

SIMULATION OF VEHICLE IMPACT WITH THE
TEXAS CONCRETE MEDIAN BARRIER
VOLUME I: TEST COMPARISONS AND PARAMETER STUDY

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Evaluation of the Roadway Environment by
Dynamic Analysis of the Interaction Between
the Vehicle, Passenger, and Roadway

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FOREWORD

The information contained herein was developed on Research Project 2-5-69-140 entitled "Evaluation of the Roadside Environment by Dynamic Analysis of the Interaction Between the Vehicle, Passenger, and Roadway" which is a cooperative research study sponsored jointly by the Texas Highway Department and the U. S. Department of Transportation, Federal Highway Administration.

Basically, the objectives of the study are to apply mathematical simulation techniques in determining the dynamic behavior of automobiles and their occupants when in collision with various roadside objects or when traversing curves in the road, shoulders, or other situations. It is a continuing study, having been initiated in September 1968.

As part of the first year's work, the computer program HVOSM (formerly known as CALSVA) was obtained from Cornell Aeronautical Laboratory and made operational on the IBM 360 computer facilities at Texas A&M University. In adapting the program, additions and modifications were made which increased its flexibility and usefulness. These changes and the input requirements of the program are documented in Research Report 140-1.

The primary emphasis of the second year's work was the development of an analytical model which predicts the dynamic response of an automobile's occupant in three-dimensional space. Research Report 140-2 presents the derivation of the occupant model, a validation study, and a description of computer input data for determining the occupant's response.

In the 1970-71 year the emphasis was on application of HVOSM to specific roadway design problems. Research Report 140-3 describes an investigation of the *traffic-safe* characteristics of different sloping culvert grate configurations. Criteria are presented for designing a *traffic-safe* sloping grate. Research Report 140-4 describes the development of criteria from which the need and location of guardrail on embankments can be determined.

Volume I of this report describes a comparison of full-scale test results of the Texas Concrete Median Barrier with simulated results as computed by a modified version of HVOSM. Also contained in Volume I is a parametric study of the performance characteristics of the CMB for various vehicle encroachment or impact conditions. Volume II contains the computer input for all runs made and some sample output.

The contents of this report reflect the views of the authors who are responsible for the facts and the accuracy of the data presented herein. The contents do not necessarily reflect the official views or policies of the Federal Highway Administration. This report does not constitute a standard, specification, or regulation.

ABSTRACT

Key Words: Accidents, Automobile, Barrier Impact, Computer Simulation, Concrete Median Barrier, Injury Level, Math Modeling, Safety, Severity Index, Vehicle Redirection, Vehicle Simulation

The Highway-Vehicle-Object Simulation Model (HVOSM), a computer program developed at Cornell Aeronautical Lab. (CAL), has been modified with generous help from CAL and used to successfully simulate a vehicle impacting the Texas Concrete Median Barrier (CMB) at speeds ranging from 50 to 80 m.p.h. and angles ranging from 5 to 25 degrees.

The Texas CMB (a New Jersey type design) was impacted by a 4000 lb. sedan at angles of 7, 15 and 25 degrees at 60 m.p.h. The results of these full scale tests were closely approximated by the modified HVOSM.

Comparison of simulation and test results are presented both visually and quantitatively in the form of computer generated drawings of the vehicle during impact alongside corresponding frames from the high speed film and plots relating predicted and measured accelerometer readings.

Following the successful simulation of the full scale tests, a parameter study on impact conditions was conducted. Using the modified HVOSM, a 4780 lb. vehicle impacting the CMB was simulated for speeds of 50, 70 and 80 m.p.h. at angles of 5, 10 and 15 degrees for each of those speeds.

For speeds less than 70 m.p.h., the results were in line with

findings of other researchers in testing similar barriers. However, it was concluded that for impact speeds of 70 m.p.h. and greater in conjunction with impact angles of 15 degrees and greater, automobile rollover can be expected.

The results of all simulated impacts with the Texas CMB are presented graphically with regard to a severity index which quantifies the severity of each crash. This index is based on vehicle accelerations.

SUMMARY

In this study, the performance of the Texas Concrete Median Barrier (CMB) was evaluated from the standpoint of severity of impact, vehicle exit angle, maximum roll angle, and maximum pitch angle for a wide range of vehicle encroachment conditions. A modified version of the Highway-Vehicle-Object Simulation Model (HVOSM) was used to simulate a 4780 lb. automobile impacting the CMB at speeds of 50, 70, and 80 m.p.h. at angles of 5, 10 and 15 degrees for each of those speeds. These parameter studies were preceded by a validation phase where full-scale tests on the CMB were successfully simulated with the HVOSM.

In summary the major findings are:

1. that the Texas CMB performs well with respect to exit angle and maximum roll and pitch angles at least up to 80 m.p.h. (highest speed considered) for impact angles of less than 10 degrees.
2. that serious injuries could occur at 70 m.p.h. for impact angles greater than 7.5 degrees (see Figure 25).
3. that rollover can be expected for speeds greater than 70 m.p.h. at angles greater than 15 degrees. Although these encroachment conditions would probably result in serious or fatal injury even without the rollover, this creates a hazard for other motorists as well.

IMPLEMENTATION STATEMENT

There is very little disagreement (if any) among Highway Engineers on the superior attributes of the Concrete Median Barrier (New Jersey or Texas type) for use in narrow medians of roadways carrying high traffic volume such as in urban areas. Tests have shown that impacting the CMB at low angles is safe and that maintenance is virtually non existant.

The results of this study provide additional detailed information which will aid the highway designer in evaluating an existing roadway for possible installation of a CMB or designing a new roadway to accomodate a CMB. For an automobile colliding with the Texas CMB, Figure 25 of this report gives combinations of speed and impact angles for which serious injury to the occupant is unlikely.

ACKNOWLEDGEMENTS

This research was sponsored by the Texas Highway Department (THD) in cooperation with the U.S. Department of Transportation, Federal Highway Administration (FHWA).

The authors sincerely thank Messrs. Norman J. Deleys and Raymond R. McHenry of Cornell Aeronautical Laboratories for providing generous professional assistance in the form of copies of their previous work on modifying their model, the Highway-Vehicle-Object Simulation Model (HVOSM), to include the effect of vehicle structural hard points in simulating automobile-barrier crashes.

Thanks are also extended to Messrs. John F. Nixon and David Hustace (THD) and Mr. Edward V. Kristaponis (FHWA) for their cooperation in meeting the uncertainties of this particular research effort and their patience in awaiting the reported results.

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I. INTRODUCTION

Evaluation of barrier systems usually include full-scale vehicle crash tests. These tests are often quite expensive and many man-hours are required for all phases of the test program. A more ideal method of studying the performance of barriers is by computer simulation.

The original version of the Highway-Vehicle-Object-Simulation Model (HVOSM)* (1,2) was capable of predicting automobile behavior for impact with certain types of barriers, provided the automobile crush was moderate (12 to 18 inches). The types of barrier systems which can be studied with the HVOSM are those whose lateral resistance to vehicle penetration is independent of the longitudinal position of the vehicle contact. The Texas Highway Department (THD) concrete median barrier (CMB), being a rigid barrier, falls within this category. To study barrier impacts in which large automobile crush occurs, the HVOSM was modified to include *hard points* within the automobile structure. These hard points simulate the effects caused when very stiff automobile members are encountered, such as the engine, a frame member, or a wheel assembly. Basic details of HVOSM are summarized in Appendix A and a description of the hard point modifications is given in Appendix B.

In Chapter II, comparisons are made between experimental data from full-scale crash tests of the Texas CMB and simulated results from TTI's modified version of HVOSM (including hard points). The

*Formerly known as CALSVA.

crash test data were obtained from another THD sponsored research program (3). In general, good correlation exists between simulated and experimental results.

A parametric study of the Texas concrete median barrier was conducted using HVOSM and the results are described in Chapter III. The parametric study was used to determine the barrier's performance characteristics for a range of vehicle encroachment conditions. Factors used in measuring the performance were the vehicle's exit angle, maximum pitch and roll angle, and a severity index which quantified the impact severity.

II. COMPARISON OF EXPERIMENTAL DATA WITH PREDICTIONS BY HVOSM

General

It was felt by sponsors and researchers alike that a good correlation between simulation and testing was a prerequisite to conducting parametric studies of the Texas CMB with the HVOSM. The crash test data used in the comparisons were obtained from another THD sponsored research program (3) and the results were available to this study at no extra cost.

The three tests used in the comparisons consisted of passenger cars (roughly 4000 lbs. in weight) being towed into a full scale model of the CMB, model I-70 (designated CMBI-70), at approximately 60 mph for impact angles of 7, 15, and 25 degrees. The CMBI-70 designation is used when illumination poles are placed atop the barrier. The exterior dimensions of the barrier are the same however, whether illumination is used or not. Reference to the barrier will henceforth be the CMB. Accelerometers were mounted to the structural framework of the vehicles, at the locations described in APPENDIX C (Figures C1 and C2). They were oriented to measure the lateral and the longitudinal components of vehicle acceleration. Vehicle motion was recorded on high speed film from rear, side, and overhead views. These films were used to determine the automobile's speed and angle at impact and to provide a comparison of the vehicle's simulated and actual motion.

The computer simulation used was a modified version of the HVOSM, as discussed in Appendix B. The modification made was the addition of

NOTE: Dotted Outline Shows the Actual Shape of the CMBI-70 and Solid Lines Show the Shape Used for Simulation.

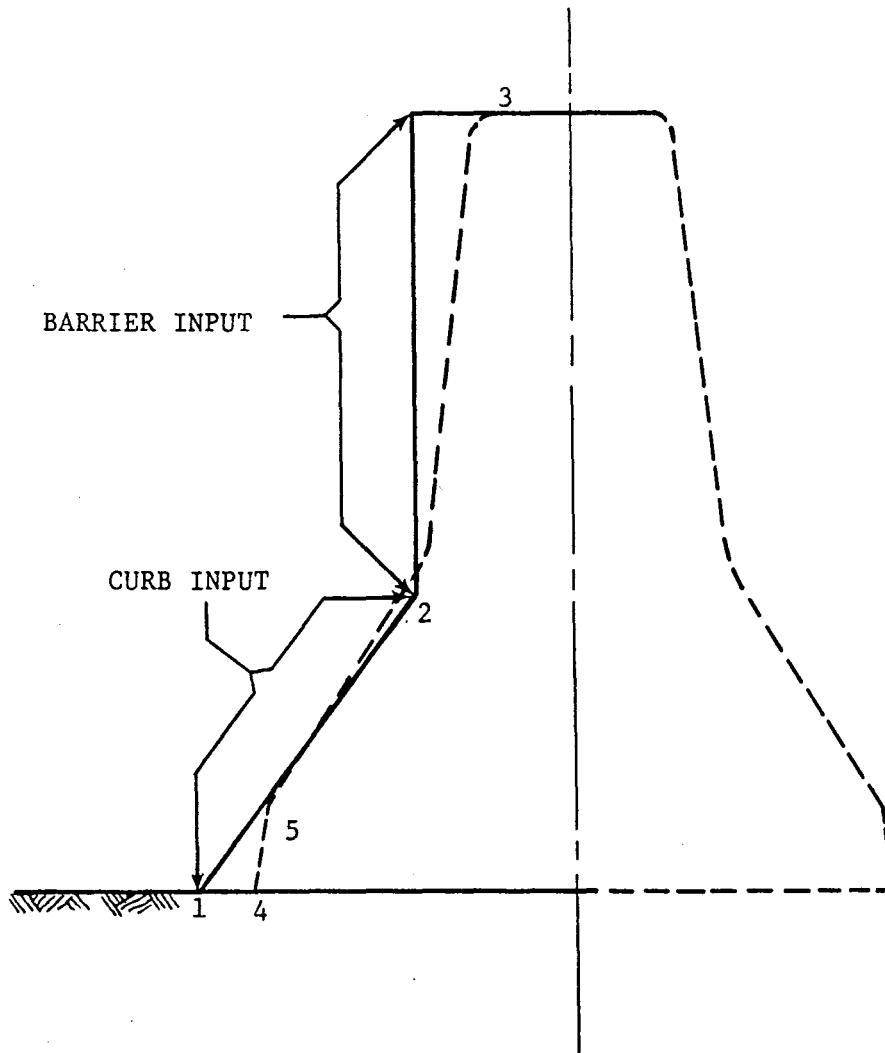


FIGURE 1. IDEALIZATION OF THE CMBI-70 FOR COMPUTER SIMULATION.

vehicle structural hard points to the existing sheet metal crushing capability, to account for stiffer portions of the vehicle (frame members, wheels, etc.) impacting the barrier.

Due to limitations within HVOSM, it was necessary to describe the CMB by a combination (or superposition) of the program's "curb impact" and "barrier impact" capabilities. As shown in Figure 1, the sloping face of the barrier was simulated as a curb (line 1-2) and the upright face was simulated as a vertical rigid barrier (line 2-3). In the simulation tire-curb interaction is accounted for but tire-rigid barrier interaction is not. However, since good comparisons between simulations and tests were obtained (for both kinematics and accelerations), it would appear that tire contact with the upright face is of secondary importance. For the same reason, the omission of slope 4-5 was apparently not detrimental to the simulation. This is not surprising in view of the relative dimensions of the tires to the length of line 4-5 and the high impact speeds. Likewise, idealizing the upright face as vertical rather than sloped a few degrees proved to be an adequate representation. For the shallow angle impact of 7° , the whole concrete median barrier could have been defined as one high curb (1), but for the sake of uniformity all cases were defined as described above.

Data corresponding to the CMB and the test vehicles, such as vehicle weight, barrier and vehicle dimensions, impact speed and angle, etc., were read into the HVOSM program. All computer input data are described in Volume II of this report (4).

Plots of the predicted and measured acceleration components at

points corresponding to the locations of accelerometers in the actual vehicle were made. Drawings of the simulated impacts were generated using a computer program (5). The program produces a perspective drawing of the vehicle and barrier at selected times, utilizing vehicle position as determined from the HVOSM. These line drawings were then compared with corresponding photographs taken from the high speed photography of the test.

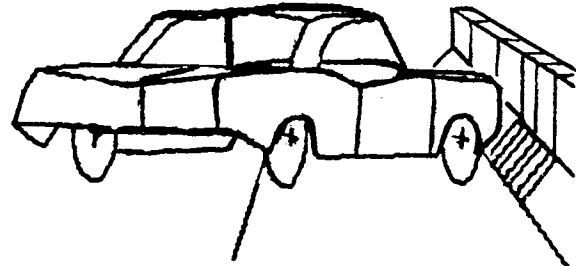
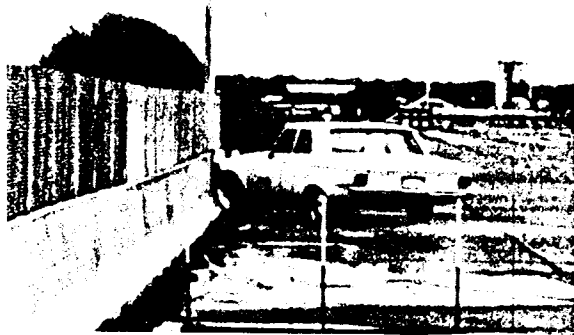
Results of the Comparisons

The results of the comparisons are given both pictorially and graphically in Figures 2 through 23. It should be noted that the test results are for a vehicle impacting on the left (driver) side of the vehicle, while the simulation impacts the car on the opposite side (passenger side). For this reason, the results are expressed in terms of "impact side" and "free side" of the vehicle, the free side being the side of the vehicle away from the wall. Time of initial impact was taken as zero and the times shown on the figures are with respect to impact time. Comparisons were stopped when the vehicle lost contact with the barrier.

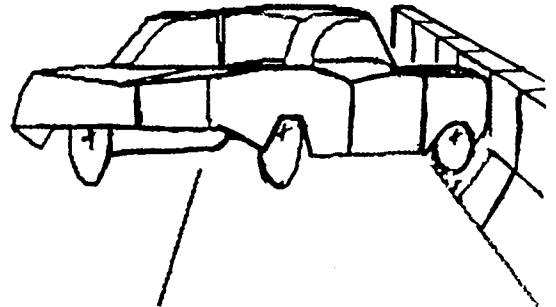
The comparison of test photographs to computer drawings in Figures 2 through 11 is very good for all three tests. Comparing the wheel positions, height of climb, and relative position of vehicle body to the ground shows that the simulation accurately computed the motions of the test vehicle. The small differences observable

TEST

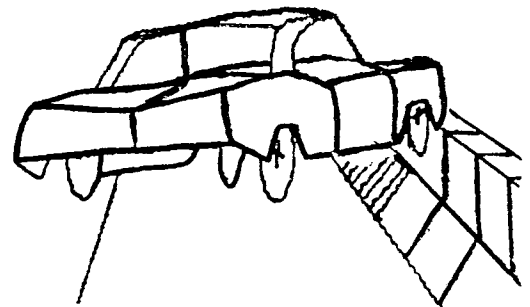
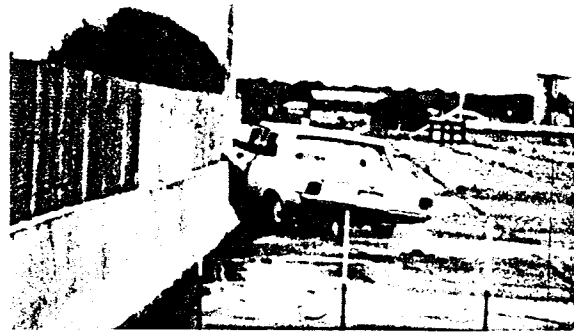
SIMULATION



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TIME = 0.035 SEC.

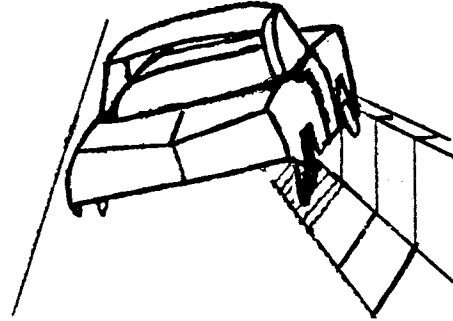
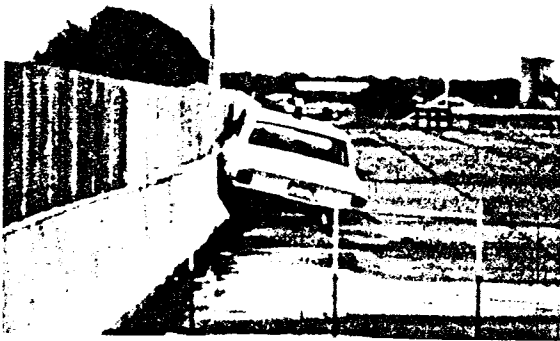


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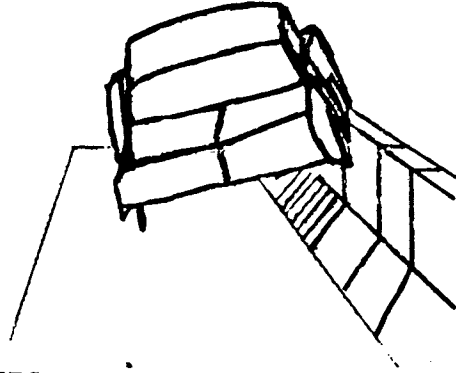
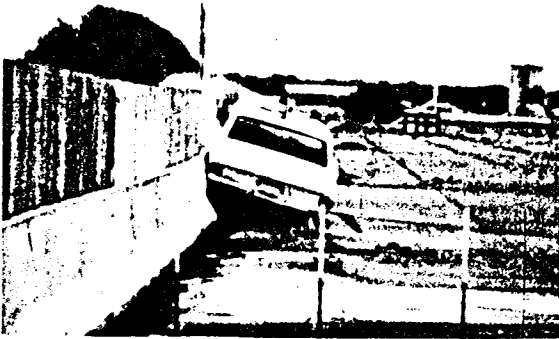
FIGURE 2.- COMPARISON OF COMPUTER SIMULATION WITH TEST RESULTS FOR 4000 LB. VEHICLE IMPACTING CONCRETE MEDIAN BARRIER AT 63 MPH AT 25 DEGREES; TIME = 0.000 SEC. TO 0.085 SEC.

