A DISCRETE-ELEMENT METHOD FOR TRANSVERSE VIBRATIONS OF BEAM-COLUMNS RESTING ON LINEARLY ELASTIC OR INELASTIC SUPPORTS

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PREFACE

This study describes a method developed to analyze beam-columns subjected to either static fixed loads or dynamic loads. The supports of the beamcolumn may be either linearly elastic or nonlinear and inelastic. The method incorporates previously developed discrete-element beam-column techniques.

This work was done under Research Project 3-5-63-56, "Development of Methods for Computer Simulation of Beam-Columns and Grid-Beam and Slab Systems," conducted at the Center for Highway Research, The University of Texas at Austin, as part of the Cooperative Highway Research Program sponsored by the Texas Highway Department and the Department of Transportation Federal Highway Administration.

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11**1**

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LIST OF REPORTS

Report No. 56-1, "A Finite-Element Method of Solution for Linearly Elastic Beam-Columns" by Hudson Matlock and T. Allan Haliburton, presents a solution for beam-columns that is a basic tool in subsequent reports. September 1966.

Report No. 56-2, "A Computer Program to Analyze Bending of Bent Caps" by Hudson Matlock and Wayne B. Ingram, describes the application of the beamcolumn solution to the particular problem of bridge bent caps. October 1966.

Report No. 56-3, "A Finite-Element Method of Solution for Structural Frames" by Hudson Matlock and Berry Ray Grubbs, describes a solution for frames with no sway. May 1967.

Report No. 56-4, "A Computer Program to Analyze Beam-Columns under Movable Loads" by Hudson Matlcok and Thomas P. Taylor, describes the application of the beam-column solution to problems with any configuration of movable nondynamic loads. June 1968.

Report No. 56-5, "A Finite-Element Method for Bending Analysis of Layered Structural Systems" by Wayne B. Ingram and Hudson Matlock, describes an alternating-direction iteration method for solving two-dimensional systems of layered grids-over-beams and plates-over-beams. June 1967.

Report No. 56-6, 'Discontinuous Orthotropic Plates and Pavement Slabs" by W. Ronald Hudson and Hudson Matlock, describes an alternating-direction iteration method for solving complex two-dimensional plate and slab problems with emphasis on pavement slabs. May 1966.

Report No. 56-7, "A Finite-Element Analysis of Structural Frames" by T. Allan Haliburton and Hudson Matlock, describes a method of analysis for rectangular plane frames with three degrees of freedom at each joint. July 1967.

Report No. 56-8, "A Finite-Element Method for Transverse Vibrations of Beams and Plates" by Harold Salani and Hudson Matlock, describes an implicit procedure for determining the transient and steady-state vibrations of beams and plates, including pavement slabs. June 1968.

Report No. 56-9, "A Direct Computer Solution for Plates and Pavement Slabs" by C. Fred Stelzer, Jr., and W. Ronald Hudson, describes a direct method for solving complex two-dimensional plate and slab problems. October 1967.

Report No. 56-10, "A Finite-Element Method of Analysis for Composite Beams" by Thomas P. Taylor and Hudson Matlock, describes a method of analysis for composite beams with any degree of horizontal shear interaction. January 1968. Report No. 56-11, "A Discrete-Element Solution of Plates and Pavement Slabs Using a Variable-Increment-Length Model" by Charles M. Pearre, III, and W. Ronald Hudson, presents a method for solving freely discontinuous plates and pavement slabs subjected to a variety of loads. April 1969.

Report No. 56-12, "A Discrete-Element Method of Analysis for Combined Bending and Shear Deformations of a Beam" by David F. Tankersley and William P. Dawkins, presents a method of analysis for the combined effects of bending and shear deformations. December 1969.

Report No. 56-13, "A Discrete-Element Method of Multiple-Loading Analysis for Two-Way Bridge Floor Slabs" by John J. Panak and Hudson Matlock, includes a procedure for analysis of two-way bridge floor slabs continuous over many supports. January 1970.

Report No. 56-14, "A Direct Computer Solution for Plane Frames" by William P. Dawkins and John R. Ruser, Jr., presents a direct method of solution for the computer analysis of plane frame structures. May 1969.

Report No. 56-15, "Experimental Verification of Discrete-Element Solutions for Plates and Slabs" by Sohan L. Agarwal and W. Ronald Hudson, presents a comparison of discrete-element solutions with small-dimension test results for plates and slabs, including some cyclic data. April 1970.

Report No. 56-16, "Experimental Evaluation of Subgrade Modulus and Its Application in Model Slab Studies" by Qaiser S. Siddiqi and W. Ronald Hudson, describes a series of experiments to evaluate layered foundation coefficients of subgrade reaction for use in the discrete-element method. January 1970.

Report No. 56-17, "Dynamic Analysis of Discrete-Element Plates on Nonlinear Foundations" by Allen E. Kelly and Hudson Matlock, presents a numerical method for the dynamic analysis of plates on nonlinear foundations. July 1970.

Report No. 56-18, "A Discrete-Element Analysis for Anisotropic Skew Plates and Grids" by Mahendrakumar R. Vora and Hudson Matlock, describes a tridirectional model and a computer program for the analysis of anisotropic skew plates or slabs with grid-beams. August 1970.

Report No. 56-19, "An Algebraic Equation Solution Process Formulated in Anticipation of Banded Linear Equations" by Frank L. Endres and Hudson Matlock, describes a system of equation-solving routines that may be applied to a wide variety of problems by using them within appropriate programs. January 1971.

Report No. 56-20, "Finite-Element Method of Analysis for Plane Curved Girders" by William P. Dawkins, presents a method of analysis that may be applied to plane-curved highway bridge girders and other structural members composed of straight and curved sections. June 1971.

Report No. 56-21, "Linearly Elastic Analysis of Plane Frames Subjected to Complex Loading Conditions" by Clifford O. Hays and Hudson Matlock, presents a design-oriented computer solution for plane frames structures and trusses that can analyze with a large number of loading conditions. June 1971. Report No. 56-22, "Analysis of Bending Stiffness Variation at Cracks in Continuous Pavements," by Adnan Abou-Ayyash and W. Ronald Hudson, describes an evaluation of the effect of transverse cracks on the longitudinal bending rigidity of continuously reinforced concrete pavements. April 1972.

Report No. 56-23, "A Nonlinear Analysis of Statically Loaded Plane Frames Using a Discrete Element Model" by Clifford O. Hays and Hudson Matlock, describes a method of analysis which considers support, material, and geometric nonlinearities for plane frames subjected to complex loads and restraints. May 1972.

Report No. 56-24, "A Discrete-Element Method for Transverse Vibrations of Beam-Columns Resting on Linearly Elastic or Inelastic Supports" by Jack Hsiao-Chieh Chan and Hudson Matlock, presents a new approach to predict the hysteretic behavior of inelastic supports in dynamic problems. June 1972.

(P) indicates Preliminary Report.

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ABSTRACT

A discrete-element method of analysis for transverse vibration of beam columns resting on linearly or nonlinearly-elastic, or nonlinearly-inelastic, supports is presented. The applied forces include static fixed loads and timedependent dynamic loads. The program is an extension of programs BMCOL 43 (Research Report No. 56-4) and DBCl (Research Report No. 56-8) to cover inelastic supports and time-dependent axial thrusts. Two multi-element models are used to simulate the inelastic characteristics of supports, one which allows the beam to lift off the support when it deflects, upward or downward, and the other which considers the resistance to either upward or downward deflection. An internal damping factor, which is related to the first derivative with respect to time of the curvature of the beam, has been included, in addition to the conventional external viscous damping factor.

The method is based on an implicit difference formulation of the Crank-Nicolson type. A computer program has been written to check the validities of the proposed multi-element models of tracing the loading paths of the nonlinearlyinelastic supports and of the implicit formulation of the Crank-Nicolson type. The results compare well with the theoretical results and with experimental data.

KEY WORDS: dynamic, static, nonlinearly-inelastic supports, multi-element model, beam column, implicit formulation, discrete-element method, computers, piles, bridges, earthquake, wave forces.

ix

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SUMMARY

A computer program, DBC5, is presented which can efficiently analyze a beam-column resting on linearly or nonlinearly-elastic, or nonlinearly-inelastic supports and subjected to either static fixed loads or dynamic loads.

The path-dependent history of loading of the nonlinearly-inelastic resistance-deflection curve of the support is considered in this study. Two multi-element models, which are used to simulate the nonlinear characteristics of the inelastic resistance-deflection curves, have been introduced. For problems with nonlinearly-elastic or nonlinearly-inelastic supports, the iteration process compares successively computed deflections until a specified tolerance is satisfied. An option available in the program allows a switch from a spring-load-iteration process (adjusting both the stiffness and the load from one iteration to the next, which is known as tangent modulus method) to a load-iteration technique (adjusting only the load) when the supports yield or disconnect from the beam-column. An internal damping factor, which is related to the first time derivative of the curvature of the beam, has been considered in addition to the external viscous damping factor which is normally encountered in a dynamic problem.

The results of an analysis can include

- (1) solutions for the member under static fixed loads,
- (2) solutions for the member under dynamic and static loads at each time station, and
- (3) plots of computed deflections or moments along either the time or the beam axis as required.

Seven example problems typical of those encountered by highway and foundation designers illustrate the uses of the program and the options available. Included are a three-span beam loaded with an AASHO standard 2-D truck moving at a uniform speed of 60 mph, a simply supported steel rod loaded by axial pulses, a partially embedded steel pipe pile loaded by idealized wave forces, and a partially embedded steel pipe pile excited by sinusoidal, earthquakeinduced forces. A guide for data input is presented which allows routine application of the method of analysis with little necessary reference to the body of the main report. Any number of analyses may be run at the same time.

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IMPLEMENTATION STATEMENT

The utilization of numerical methods to describe computer models of problems in structural dynamics has been an interesting subject to which many structural engineers have devoted their efforts in recent years.

In this study, an efficient computer program, DBC5, is developed for the analysis of beam-column problems, static or dynamic, which have either linearlyelastic or nonlinearly-inelastic supports. The path-dependent history of loading of the nonlinearly-inelastic resistance-deflection curve of the support is considered. Potential applications include study of the dynamic response of actual truck-loaded bridges, prediction of the response of offshore piles tc wave forces, analysis of railroad loadings on continuous spans which are supported on soil foundations, analysis of transverse response of partially embedded piles to earthquake-induced forces, and prediction of the hysteresis effect of inelastic supports under pavement slabs.

Recommendations are made for further research in developing other better multi-element computer models for nonlinear supports so that buckling, fracture, softening, and relaxation (creep) of the support could also be considered in the program.

It is further recommended that this program be put into test use by designers of the Texas Highway Department to further evaluate its uses, and to investigate needed extensions or modifications to make it more usable for the practicing design engineer.

xiii

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TABLE OF CONTENTS

PREFACE
LIST OF REPORTS
ABSTRACT AND KEY WORDS
SUMMARY xi
IMPLEMENTATION STATEMENT
NOMENCLATURE
CHAPTER 1. INTRODUCTION
Purpose and Scope of Program DBC5
CHAPTER 2. DEVELOPMENT OF THE IMPLICIT OPERATOR
Discrete-Element Model for Dynamic Beam-Columns
CHAPTER 3. RESISTANCE-DEFLECTION CURVES FOR THE LATERAL SUPPORTS
Multi-Element Models
Curve
CHAPTER 4. DESCRIPTION OF PROGRAM DBC5
General41Procedure for Data Input41Tables of Data Input43Error Messages46Description of Problem Results47Plots of Results48

CHAPTER 5. EXAMPLE PROBLEMS

Example Problem 1. Inelastically Supported Mass Under Free			
Vibration	•	•	53
Initial Displacements			55
Example Problem 2-1. Lateral Vibration of a Simply	-	-	
Supported Beam with Axial Compression Force	•	•	55
Example Problem 2-2. Lateral Vibration of a Beam on			<i>c</i> 1
Example Problem 3. Three-Span Beam Loaded by an AASHO	•	•	61
Standard 2-D Truck Moving at a Speed of 60 mph	•	•	61
Example Problem 4-1. Simply Supported Steel Rod - Axial			
Pulse Base Time = 0.10 Second	•	•	72
Example Problem 4-2. Simply Supported Steel Rod - Axial Pulse Base Time - 0.026 Second			76
Example Problem 4-3. Simply Supported Steel Rod - Axial	•	•	70
Pulse Base Time = 0.006 Second		•	76
Example Problem 5. Partially Embedded Steel Pipe Pile			
Loaded by Wave Forces	•	•	76
Example Problem 6. Inree-Span Beam with One-way and Symmetric Nonlinear Supports Loaded with a Transient			
Pulse	•		90
Example Problem 7. Partially Embedded Steel Pipe Pile			
Excited by Sinusoidal Earthquake-Induced Forces	•	•	95
CHAPTER 6. SUMMARY AND CONCLUSTONS		_	107
	•	•	
REFERENCES	•	•	111

APPENDICES

Appendix A. Derivation of the Dynamic Implicit Operator	
Based on the Crank-Nicolson Implicit Formula	, 115
Appendix B. Derivation for Recursive Solution of Equations	
(Extracted from Appendix 2 of Ref 4)	. 127
Appendix C. Stability Analysis for the Dynamic Implicit	
Operator with Internal Damping Coefficient	. 133
Appendix D. Guide for Data Input for DBC5	. 141
Appendix E. Glossary of Notation for DBC5	. 169
Appendix F. Flow Charts and Listing of Deck of Program DBC5	. 175
Appendix G. Listing of Data for Example Problems	209
Appendix H. Partial Sample Computer Output for Example	
Problems	• 215
THE AUTHORS	. 229
THE AUTHORS	. 229

NOMENCLATURE

Symbol	Typical Units	Definition
^a k+1, ^a k-1		Coefficients in stiffness matrix
^b k+1, ^b k-1		Coeffi cient s in stiffness matrix
С	in-1b	Concentrated applied couple
^c _{k+1} , ^c _k , ^c _{k-1}		Coefficient in stiffness matrix
D	lb-in ²	Lumped internal damping factor divided by time increment length
(D ^e)	lb-sec/in	Lumped external viscous damping factor
(D ⁱ)	lb-in ² -sec	Lumped internal damping factor
^d k+1, ^d k-1		Coefficient in stiffness matrix
e		Base of natural logarithms
Е	lb/in ²	Modulus of elasticity
^e k+1, ^e k-1		Coefficient in stiffness matrix
f _{j,k}		Coefficient in stiffness matrix
F	lb-in ²	Flexural stiffness, EI
h	in	Beam increment length
^h t	sec	Time increment length
I	in ⁴	Moment of inertia of cross section
j		Beam station number
k		Time station number
Mj	in-lb	Static bending moment at beam station j
M _{j,k}	in-lb	Dynamic bending moment at beam station j and time station k

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Symbol	Typical Units	Definition
q ²		$2 \rho h^3 / h_t^2$
Q	1b	Concentrated applied static transverse load
Q^T	1Ь	Concentrated applied dynamic transverse load
Q_j^R , Q_j^S	1b	Static support reaction at beam station j
$Q_{j,k}^{R}, Q_{j,k}^{S}$	1b	Dynamic support reaction at beam station j and time station k
$\Delta Q_1^{\rm T}, \Delta Q_2^{\rm T}, \ldots$	1b	Retained projected values of resistance of segments 1, 2, on support curve
R	in-lb/rad	Concentrated rotational restraint
^R ₁ , ^R ₂ ,	1b	Resistances in the sub-springs 1, 2, of the nonlinear support
R_1^S, R_2^S	1b	Residual resistances in sub- springs 1, 2
R _A , R _B	1b	Resistances at points A and B on support curve
s ^s j	lb/in	Concentrated transverse linear spring restraint at beam station j
S ^N j,k	lb/in	Concentrated transverse nonlinear spring restraint at beam station j and time station k
s ₁ , s ₂ ,		Retained values of slopes of segments 1, 2, on support curve
Т	16	Static axial thrust
$\mathbf{T}^{\mathbf{T}}$	1b	Dynamic axial thrust
V j	1b	Static shear of bar j
V _{j,k}	1b	Dynamic shear of bar j at time station k
W j	in	Static transverse deflection at beam station j

1

CHAPTER 1. INTRODUCTION

In recent years, a great deal of interest has been focused on the utilization of numerical methods to describe computer models of problems in structural dynamics. Many computer programs (for instance, Refs 3 and 10) are available for solving for the dynamic response of beams or slabs which are either linearly or nonlinearly supported; no program has been found, however, which considers the path-dependent history of loading of nonlinearly-inelastic supports. As a result of the necessity to consider the important effects of energy lost due to the hysteretic behavior of the supports, much of the present work has been devoted to the development of multi-element models to simulate the nonlinearly-inelastic supports. With these models, general rules in FOR-TRAN logic can be observed and used to predict the loading paths of resistancedeflection curves of the supports. The models for the nonlinearly-inelastic supports described in this work are not time-dependent; therefore, relaxation (creep) of resistance of the support is not included in the model. The retardation, or delayed elasticity, of the support can be considered by installing an external viscous dashpot in parallel with the support model. The softening of the nonlinearly-inelastic support is also not included but strain-hardening may be considered.

Purpose and Scope of Program DBC5

The primary purpose of this investigation is to develop an efficient computer program for solving for the transverse dynamic response of a beam-column resting on linearly or nonlinearly-elastic or nonlinearly-inelastic supports which are simulated by the proposed multi-element models described in Chapter 3. When the hysteretic behavior of the supports is considered, the program is able to predict more accurately the dynamic response of beams, piles, slabs, or even bridges which are supported by soil foundations.

A marching method of solution is used which is based on an implicit formula introduced by Crank and Nicolson (Ref 2) to solve second-order heat flow problems. Salani (Ref 10) is credited with applying this implicit

1

formula in determining the transverse time-dependent linear deflections of a beam or plate. Essentially, the beam is replaced by an arbitrary number of rigid bars and deformable joints, and time is divided into discrete, equal intervals. The representation readily permits the flexural stiffness, the elastic restraints, the mass densities, and the applied external loadings to be discontinuous and lumped at the deformable joints which connect the rigid bars. The governing partial differential equation at each joint is approximated by a difference equation that includes several unknown deflections of the joint, which occur at specified time intervals. All difference equations are based on the assumptions of linear elasticity and the elementary beam theory. The effects of transverse shear and rotatory inertia are neglected.

The nonlinear characteristics of the supports are considered by using either a spring-load iteration process (adjusting both the stiffness and the load from one iteration to the next, which is known as tangent modulus method) or a load-iteration technique (adjusting only the load). The iteration process compares successively computed deflections until a specified tolerance is satisfied. Only three nonlinear characteristics of support curves are considered: the first is exhibiting the same resistance to either upward to downward deflection, hereafter referred to as the symmetric resistance-deflection curve; the second is allowing the beam to lift off the support when it deflects upward, hereafter referred to as the negative one-way resistancedeflection curve; the third is allowing the beam to lift off the support when it deflects downward, hereafter referred to as positive one-way resistancedeflect ion curve. Two multi-element models to simulate the three types of support are introduced in this work.

Application

Computer Program DBC5 is versatile and efficient for

- (1) solving for the transverse response of beam-columns with linear and nonlinear supports under free or forced vibration;
- (2) computing slopes and shears of the bars, bending moments, and support reactions of the deformable joints, statically or dynamically; and
- (3) plotting computed deflections or moments along either the time or beam axis for the requested monitor stations.

Applied forces include static fixed loads, time variant axial thrusts, and time variant lateral loads. Static solutions of beam-columns under fixed loads may also be obtained.

The computer program is intended to provide an efficient tool for analyzing many problems encountered by highway and foundation designers which are complicated and unsolvable using classical methods. A variety of highway structures, such as bridge girders, guard rails, or even a whole bridge floor under moving loads or suddenly applied impact, can be simulated by the computer model of Program DBC5 and solved efficiently. The capability to treat supports that behave nonlinearly and inelastically provides for direct solutions of transverse deflections of railroad rails under moving loads and the prediction of the responses of offshore piles to wave forces, as well as the study of transverse responses of piles to earthquake-induced forces, since the parts of rails and piles supported by the soil can be reasonably represented by the proposed multi-element models for considering the path-dependent history of loading of the soil supports.

3

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CHAPTER 2. DEVELOPMENT OF THE IMPLICIT OPERATOR

Program DBC5 is developed for the dynamic analysis of beam-columns resting on linearly-elastic or nonlinear supports under time variant axial thrusts and lateral loads. The program uses a discrete-element model for developing the equation of motion of beams. An implicit formula of the Crank-Nicolson (Ref 2) type is then utilized to form a marching operator. At each point in time, a set of simultaneous equations for the unknown deflections at the deformable joints can be obtained by systematically applying the marching operator at all joints. A recursive procedure is then used to solve the simultaneous equations. A brief discussion of the method of the recursive solution procedure is included in Appendix B.

Discrete-Element Model for Dynamic Beam-Columns

Matlock and Taylor have developed a static model composed of rigid bars and springs (Fig 1) which can be used to simulate a beam-column. In Ref 5 the efficiency of this model has been proved by solving a variety of structures which can be simulated as beam-columns. By the addition of masses and damping factors lumped at the deformable joints, a dynamic model (Fig 2) is formed that can closely simulate the dynamic response of real beam-columns.

Equation of Motion of Beam-Columns

The equation of motion for transverse vibration of beams can be obtained by summing all the forces, internal and external, at a particular joint j and a particular time station k (see Fig 3). The concept is based on D'Alembert's principle. Thus an equation can be written in terms of the unknown deflections w, moments M, internal damping factors D^{i} ...,

$$-M_{j-1} + 2M_{j} - M_{j+1} - (D^{i})_{j-1} \frac{d}{dt} (\varphi_{j-1}, k) + 2(D^{i})_{j} \frac{d}{dt} (\varphi_{j,k})$$





(b)



Fig 1. Mechanical model of beam-column.



Fig 2. Dynamic model of beam-column.



Fig 3. Free-body diagram of a portion of the dynamic beam-column model.

$$- \langle D^{i} \rangle_{j+1} \frac{d}{dt} \langle \varphi_{j+1,k} \rangle - \langle T_{j} + T_{j,k}^{T} \rangle \langle -w_{j-1,k} + w_{j,k} \rangle$$

$$+ \langle T_{j+1} + T_{j+1,k}^{T} \rangle \langle -w_{j,k} + w_{j+1,k} \rangle + hQ_{j,k}^{T} + hQ_{j}$$

$$+ 1/2 \langle -R_{j-1}\theta_{j-1} + R_{j+1}\theta_{j+1} \rangle + 1/2 \langle -C_{j-1} + C_{j+1} \rangle - hS_{j}^{s}w_{j,k}$$

$$- hS_{j,k}^{N}w_{j,k} - h\rho_{j} \frac{d^{2}}{dt^{2}} \langle w_{j,k} \rangle - h\langle D^{e} \rangle_{j} \frac{d}{dt} \langle w_{j,k} \rangle = 0$$
(2.1)

By the use of Crank-Nicolson's implicit formula, Eq 2.1 can be further represented as

$$a_{k+1}^{w}_{j-2,k+1} + b_{k+1}^{w}_{j-1,k+1} + c_{k+1}^{w}_{j,k+1} + d_{k+1}^{w}_{j+1,k+1} + e_{k+1}^{w}_{j+2,k+1}$$

$$= c_{k}^{w}_{j,k} + a_{k-1}^{w}_{j-2,k-1} + b_{k-1}^{w}_{j-1,k-1} + c_{k-1}^{w}_{j,k-1}$$

$$+ d_{k-1}^{w}_{j+1,k-1} + e_{k-1}^{w}_{j+2,k-1} + f_{j,k}$$
(2.2)

where

.

$$\begin{aligned} \mathbf{a_{k+1}} &= \mathbf{F_{j-1}} + \frac{1}{h_t} \left(\mathbf{D^i} \right)_{j-1} - \mathbf{0.25hR_{j-1}} \\ \mathbf{b_{k+1}} &= -2 \left(\mathbf{F_{j-1}} + \mathbf{F_j} \right) - \frac{2}{h_t} \left[\left(\mathbf{D^i} \right)_{j-1} + \left(\mathbf{D^i} \right)_{j-1} - \mathbf{h^2_{L}T_j} + \mathbf{0.5} \left(\mathbf{T_{j,k-1}^T} + \mathbf{T_{j,k+1}^T} \right) \right] \\ \mathbf{c_{k+1}} &= \left(\mathbf{F_{j-1}} + 4\mathbf{F_j} + \mathbf{F_{j+1}} \right) + \frac{1}{h_t} \left[\left(\mathbf{D^i} \right)_{j-1} + 4 \left(\mathbf{D^i} \right)_{j} + \left(\mathbf{D^i} \right)_{j+1} \right] \\ &+ \mathbf{h^2 L_T_j} + \mathbf{T_{j+1}} + \mathbf{0.5} \left(\mathbf{T_{j,k-1}^T} + \mathbf{T_{j,k+1}^T} + \mathbf{T_{j+1,k-1}^T} + \mathbf{T_{j+1,k+1}^T} \right) \right] \end{aligned}$$

$$+ 0.25h(R_{j-1} + R_{j+1}) + 2h^{3} \frac{\rho_{j}}{h_{t}^{2}} + h^{3}(D^{e})_{j}/h_{t} + 0.5h^{3}[(s_{j}^{s} + s_{j,k-1}^{N})]$$

$$+ (s_{j}^{s} + s_{j,k+1}^{N})_{j}$$

$$d_{k+1} = -2(F_{j} + F_{j+1}) - \frac{2[(D^{i})_{j} + (D^{i})_{j+1}]}{h_{t}}$$

$$- h^{2}[T_{j+1} + 0.5(T_{j+1,k-1}^{T} + T_{j+1,k+1}^{T})]$$

$$e_{k+1} = F_{j+1} + \frac{(D^{i})_{j+1}}{h_{t}} - 0.25hR_{j+1}$$

$$= -3$$

$$c_k = \frac{4h^3 \rho_1}{h_t^2}$$

$$a_{k-1} = -a_{k+1} + \frac{2 \langle D^{i} \rangle_{j-1}}{h_{t}}$$

$$\mathbf{b}_{k-1} = -\mathbf{b}_{k+1} - \frac{4\left[\left(\mathbf{D}^{i}\right)_{j-1} + \left(\mathbf{D}^{i}\right)_{j}\right]}{\mathbf{h}_{t}}$$

$$\mathbf{c_{k-1}} = -\mathbf{c_{k+1}} + \frac{2\left[\left(\mathbf{D^{i}}\right)_{j-1} + 4\left(\mathbf{D^{i}}\right)_{j} + \left(\mathbf{D^{i}}\right)_{j+1}\right]}{\mathbf{h_{t}}} + \frac{2\mathbf{h^{3}}\left(\mathbf{D^{e}}\right)_{j}}{\mathbf{h_{t}}}$$

$$\mathbf{d}_{k-1} = -\mathbf{d}_{k+1} - \frac{4\left[\left(\mathbf{D}^{i}\right)_{j} + \left(\mathbf{D}^{i}\right)_{j+1}\right]}{\mathbf{h}_{t}}$$

$$e_{k-1} = -e_{k+1} + \frac{2(D^{i})_{j+1}}{h_{t}}$$

$$f_{j,k} = h^{3} \left[\left(Q_{j} + Q_{j,k-1}^{T} \right) + \left(Q_{j} + Q_{j,k+1}^{T} \right) \right] + h^{2} \left(-C_{j-1} + C_{j+1} \right)$$

Detailed derivations of Eqs 2.1 and 2.2 are included in Appendix A. In Fig 4, an implicit operator of the Crank-Nicolson (Ref 2) type is shown for Eq 2.2.

To start the dynamic solutions, a static model as described in Ref 2 is used twice, at time stations -2 and -1, for solving the initial deflected shape of the beam-column due to static loads. Since the implicit operator requires the deflected shape at only the two previous time stations, there are no considerations of the initial velocities and initial accelerations of the deformable joints of the beam. This is consistent with normal practice, since deformable joints of a beam normally have no initial velocities or initial accelerations.

All deflections at time station k+1 are unknown. To solve for the unknown deflections at time station k+1, the operator, which requires the deflected shapes of the beam at time stations k and k-1 to be known, is applied systematically at beam joints $j = -1, 1, 2 \dots M+1$. This procedure establishes a set of simultaneous equations wherein each equation includes five unknown deflections. These equations are solved by a two-pass, recursive technique described in Ref 4 (see Appendix B) for the unknown deflection of every joint. Once the deflected shape of a member at time station k+1 is known, the slopes, bending moments, shears, and support reactions for the member at the previous time station k can be determined by using the procedures described below.

<u>Slope</u>. Equation 2.3 is the simple-difference expression for slope of the individual bar j in terms of the deflections of joints j-1 and j and the beam increment length h (see Fig 1(c)).

$$\theta_{j} = \frac{-w_{j-1} + w_{j}}{h}$$
(2.3)

The concept applied for both the static model and the dynamic model. It will be seen in results printed from the program that the slope is printed between the beam stations; this slope is that of the bar between the indicated beam stations.



 ${}^{a}_{k+1} {}^{w}_{j-2,k+1} + {}^{b}_{k+1} {}^{w}_{j-1,k+1} + {}^{c}_{k+1} {}^{w}_{j,k+1} + {}^{d}_{k+1} {}^{w}_{j+1,k+1} + {}^{e}_{k+1} {}^{w}_{j+2,k+1}$ $- {}^{c}_{k} {}^{w}_{j,k} + {}^{a}_{k-1} {}^{w}_{j-2,k-1} + {}^{b}_{k-1} {}^{w}_{j-1,k-1} + {}^{c}_{k-1} {}^{w}_{j,k-1} + {}^{d}_{k-1} {}^{w}_{j+1,k-1}$ $+ {}^{e}_{k-1} {}^{w}_{j+2,k-1} + {}^{f}_{j,k}$

Fig 4. Implicit operator of Crank-Nicolson type used in Program DBC5.

Bending Moment. In conventional beams, the bending moment is equal to the product of the flexural stiffness and the curvature. In the finite-element model the flexibility of the beam and the curvature are lumped at the beam station points. The corresponding relation for bending moment M in the static model is

$$M_{j} = \frac{F_{j}}{h} \left(\frac{w_{j-1} - 2w_{j} + w_{j+1}}{h} \right)$$
(2.4)

In the dynamic model, the internal dampling factor lumped at the joint contributes its effect in addition to the conventional bending moment. Thus,

$$M_{j,k} = \frac{F_{j}}{h} \left(\frac{w_{j-1,k} - 2w_{j,k} + w_{j+1,k}}{h} \right) + \frac{(D^{i})_{j}}{2h^{2}h_{t}} \left(-w_{j-1,k-1} + 2w_{j,k-1} - w_{j+1,k-1} + w_{j-1,k+1} - 2w_{j,k+1} + w_{j+1,k+1} \right)$$

$$(2.5)$$

Shear. Shear V in the static model is found from the equation of moment equilibrium of bar j (Fig 1(b)). Thus,

$$V_{j} = \frac{-M_{j-1} + M_{j}}{h} - T_{j} \left(\frac{-W_{j-1} + W_{j}}{h} \right)$$
(2.6)

For the dynamic model, in addition to the effects of conventional bending moments and static axial thrusts, the moments contributed by the internal damping factors $(D^i)_{j-1}$ and $(D^i)_j$, and the time dependent axial thrusts $T^T_{j,k}$ are also found in the equation of moment equilibrium of bar j (Fig 3). Thus,

$$V_{j,k} = \frac{-M_{j-1,k} + M_{j,k}}{h} - \left[T_j + T_{j,k}^T\right] \left(\frac{-W_{j-1,k} + W_{j,k}}{h}\right)$$

$$- \left(D^{i} \right)_{j-1} \frac{d}{dt} \left(\frac{w_{j-2,\kappa} - 2w_{j-1,k} + w_{j,k}}{h^{3}} \right)$$

$$+ \left(D^{i} \right)_{j} \frac{d}{dt} \left(\frac{w_{j-1,k} - 2w_{j,k} + w_{j+1,k}}{h^{3}} \right)$$

$$(2.7)$$

Substituting

$$T_{j,k}^{T} = \frac{1}{2} \left(T_{j,k-1}^{T} + T_{j,k+1}^{T} \right),$$

$$\frac{d}{dt} \left(\frac{w_{j-2,k} - 2w_{j-1,k} + w_{j,k}}{h^{3}} \right) = \frac{1}{2h_{t}h^{3}} \left(-w_{j-2,k-1} + 2w_{j-1,k-1} + w_{j,k+1} + w_{j,k+1} \right)$$

and

$$\frac{d}{dt} \left(\frac{w_{j-1,k} - 2w_{j,k} + w_{j+1,k}}{h^3} \right) = \frac{1}{2h_t h^3} \left(-w_{j-1,k-1} + 2w_{j,k-1} \right)$$
$$-w_{j+1,k-1} + w_{j-1,k+1} - 2w_{j,k+1} + w_{j+1,k+1} \right)$$

into Eq 2.7, thus,

.

$$\mathbf{v}_{\mathbf{j},\mathbf{k}} = \frac{-M_{\mathbf{j}-\mathbf{1},\mathbf{k}} + M_{\mathbf{j},\mathbf{k}}}{h} - \left[\mathbf{T}_{\mathbf{j}} + 0.5\left(\mathbf{T}_{\mathbf{j},\mathbf{k}-1}^{\mathrm{T}} + \mathbf{T}_{\mathbf{j},\mathbf{k}+1}^{\mathrm{T}}\right)\right] \left(\frac{-w_{\mathbf{j}-\mathbf{1},\mathbf{k}} + w_{\mathbf{j},\mathbf{k}}}{h}\right)$$

$$- \frac{(D^{i})_{j-1}}{2h_{t}h^{3}} (-w_{j-2,k-1} + 2w_{j-1,k-1} - w_{j,k-1} + w_{j-2,k+1})$$

$$-2w_{j-1,k+1} + w_{j,k+1} + \frac{(D^{i})_{j}}{2h_{t}h^{3}} (-w_{j-1,k-1} + 2w_{j,k-1} - w_{j+1,k-1})$$

+
$$w_{j-1,k+1} - 2w_{j,k+1} + w_{j+1,k+1}$$
 (2.8)

As seen in Fig 1(b) (or Fig 3), V_j (or $V_{j,k}$) is the shear throughout the length of bar j. Therefore Program DBC5 is written such that the shear computed in each bar is printed between the adjacent beam stations. In the program, the rotational restraints and the applied couples, which conventionally are considered to be concentrated at a point, are acting on the beam as equal and opposite loads separated by two increments, as shown in Fig 5. Therefore, the shear for only the bars adjacent to beam stations with applied couples or rotational restraints is affected by these loads and is not the same as conventional shear.

<u>Support Reaction</u>. As described in Ref 5, the support reaction for the static model can be obtained by Eq 2.9 or Eq 2.10.

If joint j is supported on a linear spring S_{i}^{s} , the reaction Q_{i}^{S} is

$$Q_{j}^{S} = -S_{j}^{S}W_{j}$$
(2.9)

If joint j is supported on a nonyielding support (deflection w equal to zero), the support reaction is

$$Q_{j}^{R} = \frac{M_{j-1} - 2M_{j} + M_{j+1}}{h} - Q_{j} + \frac{C_{j-1} - C_{j+1}}{2h}$$
$$- \frac{M_{j-1}w_{j-2} - M_{j-1}w_{j} - M_{j+1}w_{j} + M_{j+1}w_{j+2}}{4h^{2}}$$
$$- \frac{T_{j}w_{j-1} - T_{j}w_{j} - T_{j+1}w_{j} + T_{j+1}w_{j+1}}{h}$$
(2.10)



(a) Mechanical model.



(b) Equivalent forces.

Fig 5. Rotational resistance R and applied couple C acting on the mechanical model.

For the dynamic model, the support reaction can be obtained by Eq 2.11 or Eq 2.12.

If joint j is supported on a linear (S_j^s) or nonlinear $(S_{j,k}^N)$ spring or both $(S_j^s + S_{j,k}^N)$, the support reaction $Q_{j,k}^s$ is

$$Q_{j,k}^{S} = -S_{j}^{s}W_{j,k}$$

or

$$s_{j,k}^{N}w_{j,k} + q_{i,k}^{S}$$
 or $-(s_{j}^{s} + s_{j,k}^{N})w_{j,k} + q_{i,k}^{S}$ (2.11)

where $Q_{i,k}^S$ is the iterative correction reaction of the nonlinear support at time station k .

If joint j is supported on a nonyielding support, the support reaction is

$$Q_{j,k}^{R} = \frac{M_{j-1,k} - 2M_{j,k} + M_{j+1,k}}{h} - Q_{j} + \frac{C_{j-1} - C_{j+1}}{2h}$$

$$- \left(\frac{Q_{j,k-1}^{T} + Q_{j,k+1}^{T}}{2}\right) - \frac{1}{4h^{2}} \left(R_{j-1}w_{j-2,k} - R_{j-1}w_{j,k} - R_{j+1}w_{j,k}\right)$$

$$+ R_{j+1}w_{j+2,k}\right) - \frac{1}{h} \left(T_{j}w_{j-1,k} - T_{j}w_{j,k} - T_{j+1}w_{j,k} + T_{j+1}w_{j+1,k}\right)$$

$$+ \frac{\left(D^{i}\right)_{j-1}}{h^{3}2h_{t}} \left(-w_{j-2,k-1} + 2w_{j-1,k-1} - w_{j,k-1} + w_{j-2,k+1}\right)$$

$$- {}^{2w}_{j-1,k+1} + {}^{w}_{j,k+1} - \frac{{}^{2} {}^{(b)}_{j}}{{}^{h^{3}}_{2h}_{t}} \left(- {}^{w}_{j-1,k-1} + {}^{2w}_{j,k-1} \right)$$

$$= w_{j+1,k-1} + w_{j-1,k+1} - 2w_{j,k+1} + w_{j+1,k+1})$$

$$+ \frac{(p^{i})_{j+1}}{h^{3}2h_{t}} (-w_{j,k-1} + 2w_{j+1,k-1} - w_{j+2,k-1} + w_{j,k+1})$$

$$= 2w_{j+1,k+1} + w_{j+2,k+1}) - \frac{1}{h} \left[0.5 (T_{j,k-1}^{T} + T_{j,k+1}^{T}) (w_{j,k} - w_{j+1,k}) \right]$$

$$(w_{j-1,k} - w_{j,k}) = 0.5 (T_{j+1,k-1}^{T} + T_{j+1,k+1}^{T}) (w_{j,k} - w_{j+1,k})]$$

$$+ \frac{\rho_{j}}{h_{t}^{2}} (w_{j,k-1} - 2w_{j,k} + w_{j,k+1}) + \frac{(p^{e})_{j}}{2h_{t}} (-w_{j,k-1} + w_{j,k+1})$$

$$+ w_{j,k+1})$$

$$(2.12)$$

Internal and External Damping

In Eq 2.1, two symbols $(D^i)_j$ and $(D^e)_j$ are introduced to represent the internal damping coefficients and the external viscous damping coefficients, which are both lumped at joint j. The rheological models of the two damping constants are shown in Fig 6. The typical units of $(D^i)_j$ and $(D^e)_j$ are $1b-in^2$ -sec/sta and 1b-sec/in/sta. The internal damping is due to the internal dynamic viscosity of materials. Boltzmann first proposed the hereditary theory which attributed the loss of energy, due to internal damping, to the elastic delay by which the deformation lagged behind the applied force.

Coulomb also proposed a viscous theory which assumed that the viscosity effects are proportional to the first time derivative of strain. The coefficient of proportionality (constant for each material at constant temperature) is called the coefficient of viscosity. In Ref 13, Volterra has shown a mathematical relationship between the viscous and hereditary damping theories. The relationship of stress σ to strain ε for the material based on the hereditary theory can be assumed as


(a)



(b)

Fig 6. Rheological models of internal and external damping coefficients.

$$\sigma = E \varepsilon + \sum_{n=1}^{\infty} Pn \frac{d^{n-1}\varepsilon}{dt^{n-1}}$$
(2.13)

where

$$Pn = \frac{(-1)^{n-1}}{(n-1)!} \int_{0}^{To} \tau^{n-1} \phi(\tau) d\tau$$

 τ is an instant of time between 0 and To , To is the period of heredity $(\phi(\tau) = 0 \text{ for } \tau > \text{To})$, and $\phi(\tau)$ is the memory function which can be found from the experimental data.

If the coefficients Pn with n > 1 can be neglected, Eq 2.13 reduces to

$$\sigma = E\varepsilon + z \frac{d\varepsilon}{dt}$$
(2.14)

where $z = P_1$. Equation 2.14 is the stress-strain relationship of the material based on viscous theory. Therefore, viscous theory is included in hereditary theory.

In the rheological model shown in Fig 6(a), the internal damping coefficients $(D^{i})_{j}$ are related to the first time derivative of curvature. Thus

$$M_{j} = F_{j}\varphi_{j} + (D^{i})_{j} \frac{d\varphi_{j}}{dt}$$
(2.15)

gives the relationship between the moment M. and the curvature φ_{i} .

Equation 2.15 is used in Program DBC5 for calculating the bending moments contributed by the flexural stiffness and the internal damping coefficient at the deformable joint j.

It will be shown in problem 4 described in Chapter 5 that the internal damping factors have little effect on the solutions of lower frequencies of a vibrating steel beam. The vibrations at higher frequencies, however, are damped out with time due to the effects of internal damping and rapid changes of curvatures. The external viscous damping force is defined as $f(w_{j,k}) = -(D^e)_{j}w_{j,k}$. The constant D^e (in tons per unit velocity, lb-sec/inch, or kip-sec/ft) is called the coefficient of viscous damping. This type of damping occurs for small velocities in lubricated sliding surfaces, dashpots, and hydraulic shockabsorbers. The so-called viscous resistances are produced by the slow motion of immersed bodies in fluid, either liquid or gas. The value of the coefficient $(D^e)_j$ depends essentially on the nature of the fluid, as well as on the form and the dimensions of the immersed body.

With these two types of damping factors, the Program DBC5 can solve the problems characterized by the so-called visco-elastic nonlinearity to a great extent.

Linearly-Elastic and Nonlinearly-Inelastic Supports

In the discrete-element model, lateral supports of various types can be represented by either an equivalent linearly-elastic spring or a nonlinearlyinelastic spring at each of the beam stations. For example, if the beam is supported by columns or piers, the axial stiffnesses of the columns or piers can be approximately estimated and included as equivalent linear spring constants. For more reality, the true behavior of many lateral supports can be better represented by the nonlinear characteristics of the axial resistancedeflection curves of columns, or piers, and soil supports.

If the beam is laterally supported by a soil foundation, better results are obtained by using the nonlinear characteristics of the resistance-deflection curves of the soil than by using the equivalent linear spring constants of the soil resistances, since these characteristics represent the true behavior of the soil supports. In Program DBC5, the nonlinear characteristics of each lateral support are described by a curve consisting of straight line segments. Only three types of the nonlinear characteristics of the lateral supports, symmetric, negative one-way, and positive one-way, are considered in this work.

The symmetrical nonlinear resistance-deflection curve is shown in Fig 7. The force developed against the beam-column model by the supports is plotted on the vertical axis and the model deflection is plotted on the horizontal axis. For both load and deflection, the positive sense is upward. The positive and negative one-way resistance-deflection curves are shown in Fig 8.



Fig 7. Symmetric resistance-deflection curve.



(a) Negative one-way resistance-deflection curve.



(b) Positive one-way resistance-deflection curve.

For the negative one-way support, resistance to deflection is developed only when deformable joints deflect in the negative or downward direction; for the positive one-way support, resistance to deflection is developed only when deformable joints deflect in the positive or upward direction. In order to be consistent with the multi-element model (see Chapter 3) used to simulate the nonlinear resistance-deflection curve of the supports, several limitations are placed on the nonlinear characterization:

- (1) The curve must pass through the origin.
- (2) The curve must be continuously concave as viewed from the horizontal axis, except that horizontal lines of zero stiffness can be input for representing ideally plastic behavior.
- (3) No softening of supports is permitted; that is, no reversal of slope sign of the segments on the support curve is permitted.

Stability of the Implicit Operator

Reference 10 has shown that when a uniform beam with well-defined boundary conditions under free vibration is analyzed using the implicit operator of the Crank-Nicolson form, the solution is stable for all positive values of EI , ρ , h , and ht .

If the same beam, with internal damping $(D^i)_j$ lumped at joints, is under free vibration, the quadratic equation for evaluation of the stability criterion becomes

$$e^{2\phi} + e^{\phi} \left[\frac{2q^2}{4(F+D)(\cos\beta_n - 1)^2 + q^2} \right]$$
$$+ \left[\frac{4(F-D)(\cos\beta_n - 1)^2 + q^2}{4(F+D)(\cos\beta_n - 1)^2 + q^2} \right] = 0$$

(2.16)

where

$$F = EI$$

$$D = (D^{i})_{j}/ht$$

$$q^{2} = 2\rho h^{3}/h_{t}^{2}$$

The derivation of Eq 2.16 is included in Appendix C.

From Eq 2.16, the condition for bounded solutions as time approaches infinity becomes

$$16(F^2 - D^2)(\cos \beta_n - 1)^2 + 8Fq^2 \ge 0$$
(2.17)

The above inequality is true, since D is usually less than F for a reasonable time increment length h_t , and the positive value of the term $8Fq^2$ is always greater than the negative value of the term $16(F^2 - D^2)(\cos \beta_n - 1)^2$ when h_t is extremely small.

For more complicated cases, such as a beam with internal damping, nonlinear spring supports, and rotational restraints under forced vibration, the analytical proofs for the stability of the implicit operator are not feasible. However, stable numerical solutions have been obtained for most complex practical problems that are physically stable. For beams with nonlinear supports under forced vibration, cautious selection of a reasonably small time increment length (for instance, less than 1/10 of the fundamental period) must be made, since the basic assumption of the Crank-Nicolson implicit formula is that the time-dependent forcing function and the time variant nonlinear springs are smoothly varied with time. Extremely small time increment length is not possible for most practical problems due to the present limitation of a maximum of 1000 time stations provided in Program DBC5. This page replaces an intentionally blank page in the original. -- CTR Library Digitization Team CHAPTER 3. RESISTANCE-DEFLECTION CURVES FOR THE LATERAL SUPPORTS

Three types of lateral supports are considered in this work: linearlyelastic, nonlinearly-elastic, and nonlinearly-inelastic. Chapter 2 discusses how the nonlinear support characteristics can be reasonably described by nonlinear resistance-deflection curves which are approximated by a series of straight-line segments. This chapter introduces two multi-element models each of which is made up of several sub-elements that can be used to simulate the nonlinearly-inelastic characteristics of either a symmetric or a one-way resistance-deflection curve. A short discussion of the Baushinger effect (Refs 9 and 11) as it relates to the formation of the loading paths of nonlinearlyinelastic resistance-deflection curves is included. Finally, a systematic approach to tracing the loading paths of nonlinearly-inelastic resistancedeflection curves in dynamic problems is explained.

Multi-Element Models

A multi-element model consisting of several parallel sub-elements for simulating the symmetric resistance-deflection curve is shown in Fig 9(a). Each sub-element has a linear spring connected in series with a Coulomb friction block. The sub-elements are assumed to behave as perfectly elasticplastic and their resistance-deflection curves are shown in Fig 9(c). We see that it is possible to represent a symmetric resistance-deflection curve with a number of perfectly elastic-plastic simple elements in parallel. Buckling and fracture phenomena are not considered in this work. Now let us study the behavior of the sub-elements. If the multi-element model is starting to deform downward, all of the four sub-springs will deform elastically and the resistance will be built up to point A, as shown in Fig 9(b).

At this point the resistance force in the sub-spring S_1 is just equal to the friction force created at the sliding surface of the Coulomb friction block. Beyond the point A the entire sub-element 1 will slide and no resistance, therefore, will be taken by this sub-element. If the model continues to deform downward the resistance will be taken by sub-springs S_2 , S_3 , and S_4



Fig 9. Multi-element model of symmetric nonlinear resistance-deflection curve of the support.

up to point B. At point B the sub-element 2 will start to slide. Beyond the point B the resistance will be taken by sub-springs S_3 and S_4 up to point C. At point C the sub-element 3 will also start to slide. Beyond point C the resistance will be taken only by sub-spring S_4 up to point D. Finally at point D the entire support will start to yield plastically and offer no more resistance to the deflection. Based on the above assumption, a continuously-straight-line-segment curve which represents the nonlinear characteristics of the support can be easily obtained by the summation of all ideally elastic-plastic resistance-deflection curves of the sub-elements. The procedure of constructing such a curve is shown in Fig 9(b). Theoretically, the more segments there are in a curve, the smoother the representation of the nonlinear characteristics of a maximum of 19 straight-line segments (9 sub-elements) is suitable to represent the nonlinear characteristics.

A multi-element model, as shown in Fig 10(a), is used to simulate the negative one-way resistance-deflection curve. The model is similar to the one used in the symmetric case except that there is no connection between the springs and the friction blocks. The spring-blocks work the same way as the ones used in the symmetric case when the deflection is downward (negative). While the deflection is upward (positive), the springs lift off the friction blocks and therefore no resistances can be built up in the sub-springs. Once the sub-springs are contacting the friction blocks again, the resistances in the sub-springs will again begin to build up and the spring-blocks act exactly as if the model is deflecting downward. The procedure of constructing a negative one-way curve from the individual sub-elements is shown in Fig 10(b).

The same multi-element model as shown in Fig 10(a) can be used to simulate the positive one-way resistance-deflection curve. In this case, the support model is to the right of (or above) the joint of beam-column; therefore, the sub-springs disconnect from the friction blocks and exhibit no resistances while the deflection is downward (negative).

Baushinger Effect of Nonlinear Resistance-Deflection Curve

If a specimen of mild steel is subjected to compression after a previous loading in tension, the applied compression stress, combined with the residual stress induced by the preceding tensile test, will produce yielding in the most unfavorably oriented crystals before the average compressive stress



Fig 10. Multi-element model of one-way resistance-deflection curve of the support.

reaches the value at which slip bands would be produced in a specimen in its original state. Thus, the tensile test cycle raises the elastic limit in tension, but lowers the elastic limit in compression. This phenomen is called the Baushinger effect (Refs 9 and 11).

From the multi-element models shown in Fig 9(a) and Fig 10(a), we can see that each individual sub-element is allowed to slide during the loadingunloading process. As a result of sliding or plastic deformation produced on loading, a system of residual resistances is introduced on unloading. Obviously, these residual resistances will influence the deformation produced by subsequent loads. Now let us study the so-called Baushinger effect based on the multi-element models shown in Fig 9(a) and Fig 10(a). For simplicity, consider a single spring support which has a nonlinear symmetric resistancedeflection curve with only three segments of resistances as shown in Fig 11(b). The nonlinear spring can be constructed by a multi-element model consisting of only two sub-elements each with an ideally elastic-plastic resistancedeflection curve as shown in Fig 11(c) and Fig 11(d). If a vertical load Q_1 is applied downward to the spring, it will produce resistances R_1 and R_2 in the sub-springs S_1 and S_2 , respectively. Thus,

$$R_1 = \frac{Q_1 S_1}{(S_1 + S_2)}$$
(3.1)

$$R_2 = \frac{Q_1 S_2}{(S_1 + S_2)}$$
(3.2)

By gradually increasing the load Q_1 we reach the condition at which the friction block in the sub-element 1 starts to slide while the sub-spring S_2 continues to deform elastically. This condition corresponds to point A in Fig 11b. If at point A the deflection is equal to a value w_1 , then the corresponding resistance R_A is found from the equation

$$R_{A} = (S_{1} + S_{2})w_{1}$$
(3.3)



Fig 11. Baushinger effect of symmetric nonlinear resistancedeflection curve of the support.

If we continue to increase the load, the sub-element 1 will slide and exhibit no resistance, and the additional resistance R_B will be taken by the subspring S_2 . The additional resistance R_B corresponding to the deflection w_2 will be

$$R_{\rm B} = S_2(w_2 - w_1) \tag{3.4}$$

The relation between R_B and w_2 is shown in Fig 11(b) by the inclined line AB. If we begin unloading the model after reaching point B on the diagram, both sub-springs S_1 and S_2 will behave elastically and the relation between the removed resistance and the upward displacement will be given by Eq 3.3. In the diagram we therefore obtain the line BC parallel to OA, and the total vertical displacement upward during unloading is

$$w_{3} = (R_{A} + R_{B})/(S_{1} + S_{2})$$
(3.5)

Substituting Eq 3.3 into Eq 3.5 yields

$$w_3 = w_1 + \frac{R_B}{S_1 + S_2}$$
(3.6)

From Eq 3.4 we obtain

$$w_2 = w_1 + \frac{R_B}{S_2}$$
 (3.7)

Comparing Eqs 3.6 and 3.7, we find that w_2 is greater than w_3 . We see that because of the sliding of sub-element 1, the model does not return to its initial state and the permanent set \overline{OC} is produced. The magnitude of the permanent set is found from the equation

$$\overline{\text{OC}} = w_2 - w_3 = R_B \left[\frac{s_1}{s_2(s_1 + s_2)} \right]$$
 (3.8)

Because of this permanent set there will be positive residual resistance R_2^S in the sub-spring S_2 , and this resistance will be balanced by the negative residual resistance R_1^S in the sub-spring S_1 . The magnitudes of R_1^S and R_2^S can be shown graphically in Fig 11(c) and Fig 11(d), respectively.

