7.680E+02	-1.223E-15	-2.487E-18	2.555E-10	1.511E-12	0.
17	8.160E+02	-1.310E-15	-1.907E-18	3.399E-10	1.511E-12	-1.161E-12
18	8.640E+02	-1.310E-15	-1.677E-18	3.650E-10	3.414E-13	-7.274E-13
19	9.120E+02	-1.379E-15	-1.677E-18	3.514E-10	-3.906E-13	0.
20	9.600E+02	-1.379E-15	-3.739E-19	3.344E-10	-3.919E-13	0.
21	1.008E+03	-1.379E-15	2.849E-19	3.142E-10	-3.904E-13	0.
22	1.056E+03	-1.365E-15	9.233E-19	2.912E-10	-3.880E-13	0.
23	1.104E+03	-1.249E-15	1.506E-18	2.655E-10	-3.841E-13	0.
24	1.152E+03	-1.151E-15	2.037E-18	2.375E-10	-3.796E-13	0.
25	1.200E+03	-1.030E-15	2.512E-18	2.075E-10	-3.749E-13	0.
26	1.248E+03	-8.898E-16	2.927E-18	1.756E-10	-3.705E-13	0.
27	1.296E+03	-7.324E-16	3.278E-18	1.423E-10	-3.664E-13	0.
28	1.344E+03	-5.614E-16	3.563E-18	1.079E-10	-3.629E-13	0.
29	1.392E+03	-3.801E-16	3.778E-18	7.237E-11	-3.602E-13	0.
30	1.440E+03	-1.918E-16	3.823E-18	3.674E-11	-3.584E-13	0.
31	1.488E+03	0.	3.896E-18	0.	-3.574E-13	0.

30 1.440E+03 0. -4.109E-24 3.574E+13  
 31 1.440E+03 1.918E-16 3.096E-18 8.560E-24 0.  
 TIME STA K = 1

J = 8 MONITOR STATIONS  
 15 16 17  
 FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
 -0.665E-03 -1.190E-01 -1.747E-01 -1.462E-01  
 FOR 2 ITERATION, THERE ARE 0 STATIONS NOT CLOSED  
 -0.665E-03 -1.190E-01 -1.747E-01 -1.462E-01  
 TIME STA K = 1

STA J	DIST	DEFL	SLOPE	MOM	SHEAR	SUP REACT
			-1.375E+04		-1.382E+02	
8	3.840E+02	-6.465E-03	-3.778E+04	-3.578E+04	2.847E+02	2.666E+02
15	7.200E+02	-1.190E-01	-3.275E+04	2.514E+04	3.618E+02	0.
16	7.680E+02	-1.347E-01	-2.394E+04	4.408E+04	1.123E+03	6.735E+02
17	8.160E+02	-1.462E-01	-2.394E+04	9.912E+04		7.309E+02

TIME STA K = 2

J = 8 MONITOR STATIONS  
 15 16 17  
 FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
 -1.690E-02 -3.120E-01 -3.437E-01 -3.880E-01  
 FOR 2 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
 -1.613E-02 -2.998E-01 -3.404E-01 -3.744E-01  
 FOR 3 ITERATION, THERE ARE 0 STATIONS NOT CLOSED  
 -1.613E-02 -2.998E-01 -3.404E-01 -3.744E-01  
 TIME STA K = 2

STA J	DIST	DEFL	SLOPE	MOM	SHEAR	SUP REACT
			-3.444E-04		-3.629E+02	
8	3.840E+02	-1.613E-02	-9.751E-04	-9.538E+04	4.050E+02	6.452E+02
15	7.200E+02	-2.998E-01	-8.472E-04	4.396E+04	4.451E+02	0.
16	7.680E+02	-3.404E-01	-7.084E-04	6.939E+04	2.323E+03	1.662E+03
17	8.160E+02	-3.744E-01		1.843E+05		1.798E+03

TIME STA K = 2

J = 8 MONITOR STATIONS  
 15 16 17  
 FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
 -2.991E-02 -5.467E-01 -4.468E-01 -7.180E-01  
 FOR 2 ITERATION, THERE ARE 0 STATIONS NOT CLOSED  
 -2.991E-02 -5.467E-01 -4.468E-01 -7.180E-01  
 TIME STA K = 1

STA J	DIST	DEFL	SLOPE	MOM	SHEAR	SUP REACT
			-6.041E-04		-6.800E+02	
8	3.840E+02	-2.991E-02	-1.788E-03	-1.812E+05	5.185E+02	1.196E+03
15	7.200E+02	-5.467E-01	-1.468E-03	9.998E+04	4.972E+02	0.
16	7.680E+02	-6.468E-01	-1.485E-03	9.186E+04	3.618E+03	2.887E+03
17	8.160E+02	-7.180E-01		2.727E+05		3.172E+03

TIME STA K = 4

J = 8 MONITOR STATIONS  
 15 16 17  
 FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
 -4.673E-02 -8.891E-01 -1.019E+00 -1.137E+00  
 FOR 2 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
 -4.336E-02 -8.350E-01 -9.414E-01 -1.078E+00  
 FOR 3 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
 -4.576E-02 -8.684E-01 -9.057E-01 -1.111E+00  
 FOR 4 ITERATION, THERE ARE 0 STATIONS NOT CLOSED  
 -4.576E-02 -8.684E-01 -9.057E-01 -1.111E+00  
 TIME STA K = 4

