

LINEARLY ELASTIC ANALYSIS OF PLANE FRAMES
SUBJECTED TO COMPLEX LOADING CONDITIONS

by

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Development of Methods for Computer Simulation
of Beam-Columns and Grid-Beam and Slab Systems

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PREFACE

This report presents an analytical tool for the solution of plane-frame structures. The computer program developed for the solution is intended to be well suited for solving complex structures with a minimum of hand calculations.

The solution techniques developed rely on the linearly elastic behavior that many plane frames exhibit under design loads. This linearity allows the user to solve a structure for many loading cases and combinations of loading cases at a cost only slightly greater than that of a single solution.

This is the twenty-first in a series of reports that describe work under Research Project No. 3-5-63-56, "Development of Methods for Computer Simulation of Beam-Columns and Grid-Beam and Slab Systems". Reports No. 56-1, 56-3, 56-4, 56-7, and 56-14 provide background information for this report.

Duplicate copies of the program deck and test data cards for the example problems in this report may be obtained from the Center for Highway Research, The University of Texas at Austin.

Thanks are due to the members of the staff of the Center for Highway Research for their assistance in producing this report.

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LIST OF REPORTS

Report No. 56-1, "A Finite-Element Method of Solution for Linearly Elastic Beam-Columns" by Hudson Matlock and T. Allan Haliburton, presents a finite-element solution for beam-columns that is a basic tool in subsequent reports.

Report No. 56-2, "A Computer Program to Analyze Bending of Bent Caps" by Hudson Matlock and Wayne B. Ingram, describes the application of the beam-column solution to the particular problem of bent caps.

Report No. 56-3, "A Finite-Element Method of Solution for Structural Frames" by Hudson Matlock and Berry Ray Grubbs, describes a solution for frames with no sway.

Report No. 56-4, "A Computer Program to Analyze Beam-Columns under Movable Loads" by Hudson Matlock and Thomas P. Taylor, describes the application of the beam-column solution to problems with any configuration of movable non-dynamic loads.

Report No. 56-5, "A Finite-Element Method for Bending Analysis of Layered Structural Systems" by Wayne B. Ingram and Hudson Matlock, describes an alternating-direction iteration method for solving two-dimensional systems of layered grids-over-beams and plates-over-beams.

Report No. 56-6, "Discontinuous Orthotropic Plates and Pavement Slabs" by W. Ronald Hudson and Hudson Matlock, describes an alternating-direction iteration method for solving complex two-dimensional plate and slab problems with emphasis on pavement slabs.

Report No. 56-7, "A Finite-Element Analysis of Structural Frames" by T. Allan Haliburton and Hudson Matlock, describes a method of analysis for rectangular plane frames with three degrees of freedom at each joint.

Report No. 56-8, "A Finite-Element Method for Transverse Vibrations of Beams and Plates" by Harold Salani and Hudson Matlock, describes an implicit procedure for determining the transient and steady-state vibrations of beams and plates, including pavement slabs.

Report No. 56-9, "A Direct Computer Solution for Plates and Pavement Slabs" by C. Fred Stelzer, Jr., and W. Ronald Hudson, describes a direct method for solving complex two-dimensional plate and slab problems.

Report No. 56-10, "A Finite-Element Method of Analysis for Composite Beams" by Thomas P. Taylor and Hudson Matlock, describes a method of analysis for composite beams with any degree of horizontal shear interaction.

Report No. 56-11, "A Discrete-Element Solution of Plates and Pavement Slabs Using a Variable-Increment-Length Model" by Charles M. Pearre, III, and W. Ronald Hudson, presents a method of solving for the deflected shape of freely discontinuous plates and pavement slabs subjected to a variety of loads.

Report No. 56-12, "A Discrete-Element Method of Analysis for Combined Bending and Shear Deformations of a Beam" by David F. Tankersley and William P. Dawkins, presents a method of analysis for the combined effects of bending and shear deformations.

Report No. 56-13, "A Discrete-Element Method of Multiple-Loading Analysis for Two-Way Bridge Floor Slabs" by John J. Panak and Hudson Matlock, includes a procedure for analysis of two-way bridge floor slabs continuous over many supports.

Report No. 56-14, "A Direct Computer Solution for Plane Frames" by William P. Dawkins and John R. Ruser, Jr., presents a direct method of solution for the computer analysis of plane frame structures.

Report No. 56-15, "Experimental Verification of Discrete-Element Solutions for Plates and Slabs" by Sohan L. Agarwal and W. Ronald Hudson, presents a comparison of discrete-element solutions with the small-dimension test results for plates and slabs, along with some cyclic data on the slab.

Report No. 56-16, "Experimental Evaluation of Subgrade Modulus and Its Application in Model Slab Studies" by Qaiser S. Siddiqi and W. Ronald Hudson, describes an experimental program developed in the laboratory for the evaluation of the coefficient of subgrade reaction for use in the solution of small dimension slabs on layered foundations based on the discrete-element method.

Report No. 56-17, "Dynamic Analysis of Discrete-Element Plates on Nonlinear Foundations" by Allen E. Kelly and Hudson Matlock, presents a numerical method for the dynamic analysis of plates on nonlinear foundations.

Report No. 56-18, "Discrete-Element Analysis for Anisotropic Skew Plates and Grids" by Mahendrakumar R. Vora and Hudson Matlock, describes a tridirectional model and a computer program for the analysis of anisotropic skew plates or slabs with grid-beams.

Report No. 56-19, "An Algebraic Equation Solution Process Formulated in Anticipation of Banded Linear Equations" by Frank L. Endres and Hudson Matlock, describes a system of equation-solving routines that may be applied to a wide variety of problems by utilizing them within appropriate programs.

Report No. 56-20, "Finite-Element Method of Analysis for Plane Curved Girders" by William P. Dawkins, presents a method of analysis that may be applied to plane-curved highway bridge girders and other structural members composed of straight and curved sections.

Report No. 56-21, "Linearly Elastic Analysis of Plane Frames Subjected to Complex Loading Conditions" by Clifford O. Hays and Hudson Matlock, presents a design-oriented computer solution of plane frame structures that has the capability to economically analyze skewed frames and trusses with variable cross-section members randomly loaded and supported for a large number of loading conditions.

ABSTRACT

A linearly elastic solution for the computer-aided analysis of plane frames is presented. The computer program which is developed features input formats which greatly reduce the manual preparation of data.

The solution uses a variation of the basic discrete-element beam-column model for the evaluation of member stiffness and fixed-end-force properties. The new discrete-element model allows flexural and axial rigidity as well as lateral, axial, and rotational values of loading and elastic restraint to vary randomly along the length of the member. Input is not restricted to values lumped at certain discrete stations but may be input in normal engineering values at any point on the member. In addition, options provided do not require the transferring of loads and dimensions from one axis to another by the user.

Frame displacements are obtained by standard matrix techniques modified to utilize the time and storage reductions possible for linearly elastic plane frames. The frame geometry may vary randomly and still be input in a simple and straightforward manner.

Options are provided to permit the analysis of a structure for several loading cases and combinations of cases with a minimum of new input and computer time.

KEY WORDS: structural engineering, frame analysis, plane frames, computer program, discrete element, soil-structure interaction, matrix analysis.

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SUMMARY

A computer program for the direct linearly elastic analysis of complex bridge bents and other highway structures has been developed and is reported herein. Rigid frames, trusses, continuous beams, and other planar structures may be analyzed using the program.

The beam-column model previously reported has been modified so that loads and restraints may act both normal and parallel to the members of the frame. This allows the designer to consider vertical or inclined piles as an integral part of the frame, even if the pile is supported by lateral and axial soil restraints.

The description of loads, cross-sectional properties, and soil supports is completely general as in previous beam-column models. In addition, the loads and changes in cross-sectional properties and soil supports may be specified at any point along the member.

The geometry of the frame and the directions of the loads may be input in a manner both natural and convenient to the designer.

Options are provided that permit the designer to analyze structures for the multitude of loading cases required by the AASHO code. These options allow the designer to consider a large number of loading conditions economically.

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IMPLEMENTATION STATEMENT

An extremely useful tool for the analysis of highway bridge structures has been developed in this study. The computer program described in the report is capable of handling large, skewed, randomly loaded plane-frame structures, and smaller and more regular structures may be input quickly for an economical solution. Rigid frames, beams, and trusses are analyzed by the same program.

In developing the program, emphasis was placed on maintaining complete generality of input. A skewed bridge bent with piles on lateral and axial soil supports can be easily input. At the same time, it was recognized that many frames are more regular and have simpler support conditions. These simpler problems can also be solved efficiently by the designer using the program.

One outstanding design-oriented feature of the program is its ability to superimpose the effects of a large number of loading conditions. A designer may consecutively run a dead-load analysis, a live-load analysis, a wind-load analysis, etc.; a program option then allows him to ask for any linear combination of these loadings he desires. Designers who have manually checked all the group loadings required by the AASHO specifications at their various unit stresses will appreciate this feature.

Further research in the area of linear analysis of planar structures does not appear warranted at this time. Future areas of research will be in nonlinear analysis and extensions to three-dimensional structures.

It is recommended that designers who have need for this program code some simple example problems in order to become familiar with its use. In addition, informal training sessions conducted by the research personnel would be extremely useful in implementing this work for immediate use by the Texas Highway Department.

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NOMENCLATURE

<u>Symbol</u>	<u>Typical Units</u>	<u>Definition</u>
A	in^2	Cross-sectional area of member
AE	lb	Axial rigidity of continuous element at any point
AE_i	lb	Axial rigidity of continuous element i at midpoint, used for discrete-element i
\overline{AE} , AE_1 , AE_2	lb	Effective value of AE and values of AE over partial lengths of elements
α	--	Cosine of angle between the x' and x -axes
β	--	Cosine of angle between the x' and y -axes
c	inches	Distance from nearest station to left of concentrated load to the load
c_1 , c_2	inches	Distance on element over which AE_1 , AE_2 , EI_1 , and EI_2 are acting
C_1 , C_2	--	Constants of integration
E	lb/in^2	Modulus of elasticity
EI	$\text{lb}\text{-in}^2$	Flexural rigidity of continuous element at any point
EI_i	$\text{lb}\text{-in}^2$	Flexural rigidity of continuous element i at midpoint, used for discrete-element i
\overline{EI} , EI_1 , EI_2	$\text{lb}\text{-in}^2$	Effective values of EI and values of EI over partial lengths of element
e	inches	Diameter of circle which contains loads which are being astatically equivalenced

<u>Symbol</u>	<u>Typical Units</u>	<u>Definition</u>
ϵ	inches and radians	Error term which represents the difference in displacements for a member composed of discrete-elements versus continuous-line elements
ϵ^*	inches and radians	Error term which represents the difference in displacements for one discrete element versus one continuous element
$\{f\}_i$	lb and lb-in	(6 x 1) matrix of end-forces on element i
$\{F\}_k$	lb and lb-in	(6 x 1) matrix of member-end-forces for member k measured in member coordinates
$\{F_i\}_k, \{F_j\}_k$	lb and lb-in	(3 x 1) matrix of member-end-forces at joint i, j for member k measured in member coordinates
$F_i^1, F_i^2, F_i^3,$ F_j^1, F_j^2, F_j^3	lb and lb-in	Member-end-forces at joint i, j measured in member coordinates (x'-force, y'-force, and moment about z'-axis, respectively)
$\{\bar{F}_i\}_k$	lb and lb-in	(3 x 1) matrix of member-end-forces at joint i for member k measured in structure coordinates
$\{\tilde{F}\}$	lb and lb-in	(3N x 1) matrix of frame joint loads measured in structure coordinates
$\{\tilde{F}_i\}$	lb and lb-in	(3 x 1) matrix of frame joint loads for joint i measured in structure coordinates
$\{FF\}_k$	lb and lb-in	(6 x 1) matrix of member fixed-end-forces measured in member coordinates
$\{FF_i\}_k, \{FF_j\}_k$	lb and lb-in	(3 x 1) matrix of member fixed-end-forces at joint i, j for member k measured in member coordinates
$\{\overline{FF}_i\}_k$	lb and lb-in	(3 x 1) matrix of member fixed-end-forces at joint i for member k measured in structure coordinates
h	inches	Distance between concentrated springs in discrete-element model, one-half of element's length

<u>Symbol</u>	<u>Typical Units</u>	<u>Definition</u>
i	--	Integer index
I	in^4	Moment of inertia of cross-section about member's z' -axis
j	--	Integer index
k	--	Integer index
$[k]_i$	lb/in and lb-in/rad	(6 × 6) element stiffness matrix for element i of member
$[k_{i-1,i-1}]_i$, $[k_{i-1,i}]_i$, $[k_{i,i-1}]_i$, $[k_{i,i}]_i$	lb/in and lb-in/rad	(3 × 3) submatrices of $[k]_i$ which relate forces at station of first inner subscript to displacements at station of second inner subscript
k_{pq}	lb/in and lb-in/rad	Element of stiffness matrix $[k]_i$ which represents the force corresponding to the p^{th} displacement due to a unit value of the q^{th} displacement
$[K]_k$	lb/in and lb-in/rad	(6 × 6) member stiffness matrix for member k measured in member coordinates
K_{pq}	lb/in and lb-in/rad	Element of stiffness matrix $[K]_k$ which represents the force corresponding to the p^{th} displacement due to a unit value of the q^{th} displacement
$[K_{ii}]_k$, $[K_{jj}]_k$	lb/in and lb-in/rad	(3 × 3) member stiffness matrix for member k measured in member coordinates which represents forces at i, j due to unit displacements at i, j
$[\bar{K}_{ii}]_k$	lb/in and lb-in/rad	(3 × 3) member stiffness matrix for member k measured in structure coordinates which represents forces at i due to unit displacement at i

<u>Symbol</u>	<u>Typical Units</u>	<u>Definition</u>
$[K_{ij}]_k, [K_{ji}]_k$	lb/in and lb-in/rad	(3 x 3) member stiffness matrix for member k measured in member coordinates which represents the forces at i, j due to unit displacements at j, i
$[\bar{K}_{ij}]_k$	lb/in and lb-in/rad	(3 x 3) member stiffness matrix for member k measured in structure coordinates which represents the forces at i due to unit displacements at j
$[\tilde{K}]$	lb/in and lb-in/rad	(3N x 3N) structure stiffness matrix measured in structure coordinates
$[\tilde{K}_{ii}]$	lb/in and lb-in/rad	(3 x 3) diagonal submatrix of $[\tilde{K}]$ which represents the forces at i due to the loads at i measured in structure coordinates
$[\tilde{K}_{ij}]$	lb/in and lb-in/rad	(3 x 3) off-diagonal submatrix of $[\tilde{K}]$ which represents the forces at i due to the loads at j measured in structure coordinates
γ	1/in	Slope of EI line divided by EI_i
L	inches	Length of member
m	--	Number of discrete-elements in frame member
M	--	Number of members intersecting at a joint
M	lb-in	Bending moment at any point in continuous element
M_1, M_2	lb-in	Bending moments at location of first and second rotational springs in discrete-element model
N	--	Number of joints in frame
p	--	Integer index
$[\tilde{p}_i]$	lb and lb-in	(3 x 1) matrix of forces acting at station i on member measured in member coordinates

<u>Symbol</u>	<u>Typical Units</u>	<u>Definition</u>
$\tilde{P}_i^1, \tilde{P}_i^2, \tilde{P}_i^3$	lb and lb-in	Concentrated forces acting at station i on a member, measured in member coordinates (x' -force, y' -force, and moment about z' -axis, respectively)
$\{\tilde{P}_i\}$	lb and lb-in	(3 × 1) matrix of applied forces at joint i measured in structure coordinates
$\tilde{P}_i^1, \tilde{P}_i^2, \tilde{P}_i^3$	lb and lb-in	Applied forces at joint i measured in structure coordinates (x -force, y -force, and moment about z -axis, respectively)
Ψ_1, Ψ_2	radians	Concentrated curvature (discrete angle changes) at first and second rotational springs in discrete-element model
q	--	Integer index
q_{ab}	lb/in and lb-in/in	Distributed load in the direction of the a -axis with its intensity per unit of length referenced to the b -axis, as $q_{x'x'}$, $q_{y'x'}$, $q_{z'x'}$, $q_{xx'}$, $q_{yx'}$, $q_{z'x'}$, q_{xy} , q_{yx} , and $q_{z'x'}$
Q_a	lb and lb-in	Concentrated load in the direction of the a -axis, as Q_x , $Q_{x'}$, Q_y , $Q_{y'}$, and Q_z , and $Q_{z'}$
RM, RO	--	Recursion multipliers used in recursion-inversion solution of simultaneous equations
$s_{x'}, s_{y'}, s_{z'}$	lb/in ² and lb/rad	Distributed elastic spring restraints parallel to members x' -axis, y' -axis, and acting about z' -axis
$\tilde{s}_i^1, \tilde{s}_i^2, \tilde{s}_i^3$	lb/in and lb-in/rad	Concentrated elastic spring restraints at station i on a member, measured in member coordinate (x' -restraint, y' -restraint, and rotational restraint about z' -axis, respectively)

<u>Symbol</u>	<u>Typical Units</u>	<u>Definition</u>
$\tilde{S}_i^1, \tilde{S}_i^2, \tilde{S}_i^3$	lb/in and lb-in/rad	Concentrated elastic spring restraints acting at frame joint i expressed in structure coordinates (x-restraint, y-restraint, and rotational restraint about the z-axis, respectively)
SMMT	lb/in and lb-in/rad	(13 × 1) storage matrix used by program to store 13 constants needed to generate $[K]_k$
$[T]_k$	--	(3 × 3) coordinate transformation matrix for member k
$[T]_k^t$	--	(3 × 3) matrix which is the transpose of $[T]_k$
T	lb	Value of thrust at any point in continuous element
T_i	lb	Axial thrust in discrete-element i
V_1, V_2	lb	Shear forces at locations of first and second rotational springs in discrete-element model
$\{w\}_i$	inches and radians	(6 × 1) matrix of end displacements for element i
$\{\tilde{w}_i\}$	inches and radians	(3 × 1) matrix of element displacements at station i measured in member coordinates
$\tilde{w}_i^1, \tilde{w}_i^2, \tilde{w}_i^3$	inches and radians	Displacements of station i measured in member coordinates (distances along x' , y' , and rotation about z' , respectively)
$\{W\}_k$	inches and radians	(6 × 1) matrix of member-end-displacements for member k measured in member coordinates
$\{W_i\}_k, \{W_j\}_k$	inches and radians	(3 × 1) matrix of member-end-displacements at joint i, j for member k measured in member coordinates
$w_i^1, w_i^2, w_i^3,$ w_j^1, w_j^2, w_j^3	inches and radians	Member-end displacements at joint i, j measured in member coordinates (distances along x' , y' , and rotation about z' -axis, respectively)

<u>Symbol</u>	<u>Typical Units</u>	<u>Definition</u>
$\{\bar{W}_i\}_k$	inches and radians	(3 × 1) matrix of member-end-displacements at joint i for member k measured in structure coordinates
$\{\tilde{W}\}$	inches and radians	(3N × 1) matrix of frame joint displacement measured in structure coordinates
$\{\tilde{W}_i\}$	inches and radians	(3 × 1) matrix of frame joint displacements at joint i measured in structure coordinates
$\tilde{W}_i^1, \tilde{W}_i^2, \tilde{W}_i^3$	inches and radians	Displacements of joint i measured in structure coordinates (distance along x and y, and rotation about z-axis, respectively)
x, y, z	inches	Cartesian coordinate axes for frame structure coordinates
x, y	inches	Distance along structure coordinate axes to change in loading or stiffness (referenced from members "From" joint)
x', y', z'	inches	Cartesian coordinate axes for member
x'	inches	Distance along member coordinate axis to change in loading or stiffness
x'', y'', x''	inches	Cartesian coordinate axes for element
x_i, y_i, x_j, y_j	inches	Structural coordinates of joints i, j
x_o, y_o	inches	x, y offset of joint j from joint i, projection of line going from joint i to joint j on the structure x-axis, y-axis

CHAPTER 1. INTRODUCTION

Statement of Problem

Many highway structures such as bridge bents and freeway overpasses are designed as plane frames. These structures may be composed of nonprismatic elastically restrained members and may be subjected to several complicated loading conditions. A thorough elastic analysis is economically feasible only with the aid of a digital computer program that is both versatile and convenient.

Most existing frame analysis programs are difficult or impossible to use for such real problems. Three frame solutions (Refs 2, 5, and 7) which incorporate the versatility of the discrete-element beam-column model (Refs 8 and 9) have been reported previously. References 5 and 7 use an alternating-direction iterative solution for the simultaneous equations which occur in the analysis. Recent developments in the direct solution of simultaneous equations (Ref 3) have made ADI solutions relatively less attractive on the present generation of computers. In addition, the solutions found in Refs 5 and 7 are restricted to rectangular frames.

Reference 2 gives a direct solution for linearly elastic frames and also permits an iterative investigation of the interaction of axial loads and lateral displacements. This solution while more versatile than previous ones still does not provide the designer with the convenience needed for a routine linear analysis, i.e., it allows members of the frame to be at any angle relative to the frame coordinate system, but it requires an orthogonal grid with intersections at all joints.

Purpose of Study

The primary purpose of this study is to develop a computer solution for plane frames that has the maximum convenience for the user consistent with generality of member arrangement and loading. The program is intended for routine use in a design office and has the following distinguishable features:

- (1) Input of both regular and irregular frame geometries is simple and quick.
- (2) Members may be either rigidly connected or pinned to joints.
- (3) Loads and elastic restraints may act both normal and parallel to members and may be input in the most convenient coordinate system.
- (4) Member stiffness properties and loadings may be input in normal engineering terms rather than as concentrated values at discrete stations.
- (5) Solutions of dead, live, wind, and other loads may be multiplied by appropriate load factors and superimposed to satisfy code loading requirements.

A secondary purpose of this report is to develop a solution of the frame members by the direct stiffness method (Ref 6) using a modification of the discrete-element model previously reported.

Assumptions and Limitations of the Proposed Solution

The solution developed is for the linearly elastic analysis of plane frames subjected to static inplane loads and displacements and has the following restrictions of conventional plane frame analysis:

- (1) Members are represented as straight-line elements intersecting at joints of infinitesimal size and are either rigidly connected or pinned to the joints.
- (2) Members are made of a linearly elastic material.
- (3) Displacements and deformations are small enough that the equilibrium equations can be formulated on the undeformed structure. Thus, the interaction of axial loads and lateral displacements is neglected.
- (4) Shearing deformations are neglected.

Outline of Report

The conventional theory of the direct stiffness method is presented in Chapter 2, and the method is applied to develop the joint equilibrium equations for the frame solution. Chapter 3 develops the equilibrium equations for the frame members by the direct stiffness method and discusses how they are used to obtain the member properties required for the frame solution and the member results. In Chapter 4 the equations needed to internally transform the engineering data into discretized station values are given.

The organization of and the input for the computer program are discussed in Chapter 5. Several example problems are presented in Chapter 6 to

illustrate the features of the program. The results of this study are given in Chapter 7.

Appendix 1 gives a theoretical justification of the discrete-element model developed in the report. The remaining appendices have the input guides, flow charts, FORTRAN notation, FORTRAN listing of the program, and input and selected output for the example problems.

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CHAPTER 2. LINEAR MATRIX ANALYSIS OF STATICALLY LOADED PLANE FRAMES

General Theory

Most methods of structural analysis can be classified as either displacement or force methods. The classification is based on whether the basic unknowns are displacements* or forces**. The force or flexibility method has advantages for certain structures but is not as easy to formulate in general terms as the displacement method. Hence, the displacement or the stiffness method is the only one considered in this report.

For the purpose of a stiffness analysis, a structure may be visualized as a group of elements connected at a finite number of nodal points. Each nodal point can undergo one or more pertinent displacements. For each of these displacements, there is a corresponding force. A force and a displacement are said to correspond if they act at the same nodal point, have the same line of action, and their product has the units of work. The total number of nodal point displacements for a structure is said to be the degree of kinematic indeterminacy or the number of degrees of freedom of the structure.

The elastic analysis of a statically loaded structure is basically a problem in satisfying simultaneously four sets of conditions. The governing conditions are the equations of nodal point equilibrium, compatibility of nodal point displacement, any boundary conditions applied at the nodal points, and the elements force-displacement relations. It is assumed that the force-displacement relations used for the elements insure that equilibrium, compatibility, any boundary conditions applied to the elements, and the constitutive laws for the element are satisfied continuously throughout the element.