To show the Baushinger effect let us consider a second cycle of loading. At small values of load both sub-springs deform elastically, and in Fig 11(b) the second loading process will begin at point C and proceed along the straight line CB. When we reach point D the sub-spring S_1 will be relieved of the residual negative resistance R_1^S and during further loading will have positive resistance. At point B the positive resistance in the sub-spring S_1 will equal the friction force created by the friction block, and sliding of the sub-element 1 will begin. It is seen that the sliding resistance of the sub-element 1 is raised because of the residual resistances. The sliding resistance was at point A with resistance R_A in the first cycle and at point B with resistance $R_A + R_B$ in the second cycle. The process of unloading in the second cycle is perfectly elastic and follows the line BC.

Let us now reverse the direction of the force and apply an upward load Q_1 . The deformation will then proceed along the straight line CE, which is a continuation of line BC. At point E the total negative resistance is $R_A - R_B$, and the negative resistance in the sub-spring S_1 is equal to the friction force created by the friction block. As the load is increased, the deformation proceeds along the line EF. If the model is unloaded after reaching point F, it will return to the initial state represented by point 0. It is seen that by loading the model downward to point B, the positive sliding resistance in the sub-spring S_1 was increased from R_A to $R_A + R_B$. At the same time the negative sliding resistance in the sub-spring S_1 was diminished from R_A to $R_A - R_B$. This illustrates the Baushinger effect. The area of the parallelogram OABCEFO gives the amount of mechanical energy lost per cycle. The same concept discussed above can be expanded to a model consisting of any number of sub-elements.

For a negative one-way resistance-deflection curve as shown in Fig 12b, the Baushinger effect for this model can be illustrated by the loading paths indicated by the parallelogram OABCO and the area defined by OAB'DEFO. Notice that unless the model reaches the condition at which both sub-elements are sliding during the loading process (point D) there will be no permanent set after unloading from point B in the diagram. This is because the sub-spring



Fig 12. Baushinger effect of negative one-way resistancedeflection curve of the support.

 S_1 and the friction block are disconnected after reaching the point C in the diagram, and hence no residual resistance can be built up in the sub-spring S_1 . With further unloading the deformation proceeds along the line CO and the model will return to the initial state represented by point O. At this point, if we reverse the direction of load, the deformation will proceed along the new curve line OCBB'D. This is because the sub-spring S_1 lifts off the friction block before reaching the point C.

It is seen that because the sub-springs are able to lift off the friction blocks, no negative resistances can be built up in the sub-springs and therefore the loading paths can generate only on the negative side of deflection. The above concept also can be expanded to the model which has more than two sub-elements. The Baushinger effect of a positive one-way resistance-deflection curve is similar to a negative one-way resistance-deflection curve, except that the loading paths can generate only on the positive side of the deflection.

A general procedure for tracing the loading paths of either a symmetric nonlinear or a one-way resistance-deflection curve of any support which has more than two sub-elements will be discussed below.

Systematic Approach for Following Loading Paths

In Figs 9(c) and 10(c), we see that the multi-element models for simulating the nonlinearly-inelastic supports can be represented by a number of ideally elastic-plastic sub-elements. Although it is possible to retain the ideally elastic-plastic resistance-deflection curves of each individual subelement in the program, a more efficient and easier way is to retain the overall curve which is obtained by the summation of all ideally elastic-plastic resistance-deflection curves of the sub-elements as shown in Figs 9(b) and 10(b).

In Program DBC5, nonlinear supports can be described for any beam station. The nonlinear resistance-deflection curve of the support is described by a simple tabular input which generates a curve of straight line segments with a series of point numbers. After defining the resistance-deflection curve by a series of points, which are numbered in order, the resistance and deflection of each point must be retained sequentially in the variables of Q and W, respectively. The retained values of the deflection w must be increasing positively, while that of the resistance Q must be decreasing or equal for the consecutive points.

A typical symmetric nonlinear resistance-deflection curve is shown in Fig 13a.

In order to be able to systematically generate the loading paths of the curve during the loading-unloading process, the slope of each segment, the projected horizontal (deflection) value and the projected vertical (resistance) value of each segment must be retained sequentially in the variables of S , $\triangle w$, and $\triangle Q$, correspondingly. Notice that the retained values of the variable S must be in descending order.

Now let us study the procedure of constructing the new path of the original resistance-deflection curve shown in Fig 13(a), when the deformation of the support starts to reverse its direction at point A as shown in Fig 13(b). The systematic procedure consists of (1) drawing line AB parallel to segment 4-5 so that the projected value of line AB on the horizontal axis is equal to $\triangle w_1^T$ and the projected value on the vertical axis is equal to $\triangle q_1^T$, (2) drawing line BC parallel to segment 3-4 so that the projected value of line BC on the horizontal axis is equal to $\triangle w_2^T$ and the projected value on the vertical axis is equal to $\triangle q_2^T$, (3) connecting point C to point 7 on the original curve, and (4) renumbering the new path of the resistance-deflection curve as 1, 2, ... 7 as shown in Fig 13(b).

We see that the number of points on the new curve has been reduced to 7 from 8, which is the previous number of points on the original curve. If the deformation of the support again starts to reverse its direction at point A', shown in Fig 13(b), another new path of the resistance-deflection curve, numbered 1, 2, ... 6, will be generated by a procedure similar to that described above, except that three lines, A'B', B'C', and C'D', are drawn before the new curve connects to the previous curve.

For a typical negative one-way resistance-deflection curve, as shown in Fig 14(a), the logic of tracing two new curves can be shown graphically in Fig 14(b). The procedure is similar to the one used in the symmetric case. It is seen that loading paths are generated only by downward (negative) deflections and positive resistances. We also see that the second adjusted curve is generated when the deformation of the support, after going back to the initial state at the end of the first loop, continues to proceed along the line 5CBA2A' and starts to reverse its direction at point A'. The deformation at point A' is greater than the deformation at point A, where the first







Fig 13. Logic of tracing hysteresis loops for symmetric nonlinear resistance-deflection curve.



Fig 14. Logic of tracing hysteresis loops for a negative one-way resistance-deflection curve.

adjusted curve started. The logic of tracing the loading paths of a positive one-way resistance-deflection curve is similar to the one used in a negative one-way resistance-deflection curve, except that the loading paths are on the positive side of the deflection.

A detailed flow chart relating to the FORTRAN logic, which is used in the program, of tracing the loading paths of the nonlinearly-inelastic resistance-deflection curves of the support will be found in Appendix F.

CHAPTER 4. DESCRIPTION OF PROGRAM DBC5

<u>General</u>

Program DBC5 is written in FORTRAN language for Control Data Corporation (CDC) 6600 computers. With minor changes, the program would be compatible with IBM 7090 computers and with other systems. Four available FORTRAN subroutines are included in the program:

- (1) INTERP3 (Ref 5) for interpolating the input data,
- (2) TICTOC (Ref 5) for evaluating the computation time,
- (3) SPLOT9 for plotting the results by using the printer plotting method, and
- (4) ZOT1 for plotting the results by using the microfilm or ball-point paper plotting method.

The program uses in-core storage which requires 58,500 words. Although the use of auxiliary tapes will reduce the storage, it raises the computation time to a great extent due to tape operations; therefore, the in-core solution method has been chosen rather than using many auxiliary tapes. A summary flow chart of the program is presented in Fig 15. The definitions of symbols used in the program and a listing of the program with several general flow charts are included in Appendix E and Appendix F. Subroutines SPLOT9 and ZOT1 were developed particularly for the highway research projects at The University of Texas at Austin.

Procedure for Data Input

A guide for data input is included in Appendix D. The guide is designed so that additional copies may be furnished as separately bound extracts for routine use. A parallel study of the guide will help the reader to understand the following discussion.

Any number of problems may be stacked and run together. The sequence of problems is preceded by two cards which describe the run. The first card of each problem contains the problem number and a brief description of the problem



The program continues working problems until a blank problem number or an error is encountered; then the run is terminated.

<u>Tables of Input Data</u>

Table 1 is the data-control table. It consists of a single card which must be input for each problem. The number of cards in the remaining tables and the hold options of these tables are specified in this table. The hold options for the various tables are independent of each other, but care must be exercised to insure that the data in the various tables are compatible. For example, if a previous 40-increment problem had a distributed load from station 0 to station 40, then the user should not change the total number of increments in the new problem to some number less than 40 unless he erases the loads past the end of the new beam.

Table 2 contains

- (1) number of increments into which the beam-column is divided,
- (2) beam increment length,
- (3) number of increments into which the time axis is divided,
- (4) time increment length,
- (5) printing option for the computed results,
- (6) number of monitor beam stations which are requested for printing the computed results,
- (7) iteration switch to show whether or not the problem has nonlinear supports,
- (8) number of monitor beam stations which are requested for printing the iteration data,
- (9) option to choose the method of plotting the results, and
- (10) option of using lines or points for the plots.

If the number of time increments is 0, the problem is considered to be a static case and the complete results of the static solution will be printed at every beam station. In this case, two kinds of plots, which have either deflection or moments for the vertical axis and beam stations for the horizontal axis, may be requested by inputing "0" for the time station in Table 9.

For any station in the beam, a deflection, slope, or both, in either the initial or the permanent condition, may be specified in Table 3. If an initial condition of deflection, slope, or both is specified, it will be disregarded after the dynamic solutions are started. A specified deflection is equivalent to a single lateral support that is very stiff and is unable to deform freely.

A fixed-end support can be simulated in the computer by the specification of a permanent zero deflection and a permanent zero slope at the same beam station. The ability to specify a permanent deflection other than zero provides the user with a simple method of studying problems in which the supports settle. The ability to specify an initial deflection other than zero, on the other hand, provides the user with capability to study dynamic problems with initial displacements. Due to the method of simulation (see Fig 5), the specified slopes and deflections must conform to the following minimum spacing requirements:

- (1) A slope may not be specified closer than three increments from another specified slope.
- (2) A deflection may not be specified closer than two increments from a specified slope, except that both a slope and deflection may be specified at the same beam station.

Slope and deflection conditions may be specified at no more than 20 beam stations. Each specification requires a separate card. The cards may be stacked in any order within the table.

Fixed loads and restraints are described in Table 4. The input values are beam stiffness, fixed static loads, linear support springs, static axial thrusts, rotational restraints, and applied couples. Couples and rotational restraints appear only as concentrated effects, while axial thrusts are usually distributed. The remaining values may be either concentrated or distributed. The method used for the description of distributed data is illustrated in Appendix D. All of the input values of Table 4 are accumulated algebraically in storage. Therefore, there are no restrictions on the order of the cards, except that within a distribution sequence, the beam stations must be in ascending order. Axial thrusts must be described in the same manner as other values in this table because there is no provision in the program for automatically distributing the internal effects of an externally applied axial thrust. For example, an axial thrust applied to the ends of the beam-column must be specified as either an axial tension or an axial compression at each interior beam station. The number of cards accumulated in Table 4 cannot exceed 100.

Table 5 is used to describe the lumped mass densities, the lumped internal damping factors, and the lumped external damping factors at beam stations. Data in this table are input the same as in Table 4. The maximum allowable number of cards accumulated in this table is 100.

Time-dependent axial thrusts are input in Table 6 as values applying to designated bars. The values of time-dependent axial thrusts specified in this table may vary linearly both in the beam axis direction and in the time axis direction. There are no restrictions on the order of cards, except that beam stations which are stated as the starting station and the ending station in the same card must be in ascending order. A concentrated effect of timedependent axial thrust at a beam station is not permitted to be stated in this table since it would have no physical meaning.

The method used for the description of distributed data is also illustrated in Appendix D. The number of cards accumulated in this table cannot exceed 100.

Table 7 is used to describe time-dependent lateral loadings applied at the station points. The variation and the distribution of the input data are the same as in Table 6, except that the concentrated effect of the lateral load at a single station is allowed in this table. The number of cards accumulated in this table cannot exceed 100.

Nonlinear resistance-deflection curves of the supporting beam stations are input in Table 8. Every particular nonlinear support curve is described by three consecutive cards. The values input on the first card are resistancemultiplier, deflection-multiplier, number of points, symmetry option, deflectiontolerance, and resistance-tolerance. The values input on the second card are resistance values which are multiplied by the resistance-multiplier to obtain the final resistances of each point on the curve. The values input on the third card are deflection values which are multiplied by the deflectionmultiplier to obtain the final deflections of each point on the curve.

There are no restrictions on the order of support curves, except that within any distribution sequence, the beam stations must be in regular order. More than one curve may be placed at a beam station. The maximum number of support curves cannot exceed 20 (60 cards).

Four kinds of plots may be requested by inputing the name of the horizontal axis, the name of the vertical axis, the particular beam or time station, and the multiple plot switch in Table 9. As many as five of the same

kinds of plots, each of which is described by one separate card, may be superimposed by using the multiple plot switch. There are no restrictions on the order of cards, except that beam or time stations must be in regular order within a sequence of the same kind of plots. Superimposed plots must all be of the same kind. The maximum number of cards input in this table cannot exceed 10.

Three kinds of plotting methods, namely printer plots, microfilm plots, and ball-point paper plots, are built into the program. The switch which tells the program to choose one of the above three methods for plots is input in Table 2. Various plot options are illustrated in example problems described in Chapter 5.

Error Messages

Program DBC5 provides many checks for common types of data errors as well as for particular errors which occur due to either violations of FORTRAN logic or solution errors considered by the program.

Conditions which are considered to be of the type of data errors are

- (1) the allowable number of cards for an input table is exceeded;
- (2) a slope or deflection is specified too close to another slope or deflection;
- (3) data are input at stations beyond the end of the beam-column;
- (4) the station numbers in a distribution sequence are out of order;
- (5) the symmetry option is not equal to 1 when only one point is input on a support curve;
- (6) the final values of deflection and resistance, which are input for defining the points on a support curve, are improperly specified;
- (7) the allowable number of nonlinearly supported beam stations is exceeded;
- (8) the number of cards input in Table 8 is not zero when the problem is a linear case; and
- (9) repeated data are input in Table 9.

The conditions which are considered solution errors are

- calculated displacements exceed maximum allowable deflection specified by the user,
- (2) calculated displacements exceed limits of support curves during the iteration process, and
- (3) solution does not close within the specified number of iterations.

An error message which defines the error will be printed if any of the above conditions is encountered.

In addition to the specific error messages, a general purpose error message is provided for a number of unlikely errors. If the message 'UNDESIGNATED ERROR STOP" is printed, the user must investigate the program to determine what caused the error. Any error detected by the program will cause the entire run to be abandoned.

Description of Problem Results

The output of results is arranged so that the input quantities of Tables 1 through 9 are available for all problems and are printed with explanatory headings.

Table 10 presents the calculated results which are tabulated according to the following order:

- monitor deflections during the iteration process of the static solutions (none if the problem is a linear case or if the number of monitor beam stations for printing the iteration data is zero),
- (2) complete results of the static solutions,
- (3) monitor deflections during the iteration process of the dynamic solutions at time station 0 (see explanation in the parentheses of procedure 1,
- (4) complete results of the dynamic solutions at time station 0,
- (5) monitor deflections during the iteration process of the dynamic solutions at this time station (see explanation in the parentheses of procedure 1),
- (6) monitor or complete results of the dynamic solutions at this time station, and
- (7) repeat outputs of procedures 5 and 6 until the last time station specified in this problem is encountered.

The proper headings are printed at the top of the outputs of each procedure described above.

If printout of the calculated results are not requested, the message "PRINTING RESULTS IS NOT REQUESTED" will be printed under the headings of Table 10.

Plots of Results

Options are available which allow the user to obtain plots of the deflections or moments along either beam or time axis. Three kinds of plotting methods, namely printer plots, mircofilm plots, and ball-point plots, are available in the program. Both microfilm and ball-point plots fit in an 8-inch by 9-inch space. The optimum scales for the plots, both horizontal and vertical, are automatically selected by the program. Two fictitious points, one with the minimum horizontal and vertical values of all the points to be plotted on a single frame and the other with the maximum horizontal and vertical values, are plotted on the microfilm or ball-point plots for the purpose of choosing the proper vertical scale. These two fictitious points are not in the stored values of the points to be plotted on this frame and, therefore, should be disregarded. An example plot of the deflections along the time axis for inelastically supported mass under free vibration is shown in Fig 16. The plot in Fig 16 is made using the ball-point plotting method; the problem number and the identification of the plot variables are printed beneath the plot. The plot variables which identify either a microfilm plot or a ball-point plot include three sets of characters. The first set to the right of the problem number is one of the following four symbols: DVT or MVT (deflection or moment along time axis for a particular beam station), DVB or MVB (deflection or moment along beam axis for a particular time station). In the center is either BEAM STA = or TIME STA =. On the right are the station numbers for which the deflections or moments are plotted. The plot in Fig 16 is from Example Problem 1, described in Chapter 5. Another example plot, which is plotted using the microfilm plotting method, is shown in Fig 17. In this example, the moments along the time axis for the mid-point of a three-span beam on which an AASHO standard 2-D truck is moving with a speed of 60 mph is plotted on the microfilm, which then can be developed on an enlarged magnetic print. The plot is from Example Problem 3, described in Chapter 5. Many more ball-point plots which have more than one curve plotted on one frame can be found in Example Problem 5, described in Chapter 5.

A typical example plot which is plotted by using the printer plotting method is shown in Fig 18. The printer plotting method plots the horizontal axis vertically on the sheet. The plot is from Example Problem 6, described in Chapter 5. Different symbols (as many as five) are used on each frame of



Fig 16. A typical ball-point paper plot.



Fig 17. A typical microfilm plot.



PROF (CONTD) 6 THREE-SPAN BEAM WITH ONE-WAY AND SYMMETRIC NONLINEAR SUP LOADED BY T.P ***** PLOT OF DEFLECTION VS TIME FOR BEAM STATIONS OF *****

PROGRAM DBCS - MASTER - JACK CHAN - MATEOCK - DECKT-REVISION DATE = 26 JUN 71 EXAMPLE PROBLEMS FOR PROGRAM DBCS - RY JACK CHAN JUNE 1971 DYNAMIC HEAM-COLUMN PROGRAM (5-1-5 IMPLICIT OPERATOR) the printer plots for identifying different curves superimposed on this frame. The identifications of the plot variables (such as description of program and run, problem number and problem description, description of the type of the plots, and beam stations or time stations) are printed on the top of each frame of plots. The numerical values printed vertically to the left of the plots are those of the first plot on this frame. The scale of the vertical (deflection or moment) axis is not plotted on the plots, and the horizontal (beam or time station) axis is scaled equally for each point on the plots and plotted by the symbols I. A series of numbers corresponding to the beam or time stations is printed vertically on the left end of the plots for the purpose of indexing. The printer plots fit on 8-1/2-inch by 11-inch standard letter-size paper.

CHAPTER 5. EXAMPLE PROBLEMS

In this chapter seven example problems are solved by Program DBC5. Problems 1 and 2 each have two different cases and Problem 4 has three different cases. Problems 1 and 1A demonstrate the validity of the program in predicting the formation of hysteresis loops on the nonlinear resistancedeflection curve of the spring which supports a freely vibrating mass.

Problems 2-1 and 2-2 present a comparison between the numerical solutions solved by DBC5 and the theoretical solutions of vibrating beams illustrated in Ref 12, pages 375 and 378. Problem 3 demonstrates the capability to use the computed dynamic tire forces of a moving truck from another available computer program as input data in Program DBC5 to predict the dynamic responses of a highway structure. Problem 4 compares the experimental data with the computed results of a simply supported steel rod under axial pulses. Problem 5 demonstrates the capability to predict the response of off-shore piles to wave forces. The solution of a three-span beam (30 beam increments) which is simply supported at the ends and loaded by a transient pulse at station 17 is illustrated in Problem 6. Finally, Problem 7 demonstrates the capability to predict the response of a partially embedded pile to earthquake-induced forces. A listing of input data is included in Appendix H.

Example Problem 1. Inelastically Supported Mass Under Free Vibration

Figure 19 shows a mass of density 1.0 lb-sec²/in. supported by a nonlinearly inelastic spring; the mass is loaded by a static load of 100 pounds for the initial deflection and then released by applying a constant dynamic load of 100 pounds. The weight of the mass is neglected. The nonlinear characteristics of the resistance-deflection curve of the spring are also shown in Fig 19. Time increment length is equal to 6.283×10^{-3} second, which is approximately 1/100 of the natural period. The vibration of the mass is solved at 800 time stations. This first problem is intended to provide confidence in the proposed multi-element models to predict the loading paths of the nonlinear-inelastic



Fig 19. Example Problems 1 and 1A. Inelastically supported mass under free vibration.
support. From the computer output, a plot of deflections versus support reactions can be obtained, as shown in Fig 20.

It is seen that due to partial yielding of the support at the beginning, a hysteresis loop is formed in each cycle of vibration and mechanical energy is lost in each loop. The mechanical energy lost from point A to point J in Fig 20 is equal to the area defined by JKEFGHIJ, which is equivalent to the difference between the summation of the areas defined by ALDCBA and DEFPD and the summation of the areas defined by FPNHGF and JMNIJ. Therefore, the deflections are damping out with time until the response finally becomes a free elastic vibration within the range of greatest stiffness. This can be proved from the tabular output of the program when large time increment lengths (6.283×10^{-2} second) are used. The computer plot of deflections versus time is shown in Fig 21.

Example Problem 1A. Free Vibration of Mass by Specifying Initial Displacements

An additional run on the preceding problem was made by specifying an initial displacement of 3 inches instead of applying a static load for the initial deflection. From the plot of support reactions versus deflections (Fig 22), it is seen that a permanent set is obtained because of initial yielding of the support. The mass is finally freely vibrated at a position about a mean permanent upward deflection of approximately 1 inch. This can be shown in the computer plot of deflections versus time in Fig 23.

Example Problem 2-1. Lateral Vibration of a Simply Supported Beam with Axial Compression Force

Figure 24 shows a simply supported beam which is compressed with an axial force of 3.70×10^5 pounds and placed under free vibration by specifying an initially deflected sine curve for which the maximum deflection at the center is 3.952 inches. The beam is 120 inches long and has a uniform flexural stiffness of 1.08×10^9 lb-in². The beam is divided into 10 elements, each is 12 inches long. The mass density of the beam is 1.085×10^{-1} lb-sec²/in./sta. The number of time increments is 200, and the time increment length is 3.752×10^{-4} seconds. The theoretical natural period of the vibration is 3.752×10^{-2} seconds (Ref 9, p 375).







Fig 21. Example Problem 1. Response of the nonlinearly supported mass.



Fig 22. Hysteresis loops of computed resistance-deflection curves of Example Problem 1A.

ī,



Fig 23. Example Problem 1A. Response of the nonlinearly supported mass.



EI 1.08×10^9 lb-in² Mass density 1.085×10^{-1} lb-sec²/in/sta T = -3.70 $\times 10^5$ lb Number of beam increments = 10 Beam increment length = 12 inches Number of time increments = 200 Time increment length = 3.752 $\times 10^{-4}$ seconds

Initially Deflection Sine Curve



Fig 24. Example Problem 2-1. Lateral vibration of a simple supported beam with axial compression force.

The theoretical responses (Eq d, p 375, Ref 12) and the computed responses based on a time increment length of 3.752×10^{-3} seconds (1/10 of the natural period) are plotted for the center of the beam on the computer plot of deflections versus time for beam station 5, as shown in Fig 25. The responses of the small time increment length (1/100 of the natural period) are almost perfectly matched with the theoretical responses. The responses of the large time increment length (1/10 of the natural period) show more variation when compared to the theoretical responses in the first period, and after the first period are shifted with a phase angle. The more increments, however, the beam is divided into, the better the results. The computed moments, based on both small and large time increment lengths along the time axis for the center of the beam, are shown in Fig 26.

Example Problem 2-2. Lateral Vibration of a Beam on Elastic Foundation

Figure 27 shows the beam in Problem 2-1 resting on an elastic foundation which has a spring constant of 1.2×10^4 lb/in./sta. The beam is under free vibration with an initially deflected shape of sine curve, in which the maximum deflection at the center is 6.672 inches. The number of time increments is 200, and the time increment length is 1.540×10^{-4} seconds. The theoretical natural period is 1.540×10^{-2} seconds (Ref 12, p 378).

The comparisons between the computed responses, based on both small and large time increment lengths, and the theoretical responses (Eq c, p 378, Ref 12) for the center of the beam are shown in Fig 28. The difference between the computed responses of the small time increment length (1/100 of the natural period) and the theoretical responses is slight. The computed moments, based on both small and large time increment lengths, along the time axis are shown in Fig 29.

Example Problem 3. Three-Span Beam Loaded with an AASHO Standard 2-D Truck Moving at a Speed of 60 MPH

This problem demonstrates the capability to input the dynamic tire forces interpreted from the computer output of the program described in Ref 1 into Table 7 of Program DBC5 to predict the responses of a three-span beam; the beam is loaded with an AASHO standard 2-D truck which is moving at a uniform speed of 60 mph, as shown in Fig 30.



Fig 25. Example Problem 2-1. Deflection vs time the center of beam.



Fig 26. Example Problem 2-1. Moment vs time for the center of beam.



EI = 1.08×10^9 lb-in² Mass density 1.085×10^{-1} lb-sec²/in/sta Elastic spring constant 1.2×10^4 lb/in/sta Number of beam increments = 10 Beam increment length = 12 inches Number of time increments = 200 Time increment length = 1.540×10^{-4} seconds

Initially Deflected Sine Curve



d ₁		d9		-6.672	Х	sin	180	=	-2.061	inches
d ₂		d ₈	:-	-6.672	х	sin	36 ⁰	=	- 3.922	inches
d ₃	=	d ₇	72	-6.672	Х	sin	54 ⁰	=	- 5.398	inches
d ₄		d ₆		-6.672	Х	sin	72 ⁰	=	-6.345	inches
d _5	=	- 6	. 67	$72 \times sin$	n <u>9</u>	90 ⁰	- 6	. 67	72 inche	e S

Fig 27. Example Problem 2-2. Lateral vibration of a beam on elastic foundation.



Fig 28. Example Problem 2-2. Deflection vs time for the center of beam.



Fig 29. Example Problem 2-2. Moment vs time for the center of beam.



Fig 30. Example Problem 3. Three-span beam loaded by an AASHO standard 2-D truck moving at a speed of 60 mph.

The beam is simply supported at the ends and has a length of 200 feet. The beam is divided into 40 increments, each of which is 60 inches long. The two linear springs are located at beam stations 10 and 30 and have a value of 2×10^{2} lb/in. The first and third spans of the beam have a uniform flexural stiffness of 4.5 \times 10¹¹ lb-in.² and a uniform mass density of 0.5964 lb-sec²/ in./sta. The second span of the beam has a uniform flexural stiffness of 9.0 \times 10¹¹ lb-in.² and a uniform mass density of 0.2982 lb-sec²/in./sta. The computer model used in Ref 1 for simulating the AASHO standard 2-D truck is shown in Fig 30. The input data for the springs and dashpots which simulate the tires of the truck and the static weights of the wheels are also shown in Fig 30. The computed dynamic forces of the four tires are averaged to give the two axle forces which then are input into Table 7 of Program DBC5. The distribution of these two axle forces on both the beam and time axes is shown in Fig 30. The time axis is divided into 80 increments, each of which has a length of 5.666 $\times 10^{-2}$ seconds so that the front axle will move one beam increment length per time station.

The computed deflections and moments along the time axis for the center of the beam are plotted by the program as shown in Figs 31 and 32, respectively. The plots are produced on microfilm which then can be enlarged.

A comparison of the dynamic response to the static response of the deflections along the beam axis is shown in Fig 33. This static response of the plot in Fig 33 is computed by specifying the static load of the front axle at beam station 21 and the static load of the rear axle at the center of the bar, between beam stations 18 and 10; the dynamic response of the plot, therefore, is the computed deflection along the beam axis for time station 21. The dynamic effect on the responses of the beam produced by a moving 2-D truck is fairly small compared to the static responses produced by the same truck. This is because the dynamic forces generated by the 2-D truck, which is moving on a smooth pavement, are smoothly varied with time, and therefore there are small dynamic effects. Any other AASHO standard truck or semi-trailer can be used in the program described in Ref 1 to generate the corresponding dynamic tire forces, which then can be input with a limited amount of interpretation into Table 7 of Program DBC5 to predict the dynamic responses of highway structures that can be simulated by beam-columns.



Fig 31. Example Problem 3. Deflection vs time for the center of beam.



Fig 32. Example Problem 3. Moment vs time for the center of beam.



Fig 33. Example Problem 3. Comparison of dynamic to static response (deflection along the beam axis).

Example Problem 4-1. Simply Supported Steel Rod - Axial Pulse Base Time = 0.10 Second

In the following three problems, a simply supported steel rod is loaded by three axial pulses with different peak loads and base time, as shown in Fig 34. The problem is a real experiment which was done by Matlock and Vora (Ref 7). The steel rod is 153.75 inches long and 11/16 of an inch in diameter. The steel rod is divided into 41 increments, each of which has a length of 3.75 inches. The mass density of the rod is 1.023×10^{-3} lb-sec²/in./sta. The static weight of the rod is 3.95×10^{-1} lb/sta. An internal damping factor of 20 lb-in.²-sec is input at each station.

The first axial pulse input for this problem has a peak value of 175 pounds and a base time of 0.10 seconds. The axial pulse was recorded on the oscilloscope in the experiment by Matlock and Vora described in Ref 7. The true axial pulse is described at 20 points by scaling from the recorded output of strain gages, which then can be input in Table 6 of Program DBC5. The time increment length is set equal to 5×10^{-3} seconds and 40 time stations are solved for this problem.

Figure 35 shows the comparison of the computed bending moments along the time axis to the measured bending moments in the experiment for the center station of the steel rod. It is seen that good agreement exists between the predicted results and the measured results before 0.9 second, while the measured bending moments are smaller than the predicted bending moments after 0.9 second. This deviation is mainly due to the assumption that the beam properties are fully elastic in the dynamic model of Program DBC5. Other factors, such as the errors obtained by scaling from the recorded output, the difficulty in establishing the ideal hinged supports, and failure to consider the axial deformation of the rod, also affect the accuracy of the predicted results.

An additional curve of the bending moment along the time axis for the center of the rod, which is computed without inputting the internal damping factor, is also plotted on Fig 35. Little is changed by ignoring the effect of the internal damping factor, since the change of the curvature at this point is very smooth along the time axis. The computed deflections along the time axis for the center station of the steel rod are plotted by the program, as shown in Fig 36.



Fig 34. Example Problems 4-1, 4-2, and 4-3. Simply supported steel rod loaded by axial pulses.



Fig 35. Example Problem 4-1. Comparison of moments vs time for the center of beam.



Fig 36. Example Problem 4-1. Deflection vs time for the center of beam.

Example Problem 4-2. Simply Supported Steel Rod - Axial Pulse Base Time = 0.026 Second

In this problem, the steel rod in the preceding problem is loaded by another pulse with a shorter base time but larger peak value. The base time of the pulse is 0.026 second and the peak value of the pulse is increased to approximately 635 pounds. The axial pulse is described at 26 points by scaling from the recorded output of the experiment.

The computed bending moments along the time axis, with and without internal damping effect, and the measured bending moments are plotted for comparison, as shown in Fig 37. Results similar to those for the preceding problem are observed, but the effect of the internal damping factor is a little greater than in the preceding problem. Figure 38 shows the computed deflections along the time axis for the center station of the steel rod.

Example Problem 4-3. Simply Supported Steel Rod - Axial Pulse Base Time = 0.006 Second

In this problem the loaded axial pulse has a base time of 0.006 seconds and a peak value of 850 pounds. Again the computed bending moments along the time axis, with and without internal damping factor, and the measured bending moments are plotted as shown in Fig 39. Because of the higher frequency of vibrations, the mode shapes of the beam vary rapidly with time and cause large variation of curvature; therefore, the effect of the internal damping factor is much greater than in the preceding two cases. The estimated value of 20 lb-in.² sec/sta input for the internal damping factor is higher than the real value. Figure 40 shows the computed deflections along the time axis for the center station of the steel rod.

Example Problem 5. Partially Embedded Steel Pipe Pile Loaded by Wave Forces

This problem shows how DBC5 can be used to solve the dynamic response of laterally loaded piles to wave forces. Figure 41 presents a steel pipe of 30-inch outside diameter and 28-inch inside diameter which is partially embedded in the soil to a depth of 80 feet and extends slightly above the surface of the water which is 40 feet deep. The steel pipe is divided into 31 increments, each of which has a length of 48 inches. The flexural stiffness of the pipe is 2.877×10 inch lb-in.². The mass density of the pipe is 3.21



Fig 37. Example Problem 4-2. Comparison of moments vs time for the center of beam.



Fig 38. Example Problem 4-2. Deflection vs time for the center of beam.



Fig 39. Example Problem 4-3. Comparison of moments vs time for the center of beam.



Fig 40. Example Problem 4-3. Deflection vs time for the center of beam.



Fig 41. Example Problem 5. Partially embedded steel pipe loaded by idealized wave forces.

lb-sec²/in. from station 1 to station 31. A large mass, which is arbitrarily set at 32.1 lb-sec²/in., is lumped at station 0 to represent the weights of the structure, equipment, etc. The static weights of the pipe and the large mass lumped at station 0 are considered to be the axial compression forces, which are distributed from 1.220×10^4 pound at station 1 to 4.924×10^4 pound at station 31. An estimated value of 1.918×10^6 lb-in.²-sec/sta is input for the internal damping factor.

The pipe is assumed to be simply supported at station 31 and assumed to have a linear spring support of 3×10^7 lb/in. as well as a rotational restraint of 2.1×10^{10} lb-in./rad at station 1. From station 11 to station 31, the pipe is laterally supported by soil. The soil from station 11 to station 16 is assumed to be a soft clay which has a shear strength that varies from 200 lb/ft² at the top to 333.4 lb/ft² at the bottom. The soil below station 16 is assumed to be a stiff clay which has a shear strength of approximately 475 lb/ft². Generation of the resistance-deflection curves for the soil supports at stations 11 and 13 is based on a technique presented by Matlock (Ref 6). The resistances and deflections for both support curves are described at 20 points as shown in Fig 42. The remaining support curves between stations 11 and 16 are linearly distributed.

The nonlinear characteristics of the resistance-deflection curves for the soil supports from station 16 to station 31 are obtained from the data for the experiment done by Matlock and Vora in Ref 8. The nonlinear resistancedeflection curves are also described at 20 points, as shown in Fig 43.

Current drag forces are assumed to vary from -2×10^4 pound at station 1 to -1×10^4 pound at station 11. Idealized wave forces are shown at the bottom of Fig 41. The wave forces are assumed to vary linearly from station 1 to station 11. The time increment length is equal to 1.0×10^{-2} seconds. Four hundred time stations are solved for this problem.

The computed deflections and moments versus time for stations 10 and 21 are plotted by the program, as shown in Figs 44 and 45, respectively. Figure 46 shows the computed deflections along the beam axis for time stations 60 and 160. Figure 47 shows the computed deflection along the beam axis for time stations 230 and 280. Figure 48 shows the computed moments along the beam axis for time stations 60 and 320. All plots in this problem are automatically plotted using the ball-point plotting method.



Fig 42. Symmetric resistance-deflection curves of soft clay at beam stations 11 and 13.







Fig 44. Example Problem 5. Deflection vs time for beam stations 10 and 21.



Fig 45. Example Problem 5. Moment vs time for beam stations 10 and 21.



Fig 46. Example Problem 5. Deflection along the beam axis for time stations 60 and 160.



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Fig 48. Example Problem 5. Moment along the beam axis for time stations 60 and 320.

Example Problem 6. Three-Span Beam with One-Way and Symmetric Nonlinear Supports Loaded with a Transient Pulse

This problem is intended to demonstrate that the option of switching from a spring-load (tangent modulus method) iteration to a load iteration technique which is built into Program DBC5 is useful when the tangent modulus method fails because of lift-off of the one-way support or yielding of the nonlinear supports. Figure 49 shows a three-span beam (30 beam stations) which is simply supported at the ends, one-way supported at station 8 and nonlinearly supported at stations 16 and 17 and is loaded with a transient pulse at station 17. The beam has a uniform flexural stiffness of 2.4 \times 10¹⁰ lb-in.² and a uniform mass density of 5.184 $\times 10^{-1}$ lb-sec²/in./sta. A rotational restraint of 3.0 $_{\rm X}$ 10^{10} is applied at beam station 0 and a constant axial compression force of 10⁵ pounds is assumed. The negative one-way resistancedeflection curve of the support at beam station 8 is shown in Fig 50. The symmetric resistance-deflection curve of the support at beam station 17 is also shown in Fig 50. The nonlinear characteristic of the support at beam station 16 is linearly distributed between beam station 15, which has no resistance to deflection, and beam station 17. The applied transient pulse at beam station 17 is shown at the bottom of Fig 49. The time increment length is equal to 4 \times 10⁻² seconds and 40 time stations are solved for this problem.

The computed results and the monitor deflections at beam stations 8, 15, 16, and 17 during the iteration process are printed on the computer output. It is seen that the tangent modulus method failed at time stations 5, 9, 13, 16, and 24 due to abrupt changes of time-dependent loads or lift-off of the support at beam station 8 at these time stations; therefore the program automatically switches to the load iteration method and succeeds in obtaining the closed solutions. If the time increment length is reduced to 4×10^{-3} seconds and the same problem is rerun for 400 time stations, the tangent modulus method works nicely at each time station 8. This is because smoother changes of time-dependent loads are obtained due to the small time increment length and, therefore, the tangent modulus method only fails at some of the critical time stations at which lift-off of the support occurred. The computed deflections and moments versus time for beam stations 8 and 17 are plotted using the printer plotting method, as shown in Figs 51 and 52,


EI = 2.4 $\times 10^{10}$ lb-in² $\infty = 0.5184$ lb-sec²/in/sta T = 10⁵lb R = 3 $\times 10^{10}$ lb-in/rad S^B and S^N see Fig 50 Beam increments = 30 Beam increment length = 48 inches Time increments = 40 Time increment length = 4 $\times 10^{-2}$ seconds



Fig 49. Example Problem 6. Three-span beam loaded by transient pulse.



Fig 50. Resistance-deflection curves of the supports at beam stations 8 and 17.

Fig 51. Example Problem 6. Deflection vs time for beam stations 8 and 17.

0	-6.601F-17			+	
1	-6,665F-03			+ + + T	
2	-1.613F-02			+ 45 T	
Э	-2,991F-02		•	6- T	
4	-4.576F-02		÷	~ L 84 T	
5	-3.4/4F-02	•			
Ä	-1.719F-02	•	•		
7	-2.6865-04		•	*	
α	-3 0315-03			* 9	
-) - 0				9 [0 -	•
10	-1 790r 00			-1	•
19	-2 1365-02			+ + <u>+</u>	
11	-2.000F-007		*	• I	
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13	= 3, 996F = 17	*		J 44	
14	-3.4228-02		•	* र	
15	-1.095F-02			+ +Ţ	
16	7.612E-03			*	+
17	=3.977F=03			*T	+
18	-3.483F-12		•	* T	
19	-3.544F-02	♦		ë T	
20	-2,729F-02	*		4 I	
21	-1.9455-02		+	* 1	
55	-5,084F-03			₩ ¶	•
23	8,561F=03			4	
24	3,095E-03			4	•
25	-2,837F-02			+ +Ţ	
26	-3.069F-02		*	• T	
27	1.602E-02		•		
28	4.137F-02			+ Ta	•
29	-1,8826-02			+ T	•
30	-8.241F-02			+ T	•
31	-1,388F-05			47.	
32	6,972F-J2			+ T	\$
33	4.049F-02		*	T	•
34	-6.450F-02			+ * T	
35	-8.340F-02			- 1 • T	•
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40	-1.584F-02		*	· • •	
* 0	• • • • • • • • • • • • • • • • •			• भ	

PROH (CONTD) 6 THREE-SPAN BEAM WITH ONE-WAY AND SYMMETRIC NONLINEAR SUP LOADED BY T.P ###### PLOT OF DEFLECTION VS TIME FOR REAM STATIONS OF. ######

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1 CURVF (*) = 82 CURVF (*) = 17

PT-NO #VALUE#

PROGRAM DBCS - MASTER - JACK CHAN - MATLOCK - DECK)-REVISION DATE = 26 JUN 71 EXAMPLE PROBLEMS FOR PROGRAM DBCS BY JACK CHAN JUNE 1971 DYDAMIC BEAM-COLUMN PROGRAM (5-1-5 IMPLICIT OPERATOR)

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PROGRAM DECS - MASTER - JACK CHAN - MATLOCK - DECKI-REVISION DATE = 26 JUN 71 EXAMPLE PROBLEMS FOR PROGRAM DEC5 BY JACK CHAN JUNE 1971 DYNAMIC BEAM-COLUMN PROGRAM (5-1-5 IMPLICIT OPERATOR) PROA (CONTD) 6 THREE-SPAN BEAM WITH ONE-WAY AND SYMMETRIC NONLINEAR SUP LOADED BY T.P **** PLOT OF BEND. MOM. VS TIME FOR REAM STATIONS OF. ***** 1 CURVE (#) = 8 2 CURVE (+) = 17 PT.NO **#VALUE#** -1 -3,358E-10 "3.891E"10 0 -3.578E+04 1 2 -9,538E+04 Ī -1.812E+05 З -2.722E+05 4 5 -2,115E+05 Ť -1.409E+05 6 Ţ 7 -4.657F+04 8 1.139E+04 9 -5,331E+03 -6.771F+04 10 Ť -1.828E+05 11 T 12 -2.875E+U5 T 13 -2.433E+05 Ţ 14 -1.828F+05 Ţ 15 -5.922E+04 3,178F+04 16 ĩ 17 1.222F+04 T 18 -9.157F+04 -2.589E+05 19 -2,965E+05 20 21 -1.227E+05 22 7.166E+04 23 1.018E+05 -1.266E+04 -8.175E+04 24 25 -3.985E+04 26 27 -1,949F+04 28 -4.34UE+04 29 -3.052F+04 2.709E+04 30 31 1.262E+04 -5,033E+04 32 33 -7.626E+04 34 -5.714E+04 -1.049E+04 35 2.8662+04 36 Ť 37 4.163F+04 T 38 -4.764r+04 ţ -9.793E+04 39 Ţ -7.125E+04 40

Fig 52. Example Problem 6. Moment vs time for beam stations 8 and 17.

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respectively. Figure 53 shows the computed deflections along the beam axis for time stations 4, 8, and 20. Figure 54 shows the computed moments along the beam axis for time stations 12, 16, and 30.

Example Problem 7. Partially Embedded Steel Pipe Pile Excited by Sinusoidal Earthquake-Induced Forces

The capability of Program DBC5 to analyze the response of a partially embedded steel pipe pile which is excited by sinusoidal earthquake-induced forces is illustrated with problem 7. Figure 55(a) shows a 12.75-inch outside diameter \times 0.5-inch wall steel pipe pile embedded in the soil to a depth of 40 feet and extending 10 feet above the ground surface. The pile has a flexure stiffness of 10.9 \times 10⁹ lb-in.² and is divided into 50 beam increments. Each beam increment has a length of 12 inches. An axial compression force of 10,000 pounds is assumed in the pile. From beam station 0 to beam station 40, the lumped mass density of soil and pile is assumed to be 5.184 lb-sec²/in./sta. From beam station 40 to beam station 50, the lumped mass density of soil and lateral load is assumed to be 2.592 lb-sec²/in./sta. The negative and positive one-way resistance-deflection curves, which describe the lateral soil supports from beam station 0 to beam station 40, are shown in Fig 55(b) and (c), respectively. The time increment length is 5 \times 10⁻³ seconds and 160 time stations are solved for this problem.

The pile is excited by a sinusoidal earthquake-induced force, as shown in Fig 55(d). The sinusoidal earthquake induced force has a base time of 0.1 second (equivalent to 20 time stations) and is induced from an assumed sinusoidal acceleration with a maximum value of 1/10 G (G = 386 in./sec²) and a base time of 0.1 second. Based on the discrete-element beam-column model, the applied transverse load Q^{T} is equal to the product of deflection W and spring force S^{N} . By double integration of the equation of sinusoidal acceleration and evaluation of the constants of integration, the relation between the maximum deflection (W_{max}) and the maximum acceleration (\ddot{W}_{max}) is found to be

$$W_{\text{max}} = W_{\text{max}} t_c^2 / \pi^2$$

where t is the base time and π is equal to 3.1416. If the nonlinear spring



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Fig 53. Example Problem 6. Deflection along the beam axis for time stations 4, 8, and 20.



Fig 54. Example Problem 6. Moment along the beam axis for time stations 12, 16, and 30.



Fig 55. Example Problem 7. Partially embedded pile excited by sinusoidal earthquake-induced forces.

force S^N can be assumed to be constant, then the maximum sinusoidal earthquake-induced force can be obtained by the equation

$$Q_{\max}^{T} = S^{N} t_{c}^{2} W_{\max} / \pi^{2}$$

The earthquake-induced forces are applied at beam stations 0 to 40 to approximately describe the ground movements.

The computed resistance-deflection curves of the negative and positive one-way supports at beam station 20 are shown in Fig 56. The first loading path is seen to form a loop which starts from point A and follows the path of ABCDAEFGA. The mechanical energy lost in the first loop is equal to the area defined by the loading path. The second loading path follows the path of ADHDAGIGA; no mechanical energy is lost in this path because the energy is absorbed by or transformed to adjacent supports. Between points D and G in the second loading path, the supports at beam station 20 exhibit no resistance and hence a hole is produced around the pile at this station. The third loading path is similar to the second loading path except that in the third the maximum positive deflection increases from point H to point J, and the maximum negative deflection decrease from point I to point K. The succeeding paths are similar but have progressively smaller maximum positive and maximum negative deflections until finally no more energy is absorbed by or transformed to adjacent supports and the entire pile exists in a state of free vibration since no damping factor has been included. Figure 57 shows the plot of deflection versus time for beam station 20. Here the response is tapering down to the two limits at which the entire pile is in a state of free vibration.

A similar computed resistance-deflection curve can be constructed for any other support, with a similar result expected unless the support (for instance, at beam station 40) has not yet reached the yielding condition; then the computed resistance-deflection curve will be a straight line. Figure 58 shows the computer plot of deflection versus time for beam stations 10 and 30. Figure 59 shows the computer plot of deflection versus time for beam stations 40 and 50. Figure 60 shows the computer plot of deflection along the beam axis at time stations 10 and 20. Figure 61 shows the computer plot of deflection along the beam axis at time stations 50 and 150. Figure 62 shows the computer plot of moment along the beam axis at time stations 30 and 100.



Fig 56. Computed resistance-deflection curves of the one-way supports at beam station 20 of Example Problem 7.



Fig 57. Example Problem 7. Deflection vs time for beam station 20.



Fig 58. Example Problem 7. Deflection vs time for beam stations 10 and 30.





Fig 60. Example Problem 7. Deflection along the beam axis for time stations 10 and 20.









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CHAPTER 6. SUMMARY AND CONCLUSIONS

In this study, the dynamic solution of the discrete-element beam-columns of Ref 10 (DBC1) and the static solution of the discrete-element beam-columns of Ref 5 (BMCOL 43) have been modified and extended to cover nonlinear supports and loads with time-dependent axial thrusts; the program has the added capability of producing plots of the deflections or moments along either beam or time axis of any requested monitor stations.

The path-dependent history of loading of the nonlinearly-inelastic resistance-deflection curve of the support is considered in the study. Two multielement models, which are used to simulate the nonlinear characteristics of symmetric and one-way resistance-deflection curves, have been introduced. An internal damping factor, which is related to the first time derivative of the curvature of the beam, has been considered in addition to the external viscous damping factor which is normally encountered in a dynamic problem. A computer program which has been presented, DBC5, allows the user to obtain a variety of results and plots. All of the uses and capabilities of DBC1 and BMCOL 43, except construction of the envelopes of maximums provided in BMCOL 43, have been incorporated in the DBC5 program.

This method is based on an implicit difference formula of the Crank-Nicolson type. Problems in which only linear spring supports are specified are not subject to the instability of dynamic solutions. Problems in which nonlinearly-inelastic supports are specified, however, require cautious selection of a time increment length, to pick one reasonably but not extremely small (for instance, less than 1/10 of the fundamental period), since the basic assumption of the Crank-Nicolson implicit formula is that the time dependent forcing function and the time variant nonlinear springs are smoothly varied with time. For problems with nonlinear supports, the iteration process compares the successively computed deflections until a specified tolerance is satisfied. The option of switching from the spring-load iteration technique (tangent modulus method) to the load iteration technique for problems with nonlinear

supports is useful when the support yields or disconnects from the beam-column since the spring-load iteration method fails in these cases.

This report is intended to encourage use of DBC5 as a tool in computer aided design. Highway structural and foundation designers can use it to solve many problems, static or dynamic, which they encounter and which presently are solved by rough approximations or by tedious hand calculations. A very important use would be for a study of the dynamic response of actual truck loaded beam-columns, as illustrated in Example Problem 3 described in Chapter 5. A highly sophisticated computer program (Ref 1) is available to compute the time variant dynamic forces of the tires of a truck moving at a uniform speed on highway pavements. With limited interpretation, the computed dynamic tire forces could be input in the table provided for the time dependent lateral loads in Program DBC5.

Further application to design problems that are difficult to solve by conventional means is possible. The possible variations include analyses of a bridge structure as foundation supports settle, railroad loadings on continuous spans supported on soil foundations, the effect of axial loads induced from tractive forces or temperature changes, approach slabs connected integrally with the bridge structure and supported on soil foundations, offshore piles loaded with wave forces and partially embedded in the soil, and transverse response to earthquake-induced forces.

Future extensions to the model and the program might include, for highway pavement analysis, the coupling of a vehicle model and pavement roughness characteristics to the beam-column model for generation of dynamic loads (similar to the model described in Ref 1) and the inelastic analysis of beam properties.

The nonlinear solution capabilities should be extended to the beam bending stiffness variable. Nonlinear moment-curvature relations could be incorporated in the iterative procedure, thereby permitting analysis of the beam material for stresses in the nonlinear range. The hysteresis effect of the nonlinear moment-curvature relations should also be considered.

Studies of the nonlinear closure procedure should be continued. Methods for accelerating closure in the load iteration technique should be developed. In Program DBC5, the iteration process is performed at each time step. It is possible, however, to iterate only at the critical time stations; at the rest of the time stations, the support stiffness can be assumed to be unchanged since a small time increment length must be used in these problems. The existing discrete-element model requires the user to know and specify the distribution of axial thrust throughout the beam. A valuable extension of this work would be modification of the model to include axial deformations and development of the force-deformation equations for axial thrust.

Finally, experimental data are required for further evaluation of the method, especially in predicting the path-dependent behavior of the supports. Research of this nature will furnish additional data on the nonlinear behavior of the supports, which could be applied to modification of the existing multielement models for nonlinear supports so that buckling, fracture, softening, and relaxation (creep) of the support could also be considered in the program. This page replaces an intentionally blank page in the original. -- CTR Library Digitization Team

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APPENDIX A

DERIVATION OF THE DYNAMIC IMPLICIT OPERATOR BASED ON THE CRANK-NICOLSON IMPLICIT FORMULA

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APPENDIX A. DERIVATION OF THE DYNAMIC IMPLICIT OPERATOR BASED ON THE CRANK-NICOLSON IMPLICIT FORMULA

The dynamic model used in Program DBC5 is discussed in Chapter 2 and the free-body diagram of the model is shown in Fig 3.

Since the rigid bars connected between the deformable joints are only responsible for transferring the bending moments, the shears, and the axial thrusts from one joint to another, the equation of motion of the beam-column can be obtained by summing up all the forces, internal or external, at any deformable joint where the beam properties are lumped. For the convenience of the reader, the free-body diagram of a portion of the dynamic beam-column model is shown in Fig Al.