STA J	DIST	DEFL	SLOPE	MOM	SHEAR	SUP REACT
-1	-4.800E+01	5.388E-03		0.	1.210E+03	0.
0	0.	0.	-1.122E-04	5.804E+04	1.811E+02	-1.028E+03
1	4.800E+01	5.759E-03	1.200E-04	4.617E+04	-1.028E+03	0.
2	9.600E+01	1.787E-02	2.623E-04	1.561E+04	-1.027E+03	0.
3	1.440E+02	3.148E-02	2.876E-04	-3.505E+04	-1.025E+03	0.
4	1.920E+02	4.173E-02	2.135E-04	-8.529E+04	-1.023E+03	0.
5	2.400E+02	4.379E-02	4.288E-05	-1.346E+05	-1.020E+03	0.
6	2.880E+02	3.293E-02	-2.263E-04	-1.824E+05	-1.017E+03	0.
			-5.012E-04			

7	3.360E+02	4.549E-03	-2.294E+05	0.	0.
8	3.840E+02	-4.574E-02	-2.722E+05	-1.016E+03	1.830E+03
9	4.320E+02	-1.222E-01	-2.254E+05	8.142E+02	0.
10	4.800E+02	-7.203E-01	-2.043E-03	8.105E+02	0.
11	5.280E+02	-3.353E-01	-2.397E-03	-1.767E+05	8.032E+02
12	5.760E+02	-4.625E-01	-2.450E-03	-1.267E+05	7.923E+02
13	6.240E+02	-5.970E-01	-2.002E-03	-7.592E+04	7.786E+02
14	6.720E+02	-7.339E-01	-2.052E-03	-2.510E+04	7.628E+02
15	7.200E+02	-8.684E-01	-2.002E-03	2.521E+04	7.459E+02
16	7.680E+02	-9.957E-01	-2.453E-03	7.446E+04	7.283E+02
17	8.160E+02	-1.111E+00	-2.408E-03	1.222E+05	4.283E+03
18	8.640E+02	-1.192E+00	-1.678E-03	3.652E+05	4.578E+03
19	9.120E+02	-1.238E+00	-9.560E-04	-2.547E+02	0.
20	9.600E+02	-1.250E+00	3.610E+05	-2.793E+02	0.
21	1.008E+03	-1.229E+00	3.522E+05	-3.073E+02	0.
22	1.056E+03	-1.178E+00	-2.616E-04	3.387E+05	-3.375E+02
23	1.104E+03	-1.098E+00	4.257E-04	3.204E+05	-3.684E+02
24	1.152E+03	-9.927E-01	1.067E-03	2.976E+05	-3.982E+02
25	1.200E+03	-8.640E-01	1.462E-03	2.705E+05	-4.258E+02
26	1.248E+03	-7.155E-01	2.203E-03	2.395E+05	-4.500E+02
27	1.296E+03	-5.510E-01	2.482E-03	2.050E+05	-4.704E+02
28	1.344E+03	-3.743E-01	3.092E-03	1.676E+05	-4.865E+02
29	1.392E+03	-1.892E-01	3.483E-03	1.278E+05	-4.984E+02
30	1.440E+03	0.	3.855E-03	8.621E+04	-5.062E+02
31	1.488E+03	1.892E-01	3.942E-03	4.341E+04	-5.101E+02
			0.	0.	0.
			0.	0.	0.
			0.	0.	0.

TIME STA K = 5

J = 8 MONITOR STATIONS  
15 16 17

FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-4.600E-02 -8.574E-01 -9.930E-01 -1.125E+00

FOR 2 ITERATION, THERE ARE 2 STATIONS NOT CLOSED  
-4.600E-02 -8.574E-01 -9.930E-01 -1.125E+00

\* USED ADJUSTED LOAD-DEFLECT CURVE TO CALCULATE DEFLECTIONS \*

FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-4.678E-02 -8.684E-01 -1.004E+00 -1.135E+00

FOR 2 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-4.470E-02 -8.402E-01 -9.745E-01 -1.106E+00

FOR 3 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-4.678E-02 -8.684E-01 -1.004E+00 -1.135E+00

FOR 4 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-4.470E-02 -8.402E-01 -9.745E-01 -1.106E+00

FOR 5 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-4.738E-02 -8.859E-01 -1.024E+00 -1.158E+00

FOR 6 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-4.785E-02 -8.938E-01 -1.033E+00 -1.167E+00

FOR 7 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-4.798E-02 -8.961E-01 -1.036E+00 -1.170E+00

FOR 8 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-4.801E-02 -8.966E-01 -1.036E+00 -1.171E+00

FOR 9 ITERATION, THERE ARE 27 STATIONS NOT CLOSED  
-4.802E-02 -8.968E-01 -1.036E+00 -1.171E+00

FOR 10 ITERATION, THERE ARE 26 STATIONS NOT CLOSED  
-4.802E-02 -8.968E-01 -1.036E+00 -1.171E+00

FOR 11 ITERATION, THERE ARE 20 STATIONS NOT CLOSED  
-4.802E-02 -8.968E-01 -1.036E+00 -1.171E+00

FOR 12 ITERATION, THERE ARE 15 STATIONS NOT CLOSED  
-4.802E-02 -8.968E-01 -1.036E+00 -1.171E+00

FOR 13 ITERATION, THERE ARE 0 STATIONS NOT CLOSED  
-3.474E-02 -7.216E-01 -8.534E-01 -9.930E-01

TIME STA K = 5

STA J	QIST	DEFL	SLOPE	MOM	SHEAR	SUP REACT
			-8.204E-04	-2.115E+05	-7.733E+02	1.390E+03
8	3.840E+02	-3.474E-02	-2.413E-03	-6.705E+04	-2.778E+02	0.
15	7.200E+02	-7.216E-01	-2.747E-03	-8.705E+04	-5.472E+02	7.175E+03
16	7.680E+02	-8.534E-01	-2.707E-03	-8.012E+04	5.529E+03	3.838E+03
17	8.160E+02	-9.930E-01	-2.607E-03	1.992E+05		