* Throughout this report, the word "displacement" should be considered to mean either a translation or a rotation.

** Throughout this report, the word "force" implies either a translational force or a moment.

In many cases some approximation is actually made in developing the force-displacement relations as discussed in Chapter 3. The direct stiffness method, as described by Martin (Ref 6) is the most widely used technique for applying these four governing conditions to obtain the structures equilibrium equations.

A direct stiffness analysis of a structure can be separated somewhat arbitrarily into nine steps as outlined below. The general method is independent of the type of structure to which it is applied. Note that in the outline, the word matrix is not mentioned. This is done to emphasize that the basic ideas are not dependent on matrix algebra. However, matrix algebra is a powerful tool that enables the method to be developed concisely and implemented on a digital computer with ease.

Outline of the Direct Stiffness Method

- (1) Select nodal point displacements that insure nodal point compatibility is satisfied.
- (2) Calculate the force-displacement equations for all elements in their element coordinate systems.
- (3) Transform these equations into the structure coordinate system.
- (4) Sum up the nodal point forces corresponding to each nodal point displacement from the equations developed in Step 3. This gives the nodal point equilibrium equations in terms of element properties, nodal point forces, and nodal point displacements.
- (5) Modify the equations as necessary for support (displacement type boundary) conditions at the nodal points.
- (6) Solve the equations developed in Step 5 for the nodal point displacements. For the usual assumption of linearly elastic elements and small displacements, the equations are a set of linear simultaneous equations.
- (7) Transform the nodal point displacements into element displacements.
- (8) Solve for element forces from the force-displacement equations of Step 2.
- (9) Solve for nodal point reactions and check nodal point equilibrium.

The direct stiffness method, as outlined above, will be applied to a plane frame to obtain a solution for the joint (nodal point) displacements in the remainder of this chapter. The solutions of the individual frame members (elements) needed for the frame solution is accomplished by another application of the direct stiffness method to a general frame member in Chapter 3.

Plane Frame Definition

Consider a plane frame as shown in Fig 1. It is assumed that the frame is composed of members that may be treated as straight-line elements. All of the elements lie in a plane and all loads and displacements occur in that plane, which for convenience is taken to be the x-y plane of a right-hand Cartesian coordinate system.

The end of a member or the intersection of two or more members forms a joint and this joint is assumed to be rigid and to have negligible dimensions. A member may be either rigidly connected or pinned to the joint. When a member is rigidly connected to a joint, it and all other members also rigidly connected to the joint rotate through the same angle and transmit moment to one another. When a member is pinned to a joint, it is free to rotate independently of the joint and other members intersecting at that joint. Thus, no moment is transferred from a pinned-end member to any member at the joint.

Joint (Nodal Point) Displacements

Each joint (i) will in general have three degrees of freedom, \tilde{w}_i^1 , \tilde{w}_i^2 , and \tilde{w}_i^3 as shown in Fig 1. Translational displacements \tilde{w}_i^1 and \tilde{w}_i^2 must be equal (compatible) for all members intersecting at a joint. The rotational displacement may not be the same for all members at a joint, since some or all of the members may be pinned to the joint. Hence \tilde{w}_i^3 is defined as being the rotation of the joint, and the pin is assumed to be a part of the member* occurring at an infinitesimal distance inside the member. Thus, \tilde{w}_i^3 is equal (compatible) for all members intersecting at a joint. \tilde{w}_i^1 , \tilde{w}_i^2 , and \tilde{w}_i^3 compose a vector $\{\tilde{w}_i\}$ where $\{\tilde{w}_i\}$ is a (3×1) matrix of structure displacements measured in structure coordinates.

A frame with N joints has a structure displacement vector $\{\tilde{w}\}$ where $\{\tilde{w}\}$ is a $(3N \times 1)$ matrix of structure displacements measured in structure coordinates. $\{\tilde{w}\}$ then is composed of N submatrices $\{\tilde{w}_i\}$. The basic equation of nodal point compatibility is

$$\{\tilde{w}_i\} = \{\bar{w}_i\}_k \quad (2.1)$$

* The special force-displacement equations needed for members with pinned ends are discussed in the next section.

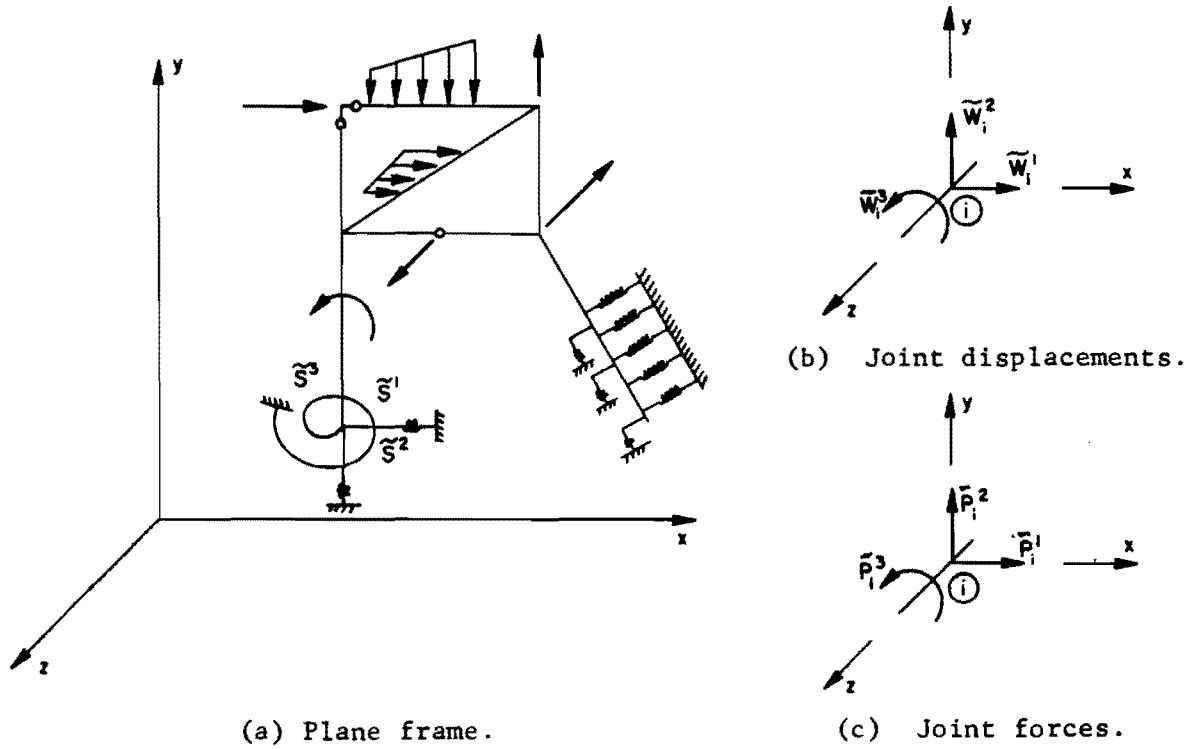


Fig 1. Plane frame.

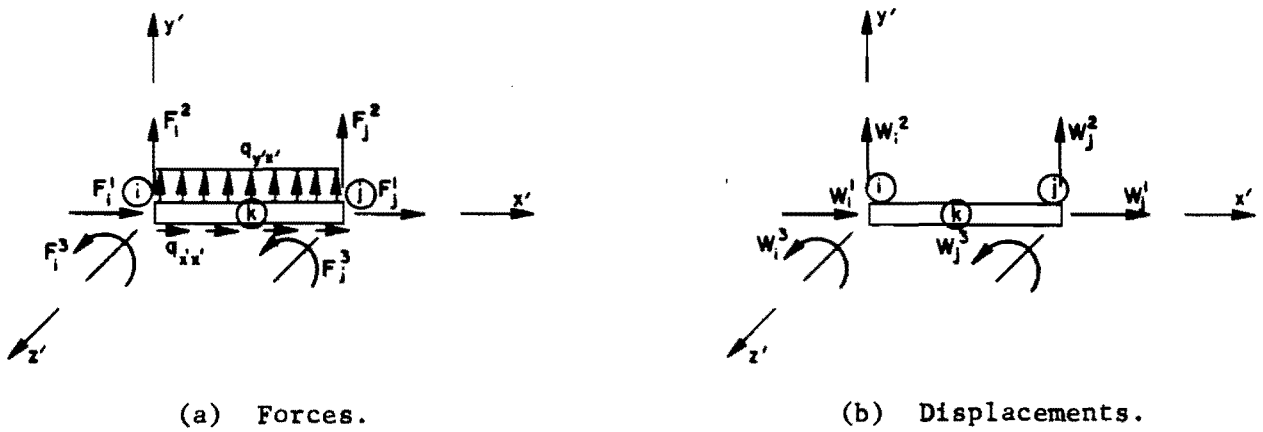


Fig 2. Prismatic, uniformly loaded plane frame member.

where

$$\{\bar{w}_i\}_k = (3 \times 1) \text{ matrix of member-end-displacements measured in structure coordinates for member } k \text{ which has one of its joints at joint } i .$$

Member (Element) Force-Displacement Equations

Consider a prismatic member as shown in Fig 2. Member (k) has its own local right-hand Cartesian coordinate system $x'-y'-z'$. The members x' -axis is directed along the members centroidal axis. To satisfy the assumption of planar behavior, the members y' -axis and z' -axis must be the members principle axes and the y' -axis must lie in the structure $x-y$ plane. All member loads and restraints must also act in the $x-y$ plane.

The prismatic member is assumed to have only uniform loads $q_{y',x'}$ and $q_{x',x'}$ acting on its full length as shown in Fig 2. The reason for the double subscripting will be discussed in Chapter 4. Neglecting the effect of shearing deformations and finite displacements, the following force-displacement equation (Ref 4) is derivable.

$$\begin{bmatrix} F_i^1 \\ F_i^2 \\ F_i^3 \\ \hline F_j^1 \\ F_j^2 \\ F_j^3 \end{bmatrix} = \begin{bmatrix} \frac{AE}{L} & 0 & 0 & | & -\frac{AE}{L} & 0 & 0 \\ 0 & \frac{12EI}{L^3} & \frac{6EI}{L^2} & | & 0 & -\frac{12EI}{L^3} & \frac{6EI}{L^2} \\ 0 & \frac{6EI}{L^2} & \frac{4EI}{L} & | & 0 & -\frac{6EI}{L^2} & \frac{2EI}{L} \\ \hline -\frac{AE}{L} & 0 & 0 & | & \frac{AE}{L} & 0 & 0 \\ 0 & -\frac{12EI}{L^3} & -\frac{6EI}{L^2} & | & 0 & \frac{12EI}{L^3} & -\frac{6EI}{L^2} \\ 0 & \frac{6EI}{L^2} & \frac{2EI}{L} & | & 0 & -\frac{6EI}{L^2} & \frac{4EI}{L} \end{bmatrix} \cdot \begin{bmatrix} w_i^1 \\ w_i^2 \\ w_i^3 \\ \hline w_j^1 \\ w_j^2 \\ w_j^3 \end{bmatrix} + \begin{bmatrix} -q_{x',x'} \frac{L}{2} \\ -q_{y',x'} \frac{L}{2} \\ -q_{y',x'} \frac{L^2}{12} \\ \hline -q_{x',x'} \frac{L}{2} \\ -q_{y',x'} \frac{L}{2} \\ +q_{y',x'} \frac{L^2}{12} \end{bmatrix} \quad (2.2)$$

where

A = cross-sectional area of the member,

E = modulus of elasticity,

L = length of member, and

I = moment of inertia of cross-section about z' -axis.

The forces F , loads q , and displacements W are defined in Fig 2.

In matrix notation, Eq 2.2 may be expressed as

$$\{F\}_k = [K]_k \{W\}_k + \{FF\}_k \quad (2.3)$$

where

$\{F\}_k$ = (6 × 1) matrix of member-end-forces due to loads and displacements,

$[K]_k$ = (6 × 6) member stiffness matrix,

$\{W\}_k$ = (6 × 1) matrix of member-end-displacements, and

$\{FF\}_k$ = (6 × 1) matrix of member-end-forces due to loads only (fixed-end-force-matrix).

All of the above are for member k derived in the members coordinate system.

For nonprismatic members or nonuniform loads, the member-force-displacement equations can still be expressed in matrix form by Eq 2.3, but $[K]_k$ and $\{FF\}_k$ will not be the same as in Eq 2.2.

In general, $\{FF\}_k$ is the matrix of fixed-end-forces for member k and can be found by working a fixed-end-member problem.

A typical element of $[K]_k$ is K_{pq} . The p represents the p^{th} row and q represents the q^{th} column of $[K]_k$. The range of p and q is from 1 to 6. For a linearly elastic member, K_{pq} represents the force corresponding to the p^{th} displacement due to a unit value of the q^{th} displacement. Thus, the q^{th} column of $[K]_k$ is the collection of member-end-forces due to a unit value of the q^{th} displacement. This is illustrated for ($q = 2$ and $W_1^2 = 1$) in Fig 3.

Chapter 3 presents the discrete-element technique for analyzing non-prismatic or nonuniformly loaded members and obtaining $[K]_k$ and $\{FF\}_k$. In the rest of this chapter, it is assumed that they have been found correctly and no distinction is made between prismatic and nonprismatic members.

Members with pinned ends are treated as follows in order to maintain compatibility of rotational displacements at a joint. The pin is assumed to be located just inside the member at a negligible distance from the joint. Special stiffness matrices $[K]_k$ and fixed-end-force matrices $\{FF\}_k$ will be used for the k^{th} member. Reference 4 gives these matrices for prismatic members and a solution technique to develop them for other members is given in Chapter 3.

Since forces will be superimposed separately at each joint, it is convenient to partition $[K]_k$, $\{F\}_k$, and $\{FF\}_k$ as suggested by the dashed lines in Eq 2.2. This then gives

$$\begin{bmatrix} F_i \\ \text{---} \\ F_j \end{bmatrix}_k = \begin{bmatrix} K_{ii} & K_{ij} \\ \text{---} & \text{---} \\ K_{ji} & K_{jj} \end{bmatrix}_k \cdot \begin{bmatrix} W_i \\ \text{---} \\ W_j \end{bmatrix}_k + \begin{bmatrix} FF_i \\ \text{---} \\ FF_j \end{bmatrix}_k \quad (2.4)$$

where

$$\{F_i\}_k = (3 \times 1) \text{ matrix of member-end-forces at joint } i,$$

$$[K_{ii}]_k = (3 \times 3) \text{ member stiffness matrix which represents the forces at } i \text{ due to unit displacements at } i,$$

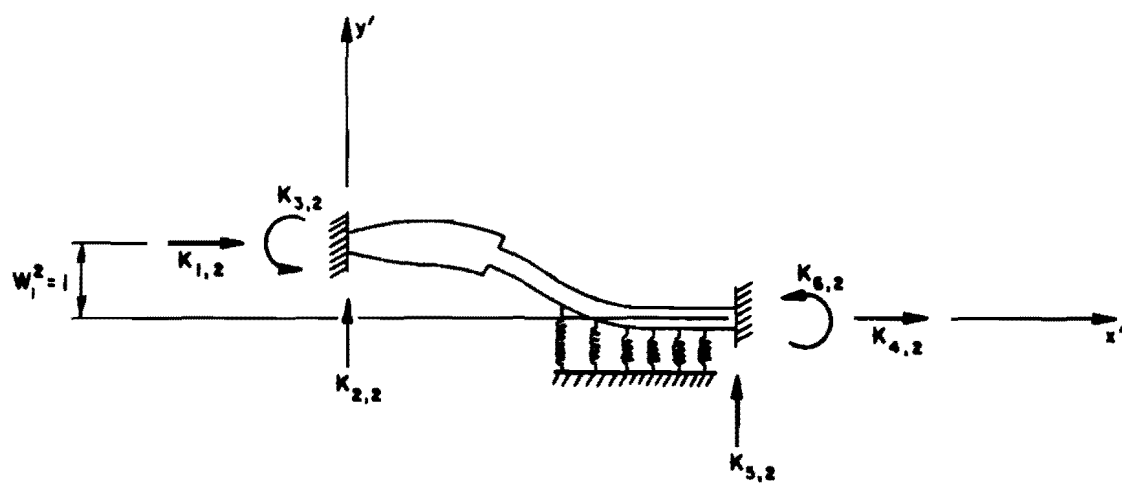


Fig 3. Nonprismatic elastic plane frame member
 subjected to unit displacement W_1^2 .

$[K_{ij}]_k$ = (3 × 3) member stiffness matrix in member coordinates which represents the forces at i due to unit displacements at j ,

$\{W_i\}_k$ = (3 × 1) matrix of member displacements at joint i, and

$\{FF_i\}_k$ = (3 × 1) matrix of member fixed-end-forces at joint i.

All of the above are for member k derived in the members coordinates.

The matrices on the second row of Eq 2.4 are defined as above except i and j are interchanged.

Equation 2.4 represents two matrix equations, the first of which is

$$\{F_i\}_k = [K_{ii}]_k \{W_i\}_k + [K_{ij}]_k \{W_j\}_k + \{FF_i\}_k \quad (2.5)$$

Equation 2.5 gives the member-end-forces acting at joint i in member coordinates. The forces and displacement in Eq 2.5 must be expressed in structure coordinates before $\{F_i\}_k$ can be added to other forces which are acting at the joint.

Transformation of Member (Element) Force-Displacement Equations into Structure Coordinates

The transformation at joint i, for member k, of member displacements in structure coordinates $\{\bar{W}_i\}_k$ to member displacement in member coordinates $\{W_i\}_k$ is given by

$$\{W_i\}_k = [T]_k \{\bar{W}_i\}_k \quad (2.6)$$

The transformation matrix for member k, $[T]_k$, is given by Eq 2.7 (Ref 4).

$$[T]_k = \begin{bmatrix} \alpha & \beta & 0 \\ -\beta & \alpha & 0 \\ 0 & 0 & 1 \end{bmatrix} \quad (2.7)$$

where

α = cosine of angle between the x' -axis and x -axis, and

β = cosine of angle between the x' -axis and y -axis.

Similarly, the transformation of member forces in structure coordinates $\{\bar{F}_i\}_k$ to member forces in member coordinates $\{F_i\}_k$ is given by Eq 2.8

$$\{F_i\}_k = [T]_k \{\bar{F}_i\}_k \quad (2.8)$$

where for member k

$\{F_i\}_k$ = (3 x 1) matrix of member-end-forces at joint i in member coordinates,

$\{\bar{F}_i\}_k$ = (3 x 1) matrix of member-end-forces at joint i in structure coordinates, and

$[T]_k$ = the previously defined coordinate transformation matrix.

Since $[T]_k$ is an orthogonal matrix, it possesses the special properties that its transpose $[T]_k^t$ is also its inverse $[T]_k^{-1}$ and that the product of $[T]_k$ and $[T]_k^t$ in any order is equal to the identity matrix $[I]$.

Therefore

$$\{\bar{W}_i\}_k = [T]_k^t \{W_i\}_k \quad (2.9)$$

and

$$\{\bar{F}_i\}_k = [T]_k^t \{F_i\}_k \quad (2.10)$$

Substituting Eq 2.6 into Eq 2.5 for $\{W_i\}_k$ and $\{W_j\}_k$, premultiplying both sides of Eq 2.5 by $[T]_k^t$ and using Eq 2.10 gives

$$\begin{aligned} \{\bar{F}_i\}_k &= [T]_k^t [K_{ii}]_k [T]_k \{\bar{W}_i\}_k + [T]_k^t [K_{ij}]_k [T]_k \{\bar{W}_j\}_k \\ &+ [T]_k^t \{FF\}_k \end{aligned} \quad (2.11)$$

Let

$$[\bar{K}_{ii}]_k = [T]_k^t [K_{ii}]_k [T]_k \quad (2.12)$$

$$[\bar{K}_{ij}]_k = [T]_k^t [K_{ij}]_k [T]_k \quad (2.13)$$

$$\{\bar{FF}_i\}_k = [T]_k^t \{FF_i\}_k \quad (2.14)$$

then for member k

$$[\bar{K}_{ii}]_k = (3 \times 3) \text{ member stiffness matrix in structure coordinates which represent the forces at } i \text{ due to unit displacements at } i ,$$

$$[\bar{K}_{ij}]_k = (3 \times 3) \text{ member stiffness matrix in structure coordinates which represents the forces at } i \text{ due to unit displacements at } j , \text{ and}$$

$$\{\bar{FF}_i\}_k = (3 \times 1) \text{ matrix of member-fixed-end-forces in structure coordinates.}$$

Then Eq 2.11 can be rewritten as

$$\{\bar{F}_i\}_k = [\bar{K}_{ii}]_k \{\bar{W}_i\}_k + [\bar{K}_{ij}]_k \{\bar{W}_j\}_k + \{\bar{FF}_i\}_k \quad (2.15)$$

To satisfy the compatibility requirement, Eq 2.1 can be substituted into Eq 2.15 for $\{\bar{W}_i\}_k$ and $\{\bar{W}_j\}_k$, to give

$$\{\bar{F}_i\}_k = [\bar{K}_{ii}]_k \{\tilde{W}_i\}_k + [\bar{K}_{ij}]_k \{\tilde{W}_j\}_k + \{\bar{FF}_i\}_k \quad (2.16)$$

Summation of Joint (Nodal Point) Forces

Each joint must be in static equilibrium due to the forces imparted to it by each of the M members acting there, any applied joint forces, and any support reactions. The effect of the support reactions will be considered in the next section. The applied joint forces \tilde{P}_i^1 , \tilde{P}_i^2 , and \tilde{P}_i^3 at joint i represent a vector $\{\tilde{P}_i\}$ (see Fig 1)

where

$$\{\tilde{P}_i\} = (3 \times 1) \text{ matrix of applied joint forces at joint } i.$$

Thus, temporarily neglecting the support reactions at joint i , $\{\tilde{P}_i\}$ is equal to the sum of $\{\tilde{F}_i\}$ from Eq 2.16 for all M members.

$$\begin{aligned} \{\tilde{P}_i\} &= \left(\sum_{k=1}^M [\bar{K}_{ii}]_k \right) \{\tilde{W}_i\} + \sum_{k=1}^M \left([\bar{K}_{ij}]_k \{\tilde{W}_j\} \right) \\ &+ \sum_{k=1}^M \{\bar{FF}_i\}_k \end{aligned} \quad (2.17)$$

Equation 2.17 can be rearranged to have all loads on the right-hand side as follows:

$$\left(\sum_{k=1}^M [\bar{K}_{ii}]_k \right) \{\tilde{W}_i\} + \sum_{k=1}^M \left([\bar{K}_{ij}]_k \cdot \{\tilde{W}_j\} \right) = \{\tilde{P}_i\} - \sum_{k=1}^M \{\overline{FF}_i\}_k \quad (2.18)$$

Equation 2.18 can be applied now to all N joints ($i = 1, N$) to form the structure equilibrium equations which can be expressed as

$$[\tilde{K}] \cdot \{\tilde{W}\} = \{\tilde{F}\} \quad (2.19)$$

where

$$\begin{aligned} [\tilde{K}] &= (3N \times 3N) \text{ structure stiffness matrix,} \\ \{\tilde{W}\} &= (3N \times 1) \text{ structure displacement matrix,} \\ \{\tilde{F}\} &= (3N \times 1) \text{ structure load matrix.} \end{aligned}$$

Clearly $[\tilde{K}]$ can be partitioned into $N^2 (3 \times 3)$ submatrices $[\tilde{K}_{ij}]$ and from Eq 2.18.

$$[\tilde{K}_{ii}] = \sum_{k=1}^M [\bar{K}_{ii}]_k \quad (2.20)$$

and

$$[\tilde{K}_{ij}] = [\bar{K}_{ij}]_k \quad i \neq j \quad (2.21)$$

Thus the structure stiffness matrix is easily composed from the individual member stiffness matrices expressed in structure coordinates. Similarly the (3×1) load matrix at the i^{th} joint $\{\tilde{F}_i\}$ is given by Eq 2.18 as,

$$\{\tilde{F}_i\} = \{\tilde{P}_i\} - \sum_{k=1}^M \{\overline{FF}_i\}_k \quad (2.22)$$

Joint Supports (Displacement Type Boundary Conditions)

Suppose that any joint i has three linearly elastic support springs, with spring constants \tilde{S}_i^1 , \tilde{S}_i^2 , and \tilde{S}_i^3 , as shown in Fig 1. Any of these can be zero as a lower limit and approach infinity as an upper limit. If a joint undergoes displacement during the application of loads, then support reactions will be generated equal to the negative of the displacements times the corresponding spring constants. Clearly these reactions must be considered in writing the joint equilibrium equations. If these terms are added to equations, the effect on $[\tilde{K}]$ is to add the corresponding spring term to the diagonal of the matrix.