If we take equilibriums for the moments in Bar j and Bar j+1 and for the vertical forces at joint J, then

<u>for Bar j</u>

$$M_{j-1} - M_{j} + (D^{i})_{j-1} \frac{d}{dt} (\varphi_{j-1,k}) - (D^{i})_{j} \frac{d}{dt} (\varphi_{j,k})$$

+ $V_{j}h + (T_{j} + T_{j,k}^{T}) (-w_{j-1,k} + w_{j,k}) = 0$ (A.1)

for Bar j+1

$$M_{j} - M_{j+1} + (D^{i})_{j} \frac{d}{dt} (\omega_{j,k}) - (D^{i})_{j+1} \frac{d}{dt} (\omega_{j+1,k})$$

+ $V_{j+1}h + (T_{j+1} + T_{j+1,k}^{T})(-w_{j,k} + w_{j+1,k}) = 0$ (A.2)

<u>for Joint j</u>

$$V_{j} - V_{j+1} + Q_{j,k}^{T} + Q_{j} + \frac{-R_{j-1}\theta_{j-1} + R_{j+1}\theta_{j+1}}{2h}$$



Fig Al. Free-body diagram of a portion of the dynamic beam-column model.

$$+ \frac{-C_{j-1} + C_{j+1}}{2h} - S_{j,k}^{N} w_{j,k} - S_{j}^{S} w_{j,k} - \rho_{j} \frac{d^{2}}{dt^{2}} (w_{j,k})$$

- $(D^{e})_{j} \frac{d}{dt} (w_{j,k}) = 0$ (A.3)

Solving for the shears V $_j$ and V $_{j+1}$ in Eqs A.1 and A.2 and substituting into Eq A.3 and multiplying by $\,h\,$ yields

$$- M_{j-1} + 2M_{j} - M_{j+1} - (D^{i})_{j-1} \frac{d}{dt} (\varphi_{j-1,k}) + 2(D^{i})_{j} \frac{d}{dt} (\varphi_{j,k})$$

$$- (D^{i})_{j+1} \frac{d}{dt} (\varphi_{j+1,k}) - (T_{j} + T^{T}_{j,k}) (-w_{j-1,k} + w_{j,k})$$

$$+ (T_{j+1} + T^{T}_{j+1,k}) (-w_{j,k} + w_{j+1,k}) + hQ^{T}_{j,k} + hQ_{j}$$

$$+ \frac{(-R_{j-1}\theta_{j-1} + R_{j+1}\theta_{j+1})}{2} + \frac{(-C_{j-1} + C_{j+1})}{2} - hS^{S}_{j}w_{j,k}$$

$$- hS^{N}_{j,k}w_{j,k} - h\rho_{j} \frac{d^{2}}{dt^{2}} (w_{j,k}) - h(D^{e})_{j} \frac{d}{dt} (w_{j,k}) = 0$$

$$(A.4)$$

Based on the Crank-Nicolson implicit formula, we can assume that

$$M_{j-1} = F_{j-1} \frac{1}{2h^2} \left[w_{j-2,k-1} - 2w_{j-1,k-1} + w_{j,k-1} + w_{j-2,k+1} - 2w_{j-1,k+1} + w_{j,k+1} \right],$$

$$M_{j} = F_{j} \frac{1}{2h^2} \left[w_{j-1,k-1} - 2w_{j,k-1} + w_{j+1,k-1} + w_{j-1,k+1} - 2w_{j,k+1} + w_{j,k+1} \right],$$

$$\begin{split} \mathbf{M}_{j+1} &= \mathbf{F}_{j+1} \frac{1}{2h^2} \left[\mathbf{w}_{j,k-1} - 2\mathbf{w}_{j+1,k-1} + \mathbf{w}_{j+2,k-1} + \mathbf{w}_{j,k+1} \right], \\ &- 2\mathbf{w}_{j+1,k+1} + \mathbf{w}_{j+2,k+1} \right], \\ &\frac{d}{dt} (\mathbf{\phi}_{j-1,k}) &= \frac{1}{h^2 2h_t} \left[-\mathbf{w}_{j-2,k-1} + 2\mathbf{w}_{j-1,k-1} - \mathbf{w}_{j,k-1} + \mathbf{w}_{j-2,k+1} \right], \\ &\frac{d}{dt} (\mathbf{\phi}_{j,k}) &= \frac{1}{h^2 2h_t} \left[-\mathbf{w}_{j-1,k-1} + 2\mathbf{w}_{j,k-1} - \mathbf{w}_{j+1,k-1} + \mathbf{w}_{j-1,k+1} \right], \\ &\frac{d}{dt} (\mathbf{\phi}_{j+k}) &= \frac{1}{h^2 2h_t} \left[-\mathbf{w}_{j,k-1} + 2\mathbf{w}_{j,k-1} - \mathbf{w}_{j+1,k-1} + \mathbf{w}_{j-1,k+1} \right], \\ &\frac{d}{dt} (\mathbf{\phi}_{j+1,k}) &= \frac{1}{h^2 2h_t} \left[-\mathbf{w}_{j,k-1} + 2\mathbf{w}_{j+1,k-1} - \mathbf{w}_{j+2,k-1} + \mathbf{w}_{j,k+1} \right], \\ &\frac{d}{dt} (\mathbf{\phi}_{j+1,k}) &= \frac{1}{h^2 2h_t} \left[-\mathbf{w}_{j,k-1} + 2\mathbf{w}_{j+1,k-1} - \mathbf{w}_{j+2,k-1} + \mathbf{w}_{j,k+1} \right], \\ &\theta_{j-1} &= \frac{1}{4h} \left[-\mathbf{w}_{j-2,k-1} - \mathbf{w}_{j-2,k+1} + \mathbf{w}_{j,k-1} + \mathbf{w}_{j,k+1} \right], \\ &\theta_{j+1} &= \frac{1}{4h} \left[-\mathbf{w}_{j,k-1} - \mathbf{w}_{j,k+1} + \mathbf{w}_{j+2,k-1} + \mathbf{w}_{j+2,k+1} \right], \\ &s_{j,k}^{N} &= \frac{1}{2} \left[s_{j,k-1}^{N} + s_{j,k+1}^{N} \right], \\ &\frac{d^2}{dt^2} \left(\mathbf{w}_{j,k} \right) &= \frac{1}{h_t^2} \left[\mathbf{w}_{j,k-1} - 2\mathbf{w}_{j,k} + \mathbf{w}_{j,k+1} \right], \end{split}$$

$$\frac{d}{dt} (w_{j,k}) = \frac{1}{2h_t} \left[-w_{j,k-1} + w_{j,k+1} \right],$$

$$w_{j-1,k} = \frac{1}{2} \left[w_{j-1,k-1} + w_{j-1,k+1} \right],$$

$$w_{j,k} = \frac{1}{2} \left[w_{j,k-1} + w_{j,k+1} \right],$$

$$w_{j+1,k} = \frac{1}{2} \left[w_{j+1,k-1} + w_{j+1,k+1} \right],$$

$$q_{j,k}^T = \frac{1}{2} \left[q_{j,k-1}^T + q_{j,k+1}^T \right],$$

$$T_{j,k}^T = \frac{1}{2} \left[T_{j,k-1}^T + T_{j,k+1}^T \right],$$

and

$$T_{j+1,k}^{T} = \frac{1}{2} \left[T_{j+1,k-1}^{T} + T_{j+1,k+1}^{T} \right]$$
 (A.5)

Substituting Eq A.5 into Eq A.4 and rearranging the order, we obtain

$$\begin{bmatrix} F_{j-1} + \frac{(D^{i})_{j-1}}{h_{t}} - 0.25hR_{j-1}\end{bmatrix} w_{j-2,k+1} + \left\{ -2(F_{j-1} + F_{j}) - \frac{2}{h_{t}} \left[(D^{i})_{j-1} + (D^{i})_{j} \right] - h^{2} \left[T_{j} + 0.5(T_{j,k-1}^{T} + T_{j,k+1}^{T}) \right] \right\}$$
$$w_{j-1,k+1} + \left\{ (F_{j-1} + 4F_{j} + F_{j+1}) + \frac{1}{h_{t}} \left[(D^{i})_{j-1} + 4(D^{i})_{j} \right] \right\}$$

$$+ T_{j+1,k+1}^{T})] w_{j+1,k-1} + [-F_{j+1} + \frac{(D^{i})_{j+1}}{h_{t}} + 0.25hR_{j+1}] \\ w_{j+2,k-1} + h^{3} [(Q_{j,k-1}^{T} + Q_{j,k+1}^{T}) + 2Q_{j}] + h^{2} (-C_{j-1} + C_{j+1})$$

$$(A.6)$$

Collecting the terms and rearranging the order in Eq A.6, we obtain a dynamic implicit operator:

$$a_{k+1}^{w}_{j-2,k+1} + b_{k+1}^{w}_{j-1,k+1} + c_{k+1}^{w}_{j,k+1} + d_{k+1}^{w}_{j+1,k+1}$$

$$+ e_{k+1}^{w}_{j+2,k+1}$$

$$= c_{k}^{w}_{j,k} + a_{k-1}^{w}_{j-2,k-1} + b_{k-1}^{w}_{j-1,k-1} + c_{k-1}^{w}_{j,k-1}$$

$$+ d_{k-1}^{w}_{j+1,k-1} + e_{k-1}^{w}_{j+2,k-1} + f_{j,k}$$

where

$$a_{k+1} = F_{j-1} + \frac{(D^{i})_{j-1}}{h_{t}} - 0.25hR_{j-1},$$

$$b_{k+1} = -2(F_{j-1} + F_{j}) - \frac{2}{h_{t}} \left[(D^{i})_{j-1} + (D^{i})_{j} \right] - h^{2} \left[T_{j} + 0.5(T_{j,k-1}^{T} + T_{j,k+1}^{T}) \right],$$

$$c_{k+1} = (F_{j-1} + 4F_{j} + F_{j+1}) + \frac{1}{h_{t}} \left[(D^{i})_{j-1} + 4(D^{i})_{j} + (D^{i})_{j+1} \right]$$

$$+ h^{2} \left[T_{j} + T_{j+1} + 0.5(T_{j,k-1}^{T} + T_{j,k+1}^{T} + T_{j+1,k-1}^{T} + T_{j+1,k+1}^{T}) \right]$$

$$+ 0.25h(R_{j-1} + R_{j+1}) + \frac{2h^{3}\rho_{j}}{h_{t}^{2}} + \frac{h^{3}(0^{e})_{j}}{h_{t}} + \frac{h^{3}}{2} \left[(S_{j}^{S} + S_{j,k-1}^{N}) + (S_{j}^{S} + S_{j,k+1}^{N}) \right],$$

$$+ (S_{j}^{S} + S_{j,k+1}^{N}) \right],$$

$$d_{k+1} = -2(F_{j} + F_{j+1}) - \frac{2}{h_{t}} \left[(0^{i})_{j} + (0^{i})_{j+1} \right] - h^{2} \left[T_{j+1} + 0.5(T_{j+1,k-1}^{T} + T_{j+1,k+1}^{T}) \right],$$

$$e_{k+1} = F_{j+1} + \frac{(0^{i})_{j+1}}{h_{t}} - 0.25hR_{j+1},$$

$$e_{k+1} = -a_{k+1} + \frac{2(0^{i})_{j-1}}{h_{t}} ,$$

$$a_{k-1} = -a_{k+1} - \frac{4}{h_{t}} \left[(0^{i})_{j-1} + (0^{i})_{j} \right],$$

$$e_{k-1} = -b_{k+1} - \frac{4}{h_{t}} \left[(0^{i})_{j-1} + 4(0^{i})_{j} + (0^{i})_{j+1} \right] + \frac{2h^{3}(0^{e})_{j}}{h_{t}} ,$$

$$d_{k-1} = -d_{k+1} - \frac{4}{h_{t}} \left[(0^{i})_{j} + (0^{i})_{j+1} \right],$$

$$e_{k-1} = -e_{k+1} + \frac{2(0^{i})_{j+1}}{h_{t}} ,$$

 $f_{j,k} = h^{3} \left[(Q_{j,k-1}^{T} + Q_{j}) + (Q_{j,k+1}^{T} + Q_{j}) \right] + h^{2} (-C_{j-1} + C_{j+1})$

This page replaces an intentionally blank page in the original. -- CTR Library Digitization Team APPENDIX B

DERIVATION FOR RECURSIVE SOLUTION OF EQUATIONS (Extracted from Appendix 2 of Ref 4) This page replaces an intentionally blank page in the original. -- CTR Library Digitization Team
APPENDIX B. DERIVATION FOR RECURSIVE SOLUTION OF EQUATIONS (Extracted from Appendix 2 of Ref 4)

Any of the fourth-order difference equations for beams or beam-columns can be written in the following form:

$$a_{j}w_{j-2} + b_{j}w_{j-1} + c_{j}w_{j} + d_{j}w_{j+1} + e_{j}w_{j+2} = f_{j}$$
 (B.1)

Definitions of the coefficients a_j through f_j will vary according to the particular beam formulation considered. The general process of simultaneous solution of a complete system of such equations will be the same in any case. The matrix formed by writing coefficients a_j through e_j at all stations has non-zero terms along only the five main diagonals, and the most efficient method of solutions, therefore, is a direct process that amounts to simple Gaussian elimination. In a forward pass, two unknowns are eliminated from each equation, resulting in a triangularized coefficient matrix of only three diagonals. On the reverse pass, the solution is completed by back substitution. The necessary recursion equations for the process are derived below.

Assume, temporarily, that w_{j-2} and w_{j-1} can be eliminated so that w_j can be written in terms of deflections at two stations to the right. In general

$$w_{j} = A_{j} + B_{j}w_{j+1} + C_{j}w_{j+2}$$
 (B.2)

Writing equations of this form for w_{j-2} and w_{j-1} ,

$$w_{j-2} = A_{j-2} + B_{j-2} + C_{j-2} + C_{j$$

$$w_{j-1} = A_{j-1} + B_{j-1} + C_{j-1}, w_{j+1}$$
 (B.4)

Substituting Eqs B.3 and B.4 into Eq B.1,

$$a_{j} \left[A_{j-2} + B_{j-2} (A_{j-1} + B_{j-1}w_{j} + C_{j-1}w_{j+1}) + C_{j-2}w_{j} \right]$$

$$+ b_{j}(A_{j-1} + B_{j-1}w_{j} + C_{j-1}w_{j+1}) + c_{j}w_{j} + d_{j}w_{j+1} + e_{j}w_{j+2}$$

$$= f_{j}$$
(B.5)

Multiplying and collecting terms,

$$(a_{j}B_{j-2}B_{j-1} + a_{j}C_{j-2} + b_{j}B_{j-1} + c_{j})w_{j}$$

$$+ (a_{j}B_{j-2}C_{j-1} + b_{j}C_{j-1} + d_{j})w_{j+1}$$

$$+ (e_{j})w_{j+2} = f_{j} - (a_{j}A_{j-2} + a_{j}B_{j-2}A_{j-1} + b_{j}A_{j-1})$$
(B.6)

Equation B.6 can be rewritten in the form assumed in Eq $B_{*}2$:

$$w_{j} = A_{j} + B_{j}w_{j+1} + C_{j}w_{j+2}$$
 (B.7)

where

$$A_{j} = D_{j}(E_{j}A_{j-1} + a_{j}A_{j-2} - f_{j})$$
(B.7a)

$$B_{j} = D_{j}(E_{j}C_{j-1} + d_{j})$$
(B.7b)

$$C_{j} = D_{j}(e_{j})$$
(B.7c)

and where

$$D_{j} = -\frac{1}{(E_{j}B_{j-1} + a_{j}C_{j-2} + c_{j})}$$
(B.7d)

$$E_{j} = a_{j}B_{j-2} + b_{j}$$
 (B.7e)

For beam and beam-column problems, the coefficients A_j , B_j , and C_j can be thought of as expressing the physical continuity of the system. In

these coefficients all of the known input data are digested and stored. The coefficients at any one station depend not only on the load and stiffness data at that station but also on effects from all previous stations. These coefficients have therefore been termed "Continuity Coefficients."

APPENDIX C

STABILITY ANALYSIS FOR THE DYNAMIC IMPLICIT OPERATOR WITH INTERNAL DAMPING COEFFICIENT

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APPENDIX C. STABILITY ANALYSIS FOR THE DYNAMIC IMPLICIT OPERATOR WITH INTERNAL DAMPING COEFFICIENT

In studying the stability criterion for the dynamic solutions of a beam with hinged ends and M increments, a solution to the equation of motion described in Chapter 2 (Eq 2.1 or 2.2) can be assumed to be

$$w_{j,k} = A \sin(j\beta_n) e^{k\phi}$$
 (C.1)

in which A is a constant, $j = 0, 1, 2, \dots M$, and $k = 1, 2, 3, \dots, \infty$.

If we consider only the flexibility of the beam F_j , the mass density of the beam ρ_j , and the internal damping coefficient of the beam $(D^i)_j$, then Eq 2.2, becomes

$$\left[F_{j-1} + \frac{(D^{1})_{j-1}}{h_{t}} \right] w_{j-2,k+1} - 2 \left\{ (F_{j-1} + F_{j}) + \frac{1}{h_{t}} \left[(D^{1})_{j-1} + (D^{1})_{j} \right] \right\} w_{j-1,k+1} + \left\{ (F_{j-1} + 4F_{j} + F_{j+1}) + \frac{1}{h_{t}} \left[(D^{1})_{j-1} + 4(D^{1})_{j} + (D^{1})_{j+1} \right] + \frac{2h^{3}\rho_{j}}{ht^{2}} \right\} w_{j,k+1} - 2 \left\{ (F_{j} + F_{j+1}) + \frac{1}{h_{t}} \left[(D^{1})_{j+1} \right] + \frac{1}{h_{t}} \left[(D^{1})_{j} + (D^{1})_{j+1} \right] \right\} w_{j+1,k+1} + \left[F_{j+1} + \frac{(D^{1})_{j+1}}{h_{t}} \right] w_{j+2,k+1}$$

$$= \left(\frac{4h^{3}\rho_{j}}{h_{t}^{2}}\right) w_{j,k} - \left[F_{j-1} - \frac{(D^{i})_{j-1}}{h_{t}}\right] w_{j-2,k-1} + 2\left\{(F_{j-1} + F_{j}) - \frac{1}{h_{t}}\left[(D^{i})_{j-1} + (D^{i})_{j}\right]\right\} w_{j-1,k-1} - \left\{(F_{j-1} + 4F_{j} + F_{j+1})\right\}$$
$$- \frac{1}{h_{t}}\left[(D^{i})_{j-1} + 4(D^{i})_{j} + (D^{i})_{j+1}\right] + \frac{2h^{3}\rho_{j}}{h^{2}}\right] w_{j,k-1}$$
$$+ 2\left\{(F_{j} + F_{j+1}) - \frac{1}{h_{t}}\left[(D^{i})_{j} + (D^{i})_{j+1}\right]\right\} w_{j+1,k-1}$$
$$- \left[F_{j+1} - \frac{(D^{i})_{j+1}}{h_{t}}\right] w_{j+2,k-1}$$
(C.2)

Assuming that the beam has a uniform cross section, uniform elastic properties, uniform mass density, and a uniform internal damping coefficient, Eq C.2 can be simplified as

$$(F + D)w_{j-2,k+1} - 4(F + D)w_{j-1,k+1} + 6(F + D)w_{j,k+1} - 4(F + D)$$

$$w_{j+1,k+1} + (F + D)w_{j+2,k+1} + (F - D)w_{j-2,k-1} - 4(F - D)w_{j-1,k-1}$$

$$+ 6(F - D)w_{j,k-1} - 4(F - D)w_{j+1,k-1} + 6(F - D)w_{j,k-1}$$

$$- 4(F - D)w_{j+1,k-1} + (F - D)w_{j+2,k-1}$$

$$= -\frac{2\rho h^{3}}{h_{t}^{2}} (w_{j,k+1} + 2w_{j,k} + w_{j,k-1})$$
(C.3)

where

F = all F terms (EI)
D = all (Dⁱ) terms divided by
$$h_t$$

 ρ = all ρ terms

Substituting Eq C.1 into Eq C.3, we obtain

$$(\mathbf{F} + \mathbf{D}) \mathbf{f} (\beta_{n}, \mathbf{j}) e^{(\mathbf{k}+1)\phi} + (\mathbf{F} - \mathbf{D}) \mathbf{f} (\beta_{n}, \mathbf{j}) e^{(\mathbf{k}-1)\phi}$$

$$= -\frac{2\rho h^{3}}{h_{t}^{2}} \left[\sin \beta_{n} \mathbf{j} e^{(\mathbf{k}+1)\phi} + 2 \sin \beta_{n} \mathbf{j} e^{\mathbf{k}\phi} + \sin \beta_{n} \mathbf{j} e^{(\mathbf{k}-1)\phi} \right]$$

$$(C.4)$$

where

$$f(\beta_n, j) = \sin \beta_n(j-2) - 4 \sin \beta_n(j-1) + 6 \sin \beta_n j$$
$$- 4 \sin \beta_n(j+1) + \sin \beta_n(j+2)$$

Let

$$q^2 = \frac{2\rho h^3}{h_t^2}$$

Then, dividing Eq C.4 by $e^{(k-1)\phi}$, we obtain

$$\left\{ (F + D) \left[f(\beta_n, j) + q^2 \sin \beta_n j \right] \right\} e^{2\phi} + (2q^2 \sin \beta_n j) e^{\phi} + \left\{ (F - D) \left[f(\beta_n, j) + q^2 \sin \beta_n j \right] \right\} = 0$$
(C.5)

Substituting the following trigonometric identities,

$$\sin \beta_{n}(j-2) = \sin \beta_{n} j \cos 2\beta_{n} - \cos \beta_{n} j \sin 2\beta_{n}$$

$$4 \sin \beta_{n}(j-1) = 4(\sin \beta_{n} j \cos \beta_{n} - \cos \beta_{n} j \sin \beta_{n})$$

$$4 \sin \beta_{n}(j+1) = 4(\sin \beta_{n} j \cos \beta_{n} + \cos \beta_{n} j \sin \beta_{n})$$

$$\sin \beta_{n}(j+2) = \sin \beta_{n} j \cos 2\beta_{n} + \cos \beta_{n} j \sin 2\beta_{n}$$

into Eq C.5 and simplifying the equation, we obtain

$$\left[2(F + D)(\cos 2\beta_{n} - 4\cos \beta_{n} + 3) + q^{2}\right]e^{2\phi} + 2q^{2}e^{\phi} + \left[2(F - D)(\cos 2\beta_{n} - 4\cos \beta_{n} + 3) + q^{2}\right] = 0$$
(C.6)

We can simplify Eq C.6 by using the trigonometric identities

$$\cos 2\beta_n = 2\cos^2\beta_n - 1$$
, $(\cos\beta_n - 1)^2 = \cos^2\beta_n - 2\cos\beta_n + 1$,

and dividing by the coefficient of $e^{2\phi}$; thus

$$e^{2\phi} + \left[\frac{2q^2}{4(F+D)(\cos\beta_n - 1)^2 + q^2}\right]e^{\phi}$$

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+
$$\left[\frac{4(F - D)(\cos \beta_n - 1)^2 + q^2}{4(F + D)(\cos \beta_n - 1)^2 + q^2}\right] = 0$$
 (C.7)

Let

$$B = \frac{q^2}{\left[4(F+D)\left(\cos \beta_n - 1\right)^2 + q^2\right]} \quad \text{and}$$

$$C = \frac{4(F - D)(\cos \beta_n - 1)^2 + q^2}{4(F + D)(\cos \beta_n - 1)^2 + q^2}$$

Hence Eq C.7 becomes

$$e^{2\phi} + 2Be^{\phi} + C = 0$$
 (C.8)

Equation C.8 is the quadratic equation which can be used to evaluate the stability criterion of the dynamic solutions of the beam. The criterion for the solution of w in Eq C.1, which is to be bounded as time approaches infinity, is that the roots of the quadratic, e^{21} and e^{22} , must satisfy the condition that

$$|e^{\phi_1}|, |e^{\phi_2}| \le 1$$
 (C.9)

The two roots of Eq C.8 are

$$e^{\phi_1} = -B + \sqrt{B^2 - C}$$

and

$$e^{\varphi_2} = -B - \sqrt{B^2 - C}$$
 (C.10)

The conditions of Eq C.9 are satisfied if the following inequality is

true

$$B^2 - C \le 0 \tag{C.11}$$

Substituting the equalities of B and C into Eq C.11 yields

$$\frac{q^2}{\left[4(F+D)(\cos\beta_n-1)^2+q^2\right]^2} \leq \frac{4(F-D)(\cos\beta_n-1)^2+q^2}{4(F+D)(\cos\beta_n-1)^2+q^2} (C.12)$$

The above inequality can be reduced to

16
$$(F^2 - D^2)(\cos \beta_n - 1)^4 + 8F(\cos \beta_n - 1)^2 q^2 \ge 0$$
 (C.13)

By omitting the positive term $(\cos \beta_n - 1)^2$, the criterion for the stability of the dynamic solution of a uniform beam with hinged ends and internal damping coefficient becomes

16
$$(F^2 - D^2)(\cos \beta_n - 1)^2 + 8Fq^2 \ge 0$$
 (C.14)

Since $\frac{(D^{i})_{j}}{h_{t}}$ is usually less than F_{j} for a reasonable time increment length h_{t} and the positive value of the term $8Fq^{2}$ is always greater than the negative value of the term $16(f^{2} - D^{2})(\cos \beta_{n} - 1)^{2}$ when the time increment length is extremely small, the above inequality is true. APPENDIX D

GUIDE FOR DATA INPUT FOR DBC5

DBC5 GUIDE FOR DATA INPUT -- Card Forms

Page 1 of 13

IDENTIFICATION OF PROGRAM AND RUN (Two alphanumeric cards per run)

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IDENTIFICATION OF PROBLEM (One card for each problem)

Prob.															
No.		Descrip	ption of	proble	m										
	ſ														
5		I			_										80
TABLE 1.	PROGRA	M CONTH	ROL DATA	(One c	ard for e	ach prob	lem)								
En	nter "1"	to keep	p d ata f	rom pri	or tables				Nu	umber of	cards i	input fo	r tables	3	
2	3	4	5	6	7	8	9	2	3	4	5	6	7	8	9
								1							
5	10	15	20	25	30	35	40	45	50	55	60	6	5 70) 75	B
CABLE 2.	CONSTA	.NTS (Or	ne, two,	or thr	ee cards; NUM	none if	preced	ing Tab	ole 2 is	held) NUM of		NUM of		Option	
	of				of					monitor	c	monito	r Plot-	for	
	beam	Beam			time	Time			Print-	STA	Itera-	STA	ting	line or	
	incre-	incre	ement		incre-	increm	ent		ing	(print-	• tion	(itera	- method	point	
	ments	lengt	th		ments	length			Switch	ing)	switch	tion)	switch	plots	
_	М	H			MT	HT		_	IPS*	MONS	ITSW**	MONI	MOP	LOP	
6	i 10		20		26 30		40	4	16 5C	55	60	0 65	5 70	75	

Complete print out at time sta interval KPS Monitor stations 25 10 30 35 40 45 50 55 60 65 70 21 ** If iteration switch is not zero, use the following card, otherwise ignore: Deflection Max num of Maximum closure iterations for allowable any time step deflection tolerance Monitor stations MAXIT WMAX WTOL 6 10 20 30 45 50 55 60 65 41 TABLE 3. SPECIFIED DEFLECTIONS AND SLOPES (The number of cards as shown in Table 1, none if preceding Table 3 is held) Initial or permanent Case switch STA *** IOPS # Deflection Slope 10 6 16 20 30 40 46 50 *** If case = 1, specified deflection # If IOPS = 1, initial deflection or slope (or both) is = 2, specified slope specified. = 3, specified both = 2, permanent deflection or slope (or both) 143 is specified.

* If printing switch is -1, use the following card, otherwise ignore:

Page 2 of 13

Page 3 of 13



TABLE 4. STIFFNESS AND FIXED-LOAD DATA (The number of cards as shown in Table 1)

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145



TABLE 7. TIME DEPENDENT LATERAL LOADING (The number of cards as shown in Table 1)

Page 4 of 13

147

Page 5 of 13

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The consecutive plots, which are entered 1 for Multiple Plot Switch, will superimpose on common axes. The last plot of the group of plots which are superimposed on common axes must be entered 0 for Multiple Plot Switch.

For consecutive plots which are entered 0 for Multiple Plot Switch, each plot is plotted separately on different axes.

STOP CARD (One blank card of the end of each run)

-4.231E+01

The data cards must be stacked in proper order for the program to run.

A consistent system of units must be used for all input data, for example, 1bs and inches.

All words of 5-spaces or less are understood to be right justified integers, whole decimal numbers, or alphanumeric words.

All words of 10-spaces are right justified floating-point decimal numbers.

Table 1. Program-Control Data

For each of Tables 2 and 3, a choice must be made between holding all the data from the preceding problem or entering entirely new data. If the hold option for any of these Tables is set equal to 1, the number of cards input for that Table must be zero.
For Tables 4 through 9, the data is accumulated in storage by adding to previously stored data. The number of cards input is independent of the hold option, except that the cumulative total of cards cannot exceed 100 in Tables 4 through 7; 60 in Table 8; and 10 in Table 9. Card counts entered in Table 1 should be rechecked carefully after the coding of each problem is completed.

Table 2. Constants

Variables:	Н	HT	WMAX	WTOL
Typical Input Units:	in	sec	in	in

The maximum number of increments into which the beam-column may be divided is 200. Typical units for the value of beam increment length are inches. The maximum number of increments into which the time axis may be divided is 1000. Typical units for the value of time increment length are seconds. If the printing switch (IPS) is set equal to -1, only results at monitor beam stations will be printed, except at every time station interval specified (KPS) where results at all beam stations will be printed. If the printing switch (IPS) is set equal to +1, the complete results of all beam stations will be printed at every time station and the second card therefore is not necessary. If the printing switch is equal to zero or left blank, no intermediate results will be printed. MONS is the number of specified monitor beam stations which are entered on the second card from column 21 to column 70, for printing the monitor stations results at every time station. The maximum number of monitor beam stations which may be used is 10.

- The second input card should be omitted when the printing switch (IPS) is set equal to zero, blank, or +1.
- For nonlinear support curves, which are stated in Table 8, the switch (ITSW) is set equal to either a negative or positive integer, for example, -1 or +1. If ITSW is set equal to zero or left blank, no nonlinear support curves may be entered in Table 8 and the third input card must be omitted.
- MONI is the number of monitor beam stations, which are entered on the third input card from column 41 to column 65, for printing monitor deflections during the iteration process. The maximum number of monitor beam stations which may be requested is 5.

MAXIT is the number of iterations to be allowed for this problem. The maximum number is 50. WMAX is the maximum allowable deflection for this problem.

- WTOL is the closure tolerance for the iteration process.
- MOP is a switch for selecting the method of plotting the results of beam or time stations which are stated in Table 9.
- If MOP is set equal to -1, microfilm plots are made.
- If MOP is set equal to zero or left blank, printer plots are made.
- If MOP is set equal to 1, 12-inch standard ball-point paper plots are made.
- LOP is an optional switch for choosing point or line plots for the microfilm or paper plots.
- If LOP = -j, point plots are made at every jth point.
- If LOP = 0 or blank, line plots are made. If LOP = j, line plots with a point plot at every jth point are made.
- If no cards are entered for Table 9, MOP and LOP may be left blank.
- If MOP is set equal to zero or left blank, LOP may be left blank.

Table 3. Specified Deflections and Slopes

The maximum number of beam stations at which deflections and slopes may be specified is 20.

- A slope may not be specified closer than 3 increments from another specified slope.
- A deflection may not be specified closer than 2 increments from a specified slope, except that both a deflection and a slope may be specified at the same beam station.
- IOPS is a switch for specifying either an initial deflection (or slope) or a permanent deflection (or slope) at the beam station stated on the same card.
- If IOPS is set equal to 1, initial deflection or slope (or both) are specified for the static solutions only.
- If IOPS is set equal to 2, permanent deflection or slope (or both) are specified, both for the static solutions and the dynamic solutions.

Table 4. Stiffness and Fixed-Load Data

Variables:	F	QF	S	Т	R	CP
Typical Input Units:	$1\mathrm{b} imes$ in 2	1b	lb/in	1b	in×1b/rad	in×1b

- Axial tension or compression values T must be stated at each beam station in the same manner as any other distributed data; there is no mechanism in the program to automatically distribute the internal effects of an externally applied axial force.
- Data should not be entered in this table (nor held from the preceding problem) which would express effects at fictitious stations beyond the ends of the real beam-column.
- The left end of the beam-column must be located at station 0.
- For the interpolation and distribution process, there are four variations in the beam station numbering and referencing for continuation to succeeding cards. These variations are explained and illustrated on page 11. There are no restrictions on the order of cards in Tables 4 and 5, except that within a distribution sequence the beam stations must be in regular order.

Table 5. Mass and Damping Data

Variables:	RHO	DI	DE
Typical Input Units:	$lb imes sec^2/in$	$1b imes$ in $^2 imes$ sec	$lb \times sec/in$

See description in Table 4.

Table 6. Time Dependent Axial Thrust

Variables:	TT1	TT2	TT3
Typical Input Units:	1b	1b	1ь

- Time dependent axial thrust values "TT1" and "TT2" are the distributed values entered at beam stations "FROM" and "TO", respectively, of time station "FROM". The values "TT3" are the distributed values entered at beam stations "FROM" of time stations "TO".
- All the values of time dependent axial thrusts at the beam stations within the two limits of "FROM" and "TO" of the time stations between the two limits of "FROM" and "TO" will be linearly

interpolated by the program. There are no restrictions on the order of cards, except that beam stations which are entered as "FROM" and "TO" in the same card must be in ascending order, or in other words, the same beam station is not allowed to be entered as "FROM" and "TO" in the same card.

See page 12 for illustrated examples of interpolation and distribution.

Table 7. Time Dependent Lateral Loading

Variables:	QT1	QT2	QT3
Typical Input Units:	1b	1b	1b

All restrictions stated in Table 6 are also true in this Table, except that same beam station is allowed to be entered as "FROM" and "TO" in the same input card.

See page 13 for illustrated examples of interpolation and distribution.

Table 8. Nonlinear Support Curves

- QMP is a constant which is multiplied by Q-VALUES to obtain the vertical resistance values of the corresponding points of the resistance-deflection curve.
- WMP is a constant which is multiplied by W-VALUES to obtain the horizontal deflection values of the corresponding points of the resistance-deflection curve.
- NPOC is the number of points input on the resistance-deflection curves.
- KSYM is the symmetry option of the input resistance-deflection curve.
- If KSYM is set equal to 1, then the total number of points on the curve will be twice the value of NPOC. In this case, the point of origin of the curve should not be specified.
- If KSYM is set equal to 0 (or left blank) or -1, the point of origin of the curve must be specified. In this case, the deflections which are stated as the first point and the last point on the curve must be equal but opposite in sign.
- If KSYM is set equal to 0, the support is a negative one-way support; that is the beam will lift off the support when it deflects upward.
- If KSYM is set equal to -1, the support is a positive one-way support; that is the beam will lift off the support when it deflects downward.
- There are no restrictions on the order of support curves, except that within any distribution sequence the beam stations must be in regular order. More than one curve may be placed at a beam station.

- The maximum number of points which may be input on any given curve is 10, although additional points up to a total of 20 may be created internally if the curve symmetry option is exercised.
- WNTOL is the closure tolerance of deflections for matching up the unadjusted part of the resistance-deflection curve when the original curve needs to be adjusted.
- QNTOL is the closure tolerance of resistances for matching up the unadjusted part of the resistancedeflection curve when the original curve needs to be adjusted.
- For any particular resistance-deflection curve, the final deflections of the points in storage (WMP times W-values) must be increasing positively, while the final resistances (QMP times Q-values) must be in descending or equaling order. In other words, the resistance-deflection curves must be continuously concave as viewed from the horizontal axis, except that horizontal lines of zero stiffness can be input for representing the plastic regions of the curve. Softening of the support is not permitted; that is, no reversal of slope sign of the segment on the support curve is permitted.
- For both one-way (negative and positive) supports, the input order of the points on the curve is the same, e.g., the final deflections of the points in storage must be increasing positively; internally, the program reverses the order of the points on the positive one-way support curve after the necessary information for tracing the loading paths has been retained.
- If continuing on next curves, the deflections must be equal for the corresponding points of each curve in the sequence of continuation. In this case, WNTOL, NPOC, and KSYM also must be equal.
- The resistance-deflection curve at every nonlinearly supported beam station within the two limits of "FROM" and "TO" will be linearly interpolated by the program.

The maximum number of beam stations which can be nonlinearly supported is 100.

Table 9. Plotting Switches

- Four kinds of plots, deflection or moment along time axis, and deflection or moment along beam axis may be requested.
- If the horizontal axis is TIME, a beam station must be entered from column 26 to column 30.
- If the horizontal axis is BEAM, a time station must be entered from column 26 to column 30.
- As many as five plots may be superimposed by using the multiple plot switch.

Superimposed plots must all be of the same kind.

There are no restrictions on the order of cards except that beam or time stations must be in regular order within a sequence of the same kind of plots.






TABLE 6. TIME DEPENDENT AXIAL THRUST (CONTINUED)

The variable TT(J,K) is input at any bar-number and time station by specifying the value of axial thrust distributed at beam station "FROM" and time station "FROM" in the columns of inputting TT1, the value of axial thrust distributed at beam station "TO" and time station "FROM" in the columns of inputting TT2, and the value of axial thrust distributed at beam station "FROM" and time station "FROM" and time station "TO" in the columns of inputting TT3.



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TABLE 7. TIME DEPENDENT LATERAL LOADING (CONTINUED)

The variable QT(J,K) is input at any beam station and time station by specifying the value of lateral load distributed at beam station "FROM" and time station "FROM" in the columns of inputting QT1, the value of lateral load distributed at beam station "TO" and time station "FROM" in the columns of inputting QT2, and the value of lateral load distributed at beam station "FROM" and time station "TO" in the columns of inputting QT3.



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APPENDIX E

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GLOSSARY OF NOTATION FOR DBC5

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CNOT	ATION FOR JACE		27JV1
с	ALL. ATEMP. AREV	CONTINUT (Y COFFETCIENT	071
c	AN1() + AN2() +	TDENTIFICATION AND REMARKS AN PRATING	07.000
č	at.) = = (J) = r (J)	RECURSION CR CONTINUITY COFFEICIENTS AT	07 101
č		REAM STA J	07.01
c	E(). HTEMP, BREV	CONTINUITY COEFFICIENT	07311
с	BDENOM	BEAM INCHEMENTS DENOMINATOR	67JU1
с	BINC	BEAM INCHEMENTS FOR INTERPOLATION IN	07 001
с		TABLES 6 AND 7	07311
с	RM()	HENDING MOMENT	07 01
с	C(), LTEMP, CHEV	CONTINUITY COFFEICIENT	07301
с	CN-()+ CP(J)	STATIC TRANSVERSE TORQUE LINPUT AND TOTAL	07301
c	L. OTE "P. OPEV	MULTIPLIER IN CONTINUITY COEFF ENS	107503
c	CEN()	SHEAR BETWEEN ADJACENT STATIONS	07.001
ç	CENOM	DENOMINATOR	07JU1
C	CENZI /, DE(C)	LUMPED EXTERNAL DAMPING COEFFICIENT (INPU	TO/JUI
r r		AND ICTAL)	
·		UIFFERINGE	
C C	C187(), D1(C)	LUPPEN INTERNAL GAVPING COEFFICIENT (INFO	07.001
c	End A	THE DEFINETURE OF OMCOUNTER A MERI	07301
·	сыст:5)	SEFECTETED VALUE OF SLOPE AT STA IS	07.001
-		TENDODARY NAME OF DUST AT ITA 10	07.001
ć	E	TERN IN CONTINUITY COFFE FOS	a7 101
ć	ESILA ESMA EST	FACTORS FOR DISTRIBUTING HALF VALUES AT	07.001
c.		FND STAS	07.001
c	ESM	MULTTPLIER FOR HALF VALUES AT END STAS	07301
1	F12()+ E(J)	FLEXIBAL STIFFNESS (FI) (INPUT AND TOTAL)	07 JU1
(,	+	REAM INCREMENT LENGTH	07301
с	Fie	H SQUARES	07303
с	HE 3	H CUAED	07JU1
c	FT	TIME INCHEMENT LENGTH	07.101
с	FT2	H TINES 2	07 いい
ç	HIE2	HT_SQUARED	07 001
¢	F311	H CURED VIVIOED BY HT	07.001
c	H312	H CUREC VIVICED BY HT SOUARES	07.001
c	14	NUM OF PLOTS TO BE PLOTTED ON ONE FRAME	07.001
C .		TA DIVIDED BY 2 (THIEGED)	47301
C C	100 11 130 14 1550	JVARIANLES FOR STOPING THE PLCT FILLS	07301
, ,	I ALGA IEvera N	NUT OF FOINIS IN JTH FLOU	07001
		- SWILCH FUR FIRST UNCONCINE CARE	07 101
	11 3 1	MENT IN TADLE D	07301 07301
	INTER D. THIER A	TNITTAL EXTENNAL DEAM STATTARLES & AND SU	07.001
~	INTOC), THISE /	TNITTAL AND DINAL STATISTICS AND STATISTICS	07.001
1 2	1,547 1, 1,567 1	FT. AL FYTECHAL BEAM CTATASIFE & AND ST	07 301
r	INSAV	TEMPORARY VALUE OF TNIAL)	07 (0)
r	IDENT	NUM OF NUNITNEAR SUPPORTS THERE OUMPUTED	07 101
r	• • • •	DEFL ARE OSCILLATING HETWEEN PRIOR	07.001
C.		AND ADJI STED D-W CHRVES WHEN UNLOAD-	07 JU 1
		ING CASE OCCUPS	07 JU'I
с	IOF()	ROUTING SHITCH FOR INITIALLY AND PERMA-	07361
c		NENTLY SPECIFIED CONDITIONS	ŋ7JU]

1055()	SPECTETED SWITCH INDUT IN TABLE 3 TO INDI	07 111
10437	CATE THAT THE EDECIFIED CONDITIONS O	
	DISPLACEMENTS APE PERMANENT OF INITI	A07.JU1
TOPSAY	TEMPORARY VALUE OF LOPS()	07.001
1058()	ROUTING SWITCH (LOAD ITERATION METHOD)	17.101
IPS	CPTINNAL SWITCH FOR PRINTING THE RESULTS	10101
	OF MONITOR STAS (-1. HLANK, OH +1)	17101
IGS	INDICATOR THAT IA- IS EVEN OR ODD	07JU1
162()+ 192()	INITIAL ANC FINAL FXTERNAL STA (TABLE 7)	07JU1
IHCLL	OPTION TO HOVE TO & NEW PLOT FRAME	07JU1
ISTA()	INTERNAL REAM STA WHERE A NONLINFAR	07JU1
	SUPPORT EXISTS	07JU1
(L) + IS	ROUTING SWITCH FOR HALF VALUES	97 JU 1
ITCS	FOUTING SWITCH (IST CYCLE OF ITERATIONS)	07JU1
ITCSP	ROUTING SWITCH (ZND CYCLE OF ITERATIONS)	07JU1
1T1()+ IT2()	INITIAL AND FINAL FXTERNAL STA (TABLE 6)	07 JU 1
ITSW	ROUTING SHITCH TO INDICATE THAT THE PHO-	07JU]
	HLEM FAS LINEAR (0 OH BLANK) + NONLI-	07JU1
	NEAR CR ROTH (OTHER THAN O) SUPPORTS	07101
IUS (L)	ROUTING SWITCH TO INDICATE THAT THE 9-W	07301
•	EV AT STA NEEDS TO BE ADJUSTED	97301
	INDEPENDENT VARIABLE LINDEXING	107301
ITSH(/	THITE THE DE REAM SIN INPUT IN TARLE	07.001
,	ADDAY CONTAINING THE MONITOD OF FOR	07 01
-1S(/	DOTITING THE STEPATION DATA	07.001
	THIEDHAL HONTOD OTA NUM	07301
	INTERNAL FUNITOR STA NUM	07.001
U.N.	PLOTS TO BE ACCUMULATED ON NEXT PLOT	07 11 1
PS()	ABRAY CONTAINING THE MONITOR STA FUR	07301
5.5.0	PRINTING THE COMPUTED RESULTS	07.001
	INTERNAL BEAM STA FOR SPECIFIED CONDITION	C07.101
•••	IN TABLE 3	07101
JSTA	INTERNAL BEAN STA FOR PRINTING THE PESULT	507JU1
UT 1	INITIAL INTERNAL STATTARLES 6 AND 7)	37363
כ ד נ	FINAL INTERNAL STATTABLES & AND 74	07 JU 1
51	INITIAL INTERNAL REAM STA USED IN TABLE P	07101
J 2	FINAL INTERNAL BEAM STA USED IN TABLE B	07JU1
ĸ	INTEQNAL TIME STA	97JU]
KASE()	CASE NUM OF SPECIFIED CONDITIONS.	07JL]
	$1 = 1$ EFL. $2 = SLOPF$, $3 = 50T^{-1}$	07JU)
KOLOSE	SWITCH TO INDICATE THAT DEFLECTIONS APE	57,001
	NOT CLOSED(IF = 0)	57 JU 1
KCNTEJ	COUNTER FOR CLOSURANCE OF DEFLECTIONS	07301
KEEP2 THRU KEEP9	IF = 1. KEEP PRIOR DATA: TABLES 2 - 9	07301
KET()	SPECIFICA FOR IDENTIFYING CASE NOW O	F07JU1
x5-01)	SPECIFIED CONDITIONS OF DISPLACEMENT	507001
NETH(/	SHILE IN INDICALE THAT PLOT AAIS IS	07001
KENDTI I. KNSTI	TEMPADARY VALUE OF REVRAILERZ)	07.07
NETELS JF KTOLS	DIOT OF HOMENY ALONG THE PEAH AVIS	07.001
**	COUNTER FOR THREE CONSECUTIVE TIME STA	07 101
W MAX	COUNTER FOR NUM OF STAS EXCEEDING MAX	67 10
	ALLONARIE OF STAD EXCLUSION OF	07 101
	ACCERACIC DEFLECTION	0,001

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			_	
VALUE OF KONTE () (FTRST STERATION)	07.001	с	NCC2 THRU NCC9	NUM OF CARDS INPUT IN TABLES 2 THRU 4 FOR 07JU1
INITIAL INTEGNAL TTHE STA TABLES 6 AND 71	07.JU3	с		THIS PROBLEM 07JUL
FINAL INTERNAL TIME STATTAHLES & AND 71	อั7มีเป็	c	NOTE THRU NOTT NUTS	TOTAL NUM OF CARDS IN THE PARTICULAR TABLEOTUUL
SWITCH TO INDICATE THAT DEFLECTIONS EXCEPT	07.001	c	Neva	TOTAL NUM OF CURVES IN TABLE # 07JU1
I THITS OF SUCODET CURVES	07.001	č	KC14 THRU NC19	INITIAL INCEX VALUE FOR THE INPUT TO THE 07JUL
SOUTING COLNTER FOR COMPLETE DEINISOUT	07 001	ĉ		PARTICINAR TABLE 07.101
AC JESH TE	67 (11)	C	NEGR	NUM OF CARES INPUT REPEATEOLY IN TABLE 9 07-01
CONDUCTE DEFUTOUT AN THE OCCUPTE OF ENDOW	07.301 07.101	è	N T T	THEEN USED IN THE TTERATION OF LOOP AT UD
COMPLETE PRINTUDI OF THE RESULTS OF EVENT	07.001	ž	NTTS	TERATION NUMBER AT WHICH THE LOAD ITERAM AT UN
STA AL EVENT KPI LIME STA	07.001	č		TION SETMON CTADTS A7 HIL
INTERNAL TIME STA OF RMI I RMITE /	07301	c c	N T T T	ENVEN VALUE OF NOT FOR THE EIDST CYCLE OF AT UN
INITIAL EXTERNAL TIME STA USED IN TABLE 7	07.001	C C	6111	Savel, Affine Ob wit bow the stant crock of 01001
FINAL EXTENNAL TIME STATISTI IN TADLE	07301	с С		
FRIOR VALUE OF KR24(), KR25(), KH28()	07,001	C .	NNC	NUM OF SUPPORTING STAS WHICH HAVE HUNA UTJUT
CONTINUE SWITCH FOR INPUT CONTINUING ON	07 JU]	ç		LINEAH RESISTANCE DEFLECTION CURVES 07JUI
NEXT CARD (TABLES 4, 5, AND 8)	07JU1	5	NOCI	ROUTING SWITCH WHICH TELLS THE PROGRAM UD 07JUI
SAVED VALUE OF KSAV() FOR EACH PLOT FRAM	F07JU1	c		ONE MORE CYCLE OF LUAD ITERATIONS 07JUI
TEMPORARY VALUE OF KASE()	07JU1	с	_	WHEN THE SOLUTION DOES NOT CLOSE 07JU1
ARRAY CONTAINING EXTERNAL BEAM OR TIME ST.	A07JU1	с	NOS	INDEX FOR COUNTING THE NUM OF SLOPES OF 07JU1
FOR THE TITLES OF PLOTS	07,001	c		NONLINEAR CV. 07JU1
ROUTING SWITCH IN TARLES 4, 5 AND 8	07 JU1	с	NOSJ(J)	NUM OF SLOPER ON THE NONLINEAR RESISTANCE-07JU1
	07.JU1	с		DEFLECTION CV AT A PARTICULAR STA J 07JUI
SYMMETRY OPTION USED IN TABLE 8	07 JU 1	с	NPE	INDEX FOR TOTAL NIN OF PLOTS 07JU1
SPECIFIED VALUE OF KSYME I AT EACH STA J	07.301	с	NPET	INDEX FOR NUM OF PLOTS OF BENDING MOMENTS 07JU1
INITIAL EXTERNAL TIME STA USED IN TABLE 6	07,001	с		ALONG THE REAM AXIS 07JU)
FINAL EXTERNAL TTUE STA USED IN TABLE &	07.11:1	¢	NPCNT	ROUTING COUNTER FOR THE NUM OF POINTS ON 07JU1
SHITCH TO INCIDATE THAT VEST PLOT ARTS IS	07 111	с		THE NONLINEAR RESISTANCE-DEFLECTION 07JU1
DEFLECTIONITE AND OP MCHENTITE 21	07.001	с		CV #HICH IS INCREASING ITS SPECIFIED 07JU1
OPTIONAL LINE OF DOINT PLOT SLITCH WHEN	07.001	č		VALUE OF DEFIECTION FROM NEG TO ZERO 07JUL
USTIC HTODORTH A DR BALL-DOTHT PLOTS	07 811	ĉ		OR FROM NEG TO POSITIVE 07.001
CHITCH WHICH CAUSES THE ATTAL LOADS TO BE	07301	ĉ	NPCT	TEMP VALUE OF NUM OF POINTS ON SUP CV 07JU1
STETDIOLITES TH USERSTAE	07 111	c	NPCTS(J)	TOTAL KUH OF POTNTS ACTUALLY ON THE NON- 07JUL
DISTRICTION OF ALLES	07.001	ć		I INFAR OFSTSTANCE-DEFI FOTION CV AT & AT ILL
INUER FUR EATERNAL BEAM DIA	07301	ř		PARTICULAR STA I
NUM OF BEAM INCHEMENTS	07.001	ř	NRCC()	NUM OF PUTKTS TAPLIT IN TABLE & TO DEFINE ANT.UL
NUM OF PUINTS FOR EACH PLUT FRAME	07301	è	ALCOL .	DIDITION IN SOM THEND DESTSTINGE AT HIS
NUM OF ITEMATIONS SPECIFIED (MAX # DC)	07,001	È.		CEFLECTION CUDVE AT 1515154CC- AT 101
NUM OF CHARACTERS IN PLOT TITLES	A7JU1	2	N0500	DUOD EN NUNDED/DDOD ETODS TE - DIANKI 07 111
NUM OF MUNITOR STATIONS FOR PRINTING THE	07.001	Č.	NPEUR	
ITERATION DATA(MAX = 5)	07.001	C C	RF3	
NUM OF MONITOR STATIONS FOR PRINTING RE-	07JU1	L L		UCFECTION AT A PARTICULAR STATES UTJUT
SULTS (MAX # 10)	07JU1	c		WITHIN THE SEGMENT CONNECTED BETWEEN UTJUT
OPTIONAL PLOTTING METHOD SWITCH	07JU1	C		A DE TETE DE CE CE CONTRE CONT
(-1=MICROFILM, A AR BLAKK=PRINTER,	07 JU 1	<u>د</u>		
+1=8ALL→POINT)	07,001	ç	NPI	INDEX FOR NOM OF PLOIS ALONG TIME AAIS 07001
WULTIPLE PLOT SWITCH SAVED FOR EACH PLOT	07JU1	C	NPTTS(J)	NUM OF PUINTS ON THE NUNLINEAR RESISTANCE-07001
TEMPORARY VALUE OF MPOL 1 FOR PLOTS OF	07JU1	с		DEFLECTION CV AT A PARTICULAR STA J 07JUI
MOMENTS ALONG THE REAM AXIS	n7JU1	c	NPZERC	PRUTING COUNTER WHICH DEFINES THE NUM OF 07 JUL
MULTIPLE PLOT SWITCH INPUT IN TABLE 911F	=07JU1	c		SLOPES SPECIFIED ON THE PAPTICULAR 07JUI
1. PLCT IS SUPERIMPOSED WITH NEXT)	07JU1	C		NONLINEAR RESISTANCE-DEFT. CV 07JU1
M PLUS ONE THRU M PLUS SEVEN	U7JU1	c	NP LS	BOUTING SHITCH TO INDICATE THAT THE 07.001
NUM OF TIME INCREMENTS	07.001	c		OEFLECTION AT A PARTICULAR SUP STA 1907JU1
NT PIUS THO	17301	c		EQUAL TO THE VALUE OF THE FIRST 07JUL
NT PLUS FOUR	07.001	c		POINT ON THE NONLINEAR SUP CV(1F =1) 07JU)
NUM OF PLOTS SUPERTMPOSED ON ONE FRAME	07.001	r	N 5	INDEX NUM FOR SPECIFIED CONDITIONS 07JU1
INDEX OF NUM OF PLATS	07.301	c	NUMOC (J)	NUM OF DECILLATING CYCLES DURING THE OTJUE
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KHCNT() KN1() KN2()

KOFFC() KPC

KPI

K°54 KO1()

KR1

K02()

KSAV()

KSYML J

KSYMJIJ

KT1() KTZ()

KYS()

LOP

LS⊭

LSIA

. Mari

PAXIT

MONI HONS

NoP

₩РС() МРСТ()

PPS()

NT FTF2

Р 1 Р.2 МТР4 NA NAC

NP1 THHU MP7

KR28(), KR25(), KR28() KS() KS4V

KS+4(), KSW5(), KS+8()