TIME STA K = 4

J = 8 MONITOR STATIONS  
15 16 17

FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-1.698E-02 -5.000E-01 -5.987E-01 -7.008E-01

24	1.152E+03	4.948E-01	-7.178E+05	0.		
25	1.200E+03	4.015E-01	-2.058E+05	1.790E+02	0.	
26	1.248E+03	3.485E-01	-1.813E+05	4.004E+02	0.	
27	1.296E+03	2.781E-01	-1.467E+05	5.885E+02	0.	
28	1.344E+03	1.936E-01	-1.022E+05	7.361E+02	0.	
29	1.392E+03	9.932E-02	-5.262E+04	8.376E+02	0.	
30	1.440E+03	0.	-4.626E+09	8.892E+02	-8.892E+02	
31	1.488E+03	-9.932E-02	0.	9.637E+11	0.	

TIME STA K = 9

J = 8 MONITOR STATIONS  
15 16 17

FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-1.559E+02 5.584E+02 1.039E-01 1.656E+01

FOR 2 ITERATION, THERE ARE 0 STATIONS NOT CLOSED  
-1.559E+02 5.584E+02 1.039E-01 1.656E+01  
\* USED ADJUSTED LOAD-DEFL CURVE TO CALCULATE DEFLECTIONS \*

FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-1.430E+02 7.719E+02 1.272E-01 1.896E+01

FOR 2 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-1.479E+02 6.642E+02 1.150E-01 1.764E+01

FOR 3 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-1.430E+02 7.719E+02 1.272E-01 1.896E+01

FOR 4 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-1.479E+02 6.642E+02 1.150E-01 1.764E+01

FOR 5 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-1.490E+02 6.719E+02 1.162E-01 1.783E+01

FOR 6 ITERATION, THERE ARE 28 STATIONS NOT CLOSED  
-1.485E+02 6.791E+02 1.171E-01 1.792E+01

FOR 7 ITERATION, THERE ARE 26 STATIONS NOT CLOSED  
-1.485E+02 6.791E+02 1.174E-01 1.791E+01

FOR 8 ITERATION, THERE ARE 15 STATIONS NOT CLOSED  
-1.485E+02 6.791E+02 1.170E-01 1.791E+01

FOR 9 ITERATION, THERE ARE 0 STATIONS NOT CLOSED  
-1.485E+02 6.791E+02 1.170E-01 1.791E+01

TIME STA K = 9

STA J	DIST	DEFL	SLOPE	MOM	SHEAR	SUP REACT
		-7.460E-05			-6.310E+00	

14	3.840E+02	-1.445E-02	-5.371E+03	5.940E+02		
15	7.200E+02	6.791E-02	7.712E-04	1.250E+05	3.071E+02	0.
16	7.680E+02	1.170E-01	1.027E-03	1.354E+05	3.004E+02	-7.505E+02
17	8.160E+02	1.791E-01	1.294E-03	1.574E+05	5.907E+02	-1.007E+03

TIME STA K = 10

J = 8 MONITOR STATIONS  
15 16 17

FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-1.738E-02 -1.844E-01 -1.970E-01 -2.008E-01

FOR 2 ITERATION, THERE ARE 0 STATIONS NOT CLOSED  
-1.738E-02 -1.844E-01 -1.970E-01 -2.008E-01

TIME STA K = 10

STA J	DIST	DEFL	SLOPE	MOM	SHEAR	SUP REACT
14	3.840E+02	-1.738E-02	-2.732E-04	-6.771E+04	-2.799E+02	6.952E+02
15	7.200E+02	-1.844E-01	-3.977E-04	6.795E+04	3.847E+02	0.
16	7.680E+02	-1.970E-01	-2.620E-04	9.100E+04	4.580E+02	8.195E+02
17	8.160E+02	-2.008E-01	-7.911E-05	1.844E+05	1.945E+03	8.928E+02

TIME STA K = 11

J = 8 MONITOR STATIONS  
15 16 17

FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-2.115E-02 -6.451E-01 -7.590E-01 -8.668E-01

FOR 2 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-2.036E-02 -6.332E-01 -7.464E-01 -8.540E-01

FOR 3 ITERATION, THERE ARE 0 STATIONS NOT CLOSED  
-2.036E-02 -6.332E-01 -7.464E-01 -8.540E-01

TIME STA K = 11

STA J	DIST	DEFL	SLOPE	MOM	SHEAR	SUP REACT
14	3.840E+02	-2.036E-02	-6.527E-04	-1.828E+05	-7.132E+02	8.144E+02
15	7.200E+02	-6.332E-01	-2.769E-03	6.249E+03	6.882E+02	0.
16	7.680E+02	-7.464E-01	-2.757E-03	5.703E+04	8.223E+02	3.285E+03
17	8.160E+02	-8.540E-01	-2.243E-03	2.373E+05	3.448E+03	3.716E+03

20	9.600E+02	-1.719E+00	-1.441E-03	4.891E+05	1.016E+03	0.
21	1.008E+03	-1.753E+00	-7.130E-04	5.271E+05	7.210E+02	0.
22	1.056E+03	-1.736E+00	3.512E-04	5.446E+05	4.002E+02	0.
23	1.104E+03	-1.667E+00	1.440E-03	5.407E+05	6.144E+01	0.
24	1.152E+03	-1.546E+00	2.522E-03	5.149E+05	-2.845E+02	0.
25	1.200E+03	-1.375E+00	3.552E-03	4.679E+05	-6.237E+02	0.
26	1.248E+03	-1.160E+00	4.487E-03	4.012E+05	-9.405E+02	0.
27	1.296E+03	-9.061E-01	5.290E-03	3.173E+05	-1.219E+03	0.
28	1.344E+03	-6.217E-01	5.925E-03	2.197E+05	-1.443E+03	0.
29	1.392E+03	-3.162E-01	6.364E-03	1.173E+05	-1.600E+03	0.
30	1.440E+03	0.	6.589E-03	-9.252E-09	1.681E+03	1.681E+03
31	1.488E+03	3.162E-01	0.	0.	1.927E+00	0.