The effect of the other matrix terms becomes negligible as the spring term becomes very large compared to the other terms in any row of $[\tilde{K}]$. Similarly the load term for that row becomes negligible.

Thus, a zero displacement can be obtained by specifying a very large spring restraint. Likewise, a specified displacement may be obtained by specifying a large spring restraint and a correspondingly large joint force equal to the desired displacement times the spring restraint.

Handling specified displacements in this way allows both real problems where supports have some reasonable value of restraint and other problems with infinitely stiff supports to be solved by the same technique.

When all members at a joint are hinged to the joint, the rotational stiffness at the joint is zero. This causes a singular set of equations for which the solution process as discussed in the next section will either cause an arithmetic error on the computer due to the attempt to divide by zero or give extremely large displacements.

One method of solving structures with such pinned joints is to renumber the joint equilibrium equations allowing only two degrees of freedom at the pinned joints. This loses much of the generality built into the equations of this chapter. Instead, the present computer program places a unit value on the diagonal of $[\tilde{K}]$ and a very large value in the load matrix. This gives, then, a very large displacement for the rotation of such a joint which indicates that it is undefined.

Similarly, when a joint is deleted in a series of problems by removing all the members intersecting at the joint, all three of the displacements of such a joint are undefined and the program handles such a joint the same way.

Unit values are placed on the diagonal for all three of the zero stiffnesses and three large values are placed in the load matrix for the three undefined displacements.

Since for both the pinned joint and the joint with all members deleted setting the displacements equal to a large value has no physical effect on the structure, the rest of the solution is valid. However, the large displacements should occur only for the rotation of joints with all members pinned or a joint that is deleted. Any other large displacements are an indication that an unstable or nearly unstable structure has been described. The user will be aware that he has such a joint so no misunderstanding of the results should occur.

Solution of Joint (Nodal Point) Equilibrium Equations

The equations developed in the preceding section are a system of linear-simultaneous equations which can be solved efficiently using a recursion-inversion process previously developed (Ref 3).

The solution of Ref 3 considers the banding of the stiffness matrix inherent in structural problems. It also takes advantage of the symmetry of the structure stiffness matrix.

The second and succeeding solutions of a structure (for additional load cases) are obtained in far less time than the initial solution. This is possible since the stiffness matrix for a linearly-elastic structure is independent of the loading, therefore the elimination process need not be repeated after the first solution.

Member (Element) Displacements from Joint (Nodal Point) Displacements

Once joint displacements are found, the member displacements can be obtained from Eqs 2.6 and 2.1. Note that for a member with a pinned end this will give the displacement on the joint side of the pin (which is actually the joint displacement). However, as outlined in the next section, this will not affect the solution of member-end-forces.

Member (Element) Forces

For prismatic members with only uniform loads the member-end-forces can be found by using Eq 2.2 or for such members with pinned ends the special equations in Ref 4 can be used. For nonprismatic members or nonuniformly

loaded members, the member-end-forces can be found by the solution developed in Chapter 3. This solution can also be used for cases in which more complete output of forces and displacements throughout the member is desired.

Joint (Nodal Point) Reactions and Check of Joint Equilibrium Equations

Once the member-end-forces have been calculated, they can be converted to structure coordinates by Eq 2.10. With all supports specified as linearly-elastic springs the joint reactions can be found merely by multiplying the spring constants times the negative of the corresponding displacements. If the proper solution of the equilibrium equations has been found, the sum of the member-end-forces applied to the joint should equal the applied joint forces plus the joint reactions. Any difference between the joint forces and the member forces is an indication of the roundoff error developed in the solution of the equations. Generally this error (joint equilibrium error) will be a very small quantity.

There is one case in which the joint equilibrium error is not a valid indication of the accuracy of the solution. When a specified displacement is enforced at a joint by an artificially large restraint and a correspondingly large force the joint equilibrium error will be as many orders of magnitude less than the artificial load as the computer is inaccurate in subtracting two numbers. This is not an indication that the solution is in error but only that the estimate of the error is invalid. This occurs only when artificially large values are used to specify displacements and has never occurred on the CDC 6600 with approximately 15 significant digits for any physical values of restraint and load.

Superposition Solution

Under sufficiently small loads structures behave in a linear manner. Thus the results of several linear solutions may be stored and combined by simple superposition to form any linear combination of loadings desired. Naturally the designer must check the results to insure that the stresses in the structure are small enough for the superposition solution to be valid.

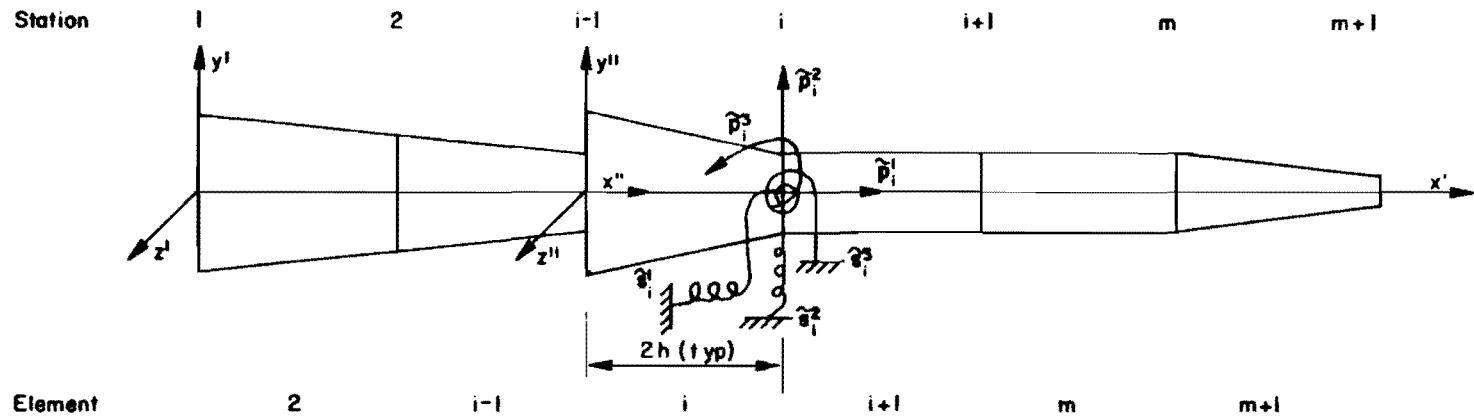
CHAPTER 3. NEW DISCRETE-ELEMENT TECHNIQUE FOR SOLUTIONS OF GENERAL FRAME MEMBERS

As previously discussed, solutions for the members stiffness matrix $[K]_k$ and the members fixed-end-force matrix $\{FF\}_k$ are required for nonprismatic, elastically restrained, and nonuniformly loaded members. The discrete-element formulation from Ref 9 was used in Ref 2 to obtain $[K]_k$ and $\{FF\}_k$. This discrete-element model does not provide for loads or restraints acting parallel to the member. This causes problems in coding frames with members that are not horizontal but have significant vertical (gravity) loads. Frames with friction piles are also difficult to input.

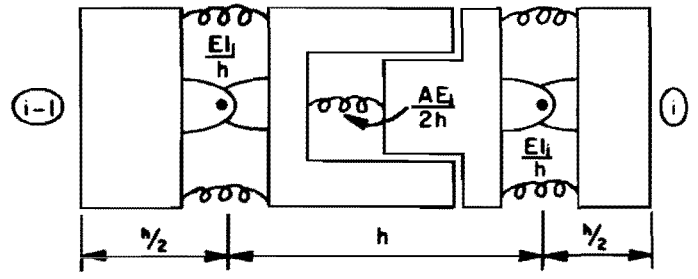
The basic element of the discrete-element beam column model (Ref 9) is a rigid bar with one degree of freedom at each end (lateral displacement). If the bar were made axially deformable it would have four degrees of freedom (a lateral and an axial displacement at each end). The stiffness or equilibrium equations for a member composed of a number of such elements connected by rotational springs could be formulated in the same manner as was done for the discrete-element in Ref 9. However, the member equilibrium equations are developed formally by the direct stiffness method (Ref 6) using a modification of the discrete-element technique proposed herein.

Frame Member with Discretized Effects

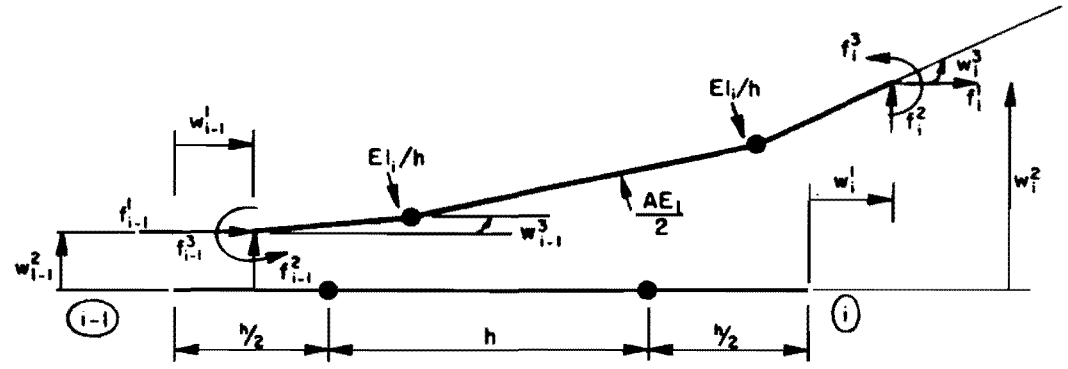
A general frame member is shown in Fig 4(a). The member is general in that at any station i , forces \tilde{p}_i^1 , \tilde{p}_i^2 , and \tilde{p}_i^3 may act, elastic restraints \tilde{s}_i^1 , \tilde{s}_i^2 , and \tilde{s}_i^3 may exist, and changes in cross-sectional properties may occur. Each continuous element i between stations $i-1$ and i is assumed to have a linear variation in cross-sectional properties. The term AE_i is the axial rigidity of the element at mid-element and EI_i is the flexural rigidity of the element at the midpoint. The terms A , E , and I are as previously defined. The member is further generalized in Chapter 4 to make it more convenient for the user by allowing loads, restraints, and changes in cross-section



(a) Frame member with discretized loads, restraints, and stiffness changes.



(b) Mechanical model of element i.



(c) Discrete-line-element model of element i.

Fig 4. Frame member and discrete-element representation of one of its continuous elements.

to occur at any point in the member. Distributed values of load and restraint are permitted and loads may be input in either frame or member coordinates. Chapter 4 presents the method of handling such effects, therefore, in this chapter the discrete station values are assumed to be available.

Application of the Direct Stiffness Method to Frame Members

The frame member may now be considered to be a structure composed of m elements, which for convenience are numbered from 2 to $m+1$ in Fig 4a. Each of the $m+1$ stations (nodal points) of the frame member has three displacements (axial displacement \tilde{w}_i^1 , lateral displacement \tilde{w}_i^2 , and rotational displacement \tilde{w}_i^3). These displacements should be equal for both elements intersecting at a station. Thus, there is no difference between the frame analysis and the member analysis other than some simplifications because of the simplified geometry of the member. (The frame member's axes $x'-y'-z'$ and the element's axes $x''-y''-z''$ are parallel as shown in Fig 4a, hence no transformation of coordinates is required.) This means no distinction need be made between properties measured in element coordinates and properties measured in the frame member's coordinates. Therefore, the element stiffness matrix can be derived with reference to its own coordinates and used in Eq 2.18.

Discrete-Element Model

The stiffness matrix for a continuous element with linearly varying stiffness properties could be derived, but to work with it would be difficult. Instead, the stiffness matrix of the element will be derived for the discrete-element shown in Figs 4b and 4c, and in Appendix 1 it is shown that the discrete-element model is an adequate representation of the continuous element. That is to say that, as the number of elements used increases, the answers obtained by the discrete-element approaches as a limit the "exact" solution.

A mechanical model of the discrete-element whose length is $2h$ is shown in Fig 4b. It is composed of two rigid end blocks, two rotational springs with spring constants EI_1/h and a rigid piston with an axial spring whose spring constant is $AE_1/2h$. The term EI_1 is the product of the modulus of elasticity and the moment of inertia at the center of the continuous element. The term AE_1 is the product of the cross-sectional area and the modulus of elasticity at the center of the continuous element.

The mechanical model of Fig 4b may be shown more conveniently as a discrete-line element model in Fig 4c. There the element is composed of three one-dimensional bars which are rigid in bending and connected by rotational springs with spring constants EI_1/h . The end bars are axially rigid but the center bar of length h is axially deformable and has an axial rigidity of $AE_1/2$. The two models are mathematically equivalent and hereafter the discrete-line element model will be shown for convenience and will be referred to as the discrete-element model.

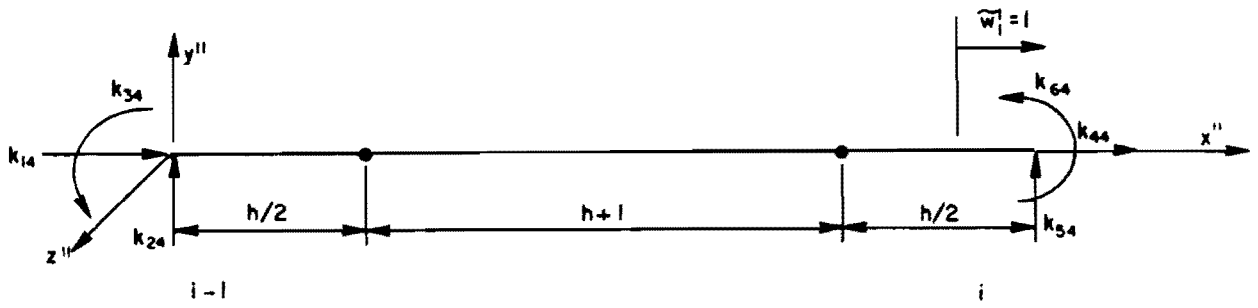
Element Stiffness Matrix

The discrete-element i has three degrees of freedom at each station and will have a (6×6) element stiffness matrix $[k]_i$, which relates forces and displacement at stations i and $i-1$. As mentioned in Chapter 2, the q^{th} column of a stiffness matrix is the set of reactions corresponding to the displacements due to a unit value of the q^{th} displacement. This is illustrated in Fig 5 for element i , $q = 4, 5, \text{ and } 6$.

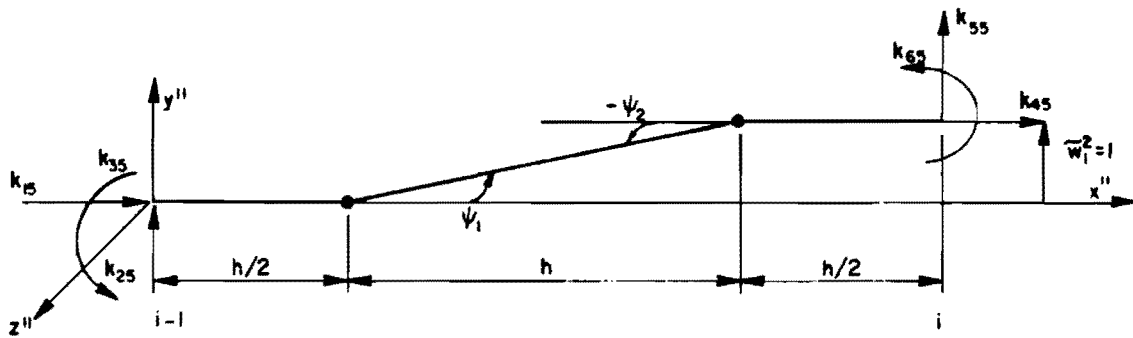
For each of the six unit displacements, the discrete angle changes and changes in length of the deformable bar may be found geometrically. The internal spring moments and the axial force can then be computed. Next a free-body analysis can be made and the forces acting on the ends of the element can be found. These forces are the desired stiffness terms. Assuming negligible displacements, the freebody analysis can be made on the undeformed element, thus neglecting the effect of the secondary moment caused by axial forces acting on lateral displacements. This secondary moment could be included in the analysis but would then require a nonlinear solution since, in general, the effect of axial forces is not linear. Therefore, in the development of this design oriented linear analysis program it is not considered. However, in the future it probably will be incorporated in nonlinear studies.

In Fig 5a ($q = 4$ and $\tilde{w}_1^1 = 1$) the axially deformable bar extends one inch, thus the force T_1 in the bar is given by Eq 3.1

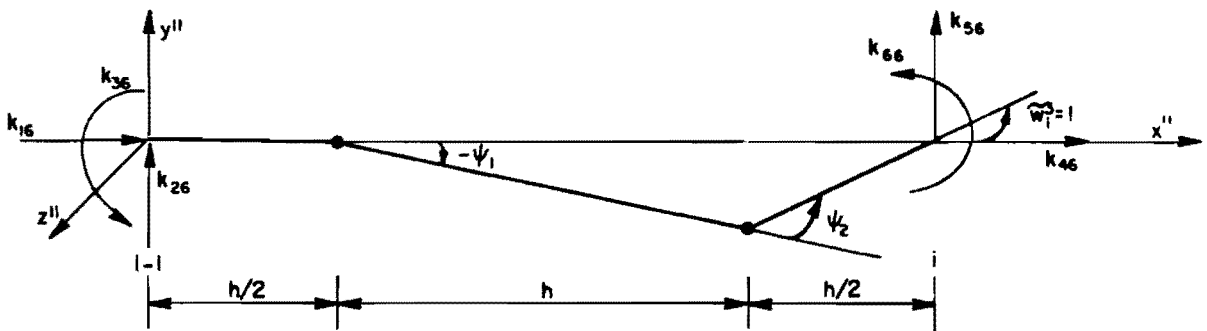
$$T_1 = \frac{AE_1(1)}{2h} = \frac{AE_1}{2h} \quad (3.1)$$



(a) Fourth column of element stiffness matrix.



(b) Fifth column of element stiffness matrix.



(c) Sixth column of element stiffness matrix.

Fig 5. Unit displacements at station i on element i .

For equilibrium of forces parallel to x''

$$k_{44} = \frac{AE_i}{2h} \quad (3.2)$$

and

$$k_{14} = \frac{-AE_i}{2h} \quad (3.3)$$

All other forces are zero because of the assumed small displacements.

In Fig 5b ($q = 5$ and $\tilde{w}_i^2 = 1$) concentrated curvatures* Ψ_1 and Ψ_2 are developed at the spring locations. For the unit displacement shown in Fig 5b

$$\Psi_1 = \frac{1}{h} \quad (3.4)$$

$$\Psi_2 = \frac{-1}{h} \quad (3.5)$$

The internal moments corresponding to Ψ_1 and Ψ_2 are M_1 and M_2 where

$$M_1 = \frac{EI_i \Psi_1}{h} = \frac{EI_i}{h^2} \quad (3.6)$$

$$M_2 = \frac{EI_i \Psi_2}{h} = \frac{-EI_i}{h^2} \quad (3.7)$$

* The sign of the curvature and corresponding moment is positive if it produces compression on the positive y'' side of the member.

Shears* V_1 and V_2 will be developed at the two spring locations and can be found from a freebody analysis of the axially deformable center bar.

$$V_1 = \frac{(M_2 - M_1)}{h} = -\frac{2EI_i}{h^3} \quad (3.8)$$

$$V_2 = -V_1 = \frac{2EI_i}{h^3} \quad (3.9)$$

Summing forces and moments on freebodies of the two end bars yields

$$\begin{bmatrix} k_{15} \\ k_{25} \\ k_{35} \\ k_{45} \\ k_{55} \\ k_{65} \end{bmatrix} = \begin{bmatrix} 0 \\ -\frac{2EI_i}{h^3} \\ -\frac{2EI_i}{h^2} \\ 0 \\ \frac{2EI_i}{h^3} \\ -\frac{2EI_i}{h^2} \end{bmatrix} \quad (3.10)$$

In Fig 5c ($q = 6$ and $\tilde{w}_1^3 = 1$) concentrated curvatures Ψ_1 and Ψ_2 are developed where

$$\Psi_1 = -\frac{1}{2} \quad (3.11)$$

* Positive shear acts in the direction of positive y' on the face of the freebody closest to the member axis.

$$\psi_2 = \frac{3}{2} \quad (3.12)$$

hence

$$M_1 = -\frac{EI_1}{2h} \quad (3.13)$$

$$M_2 = \frac{3EI_1}{2h} \quad (3.14)$$

and

$$V_1 = \frac{2EI_1}{h^2} \quad (3.15)$$

$$V_2 = -\frac{2EI_1}{h^2} \quad (3.16)$$

Summing forces and moments on the end bars yields

$$\begin{bmatrix} k_{16} \\ k_{26} \\ k_{36} \\ k_{46} \\ k_{56} \\ k_{66} \end{bmatrix} = \begin{bmatrix} 0 \\ \frac{2EI_1}{h^2} \\ \frac{1.5EI_1}{h} \\ 0 \\ -\frac{2EI_1}{h^2} \\ \frac{2.5EI_1}{h} \end{bmatrix} \quad (3.17)$$

Similarly, by applying unit displacements at $i-1$ the first three columns of the element stiffness matrix may be obtained. Thus,

$$\left[\mathbf{k} \right]_i = \begin{bmatrix} \frac{AE_i}{2h} & 0 & 0 & -\frac{AE_i}{2h} & 0 & 0 \\ 0 & \frac{2EI_i}{h^3} & \frac{2EI_i}{h^2} & 0 & -\frac{2EI_i}{h^3} & \frac{2EI_i}{h^2} \\ 0 & \frac{2EI_i}{h^2} & \frac{2.5EI_i}{h} & 0 & -\frac{2EI_i}{h^2} & \frac{1.5EI_i}{h} \\ -\frac{AE_i}{2h} & 0 & 0 & \frac{AE_i}{2h} & 0 & 0 \\ 0 & -\frac{2EI_i}{h^3} & -\frac{2EI_i}{h^2} & 0 & \frac{2EI_i}{h^3} & -\frac{2EI_i}{h^2} \\ 0 & \frac{2EI_i}{h^2} & \frac{1.5EI_i}{h} & 0 & -\frac{2EI_i}{h^2} & \frac{2.5EI_i}{h} \end{bmatrix} \quad (3.18)$$

The element end forces $\{f\}_i$ are related to the element end displacements $\{w\}_i$ as shown in Eq 3.19.*

$$\{f\}_i = [k]_i \{w\}_i \quad (3.19)$$

The matrix $[k]_i$ can be subdivided into four 3 by 3 submatrices as was done in Chapter 2 for $[K]_k$:

* Loads acting between the nodal points are transformed into equivalent nodal point loads, hence, no fixed-end-forces act as in Eq 2.3.

$$[k]_i = \begin{bmatrix} k_{i-1,i-1} & | & k_{i-1,i} \\ \hline k_{i,i-1} & | & k_{i,i} \end{bmatrix}_i \quad (3.20)$$

where the submatrices are given by Eq 3.18.

Similarly, the element stiffness matrix for element $i+1$, which connects with element i at station i , can be divided into four 3 by 3 submatrices:

$$[k]_{i+1} = \begin{bmatrix} k_{i,i} & | & k_{i,i+1} \\ \hline k_{i+1,i} & | & k_{i+1,i+1} \end{bmatrix}_{i+1} \quad (3.21)$$

The submatrices of Eq 3.21 are given by the terms in Eq 3.18 except AE_i is replaced by AE_{i+1} and EI_i is replaced by EI_{i+1} .