TEST

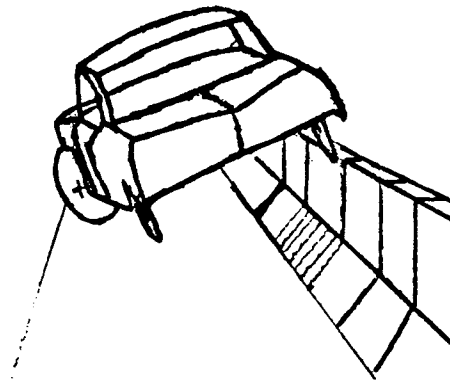
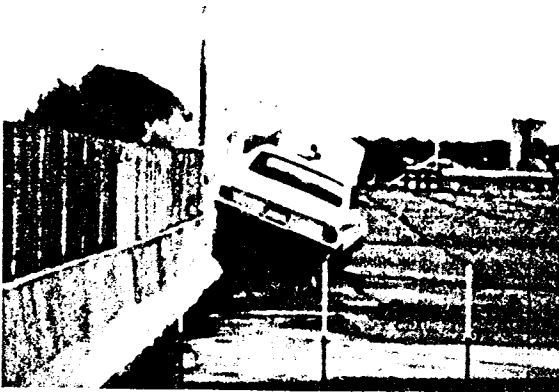
SIMULATION



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TIME = 0.201 SEC.

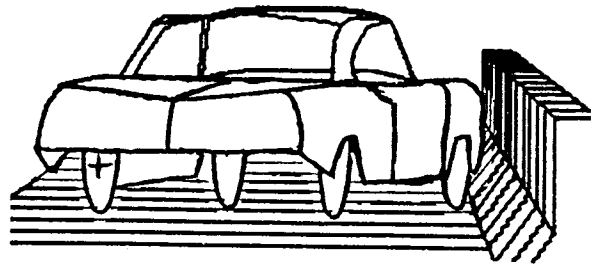


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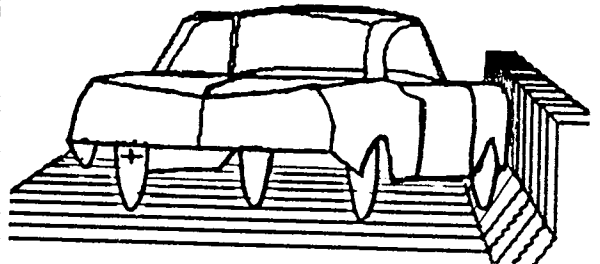
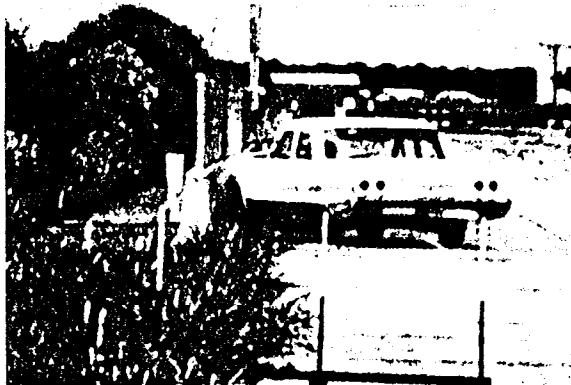
FIGURE 3. - COMPARISON OF COMPUTER SIMULATION WITH TEST RESULTS FOR 4000 LB. VEHICLE IMPACTING CONCRETE MEDIAN BARRIER AT 63 MPH AT 25 DEGREES; TIME = 0.156 SEC. TO 0.321 SEC.

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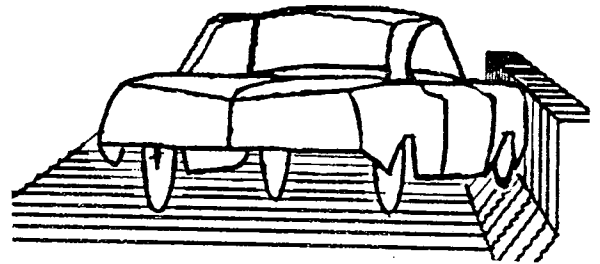
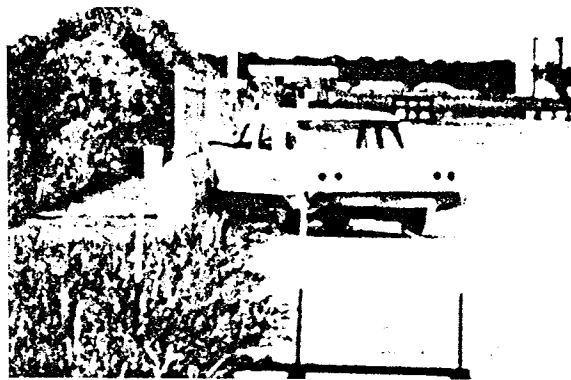
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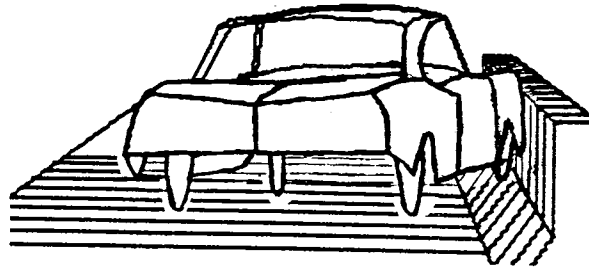
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FIGURE 4. - COMPARISON OF COMPUTER SIMULATION WITH TEST RESULTS FOR 4210 LB. VEHICLE IMPACTING CONCRETE MEDIAN BARRIER AT 59.6 MPH AT 15 DEGREES; TIME = 0.000 SEC. TO 0.060 SEC.

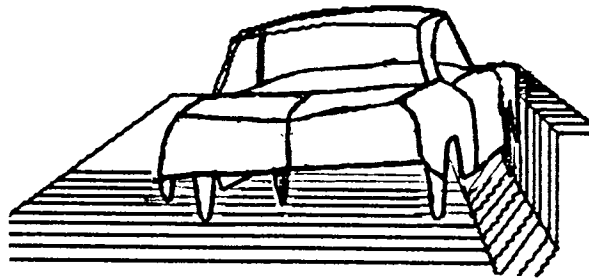
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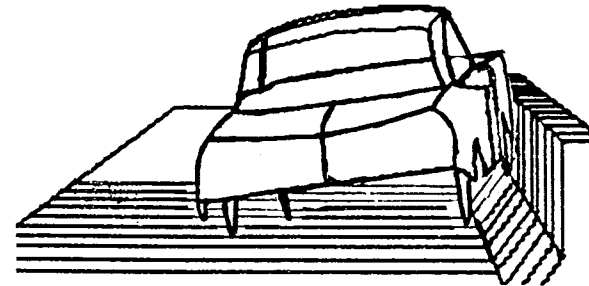
SIMULATION



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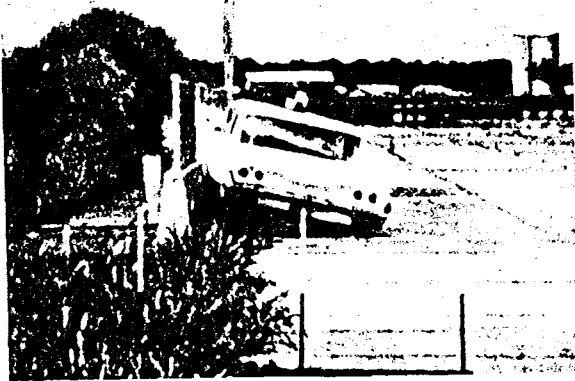
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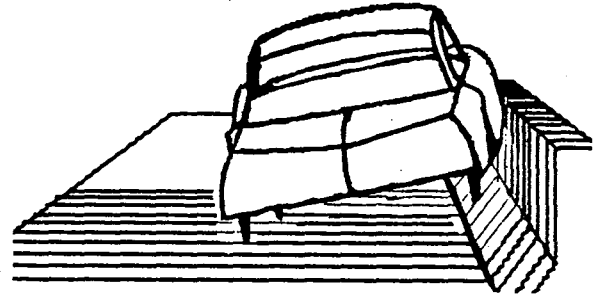
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FIGURE 5.- COMPARISON OF COMPUTER SIMULATION WITH TEST RESULTS FOR 4210 LB. VEHICLE IMPACTING CONCRETE MEDIAN BARRIER AT 59.6 MPH AT 15 DEGREES; TIME = 0.085 SEC. TO 0.135 SEC.

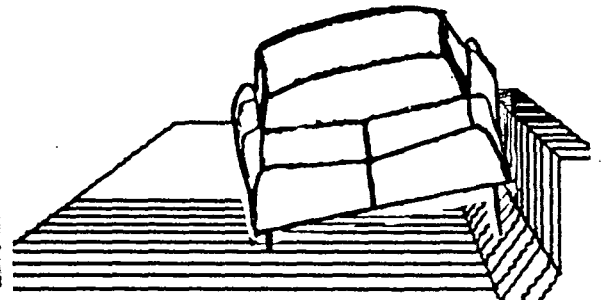
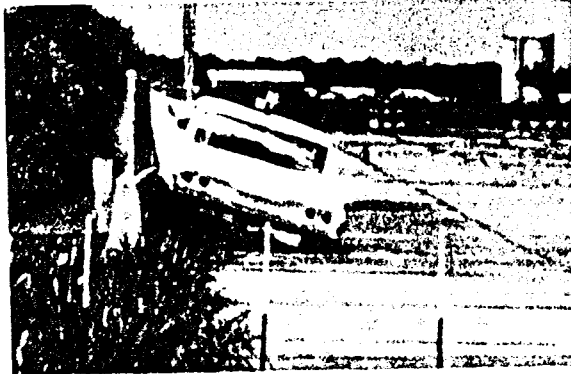
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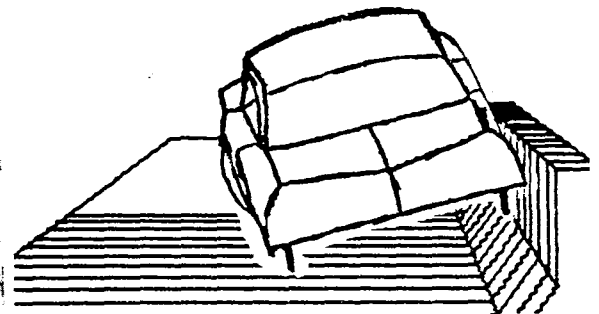
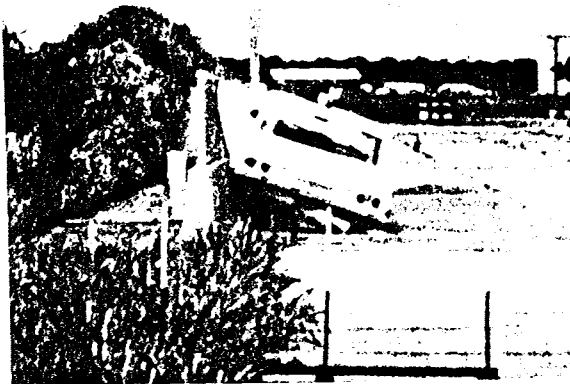
SIMULATION



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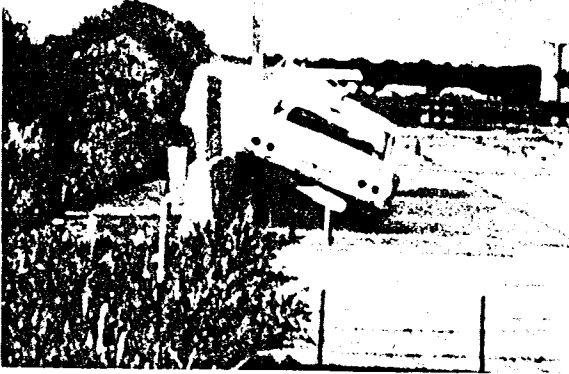
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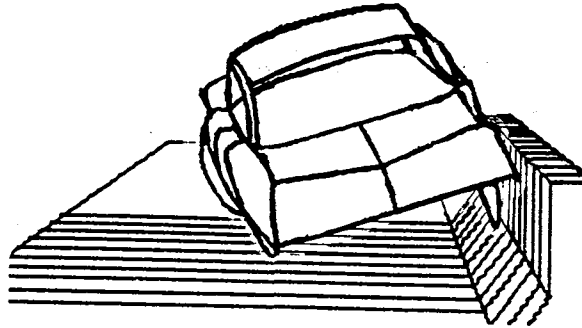
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FIGURE 6.- COMPARISON OF COMPUTER SIMULATION WITH TEST RESULTS FOR 4210 LB. VEHICLE IMPACTING CONCRETE MEDIAN BARRIER AT 59.6 MPH AT 15 DEGREES; TIME = 0.160 SEC. TO 0.210 SEC.

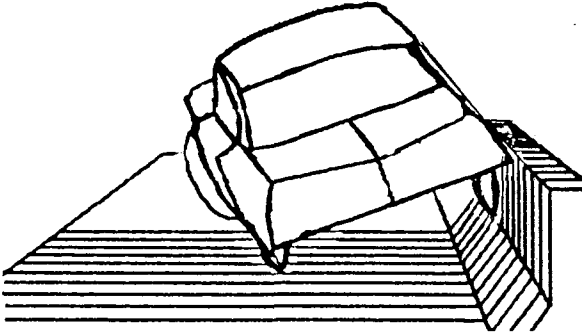
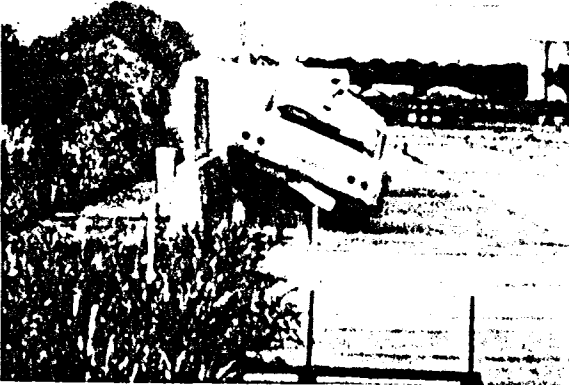
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SIMULATION



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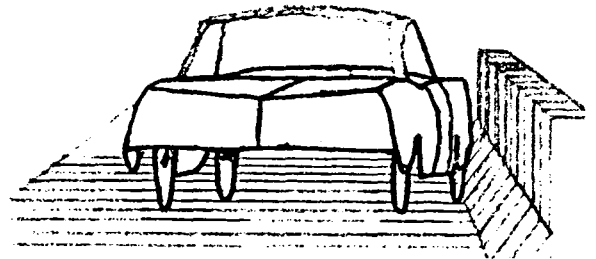
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FIGURE 7.- COMPARISON OF COMPUTER SIMULATION WITH TEST RESULTS FOR 4210 LB. VEHICLE IMPACTING CONCRETE MEDIAN BARRIER AT 59.6 MPH AT 15 DEGREES; TIME = 0.285 SEC. TO 0.335 SEC.

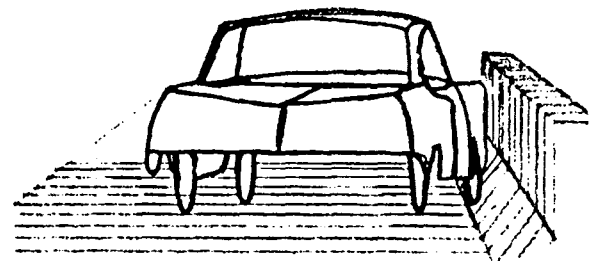
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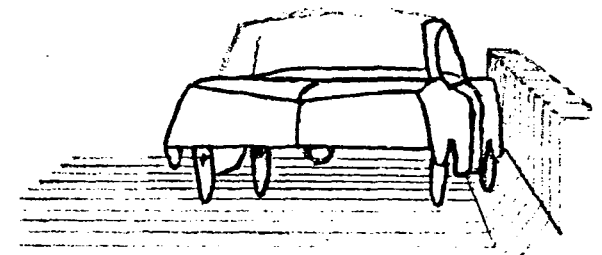
SIMULATION



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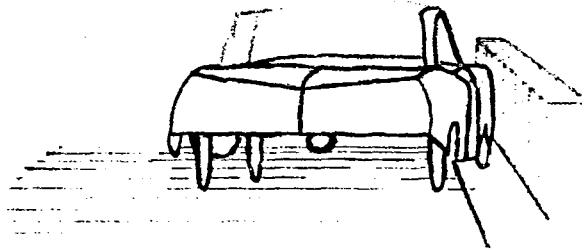
TIME = 0.050 sec.

FIGURE 8. - COMPARISON OF COMPUTER SIMULATION WITH TEST RESULTS FOR 4210 LB. VEHICLE IMPACTING CONCRETE MEDIAN BARRIER AT 61.9 MPH AT 7 DEGREE; TIME = 0.000 SEC. TO 0.050 SEC.

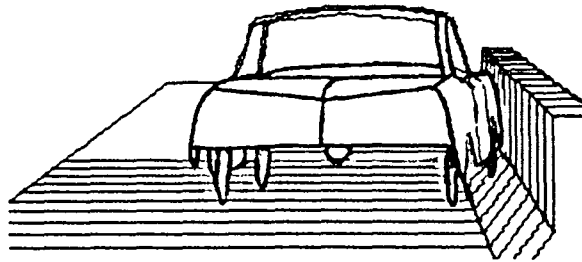
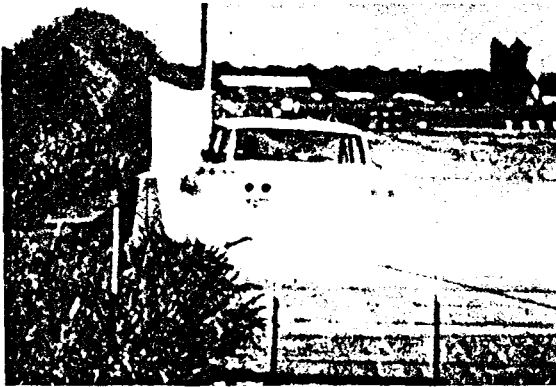
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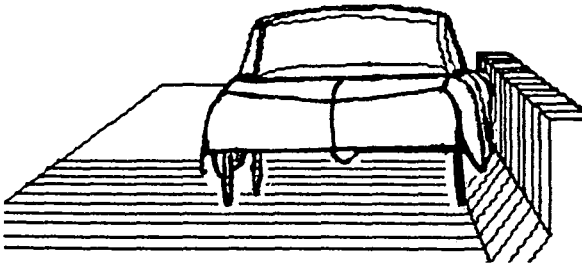
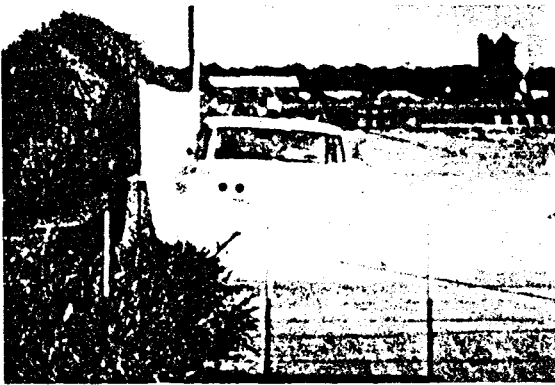
SIMULATION



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TIME = 0.100 SEC.

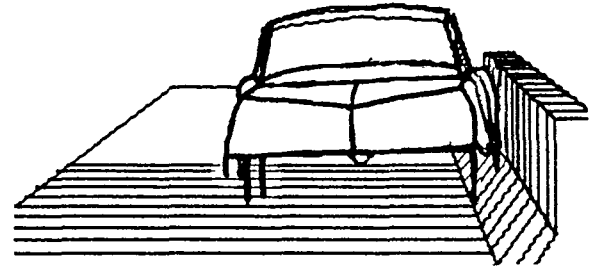


TIME = 0.125 SEC.

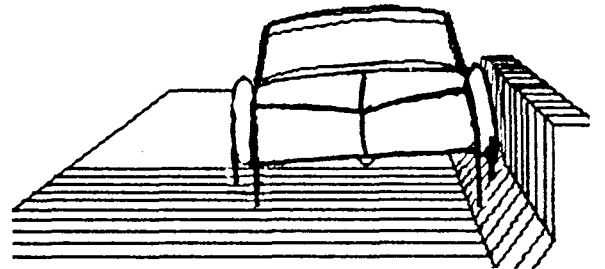
FIGURE 9. - COMPARISON OF COMPUTER SIMULATION WITH TEST RESULTS FOR 4210 LB. VEHICLE IMPACTING CONCRETE MEDIAN BARRIER AT 61.9 MPH AT 7 DEGREES; TIME = 0.075 SEC. TO 0.125 SEC.

TEST

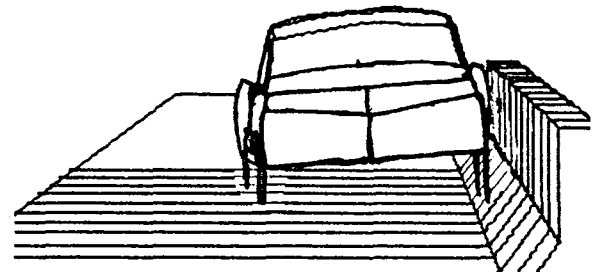
SIMULATION



TIME = 0.150 SEC.