TIME STA K = 13

J = 8 MONITOR STATIONS  
15 16 17

FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-5.080E-02 -9.677E-01 -1.130E+00 -1.296E+00

FOR 2 ITERATION, THERE ARE 28 STATIONS NOT CLOSED  
-5.080E-02 -9.677E-01 -1.130E+00 -1.296E+00  
\* USED ADJUSTED LOAD-DEFL CURVE TO CALCULATE DEFLECTIONS \*

FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-5.499E-02 -1.037E+00 -1.205E+00 -1.374E+00

FOR 2 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-5.080E-02 -9.677E-01 -1.130E+00 -1.296E+00

FOR 3 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-5.499E-02 -1.037E+00 -1.205E+00 -1.374E+00

FOR 4 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-5.395E-02 -1.021E+00 -1.188E+00 -1.354E+00

FOR 5 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-5.403E-02 -1.021E+00 -1.188E+00 -1.356E+00

FOR 6 ITERATION, THERE ARE 28 STATIONS NOT CLOSED  
-5.404E-02 -1.021E+00 -1.188E+00 -1.356E+00

FOR 7 ITERATION, THERE ARE 17 STATIONS NOT CLOSED  
-5.404E-02 -1.021E+00 -1.188E+00 -1.356E+00

FOR 8 ITERATION, THERE ARE 28 STATIONS NOT CLOSED  
-3.996E-02 -8.339E-01 -9.919E-01 -1.165E+00

TIME STA K = 12

STA J	DIST	DEFL	SLOPE	MOM	SHEAR	SUP REACT
8	3.840E+02	-3.996E-02	-9.687E-04	-2.433E+05	-8.592E+02	1.598E+03
15	7.200E+02	-8.339E-01	-3.058E-03	-1.163E+05	-7.246E+02	0.
16	7.680E+02	-9.919E-01	-1.290E-03	-1.581E+05	-1.198E+03	7.744E+03
17	8.160E+02	-1.165E+00	-3.607E-03	1.547E+05	6.156E+03	4.359E+03

TIME STA K = 14

J = 8 MONITOR STATIONS  
15 16 17

FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-3.344E-02 -5.674E-01 -6.521E-01 -7.297E-01

FOR 2 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-3.422E-02 -5.840E-01 -6.706E-01 -7.497E-01

FOR 3 ITERATION, THERE ARE 28 STATIONS NOT CLOSED  
-3.422E-02 -5.840E-01 -6.706E-01 -7.497E-01

TIME STA K = 16

STA J	DIST	DEFL	SLOPE	MOM	SHEAR	SUP REACT
8	3.840E+02	-3.422E-02	-7.196E-04	-1.828E+05	-7.239E+02	1.369E+03
15	7.200E+02	-5.840E-01	-1.888E-03	4.225E+04	4.914E+02	0.
16	7.680E+02	-6.706E-01	-1.804E-03	7.796E+04	5.635E+02	2.210E+03
17	8.160E+02	-7.497E-01	-1.448E-03	2.221E+05	2.838E+03	2.159E+03

TIME STA K = 15

J = 8 MONITOR STATIONS  
15 16 17

FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-1.099E-02 -1.445E-01 -1.785E-01 -1.102E-01

FOR 2 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
-1.095E-02 -1.430E-01 -1.778E-01 -1.096E-01

FOR 3 ITERATION, THERE ARE 28 STATIONS NOT CLOSED  
-1.095E-02 -1.430E-01 -1.778E-01 -1.096E-01

TIME STA K = 15

STA J	DIST	DEFL	SLOPE	MOM	SHEAR	SUP REACT
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      A      3.840E+02  -1.095E-02  -2.674E-04  -5.922E+04  4.378E+02
15      7.200E+02  -1.439E-01  -1.861E-04  1.558E+05  0.
16      7.680E+02  -1.378E-01  1.256E-04  2.317E+05  -1.435E+01
17      8.160E+02  -1.096E-01  5.882E-04  2.074E+05  -4.396E+02