	ITERATION PROCESS OF THE DEFLECTION	07.001
	AT A FARTICULAR STA J	07.101
PSRT	INTERPOLATION FRACTION	07301
GDN2(), U(J)	TRANSVERSE STATIC (040 (INPUT AND TOTAL)	07JU1
COROP	PROJECTED VERTICAL VALUE OF EACH SEGMENT	07JU1
	ON THE SUP CV	07 JU İ
GI(J)	ITERATIVE LOAD DISTRIBUTED AT BEAM STA J	07 JU]
	FOR THE PRESENT TIME STA	07JU1
GIM1(~)	ITERATIVE LOAD DISTRIBUTED AT BEAM STA J	07JU1
	FOR THE PREVIOUS TIME STA	07 JU 1
CIMS(J)	ITERATIVE LOAD HISTRIBUTED AT HEAM STA J	07 JU1
x	TWO TIME STAS AGO	07 JU 1
GMP()	INPUT VALUE OF RESISTANCE-WULTIPLIER	07 JU1
CNTOL ()	INPUT VALUE OF RESISTANCE-TOLERANCE	07301
GNIDLUCI	DISTRIBUTED VALUE OF RESISTANCE-TUREPANCE	07001
GP	INPUT SUP OV RESISTANCE VALUES	07301
GPT	FINAL DISTRIBUTED RESISTANCE VALUES	07 JU1
GPV GDWT	FINAL SUP CV RESISTANCE VALUES	07,011
6541	TEMPORART VALUES OF RESTSTANCE OF THE	07JU1
0.5.4.15	NOTALE ON SED CA	07001
6 E (J)	SALE CORRECTION LEAD OF BEAM SIA U FOR	07301
(T()))	THE PREVIOUS TIME STA	07,001
(J+1/	THE PROVE OF THE CEPTORNE LOAD	07,001
	CONTREPORCE TAILE CTA	07301
5.7.1.23	TOTAL VALUE OF THE YIME DEPENDENT LOAD	07301 071U1
3.131.1	THO ITME STA AGO	07 101
GT1()+GT2()+JT2(LINPUT VALUES OF TIME DEPENDENT LOADS	07.001
	(T/ BLE 7)	07.00
REACT()	NET REACTION ON THE RUCOL AT EACH BEAM ST	A07 JU1
HHEN2()+ PHE(U)	LUMPED MASS PENSITY (INPUT AND TOTAL)	07 JU 1
RN2(), R(J)	ROTATICHE RESTRAINT (INPUT AND TOTAL)	07JU1
SLCPE	SLOPF OF EACH SEGMENT ON THE SUP CV	07JU1
SN≂()• S(j)	LINFAR SPRING SUP STIFF (INPUT AND TOTAL)	07JU1
S5(J)	FINAL SPRING CONSTANT DISTRIBUTED AT BEAM	07 JU1
	STA J FOR THE PRESENT TIME STA	07JU1
22KWI(1)	FINAL SPHING CONSTANT DISTRIBUTED AT REAM	07JU1
6 6 mar 1 / 12	STA J FOR THE PREVIOUS TIME STA	07 JU1
33KM3(J)	FINAL SPRING CONSTANT DISTRIBUTED AT REAM	07101
	STA JING LIME STAS AND Syndaus USED on the DRINTCJ DIDTE	07301
TUENOV	TINE THEMELENTS DEMONTRATOR	07301
TESTU	# A LETTING #REANA USED TO INDICATE THAT	07.001
	PLOT IS ALCEA HEAM ANTS	07.001
TESTO	A LETTERS JOFFLY USED TO INDICATE THAT	07.001
	THE VERT AXIS IS DEFLECTION	07.001
TESTY	= 4 LETTERS #MONTE USED TO INDICATE THAT	07.001
	THE VERT AXIS IS MOMENT	17JU1
IESTI	# 4 LETTERS #TIME# USED TO INDICATE THAT	07001
	PLOT IS ALONG TIME AXIS	07JUI
т 1 к C	TIME INCREMENTS FOR INTERPOLATION IN	07JU1
	TABLES & AND 7	07.001
てんえく りゅうてくり	STATTC AATAL TENSION OR COPPERSION (INPL	T07 JU1
	AND ICTAL)	07 111

TT (J+1)	TOTAL VALUE OF TIME DEPENDENT AXIAL	07 J. 1
	THRUST FOR THE PRESENT TIME STA	7 10 1
11(J+ 2 /	TETAL VALUE OF TIME GEPENDENT AXTAL	07101
	THRUST AT TWO TIME STA AGO	
111()+115()+113(LINPUT VALUES OF TIME DEPENDENT AXIAL	07 Jul
	THRUSTS (TABLE 6)	27.00
# Cut KKI	COMPUTED DEFLECTION AT BEAM STA . PUP THE	37301
	PRESENT TIME STA	07 101
#(C+KK=1)	COMPUTED DEFLECTION AT REAM STA J FUP THE	07JL1
	PREVICUS TIME STA	07301
#(J+KK-2)	COMPUTED DEFLECTION AT BEAM STA J 100	07101
	TIME STAS AGO	07JC1
▶ C F O P	PROJECTED MODIZONTAL VALUE OF EACH SEGMEN	107741
	ON THE SUP CV	07JU1
M M	ITERATION DATA STORED FOR THE FIRST CYCLE	07,161
	OF ITERATION	07JL!
W ~ L X	SPECIFIEU MAXIMUM ALLOWABLE DFFLEC/ION	07JL1
WMP()	INPUT VALUE OF DEFLECTION-MULTIPLIER	1 11 70
WNTOL()	INPUT VALUE OF DEFLECTION-TOLERANCE	07.051
WNTOLJ()	DISTRIBUTED VALUE OF DEFLECTION-TOLEPANCE	073-1
wF	INPUT SUP OV DEFLECTION VALUES	07.01
WPT	FINAL DISTRIBUTED DEFLECTION VALUES	07101
WPV	FINAL SUP OV DEFLECTION VALUES	07 JU I
WPVT	TEMPORARY VALUES OF DEFLECTION OF THE	07361
	PCINTS ON SUP CV	07361
₩\$(J ^{<})	SPECIFIEU VALUE OF DEFLECTION AT STA JS	07JU1
WC	ITERATION DATA STORED FOR THE SECOND CYCL	F07JU1
	OF ITERATION	07J'
Ww()	DEFLECTION AT PREVIOUS ITERATION	07301
www.()	DEFLECTION TWO ITEPATIONS AGO	07301
XAXIS()	NAME OF HURIZONTAL AXIS INPUT IN TABLE 9	07JU1
XF()	STORED VALLES OF BEAM OR TIME STA NUM	07301
	FOR THE PLOTS	07JU1
XM()	SCALE FOR THE HORI7 PLOT AXIS	07JU1
Y(), YT(), YP()	STORED VALUES OF DEFLECTIONS OR MOMENTS	97JL]
	FOR THE PLOTS	07361
YA, AA	CUEFF IN STIFFNESS MATRIX	07361
YAXIS()	NAME OF VERTICLE AXIS INPUT IN TABLE 9	07JU:
48 · HG	COEFF IN STIFFNESS MATRIX	07JL1
YC. CC	COEFF IN STIFFNESS MATRIX	07JU1
YD, 0C	COEFF IN STIFFNESS MATRIX	07JL:
YE, EE	CHEFF IN STIFFNESS MATRIX	07 Juj
YF, FF	COEFF IN STIFFNESS MATRIX	07301
YM()	SCALE FOR THE VERT PLOT AXIS	07301

 $\mathbf{v}_{\mathbf{v}} \mathbf{v}_{\mathbf{v}} \mathbf{v}$

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APPENDIX F

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FLOW CHARTS AND LISTING OF DECK OF PROGRAM DBC5

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	PROGRAM DRCS (IFPOIL DO	1910 - J 19 DECE - NACTEO - IACK CHAR - MATLOCK - 1	(۵. ۵۰
1	EDMMAL (ENM MACHE	pr (next rows DENTRIAN DATE OF DIA 13 3 3	54 001
- 1	1	SPECKIMPEVISION OPIE = 20 JUN 11 1 J	
	DIMENSION AVI (351 + 4	N2(14)+ H(207)+ H(207)+ C(207)+	10000
1	1 F(207)+ 01	207) + 5(CO7) + (CUT) + H(CUT) + (CUC) +	10000
ā	2 KEY(207)+1	N13(20) + KASE(40) + HS(27) + SHS(40) +	10//00
- 2	3 TN14(100)+	1+74(100) + KR24(100) + FN2(100) +	05560
- 4	4 00x2(100)	St2(100), TN2(100), RN2(100), CN2(100),	055F0
ę	5 K5w4(100)+	IN15(100) + IN25(100) + KP25(100) +	05550
e	6 KSL5(100)+	0w(207) + PHO(207) + CI(207) + CE(207) +	10000
	7 Pro(207), W	(207,3), CPA(207), 4EACT(207), 58(207),	10700
\$	8 96652(100)	. 01N2(100). DEN2(100)PS(10)! SYMP(10) !	OFSEO
	DIMENSION TT (207+2) .	QT(207+2), TT1(100)+172(100)+141(100)+	10000
1	1 102(100).	<pre>kTl(lnn)+ kT2(lon), bdl(l00), kQ2(l00);</pre>	05860
-	2 TT1(100).	Tr2(100) + TT3(100) + 471(100) + 412(100) + 11	055E0
	3 073(100).	XAXIS(10), 1YS#(10), MPS(10), KSAV(10),	05550
- 2	6 KEVP(10)+	Y(10,1002), MPC(10), IDX(10), *5(10).	055F0
ė	5 YP: 50101.	1x(1002) . XF(1004), 10(7). 15(10)+755(2)	17580
	DIMENSION 1118(20)+	TN28(20) + KR28(20), GMP(20), ++P(20).	[۵ز ۵۵]
	2 KSYM(20)+	KSLA(20) + GP(20+10) + #P(20+10) + GPV(20+20)	04 341
	3 · WPV (20+2	01. NPCTS(20). JINS(5), GI(207). 105(207).	[4(4)
	4 IOE (207) -	\$5/2071. WWW (207), KOFFC (50), YAXIS(10).	14LA0
	E FCNT(5/1).	W1150.5) . HUMOC(207) . (207). (Dec(20).	04,14]
	6 T-56120714	BU(C). C), ISTA (1001 . + C: OSF 12) .	15401
- 2	7 WENT(100+2	0), GPVT(100-20), Mr.C(20), .DRVP(100,10),	1SFE1
	e Si OPE (100)	101, WDROP(100+10), KTS(10), NOSJ(100),	15FE1
	p vPT(100+20	1. OPT(100,201. NPTTS(100). WNTULJ(100).	15FF1
	A 05 101 J (100) _ WNTOL (20) . QNTOL (20) . K5YHJ(100) .	15881
,	g RECNTISOL	TEUSIONTI + KSAVTISI + KEYPTISI + MPOTISI +	1 <fe1< td=""></fe1<>
- 2	C 8YST(5), 1	7(5,207) + 55×+1(207) + YH(4) + XH(4)	1=FF]
,	OTHENSION SSEMELEOTI	- (TM1(207) - GIP2(207) - G5(207)	2] 4 7 1
	DATA SYME / 1HP. 1H++ 1	HX. 1H 1H0. 1H=. 1H\$. 1H#. 1H/. 1H0 /	11.JE0
	COMMENTSPIOT / NIDTH SPHIL	BIG.SYND	055F0
	COMMON //OT/ , OP. MC. 15	CHI. MOP	175E0
	TATEGER WIDTH		055E0
6	ECHWAT (45. 45.)		26JV]
Ē	EORMAT (15. 15.)		175F0
6	FORMAT (15. 45 1		17550
10	FORMAT (SH . PO)	• 10HTTR1M)	27FE4 10
ii	FCRMAT 1 5H1 + 803	• 10H1TRIM)	27FE4 IF
12	FORMAT (1645)		04MY3 10
13	FORMAT (5X. 1645	·)	27FE4 1C
14	FORMAT 1 45, 58, 1445 1		18FE5 [r
15	FORMAT (///10H PRCB	. /5x. 45. 5x. 1445)	IPEES IC
14	FORMAT 1///17M PEOP	(CONTD) . /5%. A5. 5%. 1445 1	IPFES ID
20	FCEMAT (1415)		300E0
21	FORMAT (21 5X+ 15+ F14	3) + 5X+ 615)	30060
22	FORMAT (5% - 15+ 10×+ 10	15)	05560
21	FORMAT (5% - 15+ 2010+1	108. 411 /	30550
รัก	FORMAT (2(5X+ 15) + 2F1((.3. 5X. 15)	175E0
41	FORMAT 5%. 315. 0110.3	Ê Ĵ	23084
51	FORMAT (5Y. 315, 10X.	F10.3 }	055E0
61	FORMAT (54. 21 215. 5%) (3830,2)	OSSEO
ĂÌ	FORMAT (5x. 315. 2810.1	+ 2I5 + 2× 10 + ×)	24 m F 0

82 FORMAT L 30X, 10F5.0)	24020
91 FCEMAT - 2(CK+ A4)+ 2(5X+ I5) }	10FE1
100 FORMAT (///35F TAPLE 1 + REOGRAM-CONTROL DATA	24 JA4*
1 / 43X, 30H TABLE NUMBER	30020
2 / 434, 405 2 3 4 5 6 7 8 9	300F0
3 // with HOLD FROM PRECEDING PROBLEM (1#HOLD)* 2x+	300E0
4 RT5+	31060
5 / 33H NUM CARDS INPUT THIS PROBLEM, 10%, 815)	30050
200 FCRMAT (///24H - TABLE 2 - CONSTANTS /)	53484*
201 FORMAT C 32H NUM OF AFAM INCREMENTS+ 43X+ IS+	055E0
1 / 32H PEAN INCREMENT LENGTH + 39X+ F10+++	05560
2 / 32W NUM OF TIME INCREMENTS: 43X+ IS+	CESEC
3 / 32H TIME INCREMENT LENGTH + 38X+ F10+3+	055E0
4 Z 35H OPTIONAL PHINTING SHITCH + 40X+ 15+	055F0
5 Z 416 NUM OF MONITOR STA (PRINTING), 34X. IS.	300E0
6 / 40H ITERATION SWITCH (O=LINEAR) + 35×+ 15+	3rDE0
7 / 42H NUM OF MONITOR STA (ITERATION) + 33X+ 15+	340F0
A Z 54H PLOTTING NETHOR (=1=MIC.+0=PRINTFR,1=PAPER)	25 JA1
R +2)X+ 15+	25 JA1
P / 3NH LINE OF POINTS PLOT OPTION + 39X + 15)	JODEO
202 FORMAT (/ F2HCOMPLETE PRINTING TIME INTERVAL AND MONITO	28JA1
1 28HH STATIONS FOR PRINTING	25 341
2 / 454 TINE STA INTERVAL OF COMPLETE PRINT, 30X.15	OFSEC
4 / 52H MONITOR STATIONS (PRINTING)	04.141
5 / 54- 1 2 3 4 5 6 7 8	04.041
6 6H9 10. /. 5X. 5HSTA J. 1015)	04.143
203 FREMAT (/ 52h	25.01
1 / JEH MAXTH W ITERATION NUMBER + 40X+ IS.	94.41
2 / 40H MAXIMUM ALLONABLE DEFLECTION + 30X+ E10+3	364.41
3 . / JAH DEFLECTION CLOSUPE TOLERANCE, 34X, E10.3.	04.41
4 / 19X. 27HMCHITOR STATIUNS (ITERATION).	04JA1
5 / 1/2+ 25+ 1 2 3 4 5+	04341
6 / Ex. EMETA J. 515)	04.141
300 FORMAT (11147H TAPLE 3 + SPECIFIFN DEFLECTIONS AND SLOPES	20344
1 // SX. 48H STA CASE DEFLECTION SLOPE .	17560
2 7x 22HTOPS (IF=1+INITIAL +	10645
3 Z K-X+ 22H]F=2,FF9HdNENT) 1	24. 11
111 FORMAT (104. 13. 7x. 12. 8x. E10.3. 9x. 44NONE. 5x. 15)	175E0
312 FORMAT (10x, 13, 7x, 12, 11x, 4HNONE, 8x, E10.3, 5x, 15)	175E0
313 FORMAT (10x+ 13+ 7x+ 12+ 3×+ 2(5×+ E10+3)+ 5×+ 15)	17SE 0
400 FORMAT (7774RH TABLE 4 - STIFFNESS AND FIXED-LOAC DATA	23004
J ZZ SIH FPCH TO CONTO F OF S	OPFES
2 20H T R CP7)	05550
411 FORMAT (5x, 214, 13, 1x, 6E11-3)	274944
4]2 FORMAT (5%, [4+ 4%+ [3+ 1%+ 6E]]+3)	23494*
413 FORMAT (9x, 14, 13, 1X, FF11,3)	23084*
500 FORMAT 17774RH TABLE 5 - MASS AND DAMPINGS DATA	0= 550
1 // 45H FECH TO CONTO PHO DI DE +/1	OFSED
AND RELAVET 1777 624 TAPES & STAME DEPENDENT AXIAL THRUDT	CESEO
1 1/ SY, 4PH REAM STA TIME STA	055F0
2 JECT ARE FROM TO FROM TO TT	05550
3 20H TI 2II ()	055F0
610 FRENAT (10x, 210, 7), 214, 10x, 3F11,3)	CESEO
TON FORMAT 1/2/ 4:00 TAPLE 7 - TIME LEPENDENT LOADING	0 = < F 0

1 1/	53+ 30+	PEAM STA	TIME S	Тд		05 S E O		1
2 /	5x	FEOM TO	FPAK	10	G T 1	055E8	973	FORM
· ,	205	012 0	13 / 1			054E0	920	FORM
BOD FORVAT	1/1/ 404	TARIE 9 - NO	NETNEAR SI	UPPORT CURVES	11	ZADED	000	FORM
1	(1), qui	ERON TO CO	NTO G-MIL	TTOL TEO	LOI TER POL	240E0		1
2	210	TE EVE ODT WITH	FRANCE -	TO: FRANCE / 1		ISFEI	0074	้รุกคม
	1 6 4 316	3013 3 313 3				15551		
RII FORMAL	1 224 217	• < !!!!!!!!!!!!!!!!!!!!!!!!!!!!!!!!!!</td <td>-12.3 /</td> <td></td> <td></td> <td>10001</td> <td>06.77</td> <td>1</td>	-12.3 /			10001	06.77	1
BIZ FORMAT	1 741 121	741 174 CF 13.34	25174 2516	e, s :		17001	9977	- F ()# #
813 FORMAL	1 10x+ 51	54 2113434 2174	CE12+3 /	•		1995.1)
B14 FORMAT	(10-	Q + 10F7.0)				24050	997A	FUHH
PIS FORMAT	с зан	W + 10F7+0)				CAPED	9979	FORM
820 FORMAT	L // 35H	TABLE 10- CA	LCULATEC +	RESULTS /		04JA1	9981	FCEN
1	354	Jargan	AXIS. KET	INE 4X15 / 1		05SE0		3
900 FOFMAT	1 111 324	TAHLE 9 - P	LOTTING	#1TCHE5		30DE0	5966	FOPM
1 11	+ 5X+49H H	PRIZONTAL VERTIC	AL BEAM CH	H TIME MULTIPL	E PLOT .	IOFE1	. 9983	FORM
2 /	- SX+48H	A315 AX15	STAT	TICN SWIT	CH +	lofel		1
3	274(1	F=1.SIPEPIMPOSE	HITH NEXT	./.		26301	9994	FORM
4	538.278 1	FRO FLOT ALL SAV	ED PLOTS))		26.001		1
ONT FORMAT	1 / 400	STA DI	57	DEFI SI	0 PF	OSSEO		2
301 FORERI	1 2 401		ELP DI	FACT Z SQ		OBSED	0085	ັຮດຍມ
	305	MU" BEERN				30050	9084	FORM
902 FOPMAT	1 // 318	STATIC	HE SUL 15			300F0	00.64	
903 FORMAT	(/ 255		=NONE - 1	40. F		05560		1
904 FORMAT	(// 40-	TCC PHCH DATA	FOR AVAL	LANCE STORAGE	//) ()(), = (), ()	05520	9987	
905 FORMAT	1 461	USING CA	A FROM T	HE PHEVIOUS PH	WHLEM I	USPEAT		1
907 FORMAT	(/ 52H	EPHOR STOP	SLOPES AN	ND DEFLECTIONS	. Імрноренц	Y310E4	9988	FUHN
3	101 5	recified)				31DE4		1
908 FORMAT	(9X: A4:	6X+ A4+ 5X+ 15+	8X: 15)			10FE1	9993	E.U.B.H
910 FORMAT	(43⊢	AUDITION	AL DATA FO	OR THIS PROBLE	м ,	31064		3
911 FORMAT	(55+	PLOI OF DE	FLECTION .	VS TIME FOR BE	AM STATICN	5055E0	4994	FORM
1	44H 0F	6H 808080 // 1	• •		-	OSSFO		1
012 609447	1 101. 12	AH CURVE (. AL		13.1		OSSEA		2
OLD FORMAT	1//. 59.16		• . / \			OSSEA	9095	FORN
SIS FORMAT	1 11 305	88888 UVLANT	E DECHISE	00000. /)		DECEN		1
915 FURMAT	1 1 2 305	TTHE STA				02320		2
916 FORMAL	12 10 8 9 1.3		2719 Z / #T+T+()55	NOT THE UDDED		001F0	0004	໌ຮຸດຊະ
917 FORMAT	1 7740	EFFOR SIOP	SIAILOUS	NOT IN ORIGIN	A		1446	
919 FORMAI	1224	Seare brui ut bf	∿D* MO⊶• /	AS LINE LON HE	HM SIDILON	510-11		1
1	. + 4 M C F + 4	Ar 400, 11)				05420		
921 FORMAT	(5X+ 14+	-Sx+ 5E15+3+ 10X	• E12.3+ 1	10/* £15+3)		05540	9996	FOHM
922 FORMAT	- (34X) El	2.3+ 10X+ F12.3)			055E0	C	-STAR
923 FORMAT	(/ 35×+	SHMCN TTOR	STATIC	CF2+ /+ SUX+ 3	HJ ##	28DE0		
1	5 (T5+ 7X } }				280E0		
924 FORMAT	1 / 108+	3HECO+ 13+ 21H 1	TERATION.	THERE ARE. 14	. •	28050		
1	214 5	TATIONS NOT CLOS	ະຄົາ			2 ADE 0		
035 506507	(20¥. 5E	12.3 1				ZPDEO		
030 EOEMAT	1 548	BOBBO PLAT OF D	EFIFCTIONS	S ALONG BEAM A	XIS AT TIM	EDSSEA		
424 - (# 1	.134 674	TONC OF	98. // V	- HETHE FC H		05550		
	•[3H 31#		KUD NOW	ALONG HEAD A	ATS AT TIN	FIAFFI		
939 FUHMAI	1 544			• ACONS SPAR A		DECEA		
1	13H STA	TIONS OF IGH POP				0		
960 FORMAT	(/49X+ 31	HasassFKHOB CLUB	IN TABLE		tur euro	00560		
1	~0H	SETTHER SPEC	LETING ON	L HEAM SIA IN	THE DAME C	+ USSE0		
2	38+	SD DP REAM STA N	UT IN ASCE	ENDING ORDER+,	./)	055F 0		
970 FORMAT	(5 X, 35H	PRINTING RESULT	5 IS NOT P	REQUESTED . /	ł.	05520		
971 F" # AT	1 SX, 29H	MICPOFILM PLOT	15 REQUEST	TFD*+ / 1		175E0		
972 FOFMAT	(SX. 50H	STANCARD 12 THC	H HALLPOTH	NT PAPER PLOT	IS REQUEST	E175E0		

1		24r+. /)			175E0
973	FORMAT L SX. 2	THOPPINTER PLD	T IS REQUESTED)•, /)	17550
980	FORMAT (///SCH	UNDESTUNAT	TER LAROA STOP	2	I CE SEO
990	FORMAT (//EX.	44HODOOD TIME	INCHEMENTS SP	PECIFIED EXCREDS	5 LIMIT # 055E0
1		5500 //)			05SE0
9976	FORMAT 1 5X, 50	HERRON INPUT	IN TABLE 9. MC	DPE THAN 5 CURVE	S TO BE PISFEL
	1	SHEOTTED IN ON	E FRAME BY PPI	INTEP PLOT 1	15FE1
9977	FORMAT (JAX, 4	ADJU ADJU	STED LOAD-DEFL	CURVE TO CALLU	JLATE CLELISFE1
	i	SHECTIONS .	1		15FE1
997P	FORMAT (// 444	• • • • • • • • • • • • • • • • • • •	NOT LERO WHEN	ITSW SHITCH IS	ZEHO) 290E0
9979	FORMAT (11 46.	- 115# Sw1	TCH IS NOT ZEF	O WHEN NOVA 19	ZER- 1 295FD
9981	FCFPAT (54H	*****\$UPPA9	T CURVE DATA I	INDHCDENTA SPECT	(FILGA====Z40E0
1	3				CACED Satetel Second
9982	FORMAT (// EX	I GEMMERTATING TO	TERALION NUMBE	TE ENCEER MICH	SCIPIER 1280PD
4943	FORMAT 1 // 467	- LALCULAIN	TATIONE AT ITM	48 STA # . 14 3	16551
000		AII 131 267 31	(A) (VNS A) (1" Fo oted Arfwer	TS FYOFFE I THIT	S OF 280EA
4424		- CHUCOLAH	te oliotne ist	CYCLE OF . /	15661
	191	11594710	N NO 13. 15:	AT TIME STA	• 14) 15FF1
0065	508WAT (// 46)		+ ERROR OF DI	AGNOSTIC	++++ 1 29"E0
9086	FORMAT 1 22 471	 SOLUTION 	DID NOT CLOSE	IN SPECIFIED N	UNBER ZADED
	341	TERATIONS	S FOP THE FIRS	ST CYCLE)	15651
9987	FORMAT (// 47)	- SOLUTION	NTD NOT CLOSE	IN SPECIFIED N	UNBER INFEI
	351	S OF ITERATION	S FOR THE SECO	DND CYCLE)	ISFE1
9948	FORMAT (SX+ 50	он ∗+++ +€квов г.	TOP DUE TO THE	E SLOPES OF THE	NONLINEADZIADI
1	+1	DH CV AT STA. "	TA, 28H 15 NOT	T IN PROPER ORDE	Reese 1 21401
9993	FORMAT (SX. S	BH ● ≢₽₽₽E₽₽OR I	VPUT IN TABLE	P. STATION NC1	IN ASCENCIEFEI
1	1	HING ORDER***	54)		15FE1
4994	FORMAT 1 1/ 461	 CALC 1 LTF 	TO DISPLACEMEN	ITS EXCEED LIMIT	S UP 24050
1	351	- CUPPERT CUPVI	ES OURING ZND	CYCLE OF + /	[
	181	- ITERATIO	N NO++ 13+ 15*	H AL TIME STA -	9 14 7 15PE1
9995	FREMAL (// 341		ANN IN TAMEE C	S INF(1)	A TEL OFFER
	58.1	ATT CARUS AR	5 585CILLING -	DANC BEAN STATIS	545EA
0004		##### FOR/	TARIF R		GSSEG
4440	54.12	JOH CAEDS ADE	SOFCIEVING SA	ME TIME STATION	115. 055F0
÷		25H FAS PLOT	ALONG REAM AX	[5])	CESEO
4000	FORMAT 1 5X. 2	THARE FRADE IN	NPS OPTION **	** 1/1	CESEC
C	START EXECUTION	OF PROGRAM	SEE GENERAL F	LOW DIAGRAM	CECEO
-	ITEST	= 5H			CECEN
	TESTT	= AFTIME			055E0
	4T>37	= AFHFAM			05560
	TESTI	≈ 4>14FL			1 HE !
	TESTM	= AHMCMT			19FE1
	WINTH	= 60			54550
	SMALL	= 0.0			955E0
	BIG .	0.0			05560
	NGT2 -	π η - •			V7589 05555
	NCT4	= 11			05550
	NUTH	* *			051C0
	NL16 -	• •			554F0
	NOVE				C6.IA)
	N	= 6			54.01

PRINT 10	JONEO	
CALL TIC TOC (1)	0 A M 4 0	
C+PRCGRAM AND FRORIEM ICENTIFICATION	04MY3 10	
$pFA() = 12 + (A^{k})(k) + k = 1, 32$	leff5 lr	
1010 READ 14. $PROB. (ANT(N), N = 1, 14)$	288G3 [r	
TE (NPHCH - TTEST) 1020+ 9999+ 1020	055F0	
IO20 PRINT 11	264G3 lr	
PRINT	18665 lC	
$PRTNT 13 + (\Delta t) (h) + h = 1 37)$	18FF5 IC	
PRINT 18. NEROP, (ANZ(N) N = 1. 14.)	24463 IC	
CINFUT (ABLE 1	10053	
READ 20+ KEEP2+ KEEP3+ KEEP4+ KEEP5+ KEEP6+ KEEP7+ KEEPA+ KEEP9+	30050	
1 NCD2+ NCD3+ NCD4+ NCD5+ NCD6+ NCD7+ NCD8+ NCD9	30DE0	
PRINT 100, KEEP2, KEEP3, KEEP4, NEEP5, KEEP6, KEEP7, KEEP8, KEEP9	,30PF0	
1 NCC2, NCD3, NCD4, NCD5, NCD6, NCD7, NCUR, NCU9	30DE 0	
CINPUT TAHLE 2	10JF3	
PRINT 200	05550	
IE (KEEP 2) 9950. 1210. 1220	055E0	
1210 READ 21. H. H. MT. HT. 1PS, MONS: ITSW: MONI: MOP. LOP	3 n n E 0	
CCCHPUTE CONSTANTS AND INDEXES	10JE3	
HT2 = H + H	11,145	
HE2 = H + H	30643	
HF3 = H + HF2	JUNAJ	
HTF2 = HT * HT	05510	
MPI = M + 1	05560	
$MP_{2} = K + 3$	05560	
MP4 = M + 4	30443	,
mpc = P + 5	20MT3	(
MPK = 1 + 6	TUJER	
MP7 = M + 7	30FT3	
MTP2 = 01 + 2	0.5550	
VIDA = HI + 4	05570	
HITZ = HEIZ Z HIEZ	OFCEA	
	30050	
IF (IPS) 1225, 1235, 1235	30050	
1235 IF (115W) 1250 (1240) (150	JORFO	
1250 PHINT 2014 W. H. MIL FT. 105. MUNCE 11344 FONT, 001. CF	JOCEO	
	300E0	
1220 PRINT 907	JODEO	
(r + (-r + r + 2 + 1) + 1 + 2 + 9) + 1 + 2 + 9) + 1 + 2 + 9 + 1 + 2 + 3 + 2 + 3 + 3 + 2 + 3 + 3 + 3 + 3	BODEC	
1240 IF CLINW COOPLESS LESS HONS, TISK, MONT, MOR, LCM	30050	
1255 FRIMI 221, 3, 4, MIT 11, 1931 MUSS FIDE OFFE	BADEN	
	30050	
[225 SEEN 224 RELATION I FILE IN MORE A	300 F0	
NOTE DETET 201- V. M. MT. DT. 105. VONS, TISK, MONI, MUP, LOP	30DE0	
$\frac{1}{2} = \frac{1}{2} = \frac{1}$	30080	
$\frac{\partial P_{1}}{\partial r} = \frac{\partial P_{1}}{\partial r} + \frac{\partial P_{2}}{\partial r} + \frac{\partial P_{1}}{\partial r} + \frac{\partial P_{1}}{\partial r} + \frac{\partial P_{2}}{\partial r} + \frac{\partial P_{1}}{\partial r} + \frac{\partial P_{2}}{\partial r} + \frac{\partial P_{1}}{\partial r} + \frac{\partial P_{2}}{\partial r} + \frac{\partial P_{2}}$	304F0	
	30050	
$\frac{1}{1} \frac{1}{1} \frac{1}$	BODEC	
1240 DOTAT 2018. MAXIT, WHAX, WTOL, (JIMS(1), 1 = 1, MUNI)	BORFO	
	30CE0	
1230 DETATION H. MT. HT. 125. MONS, ITSW. MONI, MOP. LCM	3 nnE 0	
TRAD FRANT LATE OF THE TRADE OF	01095	
CIVENT IN TE 2		

30050	13CL PRINT 300	11345
08298	1301 - 601303 = 1 + 857	05550
CANYA IN	$\kappa F Y (\omega) = 1$	03JE3
18FF5 lr	10P(J) = 2	175Fn
2PAG3 IT	TSW(_) = 0	15FE1
055F0	1303 CONTINUE	23F F5
264G3 1r	IF (KEFP3) 9980. 1310. 1305	26475
18FF5 1C	1305 PHINT 404	26M42
18FF5 IC	1326	26445
244G3 IC	1310 TE (NCO3 - 20) 1312+ 1312+ 1311	25445
10053	131) PPINT 904	25 4 7 5
BODEN	er TO 9055	055E0
30 DEO	1312 ארב גער 1312 ארב גער 131	25MY5 .
-30DEO	1F (NCT3 - 1) 1326. 1320.	08488
3 ODE 0	1320 00 1325 h = 1+ NCT3	26557
10JF3	RFAD 31+ IN13(N)+ KASE(N)+ W5(N)+ D+5(N)+ IOPS(N)	18N04
05550	1325 CONTINUE	02665
055E0	1326 IF (NCT3) 9980, 1327, 1328	25 4 15
30DE0	1327 PHIL 903	25415
10.053	GO TO 1400	25 4 75
11.45	132A IF (1.CT2 - 7) 1255, 1329, 1329	0 PMRP
30443	1329 CC 13504 ± 2+ MCT3	08MR8
30NY3	r - مز = r الان	31DE4
055F0	CO = I + A A C = I + A A C C	02665
055E0	$F = (1_{N_{1}}, (1_{N_{1}}) - I_{N_{1}}, (N)) = 1345 + 1330 + 1340$	02FE5
055E0	1330 PRINT 9(7	25MY5
30443	6C T 9999	055F0
3rMY3	CHAREANCE THE CHARG IN ASCENDING ONDER OF STA NUMBER	01 JE 🦻
10JF3	1340 THAV = $T(1)(M^{1})$	02565
30443	$F \in A_{N} = F A \in F (UM1)$	31NE4
055F0	$\psi \in \Lambda_{L} = - L (\cup K)$	310E4
05560	$(n \leq \Delta v) = [n \times ((v + 1))]$	310E4
OSSED	10050V = 1055(UM1)	17560
CSSFC	$I \wedge I \geq (I \wedge I) = I \wedge I \exists (N)$	310E4
BODEO	$\kappa \Delta SE(JMT) = \kappa \Delta SE(M)$	31054
30050	WS(UM1) = WS(N)	JIDEA
BAREO	$h_{AS}(U(1)) = \Gamma AS(K)$	- 31DE4
30150	IUBS(ONI) = IUBS(N)	175E0
300E0	TN[3(N)] = 1NSAV	31064
30060	$\kappa \Delta < F(h) = KS \Delta v$	310F4
3 ODE C	$w \leq (r) = w \leq \Delta v$	31064
30050	$hws(k) = rws\Delta v$	31054
BADED	IUB2(F) = 10824V	17560
30050	1345 CONTINIE	
300 80	1350 CONTRAC	02465
30050	1965 Fr 1980 ' = 1• (CT3	233467
30080	$JS = T(1) \Im(S) + 4$	31014
304F0	$1 + (- + \Delta < \epsilon + (k)) = - 2 - (-1) + 3 + 0 + 13 + 6 + 13 + 70$	
30010		25#15 12050
30DEC	IOP(US) = IOPS(N)	1/520
30050	PRINT 3]], TN13(F), KASE(N), WS(N), JOPS(N)	1/50
300E0	66 10 1380	U2FE5
3 nn E o	1365 TEC KEY (5-1) - 1) 3980+ 1396+ 13/5	02725
01085	1366 JF (KEY(US) - 1) 9980+ 1367+ 1375	02115

1367 PR	INT 312. IN13(N) + KASE(N) + DWS(N) + JOPS(N)	26245
	KEY/(S=1) = 3	02FE5
	A CONTRACTOR AND A CONTRA	05. IF 3
		17650
		17050
		DALLE
		02550
1370	IF C REY(US=11 = 1 2 9980+ 13(1+ 13/5	02005
1371	IF (KEY(JS) - 1) 9980, 1372, 1375	02125
1372 PR	[NT]]]] [N1] (N) + KASE(N) + WS(N) + DWS(N) + DWS(N)	17520
	KEY(JS-1) = 3	02FE5
	KEX(US) # 4	05JE3
	KEY(JS+1) # 5	05JE3
	10P(JS+1) = 10PS(N)	175E0
	top(L) = IOPS(N)	175E0
	TOP CIS+13 # TOPS (K)	175E0
	60 T0 1380	OZFES
1275 00		OZEFS
13/5 44		ORSEA
1 100		02556
1340		10163
	101 10712 4	100000
1400 PH	181 400	UAJE 3
	1F (KEFP4) 998/• 1401• 1410	04 .45
1401	NC14 = 1	SEMME
	0014 = NCC4	19004
	AC TO 1430	04.145
1410 PP	INT 905	04.145
	1 + (NC74 - 1) 1426 + 1411 + 1411 - 1411	265E7
1411	DO 1425 N = 1+ ACT4	26567
	1F(KSw4(K) = 2 + 1413, 1417, 1421	26527
1413 PR	TNT 411. TN14 (N). TN24 (N1. KR24 (N1. FN2 (N). ODN2 (N). 9N2 (N).	04.145
	TN2INI + RN2(N) + CN2(N)	055E0
,	60 Io 1425	0AJA5
1417 00	THT 412. THIA(HI. H824(N). END(N). O(N2(N). 5N2(N). TN2(N).	04.145
1.41.1	Provide a contract of the cont	05566
+		64 145
1.01.00	UN IN ENCO THE AND INDERING ADDRING AND ADDRIG AND A CORDING IN TRACKS	04 145
1421 66	int alls increase any end of the doce of a second of the second	05000
1	REAL THE LACIAL	0.4.1.6.5
1425	CONTINUE	00,045
1426	CONTINUE	615666
PR	IN1 910	UAJAS
	NC14 # +CT4 + 1	21004
	NCT4 = NCT4 + NCD4	19004
1430	IF (NCT4 - 100) 1435, 1435, 1433	04.145
1433 PR	INT 904	04.145
	60 TO 4999	055ED
1435	JF (0784) 9980: 1437: 1440	04 JA5
1437 PR	INT 903	04 JA5
-	60 TO 1500	04 145
1440	KR] = 0	04 JA5
2 0	IFI NCT4 - NC14 1 1500,1445,1445	245E7
1445	BO 1479 5 # NC14+ NCT4	26 SE 7
1440	40 41. TN14 (K) . IN24 (N1. K824 (K). FN2 (4). CONP (N). SN2 (N).	19004
1	TN2(N) + RN2(N) + CN2(N)	055E0
		23064
	econdensity of the companying of the constraints of	

	KR] = K024(N)	06N04
	TF (KSW4 (K) = 2) 1450.1455.1460	265F7
1450	POINT 411. TN1401. 1824(N1. KR24(N). FN2(N). GDN2(N). SN2(N).	04.145
1410	$T_{\rm A} = \frac{1}{2} \left[\frac{1}{1} + \frac{1}{2} \left[\frac{1}{1}$	055E0
		04 (55
		04.145
1455	PRINT #12. INIAINIE REZAINIE ENZINIE QUNZINIE SNEINIE INZINIE	04345
] RN2(N) • CN2(N)	USSED
	00 TO 1470	04JA5
1460	PRINT 413+ 124(N)+ KP24(N)+ FN2(N)+ QDN2(N)+ SN2(N)+ TN2(N)+	04.JA5
	1 9N2(N) + CL2(N)	055E0
1470	CONTINUE	04 JA5
(ATAPUT TANEF B	055E0
1600	DE 15 T 500	055E0
1900	TE (FEEDS) 9980+ 1501- 1510	DESED
10.00	JE COMMENT FOR LIGHT 1910	05550
1201		05550
		05500
	PO 10 1230	V55C0
1510	PRINT YOS	05560
	IF(NCT5 + 1) 1526+ 1511+ 1511	05560
1511	DO 1525 - M ± 1+ KCIS	055E0
	JF(FSW5(F) = 2) 1513, 1517+ 1521	OSSED
2513	PRINT 411+ INIS(K)+ IN25(N)+ KR25(N)+ RHON2(N)+ DIN2(N)+ CEN2(N)	035E0
• • •	60 TO 1525	OSSED
1517	PUTNT 412. TNISINI, KE25(N), PHOND(N), DINZ(N), DENZ(N)	055E0
1311		055E0
160.	$c_{1} = 1022$ $c_{1} = 1022$ $c_{2} = 10022$ $c_{1} = 10022$ $c_{2} = 10022$ $c_{1} = 10022$ $c_{2} = 10022$ $c_{1} = 10022$ $c_{2} $	OBSEC
1521	PHINI HISE CAPSULE APADIME PROPERTY DIRECTLY DERECT	05550
1525		0555.0
1526	C01-11-0F	05500
	PRINT 910	05510
	NC15 # NCT5 + 1	0555.0
	NCTS = KCTS + NCDS	055E0
1530	1F (HCTE - 100) 1535, 1535, 1533	055E0
1573	PRINT 964	05580
	GP 10 9999	055E0
1535	15 (NCCS) 0980. 1537. 1540	055E0
10.00		05 SF 0
12.1		DECED
		AFEFA
1540	KH] F Q	05500
	IF (NCT5 + NC15) IAUD+ 1540; 1540	03500
1545	DO 1570 N = NC15+ NC75	055E0
	READ 51. IN15(N). IN25(N), KR24(N). RHUN2(N). DIN2(N). YEN2(N)	055E 0
	KS45(N = 1 + KP25(N) + 2 + KR1	03580
	KR1 = KR25(N)	05 5 E O
	TF (KSwS(N) + 2) 1557. 1555. 1560	055E0
1664	PRINT 411 . INIS(N) . IN25(N) . KR25(N) . RHON2(N) . DIN2(N) . DEN2(N)	055E0
1 2 2 0		055E0
1665	DET AT 2 TA 5 (N), ROS (N), DUONO (N), DINO (N), DENO (N)	055F0
1022	PATRI TICE INTOTALE RESULTE RECEILE DIRECTLE DERECTLE	ACCE A
		05560
1560	BEINT 413+ INSE(N)+ KHSPIN)+ HHONS(N)+ DINS(N)+ DENS(N)	05560
1570	CONTINUE	055E0
C-+++	-INPUT TABLE 6	055E0
1600	PRINT 600	055E 0
	IF (KEFP6) 998n+ 1601+ 1610	055E Q
1601		055F0
11111	- w _ w	

	055E0	10
	OSSEO	10.
	OSSED	10
1610 PMINT 905	05550	17.
$1 \in \{ (n_1) \in -\} \} = 1 = 2 = 2 = 2 = 2 = 2 = 2 = 2 = 2 = 2$	05550	1.01
$1611 \qquad 100 1025 n = 177(10), 817(10), 817(10), 117(10), $	05550	14
PHINE BED THE REPAIR REPAIR REPAIR FOR THE PHILE PHILE	DESEA	
1625 CUNTINUE	45550 A555A	
1626 CONTINUE	055CU	18)
PRINT 910	05550	
NC14 # NCT6 + 1	V55C0	
NCTE = NCTE + NCDE	01520	187
1630 IF (NCTE = 100) 1635, 1635, 1633	05560	
1633 PRINT 904	VESED	182
CU TU PACA	05520	
1635 IF (NCDE) 9980+ 1637+ 1640	15520	
1637 PRINT 903	05SE0	182
	055E0	18;
1640 IF (NCTE - NC16) 1700, 1645, 1645	055E0	
1645 CC 1470 N # NC16+ NCT6	OSSEC	
HEAD 61+ III(N)+ IT2(N)+ NT1(N)+ KT2(N)+ TT1(N)+ TT2(N)+ TT3(N)	05 5 E 0	
PRINT 510+ TT1(N1+ 1T2(N)+ KT1(N)+ KT2(N)+ TT1(N)+ TT2(N1+ TT2(N)+	OKSEC	183
1676 CCNTINIF	05500	183
CTNPUT TTHLE 7	055F0	
1700 FRINT 700	055E0	18:
IF (KEEP7) 9980+ 1701+ 1710	055E0	18
1701 $NC17 = 1$	055E0	
NCT7 = NCD7	055E0	184
GC TC 1730	055E0	
1710 PRINT 905	055E0	184
[F [NCT7 -] 2]726.]7]], 1711	05560	
1711 DO 1725 $h = 1 \cdot hCT7$	C" SEO	
PRINT 610, T0111), 102(N), K03(N), K02(N), 071(N), 072(N), 073(N)	OFSEO	
1725 CONTINUE	055E0	
1726 CONTTAUE	055E0	
PEINT 910	OSSEO	
NC17 = NCY7 + 1	055E0	
1 CT7 = + CT7 + NCD7	05SE0	
1730 IF (NCT7 - 100) 1735, 1735, 1733	055E0	145
1733 PRINT 904	05560	
GO TO 9999	055E0	
1735 IF (NCF7) 6980+ 1737+ 1740	055E0	185
1737 PRINT 903	05580	
GO TO 1-00	055E0	
1740 IF (NCT7 - NC17) 1800+ 1745, 1745	055E0	196
1745 BO 1770 K = NC17, NCT7	055E0	
PEAD 61, 101(A) . 102(A) . KO1(A) . KO2(A) . GT1(A) . GT2(A) . GT3(A)	055E0	18
PEINT 610+ TG1(h)+ 1G2(N)+ KG1(N)+ KG2(A)+ GT1(N)+ GT2(N)+ GT3(A)	055E0	
1770 CONTINUE	055E0	
CHART TABLE P	055E0	18
1800 PRINT 800	24060	181
1F (KEEPP) 9980+ 1801+ 1819	24080	c
	240E0	
NCVB = KCC8 / 3	240E0	
60 TO 1830	24DE0	
and the second		

055E0		24050
05550	$\frac{1}{10} = \frac{1}{10} $	24050
05660		24550
02560	1811 CC THAP V = 1. SCAS	
05580	IE (K2%5() + 5) 1613* 161(* 1851	24(10)
055E0	1A13 PETAT RIL, INTROXINGALATARA (N), KEPALN), AMPLAN, WAPLAN, APPCLAN,	240E0
055Er	1 KSYMINI WATOLINI CATOLINI	24 ° E 0
05550	CC TO 1934	240F0
65°55		24050
05520	1817 PRINT 812+ INIH(K)+ RE28(b)+ UMPINI+ WMPINI+ RPOCINIE ROIMINI	
05560	1 WKTCL(N) + GNIOL(N)	15/11
05SE0	6C TA 1824	24760
05550	1021 PRINT 813. TN2P(N). KF28(N). QMP(N). WMP(N). NPOC(N). KEYM(N).	24050
ASSEA	TOUCHIA GNIOL(N)	LSFEL
0. 100		24050
VESED		31054
055E0	$PPINT PI4 \leftarrow GP(E_{F}NP) \leftarrow NP = 1 \leftarrow NPC(1)$	24020
J55E0	PRINT BIS. (WP(NAMP), NP = 1 , NP(T)	24020
645E0	1825 CONTINUE	24DE0
DECEA	1936 CONTINUE	24020
05520		24050
05520	PHINT STU	24050
USSEC	WCIP = NCV8 + 1	
05 4 E 0	NCVA = KCVB + NCDA / 3	ZAUEO
OKSEC	1830 IF (NOVA = 20) 1835, 1835, 1833	24050
05556	1933 PRINT 904	24050
05550	60 TO 9999	24050
03460		24DED
05560	THER IN CODE) AAEVA TOULA TOAD	24060
055E0	1837 PRINT 903	ENDED
055E0	GO TO 1873	15FE1
055E0	1840 KP1 # 0	24060
CSSEO	TE (NOVA - NO18) 1873+ 1845+ 1845	24068
05660	1945 DO 1870 N # NCIE. NCV8	24DE0
0.00	DEAD OF THIS AND THIS AND	24050
UT SEO	DEBUT MIE I IERCEA INFORMATION CONTRACTOR AND CONTRACTOR A	24050
CESED	I KSTATUS WITCHNE ANTOLONI	LUEU
DESEO	RPCT = KPOC(N)	20120
055E0	$PFAn = R2 + (CP(N+P) + NP = J + NPC_{1})$	CODED
055E0	PEAD P2, (WP(N,KP), NP # 1, NPCT)	ZADEO
DESED	KSWP(N) = 1 + KP2P(N) + 2 + KP1	240E0
OFFE	KP1 = K028(N)	240E0
01220	r = r = r = 2 105A, 1055, 1060	24050
05520	[P + P = P = P = P = P = P = P = P = P =	24.05.4
05560	1850 PRIMI SIL (KIH(N) - INSEN) + RP20(N) - WP (N) + MOUTH FOR (N) +	2.25.
05560] KSYM(K) + WKTOL(N) + QNIOL(N)	
055E0	GO TO 1871	24050
05550	1855 PRINT 812, IN18(K), KR28(N), OMP(N), WMP(N), NPOC(N), KSYM(N),	ZADEO
05550	1 WATGL(N) + QATOL(N)	15FE1
05560		240F0
0550	CO TO TELL BOOKS AND	24DE0
OSSEO	1960 PHINE BI3+ LOZBINIA REPORTATIONAL REPORTATION CONTATES	
055E0	1 WATOL (N) + GATOL (N)	
05560	1871 NPCT # KPOC(K)	C41120
055E0	PRINT 814, ($QP(x_{+}NP) + NP \neq 1 + NPCT$)	24DE0
05550	PRINT 815. ($P(K + NP) + NP = 1 + NPCT$)	24050
		240F0
05560		IEFEI
24RE0	1873 IF CNCVF (EG. 0) GO ID 1900	40rc.
24080	CH-H-BISTAINUTE SUPPORT CURVES	24020
24CE0	CO 1975 N # 1. NOV8	24050
740F0	NPCT = KPOC(K)	24050
24050	DO 1880 NP $= 3 + 5PCT$	24DE0
	DO 1 HO HY A 14 HIGH	

	ODVIN-NOT # CP(NAND) # UMP(N)	240E0
	WOUTS NOT = WOUNTNEY = 0.0000000000000000000000000000000000	24050
1000		24DE0
1000	TE (HECT - 1 1 1885, 1896, 1895	24050
1005 00		240F0
1000 04	60 TO 9000	24050
C====CH	FCK POINTS FCR PROFFR ORDER	24DE0
1905	PO 1896 AP = 2 APCT	24050
163,	1F (WPV(N+NE) + (F+ WPV(N+NP+1)) GO TO 1885	IOFEL
	TE 1 CPU(N.NP) .GT. OPU(A.NP+1) 1 GO TO 1985	10FE1
1996	CONTINUE	240En
1.1.23		24020
1895	IF I KSYM(N) ANE I I GO TO LARS	240E0
1001	TE (KSYN(N) .NE. 1) 60 TO 1892	ZADEO
(PLY STAMFTRY OPTION TO CONSTRUCT OTHER HALF OF SUPPORT CURVE	24050
u –	NP1 = NPOC(N) + 1	24DF0
	NP2 # NPOC(A) • 2	240E0
	00 1693 P = PT+ NP2	240E0
	NK = NP + NPCT	24080
	(PV(N,NP) = (PV(N,NN))	240E0
	$WPV(N \cdot NP) = WPV(N \cdot NN)$	24DF0
1893	CONTINUE	24060
	DO 1894 NP # 19 KPCT	240E0
	NN = NP2 + 1 - NP	24050
	$OPV(N, HP) \approx -OPV(N+NN)$	240E0
	HPU(1.+11P) = - HPV(1.+11)	24060
1894	CONTINUE	240F0
	NPCISINI # NP2	24060
	GO TO 1875	240E0
1892	NPCTS(N) # NPOC(N)	2+0E0
1875	CONTINUE	240E0
C	NEPATE ALL THE SUPPORTING STA NONLINEAR CURVES	15FE1
	× λ,ζ = 0	15FE)
	KR1 = 0	28050
	DC 6965 N = 1+ NCVB	20060
	TE (KR1 +61, 0) OU TO 6066	ZANEO
	5.1 ± 5.	20100
	IF (KP2F(N) .FQ. 1) 60 TC 6070	2868.0
6066	J1 = IN18(N1) + 4	ZADEO
	J2 = [N28(N) + 4	2ADE0
	1U - SU - MON30	ZADEO
	JEND = J2	20DE0
	1E (DEMAM .07. 0) 60 TO 6067	2PCE0
	IF (MP4 .CF, J1) ON TO 6068	24060
	GC TO 6075	26050
6067	IF (MP4 .1E. J1) GO TO 6075	ZADEO
	IF (MP4 .GE, J2) ON TO A069	ZPDEO
	JEND = MP4	2PDE0
	GO 10 6n69	2POE0
6068	DENCM = 1.0	ZEDEO
6069	DO 6080 $J = JI + JEND$	ISFE1
	IF (N1 "EQ. 1) GO TO FOAP	15FE1
	IF (KR) .EQ. 0) GO TO 60-22	26J91
	IF (KP28(1-2) .EQ. 0) GO IC 6082	26301