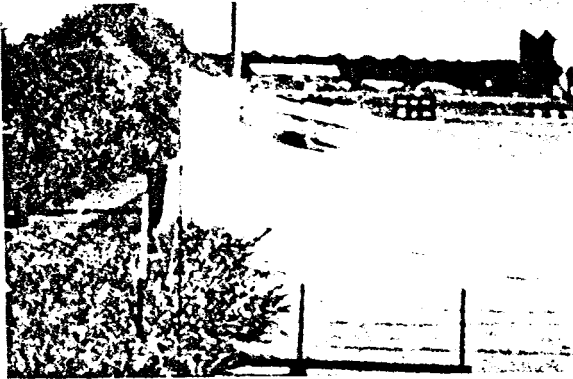
TIME = 0.175 SEC.



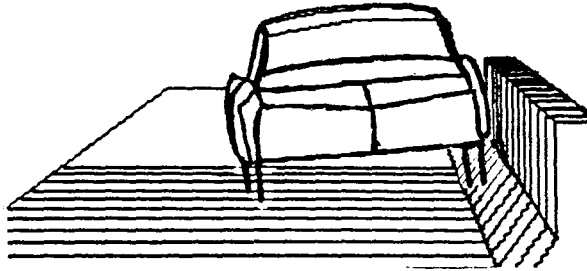
TIME = 0.200 SEC.

FIGURE 10- COMPARISON OF COMPUTER SIMULATION WITH TEST RESULTS FOR 4210 LB. VEHICLE IMPACTING CONCRETE MEDIAN BARRIER AT 61.9 MPH AT 7 DEGREES; TIME = 0.150 SEC. TO 0.200 SEC.

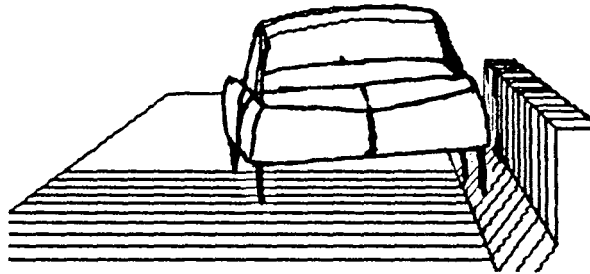
TEST



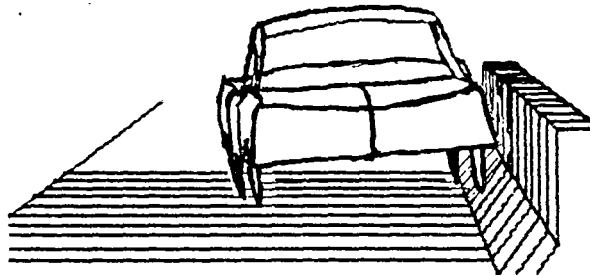
SIMULATION



TIME = 0.225 SEC.



TIME = 0.250 SEC.



TIME = 0.275 SEC.

FIGURE 11.- COMPARISON OF COMPUTER SIMULATION WITH TEST RESULTS FOR 4210 LB. VEHICLE IMPACTING CONCRETE MEDIAN BARRIER AT 61.9 MPH AT 7 DEGREES; TIME = 0.225 SEC. TO 0.275 SEC.

between the positions of simulated and actual vehicles is largely attributable to a standard automobile used for all the computer drawings, which was not necessarily of the same dimensions as the actual test vehicles. A second noticeable discrepancy is the appearance of the simulated vehicle to penetrate the barrier in a few instances. The program which produces the computer drawing cannot show sheet metal crushing, although it is accounted for in the HVOSM.

The comparisons of simulated and measured acceleration components are given in Figures 12 through 15 for the 63 mph and 25 degree impact and in Figure 16 through 19 for the 59.6 mph and 15 degree impact. Taken at face value, these two comparisons could be considered good, whereas the comparison of accelerations for the 61.9 mph and 7 degree impact (Figures 20 through 23) could only be considered poor. Fortunately, the discrepancies occurring in all three cases, whether slight or major, can be explained in terms of two basic differences between the actual and simulated vehicles:

1. The actual vehicle structure is comprised of structural subassemblies each possessing its own vibrational characteristics (natural frequencies and damping). However, the simulated vehicle structure (wheels and suspension systems excluded) is a rigid mass which is undamped and free of natural frequencies of vibration. Therefore, the actual accelerometers will respond to those structural vibrations which do not contribute to vehicle redirection and accordingly, which

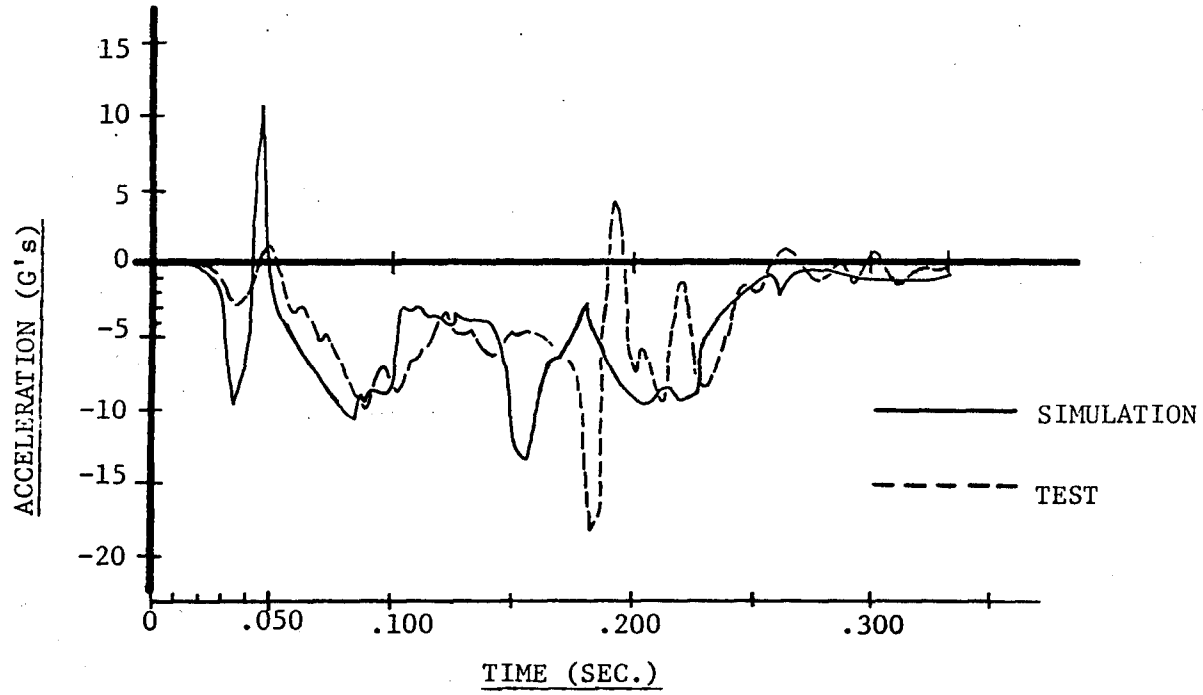


FIGURE 12.- LATERAL ACCELERATION ON IMPACT SIDE OF 4000 LB. VEHICLE IMPACTING CONCRETE MEDIAN BARRIER AT 63 MPH AT 25 DEGREES.

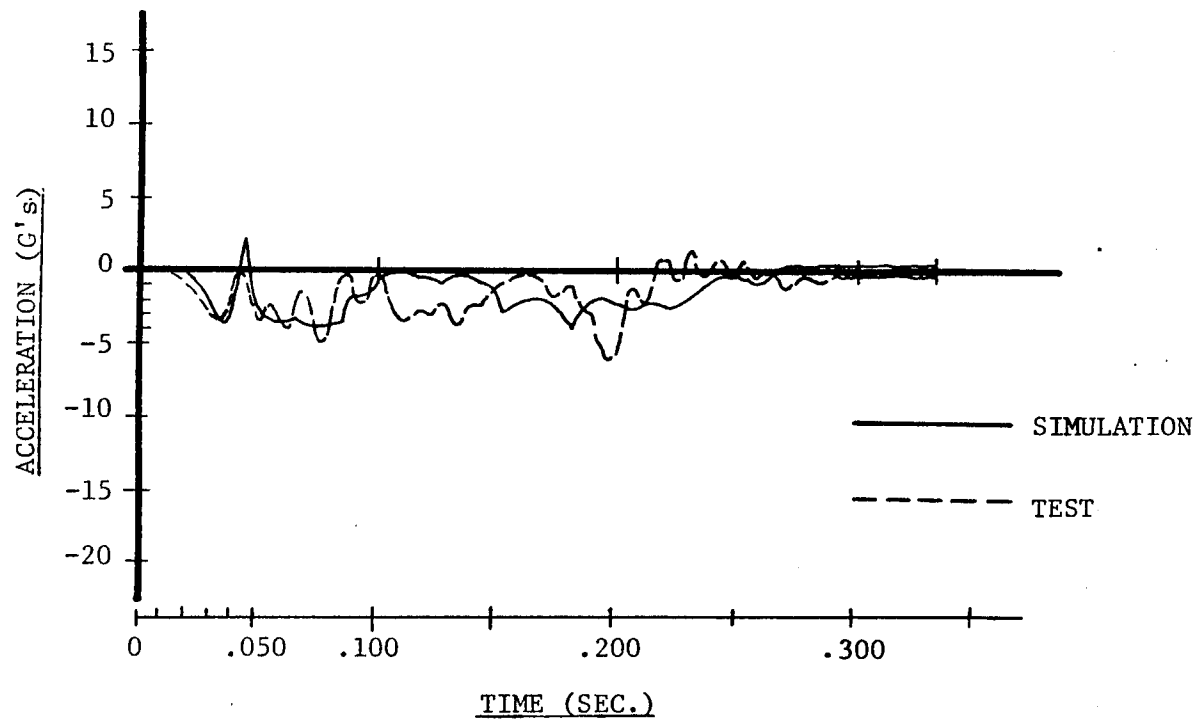


FIGURE 13.- LONGITUDINAL ACCELERATION ON IMPACT SIDE OF 4000 LB. VEHICLE IMPACTING CONCRETE MEDIAN BARRIER AT 63 MPH AT 25 DEGREES.

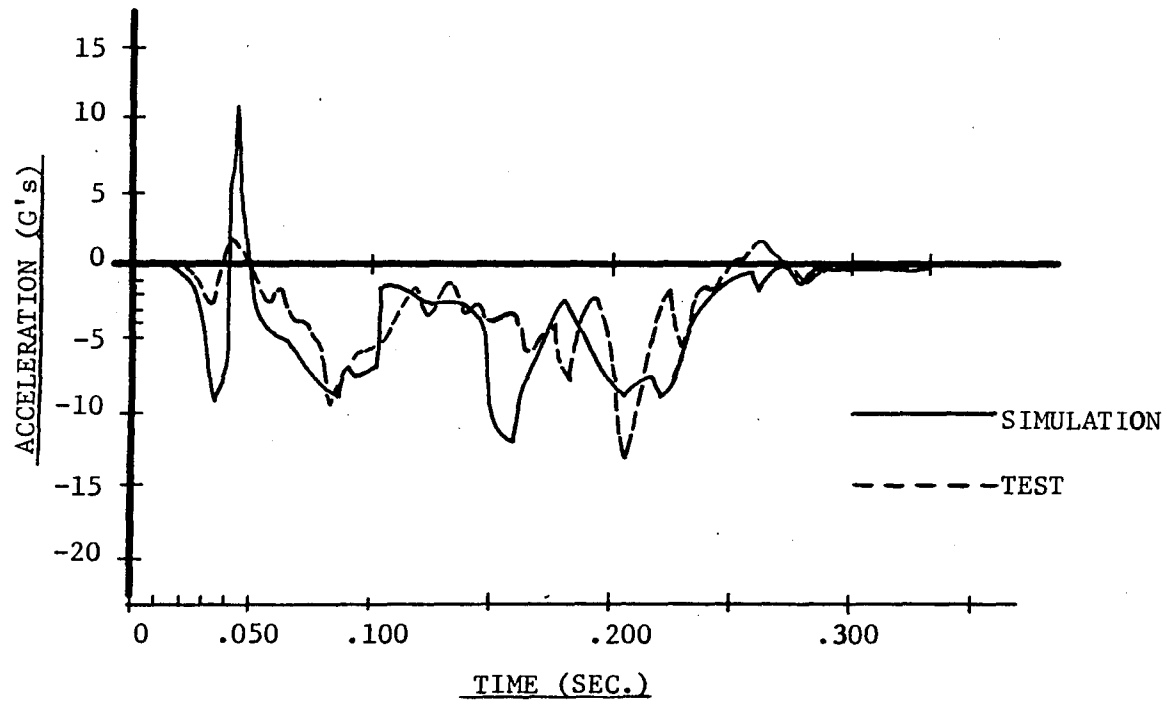


FIGURE 14. - LATERAL ACCELERATION ON FREE SIDE OF 4000 LB. VEHICLE IMPACTING CONCRETE MEDIAN BARRIER AT 63 MPH AT 25 DEGREES.

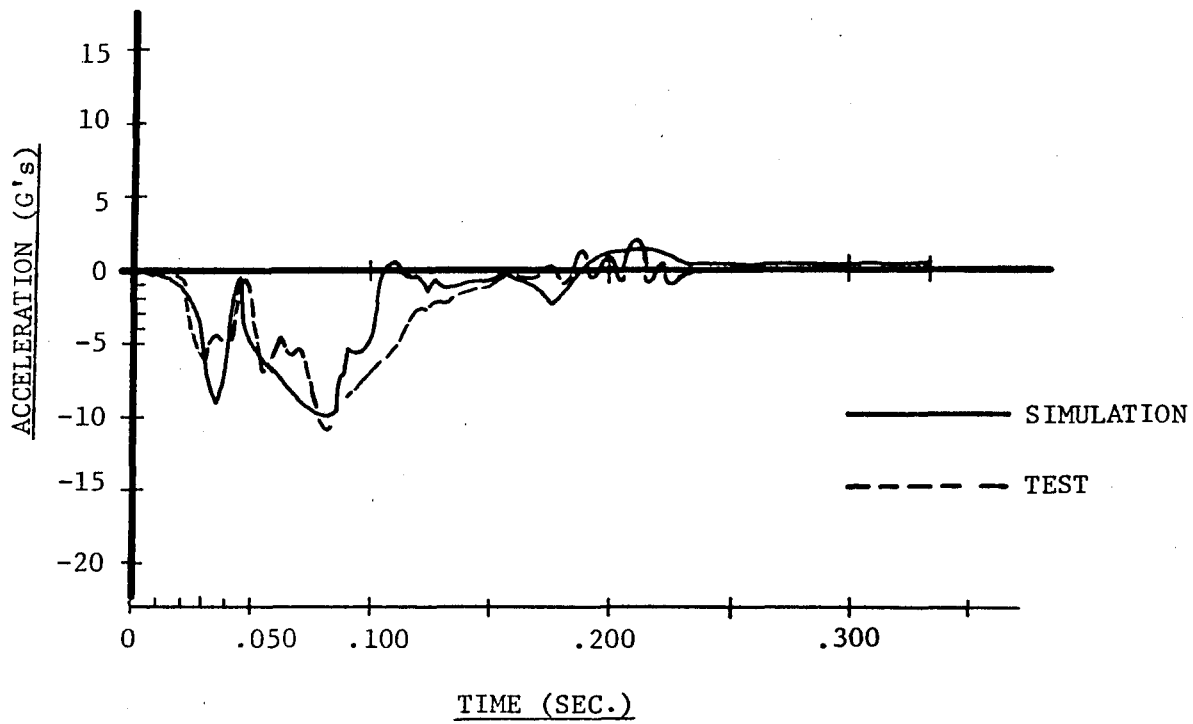


FIGURE 15.- LONGITUDINAL ACCELERATION ON FREE SIDE OF 4000 LB. VEHICLE IMPACTING CONCRETE MEDIAN BARRIER AT 63 MPH AT 25 DEGREES.

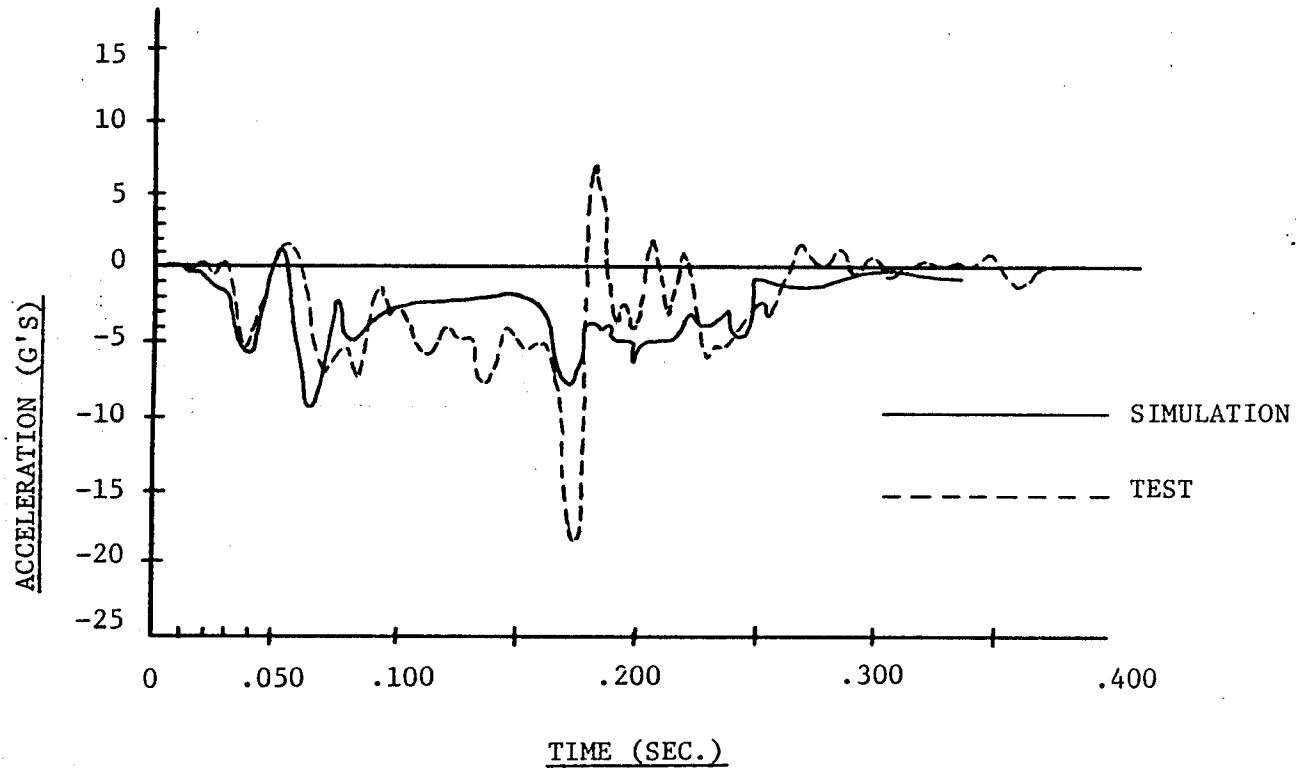


FIGURE 16- LATERAL ACCELERATION ON IMPACT SIDE OF 4210 LB. VEHICLE IMPACTING CONCRETE MEDIAN BARRIER AT 59.6 MPH AT 15 DEGREES.

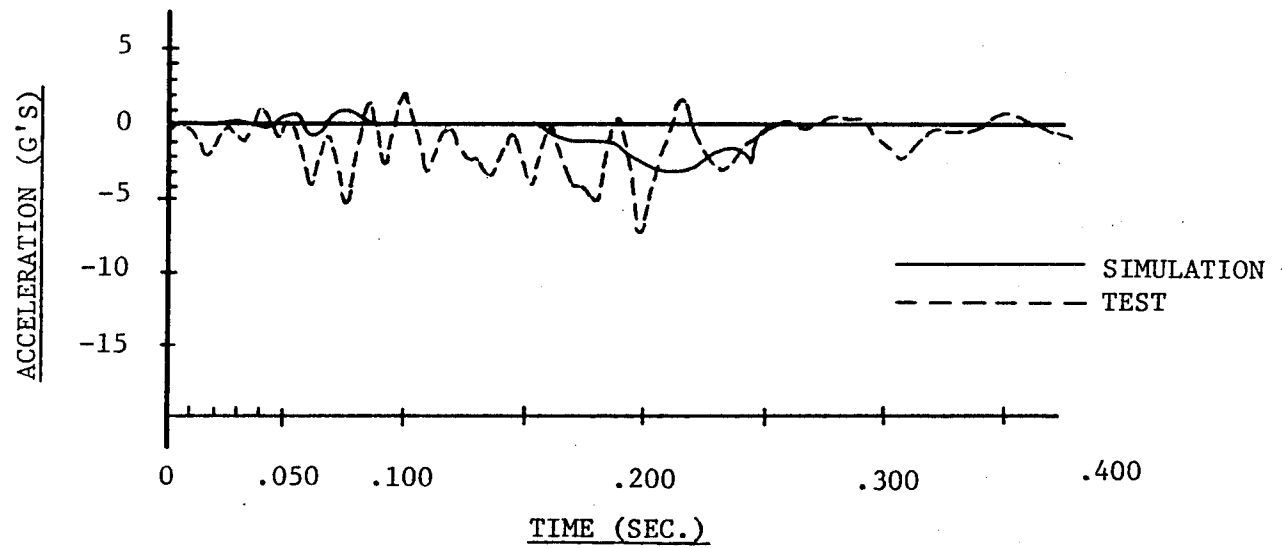


FIGURE 17 - LONGITUDINAL ACCELERATION ON IMPACT SIDE OF 4210 LB. VEHICLE IMPACTING CONCRETE MEDIAN BARRIER AT 59.6 MPH AT 15 DEGREES.