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TIME STA K = 16

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      J =      8      MONITOR      STATIONS
           15      16      17
FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED
      5.534E-04  1.768E-01  2.399E-01  3.191E-01
FOR 2 ITERATION, THERE ARE 29 STATIONS NOT CLOSED
      -2.174E-02  1.694E-01  2.351E-01  3.164E-01
FOR 3 ITERATION, THERE ARE 29 STATIONS NOT CLOSED
      5.534E-04  1.768E-01  2.399E-01  3.191E-01
FOR 4 ITERATION, THERE ARE 29 STATIONS NOT CLOSED
      -2.174E-02  1.694E-01  2.351E-01  3.164E-01
FOR 5 ITERATION, THERE ARE 29 STATIONS NOT CLOSED
      7.264E-04  1.798E-01  2.432E-01  3.224E-01
FOR 6 ITERATION, THERE ARE 29 STATIONS NOT CLOSED
      2.095E-03  1.923E-01  2.675E-01  3.372E-01
FOR 7 ITERATION, THERE ARE 29 STATIONS NOT CLOSED
      3.214E-03  1.931E-01  2.683E-01  3.379E-01
FOR 8 ITERATION, THERE ARE 29 STATIONS NOT CLOSED
      4.107E-03  1.934E-01  2.685E-01  3.380E-01
FOR 9 ITERATION, THERE ARE 29 STATIONS NOT CLOSED
      4.819E-03  1.937E-01  2.687E-01  3.381E-01
FOR 10 ITERATION, THERE ARE 29 STATIONS NOT CLOSED
      5.387E-03  1.938E-01  2.689E-01  3.382E-01
FOR 11 ITERATION, THERE ARE 29 STATIONS NOT CLOSED
      5.839E-03  1.940E-01  2.691E-01  3.382E-01
FOR 12 ITERATION, THERE ARE 29 STATIONS NOT CLOSED
      6.200E-03  1.941E-01  2.690E-01  3.383E-01
FOR 13 ITERATION, THERE ARE 29 STATIONS NOT CLOSED
      6.487E-03  1.942E-01  2.690E-01  3.383E-01
FOR 14 ITERATION, THERE ARE 29 STATIONS NOT CLOSED
      6.716E-03  1.943E-01  2.691E-01  3.383E-01
FOR 15 ITERATION, THERE ARE 29 STATIONS NOT CLOSED
      6.898E-03  1.944E-01  2.691E-01  3.384E-01
FOR 16 ITERATION, THERE ARE 29 STATIONS NOT CLOSED

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      7.044E-01  1.944E-01  2.692E-01  3.384E-01
FOR 17 ITERATION, THERE ARE 29 STATIONS NOT CLOSED
      7.160E-03  1.944E-01  2.692E-01  3.384E-01
FOR 18 ITERATION, THERE ARE 28 STATIONS NOT CLOSED
      7.252E-03  1.944E-01  2.692E-01  3.384E-01
FOR 19 ITERATION, THERE ARE 28 STATIONS NOT CLOSED
      7.326E-03  1.944E-01  2.692E-01  3.384E-01
FOR 20 ITERATION, THERE ARE 28 STATIONS NOT CLOSED
      7.385E-03  1.944E-01  2.692E-01  3.384E-01
FOR 21 ITERATION, THERE ARE 27 STATIONS NOT CLOSED
      7.432E-03  1.944E-01  2.692E-01  3.384E-01
FOR 22 ITERATION, THERE ARE 27 STATIONS NOT CLOSED
      7.469E-03  1.944E-01  2.693E-01  3.384E-01
FOR 23 ITERATION, THERE ARE 27 STATIONS NOT CLOSED
      7.499E-03  1.944E-01  2.693E-01  3.384E-01
FOR 24 ITERATION, THERE ARE 25 STATIONS NOT CLOSED
      7.522E-03  1.944E-01  2.693E-01  3.384E-01
FOR 25 ITERATION, THERE ARE 23 STATIONS NOT CLOSED
      7.541E-03  1.944E-01  2.693E-01  3.384E-01
FOR 26 ITERATION, THERE ARE 22 STATIONS NOT CLOSED
      7.556E-03  1.944E-01  2.693E-01  3.384E-01
FOR 27 ITERATION, THERE ARE 20 STATIONS NOT CLOSED
      7.568E-03  1.944E-01  2.693E-01  3.384E-01
FOR 28 ITERATION, THERE ARE 16 STATIONS NOT CLOSED
      7.578E-03  1.944E-01  2.693E-01  3.384E-01
FOR 29 ITERATION, THERE ARE 15 STATIONS NOT CLOSED
      7.585E-03  1.944E-01  2.693E-01  3.385E-01
FOR 30 ITERATION, THERE ARE 14 STATIONS NOT CLOSED
      7.592E-03  1.944E-01  2.693E-01  3.385E-01
FOR 31 ITERATION, THERE ARE 14 STATIONS NOT CLOSED
      7.596E-03  1.944E-01  2.693E-01  3.385E-01
FOR 32 ITERATION, THERE ARE 13 STATIONS NOT CLOSED
      7.600E-03  1.944E-01  2.693E-01  3.385E-01
FOR 33 ITERATION, THERE ARE 12 STATIONS NOT CLOSED
      7.603E-03  1.944E-01  2.693E-01  3.385E-01
FOR 34 ITERATION, THERE ARE 11 STATIONS NOT CLOSED
      7.606E-03  1.944E-01  2.693E-01  3.385E-01
FOR 35 ITERATION, THERE ARE 9 STATIONS NOT CLOSED
      7.608E-03  1.944E-01  2.693E-01  3.385E-01
FOR 36 ITERATION, THERE ARE 7 STATIONS NOT CLOSED
      7.609E-03  1.944E-01  2.693E-01  3.385E-01

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FOR 37 ITERATION, THERE ARE 5 STATIONS NOT CLOSED  
 7.611E-03 1.944E-01 2.593E-01 3.385E-01