Substituting the appropriate submatrices from Eqs 3.20 and 3.21 into Eq 2.18 ($M = 2$) yields

$$\begin{aligned} & \left([k_{i,i}]_i + [k_{i,i}]_{i+1} \right) \{\tilde{w}_i\} + [k_{i,i-1}]_{i-1} \{\tilde{w}_{i-1}\} \\ & + [k_{i,i+1}]_{i+1} \{\tilde{w}_{i+1}\} = \{\tilde{P}_i\} \end{aligned} \quad (3.22)$$

Substituting into Eq 3.22 the appropriate submatrices as given by Eq 3.18 and adding in the effect of the elastic restraints $\{\tilde{s}_i\}$ gives

$$\left(\begin{bmatrix} \frac{AE_i}{2h} & 0 & 0 \\ 0 & \frac{2EI_i}{h^3} & -\frac{2EI_i}{h^2} \\ 0 & -\frac{2EI_i}{h^2} & \frac{2.5EI_i}{h} \end{bmatrix} + \begin{bmatrix} \frac{AE_{i+1}}{2h} & 0 & 0 \\ 0 & \frac{2EI_{i+1}}{h^3} & \frac{2EI_{i+1}}{h^2} \\ 0 & \frac{2EI_{i+1}}{h^2} & \frac{2.5EI_{i+1}}{h} \end{bmatrix} \right)$$

$$\begin{aligned}
& + \begin{bmatrix} \tilde{s}_i^1 & 0 & 0 \\ 0 & \tilde{s}_i^2 & 0 \\ 0 & 0 & \tilde{s}_i^3 \end{bmatrix} \cdot \begin{bmatrix} \tilde{w}_i^1 \\ \tilde{w}_i^2 \\ \tilde{w}_i^3 \end{bmatrix} + \begin{bmatrix} -\frac{AE_i}{2h} & 0 & 0 \\ 0 & -\frac{2EI_i}{h^3} & -\frac{2EI_i}{h^2} \\ 0 & \frac{2EI_i}{h^2} & \frac{1.5EI_i}{h} \end{bmatrix} \\
& \cdot \begin{bmatrix} \tilde{w}_{i-1}^1 \\ \tilde{w}_{i-1}^2 \\ \tilde{w}_{i-1}^3 \end{bmatrix} + \begin{bmatrix} -\frac{AE_{i+1}}{2h} & 0 & 0 \\ 0 & -\frac{2EI_{i+1}}{h^3} & \frac{2EI_{i+1}}{h^2} \\ 0 & -\frac{2EI_{i+1}}{h^2} & \frac{1.5EI_{i+1}}{h} \end{bmatrix} \\
& \cdot \begin{bmatrix} \tilde{w}_{i+1}^1 \\ \tilde{w}_{i+1}^2 \\ \tilde{w}_{i+1}^3 \end{bmatrix} = \begin{bmatrix} \tilde{p}_i^1 \\ \tilde{p}_i^2 \\ \tilde{p}_i^3 \end{bmatrix} \tag{3.23}
\end{aligned}$$

Summing up the matrices of Eq 3.23 and multiplying the first row by h and the second and the third rows by h^3 and rearranging yields

$$\begin{aligned}
& \begin{bmatrix} -\frac{AE_i}{2} & 0 & 0 \\ 0 & -2EI_i & -2hEI_i \\ 0 & 2hEI_i & 1.5h^2EI_i \end{bmatrix} \cdot \begin{bmatrix} \tilde{w}_{i-1}^1 \\ \tilde{w}_{i-1}^2 \\ \tilde{w}_{i-1}^3 \end{bmatrix} \\
+ & \begin{bmatrix} \frac{AE_i+AE_{i+1}+\tilde{s}^1h}{2} & 0 & 0 \\ 0 & 2(EI_i+EI_{i+1})+\tilde{s}_i^2h^3 & 2h(EI_{i+1}-EI_i) \\ 0 & 2h(EI_{i+1}-EI_i) & 2.5h^2(EI_i+EI_{i+1})+\tilde{s}_i^3h^3 \end{bmatrix} \cdot \begin{bmatrix} \tilde{w}_i^1 \\ \tilde{w}_i^2 \\ \tilde{w}_i^3 \end{bmatrix} \\
+ & \begin{bmatrix} -\frac{AE_{i+1}}{2} & 0 & 0 \\ 0 & -2EI_{i+1} & 2hEI_{i+1} \\ 0 & -2hEI_{i+1} & 1.5h^2EI_{i+1} \end{bmatrix} \cdot \begin{bmatrix} \tilde{w}_{i+1}^1 \\ \tilde{w}_{i+1}^2 \\ \tilde{w}_{i+1}^3 \end{bmatrix} = \begin{bmatrix} \tilde{p}_i^1h \\ \tilde{p}_i^2h^3 \\ \tilde{p}_i^3h^3 \end{bmatrix} \quad (3.24)
\end{aligned}$$

Equation 3.24 may be applied at the $m+1$ nodal points to form the member equilibrium equation.* Since Eq 3.24 represents three scalar equations this will yield $3(m+1)$ simultaneous equations. These equations will be symmetric and have a band width of nine. Noting that the axial (\tilde{w}^1) and lateral (\tilde{w}^2 and \tilde{w}^3) displacements are uncoupled in Eq 3.24 the solution may be separated into $m+1$ axial equations with a band width of three and $2(m+1)$ lateral equations with a band width of seven.

* In order to apply Eqs 3.24, 3.25, 3.26, and 3.27 at nodal points $i = 1$ and $i = m+1$, fictitious elements $i = 1$, and $i = m+2$ must be assigned zero stiffness.

The axial equations are given by the $m+1$ application of Eq 3.25

$$\begin{aligned}
 -\frac{AE_i}{2} \tilde{w}_{i-1} + \left(\frac{AE_i}{2} + \frac{AE_{i+1}}{2} + \tilde{s}_i^1 h \right) \tilde{w}_i - \frac{AE_{i+1}}{2} \tilde{w}_{i+1} \\
 = \tilde{p}_i^1 h
 \end{aligned} \tag{3.25}$$

The lateral equations are generated by $m+1$ applications of Eqs 3.26 and 3.27.

$$\begin{aligned}
 -(2EI_i) \tilde{w}_{i-1}^2 - (2hEI_i) \tilde{w}_{i-1}^3 + [2(EI_i + EI_{i+1}) + \tilde{s}_i^2 h^3] \tilde{w}_i^2 \\
 + [2h(EI_{i+1} - EI_i)] \tilde{w}_i^3 - (2EI_{i+1}) \tilde{w}_{i+1}^2 + (2hEI_{i+1}) \tilde{w}_{i+1}^3 \\
 = \tilde{p}_i^2 h^3
 \end{aligned} \tag{3.26}$$

$$\begin{aligned}
 (2hEI_i) \tilde{w}_{i-1}^2 + (1.5h^2 EI_i) \tilde{w}_{i-1}^3 + [2h(EI_{i+1} - EI_i)] \tilde{w}_i^2 \\
 + [2.5h^2 (EI_i + EI_{i+1}) + \tilde{s}_i^3 h] \tilde{w}_i^3 - (2hEI_{i+1}) \tilde{w}_{i+1}^2 \\
 + (1.5h^2 EI_{i+1}) \tilde{w}_{i+1}^3 = \tilde{p}_i^3 h^3
 \end{aligned} \tag{3.27}$$

Both the axial and lateral equations are easily solvable by the method in Ref 3.

Once the nodal point displacements have been found they may be substituted into Eq 3.19 to find the member-end-forces. The elastic support forces may be found by multiplying the spring constants times the negative of the nodal point displacements. Then the nodal point equilibrium error (the nodal point forces minus the element forces) may be calculated. This should be a negligible quantity except for the cases where a specified displacement is enforced by a large spring value and a correspondingly large force as discussed in Chapter 2.

Calculation of Member Fixed-End-Force Vector

The fixed-end-forces $\{FF\}_k$ for each nonprismatic or nonuniformly loaded member in the frame may be obtained by a discrete-element solution of the member. The member is subjected to all its member loads and elastic restraints and the end displacements are set equal to zero by using three large spring values at stations 1 and $m+1$. The end forces acting on the member at stations 1 and $m+1$ are the desired fixed-end-forces. The member-end-forces are essentially equal to the forces acting on the end elements of the member. However, when the member has a discretized load or elastic restraint at the end of the member, the member-end-force is equal to [(the element-end-force) - (the discretized load) + (the spring restraint) \times (the element-end-displacement)].

For members with pinned ends, the rotational spring restraint is not set equal to a large value, thus leaving it free to rotate.

Calculation of Member Stiffness Matrix

The stiffness matrix $[K]_k$ for each nonprismatic member in the frame may be obtained by six discrete-element solutions of the member. The member is subject to all its member elastic restraints and in turn six unit displacements are introduced corresponding to the three degrees of freedom at station 1 and station $m+1$.

The unit displacements are obtained by using three large springs at stations 1 and $m+1$ and six correspondingly large forces. Two of the solutions will be axial solutions as generated by Eq 3.24 and four of the solutions will be lateral solutions as generated by Eqs 3.25 and 3.26. The stiffness of the member does not change, hence the second axial solution and the second, third, and fourth lateral solutions do not require an elimination of the stiffness matrix. This saves a relatively small amount of time when considering the generation of a single member stiffness matrix, but when multiplied by a large number of members represents a sizable saving.

The member-end-forces at stations 1 and $m+1$ for each of the six solutions are the six columns of the member stiffness matrix. Actually only 13 of these 36 forces need be calculated and stored. Since $[K]_k$ is always symmetrical, only 21 of its coefficients need be known. For a plane frame member subject to small displacements, 8 of these 21 will always be zero. Thus, only 13

member-end-forces need be calculated and stored to generate $\left[K \right]_k$. The savings in calculations are insignificant but the savings in storage are considerable. Further reductions can be made if the member is assumed to have no elastic restraints acting between its ends. This was not done here to maintain the generality of the solution.

For pinned-end members, the corresponding rotational spring is not set to a large value and no moment is applied at the pinned end to enforce the unit rotation.

Calculation of Member Results

Once the frame joint displacements have been found by the solution of Chapter 2, they can be transformed into member-end-displacements. Then any frame member may be analyzed as a member subject to the member-end-displacements, applied member loads, and elastic restraints. The solution is similar to the solutions for $\left[K \right]_k$ and $\left\{ FF \right\}_k$ just discussed.

The displacements of all the stations will be found from such a solution. Then the spring support reactions may be found by multiplying the spring constants times the negative of the appropriate displacement. The element-end-forces may be found from Eq 3.19. Then the nodal point equilibrium error may be evaluated.

Shears, moments, and axial forces in the member can now be found by statics at any point desired. The output used in the program is the average value of shear, moment, and axial force at every other nodal point.

Comparison of Finite-Element and Discrete-Element Methods

For an "exact" solution of a member, nodal point equilibrium, compatibility, and boundary conditions must be satisfied at the $m+1$ nodal points (stations) and the force-displacement equations for the m elements must be satisfied. The exact force-displacement equations may not be known and are often obtained by an approximate method. The difference between the finite-element method and the discrete-element method is the way in which the elements force-displacement equations are approximated.

The element force-displacement equations obtained by the finite-element method (Ref 13) satisfy the constitutive relations and compatibility throughout

the element and approximately satisfy equilibrium and the boundary conditions throughout the element. The discrete-element method satisfies the constitutive relations and equilibrium throughout the element but only approximates compatibility and the boundary conditions throughout the element. For certain special cases either method may give the "exact" force-displacement equations.

CHAPTER 4. CONVERSION OF ENGINEERING INPUT

Much of the data describing a plane frame readily available to an engineer is not in the form needed to solve the computer model of the frame. Thus either the engineer has to perform the tedious and repetitive calculations necessary to convert the data, or this conversion can be incorporated into the computer program. The equations needed for converting the engineering data into a form compatible with the theory presented in Chapters 2 and 3 are given here and are made internally in the computer program discussed in the next chapter.

Joint Coordinates

The coordinates of each joint in the frame are used to calculate the direction cosines of the frame members. The engineer is more likely to have the projections of the members on the coordinate axes than the actual coordinates of each joint. Thus the offsets or projections x_o and y_o as shown in Fig 6 are a more logical choice for the input of the frame geometry.

If the coordinates of one reference joint are given, then the coordinates of a second joint can be computed by adding the appropriate offsets to the coordinates of the first joint. A third joint can be referenced by offsets to either of the first two, etc. In general, if joint i has been located and has coordinates x_i and y_i then the coordinates of joint j , x_j and y_j are given by

$$x_j = x_i + x_o \quad (4.1)$$

and

$$y_j = y_i + y_o \quad (4.2)$$

The member's direction cosines α and β are given by

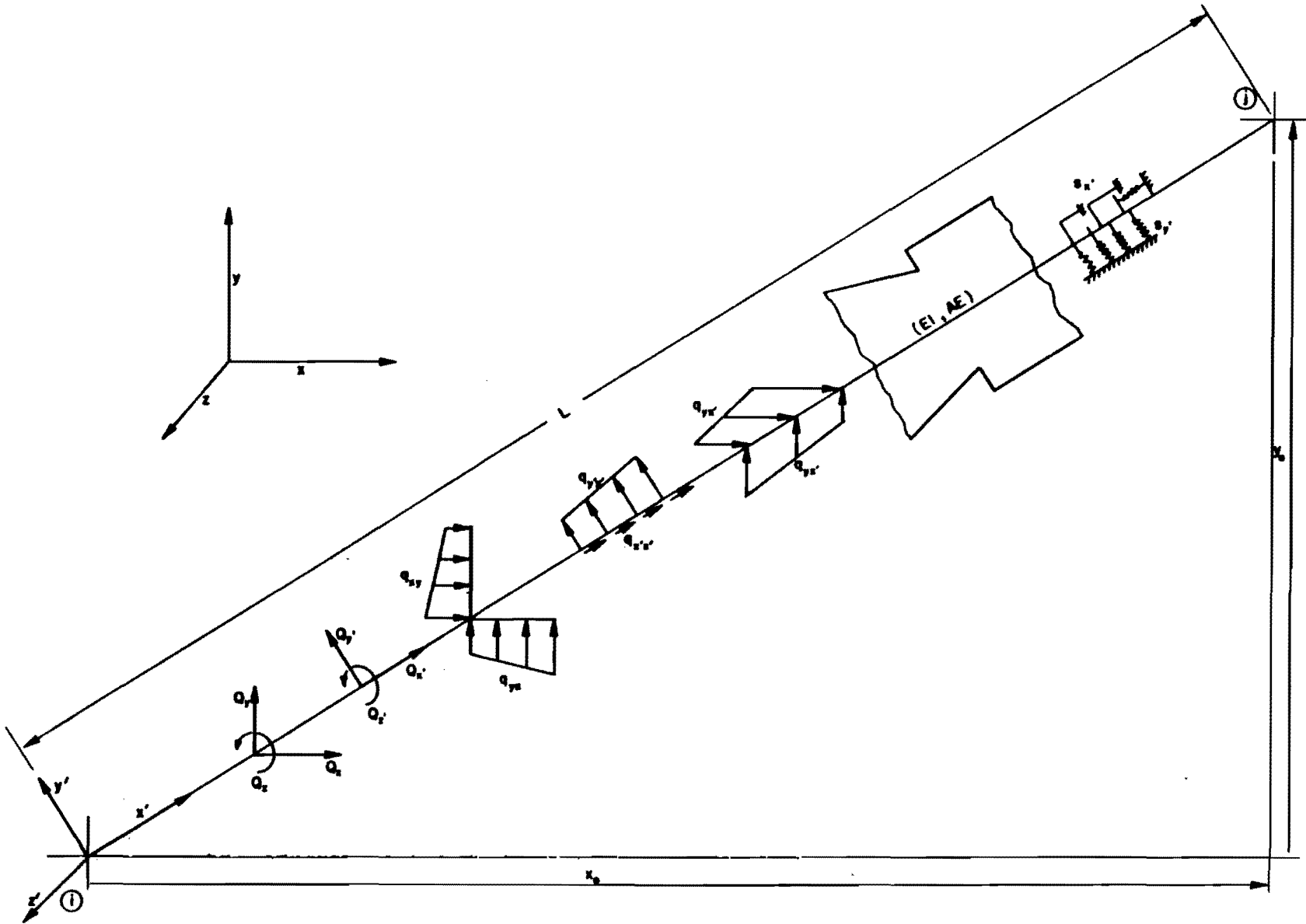


Fig 6. General frame member.

$$\alpha = (x_j - x_i)/L \quad (4.3)$$

and

$$\beta = (y_j - y_i)/L \quad (4.3)$$

where the length of the member L is given by

$$L = \sqrt{(x_j - x_i)^2 + (y_j - y_i)^2} \quad (4.5)$$

Equations 4.3 and 4.4 assume that the member x' -axis is directed from joint i to joint j . The program directs the member x' -axis in the direction that the member stiffness and load types are input.

Member Stiffness and Load Types

Many frames have several members with the same stiffness properties and/or loadings. The terms "stiffness type" and "load type" are used in the program to avoid duplication of input for such members. Two or more members have the same stiffness type if they have the same stiffness properties (length, distribution of axial and flexural rigidities and distribution of elastic restraints) and have their member axes parallel and similarly directed. Two or more members have the same load type if they have the same loadings, length, and similarly directed parallel member axes.

Member Load Data

Consider a member subjected to a variety of loads as illustrated in Fig 6. In general, Q_a is a concentrated load in the direction of the a -axis. Thus Q_x is a concentrated load in the direction of the member x' -axis, and Q_z is the concentrated moment about the z -axis. And in general, q_{ab} is a distributed loading in the direction of the a -axis and has its intensity per unit of length along the b -axis. Hence q_{yx} is a load in the direction of the structure y -axis and has its intensity per unit of length along the member

x' -axis. Such a load might be the member's own weight, if the structure y -axis is vertical.

A load is positive if directed in the positive sense of the direction axis or, in the case of a moment, if it is counterclockwise in the x - y plane.

Describing member loads in terms of Q_a and q_{ab} , giving the distance from a reference point to Q_a and the distance from a reference point to the starting and stopping points of q_{ab} is a convenient method of inputting member loads. However, the discrete-element model of Chapter 3 requires loads to be in the direction of the member axes and acting only at stations. Thus general member loads must be transformed into member coordinates and discretized to station values. The transformation of concentrated loads from the structure axes to member axes is accomplished by multiplying by the appropriate direction cosines α and β and summing as follows:

$$Q_{x'} = \alpha Q_x + \beta Q_y \quad (4.6)$$

$$Q_{y'} = -\beta Q_x + \alpha Q_y \quad (4.7)$$

and

$$Q_{z'} = Q_z \quad (4.8)$$

Distributed loads in the direction of the structure axes but with their intensity per unit of length along the member x' -axis transform in the same manner:

$$q_{x'x'} = \alpha q_{xx} + \beta q_{yx} \quad (4.9)$$

$$q_{y'x'} = -\beta q_{xx} + \alpha q_{yx} \quad (4.10)$$

and

$$q_{z'x'} = q_{zx} \quad (4.11)$$

The term $q_{z'x'}$ (not shown in Fig 6) is a moment per unit of length and probably not useful for normal design but is included for generality.

Distributed loads directed in one structure direction per unit of length in the other structure direction are transformed by Eqs 4.9 and 4.10 and then multiplied by the absolute value of the direction cosine between the member axis and the axis to which the intensity is referenced to give

$$q_{x'x'} = \alpha q_{xy} |\beta| + \beta q_{yx} |\alpha| \quad (4.12)$$

and

$$q_{y'x'} = -\beta q_{xy} |\beta| + \alpha q_{yx} |\alpha| \quad (4.13)$$

Distances to concentrated loads, etc., may be input in structure coordinates, x and y , and then divided by the appropriate direction cosine to give the distance along the member axis, x' , as follows:

$$x' = \frac{x}{\alpha} \quad (4.14)$$

or

$$x' = \frac{y}{\beta} \quad (4.15)$$

Discretizing Member Loads

The idea of replacing a complicated loading system with a simpler statically equivalent system is not new. Newmark's classic paper (Ref 12) gives a good practical discussion of the concept and a theoretical treatment is given in a paper by Mises (Ref 11). Mises points out the lack of generality of St. Venant's principle and gives a better criteria for the replacement of one load system by another.

Using Mises criteria a system of loads may be replaced by an equivalent system if the static difference of the two systems is zero and remains zero when the two systems are rotated through an arbitrary angle. Such systems are

said to be astatically equivalent. Then, if the real loading system and the astatically equivalent loading system are contained within a circle of diameter ϵ , the error in replacing the original system with the equivalent system will be of order ϵ^2 .

Consider a concentrated load Q_y' , as shown in Fig 7(a). The load may be imagined to be applied to a simple stringer supported at stations i and $i+1$. Such a stringer would give reactions at i and $i+1$ as follows:

$$\tilde{P}_i^2 = Q_y'(2h - c)/2h \quad (4.16)$$

and

$$\tilde{P}_{i+1}^2 = Q_y'c/2h \quad (4.17)$$

The terms \tilde{P}_i^2 and \tilde{P}_{i+1}^2 are the concentrated station loads required for the member analysis of Chapter 3 and are astatically equivalent to the actual load Q_y' . Both loading systems are contained within a circle of diameter $2h$ thus the error is of the order $(2h)^2$.

A load parallel to the member Q_x' , could be arbitrarily transferred to either station i or $i+1$ and it would still be statically equivalent to the original load but it would violate the principle of astatic equivalence. However, if formulas similar to Eqs 4.16 and 4.17 are used, the desired results are obtained. That is:

$$\tilde{P}_i^1 = Q_x'(2h - c)/2h \quad (4.18)$$

$$\tilde{P}_{i+1}^1 = Q_x'c/2h \quad (4.19)$$

A couple can be considered to be a system composed of two equal and oppositely directed forces a small distance apart. The forces may be transferred by Eqs 4.16 and 4.17 which has the effect of transferring the couple by Eqs 4.20 and 4.21.

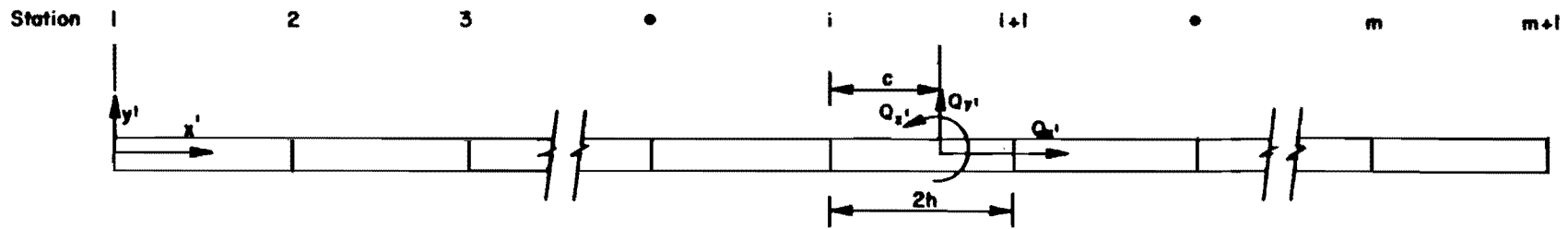


Fig 7(a). Concentrated loads.

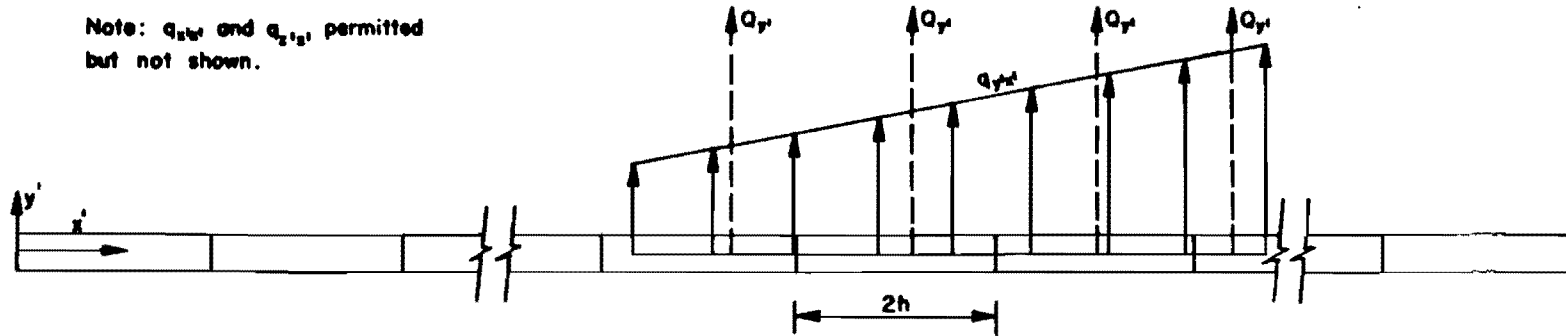


Fig 7(b). Distributed Loads.