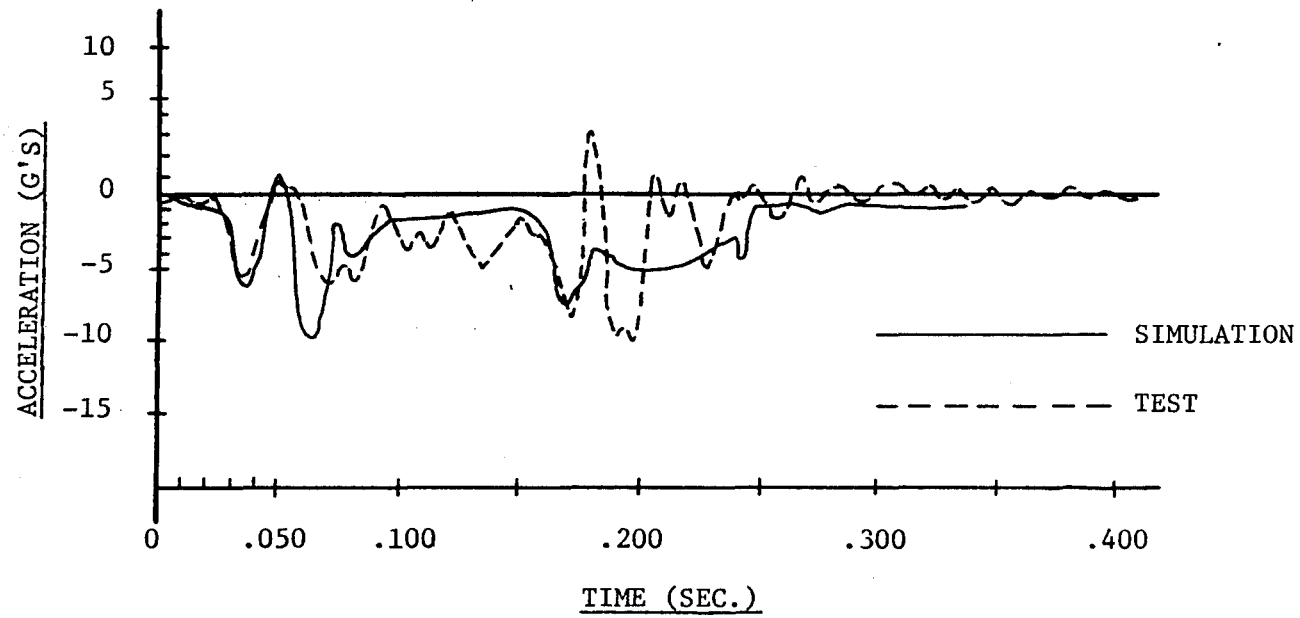


FIGURE 18 - LATERAL ACCELERATION ON FREE SIDE OF 4210 LB. VEHICLE IMPACTING CONCRETE MEDIAN BARRIER AT 59.6 MPH AT 15 DEGREES.

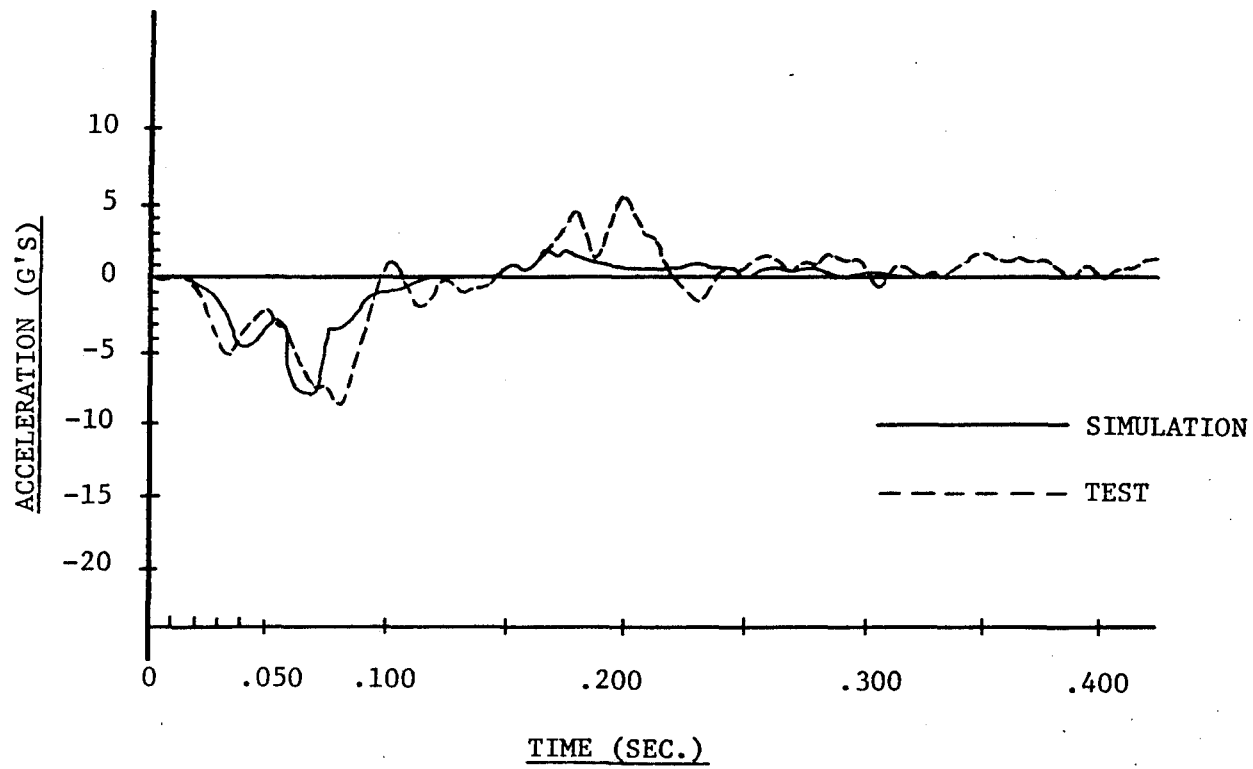


FIGURE 19 - LONGITUDINAL ACCELERATION ON FREE SIDE OF 4210 LB. VEHICLE IMPACTING CONCRETE MEDIAN BARRIER AT 59.6 MPH AT 15 DEGREES.

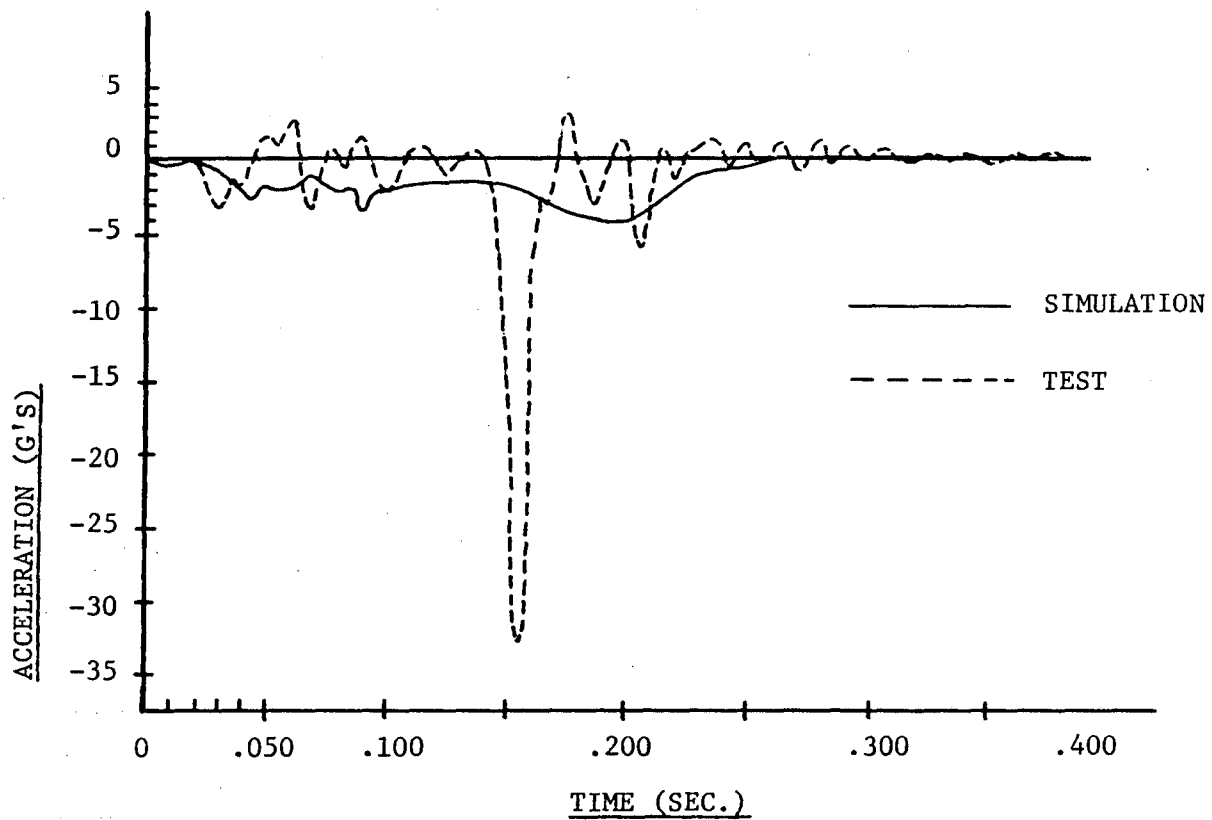


FIGURE 20 - LATERAL ACCELERATION ON IMPACT SIDE OF 4210 LB. VEHICLE IMPACTING CONCRETE MEDIAN BARRIER AT 61.9 MPH AT 7 DEGREES.

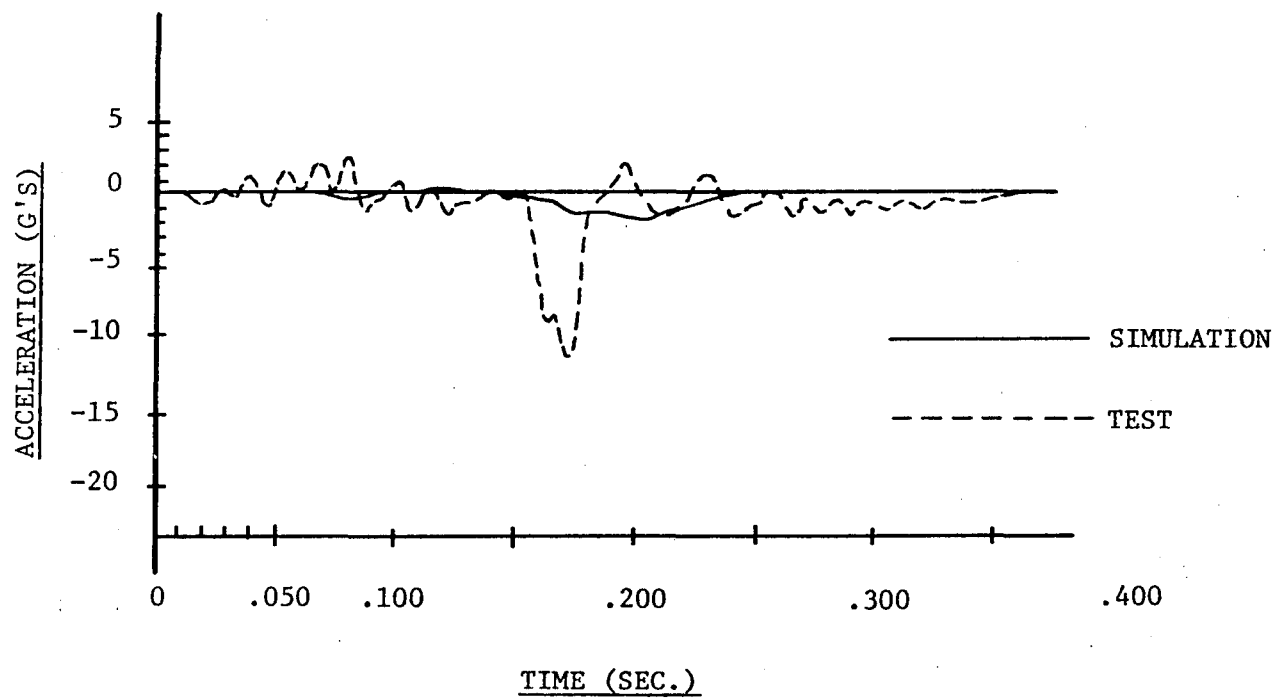


FIGURE 21 - LONGITUDINAL ACCELERATION ON IMPACT SIDE OF 4210 LB. VEHICLE IMPACTING CONCRETE MEDIAN BARRIER AT 61.9 MPH AT 7 DEGREES.

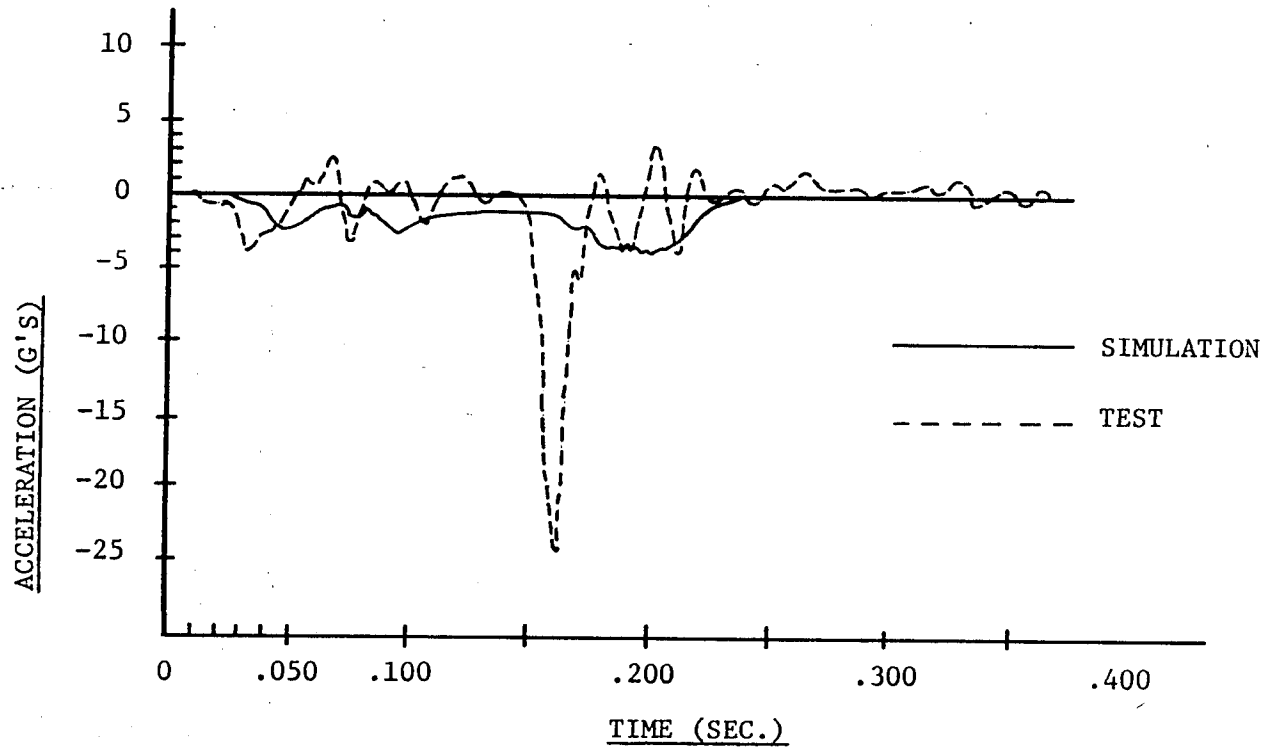


FIGURE 22 - LATERAL ACCELERATION ON FREE SIDE OF 4210 LB. VEHICLE IMPACTING CONCRETE MEDIAN BARRIER AT 61.9 MPH AT 7 DEGREES.

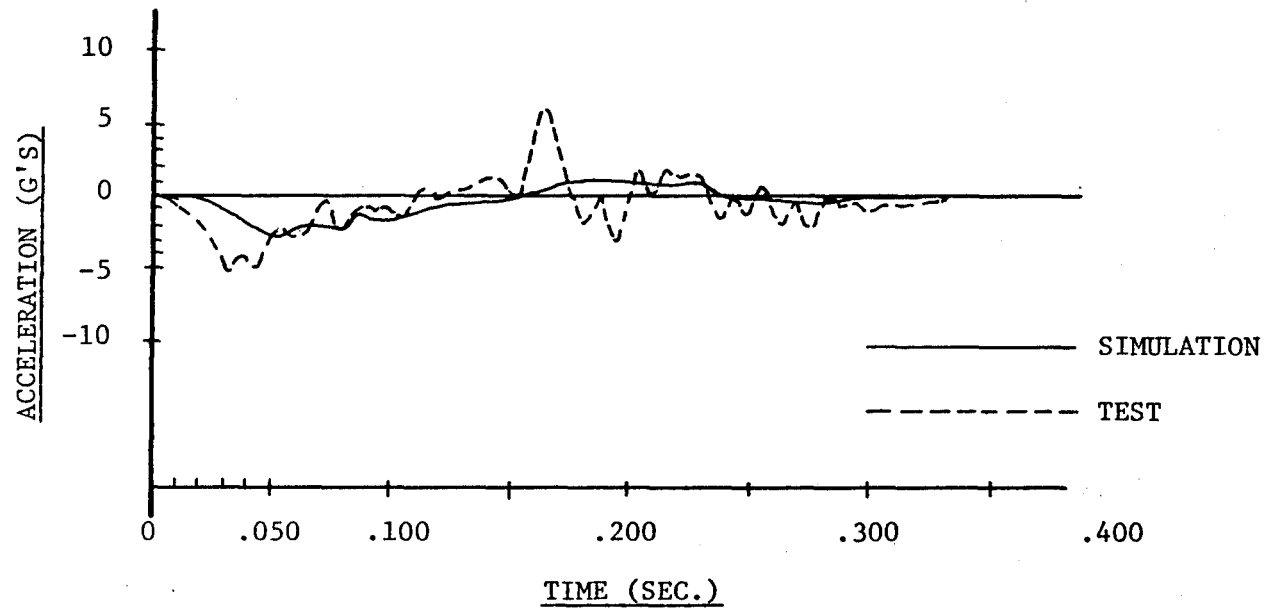


FIGURE 23 - LONGITUDINAL ACCELERATION ON FREE SIDE OF 4210 LB. VEHICLE IMPACTING CONCRETE MEDIAN BARRIER AT 61.9 MPH AT 7 DEGREES.

would not be felt by an occupant. Correspondingly, the simulated accelerometers respond only to those actions which cause vehicle redirection or rigid body motion since the simulation is devoid of structural vibrations except those stemming from the wheels and suspension systems.

2. In the actual case the effect of a force applied to the vehicle structure is diminished or damped before reaching an accelerometer located some distance away from the point of application.

In some cases if the distance is large enough and the force is of short duration, the effect may be damped out completely and hence, undetected by the accelerometer. However, in the simulated case all forces applied to the vehicle structure are transferred to the center of gravity of the rigid body as an equivalent force-couple system such that all simulated accelerometers respond instantaneously regardless of their location on the structure.

The simulated accelerometer traces in Figures 12 and 14 exhibit an oscillation of ± 11 G's between 30 and 50 milliseconds which was not recorded by the test accelerometers. This is caused by the front wheel violently engaging the suspension bumper stops as it first hits the barrier at the large impact angle of 25 degrees. The same probably occurred in the test, but since the accelerometers were located about 6.5 ft. behind the front wheel (just ahead of the rear wheel mounted to the frame member), the effects of these short duration forces were largely

damped out before reaching the accelerometers.