FOR 38 ITERATION, THERE ARE 0 STATIONS NOT CLOSED  
 7.612E-03 1.944E-01 2.593E-01 3.385E-01

TIME STA K = 16

STA J	DIST	OFFL	SLOPE	MOM	SHEAR	SUP REACT
-1	-4.800E+01	-5.278E-04	1.700E+05	0.	-1.185E+02	0.
0	0.	0.	-1.775E+05	-5.688E+03	-7.430E+00	1.111E+02
1	4.800E+01	-5.642E-04	-2.773E+05	-5.988E+03	1.100E+02	0.
2	9.600E+01	-1.703E-03	-2.492E+05	-5.926E+02	1.075E+02	0.
3	1.440E+02	-2.899E-03	-1.454E+05	4.699E+03	1.047E+02	0.
4	1.920E+02	-3.645E-03	4.041E+06	9.799E+03	1.039E+02	0.
5	2.400E+02	-3.451E-03	7.356E+05	1.476E+04	1.084E+02	0.
6	2.880E+02	-1.840E-03	7.316E+05	1.980E+04	1.218E+02	0.
7	3.360E+02	1.671E-03	1.238E+04	2.530E+04	1.474E+02	0.
8	3.840E+02	7.612E-03	1.873E+04	3.178E+04	1.874E+02	0.
9	4.320E+02	1.660E-02	2.671E+04	3.988E+04	2.433E+02	0.
10	4.800E+02	2.942E-02	3.676E+04	5.027E+04	3.104E+02	0.
11	5.280E+02	4.707E-02	4.944E+04	6.341E+04	3.794E+02	0.
12	5.760E+02	7.080E-02	6.529E+04	7.924E+04	4.382E+02	0.
13	6.240E+02	1.021E-01	8.472E+04	9.715E+04	4.763E+02	0.
14	6.720E+02	1.428E-01	1.079E+03	1.159E+05	4.887E+02	0.
15	7.200E+02	1.946E-01	1.748E-03	1.342E+05	4.825E+02	0.
16	7.680E+02	2.593E-01	1.649E-03	1.509E+05	-1.603E+03	-1.603E+03
17	8.160E+02	3.385E-01	1.799E-03	7.474E+04	-1.422E+03	-2.193E+03
18	8.640E+02	4.248E-01	1.759E-03	-1.994E+04	-1.793E+03	0.
19	9.120E+02	5.092E-01	1.759E-03	-1.759E+03	-1.651E+03	0.
20	9.600E+02	5.833E-01	1.644E-03	-1.076E+05	-1.429E+03	0.
21	1.008E+03	6.398E-01	1.776E-03	-2.440E+05	-1.140E+03	0.
22	1.056E+03	6.728E-01	1.885E-04	-2.859E+05	-8.047E+02	0.
23	1.104E+03	6.784E-01	1.746E-04	-3.079E+05	-4.450E+02	0.
24	1.152E+03	6.545E-01	-4.991E-04	-3.095E+05	-8.370E+01	0.

25	1.200E+03	6.008E-01	-1.719E-03	-2.592E+02	2.592E+02	0.
26	1.248E+03	5.192E-01	-1.701E-03	-2.917E+05	5.670E+02	0.
27	1.296E+03	4.129E-01	-2.714E-03	-2.563E+05	8.266E+02	0.
28	1.344E+03	2.868E-01	-2.626E-03	-2.040E+05	1.029E+03	0.
29	1.392E+03	1.470E-01	-2.914E-03	-1.440E+05	1.166E+03	0.
30	1.440E+03	0.	-3.062E-03	-7.404E+04	1.236E+03	0.
31	1.488E+03	-1.470E-01	-3.062E-03	-4.626E+09	9.437E+11	-1.236E+03

TIME STA K = 17

MONITOR STATIONS  
 J = 8 15 14 17

FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
 -1.726E-02 3.952E-02 4.717E-02 1.005E-01

FOR 2 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
 -6.020E-03 4.335E-02 4.969E-02 1.019E-01

FOR 3 ITERATION, THERE ARE 0 STATIONS NOT CLOSED  
 -6.020E-03 4.335E-02 4.969E-02 1.019E-01  
 \* USED ADJUSTED LOAD-DEFL CURVE TO CALCULATE DEFLECTIONS \*

FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
 -3.977E-03 7.709E-02 1.064E-01 1.398E-01

FOR 2 ITERATION, THERE ARE 0 STATIONS NOT CLOSED  
 -3.977E-03 7.709E-02 1.064E-01 1.398E-01

TIME STA K = 17

STA J	DIST	OFFL	SLOPE	MOM	SHEAR	SUP REACT
8	3.840E+02	-3.977E-03	7.454E-06	1.222E+04	1.015E+02	1.591E+02
15	7.200E+02	7.709E-02	5.747E-04	4.810E+04	-1.584E+01	0.
14	7.680E+02	1.064E-01	6.710E-04	4.777E+04	-7.083E+01	-8.385E+02
17	8.160E+02	1.398E-01	6.945E-04	1.143E+05	1.579E+03	-1.200E+03

TIME STA K = 18

MONITOR STATIONS  
 J = 8 15 14 17

FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
 -3.739E-02 -3.899E-01 -4.511E-01 -5.069E-01

FOR 2 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
 -3.483E-02 -3.654E-01 -4.249E-01 -4.802E-01

15 7.200E+02 1.489E-01 -6.700E+03 0.  
 14 7.680E+02 1.473E-01 3.443E-04 3.487E+04 4.961E+02 -1.235E+03  
 17 8.160E+02 1.891E-01 4.541E-04 5.660E+04 4.943E+02 -1.596E+03

TIME STA K = 23

J = 8 MONITOR STATIONS  
 15 14 17

FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
 1.290E-02 3.648E-01 4.548E-01 5.723E-01

FOR 2 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
 8.561E-03 3.633E-01 4.539E-01 5.718E-01

FOR 3 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
 8.561E-03 3.633E-01 4.539E-01 5.718E-01