Fig 7. Member loads in member coordinates.

$$\tilde{p}_i^3 = Q_z'(2h - c)/2h \quad (4.20)$$

$$\tilde{p}_{i+1}^3 = Q_z'c/2h \quad (4.21)$$

Linearly distributed loads such as $q_{y,x}'$, (shown in Fig 7(b)) may be first transformed into equivalent concentrated loads as suggested in the figure. A concentrated load is calculated for each element of the member that has any of the distributed loading acting on it. The formulae for the resultant of a linear load and its location may be found in any standard handbook. The concentrated loads thus found may be distributed by formulas 4.16 and 4.17. Higher ordered load distributions may be represented by the user as a series of linear loadings.

Distributed axial loadings $q_{x,x}'$, and distributed rotation loadings $q_{z,x}'$, may be handled in a similar manner. The similarity of the techniques is such that the computer program developed uses the same subroutines for all three types of loads (lateral, axial, and rotational).

Discretizing Member Stiffness Data

All member stiffness data are assumed to be given with respect to the member axes, as shown in Fig 6. That is, the cross-sectional area A is normal to the member x' -axis, the moment of inertia I is about the member z' -axis, s_x , is a distributed elastic spring restraint acting parallel to the member x' -axis, s_y , is a distributed elastic restraint acting parallel to the member y' -axis, and s_z , is a distributed rotational elastic restraint acting about the member z' -axis. Concentrated values of elastic restraint are not permitted.

It is shown in Appendix 1 that using the average values of EI and AE for a linear variation of these properties over an element gives a solution with second order converging properties. A second order converging solution is one in which the error decreases in proportion to the square of the element size. This is compatible with the astatic equivalencing of loads just described. Higher order variations in EI and AE may be specified by the user as a series of linear variations to achieve any desired degree of accuracy.

A jump in EI or AE in an element, as occurs at a cover plate, could for all practical purposes be handled by using the average values of EI and

AE for the element. However, this does not give second order convergence. To achieve this the M/EI and T/AE diagrams for an element may be balanced by choosing appropriate effective values of EI and AE to be used for the element. The term M is the bending moment at any point in the element and T is the axial thrust at any point in the element. Thus the M/EI diagram is the curvature diagram and the T/AE diagram is the axial strain diagram. These effective values should be chosen such that the errors in the positive area of the curvature and strain diagrams are balanced or offset by equal negative areas. This criteria gives

$$\overline{EI} = \frac{2hEI_1EI_2}{c_2EI_1 + c_1EI_2} \quad (4.22)$$

and

$$\overline{AE} = \frac{2hAE_1AE_2}{c_2AE_1 + c_1AE_2} \quad (4.23)$$

The terms \overline{EI} and \overline{AE} are the effective values to be used for the element of length $2h$. The terms EI_1 and AE_1 are the actual values of EI and AE for a distance c_1 , and EI_2 and AE_2 are the actual values for a distance c_2 .

Positive values of EI and AE throughout the element are assumed in obtaining Eqs 4.22 and 4.23. A zero value of EI which corresponds to an idealized pinned connection is not allowed interior to a member. However, this is not a practical limitation as a hinge can occur at any structural joint. Hence a structural joint may be defined at the location of the hinge and a member specified on both sides of the hinge.

A more realistic way of handling a reduction in moment resistance, which can only approach an idealized hinge, is to put in a reduced value of EI for a short distance around the hinge.

The discretizing of elastic spring restraints can be done in a fairly complicated manner. However, if the restraints are restricted to distributed values, then a second order converging solution can be obtained by handling

the springs exactly as distributed loads. That is by first obtaining equivalent concentrated values and then distributing these to the stations. The same formulae as used for loads can be used here. Concentrated elastic restraints, particularly very large values which might be used to set a displacement, cannot be handled in this manner without introducing unacceptable errors. Hence, the program accepts concentrated spring restraints only at structural joints where they are handled "exactly."

CHAPTER 5. COMPUTER PROGRAM

The computer program FRAME 11 has been written for the linearly elastic analysis of plane frames subjected to static inplane loads and has the restrictions outlined in the Introduction (Chapter 1).

The frame, its supports and loads are specified in input Tables 1 through 7. The results are given in output Tables 8, 9, and 10. These tables will be described after a short discussion of the four problem types the program works. A description of the internal workings of the program concludes this chapter. Chapter 6 gives the example problems of the report and additional details on the program are given in the appendices.

The program solves four distinct types of problems. The distinction of problem types is necessary both to increase computer efficiency and decrease the volume of input the user must supply. All problem types are related to the definition of a structure and its loads. For the purpose of defining the problem types, a structure should be considered to include the members of the frame, the member supports, and the joint supports. Any change in the members, their arrangement, or the supports creates a new structure.

Problem Type 1 - Regular Problem

When a structure is to be analyzed for only one loading it should be input as a problem Type 1 for the most efficient solution.

Problem Type 2 - Parent Problem

When a structure is to be analyzed for more than one loading condition, economics in computer time and man hours may be made by making the first solution a Type 2 problem.

Problem Type 3 - Offspring Problem

A structure previously analyzed as a Type 2 problem may be analyzed more economically for another loading condition by running it as a Type 3 problem.

Up to 20 Type 3 problems may follow a Type 2 problem. Type 3 problems must follow a Type 2 or a Type 3 problem. Only loads can be changed in a Type 3 problem. A computer diagnostic will appear if an attempt is made to change the structure in a Type 3 problem.

Problem Type 4 - Family Problem

When one structure is solved for several load conditions starting with a Type 2 problem the results are stored. This allows the designer to solve a structure for several basic load cases and then combine the results in a linear manner. (Exp. 1.5 dead load + 1.8 live load, etc.) Type 4 problems require only input Tables 1 and 7 and may not follow a Type 1 problem.

Input Tables

A detailed input guide is provided in Appendix 2.

Table 1. Program Control Data - consists of two cards which are required for all problems. The first card specifies the problem type, the tables for which data are held and allows the user to suppress output. The second card specifies the number of data cards in Tables 2 through 7. Data may be held for all types of problems but cannot be held on the first problem of a computer run. Data is generally held from the previous problem but after a Type 4 problem the data from the last problem other than a Type 4 problem may be held.

Table 2. Frame Geometry Data - defines the location of the structural joints of the frame. Joints are required at the intersections of two or more members and at the ends of members. Joints need not be input at concentrated loads but are required at locations of supports (concentrated linearly elastic springs) and at hinges (points of zero flexural stiffness).

The first card of Table 2 gives the total number of frame joints, the reference joint, its coordinates, and the joint location tolerance. The reference joint may be any joint and it may have any coordinates except all joints must have coordinates less than $1.0E + 50$. As many additional cards as necessary follow to specify the location of the remaining joints and check the location of as many joints as desired. When joints are located more than once the program compares the old and new coordinates. If the difference in either

coordinate (x or y) is greater than the joint-location tolerance a computer diagnostic appears, otherwise the program averages the old and new coordinates and continues.

The second and succeeding cards give the offsets of new joints with reference to previously defined joints. For example, if joint 3 is the reference joint the second card could locate joint 7 with respect to joint 3. The next card could then locate joint 1 with respect to either joint 3 or joint 7. When several joints are in a straight line and have identical offsets, they may be located with only one card. Joint offsets need not be given where members are, but all joints must be located at least once. The input data is echo-printed in Table 2 and in addition the computed joint coordinates are given.

Table 3. Member-Type Location - locates the members of the frame between the joints defined in Table 2. The use of member stiffness and load types reduces the volume of input required for large frames with repeated members. Two or more members have the same stiffness type if they have the same stiffness properties (length, distribution of axial and flexural rigidities, and distribution of elastic restraints) and have their member axes parallel and similarly directed. Two or more members have the same load type if they have the same loadings, length and similarly directed parallel member axes.

The first card of Table 3 contains the total number of stiffness types and load types in the frame. The second and succeeding cards give the location of the members in the frame and their stiffness and load types.

The members are input going "From" one joint "To" another joint. This orients the member x'-axis in the direction of the "To" joint. The orientation is given with the member output for interpreting results.

When several members with the same stiffness and load type are connected in a straight line, they may be input with only one card.

The stiffness and load types on a data card replace the old values for a member if old values exist. Thus if only one member's stiffness and load type change from the previous problem the data may be held in Table 3 and only the new values of stiffness type and load types given. And obviously, both the stiffness and load types must be given even if only one of them changes.

The input data is echo-printed and in addition the computed member numbers, lengths, and offsets are printed in Table 3.

Table 4. Joint Loads and Restraints - gives joint loads and restraints in the structure x-y-z axes. Frame supports are specified as linearly elastic restraints (springs). Realistic values may be used where available or fictitiously large values may be used to simulate unyielding supports.

A completely fixed support is obtained by specifying large horizontal (x), vertical (y), and rotational (z) springs at a joint. A pinned support would omit the rotational restraint and the free end of a cantilever would have no restraints.

A specified displacement may be enforced by inputting a large spring and a correspondingly large force equal to the spring restraint times the desired displacement.

Each card of Table 4 contains joint loads and restraints for one joint. Only joints with nonzero values need have a data card. No special order of the joints is required in Table 4. The table is accumulative and in addition to the echo-print of the data the accumulated joint data is printed.

Loads are positive if in the direction of the structure axes, thus counterclockwise couples are positive loads. Springs corresponding to stable supports will always be positive.

Table 5. Member Stiffness Data - specifies the stiffness data for the various stiffness types in the frame. One or more data cards are required to define each new stiffness type. Stiffness types must be input in ascending order and when Table 5 is held from a prior problem the first new stiffness type must be one more than the last stiffness type in the prior problem.

Prismatic members without elastic spring restraints require only one data card. Members with variable cross-sections or elastic-spring restraints require two or more data cards and the first card indicates how many additional cards follow.

Connections of members to the joint may be either pinned or rigid and are indicated on the first card for each stiffness type. This option either pins or rigidly attaches the member to the joint but does not in any way serve as a support for the frame (i.e., when a member has a pinned support at its end it must be specified in Table 4 even if a pinned connection to the joint is provided in Table 5).

Distances to locations of changes in stiffness are given from the members "From" joint and may be in either member or structure coordinates

depending upon the axis option chosen. Positive distances are in the direction of the chosen axis.

Either partial or complete member output may be requested in Table 5 for each stiffness type and all members with that stiffness type will have the specified output.

Table 6. Member Load Data - specifies the loadings for the various load types. One or more data cards are required to define each new load type. Load types must be input in ascending order and when Table 6 is held from a prior problem the first new load type must be one more than the last load type in the prior problem.

Members with only uniform loads over their full lengths may be input with only one data card. Other loadings require two or more cards and the first card indicates how many additional cards follow. Four axis options are provided which permit the user to describe the member loads in the most convenient manner. Loads are positive if they are in the direction of the chosen axes, thus, counterclockwise couples are always positive. Distances to concentrated loads and changes in distributed loads are given from the members "From" joint and are positive in the direction of the chosen axes.

Table 7. Compilation Table - specifies the problem numbers of previous problems and their appropriate load factor for a superposition solution. Table 7 has one data card for each previous problem which has a multiplier. No cards are input in Table 7 except for Type 4 problems.

A multiple of a Type 4 problem is not allowed but the data from the previous Type 4 problem may be held and combined with new multipliers. In addition to an echo-print of the data the accumulated multipliers are printed.

Output Tables

Table 8. Joint Displacements and Reactions - gives displacements and reactions for all frame joints. Only supported joints (those with elastic spring restraints) will have nonzero reactions.

Undefined displacements, such as the rotation of a joint to which all members are pinned, or all three displacements of a joint to which no members are connected, are indicated in the output by extremely large displacements. (Refer to example problem 1101.)

Joint displacements and reactions are in structure coordinates and positive in the positive coordinate directions. Thus positive rotations are counterclockwise.

Table 9. Member Results - gives, for all members, either member-end-forces or detailed output, as requested in Table 5. Member-end-force output consists of the axial forces, shears and bending moments at the ends of the member; complete member output lists the axial, lateral, and rotational displacements as well as the axial force, shear and bending moments at 21 equally spaced points along the member.

For either choice of output, the axial forces, shears, and bending moments are in normal member-sign convention (not a joint-sign convention). Positive axial force produces tension in the member, positive shear tends to raise the end of the member nearest the origin of the member's x' -axis, and positive bending moment produces tension on the bottom side of the member. Positive displacements are in the positive member axes directions, thus, positive rotations are counterclockwise. The direction of the member's x' -axis is as input in Table 3 and is given with the member output for convenience.

Distances to the 21 output points are given along the axis used to specify the members stiffness data and are positive if in the positive axis direction. Regardless of the input options used, the axial force is parallel to and the shear force is normal to the member's x' -axis.

The values of axial force, shear and bending moment are the normal engineering values except that average values are given when there is a double value at an interior point due to a concentrated load or couple.

The maximum equilibrium error in the member is also output with the member's results and should always be a negligible quantity.

Table 10. Joint-Equilibrium Errors - give the errors in equilibrium at the frame joints and should always be negligible except at a joint for which specified displacements are enforced. A valid check of such joints is not possible.

The equilibrium errors at other joints should be scanned by the designer to see that the program is working properly. In considering the relevance of an error the designer might ask what effect the error would have if applied to the joint as a load. Example problem 1201 has one of the largest joint equilibrium errors of the report, an error of 1.2 kip-inches. If this

were applied to the joint as a load it would scarcely change the distribution of moments in the members of 8,539 and 8,540 kip-inches respectively.

Program Description

FRAME 11 is written in FORTRAN IV and conforms to the requirements of "American Standard FORTRAN" (Ref 1). The program has been implemented and thoroughly checked on the CDC 6600 computer at the Computation Center of The University of Texas at Austin. Only minor modifications are necessary to convert the program to other machines.

Program flow charts, the glossary of notation, and the FORTRAN listing of programs are in Appendices 3, 4, and 5 respectively. The reader interested in developing a full understanding of the program may wish to refer to these appendices as he reads the remainder of this chapter. In particular, the flow diagram for subroutine FRAM11 should prove helpful.

There are four more or less distinct paths through the program taken by the four problem types. The program will be explained by examining these paths.

Regular Problem

A Type 1 or regular problem is the analysis of a structure for one single loading condition. Type 1 problems, as all problems after the first problem of a run, start by reading in the problem number and problem identification card. The first problem of a run also contains two run identification cards. Then unless the problem number is equal to "CEASE" the program inputs Table 1.

For Type 1 problems, calls are then made successively to JTCORD, MEMLOC, JNTDAT, RDMST, and RDMLD which input Tables 2 through 6, echo-print the data, and after making preliminary computations, print the additional data described earlier in the chapter.

Subroutine RDMST and RDMLD convert the member stiffness and load data from the input coordinates to member coordinates and from that point on all member data is expressed in member coordinates. The transformation to discrete station values is not made at this time, in order to conserve storage.

Subroutine COMP is then called which prints out the table heading for input Table 7 and indicates that there is no data in the table.

In each of the routines which inputs data, checks are made for inconsistencies. If a data error is found the program stops processing that problem, prints out an appropriate error message and searches the remaining data cards for an independent problem. All intermediate cards are listed in the output.

The next step is the calculation of the member's stiffness and fixed-end-force matrices in member coordinates. Members with the same stiffness type will have the same member stiffness matrix; thus, member-stiffness matrices are computed and stored by stiffness type. Members with the same load type need not necessarily have the same stiffness type; hence, the member fixed-end-force matrices are computed and stored for each individual member.

Subroutine FORMST is called to calculate the member's stiffness matrix. FORMST calculates the stiffness matrix using known formulae for prismatic members not elastically restrained. Other members have their stiffness matrix generated by applying the appropriate unit displacements as discussed in Chapter 3.

Members which require the unit-displacement technique first have their stiffness data discretized to station values by subroutine DISCST. Then axial and lateral solutions are performed by subroutine AXIAL and GRIP2A.

Subroutine AXIAL is a short routine for solving the members axial equilibrium equations. The routine, which is a slight modification of previous beam-column solution routines (Ref 9), takes advantage of the fact that the band width of the axial equations are only three terms wide.

Subroutine GRIP2A is the general-simultaneous equations routine of the program and is used to solve the member-equilibrium equations for lateral displacements and rotations and also the joint-equilibrium equations for the joint displacements (vertical, horizontal, and rotational).

GRIP2A calls FSUB1 which calls FSUB12 to generate the member stiffness and load matrix one row at a time. (In the frame solution, GRIP2A calls FSUB1 which in turn calls FSUB11 for the appropriate frame coefficients.)

GRIP2A is a modification of the general recursion-inversion routine GRIP2 reported in Ref 3. The modification reduces the storage required for an incore solution which proves to be the most economical for small and medium sized frames.

Only 13 of the 36-member stiffness coefficients need be calculated and stored. SMMT is the storage vector used and the relation of SMMT to the member-stiffness matrix $[K]_k$ is given in Eq 5.1.

$$[K]_k = \begin{bmatrix} \text{SMMT}(1) & & & & & \\ 0 & \text{SMMT}(3) & & & & \\ 0 & \text{SMMT}(4) & \text{SMMT}(7) & \text{(SYMMETRIC)} & & \\ \text{SMMT}(2) & 0 & 0 & \text{SMMT}(10) & & \\ 0 & \text{SMMT}(5) & \text{SMMT}(8) & 0 & \text{SMMT}(11) & \\ 0 & \text{SMMT}(6) & \text{SMMT}(9) & 0 & \text{SMMT}(12) & \text{SMMT}(13) \end{bmatrix} \quad (5.1)$$

Subroutine FORMLD is called to calculate the member fixed-end-force matrices. Prismatic members not elastically restrained have their fixed-end-force matrices calculated directly using known formulae. For other members, FORMLD does a member solution for the member subject to its member loads and zero-end displacements.

The member solutions are similar to those discussed in FORMST. Loads are discretized to concentrated station values by subroutine DISCLD.

The frame equilibrium equations are set up and solved by GRIP2A which calls FSUB1 which in turn calls FSUB11 to furnish the appropriate stiffness and load coefficients. FSUB11 is called for each row of equations but forms three rows of equations on every third call from GRIP2A in (3×3) and (3×1) submatrices following the procedure outlined in Chapter 3.

After the frame displacements are found, the corresponding reactions are then computed and the joint displacements and reactions are printed in Table 8.

The sum of the joint loads and reactions are then computed. (When the member-end-forces are subtracted away from these values, the remnants are the equilibrium errors at the joints.)

The member results are then found for each member in the frame by a call to MEMRES. Subroutine MEMRES solves each member for its member loads and the member-end displacements which are compatible with the joint displacements of the frame solution. Loads and stiffnesses are discretized in the same way as previously discussed. Subroutines AXIAL and GRIP2A are then called to solve

for the member's axial, lateral, and rotational displacements at each station. The end forces on the ends of the member elements are then computed and an equilibrium check made at each station. The average value of axial force, shear, and bending moment is then computed at alternate stations. (Output is at every other station.)

Subroutine ADJNTER is then called to subtract the member's-end-forces from the partially computed joint equilibrium error. The remnant after all members have had their forces subtracted away should be extremely small and is an indication of how accurately the frame-joint displacements have been computed. It should be noted that some error is introduced due to using the standard stiffness matrices for prismatic members and then evaluating the member-end-forces by the discrete-element solution. This could be avoided by calculating member-end-forces in a different manner (matrix techniques of Chapter 2) for prismatic member. But this would needlessly complicate the program.

As now programmed, the joint equilibrium errors thus serve to indicate the errors both in the solution process and in the two different models (continuous line element versus discrete line element). As shown in the example problems, the errors are negligible.

The member results are printed in Table 9 by subroutine PRINT9. Either complete or partial output is printed, depending on which is requested in Table 3.

The joint equilibrium errors are then printed in Table 10 and the program returns for a new problem.

Parent Problem

A Type 2 or Parent Problem differs from a Type 1 problem only in the tape operations required for a Type 2 problem.

Subroutine COMP forms a list of problem numbers for which the results are saved for future Family Problems. The first problem of the list is the Parent Problem.

After the solution of the frame equilibrium equations by GRIP2A, the recursion multipliers RM and RO are stored on Tape 2 for a Parent Problem so they will be available for future Offspring (Type 2) Problems. Note that since RM and RO are kept in core, they would be available for future solutions except that the routine GRIP2A is used for solving both frame equations and member equations.

Then the frame displacements, reactions and joint loads are stored on Tape 1 for future Family (Type 4) problems. The tape is rewound prior to writing data on it since the Parent Problem is the first of a series for which the results are stored.

Similarly, Tape 4 is rewound and the results for six members are written on it at one time. The member results are grouped in order to decrease the number of records written, thus reducing the time for tape operations.

Offspring Problem

A Type 3 or Offspring Problem skips the formation of the member stiffness matrices, since the stiffness data must be identical to the last problem. However, the member-end-forces matrices must be formed as was done for regular and Parent Problems.

Prior to calling GRIP2A for the solution of frame equilibrium equations, RM and RO are retrieved from Tape 2. Then RM and RO need not be calculated again by GRIP2A, thus greatly reducing the solution time for the equations. After the solution, RM and RO are again stored on Tape 2 for additional Offspring Problems.

The frame-joint displacements, reactions, and loads are stored on Tape 1 as was done for the Parent Problem. And member results are stored on Tape 4.

Family Problem

A Type 4 or Family Problem has a completely different path through the program. Input Tables 2 through 6 are not read in. In subroutine COMP, the problem numbers and multipliers are read in and a check is made to see that the problem numbers are in the list of problems (parent and their offspring) for which the results have been saved.

A Family Problem then calls subroutine SUM1 for a superposition solution of frame displacements, reactions, and joint loads, and outputs the displacements and reactions in Table 8.

The preliminary computation of joint equilibrium error is made by adding reactions and loads, as was done for the other problem types, and subroutine SUM2 is called for a superposition solution of the members. SUM2 also calls ADJNTER to complete the joint equilibrium error calculations and PRINT9 to print the member results requested.

The joint equilibrium errors are printed in Table 10, and the program returns for a new problem.

CHAPTER 6. EXAMPLE PROBLEMS

A number of examples have been solved to check the accuracy and usefulness of the program. The results for the truss analyzed in problems 1101-1103 are found to be almost identical to an analysis by statics and virtual work found in Ref 10. Numerous other examples not reported were worked to check the accuracy of the computer solution. Example problems 1201-1207 illustrate the use of the program in a meaningful series of problems for a two-story, two-way bent similar to certain highway structures. The results of this series were compared with independent solutions where feasible, and agreement was obtained in all cases.

The units used for all example problems are kips and inches, though any consistent set of units may be used.

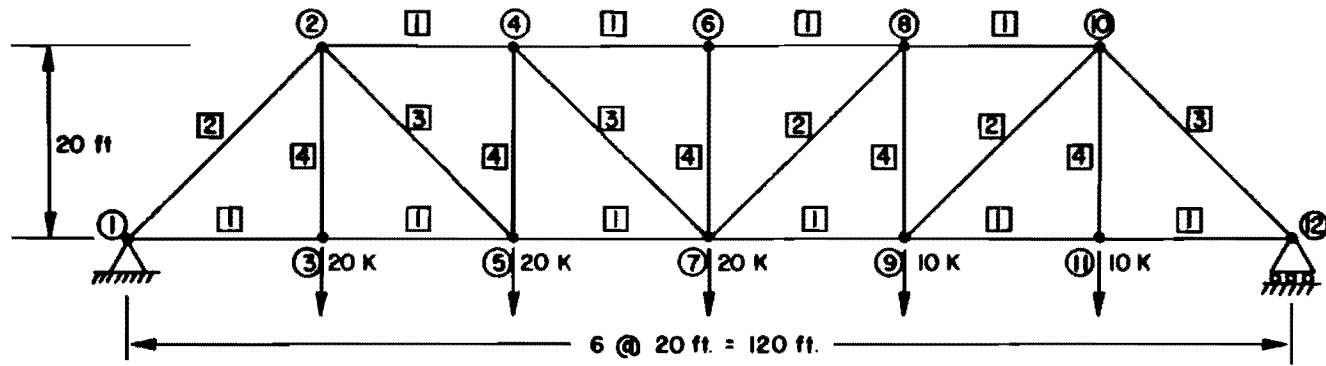
Problems 1101-1103 - Simple Truss

The truss of Fig 8 is analyzed in problems 1101-1103. This simple problem illustrates the savings in input possible when members are repeated in a structure.

In problem 1101, the joints are numbered 1 through 12 across the short direction of the truss. This numbering technique will generally give the most efficient computer solution. However, the program will accept any order of numbering as long as the difference between connected joints does not exceed nine and no joint numbers are omitted. The joints are located in Table 2, taking advantage of the uniform geometry of the problem.