The test accelerometer traces in Figures 12, 16 and 18 reveal oscillations between 175 and 200 milliseconds which were not predicted by the simulation. These represent structural vibrations of the frame member, to which the accelerometers were mounted (just ahead of the rear wheel), induced by oscillations of the rear axle assembly when the rear wheel encountered the barrier. All of the differences in accelerations explained thus far were vibrational in nature and produced negligible net changes in velocity and hence, did not contribute to vehicle redirection. This is upheld by the fact that the comparisons of vehicle position are excellent (Figures 2 through 7).

The huge spike appearing in Figures 20 through 23 has 2 possible explanations. First, it is highly probable that this spike is also the result of a structural vibration caused by the rear wheel impacting the barrier. The vibration could have been critically damped explaining the existence of only 1 spike. Furthermore, the reason that higher G levels were recorded for this test, although it was less severe (only slight sheet metal damage), could be a result of the accelerometers being more directly aligned with the blow because of the small pitch and roll motions of the vehicle. If this explanation is accepted, the spike can be disregarded as not contributing to redirection of the vehicle, and the accelerometer comparison can be considered good.

However, as a second explanation, it is conceivable that initial tire contact caused the vehicle to rotate (yaw) parallel to the barrier

without appreciably changing the vehicle's velocity vector (magnitude nor direction). The vehicle would then have impacted the barrier in this position, causing an abrupt change in lateral velocity. In fact, for 60 mph at 7 degrees, the component of velocity normal to the barrier is 10.7 ft per second which corresponds to the area under the spike in question. This comparison is justifiable in this case since the car was parallel to the barrier when the spike occurred. If this explanation is accepted, the question remains as to whether this is a reproducible phenomenon or an abnormality. Until this question is answered, the HVOSM accelerometer results cannot be discounted, especially since good accelerometer comparisons were achieved for the 2 higher angles of impact, and good vehicle position comparisons were attained for all 3 tests.

Considering all facets of the comparison, it can be concluded that the HVOSM (with added structural hard points) provides a good simulation of an automobile impacting a rigid barrier of the Texas CMB type. Hence, it follows that the results of the parameter study (Chapter III) can be treated with added confidence.

III. PARAMETER STUDY

General

The modified version of the HVOSM computer program (including hard points, as described in Appendix B) was used to study the dynamic behavior of an automobile impacting the Texas CMB. The objective of the parametric study was to determine the performance characteristics of the CMB for a range of vehicle encroachment conditions. Factors used to measure barrier performance consisted of the vehicle's exit angle, maximum pitch and roll angle, and a severity index to quantify the event severity. The index is an interaction formula involving vehicle acceleration components and tolerable vehicle accelerations. It is discussed in Appendix D.

Nine different automobile impacts with the CMB were simulated. The impact speeds were 50, 70, and 80 mph and for each speed there were three impact angles; 5, 10, and 15 degrees. The simulated automobile had the properties of a 1963 Ford Galaxie weighing 4,780 pounds. Also included in this phase of the study were the three impacts simulated in the validation study (Chapter II), consisting of impact angles of 7, 15, and 25 degrees at an impact speed of approximately 60 mph. These twelve different impact conditions are representative of the majority of accidents involving traffic barriers. The results of the twelve runs are presented in Table 1.

All impact data for the computer runs, including vehicle and barrier information, are given in Volume II of this report. Some of the significant parameters were as follows:

TABLE 1
RESULTS OF CMB BARRIER SIMULATIONS

RUN NO.	AUTO WEIGHT (LBS)	IMPACT CONDITIONS		AUTOMOBILE KINEMATICS				AVERAGE ACCELERATIONS DURING PRIMARY IMPACT			S.I.***
		SPEED (MPH)	ANGLE, θ_1 (DEG)	MAX. ROLL (DEG)	MAX. PITCH (DEG)	FRONT TIRE CLIMB (IN)	EXIT ANGLE** θ_2 (DEG)	G _{LONG}	G _{LAT}	G _{VERT}	
1	4780	50.0	5.0	1.3	0.9	4.6	1.1	0.49	1.61	0.12	0.33
2	4780	70.0	5.0	2.2	1.5	6.5	0.3	0.72	2.53	0.43	0.52
3	4780	80.0	5.0	3.3	1.8	7.1	0.1	0.21	2.90	0.54	0.58
4	4780	50.0	10.0	4.2	3.2	8.6	2.5	1.13	2.99	0.94	0.64
5	4780	70.0	10.0	19.5*	5.0	11.2	1.2	0.16	5.06	2.03	1.07
6	4780	80.0	10.0	34.6*	5.8	12.6	1.2	1.92	6.42	2.61	1.38
7	4780	50.0	15.0	15.0*	6.5	11.9	3.6	0.47	4.29	1.38	0.91
8	4780	70.0	15.0	RO	6.6	NA		2.81	6.44	3.16	1.45
9	4780	80.0	15.0	RO	6.1	NA		3.24	7.49	3.29	1.66
10	4210	61.9	7.0	4.7	2.3	7.4	2.2	1.07	3.21	0.64	0.67
11	4210	59.6	15.0	21.0*	8.2	13.2	5.5	2.78	6.86	2.92	1.48
12	4000	63.0	25.0	37.0	8.6	NA	5.1	6.47	11.23	4.38	2.54

*Estimated roll obtained by energy expression using initial conditions from computer simulation at the time it was terminated. (See Appendix E)

**When vehicle loses contact with barrier.

***From Equation D1.

RO=Rollover

NA=Not available

Hardpoint Stiffness	= 2500 LB/IN
Sheet Metal Crushing Coefficient	= 2 LB/IN ³
Auto-Barrier Coefficient of Friction	= 0.3
Tire-Curb Coefficient of Friction	= 0.50

In some instances, the roll angle of the vehicle was still increasing at the termination of the computer run. Rather than re-run those cases (which would have been uneconomical) a formula was developed to estimate the roll angle beyond the termination point. This relationship was used to determine the maximum roll angle. Its derivation is given in Appendix E.

Barrier Performance

Model simulation indicates that the vehicle will roll over during a collision with a CMB at impact speeds of 70 and 80 mph and an impact angle of 15 degrees. The roll angle reported in Table 1 is the maximum roll angle of the automobile and may or may not occur when the automobile is in contact with the barrier.

As shown in Table 1, the maximum pitch angle of the automobile appears more sensitive to impact angle than to impact speed. In any event, the pitch angle remains small for any angle of impact and appears to be insignificant when considering the motion of the automobile.

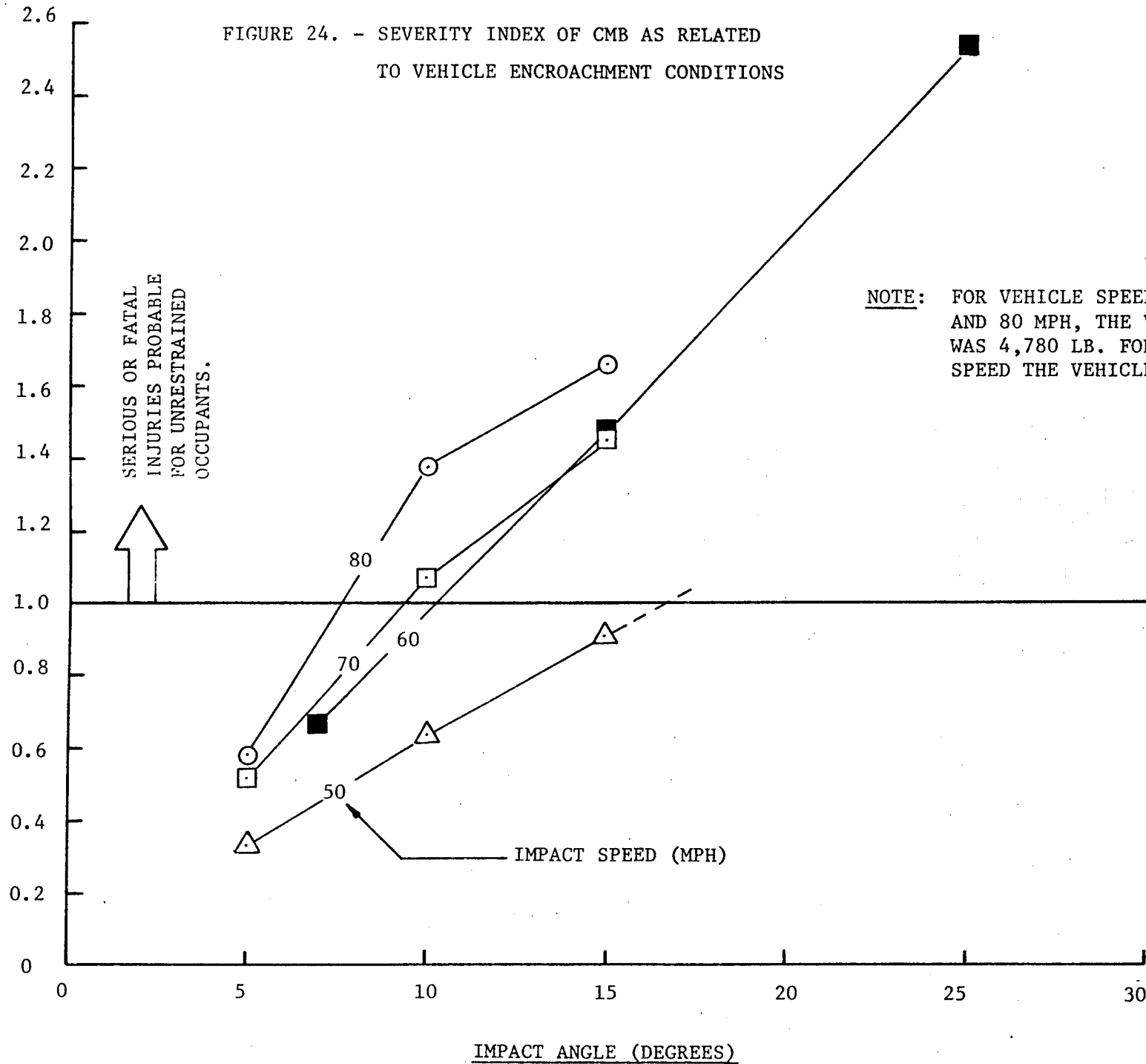
Height of climb of the front tire on the face of the CMB is given in Table 1. During a 5 degree collision, the front tire of the automobile climbs roughly 5 to 7 inches on the lower inclined CMB surface; and, during

a 10 degree collision the tire climbs roughly 9 to 12 inches on the lower surface. As indicated, the climb height was not available in some cases because tire-rigid barrier interaction is not accounted for in HVOSM (see discussion in Chapter II). However, based on an analysis of the output, it is doubtful that the tire climb would have exceeded the height of the barrier in those cases.

A desirable characteristic of a traffic barrier is that a colliding automobile be redirected at a shallow exit angle in order to minimize the danger to traffic. The exit angles shown in Table 1 were determined at the time the vehicle lost contact with the barrier. The exit angle appears to be more sensitive to impact angle than to impact speed. In all cases, however, the exit angles were shallow.

Another criterion used to determine barrier performance was the relative severity of the impact as measured by automobile accelerations. A severity index, which quantifies the severity, was computed and listed in Table 1 for each of the twelve runs studied. A discussion of the index is given in Appendix D.

Figure 24 shows the severity index versus impact angle for four different impact speeds. The apparent inconsistency of the 60 mph case is attributable to the differences in vehicle weight and dimensions. For vehicle speeds of 50, 70, and 80 mph the vehicle weighed 4,780 pounds, whereas, in the 60 mph case the vehicle weighed 4,210 pounds. The hard-point stiffness, sheet metal crushing coefficient, auto-barrier coefficient of friction, and tire-curb coefficient of friction were the same for all four speeds. The results therefore suggest that the severity of a lighter



vehicle impacting the barrier is higher than for a heavier vehicle, all other factors being the same.

Figure 25 shows impact speed versus impact angle for a severity index of one (1.0). The four points on the curve in Figure 25 were obtained from the intersection of the $SI = 1.0$ line with the four respective speed curves of Figure 24. The data as presented in Figure 25 may be useful in selecting roadway locations where the CMB can be safely used. For a given roadway, an upper limit on impact angle can be estimated as a function of the roadway's design speed and surface conditions and the distance from the roadway to the barrier.⁽⁶⁾ If the combination of design (or impact) speed and impact angle falls above the curve it may be advisable to select a more flexible barrier.

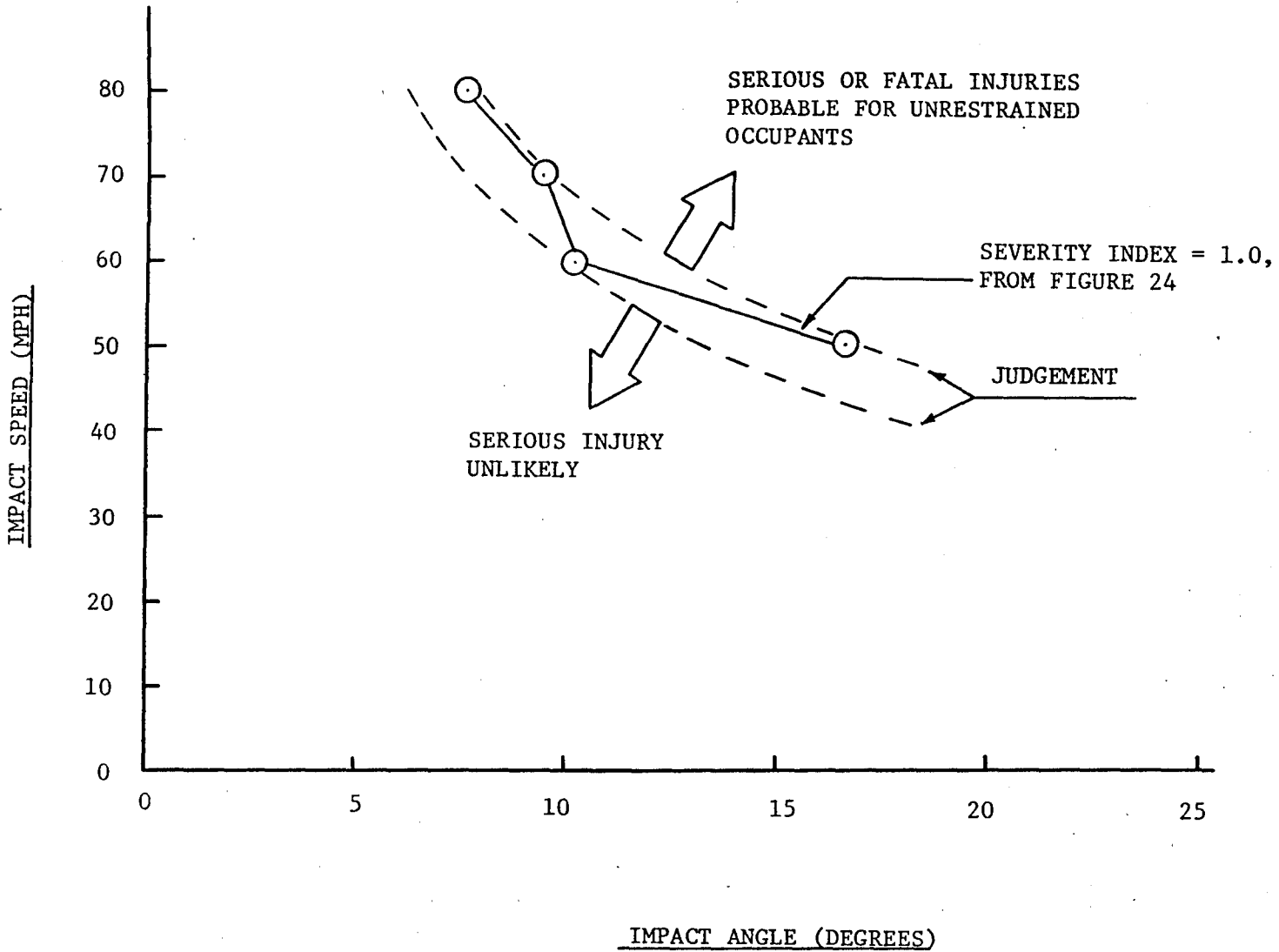


FIGURE 25. - RELATION BETWEEN ENCROACHMENT CONDITIONS AT A SEVERITY INDEX NEAR UNITY FOR CMB

IV. CONCLUSIONS

1. The impact subroutines of HVOSM were modified by TTI to account for the effects of *hardpoint* contacts (frame members, motor block, etc.) which occur when large vehicle deformations occur.
2. The modified HVOSM computer program (with hardpoints) can accurately predict automobile accelerations, motions, and external forces due to an impact with the Texas Concrete Median Barrier. This conclusion is based on a good correlation that was obtained between full-scale test results and simulations by HVOSM.
3. As a result of a parametric study with HVOSM, the following conclusions are made with regard to the Texas Concrete Median Barrier's performance:
 - (a) For impact speeds of 70 mph and greater in conjunction with impact angles of 15 degrees and greater, automobile rollover can be expected. This is a redundant finding in one respect since the severity indices indicate that serious or fatal injuries can be expected for these encroachment conditions. However, it is indeed an important consideration in view of the hazard that a rollover creates for other motorists.
 - (b) For impact speeds of 80 mph and less at impact angles of 15 degrees and less there was no tendency for the automobile to vault or climb over the barrier.
 - (c) In each of the ten impact conditions studied, where rollover did not occur, the automobile's exit angle was shallow after impact with the barrier (less than 6 degrees).
 - (d) A graphical presentation of the impact angles and speeds for which

the barrier can redirect an automobile without serious injuries to the occupants is given in Figure 25. This graph will aid the highway designer in deciding whether to install a CMB or a more flexible barrier. For instance, if the design speed is 70 mph, the width of roadway should be sufficiently narrow that an errant vehicle would impact the CMB at no greater than 10 degrees. (An estimate of maximum roadway width for a desired impact angle can be made using an equation suggested by Deleys (6)).

V. RECOMMENDATIONS

In view of the research findings the authors recommend the following:

1. further application of HVOSM in studying alternate shapes for the CMB in an effort to find one which will restrict the threat of rollover to much higher speeds and impact angles;
2. refinement of the severity index by taking a closer look at the relationship between accelerations experienced by the vehicle and the corresponding injury to the occupant. The current state-of-the-art requires one to assume that occupant accelerations equal those encountered by the vehicle. The validity of this assumption is questionable (especially for the unrestrained occupant) and as such it reduces the confidence level of the computed severity index. TTI is currently extending the development of a vehicle-victim model (on a separate study) that was initiated in Project 140 (7). This model will provide a means to determine the relationship between vehicle and occupant accelerations. This would enable a refinement of Figure 25 making it more objective and hopefully broadening its design application. For example, Figure 25 would require a safe impact angle of 7.5° maximum at 70 mph which is highly restrictive; and
3. parameter studies aimed at the performance of the CMB for a range of vehicle sizes, weights, and suspension properties.

APPENDIX A

A DESCRIPTION OF THE HIGHWAY-VEHICLE-OBJECT SIMULATION MODEL

APPENDIX A

A Description of the Highway-Vehicle-Object Simulation Model

To facilitate in the evaluation and design of a roadway and its environment, it is important to understand what effects that various roadway geometric features have on the dynamic behavior of an automobile and its occupants.

The mathematical model described herein is known as The Highway-Vehicle-Object Simulation Model (HVOSM). In general, this model can be utilized to investigate various problems associated with the roadway environment, such as highway traffic barrier collisions, rapid lane change maneuvers, handling response on horizontal curves, drainage ditch cross sections, and others.

The HVOSM was developed by Cornell Aeronautical Laboratory (CAL) (1, 2) and later modified for specific problem studies by the Texas Transportation Institute (TTI) (8). A conceptual idealization of the model is shown in Figure A1. The model is idealized as four rigid masses, which include: (a) the sprung mass (M_s) of the body supported by the springs, (b) the unsprung masses (M_1 and M_2) of the left and right independent suspension system of the front wheels, and (c) the unsprung mass (M_3) representing the rear axle assembly.