TIME STA K = 23

STA J	DIST	OFFL	SLOPF	MON	SHEAP	SUP REACT
8	3.840E+02	8.561E-03	4.439E-04	1.018E+05	1.771E+02	0.
15	7.200E+02	3.633E-01	1.517E-03	1.844E+05	1.620E+03	0.
14	7.680E+02	4.539E-01	1.886E-03	2.855E+05	2.294E+03	-2.381E+03
17	8.160E+02	5.718E-01	2.457E-03	1.747E+05	-2.063E+03	-3.127E+03

TIME STA K = 24

J = 8 MONITOR STATIONS  
 15 14 17

FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
 -1.015E-04 1.154E-01 1.943E-01 2.981E-01

FOR 2 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
 1.965E-03 1.157E-01 1.948E-01 2.983E-01

FOR 3 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
 -1.015E-04 1.154E-01 1.943E-01 2.981E-01

FOR 4 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
 1.965E-03 1.157E-01 1.948E-01 2.983E-01

FOR 5 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
 4.429E-03 1.288E-01 2.678E-01 3.111E-01

FOR 6 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
 4.571E-03 1.188E-01 1.982E-01 3.011E-01

FOR 7 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
 4.664E-03 1.194E-01 1.979E-01 3.008E-01

FOR 8 ITERATION, THERE ARE 29 STATIONS NOT CLOSED  
 4.737E-03 1.194E-01 1.979E-01 3.008E-01

FOR 9 ITERATION, THERE ARE 28 STATIONS NOT CLOSED  
 4.795E-03 1.194E-01 1.979E-01 3.008E-01

FOR 10 ITERATION, THERE ARE 27 STATIONS NOT CLOSED  
 4.841E-03 1.194E-01 1.979E-01 3.008E-01

FOR 11 ITERATION, THERE ARE 27 STATIONS NOT CLOSED  
 4.878E-03 1.194E-01 1.979E-01 3.008E-01

FOR 12 ITERATION, THERE ARE 27 STATIONS NOT CLOSED  
 4.907E-03 1.194E-01 1.979E-01 3.008E-01

FOR 13 ITERATION, THERE ARE 25 STATIONS NOT CLOSED  
 4.931E-03 1.194E-01 1.979E-01 3.008E-01

FOR 14 ITERATION, THERE ARE 24 STATIONS NOT CLOSED  
 4.949E-03 1.194E-01 1.979E-01 3.008E-01

FOR 15 ITERATION, THERE ARE 22 STATIONS NOT CLOSED  
 4.964E-03 1.194E-01 1.979E-01 3.008E-01

FOR 16 ITERATION, THERE ARE 21 STATIONS NOT CLOSED  
 4.976E-03 1.194E-01 1.979E-01 3.008E-01

FOR 17 ITERATION, THERE ARE 18 STATIONS NOT CLOSED  
 4.986E-03 1.194E-01 1.979E-01 3.008E-01

FOR 18 ITERATION, THERE ARE 15 STATIONS NOT CLOSED  
 4.993E-03 1.194E-01 1.979E-01 3.008E-01

FOR 19 ITERATION, THERE ARE 14 STATIONS NOT CLOSED  
 4.999E-03 1.194E-01 1.979E-01 3.008E-01

FOR 20 ITERATION, THERE ARE 14 STATIONS NOT CLOSED  
 1.000E-02 1.194E-01 1.979E-01 3.008E-01

FOR 21 ITERATION, THERE ARE 13 STATIONS NOT CLOSED  
 1.001E-02 1.194E-01 1.979E-01 3.008E-01

FOR 22 ITERATION, THERE ARE 12 STATIONS NOT CLOSED  
 1.001E-02 1.194E-01 1.979E-01 3.008E-01

FOR 23 ITERATION, THERE ARE 11 STATIONS NOT CLOSED  
 1.001E-02 1.194E-01 1.979E-01 3.008E-01

FOR 24 ITERATION, THERE ARE 9 STATIONS NOT CLOSED  
 1.002E-02 1.194E-01 1.979E-01 3.008E-01

FOR 25 ITERATION, THERE ARE 7 STATIONS NOT CLOSED  
 1.002E-02 1.194E-01 1.979E-01 3.008E-01

FOR 26 ITERATION, THERE ARE 5 STATIONS NOT CLOSED  
 1.002E-02 1.194E-01 1.979E-01 3.008E-01

FOR 27 ITERATION, THERE ARE 0 STATIONS NOT CLOSED  
 1.002E-02 1.194E-01 1.979E-01 3.008E-01

\* USED ADJUSTED LOAD-OFFL CURVE TO CALCULATE DEFLECTIONS \*

FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED

3.095E-03 1.477E-01 2.299E-01 3.340E-01

FOR 2 ITERATION, THERE ARE 0 STATIONS NOT CLOSED

J.045E-03 1.477E-01 2.299E-01 3.340E-01

TIME STA K = 24

STA J	DIST	DEFL	SLOPE	WOM	SHEAR	SUP REACT
-1	-4.800E+01	1.448E-04	0.	0.	3.250F+01	0.
0	0.	0.	-3.016E-06	1.540E+03	3.187E+01	-4.333E+01
1	4.800E+01	1.548E-04	3.224E-06	1.074E+03	3.187E+01	0.
2	9.600E+01	6.047E-04	2.373E-06	2.804E+03	-4.694E+00	0.
3	1.440E+02	1.324E-03	1.498E-05	1.951E+03	-1.629E+01	0.
4	1.920E+02	2.230E-03	1.988E-05	1.970E+02	-3.464E+01	0.
5	2.400E+02	3.155E-03	1.928E-05	1.970E+02	-5.540E+01	0.
6	2.880E+02	3.836E-03	1.417E-05	-2.555E+03	-7.178E+01	0.
7	3.360E+02	3.933E-03	2.031E-06	-4.088E+03	-7.628E+01	0.