	[F (JSTA (NNC) .NE. J) GO TU 6082	2630
	j5w(J) = ∩	IEFE
	0 TA 5080	15FF
6082	(1FF = J - J1	15FE
	PALT = DIFF / DFNOM	1SFE:
	fNC = NFC + 3	1SEE:
	IF I KSYN(NI) .NF. KSYM(N)) GO TO_1885	15FE
	IF (NPCTS(N)) .NE, NPCTS(N)) GD TO 1885	15FE
	τε (J *Ed* 7) *OB* η *Ed* ηEMD) IZM(η) # J	15FE
	1F (J1 - JEND) ER3+ 682+ 681	15FE
681	PRINT 9993	15FE
	60 TO 9499	15FE
692	IS+(-) = 0	15FE
693	LPCT = KPCTS(K1)	15FF:
	$PC + AP3 \rightarrow P = 1 + NPCT$	JEEE
	ΥF (WPV(N1+NP) +NF+ HPV(N+NP)) GO TO 1885	15FE
	HPT(hNC,NP) = HPV(N1,NP)	15FE
	QPT(+NC+NP) = QPV(N1+KP) + PART+ (QPV(N+NP) + GPV(N1+NP)	15FE
)	15FE
6083	CONTINUE	15FE
	15 (STOL 11)	51 Vb
	NNYALUHNE) = WATEL (N1)	2100
	ONTOLUTINO = UNTOLINI + PART + CONTOLINI - ONTOLINI -	15FF
	NPTTS(INC) # NFCTS(N1)	15FE
	ISTA(Not) = U	15FE
	KSYHU (NKC) # KSYH(N1)	15FE
	IF (ARS(WPT(NNC+1)) +NE, ARS(WPT(NNC+NPCT))) GO TU 1885	15FE
6000	CONTINUE	15FE
6075	1F (KA28(N) .NE. 1) GO TO 8070	2PDE
	$T \in \{n\}$ = $I \in \{n\}$	SUDE
	h = h	2PDE
6070	IF (FR2P(F) .FQ.]) GO TA 6095	SUDE
	KB1 = 0	SUDE
	GO TO 5045	SUDE
6095	K¤] = 1	280E
6045	CONTINUE	SADE
	TE CARE (1, 10C) 50 TO 1014	15FE
	PRINT 904	15FE
	60 TO 9999	15FE
C	-SAVE SLOPES, COPARS, WORDES FOR EACH CURVE	15FE
1874	DO 1876 A # 1. KAC	15FF
	NPCT # KPTTS(N)	15FE
	IF (NPCT - 2) 1845, 1877, 1878	15FE
1877	JE (KSYMU(N) -NE. 1) GO TO 1885	15FE
1878	JF (KSYMJ(1.)	15FE
	NOS # NECT / 2	15FE
	CO 1897 NP = 1+ NOS	15FE
	NN = NOS + NP + 1	15FE
	QPRCP(N,NP) = - (QPT(N+NN+1) - QPT(N,NN))	15FE
	WOROP(N:NP) = WPT(N:NN+1) - WPT(N:NN)	15FE
	$SLOPE(N*NP) = QOHOP(N*NP) \neq WDROP(N*NP)$	10FE
	1F (NR ,EQ,]) 60 TO 1897	LOFE
	$QDAOP(N_*NP) = QDROP(N_*NP) + 2 \cdot 0$	10FE
	$\# \Box \oplus \Box \oplus \Box = \# \Box \oplus \Box \oplus \Box \oplus (N + N^{2}) = 2 + 0$	LOFE

1007	CONTINUE	15661
10.24	NOS(N) = NCS	16FE1
	60 TO 1996	Z1AP1
1070		10FE1
1017		16551
	PO - 3 R G = 1 + NPC T I	10551
	$1 \in \{ABC(WPT(N,BC+1)\} = WPT(N+NP)\}$.NF. (ABS(WPT(N+NP+1)))	10551
	ABCTWRT(N. NPA)) CO TO 1898	1SEE 1
1		10FE1
		1PEF1
1000		1/FF1
1946	TELENDENT AF 3 1 GO TO 1885	16FF1
		10561
		15551
		165F1
	$\frac{1}{2} \left(\frac{1}{2} + 1$	15551
	WOODE (N. NO) - WEINLANDSHNPAIL - WEI(A.NOSHNP)	15FF1
	$H(M) = \{(i_1, j_1) \in \mathcal{A} = \{(i_1, j_2) \in \mathcal{A} : i_1 \in \mathcal{A} : j_2 \in \mathcal{A} : i_1 \in \mathcal{A} : j_2 \in $	15FF1
	STIRE IN THE A REPORT OF THE ADDRESS	10551
1633		15FF1
	NUCLINE HUDE The openning is a armae i go to 1864	SEFT
		15441
	S_{1} (PF (N_{NO}) = VAU HODOR(N_{NO}) = WOT(N_{NO}S_{1}) = WPT(N_{NO}S)	15661
	WOPET(N,NO3) = PT(A,NO5)T = PT(A,NO5)	ISFET
	QUEDTINANUSI = VIIO	IRFEI
	NORGINI = NOS	21421
	-60.19 THMC which the second at second s	51) ÎSFEÎ
1464	THE BOADED BOADED OF STARTS FOR THIS NONLINFAR SUP CURVE	214P1
C-seerce	FUR THE PEDEER THOSE OF SECTION OF THE THE THE THE THE	21 AP1
1860		21401
	TE 1 STORE (LANA) LE. J. OF-OD J GO TE 1887	ZIAPI
	$r = (-s_1 \circ \rho r (n_1 \circ n_N) + s_1 \circ r_1 \circ (\rho r (n_1 \circ N_N \circ 1))) = (s_1 \circ \rho r (n_1 \circ n_N) + s_1 \circ (r_1 \circ n_N))$	21401
	$\frac{1}{16} = \frac{1}{16} \sum_{i=1}^{n} \frac{1}{16} \sum_{i=1}^$	214P1
		21491
		21491
1007		21401
(HHI)	LONGING PROLE OF POINTS ON PUSITIVE ONE-WAY SUPPORTS	26.01
(THE THE COLOR OF CLOSE CONTROL TO THE	26 (1)1
		26.301
	UU Iran and the second	24.001
	APP = APP	26.001
	wowt(N, k P G) = - wot(N, N F)	24.001
		24301
JHHH	CONTROL ND A LABOR	26.001
	$\{(1, 1, 0, 0, 0, 0, 0, 0, 0, 0, 0, 0, 0, 0, 0,$	24,001
		26.001
1000		26.111
1884	CONTINUE	15FE1
1878	DHT TARLE O	30060
1000 00		30DE0
1400 00	1F (KEEP9) 9980+ 1901+ 1910	3nDE0
1001	NC19 = 1	3 OCF 0
1-101	NCTS = KCPY	30060
	0 TO 1910	3rdf0
	N I BIRD	

1010	DE 1 1 905	30050
	1F () CTG = 1 3 1526, 1911, 1911	3nDEC
1011	CG 1925 N # 1. NET9	BODEC
7.411	DOTET ONE XAXISINI YAXTSINI TYSWINI MPSINI	10FE1
1076	CONTINUE	3nnE(
1970		3 nDEC
1		3ADE C
	-1 -1 -1 -1 -1 -1 -1 -1	30DEC
		30DEC
1030	TE E NOTO - 10 1 1935, 1933	05.JA1
1011		30DEC
14.4		30DEC
1 G TE	1 F (FCP9) 2980. 1937. 1940	3 nDEC
1037		30PEr
1427		JODEC
1040	$T_{\rm E} = (1, 2, 7, 7, 7, 7, 7, 7, 7, 7, 7, 7, 7, 7, 7,$	3nDEC
1046	p_{1} (key) = key = k	30DEC
1440	$D(x_1,y_1,y_2,y_1,y_2,y_1,y_1,y_2,y_1,y_1,y_1,y_1,y_1,y_1,y_1,y_1,y_1,y_1$	LOFET
	$\frac{1}{10} \frac{1}{10} \frac$	LOFET
1074	PETRO 2007 AF 12100 AF 1200 AF	30DEC
C	TABLE S	OSSEC
2000		30DEC
¢ 0 0 0		10000
	CALL INTERPORT (MATE INTER INTER INCL. INTER INC.	10000
	$\begin{bmatrix} L \\ L \end{bmatrix} \begin{bmatrix} 1 \\ 1 \\ 1 \\ 1 \\ 1 \\ 1 \\ 1 \\ 1 \\ 1 \\ 1$	10000
	CALL INTERPOLATION (NOT, RETAIL INTAL INDAL KADA, RED. R. 1544 (SWA)	10000
	CALL INTERPORT A MOTE ACTAVITATION ACTAVITATION CONTRACTOR ACTAVITATION CONTRACTOR	lonce
	CALL INTERPORT (NOT - INTE INTE INTE KR25, RHONZ: RHO. LSMIKSWE)	1000
	CALL TATEDOS (NOT. ACTS, INIS, IN2S, KE25, DIN2; DI, (SM: KSW5)	10000
	CALL INTERDA (WOA, NOTS, INIS, IN2S, KR25, UEN2, DE, IAM, KSW5)	10000
	(ALL INTERVIEW AND A	05SEC
	CALL INTERPS (ND7. NCT4. IN14. IN24. KR24. TN24. T. ISH4 KSW4.)	10000
c	CHE THEAMACHTONE SCIETION FOR FACH TIME STATION	DESEC
		OSSEC
		05SEC
	$\frac{P_{\text{PLN}}}{D_{\text{PLN}}} = \left(\frac{1}{2} + \frac{1}{2} + \frac{1}{2} + \frac{1}{2} + \frac{1}{2} \right)$	05SEC
	$\frac{1}{100} \frac{1}{100} \frac{1}$	OSSEC
		OSSEC
		05SE0
		055E0
		15FE1
·		055E0
C	$- \int c A d d d T = 1 + v P T$	30nEC
	TF(1S(1) = 0	15FE1
		25 JA1
		15FE1
		21401
		SIAPI
	1.1.4.1.1.4.4.4.4.4.4.4.4.4.4.4.4.4.4.4	21 AP1
L007		300F
0447	CONTINUE CO 7000 X H 1. MTP4	INFEI
		154P1
		OFAPI
		ISPET
	$\mathbf{f}_{\mathbf{x}}(\mathbf{x}) = \mathbf{x}$	

		25 (A1		06491
	TTCSP - A	25.141	$\begin{array}{cccccccccccccccccccccccccccccccccccc$	16 41
	KCICSE(1) # 0	0FAP1		16.141
	$Kr(OSF(2) \approx 0$	CAPI	60 TO 748	16.41
	KPC = KPC + 1	DESFO	746 FST = 0.5	16.41
	KK = KK + 1	OSCEO	748 1F 1 1 11 . T1 .CP. J .GTT2 1 00 TO 707	USSFO
	KHI = K -]	OFSEO		10000
	KM3 = K - 2	LOFE 1	IF (KN2 + K+1) 740. 741. 742	055E0
	KM3 = K + 3	DESEC	740 PPINT 917	OSSEO
	KM4 = K = 4	05 CE O		04820
C1N11	TIALIZING	345F0	7.1 TDF* C* = 1.	055E0
	CC 6998 T = 1 + MP7	300E0	FST = 1.0	[ه ل ۲ ا
	IF (K .CT. 7) CC TO 6996	04401	BC 70 743	055E0
	4 (T.KK) = C.C	30050	742 TOFN = $HN2 = KN1$	65560
	$WW(1) \simeq 0 e^{0}$	26020	$\frac{1}{1}$	05560
	wwwsfl = 0	13_41	AIKC = A1/CT + 0.5	¢5SED
6996	IF (K .LE. >) GC TO 6995	OF AP 1	ADENAM # JT2 # JT1 # 1	DESED
	M(1*KK) = M(1*KK+))	06AP1	₹T(J+KT) = TT(J+KT) + { TT}(NC) + { TT2(NC) - [TT(NC) -	14041
	AM(1) = M(1*RK)	06491	3 • RINC / HDENOM • (TT3(NC) + TT3(NP) 3 • T	INC LEUAL
6995	S5KH2(T) = S5KM1(T)	51 4 P 1	2 / TDENOM) + EST	16141
	SSKE1(1) = 55(1)	15FF1	797 CONTINUE	OSSEO
	$a_{1+2}(1) = O(M)(1)$	21001	CHARTER CALCULATE TIME VERTART LATERAL EVADING FOR EACH STA AT THIS TIME	STASED
	OIMJ(I) = OI(I)	21401	C ANT K-2 TINE 5+A	USSEO
	TOSW(I) = 0	13541	711 IF (1 + 6T + 5CT 7) GO TO 710	10000
	NUMOC(I) = 0	13JA1	00 712 NC # 1+ NCT7	05560
	Inc(1) = 0	24 JA1	KNJ = KUJ(CL) + 3	10000
	TF (K .EC. 1) FC TO 6998	21491	KN2 = K02(NC) + 3	10000
	$O_{2}(1) = O_{2}(1) - O(1) = O_{1}(1+1)$	21 AP 1	371 = 101(NC) + 4	05520
1.958	C(0.7 (1))#	JOFFO	JTP = JrP(r,r) + 4	VESLO
	IF (1044 - K) 7002, 7003, 7003	15#E1	JE LKNP .GE. KNJ) GU TO 646	04401
7002 PEIN	T 940	UESEO	Print 917	04401
		05<60		OKAP I
1003	IF (K FR, FRF4) ON 19 6170	10*61	P46 1F ()17 ()27 ()26 (11) 10 (11 H4)	04401
		UFJA1		06661
		10000		04401
		10040	$\begin{array}{cccccccccccccccccccccccccccccccccccc$	14 11
7441	GILLERIT = GAR	OBEEC	CALE AL ALL FOLD FOR AL FURE FOR A DALLA TAR	14 141
CanadaCAL C	THEATE TIME VIRTANT ANTIN TUDUST FAR FACH BID AT THIS.	TTHE STA DESEN	$e_{31} = 1.0$	16 (41
	THE THE CLASS PRODUCT PRODUCTION CALL AND AN ANY ANY ANY ANY ANY ANY ANY ANY ANY	05550		16 161
Ç K µ.	K) = K = (KT = 1) \$ 2	10000	756 TE () TI .CP. J .GT. JT2) 60 TO 712	OSSEO
	PO 710 (= 3. MPE	10000	TE () FR. TI FP. J FR. JT2 3 GO TO 760	16.161
	TE (1 .CT. +CT6) GC TP 711	OSSFO	FSP = 1.0	16341
	PD 707 AE = 1 + FCT6	955F0	67 T 761	16.JA1
	$K_{N1} = KT1(NC) + 3$	10000	760 658 = 0.5	14 JA1
	KK2 = KT2(NC) + 3	10000	$761 \qquad TINC = KI - KNI$	10000
	JT1 = [T] (NC) + 5	05550	1F (KH2 - KN) + 750. 751. 752	055E0
	$JT_{2} = 1T_{2}(NC) + 4$	0= 5E0	750 PEINT 317	055F0
	IF (JT7 - JT1) 144. 745. 745	055E0	CO TO SAGE	055E0
744 PRTH	T 960	055E0	753 TDFA/M = 3+6	OSSEO
	60 10 9099	05480	FST = 1.0	16341
745	TE (KER .GE, KN) BO TO 845	OAAP1	Gr 10 753	USSED
PRIA	3 917	06AP1	752 TOPACH = $KL2 - KL1$	05520
	60 TO 9999	04491	753 HICE - 11	OSSEO

•

754 PPINT 917 05560 755 RDFRCM = 1+0 05560 756 RDFNCM = JT2 - JT1 05510 757 RTU+KT1 = 0T(JKT) + (CT1(NC) + (CT2(NC) - WT1(NC)) * TNC 16JA1 757 RTU+KT1 = 0T(JKT) + (CT1(NC) + (CT2(NC) - WT1(NC)) * TNC 16JA1 757 RTU+KT1 = 0T(JKT) + (CT1(NC) + (CT2(NC) - WT1(NC)) * TNC 16JA1 710 CONTINUF PINC BDENOM + UT2 - ST * ESB 16JA1 712 CONTINUF SSSC0 SSSC0 713 CONTINUF SSSC0 SSSC0 6999 IF (ITSV : A900, 6905, 6900 2PDE0 2PDE0 6991 IF (ITSV : A900, 6905, 6900 2PDE0 2PDE0 6991 CONTINUF 10 28DE0 2PDE0 751 RUMART = 1 28DE0 2PDE0 2PDE0 751 SS(J) = S(J) SS(J) = S(J) 2PDE0 2PDE0 751 IF (INCYB : EG, O.) GO TO 6554 2PDE0 2PDE0 751 IF (INCYB : EG, O.) GO TO 6557 06AP1 2PDE0 751 IF (INCYB : EG, O.) GO TO 6057 06AP1 10FE1 752 EGN		IF { JT2 - JT1 } 754, 755, 756	055E0
0.5.5.6 0.755 0.755 0.755 0.755 755 0.757 0.757 0.757 0.757 756 0.757 0.71.3.4.71 0.71.1.4.71 0.71.1.4.71 0.71.1.4.71 757 0.71.1.4.71 0.71.1.4.71 0.71.1.4.71 0.71.1.4.71 0.71.1.4.71 757 0.71.1.4.71 0.71.1.4.71 0.71.1.4.71 0.71.1.4.71 0.71.1.4.71 710 CONTINUE 0.71.1.4.71 0.71.1.4.71 0.71.1.4.71 0.71.1.4.71 710 CONTINUE 0.70.70.4.71.4.71 0.71.1.4.71 0.71.1.4.71 0.71.1.4.71 6999 CONTINUE 0.70.70.4.71.4.71 0.70.70.4.71.4.71 24000 6900 IF (INT ATT = 1.4.4.71 24000 24000 6910 C0.6000 ATT = 1.4.4.71 2	754 PPI	NT 917	055E0
755 ADFACM = 1.0 0555C 60 T0 757 G57 756 ADFACM = JT2 - JT1 757 ATLY*TT = GTL(JKT) + (GT1(AC) + (GT2(AC) - WT1(AC)) * (LGJ) 1 FINC 757 ATLY*TT = GTL(JKT) + (GT1(AC) + (GT2(AC) - WT1(AC)) * TIAC 1 FINC 710 CONTINUF 712 CONTINUF 713 CONTINUF 714 CONTINUF 715 GONG (GATTAUF 712 CONTINUF 713 CONTINUF 714 CONTINUF 715 CONTINUF 716 GATATAT 717 GATAT 718 CONTINUF 719 CONTINUF 710 CONTINUF 711 CONTINUF 712 CONTINUF 713 CONTINUF 714 CONTINUF 715 GONG TO TOTO 714 CONTINUF 715 GONG TO CONTAUF 711 CONTO	134	60 TO 9999	055E0
$ \begin{array}{cccccccccccccccccccccccccccccccccccc$	755	ADENCM = 1+0	055E0
G0 TO TY 055C0 756 RDFNOW = JTZ - JTJ 055C0 757 OTLUKTI = QTLUKTJ + (GTI(NC) + (GT2(NC) - UTI(NC)) * TINC 16JA1 1 RINC - BDENOW + (UT3(NC) - OTI(NC)) * TINC 16JA1 712 CONTINUE 055C0 713 CONTINUE 055C0 6999 CONTINUE 055C0 6999 CONTINUE 055C0 6990 F(ITSW) A900, A905, A900 2PDE0 6900 FF (ITSW) A900, A905, A900 2PDE0 6900 FF (ITSW) A900, A905, A900 2PDE0 6900 FF (ITSW) A900, A905, A900 2PDE0 6900 FF (ITSW) A900, A905, A900 2PDE0 6900 FF (ITSW) A900, A905, A900 2PDE0 CITSW = 0 + LINEAR SPRINGS, SO SFT MAXIT = 1 2BDE0 6910 CO 6000 A1T = 1, MAXIT 2BDE0 CINITIALIZING FCH NEAT ITFRATION 2PDE0 01(J) = Q(J) + OT(J+1) 21API SSIJ) = S(J) 2PDE0 CINITIALIZING FCH NEAT ITFRATION 2PDE0 NPIS = 0	140	FSB = 1.0	16101
756 RDFNOW = JT2 - JT1 055E0 757 GT(J+KT) = GT(J+KT) + (GT1(NC) + (GT2(NC) - UT1(NC)) * (LA) 1 RINC. BDEKOM + (UT3(NC) - UT1(NC)) * TINC 2 / TDFNOM) * EST * E5B 055E0 710 CONTINUF 055E0 711 CONTINUF 055E0 712 CONTINUF 055E0 713 CONTINUF 055E0 714 CITSW : A900. 4905. 6900 280E0 6909 CONTINUF 160C0 711 GT(J+KT) = 0 280E0 2000 IF (HAXIT = 1. 280E0 2001 C0 6000 ATT = 1. MAXIT 280E0 CTINTTALIZING FGR NEXT ITFDATION 280E0 6005 MAXIT = 1. 280E0 CONTINUE CONTINUE 214P1 S1J = 5(J) S1J = 0 214P1 CONTINUE S1J = 0 214P1 S1J = 5(J) S0 FT MAXIT = 1. 280E0 6500 CONTINUE 214P1 S1STA(NN) 15FE1 214P1 NDC = NDSJ(KN) 15FE1 24DE0 Conos5 N = 1. NCC		60 10 757	05580
1 1	756		055E0
1 PINC. BDEKOM. (073(KC) = 071(KC)) * TIKC 16JA1 2 7 TOFNOM) * EST * ESB 16JA1 712 CONTINUF 05500 710 CONTINUF 05500 6999 CONTINUF 0000 6990 If (HAXIT *CT* 0) GO TO 6910 20000 PRINT 9982 20000 20000 C115W * 0* LIKEAR SPRINGS, SO SFT MAXIT = 1 20000 6900 CO 6000 AIT = 1, MAXIT 20000 6910 CO 6000 AIT = 1, MAXIT 20000 CINITIALIZING FOR NEXT ITEPATION 20000 CO 6500 J = NF7 20000 CO 6005 SC = NOSJ(N) 21401 SC 00 000 J = 15XA(N) 21401 ND = SC 00 SC 00 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	757	$\sigma_{T}(1+KT) = \sigma_{T}(1+KT) + (\sigma_{T}(NC)) + (\sigma_{T}(NC) + WT(NC))$	•16JA1
712 CONTINUE 713 CONTINUE 713 CONTINUE 0550 6999 CONTINUE 0550 6999 CONTINUE 0550 6900 IF (ITSW) A 900, 4905, 6900 2PDE0 6900 IF (ITSW) A 900, 4905, 6900 2PDE0 6900 IF (ITSW) A 900, 4905, 6900 2PDE0 6900 IF (ITSW) A 900, 4905, 50 SFT MAXIT = 1 2PDE0 6900 MAXIT = 1 2PDE0 6900 NATT = 1, MAXIT 2PDE0 6900 NATT = 1 2PDE0 6900 STATATION 2PDE0 6900 STATATION 2PDE0 00 6500 J = NTATON 2PDE0 00 STATATION 2PDE0 01015 STATATION 2PDE0 01015 STATATION 2PDE0 01015 STATATION 2PDE0	(²)	RINC , BDENOM + (QT3(NC) - QT1(NC)) + TINC	16JA1
712 CONTINUE 0550 710 CONTINUE 0550 6999 (CATINUE 10000 6990 IF (ITSW : A900, A905, A900 2PDE0 6900 IF (ITSW : A900, A905, G910 2PDE0 PRINT 9982 2PDE0 2PDE0 6901 C0 6000 NT = 1, MAXIT = 1 2PDE0 6905 MAXIT = 1 2PDE0 2PDE0 6906 C0 6000 NT = 1, MAXIT 2PDE0 6910 C0 6000 NT = 1, MAXIT 2PDE0 6911 C0 6500 J = NT PAT 2PDE0 CINITIALIZING FCR NEXT ITFRATION 2PDE0 2PDE0 CINITIALIZING FCR NEXT ITFRATION 2PDE0 SIUJ = SUJ SUJ = 2DDE0 2PDE0 6500 J = NTC 2PDE0 IF (INCWB +C0.0) GO TO 6504 2PDE0 CINITIALIZING FCR NEXT ITFRATION 2PDE0 NPCT = NPTTS(NN) 1BFE1 NDC = NOSU(N) 1BFE1 NDC = NOSU(N) 1BFE1 NDC = NOSU(N) 1BFE1 SST IF (INSUJ, E0.0) GO TO 6057 06AP1 <td></td> <td>2 TDENOM 1 # 557 # 558</td> <td>16.141</td>		2 TDENOM 1 # 557 # 558	16.141
110 CONTINUE 055E0 6999 CONTINUE 100C0 111 CTSW : A900. 6905. 6900 2PDE0 6900 IF (ITSW : A900. 6905. 6900 2PDE0 0 PRINT 997 2PDE0 CITSW = 0. LINEAR SPRINGS. SO SFT MAXIT = 1 2PDE0 6900 NAXIT = 1 2PDE0 69010 C0 6000 AIT = 1. MAXIT 2PDE0 0 OC 6000 AIT = 1. MAXIT 2PDE0 0 GC 6000 AIT = 1. MAXIT 2PDE0 0 CONTIAUE 2PDE0 0 GOTALLIZINO 2PDE0 0 GOTA 6504 2PDE0 0 DC 6005 ND 1. FO. ND 1. 0 IF (INCVM .EQ. 0.) GO TO 6557 06APD 0 ND 1. FO. ND 1. 15FE1 ND 1. FO. ND 1. FO. ND 1. 15FE1 0 ST 1. ND. 1. ST 1.	*12		055F0
110 CONTINUE 100C0 6999 CONTINUE 2000 print 99R2 2000 2000 print 99R2 2000 c1TSW = 0+ LIKEAR SPRINGS, SO SFT MAXIT = 1 2000 6900 MAXIT = 1 2000 6910 C0 6000 NIT = 1, MAXIT 2000 6910 C0 6000 NIT = 1, MAXIT 2000 6911 C0 6000 J = 1, MAXIT 2000 6912 C0 6000 J = 1, MAXIT 2000 CINITIALIZING FOR NEXT ITFRATION 2000 2000 CINITIALIZING FOR NEXT ITFRATION 2000 21, 2000 CINITIALIZING FOR NEXT ITFRATION 21000 21, 2000 CC (DNTINUE 2000 2000 2000 IF (NCWB +EG. 0) GO TO 6504 20000 2000 CC (DNTINUE 2000 2000 2000 IF (INSWU) *EG. 0 1 GO TO 6557 064P1 NPCT = NPTTSINN 19FE1 19FE1 NPCT = NPTTSINN 19FE1 19FE1 Strest = NOSU(NK) 100 TO 6085 19FE1 GO TO 6085 19FE1	716		055E0
B999 CONTINUE 2000 IF (ITSW) A900, A905, A900 2000 PRINT 9982 28000 CTISW # 0, LIAEAR SPRINGS, SO SFT MAXIT = 1 28000 6905 MAXIT = 1 28000 6906 MAXIT = 1 28000 6907 MAXIT = 1 28000 6908 MAXIT = 1 28000 6909 MAXIT = 1 28000 6900 CINITALIZING FOR NEXT ITENATION 28000 CINITALIZING FOR NEXT ITENATION 28000 00 6500 J = 1+ MPT 01(J) = 0(J) + 0T(J+1) 21AP1 01(J) = 0TTSTANN 1	/10		10000
6900 IF (HARIT ACT 0) GO TO 6510 2000 PRINT 9982 28000 6900 IF (HARIT ACT 0) GO TO 6510 28000 6900 MARIT 1 28000 6900 MARIT 1 28000 6900 MARIT 1 28000 6910 C0 6000 AIT = 1. MARIT 28000 6910 C0 6000 J = 1. MARIT 28000 CINITIALIZING FOR NEXT ITERATION 28000 CINITIALIZING FOR NEXT ITERATION 28000 CINITIALIZING FOR NEXT ITERATION 28000 CINITAGE 28000 CINITAGE 28000 CINITAGE 28000 CINITAGE 28000 CINITAGE 28000 CINITAGE 28000 NDS = NOSJONN 19FE1 NDS = NOSJONN 19FE1 <t< td=""><td>6444</td><td>CONTINUE TE (TTEN) 2000- 4905, 2000</td><td>ZPDEO</td></t<>	6444	CONTINUE TE (TTEN) 2000- 4905, 2000	ZPDEO
E910 19 1 MAX11 (11: 0) NO 10 910 2010 PRINT 99P2 2010 2010 PRINT 99P2 0: LINEAR SPRINGS, SO SFT MAXIT = 1 2000 6005 MAXIT = 1 2000 6005 MAXIT = 1 2000 6006 NO 11 = 1, MAX1 2000 6007 MAXIT = 1 2000 6008 NO 6500 J = 1. MP7 2000 0100 6500 J = 1. MP7 2000 0101 = 0(J) + 0T(J+1) 21AP1 9010 CONTINUE 2000 6510 CONTINUE 2000 010 + 000 S(J) = S(J) 2000 011 + 000 S(J) + 000 S(J) + 000 011 + 000 S(J) + 000 S(J) + 000 011 + 100 S(J) + 000 S(J) + 000 01 + 100		$\frac{1}{10} + \frac{1}{10} $	28050
Definition Definition Definition 6905 MAXIT = 1 24050 6910 CC 6000 AIT = 1, MAXIT 24050 6910 CC 6000 AIT = 1, MAXIT 24050 CINITIALIZING FCH NEXT IFFNATION 24050 CC 6050 J = 1, MPT 24050 CC 7011NUE 24050 24050 CC 6056 NN = 1, ANC 24050 DC 6050 N = 1, ANC 24050 NP15 = 0 J = 1574(NN) 15FE1 NDCT = NPTTS(NN) 15FE1 10FE1 NDCT = NPTTS(NN) 15FE1 06AP1 CINITIALL/ING FOR THE FIRST LOAD ITERATION 06AP1 CINITIALL/ING FOR THE FIRST LOAD ITERATION 15FE1 NDCT = NPTTS(NN) 15FE1 SG TO 6065 15FE1 6057 ESM = 0.5 15FE1 GO TO 6065 15FE1 GO TO 6065 15FE1 IF (IFUS(J), EQ, 1) GO TO 6081 10FE1	6900	TE C MARTI (41, 0) (40 1) (7720	28050
CTISM E () (LIKEM SMINOS, SU SCIENTANI(- 1 6905 MAIT = 1 6906 C =INITIALIZING FCH NEXT ITFDATION CINITIALIZING FCH NEXT ITFDATION C =INITIALIZING FCH NEXT ITFDATION NOS = NOSJCHN) NOS = NOSJCHN) N	- 281	NI YTOL N - A ANALD CODINCE AG OFT MAKIN - 1	Zenfa
6905 MAX11 = 1 MAX17 2000 6910 CC 6000 ATT = 1, MAX17 2000 CTAUTIALIZING FOR NEXT ITFDATION 2000 01(j) = Q(j) + OT(j,1) 21AP1 CTAUTIALIZING FOR NEXT ITFDATION 2000 01(j) = Q(j) + OT(j,1) 21AP1 CTAUTIALIZING FOR NEXT ITFDATION 2000 Study = S(j) 2000 65n0 CONTINUE CTAUTIALIZING FOR THE FORTON 2000 NPTS = 0 10FE1 NPTS = 0 10FE1 NPTS = 0 10FE1 NPTS = 0 10FE1 NPT = NPTTS(NN) 15FE1 NPT = NPTTS(NN) 15FE1 NPT = NPTTS(NN) 15FE1 NPT = NPTTS(NN) 15FE1 CTINITIALI/ING FOR THE FIRST LOAD ITERATION 06AP1 CTINITIALI/ING FOR THE FIRST LOAD ITERATION 15FE1 6057 ESM = 0.5 15FF1 6057 ESM = 0.5 15FF1 6057 ESM = 1.0 10FE1 6075 ESM = 1.0 10FE1	C112	R R DI FIKER SHIKON SO AL MANICE I	28050
b910 U0 6000 K/T = 1; # MART 2000 KOPFC(NT) = 0 2000 CINITIALIZING FCR NEXT ITEDATION 2000 D0 6500 J = 1; MPT 2000 G1(J) = G(J) + DT(J,1) 2100 S1J) = S(J) 2000 6500 CONTINUE 2000 S1J) = S(J) 2000 6500 CONTINUE 2000 S1J) = S(J) CONTINUE 2000 6500 CONTINUE 2000 NDS = NOSJ(KN) 15FE1 10FE1 NDS = NOSJ(KN) 15FE1 10FE1 NDCT = MPTTS(NN) 15FE1 10FE1 NDCT = MPTTS(NN) 15FE1 06AP1 NDCT = MPTTS(NN) 15FE1 06AP1 NDCT = MPTTS(NN) 15FE1 06AP1 MU, KK) = W(J, KK-1) 00 TO 6057 06AP1 6557 IF (ISKUJ) = E0.0) GO TO 6057 06AP1 6557 IF (ISKUJ) = E0.1) GO TO 6085 15FE1 6057 ESM = 0.5 15FE1 6057 ESM = 0.5 15FE1 6057 IF (IFUS(J) = E0.1) GO TO 6085 10FE1	6905		20050
KOPPL(NT1) # 0 2000 CINITIALIZING FGR NEXT IFPRATION 2000 D0 6500 J = 1 + MP7 21AP1 01(J) = 0(J) + 0T(J+1) 21AP1 SS(J) = S(J) 21AP1 SS(J) = S(J) 2000 6500 CONTINUE 2000 D0 6500 N = 1 + MP7 21AP1 SS(J) = S(J) 21AP1 SS(J) = S(J) 21AP1 CONTINUE 2000 D0 650 NN = 1 + NNC 21F NP15 = 0 10FE1 NP15 = 0 11F J = 1STA(NN) 15FE1 NPCT = NPTS(NN) 15FE1 NPCT = NPTS(NN) 15FE1 NPCT = NPTS(NN) 15FE1 NPCT = NPTS(NN) 15FE1 S57 IF (ISK(J) * E0, 0) GO TO 6057 06AP1 CTKST TO SEE IF NEDS ADJUSTMENT OF THE G-W CUPVE FOR THE FIRST TIMESFE1 S6051 IF (ITCS * FG. 0) GO TO 6085 15FE1 IF (ARS(W(J*K*))) * ARS(W(J*K*1)) J & CO TO 603 10FE1 IF (ARS(W(J*K*))) * ANS(W(J*K*1)) J & CO TO 603 10FE1 IF (IFUS(J) * ANS TO F OF OW CUPVE AFTER THE FIRST TIME IFEE 10FE1 IF (ARS(W(J*K*1)) * ANS TO F OW CUPVE AFTER THE FIRST TIME I	6910	00 6000 KIT T T MAXI	20050
CINITIALIZING FUR NEAF ITERATION DC 6550 J = 1 + PP7 GI(J) = G(J) + DT(J,1) SI(J) = G(J) + DT(J,1) SI(J) = G(J) + DT(J,1) SI(J) = G(J) + DT(J,1) SI(J) = S(J) CONTINUE DC 6056 NN = 1 + NNC DC 6056 NN = 1 + NNC NPIS = 0 J = ISTA(NN) NPIS = 0 J = ISTA(NN) NPCT = NPTTS(NN) ISFEI NPCT = NPTTS(NN) ISFEI NULL NULL NULL NULL NULL NULL NULL NULL			20050
0 0 3 1 4 4 4 0 1() = G(J) + G(J)	CINI	TIALIZING FUR NEAT TIFRATION	20050
01(J) # 0(J) + 0(1J,1) 24050 65n0 C0NTINUE 24050 1F (NCVB + E0, 0) GO TO 6504 24050 00 + 0.56 NN = 1 + NNC 18FE1 NP15 = 0 10FE1 NP15 = 0 10FE1 NCG = NOSJ(NN) 18FE1 Stress 18FE1 Go TO 6065 18FE1 Go TO 6065 18FE1 Go TO 6065 18FE1 IF (IFUS(J) * E0 * 1) GO TO 6081 10FE1 Go TO 6065 19FE1 IF (IFUS(J) *KN) *CT * ARS(K(J*KK+1)) 1 GC TO 603 10FE1		$DO 6500 J = 1 \cdot MP7$	21401
65n0 CANTINUE 20000 65n0 IF (NCVB + EG. 0) GO TO 6504 20000 00 4056 NN = 1 + NNC 15FE1 NP15 = 0 10FE1 J = 15TA(NN) 15FE1 NDCT = NPTTS(NN) 15FE1 NPCT = NPTTS(NN) 15FE1 NPCT = NPTTS(NN) 15FE1 NPCT = NPTTS(NN) 15FE1 SC 06AP1 C*INITIALL/ING FOR THE FIRST LOAD 1TERATION 06AP1 C*INITIALL/ING FOR THE FIRST LOAD 1TERATION 06AP1 C*INITIALL/ING FOR THE FIRST COAD 15ENTION 06AP1 C*INITIALL/ING FOR THE FIRST 15EN FIRST 06050 105FE1 607 TO 6085 15FE1 607 TO 6085 15FE1 16051 16FE1 16051 16FE1 16051 16FE1 17 (IFUS(J) *EQ. 1) GO TO 6085 10FE1 187 (IFUS(J) *EQ. 1) GO TO 60			20050
6500 CONTINUE 280E0 1F (NCVB +EQ. 0) GO TO 6504 280E0 0 AD56 NN = 1+ ANC 18FE1 NP15 = 0 10FE1 J = 1STA(NN) 18FE1 NOS = NOSJ(NN) 18FE1 Statistic Not = NTS(NON) 18FE1 Statistic NOS = NOSS 06AP1 6557 FSM = 0.5 GO TO 6085 15FE1 6057 FSM = 1.0 107E1 SEC IF NEEDS ADJUISTMENT OF THE CHACUPYE FOR THE FIRST TIMESTEN 11F (INSKI) .EC, NFEDS ADJUISTMENT OF THE CHACUPYE FOR THE FIRST TIMESTEN 11F (INSKI) .CC, NISSINGAT OF GO TO 603 10FE1 <		SS(J) = S(J)	20050
IF (NCWB, EG. 0.) 50 TO 6504 LEDEO CC 6056 NN = 1, NNC IBFEI NPIS = 0 IOFE] J = ISTA(NN) IBFEI NOC = NNSJ(NN) IBFEI NPCT = NPTTS(NN) IBFEI NPCT = NPTTS(NN) IBFEI C^INITIALLIVING FOR THE FIRST LOAD ITERATION IF (IOSW(J), EG. 0) GO TO 6557 O6API C^INITIALLIVING FOR THE FIRST LOAD ITERATION IF (IOSW(J), EG. 0) GO TO 6057 O6API 6557 IF (ISW(J), EG. 0) GO TO 6057 O6API 6557 IF (ISW(J), EG. 0) GO TO 6085 ISFEI 6070 G665 ISFEI 6071 ESM = 0.5 ISFEI 6075 ESM = 1.0 ISFEI 6076 ISFEI ISFEI 6077 ESM = 1.0 ISFEI 6078 IF (IFUS(J), EG. 1) GO TO 6086 IOFEI IF (ISS(M), LE, NEST ADUISTMENT OF THE G=W CUPVE FOR THE FIRST TIMESFEI ISFEI 1 M(J+KK)) + CF. ARS(W(J+KK=1))) GO TO 603 IOFEI 1 GO TO 6066 IOFEI 1 GO TO 6067 IOFEI 1 GO TO 6068 IOFEI <tr< td=""><td>6500</td><td>CONTINUE</td><td>20050</td></tr<>	6500	CONTINUE	20050
D0 6056 NN = 1* NNC 10FE1 N015 = 0 10FE1 J = ISTA(NN) 15FE1 N05 = NNSJ(NN) 15FE1 N07 = *PTTS(NN) 15FE1 N07 = *PTTS(NN) 15FE1 IF (IOSw(J) *G0 0) GO TO 6557 06AP1 CTNITALJING FOR THE FIRST LOAD ITERATION 06AP1 15 K(N) = W(J*KK) = W(J*KK-1) 06AP1 6557 ISFE1 657 ESM = 0.5 15 FE1 0607 6057 ESM = 1.0 6058 15FE1 6057 ESM = 1.0 6058 15FE1 6057 ESM = 1.0 6058 15FE1 16059 IF (ITCS *FG, O) GO TO 6086 10FE1 IFFE1 16085 IF (IFUS(J) *EG, 1) GO TO 6081 CTEST TO SEE IF NFEDS ADJUSTMENT OF THE G-W CUPVE FOR THE FIRST TIFEFE1 17 W(J*KK)) *GO TO 603 10FE1 18 IF ((APS(W(J*KK+1)) * GA TO 603 10FE1 19 W(J*KK)) *GO, TO 603 10FE1 100 19FE1 IFE1 10FE1 10 GO TO 676		IF UNCVB .EG. 0 3 30 TO 6504	INCE)
NP15 # 0 10°C1 J = ISTA(NN) 15FE1 NOS = NOSJ(KN) 15FE1 NPCT = NPTTS(NN) 15FE1 NPCT = NPTTS(NN) 15FE1 C=INITIALL/ING FOR THE FIRST LOAD ITERATION 06AP1 C=INITIALL/ING FOR THE FIRST LOAD ITERATION 06AP1 6557 IF (ISW(J) .EQ. 0) GO TO 6057 06AP1 6557 ESM = 0.5 15FE1 607 TO 6085 15FE1 6085 IF (ITCS +FG. 0) GO TO 6086 10FE1 15FF IF (ITCS +FG. 0) GO TO 6086 10FE1 16FE1 IF (IRST TIMESTS ADJUSTMENT OF THE G=W CURVE FOR THE FIRST TIMESFE1 10FE1 17 W(J+KK)) .CT. ARS(W(J+KK+1)) GO TO 603 10FE1 18 W(J+KK)) .CT. ARS(W(J+KK+1)) .GC TO 603 10FE1 18 W(J+KK)) .CT. ARS(W(J+KK+1)) .GC TO 603 10FE1 18 W(J+KK)) .CT. ARS(W(J+KK+1)) .GC TO 603 10FE1 19 W(J+KK)) .CT. ARS(W(J+KK+1)) .GC TO 603 10FE1 19 W(J+KK)) .CT. ARS(W(J+KK+1)) .GC TO 6036 10FE1 19 W(J+KK)) .CT. ARS(W(J+KK+1)) .GC TO 6036 10FE1 10 IF (W(J+KK)) .GE, W(J+KK+1) .GC TO 6096		DO 4056 NN = 10 NNC	10551
J = 1514 (NN) 15FE1 NDCT = NPTT5(NN) 15FE1 NPCT = NPTT5(NN) 15FE1 NPCT = NPTT5(NN) 15FE1 C^INITIALL/ING FOR THE FIRST LOAD 1TERATION 06AP1 C*INITIALL/ING FOR THE FIRST LOAD 1TERATION 06AP1 W(J,KK) = W(J,KK-1) 06AP1 6557 IF (ISW(J) = E0.0) E0 TO 6057 06AP1 6557 IF (ISW(J) = E0.0) E0 TO 6057 06AP1 6557 IF (ISW(J) = E0.0) E0 TO 6057 06AP1 657 ESM = 0.5 15FE1 6057 ESM = 1.0 15FE1 6057 ESM = 1.0 15FE1 6057 ESM = 1.0 15FE1 1 IF (IFUS(J) = E0.1) GO TO 6086 10FE1 1 IF (ISS(J) = E0.1) GO TO 6081 10FE1 1 IF (ISS(J) = KARSUS ADJUSTMENT OF THE C-W CUPVE FOR THE FIRST TIMESFE1 10FE1 1 IF (ISS(U)=KK)) = C.1 ARS(W(U=KK))) = E0. ABS(W(J,KK=1) = 10FE1 10FE1 1 W(J=KK)) = 0010 603 10FE1 10FE1 1 W(J=KK)) = 0010 603 10FE1 10FE1 1 IF (W(J=KK)) = 0010 603 10FE1 10FE1 </td <td></td> <td>NP15 # 0</td> <td>10761</td>		NP15 # 0	10761
NOS = NOSJIKN) 15FE1 IF (IOSW(J) *F0. 0) GO TO 6557 06AP1 C************************************		J = 1STA(NN)	10101
NPCT = KPTTS(NN) 19701 IF (IOSW(J), FO, 0) GO TO 6557 06AP1 C====INTTIALI/ING FOR THE FIRST LOAD ITERATION 06AP1 IF (IOSW(J), FO, 0) GO TO 6557 06AP1 6557 IF (ISW(J), EO, 0) GO TO 6057 06AP1 6557 IF (ISW(J), EO, 0) GO TO 6057 06AP1 6557 ISFE1 06AP1 6070 6065 15FE1 6075 ESM = 0.5 15FE1 6076 ISFE1 15FE1 6075 ESM = 1,0 GO TO 6085 IF (IFUS(J), #EO, 1) GO TO 6081 10FE1 C====TEST TO SEE IF NFEDS ADJUSTMENT OF THE Q=W CUPVE FOR THE FIRST TIMESFE1 10FE1 IF (ARS(W(J,KK+1)) + CARS(W(J,KK+1)) + GC TO 603 10FE1 IGO TO 6066 10FE1 IF ((APS(W(J,KK+1)) + GO TO 603 10FE1 IGO TO 6066 10FE1 IF (W(J,KK)) + GE, MIJSTMENT OF Q=W CUPVE AFTER THE MENT TIME 14FE1 GO TO 6066 10FE1 IGO TO 6067 10FE1 IGO TO 6068 10FE1 IGO TO 6066 10FE1 IGO TO 6066 10FE1 IGO TO 6070 10FE1 <t< td=""><td></td><td>NGS = NOSJINN)</td><td>10721</td></t<>		NGS = NOSJINN)	10721
IF ([05w(J) *E0. 0) Gn TN 0557 064P1 C=IN [TIALJ/ING FOR THE FIRST LOAN ITERATION 064P1 IF (NIT *GT, NITS) Gn TO 6557 064P1 w(J*KK) = w(J*KK) = 0.0) GD TO 6057 064P1 6557 ISFE1 6557 ISFE1 6057 ISFE1 1 W(J*K*)) * IST * IST * ISE 1 W(J*K*)) * IST * IST * ISE 1 W(J*K*)) * IST * IST * IST * ISE 1 W(J*K*)) * IST		NPCT = NPTTS(NN)	15261
CINITIAL/FING FOR THE FIRST LOAD ITERATION		IF (IOSW(J) .EQ. 0) GO TO 0557	USAP 1
IF (NIT .6T. NITS) G0. TO 6557 064P1 w(J,KK) = w(J,KK-1) 064P1 6557 IF (ISw(J) .EQ. 0) 60 TO 6057 064P1 6557 IF (ISw(J) .EQ. 0) 60 TO 6057 064P1 6070 6068 15FE1 6075 ESM = 0.5 15FE1 6085 IF (ITUS(J) .EQ. 1) GO TO 6086 10FE1 IF (IFUS(J) .EQ. 1) GO TO 6081 10FE1 IF (IFUS(J) .EQ. 1) GO TO 6081 10FE1 IF (IFUS(J) .EQ. ABS(W(J,KK-1)) IGC TO 603 10FE1 IF (IRS(W(J,KK-1)) .ARS(W(J,KK-1)) IGC TO 603 10FE1 IF (IAS(W(J,KK-1)) .ARS(W(J,KK+1)) I.EQ. ABS(W(J,KK-1) = 10FE1 10FE1 IGC TO 6026 10FE1 GO TO 6026 10FE1 IF (IW(J,KK) .DE. MJJSTMENT OF Q-W CUNVE AFTER THE MIRST TIME IRFE1 601 IF (WYT(KN-1) .GE, W(J,KK-1)) .GO TO 6086 10FE1 106 CT 0.503 10FE1 602 TF (W(J,KK) .LE. W(J+KK-1)) .GO TO 6086 10FE1 603 IF (W(J,KK) .LE. W(J+KK-1)) .GO TO 6086 10FE1 603 IF (W(J,KK-1)) .669. A084. 503 260JJ1 503 IF (W(J,KK-1)) .669. A084. 503 260JJ1 503 IF (W(J,KK) .EC. 1) .60 TO 604 15FE1 <td>C+IV1</td> <td>TIALIZING FOR THE FIRST LOAD ITERATION</td> <td></td>	C+IV1	TIALIZING FOR THE FIRST LOAD ITERATION	
w(J,KK) = w(J,KK) = (0,0) G0 TO 6057 06AP1 6557 IF (ISWLJ) E0.0) G0 TO 6057 06AP1 ESW = 0.5 15FE1 6057 ESW = 1.0 15FE1 6057 ESW = 1.0 15FE1 6057 ESW = 1.0 15FE1 6085 IF (ITCS +FG, 0) GO TO 6086 10FE1 IF (IFUS(UJ) + E0.1) GO TO 6081 18FE1 IF (IFUS(UJ) + E0.5 ADJUSTMENT OF THE Q=W CUPVE FOR THE FIRST TIME5FE1 18FE1 IF (ARS(W(J+KK)) + LT, ARS(W(J+KK-1)) J GC TO 603 10FE1 IF ((ARS(W(J+KK))) + CH, ARS(W(J+KK-1)) J GC TO 603 10FE1 IF ((APS(K(J+KK))) + CH, ARS(W(J+KK-1)) J GC TO 603 10FE1 GO TO 6026 10FE1 IF ((M) = KK) J 60, Y0 603 10FE1 GO TO 6026 10FE1 GO TO 603 10FE1 GC TO 503 10FE1 GC TO 503 15FE1 603 IF ((W (J+KK-1)) 669* A096* 503 10FE1 GO TO 503 10FE1 GO TO 503 10FE1 GO TO 503 15FE1 603 IF ((W (J+KK-1)) 669* A096* 503 260JU1 503 IF ((KYW ((AN)) *E0* 1) 60 TO 604 15FE1		IF (NIT .GT. NITS) GO TO 6957	UEAPI
6557 IF (ISw(J), EQ. 0) 00 TO 6057 00AD1 ESM = 0.5 ISFF1 G0 TO 6085 ISFF1 6057 ESM = 1.0 ISFF1 6085 IF (ITCS +F0. 0) GO TO 6086 IDFE1 6085 IF (ITCS +F0. 0) GO TO 6086 IDFE1 C====TEST TO SEE IF NFEDS ADJUSTMENT OF THE 0=W CUPVE FOR THE FIRST TIMEFFE1 IF (ARS(W(J+KK)) + LT, ARS(W(J+KK+1))) GO TO 603 IDFE1 IF ((ARS(W(J+KK))) + ARS(W(J+KK+1))) GO TO 603 IDFE1 IDFE1 GO TO 6086 IDFE1 IDFE1 IF (W(J+KK)) + OF 0 ADS(W(J+KK+1)) + CO 4086 IDFE1 GO TO 6086 IDFE1 IDFE1 GO TO 503 GO TO 6086 IDFE1 GO TO 503 IDFE1 IDFE1 GO TO 50		₩(J, KK) = ₩(J, KK=1)	UBAPI
ESM = 0.5 GO TO 6085 15FE1 6057 ESM = 1.0 16050 IF (ITCS +FG + O) GO TO 6086 IF (IFUS(J) +EQ. 1) GO TO 6081 CTEST TO SEE IF NFEDS ADJUSTMENT OF THE Q-W CUPVE FOR THE FIRST TIMESFE1 IF (APS(W(J+KK)) +LT, APS(W(J+KK))) GC TO 603 IF ((APS(W(J+KK)) +LT, APS(W(J+KK))) +EQ. ABS(W(J+KK-1) + 10FE1 1 w(J+KK)) GO TO 603 IDFE1 1 w(J+KK)) GO TO 603 IDFE1 1 w(J+KK)) GO TO 603 IDFE1 1 f (WPT(N+1)) GO TO 603 IDFE1 1 f (WPT(N+1)) GO TO 607 6091 IF (W(J+KK) ,GE, W(J+KK-1)) GO TO 6076 10FE1 1 GC TO 503 602 TF (W(J+KK) +LE, W(J+KK-1)) GO TO 6076 10FE1 603 IF (W(J+KK) +LE, W(J+KK-1)) GO TO 6076 10FE1 1503 IF (W(J+KK-1)) 669, 6076, 503 260/01 26	6557	IF (ISW(J) _EQ_ 0) 00 70 6057	UBAP1
G0 T0 6085 15FE1 6057 ESW = 1,0 6085 IF (ITCS *F0, 0) GO TO 6086 IF (ITCS *F0, 0) GO TO 6081 10FE1 IF (ITCS *F0, 0) GO TO 6081 10FE1 IF (ITCS *F0, 0) SEC IN JUSTMENT OF THE G-W CUPVE FOR THE FIRST TIMESFE1 10FE1 IF (ARS(W(J*K*)) *LT, ARS(W(J*K*-1)) I GO TO 603 10FE1 IF (ARS(W(J*K*)) *LT, ARS(W(J*K*-1)) I GO TO 603 10FE1 IF (ARS(W(J*K*)) *LT, ARS(W(J*K*))) *EG * ABS(W(J*K*-1) * 10FE1 10FE1 G0 TO 6086 10FE1 IF (APS(W(J*K*)) *LT, ARS(W(J*K*))) *EG * ABS(W(J*K*-1) * 10FE1 10FE1 G0 TO 6086 10FE1 G0 TO 6086 10FE1 G0 TO 6086 10FE1 G0 TO 6086 10FE1 S01 IF (W(J*K*) *GE, W(J*K*-1) * GO TO 6086 10FE1 G0 TO 503 15FE1 603 IF (W(J*K*-1) * 66* * 096 * 607 260011 503 IF (W(J*K*-1) * 66* * 016 * 503 260011 503 IF (W(J*K*-1) * 66* * 016 * 0064 15FE1		ESM = 0.5	15FF.1
6057 ESM = 1,0 15FE1 6085 IF (ITUS(J), EQ, 1) GO TO 6086 10FE1 IF (IFUS(J), EQ, 1) GO TO 6081 15FE1 C====TEST TO SEE IF NFEDS ADJUSTMENT OF THE Q=W CUPVE FOR THE FIRST TIMESFE1 10FE1 IF (IARS(W(J,KK)), LT, ARS(W(J,KK+1))) GO TO 603 10FE1 IF (IARS(W(J,KK))) GO TO 603 10FE1 IF (IARS(W(J,KK))) GO TO 603 10FE1 IF (IARS(W(J,KK))) GO TO 603 10FE1 GO TO 6086 10FE1 GO TO 6086 10FE1 GO TO 6086 10FE1 GO TO 6086 10FE1 GO TO 503 10FE1 GO TO 503 </td <td></td> <td>GO TO 6085</td> <td>15FE1</td>		GO TO 6085	15FE1
6085 IF (ITCS +FG, 0) GO TO 6086 10FE1 1F (IFUS(J) *E0.1) GO TO 6081 15FE1 CTEST TO SEE IF NEEDS ADJUSTMENT OF THE Q-W CUPVE FOR THE FIRST TIMESFE1 IF (ARS(W(J*K*1)) +LT, ARS(W(J*K*1))) GC TO 603 10FE1 IF (ARS(W(J*K*1)) +LT, ARS(W(J*K*1))) GC TO 603 10FE1 IF (ARS(W(J*K*1))) GO TO 603 10FE1 IF (ARS(W(J*K*1))) GO TO 603 10FE1 1 W(J*KK)) GO TO 603 10FE1 1 GO TO 6486 10FE1 1 F(WTKN)) GO TO 603 10FE1 1 GO TO 6486 10FE1 1 GO TO 6486 10FE1 1 GO TO 6486 10FE1 1 GO TO 503 10JUSTMENT OF Q-W CUHVE AFTER THE METT TIME 18FE1 501 IF (W(J*K*) *GE, W(J*KK-1)) GO TO 6086 10FE1 503 IF (W(J*K*) *LE. W(J*KK-1)) GO TO 6086 10FE1 603 IF (W(J*K*) *LE. W(J*KK-1)) GO TO 6086 10FE1 603 IF (W(J*K*) *LE. W(J*KK-1)) GO TO 6086 15FE1 603 IF (W(J*K*) *LE. *GE + 1) GO TO 604 15FE1	6057	ESM # 1.0	15FE1
IF (IFUS(J) .EQ. 1) GO TO 6081 CTEST TO SEE IF NEEDS ADJUISTMENT OF THE Q-W CURVE FOR THE FIRST TIMESFE1 IF (ARS(W(J+KK-1)) .LT. ARS(W(J+KK-1))] GC TO 603 [OFE] IF (ARS(W(J+KK-1)) .LT. ARS(W(J+KK-1))] GC TO 603 [OFE] 1 W(J+KK-1) .CO 608 [OFE] GO TO 6086 [OFE] 6081 IF (WPT(IKN+1)] 601, 9991, 502 [OFE] 6081 IF (WPT(IKN+1)] 601, 9991, 502 [OFE] 601 IF (WPT(IKN+1)] 601, 9991, 502 [OFE] 602 IF (W(J+KK) .EE, W(J+KK-1)] GO TO 6086 [OFE] 603 IF (W(J+KK) .LE, W(J+KK-1)] GO TO 6086 [OFE] 603 IF (W(J+KK-1)] 669, 6086, 503 [OFE] 603 IF (W(J+KK-1)] 669, 6086 [OFE] 603 IF (W(J+KK-1)] 669, 6086 [OFE] 603 IF (W(J+KK-1)] 669, 6086 [OFE] 604 [OFE] 605 IF (W(J+KK-1)] 669, 6086 [OFE] 606 [OFE] 607 IF (W(J+KK-1)] 669, 6086 [OFE] 608 [OFE] 609 IF (W(J+KK-1)] 669, 6086 [OFE] 609 [OFE] 609 [OFE] 600 [OF	6085	IF (ITCS +FA+ 0) GO TO 6086	10FE1
CTEST TO SEE IF NFEDS ADJUSTMENT OF THE G-W CUMVE FOR THE FIRST TIPESFEI IF (ARS(W(J,KK+1) + LT, ARS(W(J,KK+1))) GC TO 603 10FEI IF ((APS(W(J,KK+1))) GO TO 603 10FEI 1 W(J,KK)) (GO TO 603 10FEI 60 TO 6026 10FEI 6091 IF (WPT(INN_1)) 601, 9991, 602 10FEI 6091 IF (WPT(INN_1)) 601, 9991, 602 10FEI 6010 IF (WPT(INN_1)) 601, 9991, 602 10FEI 602 IF (WPT(INN_1)) 601, 9991, 602 10FEI 603 IF (W(J,KK) , GE, WIJ,KK-1)) 60 TO 6086 10FEI 602 IF (W(J,KK) , GE, WIJ,KK-1)) 60 TO 6086 10FEI 603 IF (W(J,KK) , LE, W(J+KK-1)) 60 TO 6086 10FEI 603 IF (W(J,KK-1)) 669, 6084, 503 26JUI 503 IF (KYMJ(NK) > EG, 1) 60 TO 604 15FEI		IF (IFUS(J) .EQ. 1) GO TO 6081	15FE1
IF (ARS(W (J+KK)) + LT, ARS(W (J+KK-1))) GC TO 603 10FE1 IF ((APS(W (J+KK-1))) GD TO 603 10FE1 1 W (J+KK)) GD TO 603 10FE1 1 0 (J+KK)) GD TO 603 10FE1 1 10 (J+KK)) GD TO 603 10FE1 1 10 (J+KK)) GD TO 603 10FE1 1 10 (J+KK)) GD TO 603 10FE1 501 IF (W (J+KK) , GE, w(J+KK-1)) GD TO 6086 10FE1 602 IF (W (J+KK) + LE. W (J+KK-1)) GD TO 6086 10FE1 603 IF (W (J+KK-1)) 669 + 6086 + 503 26001 503 IF (W (J+KK-1)) 667 + 1) 60 TO 604 15FE1	CTES	T TO SEE IF NEEDS ADJUSTMENT OF THE G-W CURVE FOR THE FIRST 'I	MESPE1
IF ((APS(W(J+KK-1)) + AHS(W(J+KK))) .EQ. ABS(W(J+KK-1) + 10FE1 0 W(J+KK))) GO TO 603 6081 IF (WPT(KN+1)) 601, 9991, 502 CTFST TO SEE IF NEEDS ADJUSTMENT OF Q-W CURVE AFTER THE TIRST TIPE 18FE1 601 IF (W(J+KK) .GE, W(J+KK-1)) GO TO 6086 10FE1 602 IF (W(J+KK) .LE, W(J+KK-1)) GO TO 6086 10FE1 603 IF (W(J+KK-1)) 669, 6086, 503 16FE1 603 IF (W(J+KK-1)) 669, 6086, 503 16FE1 503 IF (W(J+KK-1)) 6670 604		IF (ARS(W(J+KK)) +LT, ARS(W(J+KK+1))) GC TO 603	10FE1
1 W(J+KK))) GO TO 603 10FE1 GO TO 606 10FE1 6081 IF (WPT(KN+1)) 601, 9991, 602 10FE1 CTFST TO SEE IF NFEDS ADJUSTMENT OF Q-W CUNVE AFTER THE LIRST TIME 14FE1 601 IF (W(J+KK), GE, WIJ+KK-1)) GO TO 6086 602 IF (W(J+KK), GE, WIJ+KK-1)) GO TO 6086 10FE1 602 IF (W(J+KK), LE, W(J+KK-1)) GO TO 6086 10FE1 603 IF (W(J+KK), EG, 1) 6070 604 15FE1		IF ((APS{W(J+KK+1)) + VHS(M(J+KK))) +EG+ ARS{M(J+KK+1) +	10FE1
GO TO 6086 10FE1 IF (WPT(NN,1)) 601, 9991, 602 10FE1 CFRT TO SEE 1F NEDS ADJUSTMENT OF Q-W CURVE AFTER THE PIRST TIPE 18FE1 501 IF (W(J_KK) GE, W(J_KK-1)) GO TO 6086 10FE1 602 IF (W(J_KK) LE. W(J+KK-1)) GO TO 6086 10FE1 603 IF (W(J_KK-1)) 669, 6086, 503 2603 IF (W(J_KK) + EE, 1) 60 TO 604	1	W(J+KK))) GO TO 003	10FE1
6091 IF (WPT(NN+1)) 601, 9991, 602 10FE1 CTFST TO SEE IF NFEDS ADJUSTMENT OF 0-W CUNVE AFTER THE LIRST TIPE IFFE1 601 10FE1 601 IF (W(J,KK), GE, W(J,KK-1)) 60 TO 6086 10FE1 602 IF (W(J,KK), LE, W(J+KK-1)) 60 TO 6086 10FE1 603 IF (W(J,KK), LE, W(J+KK-1)) 60 TO 6086 15FE1 603 IF (W(J,KK-1)) 669, 6086, 503 26001 503 IF (KSYMJ(NK) *EG, 1) 60 TO 604 15FE1		60 TO 6486	IOFE)
CTFST TO SEE IF NEOS ADJUSTMENT OF CHUYE AFTEN THE IRST THE IRST IF InfF1	6091	IF (WPT(NN+1)) 601+ 9991+ 602	LOFET
501 if i ((j,kk), GE, (j,kk-j)) GO TO 6086 10FF1 GC TO 503 15FF1 602 if (((j,kk), LE, W(J,kK-j))) GD TO 6086 10FE1 603 if ((((j,kk-j)))) 669, 6086, 503 16FE1 503 if ((((j,kk-j)))) 669, 6086, 503 16FE1 503 if (((((j,kk-j)))) 669, 6086, 503 16FE1 503 if (((((j,kk-j)))) 669, 6086, 503 16FE1	CTES	T TO SEE IF NEEDS ADJUSTMENT OF Q-W CURVE AFTER THE FIRST TIPE	15FE1
GC TO 503 15FE1 602 IF (W(J+KK) +LE, W(J+KK-1)) GD TO 6086 10FE1 GO TO 503 IF (W(J+KK-1)) 669, 6086, 503 15FE1 603 IF (W(J+KK-1)) 669, 6086, 503 26U1 503 IF (KSYMJ(NN) +EG, 1) 60 TO 604 15FE1	601	JE (W(J;KK) _GE, W(J;KK+))) GO YO ADA6	INFEL
602 1F (W(J_1KK) .LE. W(J+KK-1)) GD TO 6086 10FE1 603 1F (W(J_1KK-1)) 669. K08K. 503 15FE1 503 1F (W(J_1KK-1)) .669. K08K. 503 26001 503 1F (KSYMJ(NH) .EG. 1) .60 TO 604 15FE1		GC 10 503	15FE1
CO TO SA3 15FE1 603 IF (W(J+KK=1)) 669+ 6096+ 503 26001 503 IF (KSYWJ(NN) + EG+ 1) 60 TO 604 15FE1	602	IF (W(J+KK) +LE, W(J+KK+1)) GD TO 6086	10FE1
603 IF (₩(J,KK=1)) 669, 6096, 503 26JU1 503 IF (KSYMJ(NH) +EG+ 1) 60 TO 604 15FE1		CO TO 503	ISFE1
503 IF (KSYMJ(NK) .EG. 1) GO TO 604 15FE1	603	1F (W(J,KK-1)) 669. 6096. 503	26JU1
	503	IF (KSYMJ(NH) +EG. 1) GO TO 604	15FE1