The eleven degrees of freedom of the model include translation of the automobile in three directions measured relative to some fixed coordinate axes system; rotation about the three coordinate of the automobile; independent displacement of each front wheel suspension

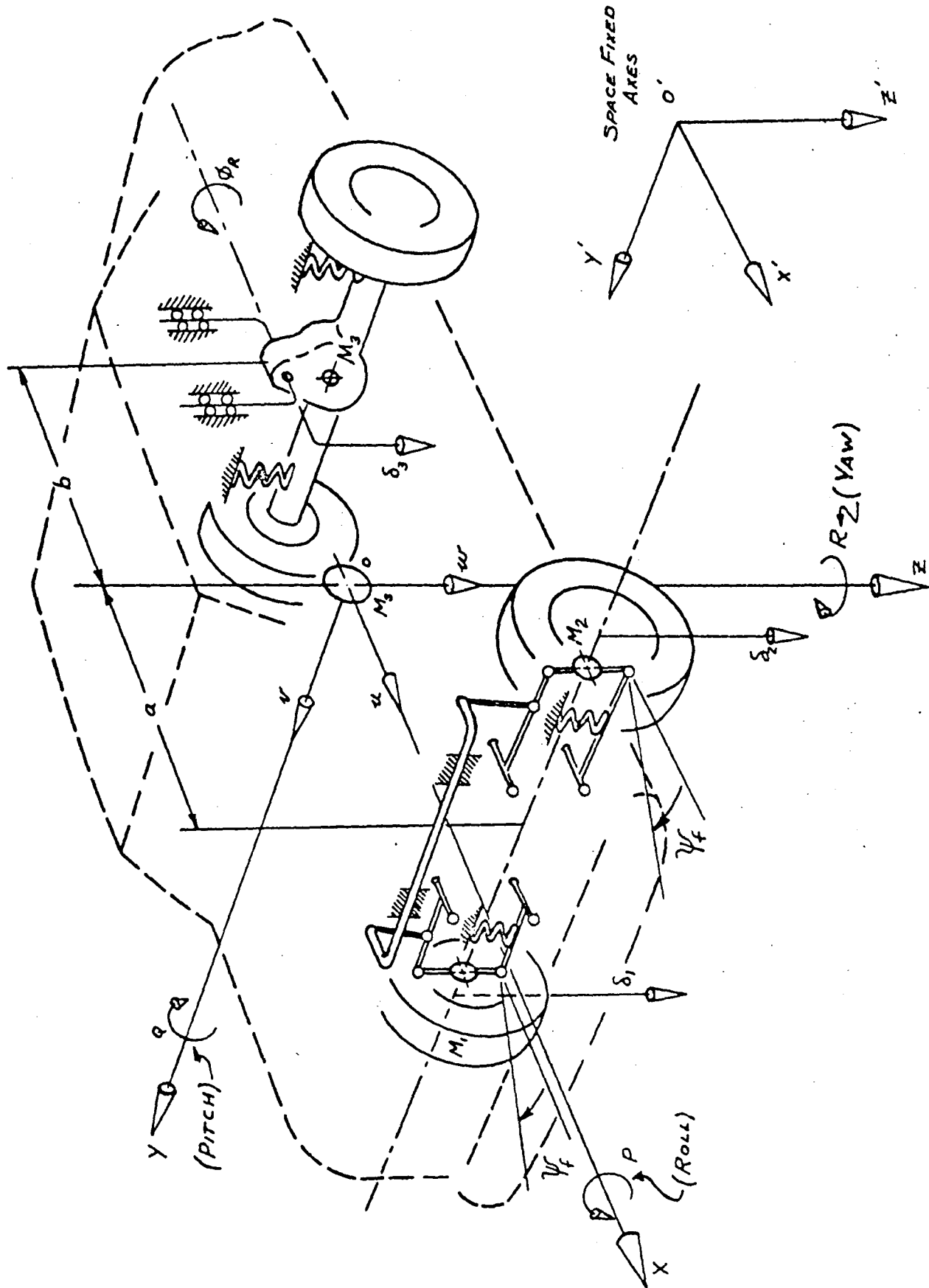


FIGURE A1. IDEALIZATION OF AUTOMOBILE (1,2)

system; suspension displacement and rotation of the rear axle assembly; and steer of the front wheels. If interested, the reader is referred to the references quoted earlier for a more in depth discussion of the mathematical model.

APPENDIX B

THE ADDITION OF STRUCTURAL HARDPOINTS TO THE ORIGINAL HVOSM

APPENDIX B

The Addition of Structural Hard Points

to the Original HVOSM

During TTI's initial attempts at simulating automobile collisions with a rigid type barrier, it became obvious that for large angles of attack (characterized by heavy sheet metal crushing), the HVOSM needed modification to account for the high impact forces resulting from stiff automobile structural components (hard points) slamming into the barrier.

According to McHenry et al.'s description of their mathematical model (2, pp. 126-129), the collision properties of the vehicle structure stem from an idealized layer of isotropic, homogenous, plastic material which surrounds the vehicle. They never intended this representation for anything more than simulating sheet metal crush of moderate penetration (i.e., 12 to 18 inches). In fact, McHenry and Deleys were the first to realize the need to account for structural hard points as they had formulated the problem (including preliminary computer programming) prior to TTI's request for assistance in the matter. They generously provided TTI with copies of their work which

proved to be a momentous contribution toward modifying the HVOSM and ultimately led to a successful simulation of an automobile impacting the Texas CMB.

Conceptually the idea of hard points was quite simple as it merely required defining (as input to HVOSM) the location of the desired points on the vehicle and the stiffness of each. The difficulty arose in trying to weave that concept into the existing fabric of sophisticated logic comprising the sprung-mass impact subroutines of the HVOSM. This is where CAL's assistance proved invaluable.

The new input variables required to define the hardpoints are as follows:

XSTIO (J)	}	-	Coordinates of original position of hard point no. J in vehicle-fixed coordinate system
YSTIO (J)			
ZSTIO (J)			
AKST (J)	-	-	Stiffness (lbs./in.) of hard point no. J, assumed to be omnidirectional

A listing of subroutine "SFORCE" is provided in this appendix.

It shows the computation and use of the following quantities:

YSTIPO (J)	-	the Y position of undeformed hardpoint no. J in space-fixed coordinates	
XSTIP (J)	}	-	the coordinates of the present position (deformed state) of hard point no. J in space-fixed coordinates
YSTIP (J)			
ZSTIP (J)			
FNSTI (J)	-	force (lbs.) on deformed hard point no. J	

SFNST	-	the sum of forces on all hard points for an assumed barrier deflection during iteration for equilibrium
FNX1	-	force due to sheet metal crushing for an assumed barrier deflection
FNX	=	$FNX1 + SFNST$, i.e., total force derived from barrier impact for an assumed barrier deflection
FN	-	force due to sheet metal crushing the barrier in its equilibrium position
FN1	=	$FN + SFNST$, i.e., total force derived from barrier contact for barrier in equilibrium position
XSTI (J)	}	- coordinates of hard point no. J (deformed state) in vehicle-fixed coordinates
YSTI (J)		
ZSTI (J)		