All members are assumed unloaded in accordance with normal truss analysis techniques. Thus, all members are assigned zero load type. However, any member loads which might actually exist could be easily accommodated in a manner similar to the member loads of the bent problem.

All chord members have the same cross-section, modulus of elasticity, length, and orientation and hence are assigned a single-stiffness type, Type 1. Stiffness Type 1 is specified for all bottom chord members with a single card. The top chord is specified in a similar manner. All of the members with



Stiff Type	E (KSL)	Prismatic I(in ⁴)	Prismatic A(in ²)
1	30,000	40	40
2	30,000	2.25	3.0
3	30,000	2.25	3.0
4	30,000	1.0	2.0

Legend :
 ○ - Joint Number
 □ - Stiffness Type
 All Members Have Load Type 0

Fig 8. Truss with unsymmetrical loads.

stiffness Type 1 have their member x' -axis going from left to right because of the manner in which the stiffness types are input in Table 3.

All diagonal members have the same cross-section, modulus of elasticity, and length but they have two different orientations. Hence, two stiffness types are required. All vertical members are identical and thus have the same stiffness type.

Joint loads and restraints are input in Table 4. The restraints used are unrealistically large to mimic the idealized pin and roller supports shown.

The data describing the member stiffness types are input in Table 5 and the pinned-end connections are indicated here. Since the members are prismatic, only one card is required per stiffness type. The minimum output is selected for all members. Axis option 1 is used for all members but does not affect the input for prismatic members.

No cards are input for Tables 6 and 7.

Table 8 gives the joint reactions and displacements. The reactions are identical to the reactions of Ref 12 and the displacement of joint 9 of 1.399 inches compares favorably with the displacement of 1.4 inches found in Ref 10. The joint rotations are undefined since all members were specified as pinned-ended and the rotations of $1.0 \text{ E}+99$ indicate this. As anticipated, all shears and bending moments for the members are trivial. The axial forces can be easily verified by statics.

Table 10 gives the joint equilibrium errors and confirms that the solution is valid.

In problem 1102, the effects of rigid connections on the truss' behavior are examined. The stiffness types are modified to indicate rigid connections rather than pinned ones. The results are very similar to 1101 except for small bending moments and shears to which the members are subjected due to the continuity.

In problem 1103, the effects of the roller freezing on the truss of 1102 are investigated. As expected, the displacements and axial forces in the bottom chord are reduced but a large horizontal reaction is developed, which the foundation must resist.

Problems 1102 and 1103 required only a few additional data cards since most of the information could be held from problem 1101. Note, however, that

neither 1102 or 1103 could be worked as an Offspring Problem, since the stiffness of the structure changed in both cases.

Problems 1201-1207 - Two-Story Bent

A two-story bent is analyzed in problem 1201 for the live loads shown in Fig 9. The two columns on the left side of the frame have the same length and cross-sectional properties and are input as the same stiffness type. In order to do this, the pin at joint 1 must be specified as a joint property in Table 4 rather than as a member property in Table 3.

The pinned support is specified in Table 4 by using large vertical and horizontal restraints and no rotational restraint. The fixed support at joint 6 requires all three restraints.

The girders are the only members loaded in problem 1201 and are assigned load Types 1 and 2, as shown in Fig 9(b). All other members are assigned zero load type.

The first five stiffness types are prismatic and require only one card in Table 5 to specify their stiffness data. Stiffness Type 6 requires additional data cards to specify its data. Axis option 2 is used for stiffness Type 6. Therefore, horizontal distances are given to the locations of changes in stiffness. The distances are referenced to the "From" joint as defined by the input of Table 3.

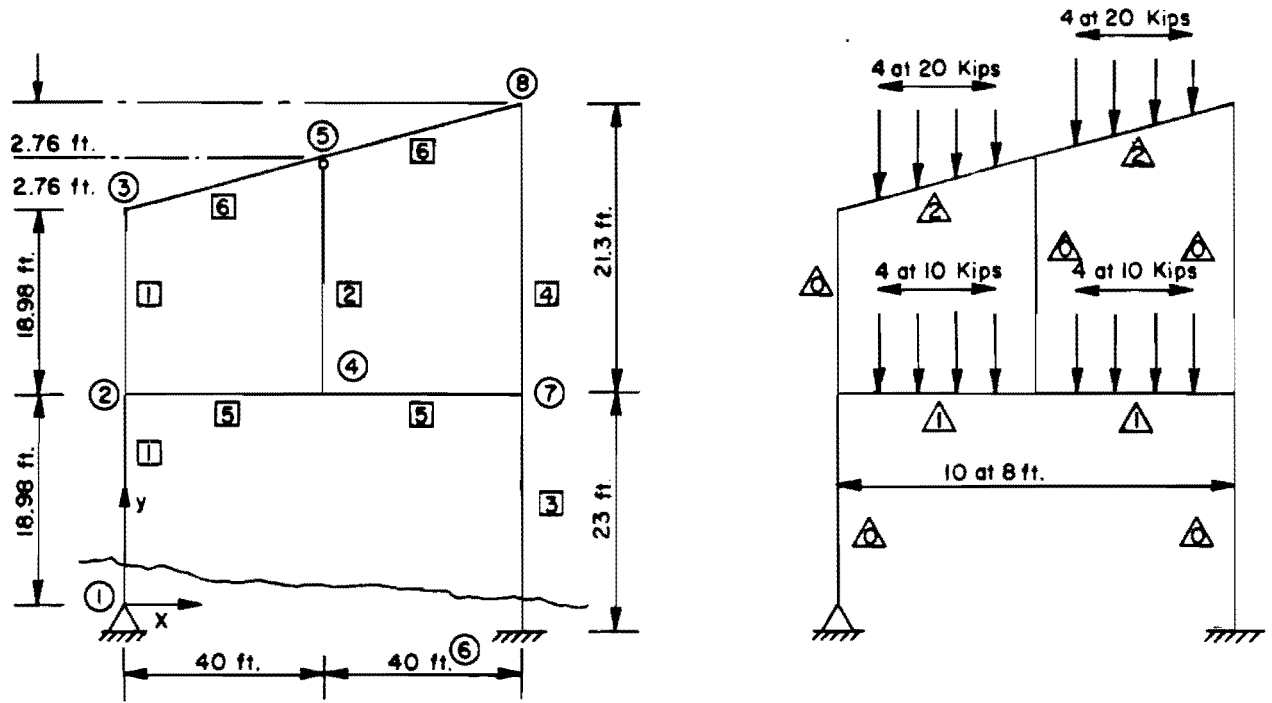
The concentrated live loads are input in Table 6 for load Types 1 and 2. Axis option 3 is used for both members so loads are input in the structure axes and distances are horizontal.

Table 8 gives the joint displacements and reactions. The reactions can be seen to be in equilibrium with the applied loads.

The member results are given in Table 9. Some of the members have only member-end-forces while others have complete output at the twentieth points, as requested in Table 3.

Table 10 gives the joint equilibrium errors, that is, the error in equilibrium of forces and moments at each joint. The maximum error is -1.2 kip-inches, which is negligible when compared with the moment in the members at the joint of 8,340 kip-inch.

In problem 1202, the frame of problem 1201 is modified by adding a column below joint 4, as shown in Fig 10. The resulting structure is analyzed for the live loads of problem 1201.



(a) Geometry and stiffness types.

(b) Live load.

- Legend**
- - Joint Number
 - - Stiffness Type
 - △ - Load Type

Stiff Type	E (K/in ²)	Prismatic I (in ⁴)	Prismatic A (in ²)	Variable Stiffness
1	29,600	1050	28	————
2	29,600	1800	20	————
3	29,600	1050	28	————
4	29,600	1050	28	————
5	29,600	1800	24	————
6	29,600	————	————	————

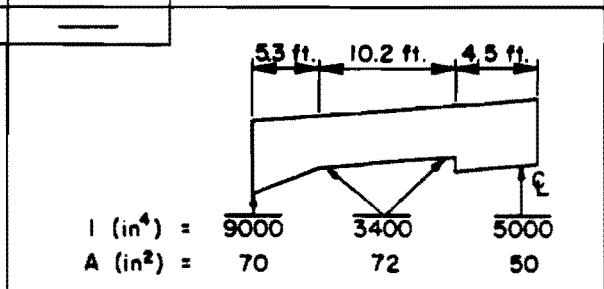
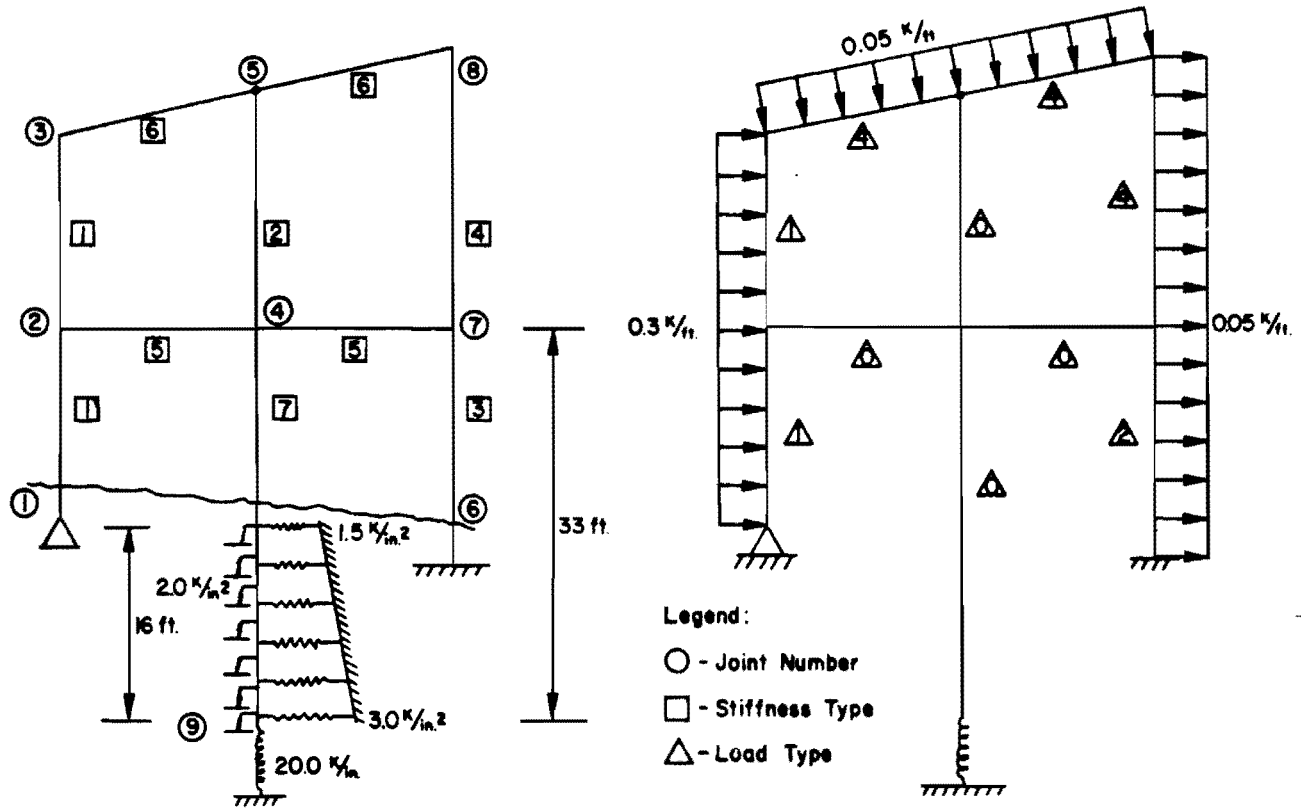


Fig 9. Two-story bent without interior column.



(a) Geometry and stiffness types.

(b) Wind load.

Stiffness Type	E (k/in ²)	Prismatic I (in. ⁴)	Prismatic A (in. ²)	Variable Stiffness
1-6	Same as Figure 9			
7	29,600	1050	28	

Dead Load:

Girders - 2.4 Kip/ft.
 Columns - 0.6 Kip/ft.
 16 Concentrated Loads of 4 Kips Each, Similar to Live Load

s_x' (k/in²)

s_y' (k/in²)

Fig 10. Two-story bent with interior column.

Only one additional joint has to be located in Table 2 and one additional member has to be located in Table 3. The member going from joint 9 to joint 4 is assigned stiffness Type 7 and load Type 0. Stiffness Type 7 is defined in Table 5. The end restraint on the column at joint 9 is input in Table 4.

The results of problem 1202 show that the displacements and bending moments are reduced by the addition of the column. The member results for the added member 10 show the variation in axial force, shear, and bending moment, as well as displacements, and could be extremely useful for design.

Problem 1202 was run as a Parent Problem since it was intended to be the first of a series in which the same structure (no change in stiffness properties) was analyzed for a group of loads. Problem 1203 is then an Offspring solution for the wind loads shown in Fig 10(b).

Four new load types are defined in problem 1203, as shown in Fig 10(b). To save the previous load types for future problems, hold Table 6 and define the new load types as 3-6. Note that in locating a new load type in Table 3, the corresponding stiffness type must also be input. And since problem 1203 is an Offspring Problem, the stiffness types must be identical to problem 1202 or a diagnostic message will occur.

Since all the wind loads shown in Fig 10(b) are normal to the members, axis option 1 is used for all load types. Only one data card is required for each load type since all loads are uniform over the full length of the members.

Problem 1204 is an Offspring solution of the frame defined in problem 1202 for the dead loads indicated in Fig 10. All of the members have a uniform gravity load (dead weight) and, in addition, the girders have concentrated gravity loads located at the same place as the live loads shown in Fig 9(b). Since all members are loaded, for convenience they are assigned a load type equal to their stiffness type. The load types are located in Table 3 and defined in Table 6 with all other data held.

All load types are input using axis option 2 which provides for loads acting in the direction of the structure axis. However, distributed loads have their intensity per unit of length along the member axis. The vertical and horizontal members could be defined using axis option 1 since their member axis coincides with one of the structure axes. If axis option 1 were used for the vertical members, the load would be $q_{x'x'}$ and be input in columns 31-40 instead of 41-50.

Problem 1205 is a superposition solution (Family Problem). A factor of 1.25 is applied to the dead load, live load, and wind load acting on the structure defined in 1202. The results could be obtained by hand from the results of 1202-1204, but the cost in man-hours would be far more than the computer costs for the Family Problem. Problem 1206 is a similar solution for 1.5 times the dead load plus 1.8 times the live load.

The column (pile) going from joint 9 to joint 4 is subdivided into two members at the point where the soil restraints start in problem 1207. The frame is then reanalyzed for the dead load and the results compared with problem 1204 (the previous dead-load solution). Subdividing the member into two elements gives a more accurate solution and an idea of the accuracy of the original solution in which the column with soil restraints over part of its length was input as one member.

Joint 10 is input at the ground line and all other joint coordinates are held. Stiffness Type 7 is deleted between joints 9 and 4 by specifying zero stiffness and stiffness Type 8 is input going from joint 10 to joint 4. Stiffness Types 9 and 10 are defined in Table 5.

The results of problems 1204 and 1207 agree within approximately one percent, except for a few locations where very small forces exist such as the tip reaction on the column-pile. Here most of the force has been removed by the axial restraints and the error is about 7 percent or approximately 1 kip. For normal design work, the original solution would certainly be accurate enough.

CHAPTER 7. CONCLUSIONS AND RECOMMENDATIONS

A direct matrix solution for plane frames has been developed that allows a designer to quickly solve problems that previously were difficult or impossible to solve.

A revision of the previous discrete element model was made to allow loads and restraints to act both parallel and normal to the members.

A computer program, FRAME 11, has been developed and is documented herein. The program has all the linear analysis capabilities of the program developed in Ref 2 and in addition to having the five features discussed in Chapter 1, on page 2, can work larger problems using the same amount of core storage. The program as presently dimensioned will work a frame with up to 75 joints and 150 members.

In order to satisfy the linear assumptions required for the superposition solutions, the beam-column effect of axial forces on lateral displacements was neglected as in normal practice under existing codes. However, various forms of nonlinear analysis are being more widely recognized by codes and the designer will soon need a more general nonlinear analysis program.

Preliminary studies indicate that the discrete-element model reported herein can be extended to include not only the beam-column effect, but other geometric and large displacement effects as well as nonlinear material properties.

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APPENDIX 1

ERROR ANALYSIS OF DISCRETE-ELEMENT MODEL

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APPENDIX 1. ERROR ANALYSIS OF DISCRETE-ELEMENT MODEL

The error in a model (either physical or mathematical) may be estimated in several ways. One widely used method is to compare the results of the model analysis with the results of another model analysis, the other model being generally accepted as an adequate model of the prototype. An accepted mathematical model of a frame member is a continuous line element.

In this appendix, the discrete-element model is shown to give results that agree with the continuous line element to within a predictably small error. The difference in resulting displacements between the two models is shown to be a function of the square of the element's length, i.e., of the order $(2h)^2$. This error is compatible with the method of discretizing loads and elastic restraints discussed in Chapter 4. Thus, as the element size decreases, the difference in the results of the two models rapidly approaches zero.

Consider the discrete element shown in Fig 4 (Chapter 3). Assume that the element is fixed at its left end, i.e.,*

$$w_{i-1}^1 = w_{i-1}^2 = w_{i-1}^3 = 0 \quad (\text{A1.1})$$

Now consider the effect of load f_i^2 only, the moment in the two springs M_1 and M_2 will be found by statics to be

$$M_1 = \frac{3f_i^2 h}{2} \quad (\text{A1.2})$$

$$M_2 = \frac{f_i^2 h}{2} \quad (\text{A1.3})$$

* The results of the derivation concern the change in displacements from one end of an element to the other; hence the actual starting values are immaterial.

These will produce corresponding angle changes Ψ_1 and Ψ_2

$$\Psi_1 = \frac{3f_i^2 h^2}{2EI_i} \quad (A1.4)$$

$$\Psi_2 = \frac{f_i^2 h^2}{2EI_i} \quad (A1.5)$$

where EI_i is the flexural rigidity of the element at midpoint.

From the geometry of the model

$$w_i^3 = \Psi_1 + \Psi_2 = \frac{2f_i^2 h^2}{EI_i} \quad (A1.6)$$

and

$$w_i^2 = \frac{3h\Psi_1}{2} + \frac{h\Psi_2}{2} = \frac{2.5f_i^2 h^3}{EI_i} \quad (A1.7)$$

The corresponding displacements will now be derived for a continuous element. The governing flexural differential equation for a continuous line is

$$\frac{d^2 y}{dx^2} = \frac{M}{EI} \quad (A1.8)$$

where y is the lateral displacement and x is the distance along the element measured from station i-1. Note that actually y'' and x'' are the element coordinates as given in Chapter 3 but are not used here in order to avoid confusion with the prime notation for derivatives.

The terms M and EI are the values of bending moment and flexural rigidity at any point along the element. The moment is easily found by statics to be

$$M = f_i^2(2h - x) \quad (A1.9)$$

For a linear variation in stiffness, EI may be expressed in terms of EI_i as

$$EI = EI_i(1 - \gamma h + \gamma x) \quad (A1.10)$$

where γ is the slope of the EI line divided by EI_i
Thus, Eq A1.8 gives

$$\frac{d^2 y}{dx^2} = \frac{f_i^2(2h - x)}{EI_i(1 - \gamma h + \gamma x)} \quad (A1.11)$$

Integrating Eq A1.11 gives

$$\frac{dy}{dx} = \frac{f_i^2}{EI_i} \left(\int \frac{2h dx}{1 - \gamma h + \gamma x} - \int \frac{x dx}{1 - \gamma h + \gamma x} \right) \quad (A1.12)$$

$$\frac{EI_i}{f_i^2} \left(\frac{dy}{dx} \right) = \frac{-x}{\gamma} + \frac{(1 + \gamma h)}{\gamma^2} \ln(1 - \gamma h + \gamma x) + C_1 \quad (A1.13)$$

C_1 is constant which can be evaluated by using the boundary condition that at the left end the slope (dy/dx) is zero. Solving for C_1 and substituting in Eq A1.13 gives

$$\frac{EI_i}{f_i^2} \left(\frac{dy}{dx} \right) = \frac{-x}{\gamma} + \frac{(1 + \gamma h)}{\gamma^2} \ln \left(\frac{1 - \gamma h + \gamma x}{1 - \gamma h} \right) \quad (A1.14)$$

Integrating Eq A1.14 and evaluating the constant of integration C_2 yields

$$\frac{EIy}{f_i^2} = \frac{-x^2}{2\gamma} + \frac{(1 + \gamma h)(1 - \gamma h + \gamma x)}{\gamma^3} \left[\ln \left(\frac{1 - \gamma h + \gamma x}{1 - \gamma h} \right) \right] + \frac{(1 - \gamma h)(1 + \gamma h)}{\gamma^3} \quad (A1.15)$$

At the right end of the element ($x = 2h$), Eq A1.14 gives

$$\frac{dy}{dx} = \frac{f_i^2}{EI_i} \left[\frac{-2h}{\gamma} + \frac{(1 + \gamma h)}{\gamma h} \ln \left(\frac{1 + \gamma h}{1 - \gamma h} \right) \right] \quad (A1.16)$$

Expanding the natural logarithm function in a Taylor series gives

$$\frac{dy}{dx} = \frac{f_i^2}{EI_i} \left(2h^2 + \frac{2\gamma h^3}{3} + \text{higher-order terms} \right) \quad (A1.17)$$

This differs from the slope at the left end of the element in the discrete-element model as given in Eq A1.6 by the error term ϵ^* ,

$$\epsilon^* = \frac{2f_i^2\gamma h^3}{3EI_i} \quad (A1.18)$$

and this can be expressed in terms of the elements length $2h$ as

$$\epsilon^* = \frac{f_i^2\gamma(2h)^3}{4EI_i} \quad (A1.19)$$

When m of these elements are connected end to end where

$$m = L/2h \quad (A1.20)$$

the total error ϵ will be the sum of the m errors given by Eq A1.19 or Eq A1.21

$$\epsilon = \frac{Lf_i^2 \gamma (2h)^2}{4EI_i} \quad (\text{A1.21})$$

Thus, the difference in the slope between the two ends of the element is a function of the square of the element size or of the second order.

Then evaluating the difference between the discrete element's displacement as given by Eq A1.7 and the continuous element's displacement at the right end of the element as given by Eq A1.15, yields an error term of

$$\epsilon = \frac{Lf_i^2 (2h)^2}{48EI_i} \quad (\text{A1.22})$$

Similarly, the lateral displacement and rotation due to a moment, and the axial displacement due to a force parallel to the member, may be shown to give a second-order error term. Thus, as the number of elements increase, the difference between the discrete element and the continuous element rapidly approaches zero. This theory was checked for a large number of examples and confirmed.

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APPENDIX 2

GUIDE FOR DATA INPUT

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Units of force (f) and distance (d) are indicated below all dimensional input.

IDENTIFICATION OF RUN (2 alphanumeric cards per run)

	80
	80

IDENTIFICATION OF PROBLEM (one alphanumeric card for each problem; program stops if problem number = CEASE)

Problem
Number

1	5	11	Description of problem	80
---	---	----	------------------------	----

TABLE 1. PROGRAM CONTROL DATA (2 cards per problem)

	Hold Options for Tables 2 through 7 Enter 1 to Hold Prior Data		Output Options for Tables 8 Through 10 Enter 1 to Suppress Output									
PROB TYPE	TABLE 2	3	4	5	6	7	8	9	10			
6	16	20	25	30	35	40	45	51	55	60	65	(1st card)
Number of Cards added in Tables 2 through 7 for this problem												
	TABLE 2	3	4	5	6	7					(2nd card)	
	16	20	25	30	35	40	45					

- PROB TYPE 1 - Regular Problem - single solution of structure.
- PROB TYPE 2 - Parent Problem - first of a series of solutions of one structure in which the stiffness properties of the structure do not change.
- PROB TYPE 3 - Offspring Problem - a solution of a structure previously solved as a Parent Problem (PROB TYPE 2).
- PROB TYPE 4 - Family Problem - a combination of multiples of Offspring Problems (PROB TYPE 3) and possibly their Parent Problem (PROB TYPE 2). A multiple of a Family Problem is not permitted but the previous Family Problem may be held.