	055E0		IF (IFUS(J) .EQ.)) GO TO 484	15 AP
	DESEO		TE (KSYMJ(NK) .EG1) 60 0 604	24001
	055E0		GO TO 6486	15FE1
	055E0	C+RFV	FREE THE ARDER OF NON THEAP SUPPORT CURVE	lorel
	16.141	604	PO + OA = NP = 1 + PCT	10FE1
	95580		NPR * NPCT = NP +)	10FF1
	05550		HPVT (NN NP) # HPT (NN NPH)	15FE1
,) (16.141		GPVT(NN,NP) = QPT(NN,NPR)	15FE1
INC	16.41	606	CONTINUE	10FE1
-	16 141		CC 506 KP = 1+ KPCT	15FE1
	055F0		WPT(N, **P) = WPVT(N, *P)	15FE1
	055E0		OPT(NN+NP) = OPVT(NN+NP)	15FE1
	10000	506	CONTINUE	15FE1
	ZPDFA	••••	TE LESTEULER SEC. 1 5 GA TE 605	154P1
	28050	CCHE	CK TO SEE TE REFDS ADJUSTMENT OF THE QHW OV FOR ONE-WAY SUPPL	CR126JU1
	28050		TF (KSYMU(NN) .NE1) GO TO 666	26301
	ZADED		IF (IFUS(J) .EQ. 0) GO TO 605	56001
	28050	666	DO 507 KP = 2+ KPCT	26JU1
	20050		STEMP = ABS { (QPT(NN+NP) - QPT(NN+NP=1)) /	2) AP 1
	280F0	1	$\{ \text{WPT}(NK_1KP) = \text{WPT}(NN_1NP-1) \}$	21 AP1
	2PDF0	•	IF I ABSISTENP - SLOPE (NN. 117 .LF. 1.0F-06) GO TO 508	15AP1
	280E0	507	CONTINUE	154P1
	21AP1	PRI	NT 980	15461
	28050		60 TO 9999	15AP1
	20050	50B	IF (WPT(KN+1) 1 509+ 9991+ 511	15AP1
	28050	509	NP = NP = 1	15401
	15FE1	511	IF [KSYNU (NA) .EC. 0) GO TO 667	26J-)1
	10FE)		}F (W(J+XK-1) +LE, WPT(NN+NP) } 60 TO 6086	26.01
	15FE1		GO TO 645	26001
	15FE1	667	IF (W(J+KK+1) +CE, WPT(NN+N+)) GO TO 6086	26001
	15FE1		GD TO 605	26JU1
	064P1	669	IF (KSYMJ(NN) +EG+ 1) GO T ^O 605	56101
			IF (KSYMJ(NK) +EG+ 0) GD TU 605	56103
	06491		GD TO 4086	26001
	OGAPI	605	$i \cup \{(j) = i \cup \{(j) + 1\}$	15FE1
	064P1		DO 607 NP = 2+ NPCT	lofe1
	15FE)		IF (WPT(NN+1)) 608+ 9991+ 809	10FE1
	1SFE1	608	1F (W(J+KK-1) + WPT(NN+NP)) 631+ 613+ 607	LOFE1
	15FÊ1	609	IF (W(J_KK-)) = WPT(NN+NP) 3 607, 613, 611	10F51
	10FE1	607	CONTINUE	10FE1
	15FE1		GO TO 612	LOFEI
T TI	PESFEI	631	IF ($W(J_{KK-1}) = WDI(NN+1)$) 612+ 632+ 614	10681
	lofEl	611	$IF (W(J_{4} \times K - 1) = WDI(NN+1)) 614 + 632 + 612$	106E1
*	10FE1	612	KOFFC(NTT) = 1	10FE1
	lofEl		GO TO 614	10FE1
	10FE)	613	NPS = NP + 1	IOFE1
	10FE1		GO TO 615	10FE1
TI≠E.	18FE1	635	NP15 = 1	LOFFI
	10FF1	61*	NDS F ND	INFFI
	15FE1	CTES	T TO SEE # (J+KK-1) IS AT WHICH SLOPE (NN+NP)	10FE1
	10FE1	615	STENP = ARS ((QPT(NN+NPS) = QPT(NN+NP5-1))	21AP1
	15FE1	1	(WPT(NN+NPS) = WPT(NN+NPS+1))	21AP1
	26JU1		1F 1 STERP .IE. 1.0E-06 1 GO TO 641	LOFEI
	15FE1		GTEMP = STEMP * ABS(w(J*KK=1) - MPT(KN+NPS=1))	TOREJ