A listing of subroutine "RESFRC" is also included in this appendix to show the manner in which the forces derived from sheet metal crushing and various hard point impacts are resolved into forces and couples at the vehicle's center of gravity.

```

C
C  COMPILER OPTIONS - NAME= MAIN,OPT=02,LINECNT=60,SOURCE,EBCDIC,NOLIST,NODECK,LOAD,MAP,
C  SINGLE VEHICLE ACCIDENT SIMULATION - SUBROUTINE SFORCE
C  SUBROUTINE SFORCE SFRC 1
COMMON/INPT/PHIO,THETA0,PSIO,PO,QU,RO,XCOP,YCOP,ZCOP,UO,VO,W0,A,B,SFRC 2
1 DEL10,DEL20,DEL30,PHIRO,DEL10D,DEL20D,DEL30D,PHIROD,TF SFRC 3
2 ,FR,ZF,ZR,RHO,RW,AKT,SIGT,XLAMT,A1,A2,A3,AKRS,AMU, XMUR,SFRC 4
3 XMS,XMU, XIX,XIY,XIZ,XIXZ,CF,AKF,XLAMF,OMEGF,CFP, EPSF,SFRC 5
4 RF,CP,AKR,XLAMR,OMEGR,CRP,EPSR,RR,TS,THMAX,DTCOMP,TO, SFRC 6
5 F1,DTCMP1,DTPRNT,MODE,EBAR,EM,AAA,HMAX,HMIN,BET,G, SFRC 7
6 HFD(36),DADE(3),XIR,X1,Y1,Z1,X2,Y2,Z2,PHIC(50),DELB, SFRC 8
7 UCLF,DEL,NDL,PSIF(50),TQF(50),TQR(50),TR,TE,TINCR, SFRC 9
8 XBDY(10),YBDY(10),ZGP(21,21),THG(21,21),PHIG(21,21), SFRC 10
9 XB,XC,XINCR,NX,YB,YE,YINCR,NY,NBX,NBY,UVWMIN,PQRMIN SFRC 11
COMMON/INPT1/YCIP,YC2P,ZC2P,DELTC,PHIC1,PHIC2,AMUC,FJP(35),XIPS, SFRC 12
1 CPSP,UMGPS,AKPS,EPSPS,XPS,RWHJB,RWHJE,DRWHJ,INJCRB, SFRC 13
2 PSIF0,PSIFD0 SFRC 14
COMMON /INTG/NEQ,T,DT,VAR(50),DER(50) SFRC 15
COMMON /DIMV/X1P,X2P,X3P,X4P,Y1P,Y2P,Y3P,Y4P,Z1P,Z2P,Z3P,Z4P,PHI1,SFRC 16
1 PHI2,PHI3,PHI4,PSI1,PSI2,PSI3,PSI4,CAYW(4),CBYW(4), SFRC 17
2 CGYW(4),ZPGI(4),THGI(4),PHGI(4),CPG(4),SPG(4),CTG(4), SFRC 18
3 STG(4),CAGZ(4),CBGZ(4),CGGZ(4),D1(4),D2(4),D3(4), SFRC 19
4 XLM1(4),XLM2(4),XLM3(4),AMTX(3,3),CMTX(3,4),XGPP(4), SFRC 20
5 YGPP(4),ZGPP(4),DMATX(10,11),DELTA(4),CAR(4),C8R(4), SFRC 21
6 CGR(4),FR(4),HI(4),FC(4),TI(4),AX(4),BX(4),CX(4), SFRC 22
7 CTXG(4),UG(4),STXG(4),AY(4),BY(4),CY(4),CPYG(4), SFRC 23
8 SPYG(4),VG(4),PSIIP(4),PHICI(4),CAC(4),CBC(4),CGC(4), SFRC 24
9 FCXU(4),FCYU(4),FCZU(4),FS(4),CAXW(4),CRXW(4),CGXW(4) SFRC 25
COMMON /DIMV/AS(4),BS(4),CS(4),CAS(4),CBS(4),CGS(4),BETP(4), SFRC 26
1 BETBR(4),FSXU(4),FSYU(4),FSZU(4),FRXU(4),FRYU(4), SFRC 27
2 FRZU(4),FXU(4),FYU(4),FZU(4),SI(4),F1FI(2),F1RI(2), SFRC 28
3 F2FI(2),F2RI(2),CAH(4),CBH(4),CGH(4) SFRC 29
COMMON /COMP/SUMM,THETN,PHIN,PSIN,PI,RAD,GAM1,GAM2,GAM3,GAM4,GAM5,SFRC 30
1 GAM6,GAM7,GAM8,GAM9,THETT,PHIT,PSIT,A12,A23,ZRD,TRO2,SFRC 31
2 TFO2,TIZ,RHO2,RHOMUR,AMUF,BMUR,ZPR,TM4,RHMR2,AU2APB, SFRC 32
3 B02APB,RFTF,TSO2,RRTS,BROMUR,XMUFO2,AXMFO2,XMTFO4, SFRC 33
4 XIZR,RTK,RHMR2I,XIXP,XIZP,XIXZP,XIYZP,D1PD2,U1MD2, SFRC 34
5 ZRD3,ZRD3R,ZFD3R,ZF012,TIZ2,TG61,DU1P2,DU1M2,RPR,PHRP SFRC 35
6 ,TANTP,SPHTP,CPHTP,SECTP,SFXS,SFYs,SFZS,SNPS,SNTS, SFRC 36
7 SNPSS,TPR,CAY,CBY,CGY,CAX,CBX,CGX,SFYU,SFXU,SFYUF, SFRC 37
8 SFYUR,SFZU,COSTH,SINTH,COSPS,SINPS,CUSPH,SINPH,ANG1, SFRC 38
9 ANG2,CPHI,SPHI,CPSI,SPSI,P1,P7,P3,P4,P5,P6,TX,TY,TZ SFRC 39
COMMON /COMP/TRH,DISTX,DISTY,DISTD,DISTS,D21,ZETA4,ZETA4D,ZETA3, SFRC 40
1 ZETA3D,SFZ1,SNPU,SNTU,HCGH1,HCGH2,HCGH3,HCGH4,TERM1, SFRC 41
2 TERM2,SNPSU,SNPR,HCBH1,HCBH2,HCBH3,HCBH4,HCAH1,HCAH2, SFRC 42
3 HCAH3,HCAH4,UQ,WP,UR,QR,VP,PR,P2,Q2,R2,VR,WQ,PQ,PHIR2 SFRC 43
4 ,PHIRD2,RPHRD,GCTH,GSTH,GCTSP,GCTCP,XXX,YYY,IX,IY,XX1, SFRC 44
5 XX2,YY1,YY2,THG1,THG2,PHG1,PHG2,ZZ1,ZZ2,LLL SFRC 45
COMMON /COMP/ OMT2M1,FRSP(4),JMEGT,ICBHIT,ICBHIT, SFRC 46
1 DPSINT,TANPC1,TANPC2,PHIC1R,PHIC2R,AMUCMP,PHI1D, SFRC 47
2 PHI2D,LCB1(4),LCB2(4),IHIT,AJMTX(3,3),BMTX(3,3), SFRC 48
3 SFRX(4),SFRY(4),SFRZ(4),T1PSI,T2PSI,XMU SFRC 49
COMMON/ADTNL/U1,U2,U3,U4,V1,V2,V3,V4,W1,W2,W3,W4,XTRK(300) SFRC 50
DIMENSION XP(4),YP(4),ZP(4),PHI1(4),PSI1(4),UI(4),VI(4),WI(4) SFRC 51
EQUIVALENCE (XP,X1P),(YP,Y1P),(ZP,Z1P),(PHI1,PHI1),(PSI1,PSI1), SFRC 52
(U1,U1),(V1,V1),(W1,W1) SFRC 53
EQUIVALENCE (U,VAR(1)),(V,VAR(2)),(W,VAR(3)),(P,VAR(4)),(Q,VAR(5)) SFRC 54
(R,VAR(6)),(DEL1,VAR(7)),(DEL1D,VAR(8)),(DEL2,VAR(9)), SFRC 55

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2	(DEL2D,VAR(10)),(DEL3,VAR(11)),(DEL3D,VAR(12)),	SFRC	57
3	(PHIR,VAR(13)),(PHIRD,VAR(14)),(THETP,VAR(15)),	SFRC	58
4	(PHITP,VAR(16)),(PSITP,VAR(17)),(XCP,VAR(18)),	SFRC	59
5	(YCP,VAR(19)),(ZCP,VAR(20)),(PSIFI,VAR(21)),	SFRC	60
6	(PSIFID,VAR(22))	SFRC	61
	EQUIVALENCE (DU,DER(1)),(DV,DER(2)),(DW,DER(3)),(DP,DER(4)),	SFRC	62
1	(DQ,DER(5)),(DR,DER(6)),(DDEL1,DER(7)),(DDEL1D,DER(8))	SFRC	63
2	,(DDEL2,DER(9)),(DDEL2D,DER(10)),(DDEL3,DER(11)),	SFRC	64
3	(DDEL3D,DER(12)),(DPHIR,DER(13)),(DPHIRD,DER(14)),	SFRC	65
4	(DTHTP,DER(15)),(DPHITP,DER(16)),(DPSITP,DER(17)),	SFRC	66
5	(DXCP,DER(18)),(DYCP,DER(19)),(DZCP,DER(20)),	SFRC	67
6	(DPSIFI,DER(21)),(DDPSFI,DER(22))	SFRC	68
	DIMENSION YCIP(2)	SFRC	69
	LOGICAL LCB1,LCR2	SFRC	70
	COMMON/INPT2/A4,YBPO,ZBTP,ZBBP,XVF,XVR,YV,ZVT,ZVB,AKV,SIGR(11),SET	SFRC	71
1	,CONS,AMU8,EP5V,EP5B,XM,EPST,DDD,IND8,DELYBP,	SFRC	72
2	DELTR, AO, DATDKV(9), XINPT(100)	SFRC	73
	COMMON/BARRIER/FN,IBHIT,JBHIT,XCPNP(3),YCPNP(3),ZCPNP(3),XCPN(3),	SFRC	74
1	YCPN(3),ZCPN(3),AA1(17),BB1(17),CC1(17),RR1(17),	SFRC	75
2	AA2(17),BB2(17),CC2(17),RR2(17),CAB,CBB,CGB,CABT,	SFRC	76
3	CB8T,CG8T,RB,XBT,YBT,ZBT,XBB,YBB,ZBB,RR2P(17),	SFRC	77
4	Y8PT,XNN(17),YNN(17),ZNN(17),XMTX(3,4),IDPT(17),IPT	SFRC	78
5	,ININD,UNP(17),VNP(17),WNP(17),VMAX(4),I1,I2,I3,I4,	SFRC	79
6	XCPTP,YCPTP,ZCPTP,XCPBP,YCPBP,ZCPBP,YCPMP,AINTI,	SFRC	80
7	AINTP,SXR,SYR,SZR,SDEN,XRI,YRI,ZRI,FRICT,DELB8,VTAN,	SFRC	81
8	FNP,FB,URP,VRP,WRP,EP5L,XLUP,DELX,VL,NCYC,EEE,ENRGY,	SFRC	82
9	SWORK,SPENGY,DISS,IPLN,ILOAD	SFRC	83
	DIMENSION INDXPT(4)	SFRC	84
	EQUIVALENCE(INDXPT,11)	SFRC	85
	EQUIVALENCE(YCIP,YC1P)	SFRC	86
	EQUIVALENCE(XIYP,XTRA(1)),(SPHIC,XTRA(2)),(CPHIC,XTRA(3))	SFRC	87
	EQUIVALENCE(NSEG,XTRA(4))	SFRC	88
	EQUIVALENCE(YBTP,XTRA(7)),(PCAB,XTRA(8)),(PCBB,XTRA(9)),	SFRC	89
1	(PCGB,XTRA(10)),(PPRB,XTRA(11)),(CAB1,XTRA(12)),	SFRC	90
2	(CBB1,XTRA(13)),(CGB1,XTRA(14)),(RBI,XTRA(15))	SFRC	91
	EQUIVALENCE(NUNLD,XTRA(16))	SFRC	92
	EQUIVALENCE(NLDCTR,XTRA(17))	SFRC	93
	EQUIVALENCE(VDEF,XTRA(18)),(PVDEF,XTRA(19))	SFRC	94
	EQUIVALENCE(PSZR,XTRA(20))	SFRC	95
	COMMON/BARSTR/ XSTIO(3),YSTIO(3),ZSTIO(3),XSTI(3),YSTI(3),ZSTI(3),		
<i>NEW</i>	YSTIPU(3),XSTIP(3),YSTIP(3),ZSTIP(3),FNSTI(3),AKSI(3)		
1	SFXS = 0.0	SFRC	96
	YBP = 0.0	SFRC	97
	SFYS = 0.0	SFRC	98
	SFZS = 0.0	SFRC	99
	SNPS = 0.0	SFRC	100
	SNTS = 0.0	SFRC	101
	SNPSS = 0.0	SFRC	102
	FN = 0.0	SFRC	103
	IBHIT = 0	SFRC	104
	IPLN = 0	SFRC	105
	NAXIS = 0	SFRC	106
	FRICT = 0.0	SFRC	107
	VTAN = 0.0	SFRC	108
	VMAX(1) = 0.0	SFRC	109
	NSLCE = 0	SFRC	110
	NUNLD=0		
	NUNLD2=0 <i>NEW</i>		

XMTX(2,3) = PCGB	SFRC 168
XMTX(2,4) = PPRB	SFRC 169
XMTX(3,1) = 0	SFRC 170
XMTX(3,2) = 0	SFRC 171
XMTX(3,3) = 1	SFRC 172
XMTX(3,4) = PSZR	SFRC 173
CALL SIMSOL(XMTX,3,3,1)	
XB1 = XMTX(1,4)	SFRC 175
YB1 = XMTX(2,4)	SFRC 176
ZB1 = XMTX(3,4)	SFRC 177
IF (XVR.LE.XB1.AND.XB1.LE.XVF.AND.ABS(YB1).LT.YV.AND.ZVT.LE.ZB1	SFRC 178
1.AND.ZB1.LE.ZVB) NAXIS = 1	SFRC 179
IF(NAXIS.EQ.0.AND.VDEF.LT.PVDEF.AND.XB1.LT.XVR) GO TO 41	SFRC 180
TMPA = CB0*PCGB-CB0*PCBB	SFRC 181
TMPE = CGB*PCAB-CAB*PCGB	SFRC 182
TMPC = CAB*PCBB-CB0*PCAB	SFRC 183
TMPAP = TMPE*CGB-TMPC*CB0	SFRC 184
TMPBP = -TMPC*CAB-TMPA*CGB	SFRC 185
TMPCP = -TMPE*CB0-TMPE*CB0	SFRC 186
TMPD = SQRT(TMPAP**2+TMPE**2+TMPC**2)	SFRC 187
CAB1 = TMPAP/TMPD	SFRC 188
CB01 = TMPE/TMPD	SFRC 189
CGB1 = TMPC/TMPD	SFRC 190
RB1 = XB1*CAB1+YB1*CB01+ZB1*CGB1	SFRC 191
YB1VF = 1.0E6	SFRC 192
IF(CB01.NE.0.) YB1VF=(RB1-XVF*CAB1)/CB01	SFRC 193
78 DO 79 I=12,17	SFRC 194
AA2(I) = CAB1	SFRC 195
BB2(I) = CB01	SFRC 196
CC2(I) = CGB1	SFRC 197
RR2(I) = RB1	SFRC 198
79 CONTINUE	SFRC 199
C PRESENT LOCATION OF HARDPPOINTS IN SPACE FIXED COORDINATES	
80 DO 81 I=1,3	
XSTIP(I)=XCP+AMTX(1,1)*XSTI(I)+AMTX(1,2)*YSTI(I)+AMTX(1,3)*ZSTI(I)	(deformed hardpoints)
YSTIP(I)=YCP+AMTX(2,1)*XSTI(I)+AMTX(2,2)*YSTI(I)+AMTX(2,3)*ZSTI(I)	
ZSTIP(I)=ZCP+AMTX(3,1)*XSTI(I)+AMTX(3,2)*YSTI(I)+AMTX(3,3)*ZSTI(I)	
81 CONTINUE	
XRI=0.	
YRI = 0.0	SFRC 201
ZRI = 0.0	SFRC 202
AINI = 0.0	SFRC 203
SXR = 0.0	SFRC 204
SYR = 0.0	SFRC 205
SZR = 0.0	SFRC 206
SDEN = 0.0	SFRC 207
FNX=0.	
FNX1=0.	
FB=0.	
FBFN=0.	
SFNST=0.	
NSEG = (YCPMP-YBPTP)/DELYBP+1.0	
IPLN=NSEG	SFRC 208
YBP=YBPTP+IPLN*DELYBP	
NSG111 = NSEG+1	
I111 = 1	SFRC 209
9 DO 3R I=I111,NSG111	SFRC 210
IPLNP=IPLN	SFRC 211

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NEW

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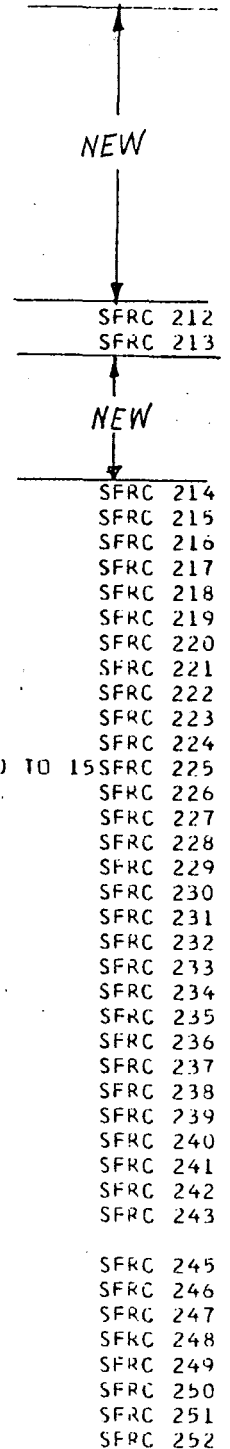
PYBP=YBP
PDELBB=DELBB
PPSXR=SXR
PPSYR=SYR
PPSZR=SZR
PSDEN=SDEN
PFNX=FNX
PFNX1=FNX1
PFB=FB
PFBFN=FBFN
PSFNST=SFNST
SFNST=0.
IPLN = NSEG-I+1
YBP = YBPTP+IPLN*DELYBP
DO 91 IJ=1,3
FNSTII(IJ)=0.
IF(YSTIP(IJ).GE.YBP)FNSTI(IJ)=AKST(IJ)*(YSTIPO(IJ)-YBP)
SFNST=SFNST+FNSTI(IJ)
CONTINUE
IF(YBP.LT.YBPO+EPSL+SET*DELX)GO TO 40
TMP = YBP-YCP
XBI = -AMTX(1,1)*XCP+AMTX(2,1)*TMP-AMTX(3,1)*ZCP
YBI = -AMTX(1,2)*XCP+AMTX(2,2)*TMP-AMTX(3,2)*ZCP
ZBI = -AMTX(1,3)*XCP+AMTX(2,3)*TMP-AMTX(3,3)*ZCP
RBI = XBI*CAB+YBI*CBB+ZBI*CGB
IPT = 0
10 DO 15 J=1,17
IDPT(J) = 0
IF(PSIT.LF.U.O.AND.J.LE.2)GO TO 15
IF(ININO.LT.2.AND.J.GT.11) GO TO 15
IF(CAB.EQ.0..AND.(J.EQ.4.OR.J.EQ.5.OR.J.EQ.10.OR.J.EQ.11))GO TO 15
IF(CBB.EQ.0..AND.(J.LE.2.OR.J.EQ.7.OR.J.EQ.8)) GO TO 15
IF(CGB.EQ.0..AND.(J.EQ.3.OR.J.EQ.6.OR.J.EQ.9)) GO TO 15
IF(CAB1*CBB.EQ.CBB1*CAB.AND.(J.EQ.12.OR.J.EQ.13)) GO TO 15
IF(CBB1*CGB.EQ.CGB1*CBB.AND.(J.EQ.14.OR.J.EQ.15)) GO TO 15
IF(CGB1*CAB.EQ.CAB1*CGB.AND.J.GE.16) GO TO 15
IF(NAXIS.EQ.0.AND.J.GT.11) GO TO 15
11 XMTX(1,1) = CAB
XMTX(1,2) = CBB
XMTX(1,3) = CGB
XMTX(1,4) = RBI
12 XMTX(2,1) = AA1(J)
XMTX(2,2) = BB1(J)
XMTX(2,3) = CC1(J)
XMTX(2,4) = RR1(J)
13 XMTX(3,1) = AA2(J)
XMTX(3,2) = BB2(J)
XMTX(3,3) = CC2(J)
XMTX(3,4) = RR2(J)
14 CALL SIMSOL(XMTX,3,3,2)
XNN(J) = XMTX(1,4)
YNN(J) = XMTX(2,4)
ZNN(J) = XMTX(3,4)
IF(XNN(J).LT.XVR.OR.XNN(J).GT.XVF) GO TO 15
IF(ABS(YNN(J)).GT.YV) GO TO 15
IF(ZNN(J).LT.ZVT.OR.ZNN(J).GT.ZVH) GO TO 15
IDPT(J) = 1
IPT = IPT+1

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Store Info. from previous slice

Compute Force on hardpoint

based on undeformed position



IPPT = J	SFRC 253
15 CONTINUE	SFRC 254
IF(IPPT.LE.11.AND.(NAXIS.EQ.1.AND.YB1VF.GT.YV.AND.ININD.EQ.2))	SFRC 255
1 GO TO 38	SFRC 256
IF(MOD(INOB,2).EQ.0) GO TO 23	SFRC 257
IF(CGB.EQ.0.0.AND.CGBT.EQ.0.0) GO TO 23	SFRC 258
RR2P(1) = RBT	SFRC 259
RR2P(2) = RBB	SFRC 260
RR2P(4) = RBT	SFRC 261
RR2P(5) = RBB	SFRC 262
RR2P(7) = RBT	SFRC 263
RR2P(8) = RBB	SFRC 264
RR2P(10) = RBT	SFRC 265
RR2P(11) = RBB	SFRC 266
RR2P(12) = RBT	SFRC 267
RR2P(13) = RBB	SFRC 268
RR2P(14) = RBT	SFRC 269
RR2P(15) = RBB	SFRC 270
RR2P(16) = RBT	SFRC 271
RR2P(17) = RBB	SFRC 272
16 DO 22 J=1,17	SFRC 273
IF(PSIT.LE.0.0.AND.J.LE.2) GO TO 22	SFRC 274
IF(J.EQ.3.OR.J.EQ.6.OR.J.EQ.9) GO TO 22	SFRC 275
IF(CAB*CGBT.EQ.CGB*CBT.AND.(J.EQ.4.OR.J.EQ.5.OR.J.EQ.10.OR.	SFRC 276
1 J.EQ.11)) GO TO 22	SFRC 277
IF(CGB*CBBT.EQ.CBB*CGBT.AND.(J.LE.2.OR.J.EQ.7.OR.J.EQ.8)) GO TO 22	SFRC 278
IF(CAB*(CBB1*CGBT-CBBT*CGB1)-CAB1*(CBB*CGBT-CBBT*CGB)+CBT*(CBB*	SFRC 279
1 CGB1-CBB1*CGB).EQ.0.0.AND.J.GE.12) GO TO 22	SFRC 280
IF(J.GE.12.AND.IDPT(J).NE.1) GO TO 22	SFRC 281
IF(IDPT(1).EQ.1.AND.IDPT(2).EQ.1.AND.J.EQ.14) GO TO 173	SFRC 282
IF(IDPT(7).EQ.1.AND.IDPT(8).EQ.1.AND.J.EQ.15) GO TO 173	SFRC 283
IF(IDPT(4).EQ.1.AND.IDPT(5).EQ.1.AND.J.EQ.16) GO TO 173	SFRC 284
IF(IDPT(10).EQ.1.AND.IDPT(11).EQ.1.AND.J.EQ.17) GO TO 173	SFRC 285
XMTX(1,1) = CAB	SFRC 286
XMTX(1,2) = CBB	SFRC 287
XMTX(1,3) = CGB	SFRC 288
XMTX(1,4) = RBT	SFRC 289
IF(J.GE.12) GO TO 170	SFRC 290
XMTX(2,2) = BB1(J)	SFRC 291
XMTX(2,3) = CC1(J)	SFRC 292
17 XMTX(2,1) = AA1(J)	SFRC 293
XMTX(2,4) = RR1(J)	SFRC 294
GO TO 18	SFRC 295
170 XMTX(2,1) = AA2(J)	SFRC 296
XMTX(2,2) = BB2(J)	SFRC 297
XMTX(2,3) = CC2(J)	SFRC 298
XMTX(2,4) = RR2(J)	SFRC 299
18 XMTX(3,1) = CABT	SFRC 300
XMTX(3,2) = CBBT	SFRC 301
XMTX(3,3) = CGBT	SFRC 302
IF((IDPT(1).EQ.1.AND.J.EQ.14).OR.(IDPT(4).EQ.1.AND.J.EQ.16))	SFRC 303
1 GO TO 171	SFRC 304
IF((IDPT(8).EQ.1.AND.J.EQ.15).OR.(IDPT(11).EQ.1.AND.J.EQ.17))	SFRC 305
1 GO TO 172	SFRC 306
XMTX(3,4) = RR2P(J)	SFRC 307
GO TO 19	SFRC 308
171 XMTX(3,4) = RBB	SFRC 309
GO TO 19	SFRC 310

172	XMTX(3,4) = RBT	SFRC 311
19	CALL SIMSOL(XMTX,3,3,3)	
	IF(XMTX(1,4).LT.XVK.UR.XMTX(1,4).GT.XVF) GO TO 22	SFRC 313
	IF(ABS(XMTX(2,4)).GT.YV) GO TO 22	SFRC 314
	IF(XMTX(3,4).LT.ZVT.CR.XMTX(3,4).GT.ZVB) GO TO 22	SFRC 315
	IF(IDPT(J).NE.0) GO TO 20	SFRC 316
	IDPT(J) = 1	SFRC 317
	GO TO 21	SFRC 318
20	IF(ABS(XMTX(3,4)).GE.ABS(ZNN(J)))GO TO 22	SFRC 319
21	XNN(J) = XMTX(1,4)	SFRC 320
	YNN(J) = XMTX(2,4)	SFRC 321
	ZNN(J) = XMTX(3,4)	SFRC 322
	GO TO 22	SFRC 323
173	IDPT(J) = 0	SFRC 324
	IPT = IPT-1	SFRC 325
22	CONTINUE	SFRC 326
23	IF(IPT.LT.3) GO TO 38	SFRC 327
24	DO 25 J=1,17	SFRC 328
	IF(IDPT(J).EQ.0) GO TO 25	SFRC 329
	TMPU = U-YNN(J)*R+ZNN(J)*Q	SFRC 330
	TMPV = V+XNN(J)*R-ZNN(J)*P	SFRC 331
	TMPW = W+YNN(J)*P-XNN(J)*Q	SFRC 332
	UNP(J) = AMTX(1,1)*TMPU+AMTX(1,2)*TMPV+AMTX(1,3)*TMPW	SFRC 333
	VNP(J) = AMTX(2,1)*TMPU+AMTX(2,2)*TMPV+AMTX(2,3)*TMPW	SFRC 334
	WNP(J) = AMTX(3,1)*TMPU+AMTX(3,2)*TMPV+AMTX(3,3)*TMPW	SFRC 335
25	CONTINUE	SFRC 336
26	DO 27 J=1,4	SFRC 337
	VMAX(J) = -1.0E30	SFRC 338
	INDXPT(J) = 0	SFRC 339
27	CONTINUE	SFRC 340
28	DO 34 J=1,17	SFRC 341
	IF(IDPT(J).EQ.0) GO TO 34	SFRC 342
29	DO 33 K=1,4	SFRC 343
	IF(VNP(J).LT.VMAX(K)) GO TO 33	SFRC 344
	IF(K.EQ.4) GO TO 32	SFRC 345
	K1 = K+1	SFRC 346
30	DO 31 L=K1,4	SFRC 347
	M = 4-L+K1	SFRC 348
	VMAX(M) = VMAX(M-1)	SFRC 349
	INDXPT(M) = INDXPT(M-1)	SFRC 350
31	CONTINUE	SFRC 351
32	VMAX(K) = VNP(J)	SFRC 352
	INDXPT(K) = J	SFRC 353
	GO TO 34	SFRC 354
33	CONTINUE	SFRC 355
34	CONTINUE	SFRC 356
	IPT = 4	SFRC 357
	IF(INDXPT(4).EQ.0) IPT = 3	SFRC 358
37	J3 = I3	SFRC 359
	J1 = I1	SFRC 360
	J2 = I2	SFRC 361
	J4 = I4	SFRC 362
	CALL AREA	SFRC 363
	IF(IB.EQ.1) GO TO 38	SFRC 364
	FNX1=AKV*DELYBP*SDEN	
	FNX=FNX1+SFNST	
	IF(NSLCE.NE.0) GO TO 38	SFRC 366
40	DELBB = AMAX1(YBP-YBPO,EPST+SET*DELX)	SFRC 367

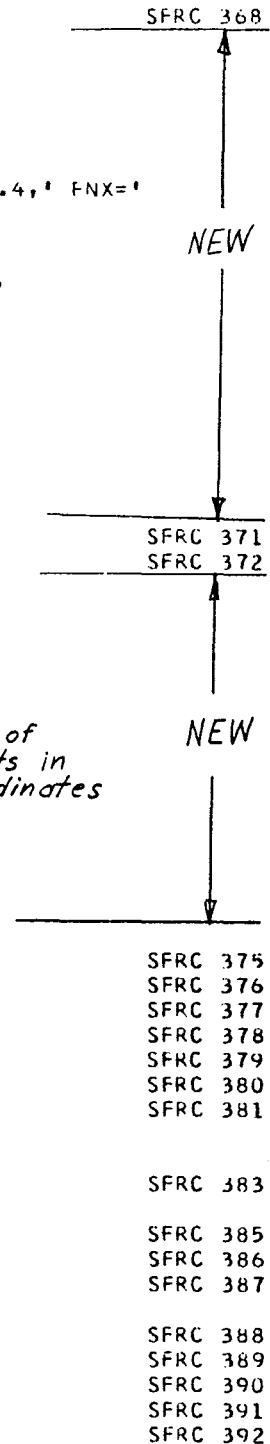
```

CALL NLDFRC
FBFN=FB-FNX
IF(EPSB.LT.FBFN)GOTO38
IF(I.EQ.1)GOTO105
IF(FBFN.GE.0.)GOTO105
IF(ABS(FBFN).LT.ABS(PFBFN))GOTO105
PRINT 1001,T,I,YBP,PYBP,PNX,PFNX
1001  FORMAT(T2,' T=',F7.4,' I=',I3,' YBP=',F10.4,' PYBP=',F10.4,' FNX='
1,G13.5,' PPNX=' ,G13.5,' EQUILIB AT PREV SLICE RESET')
IPLN=IPLNP
YBP=PYBP
DELBB=PDELBB
SXR=PPSXR
SYR=PPSYR
SZR=PPSZR
SDEN=PSDEN
FNX=PFNX
FNX1=PFNX1
FB=PFB
SFNST=PSFNST
105  YBPT=AMAX1(YBP,YBPO+EPSL+SET*DELX)
NSLCE = NSLCE+1
IF(NLDCTR.EQ.3)CALL NLDFL
NUNLD2=0
IF(NUNLD.EQ.0)GOTO38
NUNLD2=1
GOTO110
38  CONTINUE
110  DO 111 IJ=1,3
IF(YSTIP(IJ).GT.YBPT)YSTIP(IJ)=YBPT
AA=XSTIP(IJ)-XCP
BB=YSTIP(IJ)-YCP
CC=ZSTIP(IJ)-ZCP
XSTI(IJ)=AMTX(1,1)*AA+AMTX(2,1)*BB+AMTX(3,1)*CC
YSTI(IJ)=AMTX(1,2)*AA+AMTX(2,2)*BB+AMTX(3,2)*CC
ZSTI(IJ)=AMTX(1,3)*AA+AMTX(2,3)*BB+AMTX(3,3)*CC
111  CONTINUE
IF(NUNLD2.NE.0)GOTO103
IF(NUNLD.NE.0) GO TO 100
IF( IB .NE. 1) GO TO 50
45  NEGPT=0
DO 46 J=1,IPT
IF( VMAX(J) .LT. 0.0 ) NEGPT=NEGPT + 1
46  CONTINUE
IF( NEGPT .GE. IPT) GO TO 41
50  FN =AKV*DELYBP*SDEN
FN1=FN +SFNST
IF(ININD.EQ.0) ININD = 1
IF(FN1.NE.0.0.AND.NUNLD.EQ.0) CALL RESFRC
IF(NSLCE.EQ.0.AND.IB.EQ.1) GO TO 103
IF(NSLCE.EQ.0) GO TO 100
103  TMP = YBPT-YCP
NUNLD2=0
XBPP = -AMTX(1,1)*XCP+AMTX(2,1)*TMP-AMTX(3,1)*ZCP
YBPP = -AMTX(1,2)*XCP+AMTX(2,2)*TMP-AMTX(3,2)*ZCP
ZBPP = -AMTX(1,3)*XCP+AMTX(2,3)*TMP-AMTX(3,3)*ZCP
RB = XBPP*CAB + YBPP*CBB + ZBPP*CGH
GO TO 39

```

*Previous Slice had better Equilibrium
∴ reset values*

*Compute Position of
deformed hardpoints in
vehicle-fixed coordinates*



100 NUNLD = NUNLD+1	SFRC 393
NSG111 = NSG111+1	SFRC 394
I111 = NSG111	SFRC 395
GO TO 9	SFRC 396
39 NUNLD = 0	SFRC 397
41 IF(NLDCTR.EQ.3.AND.IPT.GE.3)WRITE(6,1000)T,XB1,YB1,IPT,J1,J2,J3,	SFRC 398
1 J4,XNN(J1),YNN(J1),ZNN(J1),XNN(J2),YNN(J2),ZNN(J2),	SFRC 399
2 XNN(J3),YNN(J3),ZNN(J3),XNN(J4),YNN(J4),ZNN(J4)	SFRC 400
1000 FORMAT(F7.4,2F7.1,5I3,12F8.1)	SFRC 401
NLDCTR = NLDCTR+1	SFRC 402
RETURN	SFRC 403
END	SFRC 404