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TABLE 2. FRAME GEOMETRY DATA (number of cards per Table 1)

Num of Joints		Ref Joint		x-Coordinate		y-Coordinate		Joint Location Tolerance							
11	15	21	25	31	(d)	40	d	50	61	(d)	70	(1st card)			
From Joint	x-Offset		y-Offset		To Joint	To Joint	To Joint	To Joint	To Joint	To Joint	To Joint				
11	15	21	(d)	30	(d)	40	46	50	55	60	65	70	75	80	(2nd and succeeding cards)

TABLE 3. MEMBER TYPE LOCATION (number of cards per Table 1)

Num of Stiffness Types		Num of Load Types														
11	15	21	25									(1st card)				
From Joint	Stiffness Load Type		Type		To Joint	To Joint	To Joint	To Joint	To Joint	To Joint	To Joint	To Joint				
6	10	16	20	25	31	35	40	45	50	55	60	65	70	75	80	(2nd and succeeding cards)

TABLE 4. JOINT LOADS AND RESTRAINTS IN STRUCTURE x,y,z-AXES (number of cards per Table 1)

Joint	Load // to x-Axis	Load // to y-Axis	Moment about z-Axis	Restraint // to x-Axis	Restraint // to y-Axis	Rotational Restraint about z-Axis								
6	10	(f)	20	(f)	30	(f)	40	(f/d)	50	(f/d)	60	(f/d)	70	(All cards)

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TABLE 5. MEMBER STIFFNESS DATA (number of cards as per Table 1; number of sets of cards equal to the number of stiffness types defined in this problem)

(1st card of set)	Stiffness Type	Modulus of Elasticity	Blank, if 2nd Card Used		Prismatic Moment of Inertia	Prismatic Area	Num Cards Follow	Axis Option	Output Option	Pin (From)	Pin (To)	
6	10	($1/d^2$)	31	40	(d^4)	(d^2)	56	60	65	70	75	80
(2nd and succeeding cards of set)	From (Distance)	To (Distance)	Moment of Inertia	Area	Restraint // to x'-Axis	Restraint // to y'-Axis	Rotational Restraint about z'-Axis					
11	(d)	(d)	I	A	$S_{x'}$	$S_{y'}$	$S_{z'}$					
			(d^4)	(d^2)	($1/d^2$)	($1/d^2$)	(1)					

+ If equal to 1, distances are measured along the member x'-axis; if equal to 2, distances are measured along the structure x-axis; if equal to 3, distances are measured along the structure y-axis. Member output distances for shear diagram, etc. are controlled by this option. In all cases the restraints are with reference to the local member (primed) axis. See page 10 of this appendix for an example using the various axis options.

‡ If blank, detailed output is given; if equal to 1, only member-end-forces are given.

‡ If blank, the member is assumed rigidly connected to joint at "From" end. If equal to 1, the member is assumed pinned to joint at "From" end.

‡ If blank, the member is assumed rigidly connected to joint at "To" end. If equal to 1, the member is assumed pinned to joint at "To" end.

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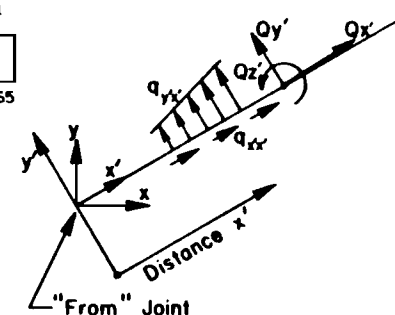
TABLE 6. MEMBER LOAD DATA (number of cards per Table 1; number of sets of cards equal to the number of load types defined in this problem)

Member loads may be input by any one of the four axis options outlined below. Q_a is the concentrated load in the direction of the a-axis. q_{ab} is the distributed load in the direction of the a-axis and has its intensity per unit of length along the b-axis.

Note: Concentrated loads may not be input at a distance of 0.0.

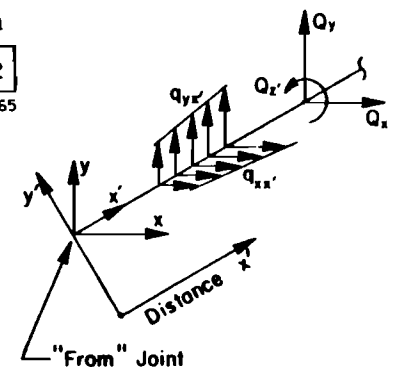
AXIS OPTION 1

(1st card of set)	Load Type	←Blank, if 2nd card used→ Uniform Load // to x'-Axis Uniform Load // to y'-Axis			Num Cards Axis Follow Option
6	10	31	40	50	56 60 65
		$q_{x'x'}$ (1/d)	$q_{y'x'}$ (1/d)		1
(2nd and succeeding cards of set)	From (Distance along Member)	To (Distance along Member)	Load // to x'-Axis	Load // to y'-Axis	Moment about z'-Axis
11	x' (d)	20	x' (d)	$Q_{x'}$, $q_{x'x'}$ (1), (1/d)	$Q_{y'}$, $q_{y'x'}$ (1), (1/d)
					$Q_{z'}$, $q_{z'x'}$ (1d), (1)
					60



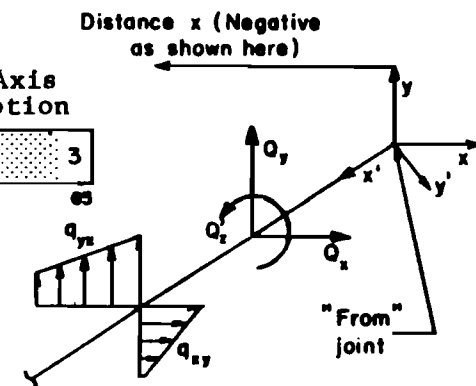
AXIS OPTION 2

(1st card of set)	Load Type	←Blank, if 2nd Card used→ Uniform Load // to x-Axis Uniform Load // to y-Axis			Num Cards Axis Follow Option
6	10	31	40	50	56 60 65
		$q_{xx'}$ (1/d)	$q_{yx'}$ (1/d)		2
(2nd and succeeding cards of set)	From (Distance along Member)	To (Distance along Member)	Load // to x-Axis	Load // to y-Axis	Moment about z'-Axis
11	x' (d)	20	x' (d)	Q_x , $q_{xx'}$ (1), (1/d)	Q_y , $q_{yx'}$ (1), (1/d)
					Q_z , $q_{z'x'}$ (1d), (1)
					60



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(1st card of set)		Load Type		AXIS OPTION 3						Num Cards Follow		Axis Option	
				Blank if 2nd Card Used			Uniform Load // to x-Axis		Uniform Load // to y-Axis				3
							q_{xy}		q_{yx}				
		6 10		31 (f/d) 40			40 (f/d) 50		36 60		60 65		
(2nd and succeeding cards of set)		From (Distance along Structure Axis)		To (Distance along Structure Axis)		Load // to x-Axis		Load // to y-Axis		Moment about z'-Axis			
		x		x		Q_x, q_{xy}		Q_y, q_{yx}		$Q_{z'}, q_{z'x'}$			
		11 (d) 20		20 (d) 30		30 (f) (f/d) 40		40 (f) (f/d) 50		50 (fd) (f) 60			



AXIS OPTION 4

Axis Option 4 is identical to Axis Option 3 except distances are in structure y-axis and 4 is input in column 65 of first card.

See page 11 of this appendix for an example using the various axis options.

The member x'-axis goes from the "From" joint to the "To" joint. The "From" and "To" joints are determined by input of Table 3.

TABLE 7. COMPILATION TABLE (number of cards per Table 1; no cards unless PROB TYPE 4)

Problem Number		Multiplier		(All cards)	
6 10		21 30			

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The data cards must be stacked in proper order for the program to run.

A consistent system of units of force (f) and distance (d) must be used for all input data, i.e., pounds and inches.

All 5-space words are understood to be integers

+ 4 3 2 1

All 10-space words are floating point decimal numbers

- 4 . 3 2 1 E + 0 3

All numbers must be right justified.

The problem number may contain alphanumeric characters.

Blank fields on data cards, except the first five columns, may be used as desired to aid in coding problems. Information in these fields is ignored by the program.

TABLE 1. PROGRAM CONTROL DATA

Type 4 (Family) Problems require only the problem type on the first card and the number of cards in Table 7 on the second card.

Data are accumulated in Tables 2 through 7 until the corresponding Hold Option is left blank.

When a nonfamily problem follows a Family Problem the data in Tables 2 through 6 may be held from the last nonfamily problem worked.

The maximum number of cards accumulated in Table 5 is 75 plus the number of stiffness types.

The maximum number of cards accumulated in Table 6 is 150 plus the number of load types.

Type 1 (Regular) and Type 2 (Parent) Problems may appear at any location in a run. However, Type 3 (Offspring) Problems must follow either their Parent or a related Offspring. Type 4 (Family) Problems must follow either their Parent, a related Offspring, or another Family Problem.

TABLE 2. FRAME GEOMETRY DATA

The first card gives the total number of joints in the frame, which must not exceed 75.

The reference joint, its coordinates, and the joint location tolerance are given only if the Hold Option for Table 2 is not exercised.

A joint number may not be deleted in a series until the Hold Option is not used. However, the joint may be structurally deleted by removing all members intersecting at the joint.

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The maximum difference in joint numbers, for joints that are connected by members is 9.

Joints are numbered from 1 to the total number of joints.

The reference joint may be any joint and it may have any coordinates, except that it and all other joints must have coordinates less than 1.0E50.

The second and succeeding cards in Table 2 specify the location of all additional joints in the frame at least once. If the Hold Option is used, only the new joints must be specified.

All offsets must be "From" a previously located joint "To" another joint. The "To" joint may be a previously defined joint. This allows the user to check the locations of the joints. If the error in the location of the joint is within the joint location tolerance then the solution continues; otherwise, the solution terminates with an appropriate diagnostic.

The joint location tolerance should allow for normal round-off error. If offsets are input to the nearest 0.01 foot then a joint location tolerance of 0.03 foot usually will be sufficient for a moderate sized frame.

The repetition of the "To" joint allows the user to locate up to seven joints with one card, if the offsets between each new "To" joint are the same as between the "From" joint and the first "To" joint.

It is not necessary for offsets to be given at locations where members are. However, the location of all joints must be specified at least once.

TABLE 3. MEMBER TYPE LOCATION

The first card in Table 3 gives the total number of stiffness types and the total number of load types.

Stiffness and load types (other than zero) are numbered from one to their total number. The total number of stiffness types must not exceed 50. The total number of load types must not exceed 50.

The total number of members in the frame must not exceed 150.

Type zero stiffness is used to delete a previously defined stiffness. Type zero load is used to indicate no load on a member. The restrictions on length, orientation, etc., outlined below do not apply to members with type zero stiffness and type zero load.

In order for two members to have the same stiffness type they must have the same length, the same angular orientation in the frame, and the same stiffness properties with respect to their "From" and "To" joints, i.e., they must have the same member stiffness matrix both in their member coordinate system and the global coordinate system.

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In order for two members to have the same load type they must have the same length, the same angular orientation in the frame and the same loading with respect to their "From" and "To" joints.

The member coordinate axes are defined by the "From" and "To" joints specified. The member x' -axis starts at the "From" joint and goes to the "To" joint. The member y' -axis and z' -axis are located from the member x' -axis by the right hand rule.

All members in the frame must be assigned a stiffness type and a load type. This assignment is not accumulative for a member in the frame, i.e., the last values of stiffness type and load type specified replace the previous values.

Stiffness and load types for a member must be specified on the same card.

Up to ten members with the same stiffness and load type may be located with a single card if the "From" joint of each new member is the "To" joint of the previous one.

TABLE 4. JOINT LOADS AND RESTRAINTS

All joint loads and restraints are specified with respect to the structure axes.

Joint loads and restraints are accumulated in Table 4.

Structure supports are input as joint restraints (linearly elastic springs). Complete fixity of a joint may be achieved by putting in very large spring values. No round-off errors are encountered when extremely large values are used unless large values are input and then subtracted away.

Complete freedom of joint movements is obtained by not specifying any restraints at a joint.

A specified displacement may be obtained by specifying a very large restraint and a corresponding force equal to the specified displacement times the large restraint.

TABLE 5. MEMBER STIFFNESS DATA

Stiffness types must be input in ascending order. If Table 5 is held from the previous problem then the first new stiffness type in Table 5 (if any) must equal the number of stiffness types in the last problem plus one.

Prismatic members may be input with one card. Members with varying stiffness and/or elastic restraints along their length require two or more cards.

If more than one card is used to describe a member stiffness type, the prismatic stiffness properties must be left blank.

Variable stiffness properties must be input continuously in sections starting at the "From" joint and continuing uninterrupted to the "To" joint. This format is illustrated in page 10 of this appendix.

Distances are given from the "From" joint.

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Each section must describe all of the stiffness properties of the member for a length greater than $1/40$ of the member's length.

Constant values of stiffness require one card per section.

A linear variation in stiffness requires two cards per section.

Concentrated values of stiffness are not allowed. A point of zero flexural stiffness is input as a pin at a joint. A concentrated spring restraint must be input as a joint restraint.

TABLE 6. MEMBER LOAD DATA

Load types must be input in ascending order. If Table 6 is held from the previous problem then the first new load type in Table 6 (if any) must equal the number of load types in the last problem plus one.

Load types with only uniform loads over their full length may be input with only one card. Other loadings require two or more cards.

If more than one card is used to describe a member load type, the uniform loads on the first card must be left blank.

Variable, concentrated, and partial uniform loadings must be input in sections but need not be input consecutively and sections may overlap. This format is illustrated on page 11 of this appendix.

Section lengths must exceed $1/40$ of the member's length except for concentrated loads where the "From" and "To" distances are equal. Concentrated loads may not be specified at a distance of 0.0.

All sections except concentrated loads must have their "To" distance larger in absolute value than their "From" distance.

Concentrated loads and sections with constant loading require one card. A linear variation in loading requires two cards per section.

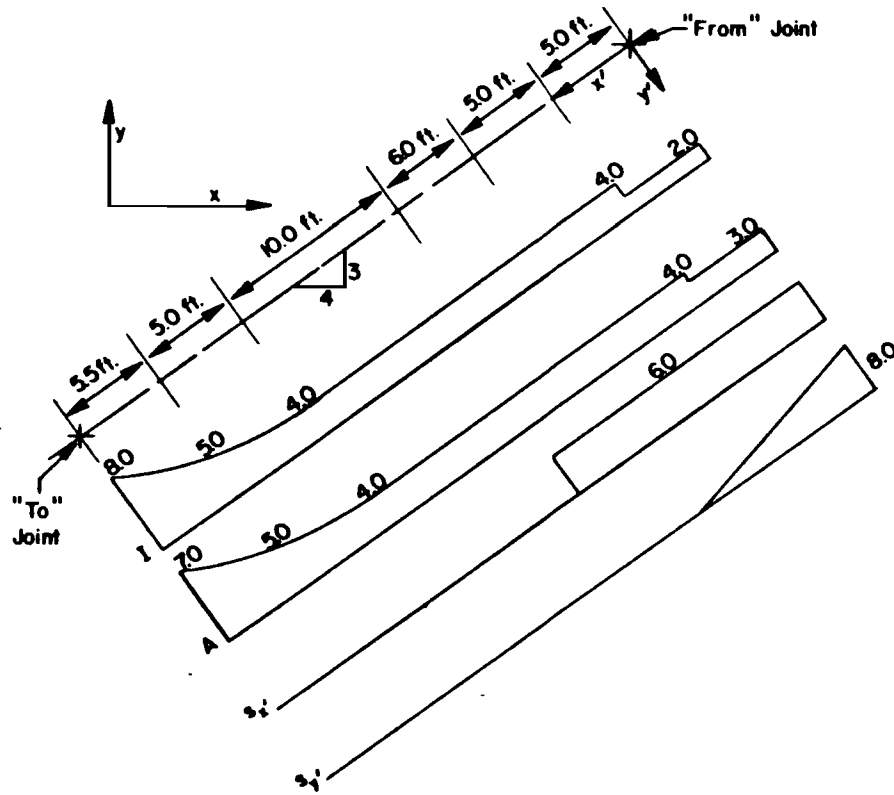
TABLE 7. COMPILATION TABLE

Each Parent Problem starts a series where the Parent solution and succeeding Offspring solutions are stored. Each of these solutions may be multiplied by a multiplier (load factor) and accumulated. The maximum number of consecutive Offspring problems is 20.

If the Hold Option is used the solution of the preceding Family Problem is added to the solutions of the additional load cases specified in the new Family Problem.

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Variable Member Stiffness



Notes: Sections must be input in order.

Linear format is required even if only one of the stiffness properties varies linearly.

A new section must be started when a change in the variation of any of the stiffness properties occurs.

There is no restriction on the length of a section except it must exceed 1/40 of members length.

"From" and "To" joints are set by input in Table 3.

From	To	I	A	s_x	s_y	s_z
* 0.0		2.0	3.0	6.0	8.0	
	5.0	2.0	3.0	6.0	4.0	
* 5.0		4.0	4.0	6.0	4.0	
	10.0	4.0	4.0	6.0	0.0	
• 10.0	16.0	4.0	4.0	6.0		
• 16.0	26.0	4.0	4.0			
* 26.0		4.0	4.0			
	31.0	5.0	5.0			
* 31.0		5.0	5.0			
	36.5	8.0	7.0			

Axis Option 1

* 0.0		2.0	3.0	6.0	8.0	
	-4.0	2.0	3.0	6.0	4.0	
* -4.0		4.0	4.0	6.0	4.0	
	-8.0	4.0	4.0	6.0	0.0	
• -8.0	-12.8	4.0	4.0	6.0		
• -12.8	-20.8	4.0	4.0			
* -20.8		4.0	4.0			
	-24.8	5.0	5.0			
* -24.8		5.0	5.0			
	-29.2	8.0	7.0			

Axis Option 2

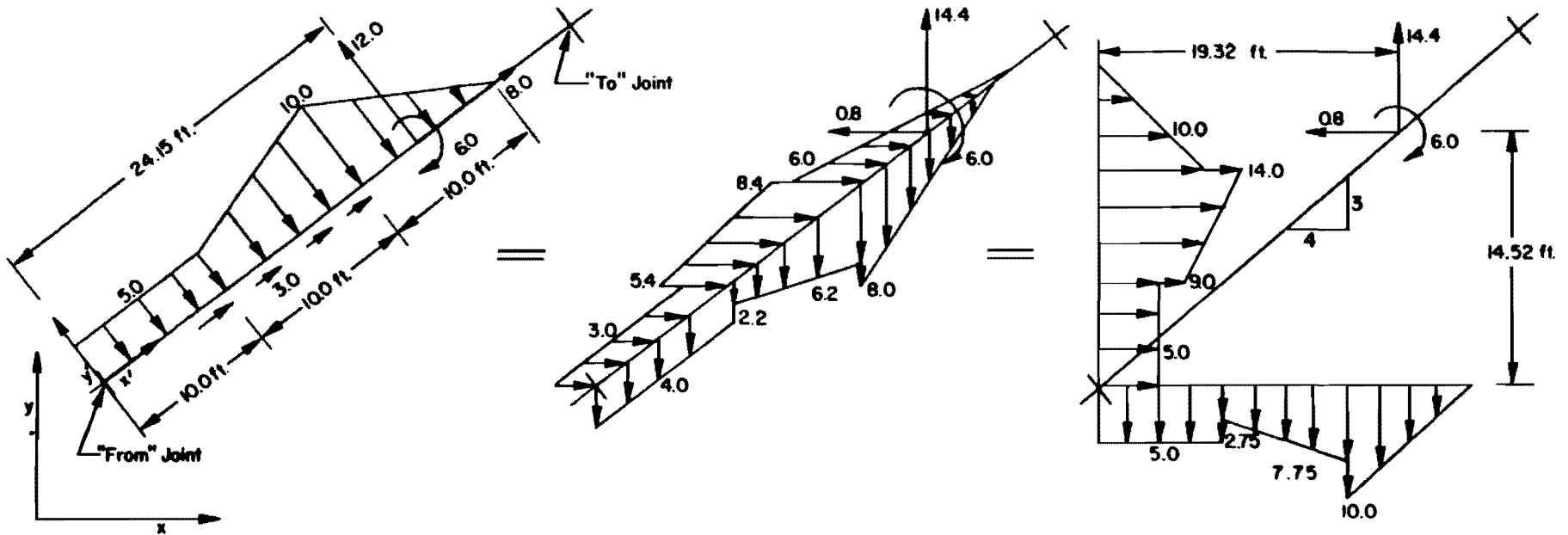
* 0.0		2.0	3.0	6.0	8.0	
	-3.0	2.0	3.0	6.0	4.0	
* -3.0		4.0	4.0	6.0	4.0	
	-6.0	4.0	4.0	6.0	0.0	
• -6.0	-9.6	4.0	4.0	6.0		
• -9.6	-15.6	4.0	4.0			
* -15.6		4.0	4.0			
	-18.6	5.0	5.0			
* -18.6		5.0	5.0			
	-21.9	8.0	7.0			

Axis Option 3

* - Two Cards for Sections with Linearly Varying Stiffness

• - One Card for Sections with Constant Stiffness

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Axis Option 1					Axis Option 2					Axis Option 3					Axis Option 4				
From	To	$Q_x, q_{x,x'}$	$Q_y, q_{y,y'}$	$Q_z, q_{z,z'}$	From	To	$Q_x, q_{x,x'}$	$Q_y, q_{y,y'}$	$Q_z, q_{z,z'}$	From	To	$Q_x, q_{x,x'}$	$Q_y, q_{y,y'}$	$Q_z, q_{z,z'}$	From	To	$Q_x, q_{x,x'}$	$Q_y, q_{y,y'}$	$Q_z, q_{z,z'}$
0.0	10.0		-5.0		0.0	10.0	3.0	-4.0		0.0	8.0	5.0	-5.0		0.0	6.0	5.0	-5.0	
10.0		3.0	-5.0		10.0		5.4	-2.2		8.0		9.0	-2.75		6.0		9.0	-2.75	
	20.0	3.0	-10.0			20.0	8.4	-6.2			16.0	14.0	-7.75			12.0	14.0	-7.75	
24.15	24.15	8.0	12.0	-6.0	24.15	24.15	-0.8	14.4	-6.0	19.32	19.32	-0.8	14.4	-6.0	14.52	14.52	-0.8	14.4	-6.0
20.0			-10.0		20.0		6.0	-8.0		16.0		10.0	-10.0		12.0		10.0	-10.0	
	30.0		0.0			30.0	0.0	0.0			24.0	0.0	0.0			18.0	0.0	0.0	

- o - One Card for Concentrated Loads
- - One Card for Sections with Uniform Loads
- * - Two Cards for Sections with Linearly Varying Loads

Notes:

There is No Restriction on the Length of a Section Except that it Must Exceed $\frac{1}{40}$ of the Member's Length.

'From' and 'To' Joints Set by Input in Table 3.

Sections Need Not be Input in Order.

Concentrated Loads May Not be Input at a Distance of 0.0.