		14551		HOW (AN NOT) - HOW (AN NOT A GODODI / CLOCK NUMBER
	60 TO 542	10701		CONTINUES A CONTINUES A COROLE STORE
641	QIERE ARAGEUGERETI - BEIGNERESTII	10FC1		CC TO ADA
642	UU KIK NOU Z IY NUG 12 7 Even - Clooping Neuron (346, 517, 530	06401	410	WOUT AN AND A WOUT AN AND AN A WOROP ANA AND A
	IF (SIEPP + SLOPE (ANINGL)) BIGE OFFE DU	06401	614	$\frac{49}{1000000}$
010	CONTINUE.	04401	•	
		04 4 0 1		$\frac{1}{2} = \frac{1}{2} = \frac{1}{2} = \frac{1}{2} = \frac{1}{2} = \frac{1}{2}$
230	$\frac{1}{2} + \frac{1}{2} + \frac{1}$	04 491	433	TE (SLOPE(NEARSC) - F 1.0FT06 1 GO TO 640
	SUMALF E A SLUFEANNINSCELL = SLUFEANNINSCELL = SLUFEANNINSCELL	04481	123	PUT (NN, NSI) - PUT (NN, NSI -) - GRROPI / SLOPF (NN, NS
	$\frac{\partial P A H}{\partial r} = \frac{\partial P A H}$	04 AP1		ODVI (NELASI) = OPVI (NELASI +) + GOROPI
417	$[F \cup SFAPT + M] + SOBALF F ARC = ASC = A$	INFEI		
617		INFEI	64.0	WEVT (NN.NSL) = WEVT (NN.NSL-1) + WEROP (NN.1.SC) +
	$\begin{array}{cccccccccccccccccccccccccccccccccccc$	15551	··~ · ·	Abe (onenP))
Cananakia	LEED OF AN DISTING THE ALW PROVES SINCE THE DEFITIOUS DEFITIT	CN 21481	•	APVTINN-NSLL = OPVTINN-ASL-11
CNG	ALLO WE ALCOPTION THE ARE LOWED SINCE THE PRETENCE THE LECT	21401	F+====5AV	F THE REMAINING WET AND OPT IN WEVT AND OPVE ARRAYS
(Alfolia (MC Cladit Hende	16551	674	DO A25 NPC - NP. NPCT
	103101 - 0 TE 1 TEVELIS - ED. 1 - 00 TO 6086	18551	95.4	TE (ARSIMPUTION ANSL) - MPTINN AND()) - GT. WATOLJINA)) GC
		ISPEI	1	A2#
C	FREE THE ORDER OF POINTS OF ONE CURVE BACK TO THE STARTED OR	DERISFET	*	TF (ABS (GPVT (NN.NSL) - QPT (NN.NPC)) .GT. QNTOLJ (NN)) GC
G 1161		ISFFI	1	625
		INFFI		GO TO 626
	WOVT (NN. ADA) WPT (NN. NPR)	ISEET	625	CONTINUE
	OPVT (NN-NPA) # OPTINN-NPR)	ISFFI	626	NPCTL = NPCT = NPC
510	CONTINUE	ISFEI		IF (NPCTL) 9980: 627. 624
	DO SIS NPA = 1. NPCT	15FE1	628	D0 6628 NA # 1+ NPCTL
	WPYINNEPAL WPVT(NNENPAL	15FF1		WPVT(NN,NSLANA) # WPT(NN+NPC+NA)
	QPT(NN.NPA) = QPVT(NN.NPA)	15FE 1		QPVTINN+NSL+NA) = QPT(NN+NPC+NA)
515	CONTINUE	15FE1	6678	CONTINUE
~~~~	GO 10 6086	15FE1		NSL = KEL + NPCTL
C54V	F KPT AND GPT FROM POINT 1 TO POINT NP-1 FOR THIS CUNVE NN	21401	CSAV	VE THE NEW ADJUSTED OFW CURVE IN WPT & OPT ARRAYS
520	NPM = $NP = 1$	15FE1	627	DC 689 NP = ]+ NSL
	DO 618 NPVS # 1. NPH1	IOFE1		WPT(NN, NP) = WPVT(NN, NP)
	WPYT (NN. NPVS) = WPT (NN. NPVS)	lorel		QPT(NN+NP) = QPVT(NN+NP)
	QPVT (NN. NPVS) = QPT (NN. NPVS)	10FE1	¥29	CONTINUE
61 B	CONTINUE	loFE1		NPTTS(NN) = NSL
	1F ( NP15 .EC. 0 ) GO TO 633	10FF1		[FUS(J) = 1
	NP = NP + 1	LOFEL	6086	NPCT # APTTS(NN)
	GD TO 634	10FE1	CINT	TERPOLATE TO FIND PESISTANT STIFFNESS, AND LOAD VALUES FROM
CRFG	SEREFATE THE NONLINEAR SUPPOPT CURVE FOR THIS CURVE NN	10FE1		DO HONO NP # 2+ NPCT
633	HPVT(NN+NP) = H(J+KK+1)	10FE1		TF ( WPT(NA.1) ) 6091, 9991, 6092
	QPVT(NN+NP) = ( WPT(NN+NPS) - W(J+KK+1) ) + STEMP +	10FE1	6091	IF ( WIJ,KK) - WPT(NN,NP) ) 6093, 6096, 6090
1	QPT (NN NPS)	loFE1	6092	[F [ W(J_KK) = WPT(NN,NP) } 6090+ 6096+ 6097
634	00 519 NSCC = 1+ NSC1	10EE1	6090	CONTINUE
-	NST = NSCC + NP	lofel		NP = NPTISINN)
	IF ( WPT (NN+1) ) 620, 9991, 621	10FE1		CC 10 449P
620	WPVT(NN+NST) = WPVT(NN+NST+1) + WDROP(NN+NSCC)	lofE1	6093	1F ( w(U.*K) = wPT(NN+1) ) 6098+ 6111+ 6110
	QPVT(NN+NST) # OPVT(NN+NST=1) = QOROP(NN+NSCC)	LOFE1	6097	IF ( W(J,KK) - WPT(NN,1) ) 6110+ 6111+ 6098
	GO TO 619	lofel	6111	IF ( 10=+(J) .EQ. 1 ) GO TO 0196
621	WPVT(NN+NST) = WPVT(NN+NST+1) + #DROP(NN+NSCC)	10FE1		NP K NP + 1
	QPVT(NN+NST) = QPVT(NN+NST-1) + QDROP(NN+NSCC)	10FE1		60 17 6096
619	CONTINUE	10FE1	609A	KOFFC(NTT) = 1
	NSL # NSC + NP	LOFEL		
	IF ( WPT(NN+1) ) 622+ 9991+ 623	10FE1	6096	IF ( 105m(U) ,EQ, 1 ) GO TO 0196
622	IF & SLOPF (NR +NSC) +LE. 1.0E=06 ) GO TO 639	LOFE1	C194	GENT WORKLUS WETHOD I LOAD+SHMING TIEMATION HETHOD T

			WPV	TINA	1.N<	L) 1	∎ ₩P	VT ()	NN +N	5L-1	) +	ODR	OPL .	/ SLOP	EINN.	NSC.	}	10FE1
			OPV	T (NN	1+NS	L) :	∎ ŋP	VT ()	NN + N	SL-1	) -	QDR	OPL					10FE1
	GĆ	ΤŪ	62	4														LOFE1
639			WPV	t (NN	I.NS	L) 1	= ⊮P	VT ()	44.44	SL=1	) +	<b>WDR</b>	OPIN	NINSCI	-			15FE1
1							۵ê	5 ( 01	DROP	'L)								15FE1
			QPV	T (NN	4NS	L) (	N OP	V T (P	4N, #N	'SL-1	)							10FE1
	GC	10	62	4								_						10FE1
623	IF	(	SLO	PEIN	(K. 9 N	SC)	•LE	. 1.	.0E-	06 1	GO	TO	640					10FE1
			#PV	T {NN	1, NS	L) :	∎ wP	VT (P	11. • 1	·SL-1	) -	ONR	OPL .	/ SLOP	EINNA	~ * C	7	ICFE1
			OPV	T (NN	1115	L) 1	∎ QP	VT (1	4Nsh	SL-1	} •	008	OPL					10FE1
	GO	10	67	4														16FE1
640			₩P¥.	t (NN	1 <b>.</b> NS	L) 4	e wP	VIEN	VN + N	SL-1	) +	#DH	OP (N	V+P2C3	•			15FE1
1							٨B	\$ (00	DROP	11								15FE1
			OPV	T (NN	1.NS	L) i	I OP	VTO	NN + N	SL-1	)				_			10FE1
SAVE	. ⊺⊁	ŧΕ	PFM	AINN	ITNG	WPT	Γ ΔN	0 06	27 1	N WP	VI I	AND	QPVT	ARRAY	5			10FE1
674	Ĉ0	65	5 1	NPC	# N	P, M	PCT					_				• •		IOFE1
	1F	ţ	A85	(¥P\	17 (N	N+N!	SLI	- #6	PT (N	IN + NP	(C))	• G T	• WN	IDE T IN	IN 1	60	16	ISFE1
1			65 <u>e</u>													~ ~	• *	ISFEI
	1F	{	ABS	(CPV	/T {N	N. N.	5L)	- QE	PT (N	N + NP	¢11	• 6 1	• QN	INC J (N	KI 1	<b>G</b> C		15FE1
1			625															15FE1
	60	10	62	ŧ –														10551
625	COM	11	NUE	<b>.</b> .		•												10761
626			MPC		NP	CT •	NP	с.										10+21
	11		NPC	1	. 99	00+	677	. 62	Şμ									10-01
628	CO	66	28	ħA.	* 1	• •	2611											10761
			WPV	TINN	1.NS	Lehi		WP1		NPC	4 N A 1							10751
			QPV	IINN	403	L+N/		QP 1	I	INPC	eng;	'						10701
678	COP	11	NUF															10161
		~	NSL	5 1	SL.	• •	at the					0.01	100					10761
	11	•	NEN	_ADJ	1051	EC (	) • W	COM	** *	N PP	3 3	QP I	AHH	415			1	
027	Сe	50	9 1	ςμ <b>Ξ</b>	11	~ 21												10551
			MPT	(NN)	NPJ	E 1	DUT	INN	NP/									10751
			UP1	(NN)	KP7		14.61	[[]]	11.61									10551
¥58	COV	et 1	NUF															10551
			NPI	1315	1 1	•	SL.											18551
			160	5 ( ) )		1												ISFEI
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0.94	TE	÷	105	 	F	<u>م</u>		c 0 1	rn 6	196								06AP1
/////	15	ر ¹ ب	1000	115	UET	HOC	6	040-	- 504	TNG	TIFF	1105	-	FTHOP	,			06 AP1
	1214		S THUR	ີ່ພະ	- E 1	V L -	· •	い具い	- 36	* ****	- 16 T		··· ···					

	1 F	(NP	. t	τ.		P (ΞŢ.)	G	Ċ.	TO		519	8														21 A P 1
	•		SL	CPE	21	1 =	: (0.0	່																				214º1
			o T	21	=	۰.	0																					÷	ZIAP1
	60	TO	6	112	,																							;	214P1
6108			SL	OP	21		: 4	18	5	(Ċ	QF	7	(Nt	1	NP.	• 1)		٩Ø	71	NN+	NP))	1				21 AP1
0140						•					ć.	ŴF	7	(Nł	1.	NP.	٠î)	-	¥P	>T ()	NN+	NP)))			1	ZIAPI
•	1F	t	SI	nPl	21	۱.	LE		1.	- 0	F	0.6	• •	1 \$	iL.	OPI	ΞŻ	1		٥.	.0							;	214P1
			QT	21		οP	T	INF	(j. j	VP	1	•	้รเ	0	È	21	•	jw	PT	1 N	N + I	NPI							LOFE1
	TE	ł	NP		GΤ.	. 1	1) (ŝ'n.	т	ò.	61	1	2	-														OFEL
			Si.	nPI	22	1 .		51 0	P	E 2	1		2.	. 0															LOFEL
			ōī	21		01	12	1		Ζ.	0			-															LOFE1
			SL	0 PI	72	2 .	- (Ō.(6																				IOFE1
			ō1	22		۰.	. 0																						10FE1
	GO	11	6	11	2																								10FE1
6112			SL	ή PI	F 2 2	2 •	e ;	48	5	((QP	T	(N	••1	NP)	•	GP	T ((NN	, NP	-11)	1			1	21 A P 1
1											t	WP	7	(N	1.	NP)	-	٧P	T ((NN	٠NP	=1)))				21 AP1
	ŢF	(SL.	nPl	22	2,	,ι	Ε,	1	• 0	F •	0.6	5 1		۶LI	OPI	E 2	2		٥.	0								21 AP1
			QŢ	72		Q.F	7	(1; #	•	NP	- 1)	٠	S١	-0	PE	22	•	W	PT	r (Ni	N,N	°-1	}					10FE1
	IF	t	NP		LT.	• •	vP (CΤ)	6	0	TC) (51	13														ZIAPI
			SL	0 PI	E 27	۰ ۶	<u>۽</u> ۽	SL (CP	E ?	5	٠	2	• 0															21AP1
			ΟĮ	22	=	01	122	2 '	•	г.	0																		21491
6113			55	١J) :	= (55	(J))	•	ο.	5	٠	1	S	ΓO	PE	21			SLO	PE 2	2)	•	ĔS	M			15FE1
			Q1	IJ	1 :	= (21	ເມ)	•	0.	5	٠	(Q	12	1	•	Q 1	22	2)	•	E SP						ISPE1
	60	TC	6	n 5	K		-	~					-																15PE1
6110	IF	(10	5¥	(J	,	E.	۰.	0)	6	0	11	9 1	o 1 '	<i>41</i>													04401
	60	.т.	6	19	6				-					- 01				TE			104	мF	THO		•				064F1
CTAN1	sth		10U	011	10.2	NK		-00 2 e	le y	٠,	L0	AL O	,,	200		D I	-	10	DI	1.4.6	U N B I	NR-	11	1	2			:	21AP1
6197			56	n Pi	2		A (93	(2			N 11.	• • •	5	-	ž				N D P	11	ί.	ŝ				21401
1		,	-	0.01			5		1	۰÷		1 M		- C	0	DF	ຸ ້	•	ά.	à.	114 4		• ·	'	, ,				21401
	15	•	50	1.1	ς,	-'`		•	÷*.	•	5,	06	,	, ⁻	•	E SI	с м												15FE1
			33	10		- 2	37	2	í		1	ň	5	iNI	<u>ا</u> م	NP	•	٠	SI.	06	2=2		#PT	1N	6. e N	P)			ISFF1
			01	1,2	, ,	··· `	• •	.ي.	,		`		сы		.,		,							•			•		15FE1
1	60	10										· -	2,																DEAP)
	- i i			10	č,	we 1	tiet	ön																					064P1
4104	ົກດັ່	Ā	204	10	κi i		5	* 1	NP	Ċт																			26.001
0140	00		57	C 14 1	P a	= 1	۰Å؛	ŝ	t.	ĩ	Q۲	т	i Nî	v . I	N	; .		Qp	11	NN	V . N	L-1	1 1	1					26.001
1				.				-		i	WP	T	(NI		١Ē)		WP	T I	NN	No Ki	L-1	> >	;					26JU1
•	1F	C	٨B	s	()	STE	EMI	P,		SL.	0P	È.	IN	4.	LĴ.)		LE	•	1.	. 0E	-06)	GO	Ιo	62	97		26,011
6296	cor	NT I	I NU	F																									26001
PRI	VT C	986	1																										26101
	GÒ	TO) 9	99	9																								26JU1
6297	1 F	(×S	¥۲	J (I	NK.))	• E (с.	1)	(50	Ţ	2	62	98												26JU1
CSFT	APP	PR	1 × 1	¥ A	TE	11	[6]	E۸I	P	SP	H I	NC	; (t0!	۰s	T A	NT	F	06	. 0	DNE	-wA	YS	υp	¢۷				56001
	1F	(48	s ()	QP 1	T (P	٩N	• 1))		LE		1	• 01	-	12)	ુઉ	0	10	0.6	310							24JU1
			\$1		ΔĘ	85	ł	QI	ΡT	(N	Ν.	NL		1)	1	M.	ΡT	٤Ņ	N+	NL	1	, ,							26 JU1
	GO	T	16	29	۹.									,															26,001
6310			S [x	4	H S	(Q	PŤ	(N	N,	NL	- 1	1	W	۲Y	(N	N 1	NL	.)	1								24JU1
	60	10	6	24	<u> </u>				-	e -	n							r		, -					9 ,19	r.,			24 111
CSF1	4PF	pp	iX [мΔ	TĘ.	11	IN	L &	H	эP	rt]	N	, 1	CO	13	14	ni f		٥H		311	- 1	410		υŢ	C V			24 001
6298			SI		5	í ur	Έ.	4 NI	<u>, </u>	11		~			6.7														24 111
	TF	(- EF	٥ŝ	1.1	1.	۰ <u>۲</u>	4.	0	, 		д) 	11	_	1	-7	~	P 7	15		. 6.1			,					26 811
			51	E	A (45	ţ	ç	a		10	i Ni			1	-	ᇉ	PT	10	1 I (1	611	-1' -1)		1					24 101
1								•		r: 1		104.1	1.41		• •		-						'	,					- 901

6799			55	(2)	E	55	(ن)	٠	SI	•	E	5 M														- 2	#JU1
			ŵĉ.	=	- v	e Car	, K K	1		PT	UKP		P)													0	AP1
			SI	÷	ARC	. ((QP.	τŧ	ulu e	ND :	-	QF	тί	NN.	•KP	-1))	1							5	1401
1							(WP'	T (+	N.	ME	-	wP	T (NN.	+ N F	-1))	1							2	JAP]
	τF	t	¢1	• L	F.	1.1	0 E -	06)	\$1	-	G.	0													2	JAP1
	•		οċ	=	OPI	r (NI	N + N	PI	٠	wc		\$ĵ.	-													0	EAP1
			05	Ŧ	÷.	1.1		۰ (• •	s٣Ĩ		-														2	6 JU1
			o T	ĒΣ	Ŧ	n T	1.1)	· •	Ē	ar	-	45	3	٠	E SP	4										0	AAPI
	e Di		MÜ			÷.									-											1	SFF1
4564	1.6	1	170	- 6		n. I	n)	G	n '	τn	651	15														2	5JA1
Caractes'	τ ¹ τ/	۱Ì9	FF	TF	TL	FOI	ř i	ເັ	۵.	ST 6	0.	Ë e	UPP	OR	Ťι	UNL	OA.	DED)							2	SJA1
C	'nr	×٩	IDA	1	-	<u>.</u>	νp	á				-				-										2	5J41
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	- 601	. 11	N I H		· •	ir a	• •						-													2	5JA1
0000	20	+ ~		75																						2	5JA1
****	00		1 7	1/~																						1	SFEI
0203			1 7	- 60	`	` 1																				1	SFE1
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					- 1																					a	SFO
			A 14		2 :																					ō	SSEO
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	-		012	1) • - •	-																			ŏ	SSEO
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			VD.	-	25				<i>د</i> ی		<u></u>	57		1	- 1	HE 2		T	.11							ī	60
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2			۲D	-		<u>, , , , , , , , , , , , , , , , , , , </u>		F	6.11		c.		11	1	-)	HE 2		T	J+	1)						ī	0000
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(< a 1		- L I		-11	. /	· · ·	. /	-		* .	~ * *				*						•••	v

2	$DI(J=1) + DI(J) + HI + W(J=1+KK=2) + \{-CC + 2+(J=1+KK=2) + \{-CC + 2+(J=1+KK=2) + (-CC + 2+(J=1+KK=2) + (-CC + 2+(J=1+KK=2)) + (-CC + 2+(CC	00001
3	* (D1(J+1) + 4,0 * D1(J) + D1(J+1) 1 / HT + 2.0 *	10000
Ā	HITT = $CF(1)$ = $H(1,KK=2)$ + 1 + DC = 4.0 = ($DI(1)$	10000
5	D[1] = 1 + 1 + 1 + 1 + 1 + 1 + 1 + 1 + 1 + 1	10000
1	$\sum_{i=1}^{n} \sum_{j=1}^{n} \sum_{i=1}^{n} \sum_{j=1}^{n} \sum_{i$	21401
CCOMPLITE	RECHOSION OR CONTINUITY COFFES AT EACH STA	05550
7006	r = A A B C (a)	DECED
1000	E = AA = Diversion of Delivery AA = Classic ACC	GREED
	DENOM $\mathbf{x} \in \mathbf{z}$ and $\mathbf{y} \in \mathbf{z}$ and $\mathbf{z} \in (\mathbf{y} - \mathbf{z})$ and $\mathbf{z} \in (\mathbf{z} - \mathbf{z})$	05500
C	DENDE I OUIDE OUUDE OUIDE DENDE IE ZERO, DEAN OOFE NOT EXIST. D - A SETS UEFL - A.	000000
CNUTE IF I	NUM IS SENDE BEAM DOLD NOT FUISTE D # 0 3615 ACT = D.	05500
6005		05550
60 1		05560
0010	D = = 1.0 / DENOM	02320
6615		03220
	H(J) = P = (E = C(J-1) + 00)	05520
	$\Delta[J] = D = \{F \in \Delta[J+1] \in AA = A(J+2) \in FF\}$	05520
CCUNTPOL	RESET ROUTINES FOR SPECIFIED CUNDITIONS	USSED
	KEAN = KEA(N)	05580
IF	K = 3) 6019. 6019. 6018	10000
6018 IF (10P(J) .NE. 1) GO TO 6019	10000
	NS = NS + 1	10000
GA TI	0 6160	10000
6019 GO TI	0 (6060, 6020, 6030, 6020, 6050), KEYJ	055E 0
CRESET FO	R SPECIFIED DEFLECTION	055E0
6020	C(J) = 0.0	055E0
	B(J) = 0.0	055E0
	د) = ws(N\$)	055E0
TF L	KEYJ - 7) 6059. 6030. 6060	055E0
CRESET FOR	R SPECIFTED SLOPE AT NEXT STA	05560
6030	NTEMP = 0	05520
	стямр = C(J)	055E0
	ATFMP = B(J)	055E0
	ATEMP = A(J)	055E0
	C(1) # 1.0	05550
	P(J) = 0.0	OSSEO
	A(1) = HT2 + DHS(NS)	OSSEO
60 1	0 6040	055F0
C PESET FOI	P SPECIFIED SLOPE AT PRECEDING STATION	055f.0
6050	DREV = 1.0 / (1.0 - (BTEMP * B(J-1) + CTEMP = 1.0) *	OSSEO
,	D Z DIEMP 3	055FO
•	CREV = DREV # C()	055F0
	DEEV & DREV * (A/ I) + (BTEMP * ((J-1)) * D / DTEMP)	OSSEA
	ADEV & DREV # (ATH + 1 HT2 # DWS(NS) + ATEHP + BTENP	OSSED
,	• Atials + D / DTFMP)	05550
1	CLU # CREV	055F0
	DIN = DEV	OBSED
		05570
6460	NG W NSAT	ORCEA
4040 CONT	1920 - 1927 B. 1663 E.	05550
		08550
UUNFFULE 1		0,500
UU 0.	100 G = 07 05 0 1 G = 0 5	050EV
	L M F F D F U Non-Alt L WWALL	13.44
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	PHYLL/ HILLERRY	300000

-0100C0		₩([+KK) ± Δ([] + B([) + W([+]+KK) + C([) + w([+2+KK)	05520
10000	6100	CONTINUE	05520
+10000		W(2+KK) = 2.0 * W(3+KK) + W(4+KK)	USSED
0*100C8	6	W(MHRANK) = 2.0 = W(MHRANK) = W(MHRANK) IN T NUMBER OF STATIONS SVEREDING MANTHUM ALLOWADLE DEFLECTIONS	20550
05550		NU PREME AL STATIONS FROTEILLA MARTAN ACCORROLD DE COLUMN	20050
05550	(,===== д	NU MOMER (F STATIONS NOT CLUSED	18551
GESEO			28050
055F0		15 / NCVA	ZRDEN
OSSEO		CO 6150	ZADED
OSSED		TF (WMAX + ABS(W(J+KK))) 6155+ 6160+ 6160	2PCE0
055E0	6155	KMAX # KMAX + 1	28050
055E0	6160	IF (10CNT .GT. 0) GO TO 6150	15FF1
055E0		TF (ARS(W(J,KK) - WW(J)) - WTOL) 6150+ 6150+ 6165	15FE1
055E0	6165	KCAT(AIT) = KCNT(NIT) + 1	05 141
055E0	6150	CONTINUE	2PDE0
05SE0		IF (K .LT. 3) GC TO 6541	23 AP1
OSSEO		IF (NDCI .EA. 1) GO TO 6541	21491
10000	C=+TE	ST TO SEE IF THERE IS A SUP STA WHERE THE DEFLECTION IS OSCILL.	ATING
10000	c0u	FING THE ITERATION PROCESS	21AP1
10000		00 6540 NN # 1+ NNG	UGAPI
10000		J T ISTA(NN)	USAPI
05520		IF (ABS(W(J_KK) = WWW(J)) +01+ 1+02700 / 00 10 0040	21481
V55E0		TE (ABS(#(J+KR)) +LE+ #102 / 00 /0 0040	21401
075E0		The C BR2(h() - MURDC()) - 7 - 7	21401
1000LU		TE (MUMOC (), - 2) 4540, 4154, 6164	04AP1
05520	£144		21 AP1
04460	6540	CONTINUE	06 AP1
05570	0,140	IF (NITS . IT. 2) GO TO 6541	ZIAPI
055F0	CSF	TTING SWITCH INSWLUD FOR LOAD ITERATION METHOD NEXT ISERATION	21 4P1
055E0		CC 4542 NN # 1+ NNC	21 AP1
055E0		J # 1STA(NN)	21 AP1
05550		105x(J) # 1	214P1
OSSEO		NUMOC(J) = 0	SIVDI
055E0	6542	CONTINUE	21 4 P 1
055E0	6541	IF (10CNT .FQ. 0) GO TO 4151	15FE1
055f.0	C=+TE	ST TO SEE IF THE DEFLECTIONS COMPUTED FROM THE INITIAL SPHINGS	\$23AP1
OSSEO	C=1T	ERATION LEADS ARE WITHIN THE AUJUSTED HANGE OF THE GAR CURVE	21491
OSSEO		DD 6152 A # 1. NNC	15461
05520		U = 1514(N) 	10001
05560		$\frac{1}{10} + \frac{1}{10} $	16551
05510		$ \left\{ \begin{array}{cccc} \left\{ x \\ y \\ y \\ z	26 0.1
05550		TE (W/) KKAIN (E. 0.0) CO TO 6152	26.001
05550		TE (#(J)#KH() #[C% **0) HO (0 0142	26.001
05520	6749	TE (W/ 1-KK=1) -6E- 0+0) 60 TO 6152	26.001
05550	6149	1F (wPT(N+1)) 6153, 9991, 6154	15FF1
055E0	6151	TF (W(J+KK) .GT. W(J+KK+1)) GQ TO 6152	15651
055E0		W(J+KK) # W(-+KK+1) + 1+0E+06	15FE1
055E0		GO TO 6152	15FE1
05SE0	6154	1F (WIJ+KK) .LT. WIJ+KK-11) GO TO 6152	15FE1
13JA1		W(J+KK) = W(J+KK-1) - 1+0E-06	15FE1
JODEO	6152	CONTINUE	15FF.1

6151	IF (MONT .FO. 0) GO TO 6174 15FE1	
C	TALES ATAI ATAI T	E# 35
	IF (IT(SP .F0. 1) GO TO #178 10FE1	
	KMCNT(HTT) = KCNT(NTT) 15FE1	
	CO 6170 K = 14 MCNI 280E0	6430
	JM = JINS(N) + 4 ZADEQ	9173
	WM(NIT+N) = W(JM+KK) 280E0	
6170	CONTINUE	Ceessinit
	GO TO 6174 10FE1	
617B	DO 6179 N =]+ MCN1 10FE1	
	JM = JINS(N) + 4 IOFEI	
	WEELATER WEELATER WEELATER	
6179	CONTINUE 10FE1	6177
CTI	FST CLOSHRANCE 28DE0	
6174	IF (KMAX +NE+ 0) GO TO 8500 280E0	
	IF (KOFFC(NTT) .EQ. 0) GO TO 6274	
	IF (ITCSR .FQ. 0) PHINT 9984, NIT, KH3 15FE1	
	IF (ITCSR .FQ.)) PRINT 9994. NIT, KM3 15FE1	6176
P	PINT 9985 154P1	
	GO TO 9999 15AP1	
6274	IF (KCNT(NIT) +NE+ 0 3 GO TO 6173 25.JA1 25.JA1	PRIN
	1F (IOCNT .GT. 0) 60 TO 6175	PRIN
	IF (17CSP .FQ. 0) 60 TO 6230	5949 PRIN
CTI	EST TO SEE IF THE DEFLECTIONS COMPUTED FHOM THE NEW ACH. G-W CV 21AP1	PRIN
CA	RE CLOSED RACK ON THE OLD CV OR NOT. IF TEST USING THE GHEATEST 26JUL	5998
C5	TIFFNESS & ITERATION LOAD OF THE NEW GAW CV AND RECOMPOSE THE DEFEGUI	6000
	DO 6235 N = 1+ NNC ISFEI	
	J = ISTA(N) 15FE1	
	IF (IUS(J) .EQ. 0) GO TO 6235	
	NPCT = NPTTS(N) 15FE1	C+++-SINC
	IF (ISW(J) "EG+ 0) GO TO 6236 15FE1	
	ESM = 0.5	
	GO 10 6237 15FL1	
6236	FSH # 1.0 15FE1	
6237	IF (WPT(N,1)) 6240, 9991, 6244	
6240	IF (KSYMJ(N) .EG. 1) GO TO 6241	
	1F (KSYMU(N) ,EG, 0) 6N TO 6241 28JUI	6684
	GO TO 6235 26JU1	
6241	1F (W(J+KK) .GE. W(J+KK+)) GO TU 6235 26JU1	6594
	DO 6250 NP # 1+ NPCT (3PE)	
	IF (W(J+KK+1) +EQ, WPT(N+KP)) GO IO 6260 10FC1	6544
6250	CONTINUE	6595
	GO TO 6235	PRIN
6244	IF (KSYMJ(N) - LU, 1) GO JU 6245	_
		6598
		PRIN
6245	[F [W(J;KK)] [E, W(J;KK=]) / GU (U GESS)	
	UU 6200 NF # 13 NFC1 1761 1 00 70 (260 1755)	6=====867
		CHPHPETHE
0255		6608
4748	60 10 6735	3000
0200	51(1 - 51(1 - 51(1 - 51(1 - 55))) + 550 - (001(N+ND) + 51(00F(k+1) - 2140)	C=====CALC
	$\frac{1}{2} \left(\frac{1}{2} - \frac{1}{2} \right) = \frac{1}{2} \left(\frac{1}{2} + \frac{1}{2} - \frac{1}{2} \right) = \frac{1}{2} \left(\frac{1}{2} + \frac{1}{2} - \frac{1}{2} \right) = \frac{1}{2} \left(\frac{1}{2} + \frac{1}{2} - \frac{1}{2} \right) = \frac{1}{2} \left(\frac{1}{2} + \frac{1}{2} - \frac{1}{2} \right)$	CasesTHF
1	ALL COMPLETE A CINET	449R

	TACHT - TACHT - L	
15FE1	ADE CONTENE	12721
25041	BESS CUNITRE TO A 1 ON TO 1178	13721
10FE1	17 L LOCHI - EMA 8 7 80 10 81/2	10051
15FE1		13761
ZEDEO		38741
SEDEO		E DUAL
28060	$1 \in \{ K_{1} \in \{1\}, 3\} \in [0] \cap [0] \cap [1] \cap [0] $	1961
CADEO	Communitie (Fine For Starting Inc Second Stole of Thermitons	121871
IOFEI	UO BIT N = 1, NNL	19471
10FE1		15471
IOFEI	103#(J) = 0 NUMOCIN _ 3	12451
LOFEL		36 744
10561	SI// CONTINCE	ESUM1
ZEDEO		31101
CNDLO		5 8 4 8 4
LOFE!		10551
10721		24060
12761		31481
15451	17 (HUNI 420, 0) 00 10 3776	
28 (41	The state of the second state of the state o	28052
18651	PRINT VIGT RES Basing Astronomics II - C. MONT I	24050
18551	PRINT YEAR (JING(N)) N H IS MURL S	22050
DHW EV 21AP1	BATHA AND VIIS WENICUIST Batha VIIS KENICUIST	24050
FATEST 26 HIT	PRINI 723 (HEINII)733 A = 17 TURI /	18881
THE DEFRAUUT	1999 IF CRUITER B 1 00 10 01/3	34550
15FE1	TE CELT INC.	21481
15FE1	TE C N 101 80 1 0 00 70 4599	1.401
15FE1		13802
15FE1	CHARTER TANDENT MODILIES METHOD DTD NOT CLOSE, GO BACK USE LOAD IT.	HD.AP1
15FE1	NOT # 1	15AP1
15FE1		15AP1
15FE1	DO AARU N TI NC	15421
15FE1	Y # TRTA(N)	154P1
15FE1	TDAW(J) # 1	15AP1
26,001	NUMDC(J) = 0	15AP1
26001	6684 CONTINUE	15AP1
Senni	GO TO 6910	15AP1
26101	6599 IF (ITCSR .Eg. 1) GD TO 6594	15AP1
15761	NITT # NIT	86AP1
ISFEI	6544 IF (ITSW) 6595, 6595	86AP1
15721	6595 TF (NCVS .NE. 0) GD TO 6600	ZODEO
10161	PRINT 9979	29DE8
24 111	GD TO 9999	29DE0
16551	6598 IF (NCV8 .EQ. 0) GO TO 6175	29020
15FE1	PRINT 9978	29028
14551	GD TO 9999	29060
15725	CHANNESS OF THE FOR PRINTING THE ERROR MESSAGE OF HOT CLOSED HITHE	4 21AP1
15FE1	CTHE SPECIFIED NUMBER OF ITERATIONS	21AP1
15FE1	6600 IF (17C3A .EG. 0) KCLO8E(1) = 1	06AP1
21 AP1	IF (ITCSR EG. 1) KCLOBE(2) = 1	86AP1
PE (N.1) 21AP1	CHEMMARCALCULATE SLOPES, SHEARS OF THE BARS AND MOMENTS, SUP, REACTIONS A	r 15PE1
214P1	CHERRENTHE JOINTS FOR THE PREVIOUS TIME STA	15FE1
-	\$175 IF (K ,LT, 3) GO TO 8101	15FE1
	DO 8000 J = 3, MP5	85858
	Ω#(j) = { ~₩(J+]+KK+1) * W(J+KK+1)) / H	05580
----------	--	---------------------------
	HM(J) = F(J) + (W(J+1)KK+1) = 2+0 + V(J+KK+1) +	055E0
1	W (J+1+K1-1) 1 / HF2	05550
•	$TE \{ K = 3 \}$ BOCO, BOCO, BOCO, BOCO	21401
	$(1 + \alpha - \beta) + 2000 + 00000 + 00000 + 00000 + 00000 + 00000 + 00000 + 00000 + 00000 + 00000 + 00000 + 00000 + 00000 + 00000 + 00000 + 000000$	-2121401
C		21101
1	* #(J+1)RR*2) * #(J+1)RK1 * C10 = #(J+RK) *	210-1
2	W(J+1+RK)) / (HE2 = 2+0 = HT)	21441
8000	CONTINUE	055E0
	RM(2) = 0.0	055E0
	AN(MPA) = 0.0	05520
	CC 8108 J # 3+ MP5	055E0
	IF (K = 3) 8105, 8105, 8106	055E0
8105	DRM(J) = {₽₩(J=>} + ₽₩(J) / H = Ť(J) + { +#(J=>}+KK	-1)055E0
1	+ $w(1+kK+1) \rightarrow \mathcal{I}$	OSSEO
,	60 10 8107	OSSED
81.06	DBM(1) = (-BM(1-1) + BM(1) + Z + + (-5.5 + (-1)(+1)))	 10000
0100	$TT = \{0, 1, 2, 3, 3, 4, 7, 1, 3, 3, 4, 6, 1, 1, 1, 1, 1, 1, 1, 1, 1, 1, 1, 1, 1,$	10000
-	(1, 1, 2, 2, 3, 3, 3, 3, 3, 4, 3, 4, 3, 3, 3, 3, 3, 3, 3, 3, 3, 3, 3, 3, 3,	10000
5	 A CALCENT AND A CALCENT A CALCENTARY - 3 A CALCENTARY - 1 - 1 - 1 - 1 - 1 - 1 - 1 - 1 - 1 -	
3		• • • • • • • • •
4	= W(J=[+KK] + =(J+KK+) / (HE3 = H(- 2.0))	10000
5	+ DI(J) = { $-W(J=1+KK=2) + 2+0 = W(J+WK+2) =$	N 100C0
ő	j=l=KK+2) + w(j=l=KK) + 2×0 + N(j=KK) + W(j=1	•KK100C0
7)) / [HE3 + MT + 2+0]	10000
8107	KEYJ = KEY(J)	055E0
	1F (K +FQ+ =) 60 TO 8108	2) 4P 1
	1F (10P(J) .NE. 1) GO TO 8108	214P1
	GD TO 8140	214P1
8108	60 TO (8140, 8120, 8140, 8120, 8140), KEYJ	21421
8120	PEAC = d (BM(J=1) + 2.0 + BM(J) + BM(J+1)) / H	OSSEA
0120		3 04 141
	= (0 + 0) + (0 + 0) + (0 + 1) + (0	1 05650
2		1 05550
5	= H(J=1) = H(J)(J*K*1) = H(J=1) = H(J=2*K*1)	1 03500
A	7 + 4 + 0 + HEC = 3 + 3 + 100 - 100 + 10	U VSSE0
5	= M(JeKK+1) = I(JeT) = M(JeKK+1) + I(JeT) =	05560
6	W(J+1+KK+1) / / H	OSSEC
	IF (K = 3) F121, 8121, A122	055E0
H121	PEACT(J) = PEAC	055E0
	GC TO 8108	055E0
8122	REACT(J) # REAC + (QT(J+1) + QT(J+2)) * 0.5 * PHO(J)	• 10000
1	(W(J+KK+2) = 2+0 * W(J+KK+1) + W(J+KK) 1 /	10000
2	HTF2 + DF(J) + (-++(J+KK+2) + +(J+K+)) / (2.10000
ä	● HT1+ (D1(J+1) ● (-H(J-2+KK+2) + 2+0 ●	10000
4	W(1+1+KK=2) = W(1+KK=2) + W(1=2+KK) = 2.0 +	10000
2		10000
2	AND TRACT A TOTAL A CONTRACT TOTAL AND A CONTRACT	10000
2		10000
	$\mathbf{x}(\mathbf{J}^{-1}\mathbf{x}\mathbf{K}\mathbf{K}) = \mathbf{z}_{\mathbf{z}}\mathbf{U} = \mathbf{x}(\mathbf{J}^{-1}\mathbf{K}\mathbf{K}) + \mathbf{z}(\mathbf{J}^{-1}\mathbf{k}\mathbf{K}) + \mathbf{z}(\mathbf{J}^{-1}\mathbf{K}) + \mathbf{z}$	10000
8	$D_1(J+1) = (-W(J_1KN+2) + 2, 0 - K(J+1NK+2))$	10000
9	W(J+2+KK-7) * W(J+KK) + Z+O * W(J+1+KK) +	10000
۵	W(J+2+KK))) / (HE3 + HT + 2+0)	10000
	PEACT(J) = FEACT(J) = (0+5 = (TTIJ+1) + TTIJ+2)) =	10000
1	₩{J=1+KK=1} = 0.5 = { TT{J+}} + TT(J+2) } =	10000
7	w(J*KK=1) = 0.5 = (TT(J+1.1) + TT(J+1.2))	• 10000
3	W(J+KK-1) + 0.5 + (TT(J+1.1) + TT(J+1.2))	100C0
-		10000
		10000

05560	60 10 A100	055F0	
05550	PFACT(1) = -SKM(1) + W(1aKK+1) + QS(J)	214P1	
05550	BIAN CONTINUE	DESED	
21 AP1	CHARLE DELECTIONS FOR PLOTS ALONG THE TIME AX15	15FF1	
(J+KK-2)214P1		280F0	
• 21 AP1		290F0	
21481		15551	
OSSEC		29050	
055E0	TE () 4980, 6180, 6185	290F0	
05570	5195 TE ($h = 2$) 8317. 6190. 6190	06AP1	
OSSFO		29050	
055E0	$r = (1 \times 4 \times 15 \times 10^{-5})$. NE. TESTT 1 GO TO 6195	290F0	
-1.KK-11055E0	\mathbf{F} (YAXIS(NC9) .NE. TESTO) GO TO 6195	15FE1	
OSSEO	TE (TYSE(NCO) .NE. (STA) GC TO A195	29050	
OSSED		290E0	
J+11 + 100C0	KSAV(NPT) = ISTA	29050	
10000	KFYP(NPT) = 1	29050	
-2) + 10000	MPC(NPT) # MPS(NCg)	290E0	
- 2.0 10000	KYS(NPT) = 1	10FE1	
0 1 10000	1F (K FG. MTP4) GO TO 6195	15FF1	
2) - 10 10000	YINPTAKM13 # W(JakK)	10FE1	
#(J+1+KK100C0	NFRR # NFPR + 1	10FE1	
10000	6195 CONTINUE	29DE0	
OSSEO	IF (NESS .LF. 1) GO TO A180	15FE1	
214P1	PPINT 9995. KFPR. LSTA	29DE0	
214P1	GD TO 9999	29060	
214P1	6180 CONTINUE	29020	
21491	CHHFAVE MUMENTS FOR PLOTS ALONG THE TIME AXIS	15FE1	
H 055E0	D0 6280 U # 4+ MP4	15FE1	
3 4 H) QAJA1	NEAR = 0	15FE1	
J≠KK-1) 055E0	LSTA = J - A	15FE1	
(K=1)) 055E0	IF (NCT9) 9980+ 6280+ 6285	15FE1	
- T(J) 055E0	6285 DO 6290 NC9 = 1+ NCT9	15FE1	
1) * 05SE0	IF (XAXIS(NO9) .NE. TESTT) GO TO 6290	15FE1	
055E0	IF (YAXIS(NC9) .NE. TESTH) GO TO 6290	15FE1	
055E0	IF (IYSN(NCG) .NE. LSTA) GO TO 6290	15FE1	
455E0	NPT = NPT + 1	15FE1	
055E0	KSAV (NPT) = LSTA	15FE1	
-D(J) * 100C0	KFYP(NPT) #]	15FE]	
1 1 10000	MPC(NPT) = MPS(NC9)	ISFE1	
/ 1 2.10000	KY5(NPT) = 2	LOFEI	
• 100C0	IF (K 11- 3) 60 10 6240	15FE1	
10000	$A(V \square I + K \square S) = D \square (2)$	10461	
10000	NERA = AFRE + 1	15Ft]	
**-21 +100C0	6290 CONTINUE	15Ft1	
+ 100C0	IF (NEPA .LF. 1) GO TO APRO		
(-2) - 10000	PRINT 9995, KEPP, LSTA	45FE1	
• 10000	60 TO 9999	17751	
10000	OPPO CUNTINUE	10551	
2 10000	CTTTTTSAVE REFLECTIONS FOR PLUTS ALONG THE REAM AND	15FE 1 2005a	
	IF (NUTY) 99800+ 6609+ 6205	20000	
2) 1 - 10000	0205 IF (K = 3) 0200, 0210, 0212	20054	
2) 1 - 10000	6210 NPH # NPT	ENGE 0	
10000	6215 NERR # 1	C 907.0	

	UU 6220 NC9 # 1, NC17	
	IF (XAX;S(NF9) .KE (ESTR) 00 10 6220	
	[F (TAXIS(NEW) .NE. (ESH) / GO TO BELU	
	[F (ITSWINCG) +NE+ KM3) 60 10 8240	
	NPH = NPH • 1	
	$RETP(NPP) = \mathcal{L}$	
	MPD(NPH) = MPS(NC9)	
	RTS(RPH) = 1	
	UU 6225 NPP = 14 MP3	
	NEELE PEELE (NDDD, KK)	
(CONTINUE	
0225	NEDD + LEDD + 1	
4 3 3 4	CONTINUE	
9220	TE (NERR . LE. 1) GO TO 6295	
	DDINT 9004. NEPD. KHR	
	GO TO 9999	
(-SAVE MOMENTS ALONG THE BEAM ANTS FOR PLOTS	
6295	NFRR # 0	
,	DO 6300 NC9 # 1, NCT9	
	IF (XAXIS(NC9) .NE. TESTA) GO TO 6300	
	IF (YAXIS(NC9) .NE. TESTH) GO TO 6300	
	IF (IYSW(NC9) .NE. KM4) GO TO 6300	
	NPRT = NPRT + 1	
	KSAVT(NPBT) = KM4	
	KEYRT(NPRT) = 2	
	MPOT(NPAT) = MPS(NC9)	
	KYST(NPAT) = 2	
	DO 6305 NPP = 1, MP3	
	NPP] = NPP • 2	
	YT(NPRT,NPP) = RM(NPPI)	
6305	CONTINUE	
	NERF = NERF + 1	
6300	CONTINUE	
	IF (NERH . LF. 1) ON TO 6200	
	PRTNT 9996+ NFPR+ KM4	
	GU 10 7999 TE (K) T = 5 CC TO 0317	
0200	17 (K (L), 3) UL (17 H31/ 17 (TDC) 0.45, 8335, 8103	
6 -	IT & IT > 3 81029 53639 8102 	
0107	-PRINT REJULTS	
0102		
	CO TO 8205	
8200	TE (K .GT. 4) GC TO 8201	
02.00	DRINT 915	
8201	PRINT 916. KM4	
8205	PRINT 901	
~ ~ ~ 7	1F (1P5) H200+ 8325+ H320	
8300	IF (K .LE. 4) GC TO 8320	
0.500	IF (KPC .EQ. KP54) GO TO 8320	
	IF (MONS . EC. 0) GO TO 8325	
c	-PRINT MONITOR STATIONS RESULTS	
-	00 P340 J = 1. HCNS	

29DE0 29DE0 29DE0 115FE0 29DE0 105FE0 29DE0 15FE1 15FE

LPS(()) + 4 ()) + 510 ⊑	055E0
71 - 105(1)	055E0
	05550
	05550
PRINT 922, (WUSIA), LHM(JSIA)	02560
PRINT 921. JPS(J), X, W(JSTA.KK=1), AM(JSIA), REALT(JSIA)	05520
B340 CONTINUE	055E0
GO TO 8325	290E0
C	05550
	05550
	05550
71 = JSTA	V552 U
x ∎ Z1 ↔ ⊢	055E0
PRINT 921, JSTA, X+ W(3+KK+), RM(3), REACT(3)	055E0
CO 8210 J ≖ 4∍MP5	055E0
	055F0
	05550
ZI = 351A	05500
X # /1 + H	05560
PRINT 922, DW(J) + DBM(J)	05SED
PRINT 921, JSTA, X. W(J,KK=1), HP(J), REACT(J)	055E0
B210 CONTINUE	055E0
	055F0
8-25 TE (TEL) 0324, 0315, 0326	290En
	04 141
E326 IF (K,EG, PIPA) GO TO B319	06141
IF (MONI .EO. 0) GO TO 8317	ZE JA1
CPRINT ITERATION PONITOR DATA	06JA1
PRINT 916. KH3	06JA1
DETNT 923. (ITMS(N). N # 3. MONI)	06JA1
	10FE1
	1651
	30050
PHINT 925. (WM(NIN). N # 1. MONI)	29020
B316 CONTINUE	29060
IF (ITCSP •FQ• 0) GO TO A317	10FE1
PRINT 9977	10FE1
EC 8330 NI # 1+ NIT	10FE1
	10FF1
	10551
PHINT 9255 (#U(KIEN) + N # IE #UNI)	10751
8330 CONTINUE	IGPEI
8317 IF (KCLOSE(1) .EG. 0 .AND. KCLOSE(2) .EG. 0) GO TY 8318	06AP1
IF (KCLOSE(1) +EG+ 1) PRINT 9986	06AP1
1F (KCLOSE(2) .FG. 1) PRINT 9987	06AP1
DDINT GGRE	154P1
C0 10 0008	15AP1
	04 4 0 1
	06461
C = + RESET W(J = L) = W(J = RK = 1) AND W(J = 2) = W(J = RK = AND RK = L	0555.0
8315 CO P214 J = 1+ MF7	10000
w(j+1) = w(j+kK-1)	055E0
₩(I+7) a ₩(I+KK)	055E0
BOILS CONTINUE	055F0
	05550
	455EU
7000 CONTINUE	07760
CPLOTTING CURVES	USSED
IF (NPAT .EQ. 0) GO TO 8500	15FE1
00 8600 K = 1+ KPAT	15FF1
KSAV (NPR+N) & KSAVT (N)	15FE1
$K_{YP} (N_{P} o e h) = K_{YP} v f (h)$	15FF1
CELENCERTSA - NETELANA	

	_	
	MPO(NPB+N) = MPOT(N)	15FE1
	KYSINPRAN) # KYST(N)	15FE1
	00 8605 NN ± 1. MP3	15FE1
	$Y(NPP+N_{*}NN) = YT(N_{*}NN)$	15FE1
8605	CONTINUE	15FE1
8600	CONTINUE	15FE1
	NPR # NPB + NPBT	15FE1
8500	IF (KMAX +EQ. 0) GO TO 8501	290E0
	PRINT 9983. KMAX. KM3	15FE1
	PRINT 9985	29060
	60 10 9949	29060
8501	IF (IPS) 8502+ 8503+ 8502	15FE1
8503	PPINT 970	15FE1
P502	IF (NCT9) 9920. 1010. 7050	055E0
7050	IF (MOP) 8312+ 8314+ 8313	17580
8312	PRINT 971	175E0
	GC TO 7040	17560
8314	PRINT 973	175E0
	GG TO 7040	175E0
8313	PRINT 972	17580
7040	YM(1) = 0.0	15FE1
	YM(2) = 0.0	15FE1
	NAC = 0	055E0
	NA =]	055E0
	195 = 0	055E0
	00 7065 NP # 1: NP8	055E0
	NAC = NAC + 1	055E0
	[F (NAC .EQ. 1) UN] = 0	25JA1
	1F (KEYP(AP) .EO. 1) GO TO 7051	175E0
	IF (KEYP(NP) .EG. 2) IDX(NAC) = HP3	05560
	KPS = 2 + KYS(NP)	lÓFEI
	KS(NAC) # KSAV(NP)	05 SE 0
	GO TO 7052	05580
7051	IDX(NAC) # MTP2	055E0
	KS(NAC) = KSAV(NP)	055E0
	KPS = KyS(NP)	10FE1
7052	HAXI = IDX(NAC)	05560
	[F (MOP) 7250+ 7255+ 7250	15661
7250	IF (NP .NE, NA) GO TO 7255	15FE1
C	-SAVE TIME OR REAM STAS FOR TITLES ON THE PLOT	15FE1
7158	14 # NA + NP + 1	175E0
	15(IA) = KSAV(NA)	175€0
	IF (MPO(NA) .EQ. 0) GO TO 7159	17560
	NA = NA + 1	175E0
	GO TO 7158	175E0
7159	NA # NA +]	175EQ
	155(1) = NPRCB	10692
	1F (KEYP(NP) .E0. 1) GO TO 7059	175E0
	ID(2) = 10HTIME STA =	17550
	IF (KPS .EQ, 3) ISS(2) = 5H DVA	Senni
	IF (KPS .FQ, 4) TSS(2) # 5H HVA	
	FNCODE (10+ 4+ ID(1)) ISS	26JU1
	GO TO 7079	175E0
7659	ID(2) = 10HBEAM STA #	175E 0

	1F (KPS .EQ. 1) ISS(2) = 54 DVT	26.091
	IFI	KPS .FG. 2) (55(2) = 5H HVT	26301
	-	FNCOFF (10, 4, 10(1)) 155	26.JU1
7079			175E0
		11 • 2 • TAD2	17580
	15 /	14 FG 17 1 GG TO 2078	175F0
	11 1		175F 0
			ITSEA
			17550
7 . 7 .			16557
1018			19854
	D0 /	076 N = 1+ 1A2	17520
		NP2 = N + 2	17520
		N # 2 + f. = 1	ITSEU
		IS<(1) = IS(KN)	175E0
		155(2) = I5(N+1)	17560
	1F (N .KE. 142) GO TO 7077	175E0
	1F (105 .EQ. 1 1 GO TO 7041	17560
7077		FNCCCE (10+ 5+ 10(NP2)) ISS	175E0
	GO T	0 7076	175E0
7081		ENCOPE (10+ 6+ ID(NP2)) ISS	175E0
7076	CONT	INUF	175E0
	-	195 - 0	175E0
		MC # 1 42 + 2 + 2 + 10	175E0
c	-SAVE THE	VERTICAL VALLES OF THE ACCUMULATED PLOTS IN AN ONE-DIMEN	SIGNAL
Č++	-ARRAY OF	YP	5) V b J
7766	00.7	060 NT # 19 MAXT	15FE1
		YP(.:K1+KT) = Y(NP+NT)	055E0
7060	CONT	INUE	OSSED
	TEL	MPO(NP) -FQ- 1 3 60 TO 7090	055E 0
	TEL	MPO(1)P1 -NF. 0 3 60 TO 9990	055F0
	TE C	MOR 3 7053+ 7056+ 7053	17SEO
·	SET SCAL	F FOR THIS FRANK OF ACCUMULATED PLOTS BY MICROFILM OF BAL	L-POINT
C====	DAPES PI	ATTING METHOD	KIAP1
7453		NDD E MAXT	IRFFI
.0.23	DO 7		ISFEI
	16 1	$\nabla H(1) = A^{-1} - \nabla D(N) + \nabla H(1) = \nabla D(N)$	16551
	11	$\{m_1\}_{i=1}^{i} = \{r_i \in \{i, j\} \mid i \in \{1, j\} = \{r_i \in \{i\}\} \}$	ISFEI
1	10.00	$1 = \{g\}$ at $1 = 1 = (n_f)$ f $T = (g)$ $T = (n_f)$	10001
1500	LUNI		16551
			Jerei
		3M(2) = MPA(= 2	19751
		1R0((* 1	1 SPE I
			15PC1
			15761
	CALL ZUT	1 C XH, YH 2, 10 J	13751
		LOP = LOPI	1551
	DO 7	OSH J # LA MAXI	17520
		xF() = J = Z	V55E.0
7058	CONT	INUE	OSSEO
7058	CONT	INUE XF(MAXI+1) = XM(3)	055E0 15FE1
7058	CONT	TNUE XF(MAXI+1) = XM(3) XF(MAXI+2) = XM(4)	055E0 15FE1 15FE1
7058 C	CONT	INUE XF(MAXI+1) = XM(3) XF(MAXI+2) = XM(4) S FPAME OF ACCUMULATED PLOTS BY MICHOFILM OR BALL-POINT P	055E0 15FE1 15FE1 2PER
7058 C+	CONT -PLOT THI -PLOTTING	THUE SF(HAXI+1) = XM(3) SF(HAXI+1) = XM(4) S FDAME OF ACCUMULATED PLOTS BY MICHOFILM OR BALL-POINT P METHOD	055E0 15FE1 15FE1 2PER 21AP1
7058 C	CONT -PLOT THI -PLOTTING DO 7	INUE XF(MAXI+1) = XM(3) XF(MAXI+1) = XM(4) S FDAME OF ACCUMULATED PLOTS BY MICHOFILM OR BALL=POINT P METHOD 265 N = 1+ NAC	055E0 15FE1 15FE1 2PER 21AP1 15FE1

	DO 7270 NI # 1. MAXT	15FE1
	y p(kT) = y p(NN+NT)	15FE1
7270	CONTINUE	15FE1
1210	$\nabla H(1) = \nabla P(MAXIAI)$	15FE1
	VM(3) = VD(MAXIAO)	15FE1
		1SEE1
	THIMMALED W THIAS	15861
	t = m (r)	18551
	$\frac{1}{1} \frac{1}{1} \frac{1}$	18661
		18551
	IF UN LEGANACI INCLUT	18861
1275	CALL 20T 1 / AF, YP. PAAL, IU I	ISPEI
	Ab (Waxi4b) = Ab(1)	19701
	Ab(hvx1+5) = Aw(5)	13751
T265	CONTINUE	137 2 1
	$AW(1) = 0^{+}0^{-}$	13461
	YM(2) = 0.0	15761
	NAC = 0	ISFE1
	GO TO 7065	17500
C	PLOT THIS FRAME OF ACCUMULATED PLOTS BY PRINTER PLOTTING METHOD	21401
7854	00 7080 J = 1+ MAXI	15FE1
	S = L = 2	055E 0
7080	CONTINUE	055E0
• • •	PRINT 11	055E0
	PRINT 1	055E0
	PRINT 13+ (4N1(N)+ N = 1+ 32 }	05580
	DRINT 16. NDDOR. (ANZINI N # 14 14)	055E0
	1F (KPS _EQ. 1) PRINT 911	05520
	TE (KPS .EQ. 2) PRINT 919	05580
	TE (KDE - FO. 3) PRINT 929	lorel
	TE (KPE - FO. 4) PRINT 939	LOFEL
		1SFE1
	ADTAL DOTA	15FE1
		15FE1
7.04		055E0
1004		05550
7.00	PRINT TICE KIN STRUKTIT KSKKI	05580
1692		DESEA
	PRINT 713 Only PRIOT O. (. MAR, YD, TDY, TX.)	OSSEO
	CALL SPECI O CRACE THE INAL IN .	OSSEA
		OSSED
	60 10 7065	OBSEC
7090	JNI - JNI - IDAINACI	05550
7065	CONTINUE	16551
	PRINT 11	16861
	PRINT 16+ NPROS, [AN2(N); N # 1* 1*]	08550
	CALL TIC TOC (4)	00000
	60 10 1010	00000
9990	PRINT 9998	05520
	GO_TO_9999	16551
9991	BRINI AAUT	10721
	GO TO 9999	IUPE1
9980	PRINT 980	05560
9999	CONTINUE	0552.0
с	END OF DRC5	05 JA1
	END	055E0

.

	SURROUTINE INTERPS (MP7. NCT. IN1. IND. KR2. 7N. Z. ISH. KSW)	055FD
	DIMENSION (N) (1300), (N) (100), (N) (100), (100)	1 DEFA
0	PERMATE CHAILED FY GREELED FY REVISION A COURT AND AND A COURT ASSAULT OF	TANKED
400	FORMAT (7740H ERROR SIND W STATIONS NOT IN ODER)	14415
440	FORMAT (77-34 UNDESTGNATED FRED CIDD IN SUBRUUTINE)	14413
9.9	FURMAI (7746H ERHOR KON_FERO DATA REYOND END OF REAM)	14MY5
	DO 1603 J = 1, MP7	15550
	2() * 0.0	26MP5
16n3	CONTINUE	26495
	∆5M == 15H	12425
	ASM = ASM / Z_	09405
	на: ч на; г су м = 407 7	-5eca
		12145
	RPL W U	12043
	AF(NC) = 1 ; [0/07]6047160471604	20361
1004	DU 1575 NC = 1: NCT	20547
	1F (KH1) 169P. 1605, 1407	10229
1692	NC1 = NC	12 J45
	UV = JN1(NC1) + LSM	12405
	IF (KR2(NC)) 1698, 1607, 1670	10FF6
1607	IF (JN2(NC) - M) 1609. 1609. 1619	24FE6
1649	JSZ = JNZ(NC)	10FF6
	7N5 = 7N(NC)	21FF6
	90 TO 1619	10FF6
1461	52 m M	INFEA
1011	$705 = 7010055 \pm 7$ Turnes = Thinests 1 + 1 (2 = 10 + 1 CM)	10.00
,	$\mathbf{x}_{i} = \mathbf{x}_{i} + \mathbf{x}_{i} $	10406
	$1 \qquad 7 \{ \text{ JNZ}\{N_{i}\} \neq \text{ JV} + \text{ LOM } \}$	10446
	RASS = RSW(NC)	21566
	GO 10 (1013, 1615, 1617, 1617), KASS	12600
1913	IF (JN1(NC) = M) 1619, 1619, 1670	10556
1675	IF (JN1(NC1) - H) 1619, 16 <u>7</u> 9, 1670	10556
1617	IF (KSW(NC1) - 2) 1618, 1615, 1618	09426
1618	IF (JN2(NC1) - M) 1619, 1679, 1670	(19MP6
1619	4 + VL = EL	10FE6
- /	4 → \$ي = \$ر	10526
	DENUM # J2 w J1 + 1 4M	12405
	JINCR = 1	12,145
	FSM m 1.0	12.145
		12.145
	IF (DENDU) 1695, 1620, 1630	12 145
		12185
1030		10,000
		12345
	ir (J2 + J1) 1651+ 1630+ 1630	NOMRE
1010	DO 1050 J # J1+ J2+ JINCB	12 143
	F = J	19495
		09495
	DIFF # F + F1 + 45M	094 05
	PART = OTFF / DENOM	12345
	2(J) = Z(J) + (ZN(NC1) + PART + (ZNS + ZN(NC1) 1) + ESH	10456
1640	CONTINUE	12 145
1651	IF (LSM) 1698+ 1652. 1460	BHRR
1642	IF (ISW) 1698. 1660. 1655	19405
6.5	JINCR = 12 - 13	12 145
		17 185
	ETT = -U.3 754 - A	17 140
		12343
	a0 10 1030	14045

1660	1F (KR2(1;C)) 1698. 1678. 1665	12,145
1645	W = JNZ(NC) + 15M	12405
1670	*61 = K ^R 2(NC)	12,145
	NCI = NC	12345
1675	CONTINUE	12.145
16+4	CONTINUE	215F66
	MP5 = MP7 - 2	05540
	MP6 = MP7 - 1	655g0
C	TEST FOR DATA ERRONEDUSLY STARED BEYAND ENDS OF REAL REAM	ORVOR
	ZCK = Z(1) + Z(2) + 7(3) + 7(4P5) + Z(MP6) + 7(MP7)	^5 <f0< td=""></f0<>
	IF (7CK) 1699, 1676, 1499	14445
1675	RETURI	14445
16-5	PRINT 905	14445
•	GO TO 1799	^5J467
1628	PRTNT 904	14475
	60 TO 1749	°5.1467
1699	PRINT 904	14475
17-9	CONTINUE	95 1647
	ENO	12 145

	SUAROUTINE TIC TOC (J)	240066
C===1	-+ TIC TOC (1) = COMPLIE TIME	^ 840 B
č	TTC TOC (2) = FLAPSED CPU TTME	ABMER
c	TIC TOC (3) . TIME FOR THIS PROBLEM	n BHPB
ř.	TIC TOC (4) = TIME FOR THIS PROBLEM AND FLAPSED CPU TIME	AMPR
~ · ·	O FORMAT //// JULI SHEL APSED COLL TIME & IS. OH WINUTESF9.3. AH SECONDS	1 255F66
	FORMAT ////JOAISHCOMPTLE TIME = . TS. AH HINHTES. F9. 3+ AH SECONDS	1 255566
	FORMATI /// 30 26 HTIME FOR THIS PROBLEM # . 15.8H MINUTES. F9.3.	255F66
	1 BH SECONDS 1	255E66
	T = .1 - 2	240066
	1F(T-1) 40,30,30	255566
31		255F66
	G CALL SECOND (F)	255F66
	III = F	755F66
	II = III / 60	255E66
	F12 = F = T1*60	255F66
	1F(I) 50+70.60	255566
51	n PRINT 11+ 11+F12	25SE66
	GO TO 990	25SE66
61	6 F13 = F - F14	255E66
	12 = F13 / 60	255E66
	FI3 = FI3 - 12#60	255566
	PRINT 12+ 12+ F13	255866
	1F(I-1) 990.990.70	255F66
70	0 PRINT 10+ 11+F12	225560
99(D CONTINUE	25 SE66
	RETURN	255566
	END	25SE66

	SUR	OUTINE SPLOT 9 (NPLT, X	, ID*• IX)	11JE0	_
С		THE LATEST REVISION NATA	FOR THIS POUTINE IS 11 JUN /0	PEVISED	55
	COMMON / SPLOT / WIDTH: SMALL, RIG, SYMB			11JF0	
	DIM	INSTON IDX(1) . X(1) . IX(1) . SPACF(A2) . SYMB(10)	11JF0	
	INŢI	GER WIDTH		14JE0	
	DAI	PACE / 62+1H / . SYMA	/ 1HT_/ + RLANK / 1H_/	11.JE0	
C.	******	THIS ROUTINE SUPERIMPOSE	S UP TO 10 PLOTS PER FRAME	ngJF0	
ç		THE FIRST PLOT SHOULD BE	THE LONGEST (MOST POINTS)	05JE0	
С		THE PAPER SHOULD BE POST	TIUNED PROPERLY AND ALL	10000	. •
С		HEADINGS PRINTED BEFORE	ALLING.	10009	50
ç	****	INPUT -NPLT. THE NUMBER	NE PLOIS FOR THIS FRAME	P4.IE0	
C .		IDA, ARMAY CONTA	INING LENGTH OF THE RESPECTIVE PLOTS	IN JEU	
C		IDX(1) = NU	WAEN OF POINTS IN FLAST PLOTE FIC.	∧ 4 J ⊭n	
C		A STARTING AN	TRESS OF THE FIRST PLOT	04JE0	
č		- NOTE - TH	PLATS MUST RE STORED CONTIGNUSLY	D T JEU	45
C C		IX, INDEPENDENT	VARTABLE (INDEXING)	11,000	
Č,		WIDTH WIDTH OF PLO	AT LESS THAN 63 7	14009	
č		STALL UNER LIMIT	OF VALUES TO BE CONSIDERED	11 150	70
č		OUTDUT NO COTUS VIEW	OF VALUES IN HE CONSIDERED	11000	
č	••••	CALL THE POURTON	THE WALLES ARE DRINTED	10009	
č		AND PLOTTED VE	E - THE VALUES ARE ERIVIED	10009	
č		THE VALUES PRINTED ARE T	HILACCY.	DE IEO	
č		TE HIDTH IS COLATED THAN	O THE YEAVIE () IS DIOTIED	19.1E0	
ř		WITH THE APPROPRIATE AD I	ISTMENTS IN SCALE MADER IF NECCESARY	04.100	
ř		TE WIDTH IS LESS THAN A	THE WANTE IS NOT PLOTTED AND THE	AA IEO	
ř		ROUTINE EUNCTIONS HIST	WE SPLAT T	04.150	
č		FRANK I FNORFS - PAY 1		11.JF0	A5
C	12 E00	AT (4x . 14 . 3x E10 . 3. 3. 4		11.100	88
		NENO = O		04.150	
		MMAK = 0		04.1F0	90
		TMSK = 1		11.JF0	1.00
		DO 30 I = 1. NPLT		14JF0	990
		IF (MMAX.LT.IDX(I))	MMAX = IDX(I)	n4JF0	С
		NENU = NEND + IOX (I		n4JE0	
	30	CONTINUE		04JE0	
		IF (NEND.LE.0)	60 TO 996	11009	
		IWIDE # WIDTH		∩4_IF0	
		IAX = 0		04JE0	
		IF (IWIDE+LT.0)	IAX = 1	r4JE0	
		IF (IWIDE+LT.0)	IWIDE = -IWIDE	n4JE0	
		IOTA = IWIDE / 2		11JE0	
		DHEGA = X(1)		10009	
		THETA = $X(1)$		10009	
		IF (NEND.EG.1)	GO TO 55	11009	
		DO 50 I = 2, NEND		10009	
		IF (OMEGA+LT.X(I))	OMEGA = X(I)	10009	
		IF (THEIA+GT.X(1))	THETA = X(I)	10009	
	50	CONTINUE	0 -0	LIJEO	
		IF (SMALL.GE,RIG)	QU 10 53	TOJEO	
		IF (UMEGA.GT.RIG)	OMEGA . BIG	NUJEO	
		IF (THEIA.LT.SHALL)	THETA = SMALL	NBUEU	
	53	AF (1AA+E"+1)		I JEU	
		AF (IME'A+GI+Q+)	TMFTA = 0.	n + JF ()	

	$IF (OMEGA+LT_0+) OMEGA = 0$	n4 IE0
5	IF (OMEGA EQ. THETA) GO TO 60	11N09
,-	SIGMA = (IWIDE = 1) \angle (OMEGA = THETA)	04JE0
	IF (IAX+FQ.1 . OR. NEND-FQ.1) GO TO 60	11JE0
	BETA = 1 SIGMA + THETA	11 JE 0
	IOTA = BETA	04JF0
	IF ($(8ETA = TOTA)$, GF, 5) $TOTA = TOTA + 1$	04 JE0
	ISKX = TOTA	11 JE0
	IMSK = ISKX	11JE0
50	DO 100 I = 1, MMAX	11JE0
	IF (OMEGA.NE.THETA) GO TO 45	11 JE 0
	INSK = INTA	11 JE0
	SPACE(IOTA) = SYUR(=)	11 JE 0
	GO TO BH	11 JEO
45	$II \times X = 0$	11JF0
	IF (IAX+EQ+1) GO TO 70	11 JE0
	SPACE(ISKX) = SYMA	11JE0
70	D ^O 85 N ^P 1, N ^P LT	^4JE0
	IF $(NP+GT+1)$ IIXX = TDX $(NP-1)$ + IIXX	n4JE0
	IF (I.GT.IDX(NP)) GO TO 85	04JF0
	IF (X(I+IIXX)+LT_THFTA) GO TO 85	11 150
	IF (X(1+I1XX)+GT+OMEGA) GO TO A5	TIJEO
	BETA # SIGMA # (X(T+ITXX) + THETA) + 1+	04 JE0
	1CTA = BFTA	10009
	IF ((BETA = TOTA).GF.n.S) IOTA = IOTA + 1	10009
	IF (IOTA+GT+IMSK) IMSK = TOTA	11 1EO
	SPACE(10TA) = SYMR(NP)	11 JE0
45	CONTINUE	ne JEO
A8	PRINT 12+ IX(I)+ X(I)+ (SPACE(L)+ L=1+IMSK)	11 JE0
	$00 \ 40 \ J = 1, \ 62$	1 I JFO
90	SPACE(J) = RLANK	11.JE0
100	CONTINUE	10009
990	RETURN	10009
С	END SPLOT 9	10.160
	END	

ENU	366	01	
END			

SUAROUTINE ZOT 1 (XF+ YF+ NP+ TD)	11JE0
C • • • • THE LATEST REVISION NATA FOR THIS POULTINE IS 11 JUN 70	PEVISEN
COMMON / ZOT / LOP+ MC+ TROLL+ MOP	11.000
DIMENSTON XF(1) + YF(1) + ID(1)	09160
DATA INC: 19, X: Y: XL: Y: X0; Y0 / 1:12:0:+0:+9:+8:+0.+90: /	va7£0
DATA 171+ 172 / -1+.0 /	r.9JF0
C XF - ARRAY CONTATNING THE X - COORDINATES	0.976.0
C YF - ARRAY CONTAINING THE Y - COORDINATES	09150
C NP = NUMBER OF POINTS TO BE PLOTTED	14760
C LOP - LINE OR POINT PLOT OPTION	19,150
C = 0 , LINE PLOT	09.000
C -J + POINT PLOT AT EVERY J-TH POINT	09160
C # +J , LINE PLOT WITH A POINT PLUT AT EVERY JAHR PLA	19380
C ID - VARIABLE OF ARRAY CONTAINING TITLE OF PLOT	19360
C MC - NUMBER OF CHARACTERS IN TITLE CO IF NO IIILE C	20160
C IROLL - OPTION TO MOVE TO & NEW FRAME - AFTER THIS PLUT	19.000
	09360
C EUDAL IU U • NEW FRAME	19000
	09 150
	69.JFn
	19.150
C ARR FRANK L ENDRES - PAY 1897 THE	09150
	09.150
iTl = iTl + 1	09JF0
IF (IT1.NE.0) GO TO 20	09JF0
IF (MOP+EQ.1) CALL AGNPLT	03UP0
IF (MOP.EQ1) CALL AGNPLT (ALFILMPL)	03160
20 IF (NC+NE+0) GO TO SA	03LPr
NC # 10	03160
ID = 10H	09 JE0
50 CONTINUE	09JE0
C POSITION ORIGIN	09-1E0
IF (IT2+EQ.0) CALL PLT (1-9+ 1-5+ -3)	9.JE0
$IF (IT2 \cdot EQ \cdot 1) = 30 To - 100$	19760
C = CALE X - AXIS	03160
CALL SCALE (AF+ XL+ NP+ INc)	09.050
C = = = SUALE Y = RAIS	09150
CALL SCALE (TF; 1L; NF; INT)	49.JE0
v = v = v = v = v = v = v = v = v = v =	19.10
r = 1 $r = 1$ $r = 1$ $r = 1$ $x = x = x = x = x = x = x = x = x = x$	09.10
C SFT UP Y+AXIS	79.JF0
$x_N = -x_E(NP+1) / x_E(NP+2)$	09.150
CALL AXIS (XM+ Y+ 1H + 1+ YL+ YO+ YF(NP+1+ YF(NP+7))	09.180
C	09160
CALL SYMROL (+1+ -+5+ +14+ TD+ ¥0+ MC)	03160
C PLAT THE FUNCTION	03L64
IND CALL LINE (XF, YF, NP, 1, LOP, 10)	19JEO
1F (IT2+EQ+1) TT2 # 0	09160
IF (IROLL+EQ.0) CALL PLT (+0+ +0+ 999)	0.47£0
IF (IROLL+LT+0) CALL ENDPLT	1.476.0
IF (IROLL+GT_0) TT? = 1	11 10
IF (IROLL+LT+C) TTL = =1	• • J • 0

RETURN END -9JE0 19JE0

APPENDIX G

LISTING OF DATA FOR EXAMPLE PROBLEMS

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	EXAMPLE	PROBLEMS F	OR DP	ACE PRU	GPAM	+ 1	JACK CI	- din Deit	11, 191	'1	
	DYNAMIC	REAM-COLUM	N FR	GRAM L	51NG 5-	1-5 1#8	PLICTT	IPERATI) ~		
1	INELAST	TCALLY SUPP	DRTER	1 MASS	UNDER 1	REF VI	NOTTARS				
, n	0 0	n n	0	0	0 2	0	1	1 0	3	7	1
	1 1.000E	ANO 8	no 6.	283F-0	έ. Έ	Ū	-	i c	1	1	
	50 1-0000	-01 1 000F=	06		-	-		•			
	1 1 1	*01 1 * 00000-	1	000545	19						
	1 1	ò		0000F + 0	10						
	· · ·	× ^	· · ·		DOF-01	-5-00BE	c1=5 a	0.01 + 0.1			
	1 1				000.001	-1 0025		0.0H + 0.0			
	1 1		00	-1-1		-1.199.CC					
	1 1	н(-0 й	00		1005+01	+ V L L -	01-5.0		11.01		
	1 1	u#1*0906*	00 1	- <u>2005</u> + C		1 1	L000F 40.	* **044			
				50 5	20 20	120	120				
	- ·			2	4 0	20	100				
	TIME D	FFL	1		0						
14	FREE VI	ARATION OF	MASS	RY SPE	CIFYIN	3 1/1/1	AL UTSP	ALEME	415		
0	0 0	0 0	0	o	0 4	1	P .	1 1	L.		1
	1 1.000E	+00 8	00 6.	, 283F - L	13	Ð	-	1 0	1	1	
	50 1+000E	+01 1,000E-	06			_					
	1	1 3,000E+	n Q			1					
	1 1	0	1.	• U U U E + C	0						
	1 1	0-1.040E+	00 1.	, an aF−L	11 2	1 1	,0005-0	2 1*000)F + 0 4		
				20 -	36 60	120	120				
				2	4 8	20	100				