8	3.840E+02	3.095E-03	-1.745E-05	-9.739E+03	-7.628E+01	0.
9	4.320E+02	1.042E+03	-4.278E-05	-1.266E+04	6.572E+02	0.
10	4.800E+02	8.215E-04	-4.599E-06	1.909E+04	7.112E+02	0.
11	5.280E+02	8.215E-04	1.019E-04	5.325E+04	7.724E+02	0.
12	5.760E+02	5.713E-03	2.018E-04	9.983E+04	8.199E+02	0.
13	6.240E+02	1.923E-02	5.322E-04	1.278E+05	8.283E+02	0.
14	6.720E+02	4.501E-02	8.673E-04	1.650E+05	7.739E+02	0.
15	7.200E+02	8.664E-02	1.283E-03	1.980E+05	6.447E+02	0.
16	7.680E+02	1.673E-01	1.709E-03	2.229E+05	4.533E+02	0.
17	8.160E+02	2.293E-01	1.709E-03	2.364E+05	4.533E+02	-1.258E+03
18	8.640E+02	3.340E-01	2.182E-03	1.645E+05	-1.280E+03	-1.938E+03
19	9.120E+02	4.546E-01	2.511E-03	2.494E+05	-2.494E+03	0.
20	9.600E+02	4.546E-01	2.576E-03	3.274E+04	-2.454E+03	0.
21	1.008E+03	5.782E-01	2.576E-03	-9.742E+04	-2.250E+03	0.
22	1.056E+03	7.92E-01	2.382E-03	-2.149E+05	-2.250E+03	0.
23	1.104E+03	8.925E-01	2.382E-03	1.948E-03	-1.898E+03	0.
24	1.152E+03	7.880E-01	1.313E-03	-3.173E+05	-1.430E+03	0.
		8.491E-01	1.313E-03	-3.922E+05	-8.868E+02	0.
		5.288E-04	-4.373E+05	-3.123E+02	0.	0.
		8.745E-01	-3.459E-04	-4.507E+05	2.534E+02	0.
		8.579E-01	-1.247E-03	2.534E+02	0.	0.

25	1.200E+03	7.080E-01	-2.112E-03	-4.325E+05	7.750E+02	0.
26	1.248E+03	6.966E-01	-2.083E-03	-7.852E+05	1.224E+03	0.
27	1.296E+03	5.582E-01	-3.608E-03	-3.126E+05	1.579E+03	0.
28	1.344E+03	3.899E-01	-3.048E-03	-2.200E+05	1.823E+03	0.
29	1.392E+03	2.004E-01	-4.175E-03	-1.135E+05	1.944E+03	0.
30	1.440E+03	0.	-4.175E-03	-1.450E+08	3.855E+03	-1.948E+03
31	1.488E+03	-2.004E-01	0.	0.	0.	0.

TIME STA K = 24

J = 8 MONITOR STATIONS  
15 16 17

FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED

-7.518E-02 -2.391E-01 -2.430E-01 -2.520E-01

FOR 2 ITERATION, THERE ARE 29 STATIONS NOT CLOSED

-2.837E-02 -1.998E-01 -2.058E-01 -2.187E-01

FOR 3 ITERATION, THERE ARE 0 STATIONS NOT CLOSED

-2.837E-02 -1.998E-01 -2.058E-01 -2.187E-01

TIME STA K = 25

STA J	DIST	DEFL	SLOPE	WOM	SHEAR	SUP REACT
8	3.840E+02	-2.837E-02	-4.495E-04	-8.175E+04	-4.493E+01	1.135E+03
15	7.200E+02	-1.986E-01	-1.784E-04	1.404E+04	-9.609E+02	0.
16	7.680E+02	-2.058E-01	-1.802E-04	-5.965E+04	-1.551E+03	8.575E+02
17	8.160E+02	-2.187E-01	-2.495E-04	9.744E+03	1.419E+03	6.354E+02

TIME STA K = 24

J = 8 MONITOR STATIONS  
15 16 17

FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED

-3.069E-02 -3.255E-01 -4.270E-01 -5.527E-01

FOR 2 ITERATION, THERE ARE 0 STATIONS NOT CLOSED

-3.069E-02 -3.255E-01 -4.270E-01 -5.527E-01

TIME STA K = 24

STA J	DIST	DEFL	SLOPE	WOM	SHEAR	SUP REACT
8	3.840E+02	-3.069E-02	-2.731E-04	-3.985E+04	-8.745E+01	1.228E+03

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## THE AUTHORS

Jack Hsiao-Chieh Chan was a Research Engineer Assistant with the Center for Highway Research, The University of Texas at Austin. His experience includes work as a teaching assistant in Taiwan and work as a highway engineer with Howard, Needle, Tammon, and Bergendoff, Consulting Engineers, Kansas City, Missouri. His research interest is primarily in the area of discrete-element analysis in structural dynamics.



Hudson Matlock is a Professor of Civil Engineering at The University of Texas at Austin. He has a broad base of experience, including research at the Center for Highway Research and as a consultant for other organizations such as the Alaska Department of Highways, Shell Development Company, Humble Oil and Refining Company, Penrod Drilling Company, and Esso Production Research Company. He is the author of over 50 technical papers and 25 research reports and received the 1967 ASCE J. James R. Croes Medal. His primary areas of interest in research include (1) soil-structure interaction, (2) experimental mechanics, and (3) development of computer methods for simulation of civil engineering problems.