```

CMPILER OPTIONS - NAME= MAIN,OPT=02,LINECNT=60,SOURCE,EBCDIC,NOLIST,NODECK,LOAD,MAP,N
C SINGLE VEHICLE ACCIDENT SIMULATION - SUBROUTINE RESFRC RSFR 0
SUBROUTINE RESFRC RSFR 1
COMMON/INPT/PHIO,THETA0,PSIO,PO,QO,RO,XCOP,YCOP,ZCOP,UO,VO,W0,A,B,RSFR 2
1 DEL10,DEL20,DEL30,PHIR0,DEL10D,DEL20D,DEL30D,PHIR0D,TFRSFR 3
2 ,TR,ZF,ZR,RHO,RW,AKT,SIGT,XLAMT,A1,A2,A3,AKRS,AMU,XMUR,RSFR 4
3 XMS,XMUF,XIX,XIY,XIZ,XIXZ,CF,AKF,XLAMF,OMEGF,CFP,EPSP,RSFR 5
4 RF,CR,AKR,XLAMR,OMEGR,CRP,EPSP,RR,TS,THMAX,DTCOMP,TO, RSFR 6
5 T1,DTCMP1,DTPRNT,MODE,EBAR,EM,AAA,HMAX,HMIN,BET,G, RSFR 7
6 HED(36),DADE(3),XIR,X1,Y1,Z1,X2,Y2,Z2,PHIC(50),DELB, RSFR 8
7 DELE,DDEL,NDEL,PSIF(50),TQF(50),TQR(50),TB,TE,TINCR, RSFR 9
8 XBDRY(10),YBDRY(10),ZGP(21,21),THG(21,21),PHIG(21,21),RSFR 10
9 XB,XE,XINCR,NX,YB,YE,YINCR,NY,NBX,NBY,UVWMIN,PQRMIN RSFR 11
COMMON/INPT1/YC1P,YC2P,ZC2P,DELTC,PHIC1,PHIC2,AMUC,FJP(35),XIPS, RSFR 12
1 CPSP,OMGPS,AKPS,EPSPS,XPS,RWHJB,RWHJE,DRWHJ,INDCRB, RSFR 13
2 PSIFIO,PSIFDO RSFR 14
COMMON /INTG/NEQ,T,DT,VAR(50),DER(50) RSFR 15
COMMON /DIMV/X1P,X2P,X3P,X4P,Y1P,Y2P,Y3P,Y4P,Z1P,Z2P,Z3P,Z4P,PHI1,RSFR 16
1 PHI2,PHI3,PHI4,PSI1,PSI2,PSI3,PSI4,CAYW(4),CBYW(4), RSFR 17
2 CGYW(4),ZPGI(4),THGI(4),PHGI(4),CPG(4),SPG(4),CTG(4),RSFR 18
3 STG(4),CAGZ(4),CBGZ(4),CGGZ(4),D1(4),D2(4),D3(4), RSFR 19
4 XLM1(4),XLM2(4),XLM3(4),AMTX(3,3),CMTX(3,4),XGPP(4), RSFR 20
5 YGPP(4),ZGPP(4),DMATX(10,11),DELTA(4),CAR(4),CBR(4), RSFR 21
6 CGR(4),FR(4),HI(4),FC(4),TI(4),AX(4),BX(4),CX(4), RSFR 22
7 CTXG(4),UG(4),STXG(4),AY(4),BY(4),CY(4),CPYG(4), RSFR 23
8 SPYG(4),VG(4),PSIIP(4),PHICI(4),CAC(4),CBC(4),CGC(4),RSFR 24
9 FCXU(4),FCYU(4),FCZU(4),FS(4),CAXW(4),CBXW(4),CGXW(4)RSFR 25
COMMON /DIMV/AS(4),BS(4),CS(4),CAS(4),CBS(4),CGS(4),BETP(4), RSFR 26
1 BETBR(4),FSXU(4),FSYU(4),FSZU(4),FRXU(4),FRYU(4), RSFR 27
2 FRZU(4),FXU(4),FYU(4),FZU(4),SI(4),F1FI(2),F1RI(2), RSFR 28
3 F2FI(2),F2RI(2),CAH(4),CBH(4),CGH(4) RSFR 29
COMMON /COMP/SUMM,THETN,PHIN,PSIN,PI,RAD,GAM1,GAM2,GAM3,GAM4,GAM5,RSFR 30
1 GAM6,GAM7,GAM8,GAM9,THETT,PHIT,PSIT,A12,A23,ZR0,TR02,RSFR 31
2 TFO2,TIZ,RHO2,RHOMUR,AMUF,BMUR,ZPR,TM4,RHMR2,AQ2APB, RSFR 32
3 BQ2APB,RFTF,TQ2,RRTS,BROMUR,XMUF02,AXMFO2,XMTFO4, RSFR 33
4 XIXZ,RTR,RHMR2I,XIXP,XIZP,XIXZP,XIYZP,D1PD2,D1MD2, RSFR 34
5 ZRD3,ZRD3R,ZFD3R,ZFD12,TIZ2,TG61,DD1P2,DD1M2,RPR,PHRPRSFR 35
6 ,TANTP,SPHTP,CPHTP,SECTP,SFXS,SFYS,SFZS,SNPS,SNTS, RSFR 36
7 SNPSS,TPR,CAY,CBY,CGY,CAX,CBX,CGX,SFYU,SFXU,SFYUF, RSFR 37
8 SFYUR,SFZU,COSTH,SINTH,COSPS,SINPS,COSPH,SINPH,ANG1, RSFR 38
9 ANG2,CPHI,SPHI,CPSI,SPSI,P1,P7,P3,P4,P5,P6,TX,TY,TZ RSFR 39
COMMON /COMP/TRH,DISTX,DISTY,DISTD,DISTS,U21,ZETA4,ZETA4D,ZETA3, RSFR 40
1 ZETA3D,SFZ1,SNPU,SNTU,HCGH1,HCGH2,HCGH3,HCGH4,TERM1, RSFR 41
2 TERM2,SNPSU,SNPR,HCBH1,HCBH2,HCBH3,HCBH4,HCAH1,HCAH2,RSFR 42
3 HCAH3,HCAH4,UQ,WP,UR,QR,VP,PR,P2,Q2,R2,VR,WQ,PQ,PHIR2RSFR 43
4 ,PHIRD2,RPHRD,GCTH,GSTH,GCTSP,GCTCP,XXX,YYY,IX,IY,XX1,RSFR 44
5 XX2,YY1,YY2,THG1,THG2,PHG1,PHG2,ZZ1,ZZ2,LLL RSFR 45
COMMON /COMPN/ OMT2M1,FRSP(4),FRCP(4),OMEGT,ICBHIT,ICBHIT, RSFR 46
1 DPSINT,TANPC1,TANPC2,PHIC1R,PHIC2R,AMUCMP,PHI1D, RSFR 47
2 PHI2D,LCB1(4),LCB2(4),IHIT,AJMTX(3,3),BMTX(3,3), RSFR 48
3 SFRX(4),SFRY(4),SFRZ(4),T1PSI,T2PSI,XMU RSFR 49
COMMON/ADTNL/UI,U2,U3,U4,V1,V2,V3,V4,W1,W2,W3,W4,XTRA(300) RSFR 50
DIMENSION XP(4),YP(4),ZP(4),PHI1(4),PSI1(4),UI(4),VI(4),WI(4) RSFR 51
EQUIVALENCE (XP,X1P),(YP,Y1P),(ZP,Z1P),(PHI1,PHI1),(PSI1,PSI1), RSFR 52
1 (UI,U1),(VI,V1),(WI,W1) RSFR 53
EQUIVALENCE (U,VAR(1)),(V,VAR(2)),(W,VAR(3)),(P,VAR(4)),(Q,VAR(5))RSFR 54
1 (R,VAR(6)),(DEL1,VAR(7)),(DEL1D,VAR(8)),(DEL2,VAR(9)),RSFR 55
2 (DEL2D,VAR(10)),(DEL3,VAR(11)),(DEL3D,VAR(12)), RSFR 56
3 (PHIR,VAR(13)),(PHIRD,VAR(14)),(THETTP,VAR(15)), RSFR 57

```

```

VTAN=SQRT(URP*URP+WRP*WRP)
TEMP1=0.
TEMP2=0.
IF(VTAN.LT.EPSV)GOTO6
AA=AMUB*FN
FRICT=AA
AA=AA/VTAN
TEMP1=-AA*URP
TEMP2=-AA*WRP
CONTINUE

```

Magnitude and Direction of Friction Force

6

```

DOSJ=1,4
FX=AMTX(1,1)*TEMP1-AMTX(2,1)*F(J)+AMTX(3,1)*TEMP2
FY=AMTX(1,2)*TEMP1-AMTX(2,2)*F(J)+AMTX(3,2)*TEMP2
FZ=AMTX(1,3)*TEMP1-AMTX(2,3)*F(J)+AMTX(3,3)*TEMP2
TEMP1=0.
TEMP2=0.

```

Components of each Force in vehicle-fixed Coordinates

```

SFXS=SFXS+FX
SFYS=SFYS+FY
SFZS=SFZS+FZ

```

Summing all Forces

```

SNPS=SNPS+FZ*Y(J)-FY*Z(J)
SNTS=SNTS+FX*Z(J)-FZ*X(J)
SNPSS=SNPSS+FY*X(J)-FX*Y(J)

```

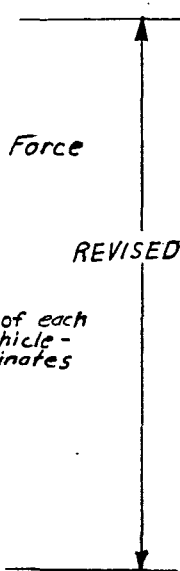
Computing and Summing Moments

5

```

RETURN
END

```



RSFR 108
RSFR 109

APPENDIX C

ACCELEROMETER LOCATIONS

Vehicle Dimensions - 1963 Plymouth

Weight - 4000 lb
Length - 16'-11" = 203"
Width - 6'-2" = 74"
Wheel base - 9'-8" = 116"
Track width - 5' = 60"
Bumper to front wheel center - 2'-8" = 32"
Front bumper to center of gravity - 7'-7 1/2" = 91 1/2"

Location of accelerometers on frame members (symmetrical about vehicle centerline)

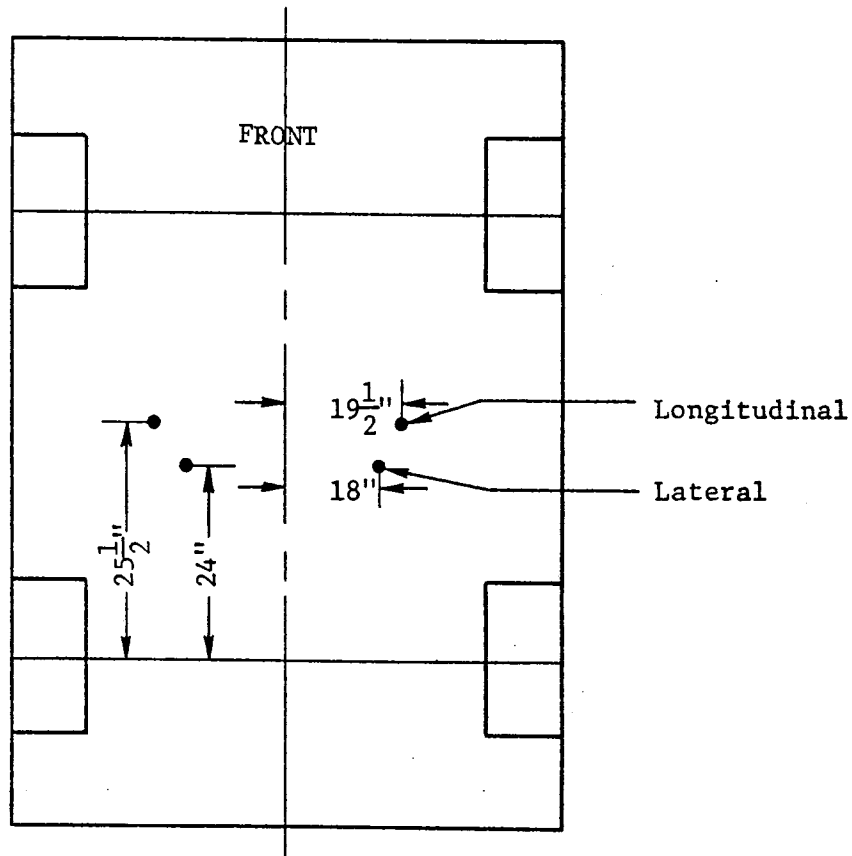


FIGURE C1- Vehicle data for test impacting concrete median barrier at 63 mph at 25 deg.

Vehicle Dimensions - 1963 Chevrolet

Weight - 4210 lb
Length - 17'-3" = 207"
Width - 6'-6" = 78"
Wheel base - 9'-11" = 119"
Track width - 4'-11" = 59"
Bumper to front wheel center - 2'-8" = 32"
Bumper to center of gravity - 7'-2" = 86"

Location of accelerometers on frame members (symmetrical about vehicle centerline)

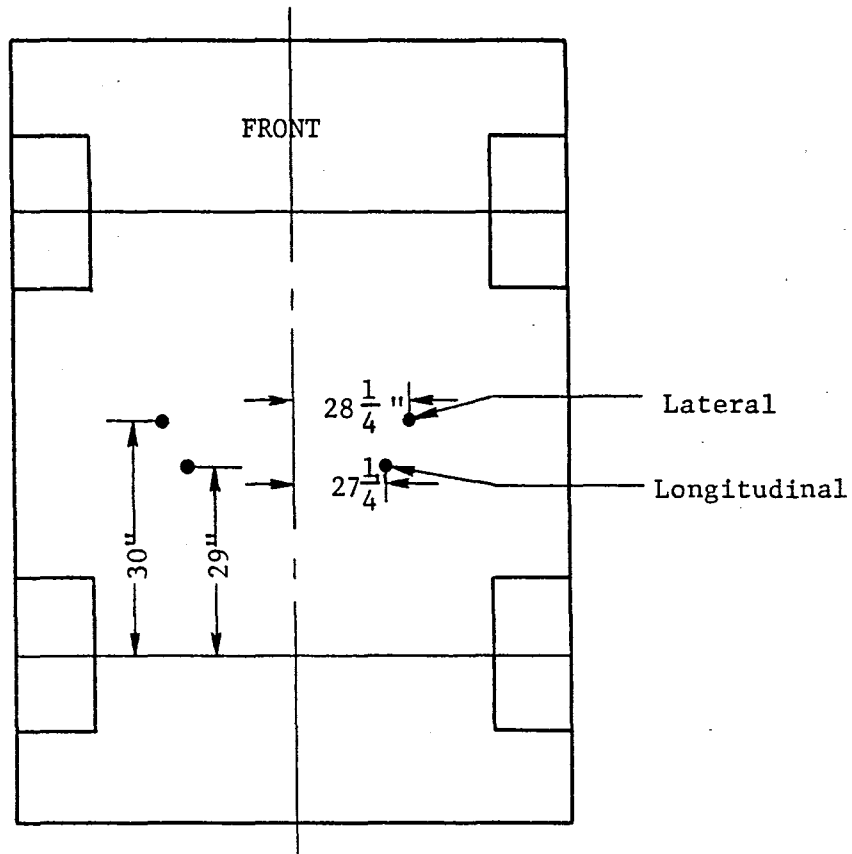


FIGURE C2- Vehicle data for tests impacting concrete median barrier at 61.9 mph at 7 deg, and at 59.6 mph at 15 deg.

APPENDIX D

SEVERITY INDEX

SEVERITY INDEX

An automobile acceleration severity-index was used in this study to quantify the relative severity an automobile impacting a traffic barrier. The severity-index takes into consideration the combined effects of the longitudinal, lateral, and vertical accelerations of the automobile at its center-of-mass. The severity-index (SI) is computed by Eq. D1.

$$SI = \sqrt{\left(\frac{G_{LONG}}{G_{XL}}\right)^2 + \left(\frac{G_{LAT}}{G_{YL}}\right)^2 + \left(\frac{G_{VERT}}{G_{ZL}}\right)^2} \quad (D1)$$

The terms in the numerator of Eq. D1 are the computed or measured accelerations of the automobile; whereas, the terms in the denominator are automobile accelerations which represent a "limiting condition" for the occupant.

An indepth discussion of the background and development of Eq. D1 was given in TTI Research Report 140-4 (7). Information relating tolerable accelerations to degree of occupant restraint, rate of onset or rise time, and time duration of accelerations was included in the discussion. In the study presented herein, the tolerable accelerations were for an unrestrained occupant, rise times greater than 0.03 seconds, and for a time duration of 0.050 seconds. The limit or tolerable accelerations for these conditions are (10):

$$\left. \begin{array}{l} G_{XL} = 7 \text{ G's} \\ G_{YL} = 5 \text{ G's} \\ G_{ZL} = 6 \text{ G's} \end{array} \right\} \quad (D2)$$

There has been much discussion as to the relationship of the severity-index to the probable level of occupant injury. The writers have interpreted an SI of unity to imply that occupants will sustain injuries which border on the "serious" type. Until more data are available on limit accelerations and the interaction relationship itself, there appears to be no other logical way to interpret the index.

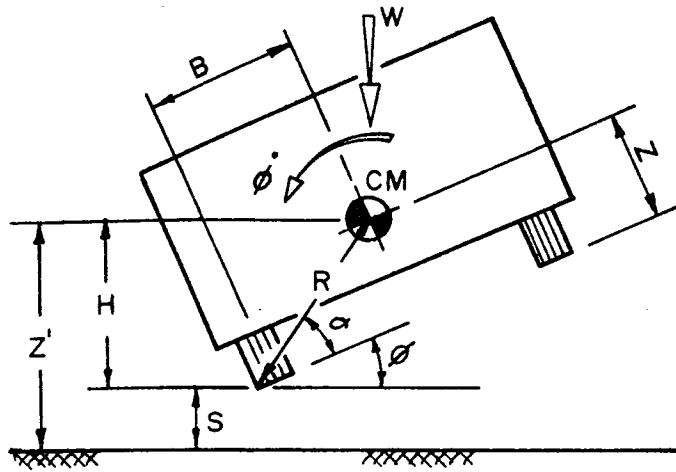
In addition, vehicle accelerations have never been translated into expected g-levels on the occupant, and until such a correlation becomes available the possible applications of the severity-index must be qualified. The index in its present form is intended for comparing the severity of one event to another and can also serve as an aide in making decisions concerning highway modifications which should effect a reduction in occupant injury and loss of life. However, it must be emphasized that the index, as defined by TTI researchers (here and elsewhere), has never been intended for direct assessment of human injury and therefore should not be used in that regard.

APPENDIX E

FORMULAS FOR MAXIMUM ROLL ANGLE

VALUES OBTAINED FROM PROGRAM AT TIME OF ITS TERMINATION

Z' (IN), ϕ (DEG), ϕ' (DEG/SEC)



$$\alpha = \tan^{-1} (Z/B)$$

$$R = Z / \sin \alpha$$

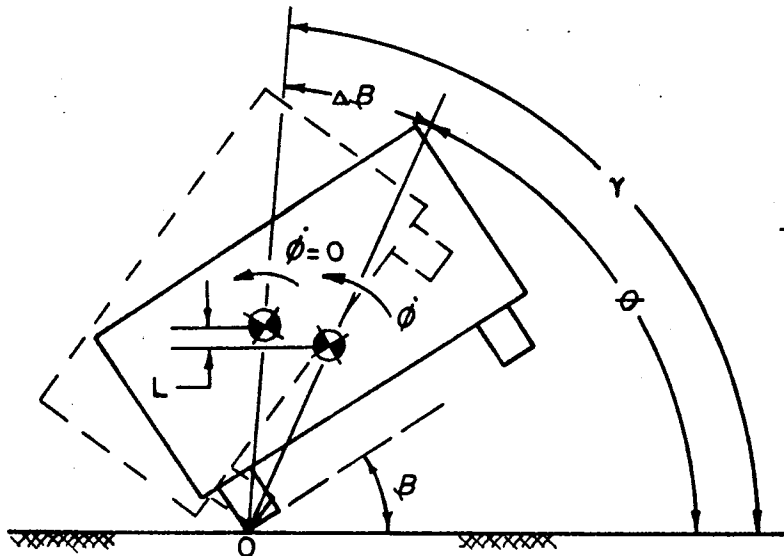
$$H = R \sin (\alpha + \phi)$$

$$S = Z' - H$$

ASSUMPTIONS

1. NEGLECT PITCH ANGLE AND PITCH RATE
2. NO VERTICAL VELOCITY ($Z' = \text{MAX}$)

FIGURE B I A. AUTO AIRBORNE



$$\beta = \phi + \sqrt{\frac{2S}{g}} \phi'$$

$$\theta = \beta + \alpha$$

$$L = R (\sin \gamma - \sin \theta)$$

$$I_0 = I_{CM} + \frac{W}{g} R^2$$

FIGURE B I B. MAXIMUM ROLL

KINETIC ENERGY = WORK DONE BY AUTO WEIGHT

$$\frac{1}{2} I_0 (\phi')^2 = WL = WR (\sin \gamma - \sin \theta)$$

$$\gamma = \sin^{-1} \left[\frac{1}{2} \frac{I_0 \phi'^2}{WR} + \sin \theta \right]$$

$$\text{MAX ROLL} = \beta + \Delta \beta$$

$$\text{MAX ROLL} = \beta + \gamma - \theta$$

IF: $\gamma > 90^\circ$ ROLLOVER WILL OCCUR

IF: $\beta > 90^\circ - \tan^{-1} (Z/B)$ ROLLOVER WILL OCCUR

FIGURE B I. PROCEDURE TO ESTIMATE MAXIMUM ROLL ANGLE

APPENDIX F

REFERENCES

APPENDIX F

References

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