	TTME D	FFL	1		0						
2-1	LATERAL	VIBRATION	OFA	SIMPLI	SUPPOI	ALEU HE	лм жутн	9 X I VI	CUMARS	5510N	F.
٨	n 0	0 1	0	3	n 1	11	1	1 0	0	0	2
	•	<i>v</i> ,			v •	• •					-
v	10 1.200E	+01 2	00 3.	,752F-0	4	Ô		0	i	ĩ	-
v	10 1.200E	+01 2 1 0.000E+	00 3. 00	,752F-0)Å	2 2	·	0	i	1	-
v	10 1.200E	+01 2 1 0.000E+ 1-1.221F+	no 3, 00 00	,752F-0)Å	0 2 1		0	i	1	-
v	10 1.200E 0 1 2	+01 2 1 0.000E+ 1-1.221F+ 1-2.324E+	00 3, 00 00 00	,752F-0	4	0 2 1 1	·	0	i	1	-
v	10 1.200E 0 1 2 3	+01 2 1 0.000E+ 1-1.221F+ 1-2.324E+ 1-3.197E+	00 3, 00 00 00 00	,752F-0	4	0 2 1 1 1		0	i	1	-
v	10 1.200E 0 1 2 3 4	+01 2 1 0.000E+ 1-1.221F+ 1-2.324E+ 1-3.197E+ 1-3.758E+	60 3, 00 00 00 00 00	,752F-0	4	0 2 1 1 1 1		0	ĩ	1	-
v	10 1-200E 0 1 2 3 4 5	+01 2 1 0.000E+ 1-1.221F+ 1-2.324E+ 1-3.197E+ 1-3.758E+ 1-3.952E+	no 3, 00 no no no no	,752F-0	4	0 2 1 1 1 1		0	ĩ	1	-
v	10 1+200E 0 1 2 3 4 5 5	+01 2 1 0.000E+ 1-1.221F+ 1-2.324E+ 1-3.197E+ 1-3.758E+ 1-3.952E+ 1-3.758E+	00 3, 00 00 00 00 00 00	,752F-U)4	0 2 1 1 1 1		D	i	1	-
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	1 1	0		3,000	E+07	2.10	0E+10			1146		ONT	10	1	
	0 31	0 2.877	•11	1.000	E*#V					TIME	M	DNT	21	ò	
	i	1	•	2.000E+04						BEAM	D	EFL	60	I	
	11	0	•	1.000E+04						BEAH	0	EFL FRI	160	0,	
	· .,	1			-1+2	2VE+U+ 24F+04				BEAM	ŏ	FFL	280	1	
	1 31	ō		3,2102+00 1,918	E+06					BEAN	Ň	OHT	60	i	
	0 0	0		3.210E+01						BEAM		ONT	320	0	TTOTO NOW THESE FUEL GADED BY T. P.
	1 11	1	10	-3.0906.02	0.	-3.090E+03				1 O	*****	DAN BEAM	#11M	UNE WAT ANU STAM	2 2 1 0 5 9 10
	1 11	20	30	-3.0902.03	V.	-8.96AE+03				30	4.800E	+01	40	4.0008-02	-1 4 -1 4 0
	i ii	30	40	-8.968E+03	ö.	-1.118E+04				4		8	15	16 17	··· ·· ··
	1 11	40	50	-1,118E+04	0.	-1.260E+04				50	4.000E	·00 1.000	E-06	A	15 16 17
	1 11	50	60	-1,260E+04	0 .	-1,309E+04				30		10.			2
	1 11	70	80	-1.2605+04	V.	-1.118E+04				õ	0	õ			3,000E+10
	i ii	e õ	90	-1.118E+04	0.	-8.968E+03	1			Ō	30	0 2.400	E+10		-1.000E+05
	1 11	90	100	-8.968E+03	ĝ.	=6.180E+03				0	30	0		5.184E-01	
	1 11	100	110	-0,160g+03	1 .	-3,0908+03				17	17	2	Â	0.0 −1.000F+04+1	+0 ==1,000E+04 +000E+04 1,000E+03
	1 11	121	130	2.7852+02	9. 0.	2.784E+03				17	17		12	1.000E+03 1	.000E+03+1.000E+04
	i ii	130	140	2.788E+03	0.	5.000E-03				17	17	12	16	-1.000E+04-1	.000E+04 1.000E+03
	1 11	140	150	5.000E+03	0.	6.421E+03	1			17	17	16	20	1.000E+03 1	.000E+03-1.000E+04
	1 11	150	160	6.4218.03	0.	6.910E+03				8	8	0-1-000	£+03	-40 -40 -39	-35 -20 0 0
	1 11	170	180	6.421E+03	0.	5.000E+03								-40 -30 -20	-10 -5 0 40
	i ii	180	190	5.000E+03	0.	2,788E+03	1			15		1-1.000	E+02	1.0008-01 4	1 1,000E-04 1,000E+00
	1 11	190	199	2.788E+03	¢•	2.788E+02								0 0 0	0
	1 11	201	210	-Z+212E+02	0.	-2.212E+03					17	0-1 000	F+02	1 0005-01 4	1 1.000F=04 1.000F+00
	1 11	210	230	-2.2122-03	0.	-4.1228+01	r I				1,	*-1+400		30 86 136	136
	1 11	230	240	-4.122E+03	0.	-3,633E+03								3 10 20	40
	i ii	240	250	-3.6332+03	Ö•	-2,212E+03	1			TIME	0	EFL	8	1	
	1 11	250	259	-2.212E+03	Q.	-2.2122.02				TIME	0	EFL	17	0	
	1 11	261	270	1.4216.02	2.	1.421E+03				1 I ME T 1 MF		OM7	17	1	
	* *1	210	-00	194515403	17.0	***********						.	• '	••	

.

	BEAM	DEF	L	4	1											
	BEAM	DEF	L	8	ĩ											
	BEAM	DEF	L	20	0											
	BEAM.	MOM	т	12	1											
	BEAH	HOH	T	16	ĩ											
	BEAM	MOM	T	30	Ó											
7	5	PARTIALLY	EMBEDDI	ED PILE	EXCITED	RY EA	RTHOU	AKE	INDUC	ED F(RCESI	1/10	8)			
0	0	Ö	0 0	0	0 0	3	0	1	2	0	19	6	10			
	50	1.200E+0	1	160 5.	000E-03		+1	5	+1	0	1	1				
	10		10	20	30 40	50										
	50	1.000E+0	1 1.0004	E=06												
	0	50	0 1.0908	E+10			-1	.00	0E+04							
	0	40	0	5.	184E-01											
	41	50	0	Ζ,	592E+00											
	0	40	1	1	1,223E	+01 1	.223E	+01	1,223	E+01						
	0	40	2	2	2.417E	•01 2	.417E	+01	2.417	E+01						
	Ö	40	3	3	3.550E	+01 3	.550E	+01	3,550	E+01						
	Ó	40	. A	4	4.597E	+01 4	.597E	+01	4.597	E+01						
	Ó	40	5	5	5.5308	+01 5	.530E	+01	5.530	E+01						
	Ô	40	6	6	6.327E	•01 6	. 327E	+01	6.327	E+01						
	0	40	7	7	6.9685	+01 6	.968E	+01	6,968	E+01						
	ō	40	8	8	7.4372	+01 7	+437E	+01	7.437	E+01						
	ō	40	9	9	7.724E	+01 7	.724E	+01	7,724	E+01						
	ö	40	10	10	7.820E	+01 7	BZOE	+01	7.820	E+01						
	ō	40	11	11	7.724E	+01 7	.724E	+01	7.724	E+01						
	0	40	12	12	7.437E	+01 7	.137E	+01	7.437	E+01						
	ō	40	13	13	6.9685	•01 6	.968E	•0ï	6.968	E+01						
	Ö	40	14	14	6.327E	+01 6	.327E	+01	6.327	E+01						
	0	40	15	15	5.530E	+01 5	+530E	+01	5.530	E+01						
	Ö	40	16	16	4, 597E	+01 4	. 597E	•0ī	4,597	E+01						
	0	40	17	17	3.550E	+01 3	.550E	•01	3,550	E+01						
	0	4 0	18	18	2.417E	• 01 2	+417E	+01	2,417	E+01						
	0	40	19	19	1.223E	+01 1	•223E	+01	1,223	E+01						
	0	40	0 1,0000	E+00-1.	000E-02		0 1	.000	E-03	1.000	E+00					
				1	00 100	0	0									
				10	00 5	0-1	000									
	0	40	0 1.0008	*00-1.	000E-02	•	-1 1	.000)E=0₽ :	1.000	E+00					
				1	00 100	0	0									
				10	00 5	0-1	000									
	TIME	DEF	L	10	1											
	TIME	DEF	L	30	0											
	TIME	DEF	L	40	i											
	TIME	DEF	L	50	Ó											
	BEAM	DEF	L	10	1											
	BEAM	DEF	L.	20	,											
	BEAM	DEF	L	50	i											
	6EAM	DEF	L	150	G											
	BEAM	HOH	т	30	1											
	BEAH	HOH	т	100	Ó											

APPENDIX H

PARTIAL SAMPLE COMPUTER OUTPUT FOR EXAMPLE PROBLEMS

This page replaces an intentionally blank page in the original. -- CTR Library Digitization Team

TABLE 5 - MASS AND CAMPINGS DATA .

s т FROM TO CONTO F QF R 1 1 0 -0. 1.000E+02 -0. -1. = 0 **.** -0.

STA CASE DEFLECTION

TABLE 4 - STIFFNESS AND FIXED-LOAD DATA

NONE

PROR

1

TABLE I - PROGRAM-CONTROL DATA

TABLE 3 - SPECIFIED DEFLECTIONS AND SLOPES SI OPF IOPS (TF=1.INITIAL .

	2	3	4	5	6	7	R	9
HOLD FROM PRECEDING PROBLEM (l=HOLD)	0	٨	0	0	0	0	o ·	0
NUM CARDS INPUT THIS PROBLEM	?	0	1	۱	٨	3	3	1
TABLE 2 - CONSTANTS								
NUM OF BEAM INCREMENTS							1	
BEAM INCREMENT LENGTH						1.00	06+00	
NUM OF TIME INCREMENTS							800	
TIME INCREMENT LENGTH						6.28	38-03	
OPTTONAL PRINTING SWITCH							^	
NUM OF HUNITOR STA (PRINTING)							-0	
ITERATION SWITCH (0=LINEAR)							- 1	
NUM OF MONITOR STA (ITERATION)	• -						0	
PLOTTING METHOD (-1=MICOEPRINTER.	INDVAN	R)					1	
LINE OR POINTS PLOT OPTION							1	
ITERATION DATA FOR NONLINEAR SUP.	cv							
MAXIMUM ITERATION NUMBER							50	
MAXIMUM ALLOWABLE DEFLECTION						1.00	0E+01	
DEFLECTION CLUSURE TOLERANCE						1.00	0E=0K	
MONITOR STATIONS (ITERATION)								
1 2 3 4 5								
STAJ 0								

PROGRAM NBCS - MASTER - JACK CHAN - MATLOCH - DECKI-REVISION DATE # 26 JUN 71 EXAMPLE PROHLEMS FOR DRCS PODGRAM - RY JACK CHAN APPIL 1971 UYNAMIC DEAM-COLUMN PROGRAM USING 5-1-5 IMPLICIT OPFRATOR

ONE MASS SUPPORTED BY A MONETNEAR SPRING UNDER FREE VIRRATION

TABLE NUMBER

TF=2, PERMANENT)

CP

TABLE 9 - PLATTING SWITCHES

STATION

1

HORIZONTAL VERTICAL BEAM OR TIME WHETTPLE PLOT

AXIS

DEFL

AXIS

∃≓IT

0 -1.000F+00 1.000F-01 36 60 120 120 1 5 1 1.000F-02 1.000F+00 1 20 ۵ 2 8 20 100 ы 4

TABLE 8 - NONLINEAR SUPPORT CURVES FROM TO CONTU Q-MULTIPLIER W-MULTIPLIER POINTS SYM OPT W-TOLEPANCE Q-TOLEPANCE

0

.

SWITCH (IF#1+SUPERTMPOSE WITH NEXT IF#0+PLAT 4L+ SAVED PLOTS:

BEA.⊯ FROM	STA TO	TIMF From	STA TO	οτj	STO	ετα
1	1	0	0	-5.000E+01	-5.000F+01	-5.000E+01
1	1 1	800	800 800	-1.000E*02 -5.000E*01	-1.000F*02 -5.000£+01	-1.000E*02 -5.000E+01

TABLE 7 - TIME DEPENDENT LOADING

.NONE.

BEAM STA TIME STA FROM TO FROM TO 112 773 ττī

TABLE 6 - TIME DEPENDENT AXIAL THRUST

1 1 0 1.000E+00 -0. -0.

FROM TO CONTO HHO 01 DE

1 ONE HASS SUPPORTED BY A NONLINEAR SPRING UNDER FREE VIRRATION

PRON (CONTD)

TABLE 10- CALCULATED RESULTS JEBEAM AXIS: KETIME AXIS

PRINTING RESULTS IS NOT REQUESTED.

STANDARD 12 INCH BALLPOINT PAPER PLOT IS REDUESTED.

PROGRAM DBCS - MASTER - JACK CHAN - MATLOCK - DECK1-REVISION DATE = 26 JUN 71 Example Problems for drog program By Jack Chan April 1971 Dynamic Beam-column Program Bing 5-1=5 Implicit Opfrator

ELAPSED CON TIME . O MINUTES 31.486 SECONDS

TTHE FOR THIS PROBLEM = 0 MINUTES 6.841 SECONDS

PRDR (CONTD) 1 ONE MASS SUPPORTED BY A HONLINEAR SPRING UNDER FREE VIRGATION

PROGRAM nBC É	S - MASTER Xample Prob Ynamic Beam	- JACK CHAN - MAT LENS FOR PROGRAM -COLUMN PROGRAM (100× - DEC DHC5 - 5 5-1-5 [MPL]	KI-REVISI RY JACK C ICIT OPER	NN DATE Han jun ator }	= 26 E 197	5 JUN ∤1	71	n n 3	0 U 0 0	-0. 2.400E	-0. 10 -0.		-0. -n.	-n. -1.	000€+05	3.010F+10 -0.	-0. -0.
									TABLE 5	_ MA	SS AND DA	MPINGS	DATA					
6 T	HREE-SPAN B	EAM WITH ONE-WAY	AND SYMMETR	TC NONL IN	FAD SUP	LOAD	DED AN	r T.P	FROM TO	CONT	n RHO		01	DE				
									o 3	0 0	5.1846-	01 -0.		-0.				
TABLE 1 - F	ROGRAM-CONT	ROL DATA																
	-		ר ל	4 5	6 6	7	e	9	TABLE 6	- TI	ME DEPEND	DENT AX	IAL TH	RIST				
HOLD FROM P NUM CARDS 1	PRECEDING PR	ORLEM (l=Holo) RorleM	יים קייני	0 0 2 1	0	0 5	0 9	0 1 0	BE	AM ST Om Ti	Δ 1 Ο Ρ	THE ST	ג	+	τī	112	ŢT3	
											•NONE •							
TABLE 2 - 0	CONSTANTS																	
NUM DE	F REAM INCRE Incrément le	MENTS NRTH				4.800	30 DE+01		TABLE 7	- TI	ME DEPEND	ENT LO	DING					
NUM OF	TIME INCRE Increment le	MENTS NGTH				4.000	40 20-30		8E	AH ST	۲ ۵	INE ST	4				_	
OPTTON NUM OF	NAL PRINTING	SWITCH					-1		FR	0 M T	0 F	ROM TO)	0	Tì	QT2	013	
1TERAT	ION SWITCH	(D=LINEAP)					-1			17 1	7	Ō		0.		0.	-1,000E	+04 *03
NUM OF	MONITOR ST Ing Method(-	A { JTERATION] 1=MJC++0=PRJNTED+	1=PAPFR)				ō			17 1	, 7	A 1	2	1.0	00E+03	1,000F	+03 -1.000E	• 0 4
LINE (DR POINTS PL	OT OPTION					- î			17 1 17 1	7 7	12 1	5	-1.0 1.0	00E+04	-1,000E	+04 1,000E	+03 +04
COMPLE	ETE PRINTING	TIME INTERVAL AN OF CONPLETE PRIM	ND MONITOR S	TATIONS F	OR PRIN	TING-	•					-						
MON 1	2 3	4 5 A	7 R 9	10					TABLE 6	- NO	NLINEAR S	UPPORT	CUBAE	5				
STAJ B	15 16	17							FROM	Tn C0	NTD G+MUL	TIPLIE	₩-40	TIPLTER P	DINTS	SYM OPT	W-TOLEPANCE	Q-TOLFPANO
ACX101	TION DATA FO	R NONLINEAR SUP.	Cv				50		8	8	0 -1.	00nE+3	• 1	100E-01	7	-0	1.000F-02	1.0005+0/
MAXIM	ALLOWABLE	DEFLECTION				4.000	00+30		9	-40	- 4 U	- 39	- 35	- 20	î	0		
DEFLEC	CTION CLOSUR Dr Stations(E TOLERANCE ITERATION)				1.000	05=08		# 15		1 +1.	0005+0	2 1			1 01	1.000F-04	1.000#+00
1 STA (B	2 3	4 5 17							9	0	0 10	0	0					
JIN U U	13 10	1,							-	17	0 -1.	000E+0	2 1	0005-01	4	1	1.000F-04	1.000
									4	30	lu	136	40					
TABLE 3 - 5	SPECIFIED DE	FLECTIONS AND SLO	IPES															
STA	CASE	DEFLECTION	SIORE	IOPS	(TF=1+ TF=2+	PERMA	IAL	•	TABLE 9	- PL	nTTING S≭	ITCHES						
0 30	1	0. 0.	NONE	2					HORIZO	NTAL	VERTICAL	BEAM OF	A TIME	HULTTPLE	Ρίοτ	1001-5-15		TH NEXT
									AXI	3	AX 1 2	514	1108	5#1*64		IF=0+PL(T ALL SAVED	PLOTS
TADIE & -	STIFFNESS	AND EIXED-1040 DA	TA						тім ттм	E	DEFL OFFL		3	1				
HOLE	51 TE - 140 23	More & TVC	-				~ •		TIM	Ē	HOPT	1	3	ĭ				
FROM TO COM	NTD F	QF	S	T	8		CP		TIM	E	MONI	1	7	0				

BEAM	UEFL	4	1
BEAM	DEFL	8	1
BEAM	DEFL	20	0
BEAM	HOHT	12	1
BEAM	MDHT	16	1
BEAM	HOMT	30	0

PROGRAM DBC5 - MASTEM - JACK CHAN - MATLOCH - DECMI-REVISION DATE = 26 JUM 71 Example problems for program obcs - Ry Jack Chan Jume 1971 Oynamic Jeam-column program / 5=1=5 implicit operator }

PROP (CONTU)

6 THREE-SPAN BEAM WITH ONF-WAY AND SYMMETRIC NONLINEAD SUP LOADED BY T.P.

TABLE 10- CALCULATED RESULTS JEBRAM AXIS, KETIMF AXIS

TIME STA K = -2

FOR 1 ITERATION, THERE ARE 0 STATIONS NOT CLOSED -1.313E-16 -1.125E-15 -1.236E-15 -1.317F-15

TIME STA # = 1

MONITOR STATIONS J= 9 14 14 17

FOR 1 TTERATION, THERE ARE 0 STATIONS NOT CLOSED -1.313E-16 -1.125E-15 -1.236E-15 -1.317E-15

***** STATIC RESULTS *****

STA J	DIST	DEFL	SLOPE	мом	SHEAR	SUP REACT
-1	-4.800E+01	5.001E-18		ñ.		0.
			-1.6425-19		1-1536-15	
•	0.	0,		5.390E-11		-1-1892-12
			1.114F-19		-6.651F~]4	_
,	4.800E+01	5.346E-18		5,017E-11		0.
			2,1175-19		-1,189F-12	
,	9.600E+01	1,5516-17		-7.9365-12		0.
			1.05AF-19		-1,1895-12	
3	1.440E+02	2.491E-17		-A.596F-11		0.
			6, 192F-20		-1,1995-12	
	1.920E.02	2.7985-17		-1.234F-10		0 .
			-1. aZAF-19		-1,1895-12	
٩.	2.4002.02	1.9205-17		-1.796E-10		0.
			-5,4195-19		-1,1895-12	
4	2.8406.02	-6.810g-18		-2,3416-10		0.
			-1, n10p-18		-1.189F-12	
7	3.340E+02	-5.5296-17		-2.843E-10		0.
			-1.5938-18		-1.149F-12	
A	3.840E+02	-1.313E-16		-3.35AE-10		2.625E-12
			-7,2545-18		5 i-36E4.1	
•	4.3202.02	-2.395F-16	•	-2.5A0F-10		0.
•			-2.766F-18		1.4365-17	

10	4.800E+02	-3,7225-16		-1.739E-10		0.	-1	-4+800E+01	7.3295-14		n.		0.
11	5.280F+02	-5-2175-16	-3.ĭleF-18	-8.9985-11	1.436F-12	o.	n	0.	٥.	-1,627F-19	7.898F-11	1.645F=12	-1.397E-12
1.2	5.760E+02		-3,294E-18		1.4365-12	0.	۱	4.810E+01	7.A35E-18	1,4325-19	9.014E-11	2,4885-13	٥.
12	5+7602+02	-0./405-10	-3.104F-18		1,436F-12	•	,	9.600E+01	2.432F-17	3.435F-19	2.1376-11	-1 . 3986-12	a.
17	6.240E+02	~9.384F-16	-3. ¹⁴⁵ F-18	7,953F-11	1.436F-12	U .		1-4405-02	4 3865-17	3.¤62F=19	-4 7975-11	-1,4045-12	0.
14	6.720E.02	~9.894E-16	-7.818E-18	1.676F-10	1,4365-12	o.	,	1.0005.02		2.005F-19		-1.4165-12	0
15	7.200E+02	-1.1258-15	-2.326F-18	2.460F-10	1,436F-12	0.	4	1.9200402	5+841E+17	5.606F-20	-1.1721-10	-1.434F-12	•
14	7.680E+02	-1.236E-15	-1 4745-18	3.261F-10	3.417F-13	-1.094E-12	5	2.400E+02	5.9505-17	-3.1665-19	-1.863E-1V	-1.460F+12	v.
17	8.160E+02	~1.317E-15		3.5055-10	3 5075 53	- j.924E-13	6	2.890E+02	4.430E-17	-8.264F-19	-2.5495-10	-1.4945-12	0.
19	8.640E+02	-1+363E-15	=4.730F=14	3.3846-10	•3.507+=13	0.	7	3+360E+02	4,K31E-18	-1 4725-18	-3,2566+10	-1.5325-12	0.
19	9.120E+02	-1.378E-15	-2,9635-19	3,2296-10	-3,507F-13	0.	A	3+840E+02	-6.601E-17	2.000 19	-3.A91E-10	1.7185-12	2.6+0[+12
20	9.600E+02	-1.361E-15	3.4965-19	3.044F=10	-3,507F-13	0.	9	4.320E+02	-1.7405-16	-/, /5//-10	-2.958F+10	1.710-17	0.
2)	1.00AF+03	-1-3158-15	9.5845-19	2.8305-10	-3,507E-13	0.	10	4.800E+02	-3,]^4E-16	-2.A42F-18	-2.01AE-10	1.0/05-12	0.
20	1.0565403	-1 2425-15	1,524F-18	2 5995-10	-3,507F-ī3	0.	11	5.2A0E+02	-4.662E-16	-3,245F-18	-1.077E-10	1.6365-12	0.
22	1.1045403	-1 1645-15	2.042E=18	3 3335-10	+3,507 <u>F</u> -13	0	12	5.760E+02	-6.373F-15	-3,460F-18	-1.426F-11	1.6005-12	0.
23	1.1-05	-1.1	2.507E-18	2.3722-10	-3.507F-13	•	15	6.240F+02	=7.997F-16	-3.4095-18	7.779F+11	1.5695-12	0.
24	1.1521+03	-1.023E+15	2.013E-18	\$*033E+10	-3,507F-13	U .	• •	6-730E+02	-9 -975-16	-3,33E-18	1 4705-10	1,544F-12	0.
25	1.200E+03	-8.A35E-16	3.258F-18	1.7258-10	-3.507#-13	0.	14	3,7202002	-4.54/2-15	-7,998F-18	1.0792-10	1.5255-12	0
26	1.248E+03	-7.271E-16	3.538F=18	1.401E-10	-3.507F-13	0•	15	7.200E+02	-1.1046-15	-2.487F-18	2.0000-10	1.5115-12	•
27	1.296E+03	-5.573F-16	3 7516-18	1.062E-10	-1.507F-13	0.	16	7.689E+02	-1.223E-15	-1,0075-18	3.399E-10	3,414E-13	-1.101E-12
28	1+344E+03	-3.772E-16	3., 51F 410	7,1398-11	-1 =0.25 12	0.	17	8.140E+02	-1.310E-15	-1.077F-18	3.5508-10	-3,906F-13	-7.274E-13
29	1.302E+03	-1.9036-16	3.8941-10	3.5A7F-11	-3.50/5-13	0.	18	8.640E+02	-1.361E-15	-1. 7395-19	3.514E-10	+3.919F-13	0•
30	1.440E+03	٥.	3.9655-18	٩.	-3.507E-13	3.507E-13	19	9.120E+02	-1.379E-15	-3,737-19	3.344E-10	-1 9085-13	0.
31	1.48BE+03	1.903E-16	3,9655-18	0.	0.	0.	2 n	9+610E+02	-1.365E-15	2,0492-10	3.1425-10		0.
		TIME S	TA # =	n -			21	1.005E+03	-1.321E-15	9,233F=19	2.912E=10	-3.8801-13	0.
							22	1.056E+03	-1.2495-15	1.506F=18	2.6558-10	-3,841E-13	0.
	. =	м	ONITOR	STATIC	INS		23	1.104E+03	-1.1516-15	2,037E+18	2.375F-10	-3.796F-13	0.
FOI	J -				17		24	1+152E+03	-1.0305-15	2.5125.18	2.075F=10	-3,749F-13	0.
FUF	-0	-501F=17 -1	+104E-15 -	1.223F-15	-1.3105-15		24	1 3005+03		2.027E-18	1 75(5-10	-3,705F-j3	0.
							25	1.2002+03		3,278F-18	1. (775 - 10	-3.6645-13	0
***** D1	NAMIC RESUL	TS *****					26	1	-7-3245-16	3,563 <u>5</u> -18	1.4436-10	-3,6295-13	·•
		TIME S	TA K =	n			27	1+544E+03	-5+614E-16	3.778=18	1.0796-10	-3,602F-13	U .
							29	1.344E+03	-3.AnlE-16	3_023=-18	7.237E-11	-3,584F-13	0.
STA J	DIST	DEFL	SLOPF	MOM	SHEAR	SUP REACT	29	1.392E+03	-1.918E-16	2 0945-19	3.634E-11	-3.5745-13	9.
										14 490-410			

STA J	DIST	OEFL	SLOPF	MOM	SHEAR	SUP REACT
+ 1	-4+800E+01	5.3888-03		o.		0.
	0.	0.	-1.1228-04	5-80AF+04	1.2105+03	-1.028E+03
0	•	••	1.200F-04		1.8115+02	
۱	4+800E+01	5.759E-03	2 5235-04	4,617E+04	-1.0285+63	0.
s	9.6006.01	1.7875-02	/ J/	1.561E+04		0.
	1-6405-02	3.1485-02	2.436E-04	-3.508E+04	-1.0275.03	0.
			2.1355-04		+1.025F+03	
•	1.920E.02	4.173E-02	4 3995-05	-8.529E+04	-1.0235+03	0.
5	2.400E+92	4.379E-02		-1.346E+05		0.
	2. AA0F+02	3.2935-02	-2,763F-04	-1.8245+05	-1.020E+03	0.
-			-5.012E-04		=1.017F+#3	

TIMESTAK# 4

FOR 4 ITERATION. THERE ARE & STATIONS NOT CLOSED -4.576E-02 -8.684E-01 -9.957E-01 -1.111E+00

FOR 3 ITERATION, THERE ARE 29 STATIONS NOT CLOSED -4.576E-02 -8.684E-01 -0.057E-01 -1.111E.00

FOR 2 ITERATION, THERE ARE DO STATIONS NOT CLOSED -+.336F-02 -4.350E-01 -4.414E-01 -1.078E+00

FOR 1 ITERATION. THERE ARE 29 STATIONS NOT CLOSED **.673F-02 -8.891E-01 -1.019E+00 -1.137E+00

MONITOR STATIONS 17 ≖ ز 16 15

TIME STA #

			-6.041E-04		-6.800F+62	
8	3+840E+02	-2.991E-02	-1.788E-03	-1.8125+05	5.185F*02] +196E +03
15	7.200E+02	-5.6675-01	-1 4695-03	5.998E+74	4.9725+02	е.
14	7.6406.02	-6.468F-01	1 075 03	9,186E+ ¹⁴	3 41003	2.887E+03
17	8.160E.02	-7.1A0E-01	-1+4858-03	2.7278+05	3.010.103	3,172E+03

SUP REACT MOM SHEAD STA J 0157 DEFL SLOPF

TIME STA # # ٦

FOR 2 TTERATION, THERE ARE & STATIONS NOT CLOSED -2.991E-02 -5.667E-01 -4.468E-01 -7.180F-01

FOR 1 ITERATION. THERE APP 29 STATIONS NOT CLOSED -2.991E-02 -5.6575-01 -6.4645-01 -7.1805-01

STATIONS HONITOP 16 17 J≠ A 15

		-A.472F-04		4.4515+0
7.640E+02	-3.404E-01	-7. PAF-04	8*A34E+04	2.323F+0
8.160E+02	-3.744E-01	at for a factor	1,843£+05	••••

3

1,798E+03

-9.538E*04 3-840E+02 -1-613E-02 A 4.050-+02 -9.1515-04 с. 4.3965+04 7.200E+02 -2.098E+01 15 ñz 1.662E.03 7.68 14)3

-3.44E-04 -3,6295.02 6.452E'02

SUP REACT MOH SHEAR STA J DIST DEFL SLOPE

TIME STA # # 2

TIME STA K =

17

FOR 3 ITERATION. THERE ARE & STATIONS NOT CLOSED -1.613E-02 -2.998E-01 -3.404E-01 -3.744F-01

FOR 2 ITERATION. THERE APE 39 STATIONS NOT CLOSED -1.613E-02 -2.998E-01 -3.404E-01 -3.744E-01

FOR 1 ITERATION. THERE ARE 20 STATIONS NOT CLOSED -1.690F-02 -3.120E-01 -3.837F-01 -3.880E-01

STATIONS MONITOR 17 J∎ 16 ĩ. - 8

TIME STA # . 2

-1.375E-04 -1.3825+02 50+3666.5 3.840E+02 -6.665E-03 -3.57RE+04 8 -3.778E-04 2.8475*02 0. 7.2002+02 =1.1902=01 2.514E+04 15 -3.275E-04 3.618F+02 6.735E+02 7+680E+02 -1.347E-01 4.408E+04 14 -2,3948-04 1,1235+03 9,9126+04 8.160E+02 -1.462E-01 7.3092+02 17

SUP REACT SHEAR STA J DIST DEFL SLOPE MOM

TIME STA # # 1

FOR 2 TTERATION, THERE ARE & STATIONS NOT CLOSED -0.665E-03 -1.190E-01 -1.347E-01 -1.462F-01

FOR 1 TTERATION, THERE ARE 29 STATIONS NOT CLOSED - -0.665E-03 -1.190E-01 -1.347E-01 -1.462F-01

MONTTOP STATIONS J ¥ 16 17 8 15

TIHE STA # =

-4.109E-24 3.5748-13 1.440E+03 0. 30 1,0065-18 8.560F-24 1+468E+03 1+918E-16 0. 31

-1,VIBF+0:	1 9305.93	~++6/8F-02 -8,684F-01 -1.604F+00 -1.135F+60
-2.722F+05 8.142F+02	1+9302+03	FOR 2 ITERATION, THERE ARE TO STATIONE NOT CLOSED
2.254E+05	0.	-++70F-02+402E-019.745F-011.106F+00
8,105F+02	2	
·1.767F+05	0.	FOR 3 ITERATION, THERE ARE 29 STATIONS NOT CLOSED
1.247F+05	0.	-4.078F-02 -8.684E-0) -1.004F+00 -1.135F+00
7.9235+02	2	FOR 4 ITERATION, THERE ARE 29 STATIONS NOT CLOSED
-7.592E+04	0.	-4.470E-02 -A.402E-01 -9.745E-01 -1.106E+00
/ / 80E+04	0.	FOR 5 ITERATION, THERE ARE 30 STATIONS NOT CLOSED
7.628F+02	2	
2,5216+04	0.	
7.4595+12	2	FOR 6 ITERATION, THERE ARE 29 STATIONS NOT CLOSED
7.2835+02	2	-4.785F-02 -8.938E-01 -1.033E+00 -1.16/F+90
1,227E+05	4,2836+03	FOR 7 ITERATION, THERE ARE 29 STATIONS NOT CLOSED
4.823F+03	1 5305-03	-4.798F-02 -8.9KjE~01 -1.ñ35F+00 -1.170F+00
3.657E+05	+•5/8E+V3	FOR B TTERATION THERE ARE TO STATIONE NOT CLOSED
3.610F+05	0.	
=2.793E+12	2	
3.5225+05	, ^U .	FOR 9 ITERATION, THERE ARE 27 STATIONS NOT CLOSED
3.387F+05	. 0.	-*+802E-02 -8.968L*n1 -1.036E*00 -1.1/15+90
-3,375E+02	2	FOR 10 ITERATION, THERE ARE 26 STATIONS NOT CLOSED
3,204E+05	0.	-4+B02E-02 -8+968E-01 -1+ñ36E+00 -1+171E+00
2.976F+05	0.	FOR 11 ITERATION, THERE ARE 30 STATIONS NOT CLOSED
-3.9825+02		-4.802F-02 -8.96AE-01 -1.636E+00 -1.171E+00
2.7052+05	0.	
-4.258E+84	0-	FOR 12 [TERATION, THERE ARE 15 STATIONS NOT CLOSED
-4,500E+02	2	***802E=02 =#*96AE=01 =1*n35E*00 =1*1/15*n0
2.0506+75	0.	FOR 13 ITERATION, THERE ARE A STATIONS NOT CLOSED
-4.7046+02	0	-3.474E-02 -7.21KF-01 -8.434E-01 -9.930F-01
-4.865F+02	2	TIME STAK . S
1.278E+05	0.	
-4.984E+02	2	
-5.062F+02		STA J OIST NEFL GLOOF HOM SHEAR SUP HEALT
4,341E+04	0.	-A,204F-04 -7,733E+02
=5,101g+02	- 101- 00	A 3+840E+02 -3+474E-02 -2.115F+05 1+390E+03
°• 0.	2.101E+05	-2. k13F=03 -2.77RE +0?
n	0.	15 /*2006*02 =/*216E=01 =***/*********************************
-		16 7.6A0E.02 -8.534E-01 -A.012E.04 7.175E.03
,		-2,907r=03 5,529r+03 3,920r,03
		17 8+160E+02 -9,930E=01 1,992E+"5 3,838E+03
STATIONS		TIME STAK = A

MONITOR STATIONS J = 8 15 16 17

FOR 1 ITERATION, THERE ARE >9 STATIONS NOT CLOSED -1.698F-02 -5.000E-01 -5.987F-01 -7.008F-01

FOR 1 TIERATION, THERE ARE 29 STATIONS NOT CLOSED

. . . .

7 3+360E+02 4+549E-03 -2,2946+05 -1.0165+03 -1.04AF-03 3+840E+02 -4.576E-02 4 -1.592F-03 4.320E+02 -1.222E=01 ۹ -2,043F-03 10 4.800E+02 -7,203E-01 -2.197F-03 11 5.287E+02 -3.353E-01 -2.450E-03 5.760E+02 -4.625E-01 12 -2,802F-03 6.240E+02 -5.970E-01 13 -2,9526-03 14 6.720E+02 -7.339E-01 -2.#02F-03 15 7.200E+02 -8.684E-01 -2.453F-03 7.680E+02 -9.957E-01 16 -2.40AF-03 17 8.150E+02 -1.111E+00 -1.678F-03 8.640E+02 =1.192E+00 1 A -9,560F-04 9.1206+02 -1.2385+00 19 -2.516F=04 9.6002+02 -1.2502+00 20 4.257E-04 1.008E+03 -1.229E+00 21 1.0675-03 22 1.056E+03 -1.178E+00 1.4625-03 1.174E+03 -1.098E+00 21 2.203E-03 1.152E.03 -9.927E-01 24 2.482F-03 1.200E+03 -8.640E-01 25 3.092F=03 56 1.248E+03 -7.155E-01 3,4275-03 27 1.296E+03 -5.510E-01 3,4835-03 1.344E.03 -3.743E-01 24 3.455<u>e</u>03 1,3926+03 -1,4926-01 29 1,942F-03 1.440E.03 0. 30 3.042F-03 1.488E+03 1.892F-01 31

TINE STAK = 5

MONITOR STATIONS J= 6 15 16 17

FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED -4.600E-02 -8.578E-00 -9.930E-01 -1.125F+00

FOR 2 ITERATION, THERE ARE & STATIONS NOT CLOSED -4.600E-02 -8.574E-01 -9.930E-01 -1.125F+00 • USED ADJUSTED LOAD-OFFL CURVE TO CALCULATE DEFLECTIONS •

31	1.	488E+(13 -9.932E-	02	۰.		0.
			TIM	E STA K #	Q		
		L	≖ Ą	MONITOR	STATI(16	DNS 17	
FC	R 1	ITER	ATION: THERE -1.559E-02	ARE 29 5TA 5.5845=02	TIONS NOT CLI 1+839E=01	0\$ED 1.656€+01	
FC	R 2	ITER	TION, THERE -1.559E-02	ARE 0 474 5.584E-02	TINNS NOT CLI 1.0398-01	0SED 1.656E-01	
•	USED	AD JUS	STED LOAD-DE	ADE SO STA	CALCHLATE DEL TIONS NOT CLU	FLECTIONS *	
	•	1.110	-1.430E-02	7.719E-02	1.2728-01	1.8965-01	
FC)R 2	I TER	ATION: THERE -1.4795-02	ARE 29 5TA 6.642E-02	1.150F-01	05ED 1.764E-01	
FC)R 3	T TERA	ATION, THERE -1,430E=02	ARE >9 STA 7.719E-07	TIONS NOT CLO 1+272E+01	05E0 1.896E-01	
FC	R 4	ITER	TION: THERE -1.479E-02	ARE 29 STA 6.542E-02	TIONS NOT CLI 1.1506-01	SED 1.764F-01	
FC	R 5	17ER	ATION: THERE -1.490E-02	ARE 29 STA 6.719E-02	TINNS NOT CLU 1.1625-01	05E0 1.783E-01	
FC)R 6	ITER	ATION: THERE -1-485E-02	ARE 28 STA 6.794E-02	TIONS NOT CLO 1.1718-01	0SED 1.7925-ô1	
F()R 7	ITER/	TION, THERE	485 26 STA 6.7915-02	TIANS NAT CL(1.1745-01	D\$ED 1+7915+ñ1	
FC	R 8	ITER	ATION: THERE -1.485F-02	ARE 15 STA 6.791F-02	TIONS NOT CLU 1_1708-01	05ED 1.7915-01	
FC)R 9	TTER	ATION: THERE -1.485E-02	APE & STA 5.791E-02	TIONS NOT CLO 1.1705-01	0SED 1,791F-01	
			TIM	E ST4 # #	9		
STA J	1	DIST	DEFL	SLOPE	404	SHEAR	SUP REACT
				-7.460F-0	5	-6.3105.00	

24	1+152E+03	4.348E-01	-2,178E+15	1 7000.67	0.
25	1.200E+03	4.015E-01	-7.05AE+05	A 0045-02	0.
24	1.248E+03	3.485E-01	=1,813E+05	5.8855+02	0.
27	1.296E.03	2.7815-01	-1,460E+05	7-3615-02	ο.
24	1.344E+03	1.936E=01	-1,022E+05	8.3765+02	٥.
29	1.392E+03	9.932E-02	-5.262E+04	8.8925+02	0.
30	1.440E+03	0.	-2.069F=03	9.6375-11	-8.892E+02
31	1.488E+03	-9.932E=02	0.		0.

TIME STA # = 11 SHEAR SUP REACT STA J OTST NEFL SLOPF MOM -6.527F+04 -7.132F+n2 8 . 144E . 02 3.840E+02 -2.036E-02 -1.828E+05 -2.369E-03 A 5.482F+62 6.2496+03 15 1.200E+02 -6+332E-01 ο. -2.1576-03 8.223F+02 7.680E+02 -7.464E-01 5.703E+04 3.285E.03 16 -2,243F-03 8.140E+02 -8.540E-01 3.448F+03 2,3136+05 3.716E.03 17

STA J	OIST	nEFL	SLOPF	HOH	SHEAR	SUP REACT
			-2.732F-04		-2.7995+62	
8	3+840E+02	-1+738E-02		-6.771E+04		6+952E+02
			-3.9775-04		3.847F*02	
15	7.200E+02	-1.844E-01		A, 785E+84		σ.
			-2,620F-04		4,5905+02	
16	7.6R0E+02	-1.970E-01		9,1005+04		8,1956.02
		_	-7.0A1F-05		1,9456+03	
17	8.140E+02	-2.0088-01	•	1,848E+*5		8.958E+05

STATIONS 16 17

TIME STA # = 11

MONITOR

FOR 2 ITERATION, THERE ARE 29 STATIONS NOT CLOSED

FOR 3 ITERATION, THERE ARE & STATIONS NOT CLOSED

15 FOR & ITERATION, THERE ARE 29 STATIONS NOT CLOSED

-2.115E-02 -6.451E-01 -7.590E-01 -8.668E-01

-2.036E-02 -6.332E-01 -7.464E-01 -8.540F-01

-2,036E+02 -6,332E-01 -7.464E-01 -8.540F-01

J #

8

TIME STAK = 10

FOR 2 ITERATION, THERE ARE & STATIONS NOT CLOSED -1.7385-02 -1.8445-01 -1.9705-01 -2.0085-01

FOR I ITERATION, THERE ARE 29 STATIONS NOT CLOSED -1.738E-02 -1.844E-01 -1.970F-01 -2.008F-01

STATIONS 15 17 MONITOR J= 8 15

TIME STAR = 10

R	3.8495.02	-1.485E-02		-5.331E+03		5.940E.02
15	7.200E.02	6.791E-02	7.712E-04	1,2596+05	3.0715+02	٥.
14	7.640E.02	1,170E=01	1.001-03	1,354E+05	5,0046+02	-7.505E+02
17	8.160E.02	1.791E-01	1.7446.003	1.574F+05	3.4016.00	_1,007E+03

HOH SHEAD

-1.095F-02 +1.439E-01 -1.378E-01 -1.096F-01 FOR 3 ITERATION, THERE ARE & STATIONS NOT CLOSED

TIME STA K #

-1.099F-02 -1.445E-01 -1.102F-01 -1.102F-01 FOR 2 ITERATION, THERE ARE 29 STATIONS NOT CLOSED

FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED

HONTTOR STATIONS 17 is. 16

-1.095E-02 -1.439E-01 -1.378E-01 -1.096F-01

15

TIME STA # = 15

		-7, 116F-04		-7.239E+02	
3.840E.02	-3.422E-02		-1.828E+05		1•369E•03
		-1,AB8F-03		4.914F*n2	
7.200E+02	-5.840E-01		4.275F+94		0.
		-1. A04E-03		5,635F+02	
7.680E+02	-6.706F-01	-	7.796F+14		5°510E+03
		-1.44AF-03		2.838F+03	
8+160E+02	-7.497E-01		2.221F+05		2.159E+03
	3+840E+02 7.2n0E+02 7.6r0E+02 8+160E+02	3.840E+02 -3.422E-02 7.200E+02 -5.840E-01 7.680E+02 -6.706E-01 8.160E+02 -7.497E-01	-7.136F-04 3.840E+02 -3.422E-02 7.200E+02 -5.840E-01 7.680E+02 -6.706E-01 8.160E+02 -7.497E-01	-7.176F-04 3.840E+02 -3.422E-02 7.200E+02 -5.840E-01 7.680E+02 -5.840E-01 7.680E+02 -6.706E-01 8.160E+02 -7.497E-01 -1.448E=03 2.221F+05	-7.936F-04 -7.239E+02 3.840E+02 -3.422E-02 -1.888F-03 4.914F+02 7.200E+02 -5.840E-01 4.225F+04 7.680E+02 -6.706E-01 7.796F+04 8.160E+02 -7.497E-01 2.838F+03 8.160E+02 -7.497E-01 2.221F+05

SHEAR SUP REACT STA J DIST DEFL SLOPF HOM

TIME STA # = 14

FOR 3 ITERATION. THERE ARE & STATIONS NOT CLOSED -3.422F-02 -5.84nE-01 -6.706F-01 -7.497E-01

FOR 2 ITERATION, THERE ARE 29 STATIONS NOT CLOSED -3.422E-02 -5.840E-01 -6.706E-01 -7.497F-01

FOR 1 ITERATION, THERE APE 29 STATIONS NOT CLOSED -3.344E-02 -5.676E-01 -6.521E-01 -7.297E-01

STATIONS MONITOR J = 15 16 17

TIME STA W 14

7	200400-01	314105-02	-2 OFHE 03		-7.2465.02	
15	7.200E+02	-8.339E-01	-3	-1.163F+15		٥.
			=3,290E=03		-1,1986+03	
16	7.6A0E+02	-9.919E-01		~1,581F+05		7.744E+03
17	8.160E+02	-1.145E+00	-3,607E-03	1.5478+05	6,156F+13	4.359E+03

SHEAR

-8.592F+02

MOM

-2 4335+05

SUP REACT

1.598E+03

TIME STA M # 1 7

SL OPF

-9.5675-04

DEEL

3.84.05+02 -3.0845-03

.) =

DIST

STA I

-

-1.491F-03 1.0165+03 9.600E+02 -1.719E+00 20 4.8915+05 ٥. -7.130F-04 7.2105+02 21 1.008E+03 -1.753E+00 5.2716+15 ø., 4.002F+02 3.5125-04 5.446E+05 23 1+056E+03 -1.736E+00 n 1,440F-03 6.144F.01 23 1.104F+03 -1.467F+00 5.407E+05 Ο. 2.522F-03 -2.845E+02 5.149E+05 24 1.152E.03 -1.546E.00 ٥. 3.525-03 -6.237E+ñ2 4.6795+05 25 1.200E+03 -1.375F+00 0. 4.487F-03 -9.405F+02 4,012E+05 24 1.2485+03 -1.1605+00 0. 5.290E-03 -1,219F+03 3,173F+05 1.296E+03 -9.061F=01 27 0. 5.925F-03 -1,443E.03 2,197E+05 1.344E+03 -6.217E-01 2A ٥. 6.364F-03 -1.600E+03 1,123E+05 1.392E+03 -3.162F-01 29 0. A.549F-03 -1.681F+03 30 1.4405.03 -9,252E-09 1.681++03 0. 1.9275-10 6.589F-03 1.498E+03 3.162F=01 ۵. 31 ٥.

TIME STAK = 13

MONITOR

* USED ADJUSTED LOAD-DEFL CURVE TO CALCULATE DEFLECTIONS * FOR 1 ITERATION, THERE APE 29 STATIONS NOT CLOSED

FOR 2 ITERATION, THERE ARE & STATIONS NOT CLOSED

FOR 2 ITERATION, THERE ARE 29 STATIONS NOT CLOSED

FOR 3 ITERATION, THERE ARE 29 STATIONS NOT CLOSED

FOR 4 ITERATION, THERE ARE 29 STATIONS NOT CLOSED

FOR 5 ITERATION, THERE ARE 29 STATIONS NOT CLOSED

FOR 6 ITERATION, THERE ARE 28 STATIONS NOT CLOSED

FOR 7 ITERATION, THERE ARE 17 STATIONS NOT CLOSED

FOR 8 ITERATION, THERE ARE & STATIONS NOT CLOSED

15

-5.080E-02 -9.677E-01 -1.130E+00 -1.296E+00

-5.4995-02 -1.0375+00 -1.2055+00 -1.3745+00

-5.080E-02 -9.677E-01 -1.130E-00 -1.296E+00

-5.499F-02 -1.037E+00 -1.205E+00 -1.374E+00

-5.395E+02 -1.020E+00 -1.186F+00 -1.354F+00

-5.403F-02 -1.n2jE+nn +1.j84F+00 +1.356F+nn

-5.404E+02 -1.021E+00 -1.184F+00 -1.356E+00

-5.404F-02 -1.021F+00 -1.189E+00 -1.356F+00

-3,996F-02 -8,339E-01 -9,919F-01 -1,165F+00

16

J =

A

STATIONS 17 FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED -5.080F-02 -9.677E-01 -1.130E+00 -1.296F+00

7.0445-03 1.9468-01 2.5925-01 3.3845-01

FOR 17 ITERATION, THERE ARE 29 STATIONS NOT CLOSED 7.160F-03 1.944E-01 2.592F-01 3.384F-n1

FOR 18 ITERATION, THERE ARE DA STATIONS NOT CLOSED 7.252F=03 1.945E=01 2.592E=01 3.384F=01

FOR 19 ITERATION, THERE ARE DB STATIONS NOT CLOSED 7.326F-03 1.945E-01 2.592F-01 3.384F-01

FOR 20 ITERATION, THERE ARE OR STATIONS NOT CLOSED 7,385E-03 1,948E-01 2.892E-01 3.384F-01

FOR 21 ITERATION. THERE ARE \$7 STATIONS NOT CLOSED 7.432E-03 1.945E-01 2.592E-01 3.384F-01

FOR 22 ITERATION, THERE ARE 27 STATIONS NOT CLOSED 7.469F-03 1.944E-01 2.893F-01 3.384F-01

FOR 23 ITERATION, THERE ARE \$7 STATIONS NOT CLOSED 7.499F-03 1.944E-01 2.593E-01 3.384F-01

FOR 24 ITERATION, THERE ARE 25 STATIONS NOT CLOSED 7.522E-03 1.944E-01 2.593E-01 3.384E-01

FOR 25 ITERATION, THERE ARE >3 STATIONS NOT CLOSED 7.541E-03 1.944E-01 2.593E-01 3.384F-01

FOR 26 ITERATION, THERE ARE 22 STATIONS NOT CLOSED 7.5566-03 1.9446-01 2.6936-01 3.3846-01

FOR 27 ITERATION, THERE ARE 20 STATIONS NOT CLOSED 7.568E-03 1.944E-01 2.593F-01 3.384E-01

FOR 28 ITERATION, THERE ARE 36 STATIONS NOT CLOSED 7.578E-03 1.946E-01 2.593E-01 3.384E-01

FOR 29 ITERATION. THERE ARE 15 STATIONS NOT CLOSED. 7.585E-03 1.944F-01 2.593E-01 3.385F-01

FOR 30 ITERATION. THERE ARE 14 STATIONS NOT CLOSED 7.592E-03 1.944E-01 2.593E-01 3.385E-01

FOR 31 ITERATION, THERE ARE 14 STATIONS NOT CLOSED 7.596E-03 1.944E-01 2.593E-01 3.385E-01

FOR 32 ITERATION, THERE ARE TO STATIONS NOT CLOSED 7.600F-03 1.946F-01 2.593F-01 3.385F-01

FOR 33 ITERATION, THERE ARE 12 STATIONS NOT CLOSED 7.603F-03 1.944E-01 2.593E-01 3.385F-01

FOR 34 ITERATION, THERE ARE 11 STATIONS NOT CLOSED

7.606F-03 1.946E-01 2.593F-01 3.385F-01 FOR 35 ITERATION, THERE ARE 9 STATIONS NOT CLOSED

7.608F-03 1.944E-01 7.893F-01 3.385F-01

FOR 36 ITERATION, THERE ARE 7 STATIONS NOT CLOSED 7.609E-03 1.946F-01 2.593E-01 3.385F-01

FOR 12 ITERATION, THERE ARE 29 STATIONS NOT CLOSED 6.200F-03 1.941E-01 2.590F-01 3.383F-01

FOR 13 ITERATION. THERE ARE 29 STATIONS NOT CLOSED

FOR 14 ITERATION, THERE ARE 29 STATIONS NOT CLOSED

FOR 15 ITERATION, THERE ARE 29 STATIONS NOT CLOSED

FOR 16 ITERATION: THERE ARE 29 STATIONS NOT CLOSED

5.839F-03 1.946E-01 2.689E-01 3.382E-01

6.487E-03 1.942E-01 2.490E-01 3.383F-01

0.716F-03 1.943E+01 2.591E+01 3.383E+01

6.898E-03 1.944E-01 2.691F-01 3.384F-01

FOR 11 ITERATION, THERE ARE 39 STATIONS NOT CLOSED

FOR 10 ITERATION, THERE ARE 29 STATIONS NOT CLOSED 5,387E-03 1.934E-01 2.684E-01 3.382F-01

FOR 9 ITERATION, THERE ARE 29 STATIONS NOT CLOSED 4.819E-03 1.937E-01 2.587F-01 3.381F-01

FOR B ITERATION. THERE ARE 29 STATIONS NOT CLOSED 4.107E-03 1.934E-01 2.585E-01 3.380E-01

FOR 7 ITERATION, THERE ARE 29 STATIONS NOT CLOSED 3.214E-03 1.931E-01 2.583F-01 3.379F-01

2.0955-03 1.9235-01 2.6755-01 3.3725-01

FOR 6 ITERATION, THERE ARE 29 STATIONS NOT CLOSED

FOR 5 ITERATION, THERE ARE 29 STATIONS NOT CLOSED 7.264F-04 1.79AE-01 2.432E-01 3.224E-01

FOR & ITERATION, THERE ARE 29 STATIONS NOT CLOSED -2.174E-02 1.586E-01 2.351F-01 3.164F-01

5.534F-04 1.76hE-01 2.399E-01 3.191F-01

FOR 3 TTERATION, THERE ARE 29 STATIONS NOT CLOSED

FOR 2 ITERATION. THERE ARE 20 STATIONS NOT CLOSED -2.174E-02 1.444E-01 2.351E-01 3.164E-01

FOR 1 ITERATION, THERE ARE 39 STATIONS NOT CLOSED 5.534F-04 1.76AF-01 2.399E-01 3.191E-01

17 16 j = A 15

MONITOR STATIONS

TIME STA K . 16

7.680E+02 -1.378E-01 2,3138+05 16 5.882E-04 +4.396F+02 2.0746+05 -4.011E+02 8.160E.02 -1.096E-01 17

-2.074F-04

-5.922F+04 4,378E+02 3.840E+02 -1.095E-02 A -1.861E-04 1,235F+03 1,558E+05 ٥. 7.200E+02 -1.439E-01 15 1.2568-04 1,585E+03 -1.435E+0)

-2.756F+82

-3.739F-02 -3.899E-01 -4.511E-01 -5.0685-01

-3.483F-02 -3.654E-01 -4.249F-01 -4.802F-01

FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED

FOR 2 ITERATION, THERE ARE 29 STATIONS NOT CLOSED

MONITOR STATIONS J = 15 15 17

TIME STA * = 1.9

6.110E-04 -7.083E+01 4.177E+04 _A. 385E .02 7.6A0E+02 1.064E-01 16 6.945E-04 1.579F+03 1.143E+05 -1.200E+03 8.149E+02 1.39AE+01 17

1.5918.02 3.840E*02 *3.977E-03 1.222E*04 A, 5.147E-04 -1.584F+01 4.810F+04 15 7.200E.02 7.704E-02 Λ.

STA J DIST SHEAD SUP REACT DEFI SLOPE мом 1.015F+02 3.454F=06

-3.977F-03 7.709E-02 1.064F-01 1.398F-01 TIME STA # = 17

-3.977E-03 7.709E-02 1.064E-01 1.398E-01 FOR 2 ITERATION, THERE ARE & STATIONS NOT CLOSED

FOR 1 TTERATION, THERE ARE 29 STATIONS NOT CLOSED

FOR 3 ITERATION. THERE ARE & STATIONS NOT CLOSED -6.020E-03 4.335E-02 4.964E-02 1.019E-01 . USED ADJUSTED LOAD-DEFL CURVE TO CALCHLATE DEFLECTIONS

-0.020E-03 4.335E-02 6.968E-02 1.019E-01

FOR 2 ITERATION. THERE ARE 29 STATIONS NOT CLOSED

FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED -1.726E-02 3.957E-02 4.717E-02 1.005E-01

17 .t. = 15 15

MONITOR STATIONS

TIME STA K # 17

-2.A26F-03 1.029F+03 -1,4405+05 1.344E.03 2.868E-01 -2.914F-03 1,166F+03 1+392E+03 -7.494E+04 0. 1.470E-01 -3.062E-03 1.236E+03 -4.626E-09 -1.236E+03 Зn 1.440E+03 0. -3.1625-03 9.6375-11 0. Зĩ 1.488E+03 -1.470E-01 Π.

-2.214F-03 8.266F+02 -2.0475+05 27 1.296E+03 4.129E-01 0. 2 A 29

-1.11AF-03 2,592F+02 6.008E-01 1.200E+03 -2.917E+05 0. 25 -1.701E-03 5,670F+02 -7.5635+05 26 1.24 BE + 03 5.192E-01 0.

FOR 38 ITERATION, THERE ARE IN STATIONS NOT CLOSED 7.612E-03 1.944E-01 2.593E-01 3.385E-01

TIME STA # =

DEFL

1.671E-03

7.612E-03

1.6605-02

2.942E-02

4.707E-02

7.080E-02

1.021E=01

1.428E=01

1.946E-01

2.593E-01

3.3A5E+01

4.2486-01

5.092F-01

5.833E-01

6.398E-01

6.728E+01

6.784E-01

6.545F-01

-4.800E+01 -5.278E+04

4.800E.01 -5.642E-04

9.600E.01 -1.703E-03

1.440E.02 -2.899E-03

1.920E+02 -3.645E-03

2.400E.02 -3.451E-03

2.880E+02 -1.840E-03

3.360E.02

3.840E+02

4.320E+02

4.800E.02

5.2A0E.02

5.760E+02

6.240E+02

6.720E.02

7.200E.02

20+3089+1

8.160E.02

8.640E.02

9.1208.02

9.600E+02

1.008E+03

1.056E+03

1.104F.03

1.1525.03

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DIST

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7.6115-03 1.9445-01 2.5935-01 3.3855-01

SLOPF

1.100E-05

-1. 175E-05

-2,373F-05

-2,492F-05

-1.554F-05

4, ñ41g-06

3,356E-05

7.3168-05

1,73Ag=04

1.A73E-04

2.471E-04

3.476E-04

4.944E-04

4.529E-04

8.477E-04

1.079==03

1,1486-03

1,6495-03

1,7995-03

1.7595-03

1.5445-03

1.176F-03

6.A85F-04

1.1665-04

-4.991F-04

FOR 37 TTERATION, THERE ARE 5 STATIONS NOT CLOSED

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мом

-5.688F*03

-5.988E+03

-5.926E.02

4.699F+03

9.7996.03

1.476E+04

1.9802+04

2.530E+04

3.178E+04

3.9886+04

5.0?7E+04

6.341E+04

7.924E+04

9.715E+04

1.159E+05

1.342E+05

1.5096+05

7.474E+04

-1.994E+04

-1.0765+05

-1.836E+05

-2.440E+05

-2.859£+15

-3,^{079F+05}

-3.0956+05

SHEAR

=1 +185F +ñ2

-7.430F+00

1,100F.02

1.0756.02

1,0475+02

1.0395+02

1.0846.02

1.216F+02

1.4745+02

1.8745+02

2.4335.02

3,1045+02

3,7945.02

4,3825+02

4.7635+02

4.8875+02

4.8255+02

+1+455E+03

-1.793F+ŕ3

-1.6510+03

~1,429F+03

-1.140E+03

-8.047E+02

-4.450F+02

-8.370F+01

SUP REACT

1.111E*02

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~1.603E+03

-2.193E+03

FOR 1 ITERATION, THERE APE 39 STATIONS NOT CLOSED

. USED AD JUSTED LOAD DEFL CHRVE TO CALCULATE DEFLECTIONS .

FOR 27 ITERATION, THERE ARE O STATIONS NOT CLOSED 1.002E-02 1.194E-01 1.979E-01 3.008E-01

1.002E-02 1.19AE-01 1.979F-01 3.008F-01

FOR 26 ITERATION, THERE ARE 5 STATIONS NOT CLOSED

1.002E-02 1.194E-01 1.979E-01 3.004E-01

FOR 25 TTERATION, THERE ARE 7 STATIONS NOT CLOSED

1.002F-02 1.194E-01 1.979F-01 3.009F-01

FOR 24 ITERATION, THERE ARE . 9 STATIONS NOT CLOSED

1.001E-02 1.194E-01 1.979E-01 3.008E-01

FOR 23 ITERATION, THERE ARE IN STATIONS NOT CLOSED

1.001F-02 1.194F-01 1.979F-01 3.008F-01

FOR 22 ITERATION, THERE ARE 12 STATIONS NOT CLOSED

1.001E-02 1.194E-01 1.979E-01 3.008E-01

FOR 21 TTERATION, THERE ARE 13 STATIONS NOT CLOSED

1.000E-02 1.194E-01 1.979F-01 3.008F-01

FOR 20 ITERATION. THERE ARE 14 STATIONS NOT CLOSED

4.999E-03 1.196E-01 1.979E-01 3.008F+01

FOR 14 ITERATION, THERE ARE 14 STATIONS NOT CLOSED

FOR 18 ITERATION, THERE ARE IS STATIONS NOT CLOSED 9.993E-03 1.194E-01 1.979F-01 3.008F-01

9.986E-03 1.194E-01 1.979E-01 3.008E-01

FOR 17 ITERATION. THERE ARE TO STATIONS NOT CLOSED

9.976F-03 1.194E-01 1.979F-01 3.008F-01

FOR 16 ITERATION, THERE ARE ON STATIONS NOT CLOSED

FOR 15 ITERATION, THERE ARE 22 STATIONS NOT CLOSED 9.964F-03 1.194E-01 1.979F-01 3.008F-01

FOR 14 ITERATION, THERE ARE 24 STATIONS NOT CLOSED 4.949E-03 1.194F-01 1.079F-01 3.008F-01

9,931E-03 1,194F-01 1,979E-01 3,008F-01

9.907F-03 1.194E-01 1.979E-01 3.008F-01 FOR 13 ITERATION, THERE ARE 25 STATIONS NOT CLOSED

FOR 12 ITERATION, THERE ARE \$7 STATIONS NOT CLOSED

9,878E-03 1.194E-01 1.979E+01 3.009F-01

FOR 11 ITERATION, THERE ARE \$7 STATIONS NOT CLOSED

7.841E-03 1.194E-01 1.079F-01 3.008F-01

#.795F-03 1.194E-01 1.979E-01 3.008F-01 FOR 10 ITERATION, THERE ARE OF STATIONS NOT CLOSED

FOR & ITERATION, THERE ARE OR STATIONS NOT CLOSED

FOR & TTERATION, THERE ARE DO STATIONS NOT CLOSED 9.737F-03 1.19cE-01 1.979F-01 3.008E-01

STATIONS MONITOR 17 ງ = 16 8 15

-1.015F-04 1.150F-01 1.043F-01 2.081F-01

1.965E-03 1.157E-01 1.948E-01 2.983E-01

9_4295-03 1.284E-01 2.0785-01 3.111E-01

9.571F-03 1.180E-01 1.982E-01 3.011E-01

9.664E-03 1.195E-01 1.979E-01 3.008F-01

FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED

-1.015E-04 1.15nE-01 1.943E-01 2.981E-01

FOR 2 ITERATION. THERE ARE 29 STATIONS NOT CLOSED

1.965F-03 1.157E-01 1.948F-01 2.983E-01 FOR 3 ITERATION, THERE ARE PO STATIONS NOT CLOSED

FOR 4 ITERALION, THERE ARE DO STATIONS NOT CLOSED

FOR 5 ITERATION. THERE ARE 29 STATIONS NOT CLOSED

FOR 6 ITERATION, THERE ARE 29 STATIONS NOT CLOSED

FOR 7 TTERATION: THERE ARE 29 STATIONS NOT CLOSED

TIME STA # = 24

4.439F-04 1.018E+05 3.8402+02 8.5611-03 1.5176-03 1.6208+03 1.8445+05 3,4338=01 7.2005.02 1.AR6E-03 2.2945+03 2,8555+05 -2.381E+03

2.457E-03

SLOPF

FOR 3 ITERATION, THERE ARE a STATIONS NOT CLOSED 8.561F-03 3.433E-01 4.539E-01 5.718E-01

TIME STA K .

OFFI

4.539E=01

5.71AE-01

DIST

7.6902.02

8.160E.02

STA J

15

16

17

2 ITERATION. THERE ARE 29 STATIONS NOT CLOSED FOR 8,561F-03 3.633F-01 4.539F-01 5.718E-01

FOR 1 ITERATION, THERE ARE 29 STATIONS NOT CLOSED 1.290F-02 3.648E-01 4.548F-01 5.723E-01

23

MOM

1.7476+05

STATIONS MONITOR 17 J = 1 = 16 8

TIME STA K # 27

7.200E+02 1.489E-01 -A.300F+03 15 8.9615+12 3.943F=04 3.487E+04 -1.2356+03 1.473E-01 7.680E+02 14 4,541F-04 4.935.02 -1,596E+03 8.160E.02 1.891E-01 5.660E+04 17

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SUP REACT

-3.127E+03

SHEAP

1.7715-02

-2,063F+03

SUP REACT

=3.985E+04 1+228E+03 3+840E+02 -3+069E-02 A

STA J DIST DEFL SLOPE MOM

SHEAR

-3.069E-02 +3.255E+01 -4.270E-01 -5.527E-01 TIME STA # = 24

-3.069E-02 -3.255E-01 -4.270E-01 -5.527E-01 FOR 2 ITERATION, THERE ARE & STATIONS NOT CLOSED

FOR 1 ITERATION, THERE ARE 39 STATIONS NOT CLOSED

STATIONS MONTTOR 17 . . 15 14

TIME STA # # 26

-2.495E-04 8.140E.02 -2.187F-01 1.4195+03 9.7445+03 6.354E+02 17

-1,502#=04 -1.551F+03 -5,9456+04 7.680E.02 -2.058E-01 8.575E+02 16

1.40AF+04 7.200E+02 -1.986E-01 0. 15

1.135E+03 3+840E+02 -2+837E-02 -A.175F+04 A -1.7845-04 -9.609E+05

-4,495F-04 +4.693E+01

SUP REACT мон SHEAD STA J 0157 DEFL SLOPF

TINE STA # # 25

FOR 3 ITERATION. THERE ARE & STATIONS NOT CLOSED -2.8375-02 -1.986E-01 -2.058E-01 -2.187E-01

FOR 2 ITERATION, THERE ARE 29 STATIONS NOT CLOSED -2.837E-02 -1.994E-01 -2.658F-01 -2.187E-01

-7,516F+02 -2.391F-01 -2.430E-01 -2.520E-01

FOR 1 TTERATION. THERE ARE 29 STATIONS NOT CLOSED

STATIONS MONTTOR 17 J = 15 16

-1.8505-08 -1.948F+03 1.440E.03 ۵. 3.8555-10 -4.175--03 1+485E+03 -2+004E-01 ۰. ۸. TTME STA K # 26

-7.200F+05 3.8995-01 η, 20 1.344F+03 -3,048E+03 1.8230+03 2.004F+01 -1.135E+05 0. 1.3426.03 20 -4.175F=03 1.0485.03 30

+1.452E+05 ο. 6.966F-01 1.249F+03 1.2245.03 -2.ca3r-03 -3.126F+95 5.5A2F-01 Ω. 1.2966.03 -3.5085-03 1.5796+03

-4,3256.05 Ο. 1.200E.03 7.080E-01 7.7505.02 -2.112F-03 25 27

25

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SUP REACT

-4.333E-01

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-1.258E+03

-1,936£+03

3.0955-03 1.4795-01 2.2935-01 3.3405-01 FOR 2 ITERATION. THERE APE IN STATIONS NOT CLOSED 3.045F+03 1.477E-01 2.293E+01 3.340F+01

SLOPF

-3.016E-06

3.7245-06

9.3734-06

1.4985-05

1.9885-05

1.928g-05

1.417E-05

5.031E-06

-1.745E-05

-4.278E-05

-4,5998-06

1, -19E-04

2.a165.04

5.372F=04

A. 673E-04

1.2A3F-03

1.7095-03

2. 182E-03

2.511E-03

2.576F-03

2,382F=03

1,948E-03

1.733E-03

24

NON

1.560E*03

3.074E+03

2.8045.03

1,9516*03

1,970€+02

-2,5555+03

-6.068E+03

=9,739F+03

-1,266E+04

1,9098+04

5.3755+04

8,983E+04

1.2786+05

1,650E+05

1.9806+05

2,2296+05

2.3645+05

1,6452+05

3,2746.04

-9.7472+04

-2,149E+05

-3,1735+05

-3,922F+95 5,288F=04

-4,373<u>E</u>+45 -3,459E-04

-1.247E-03

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SHEAD

3.250F+01

3,1876.01

-4.6945.00

-1.629F+61

-3.4648+01

-5.540F+01

-7.178E+01

-7.6285+01

-6.268£+01

6.572E+02

7.1126.05

7.7247.02

8.1995.12

8.2835.02

7,7346+02

6.4475+02

4,5335+02

=1.280E+03

-2.4945.03

-2.454F+03

-2,2505+03

-1.8988+03

-1.4305+03

-8.868E+02

-3.123F+02

2,5340+02

TTHE STA # =

DEFL

1.448F-04

1.54HE-04

6.047E-04

1.3246-03

2.230F-03

3.155E-03

3.836E-03

3.933E+03

3.0956-03

1.042E-03

8,2156-04

5,713E=03

1.923E-02

4.501F-02

8.664E-02

1.473E-01

2.2936-01

3.340E-01

4.546E-01

5.792E-01

6.975E-01

7.840E-01

8.491E-01

8.7455-01

8.5796-01

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DIST

-4.800E.01

4.800E+01

9.6n0E+01

1.4406.02

1.920E+02

2,400E+02

2.840E+02

3.3406.02

3.840E.02

4.3206.02

4.800E+02

5.280E.02

5.7502.02

6.2406.02

6.720E+02

7,2006+02

7.6406.02

8.140E+02

8.6406.02

9.120E.02

9.600E.02

1.098E+03

1.0566.03

1.1046.03

1.152E.03

0.

This page replaces an intentionally blank page in the original --- CTR Library Digitization Team

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