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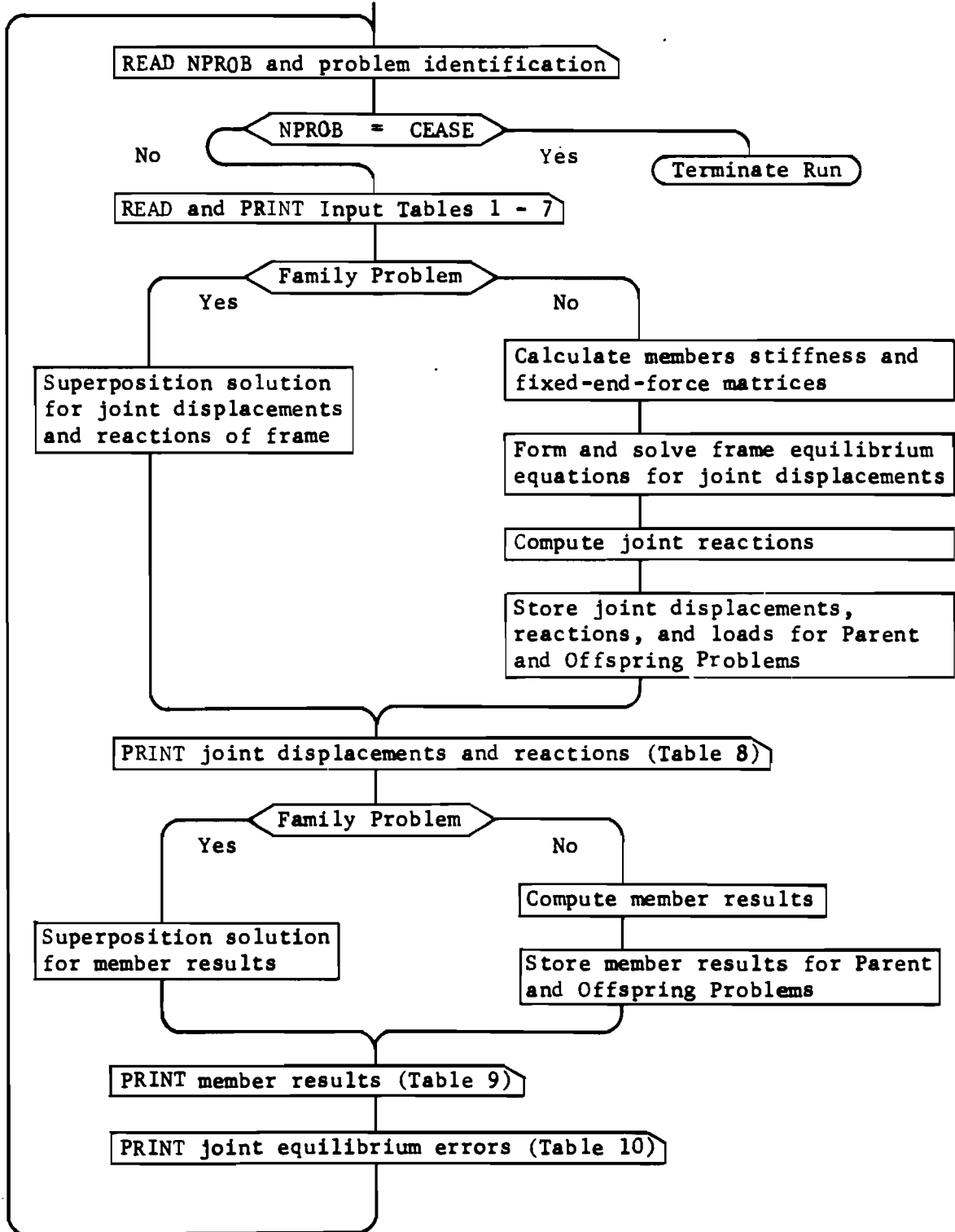
APPENDIX 3

PROGRAM FLOW CHARTS

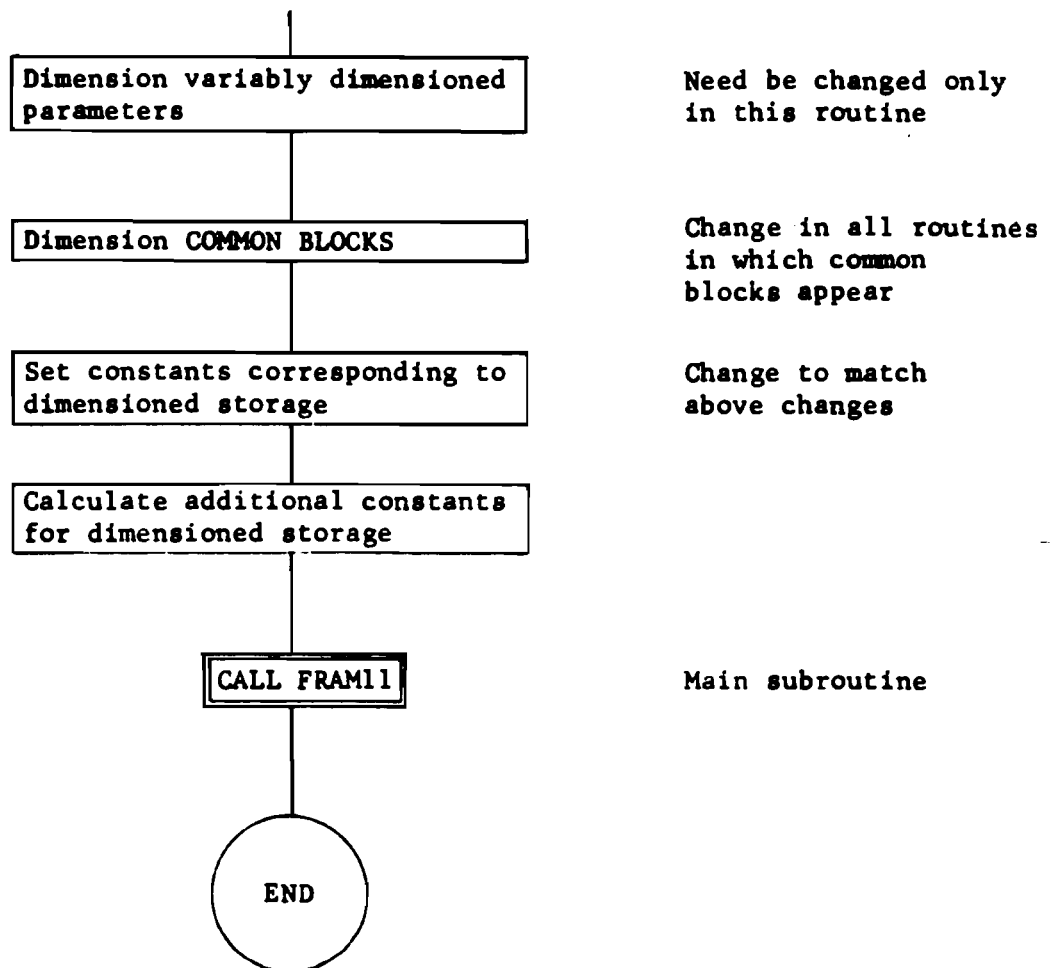
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SUMMARY FLOW DIAGRAM FOR FRAME 11

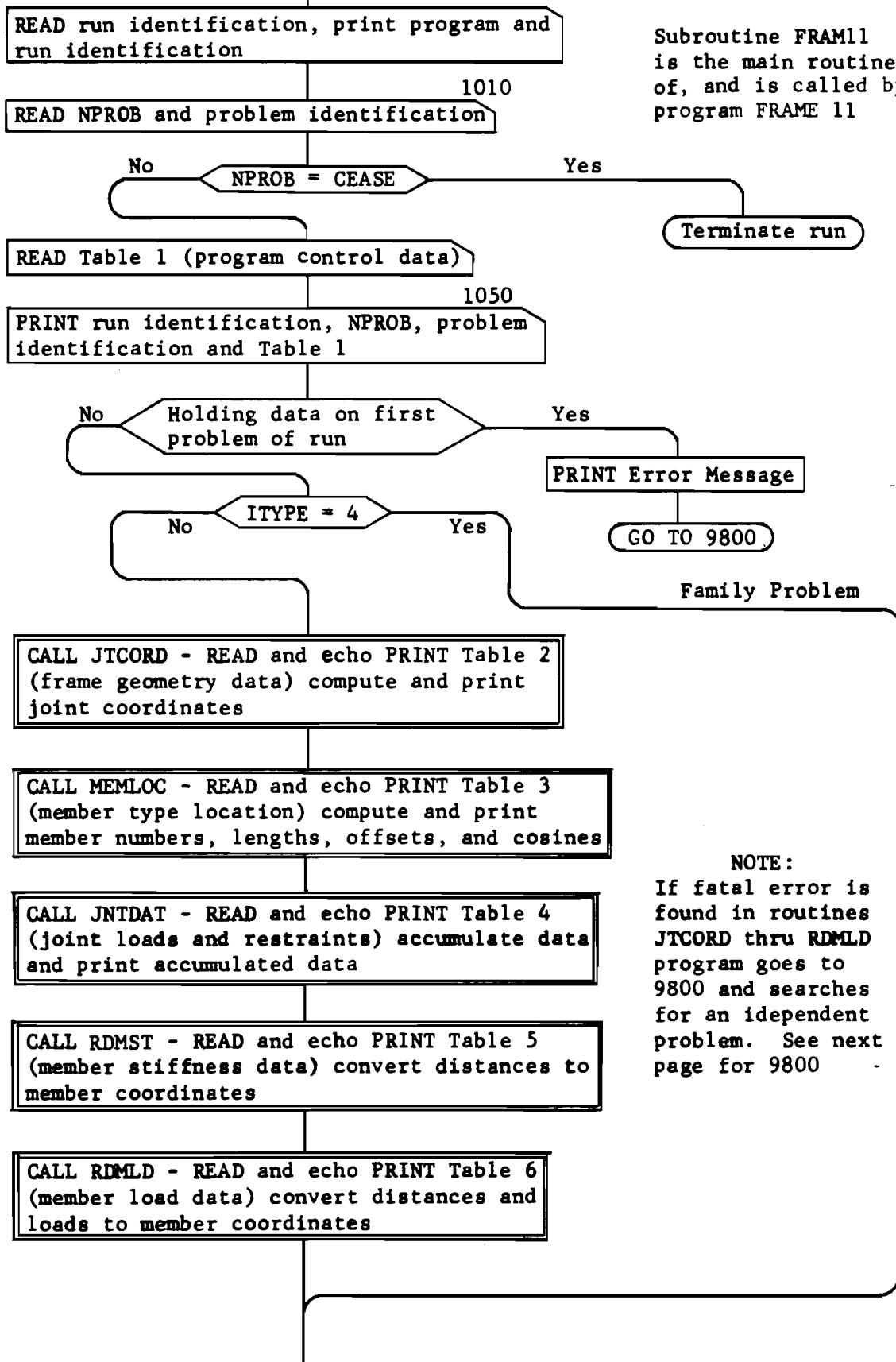


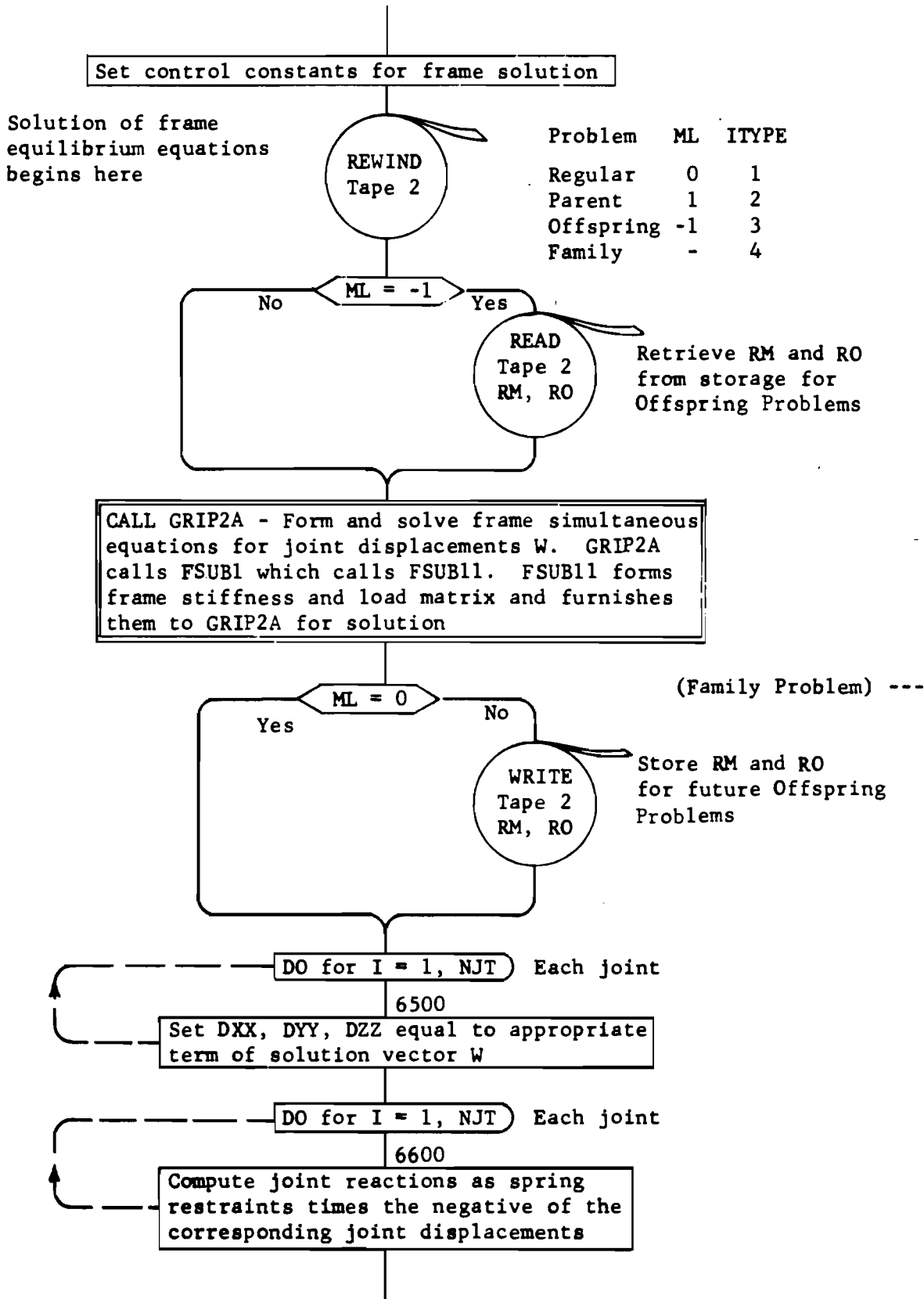
FLOW DIAGRAM FOR FRAME 11

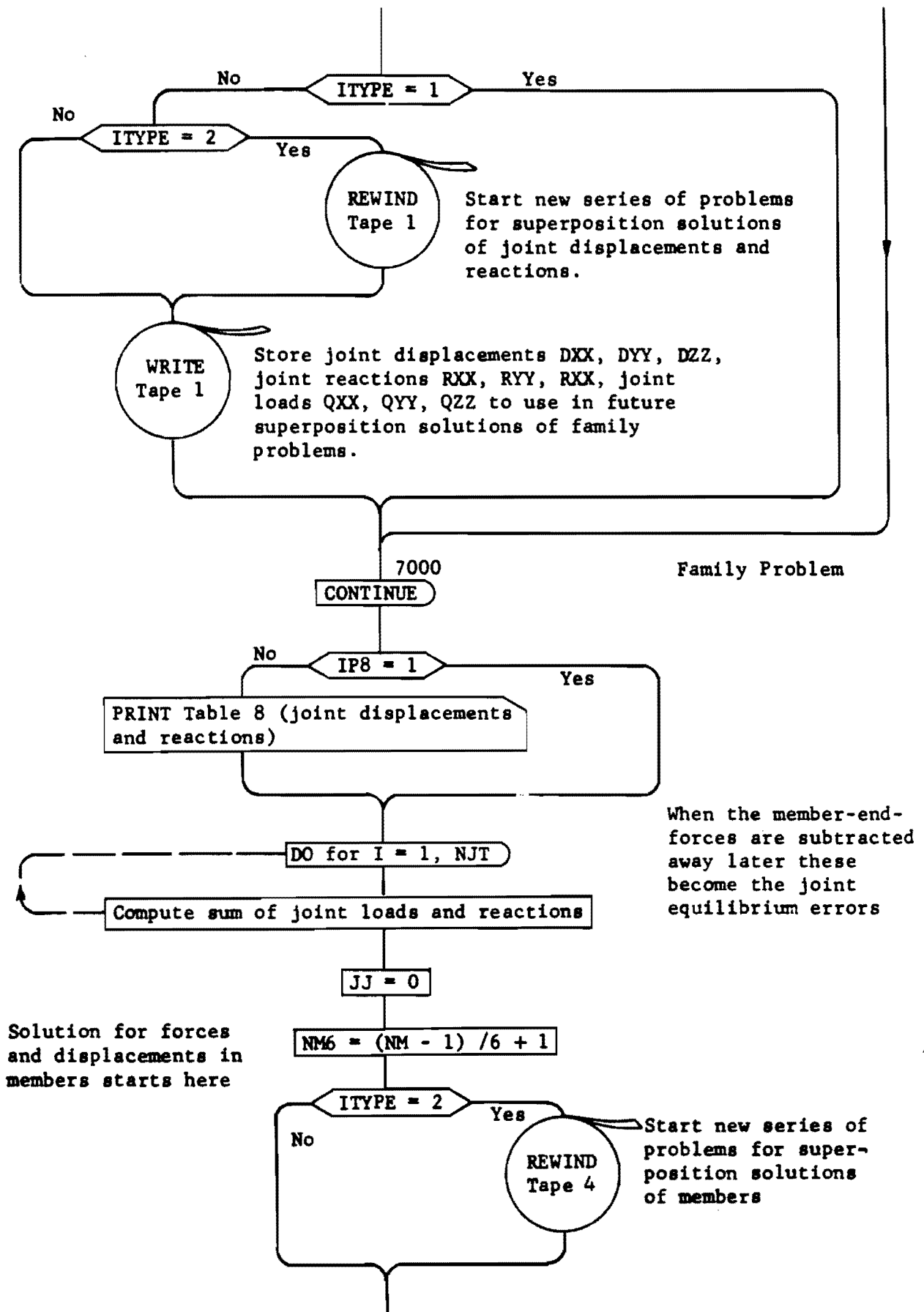


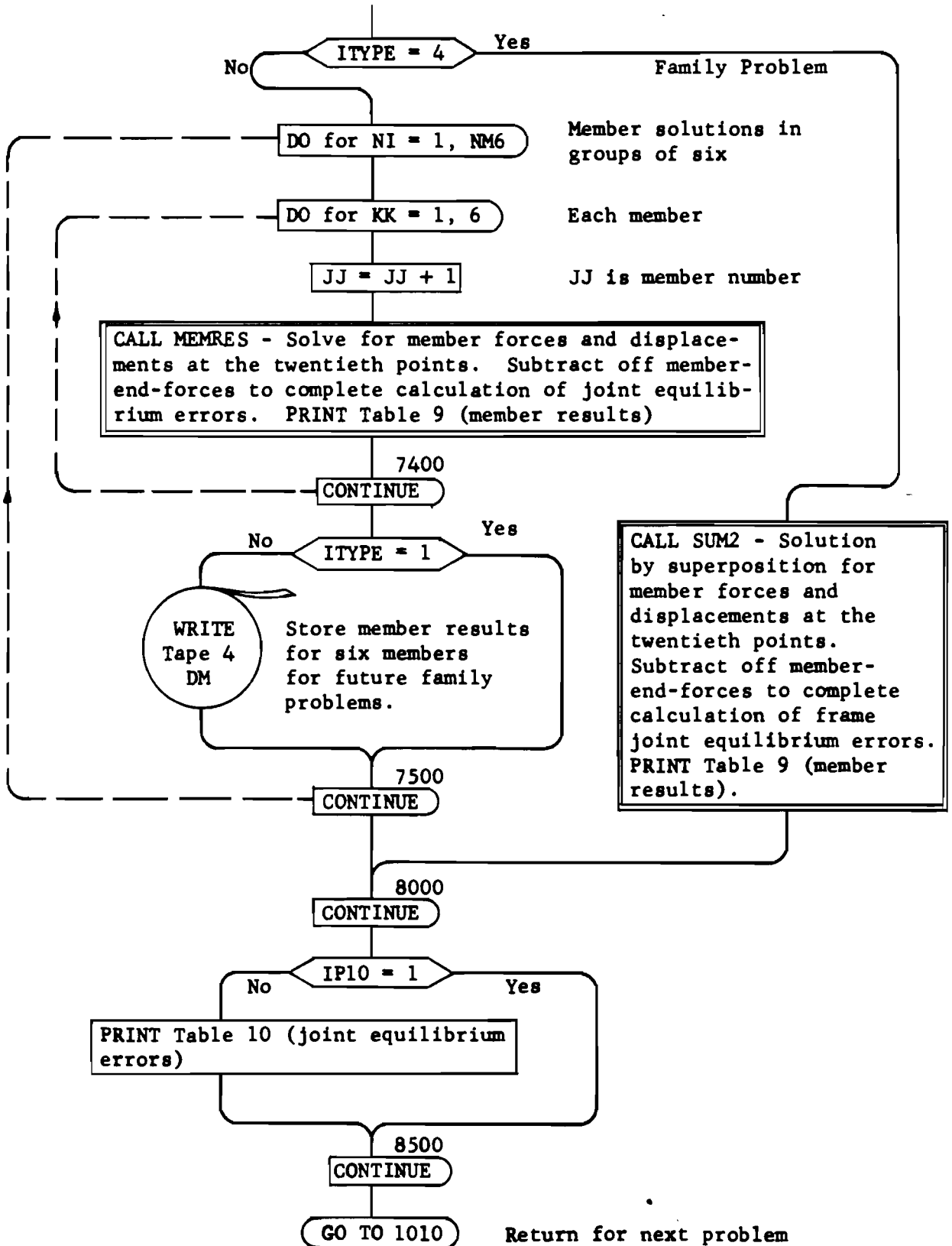
Program dimensions may be easily changed as indicated above. See the program listing and the notation.

FLOW DIAGRAM FOR SUBROUTINE FRAM11

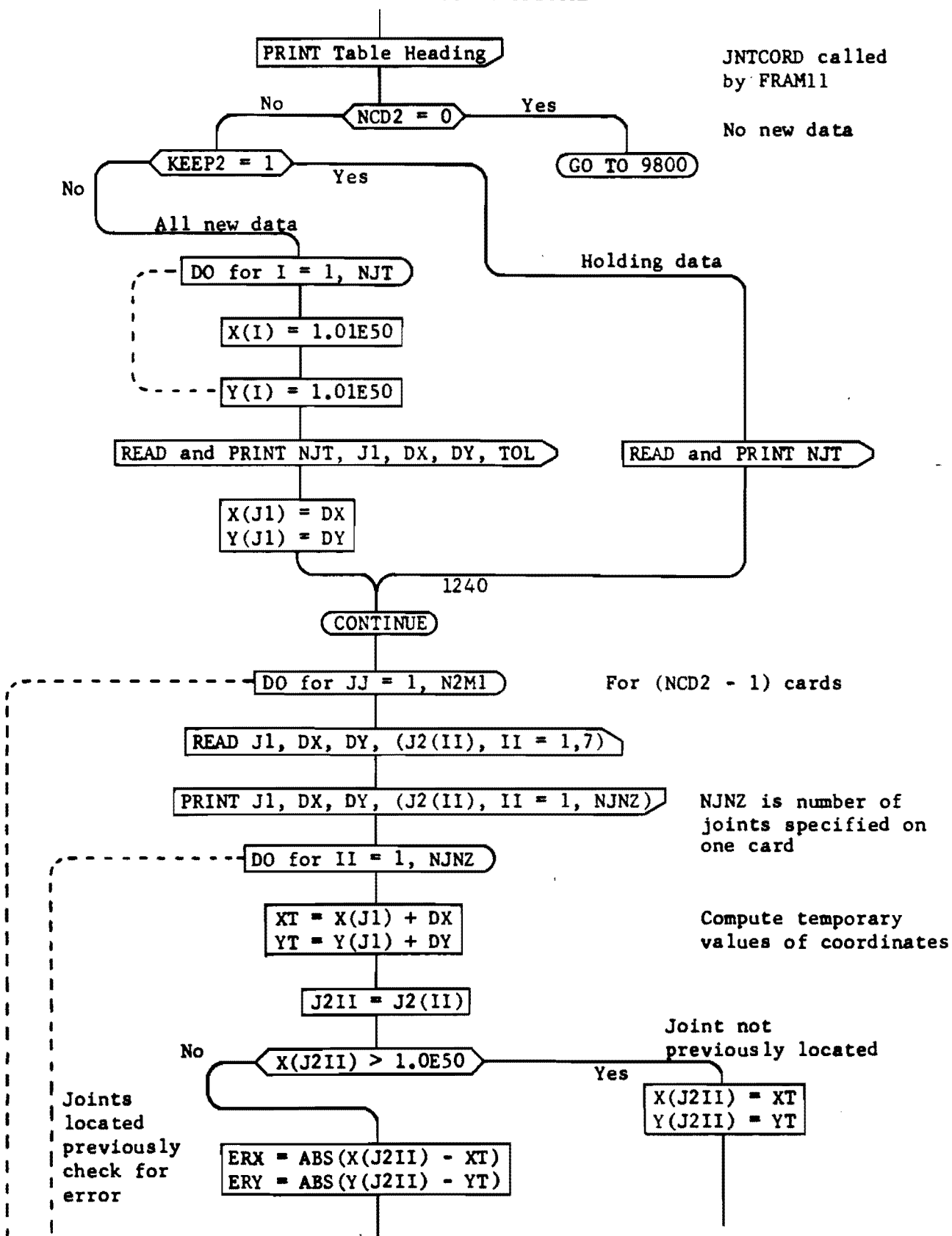


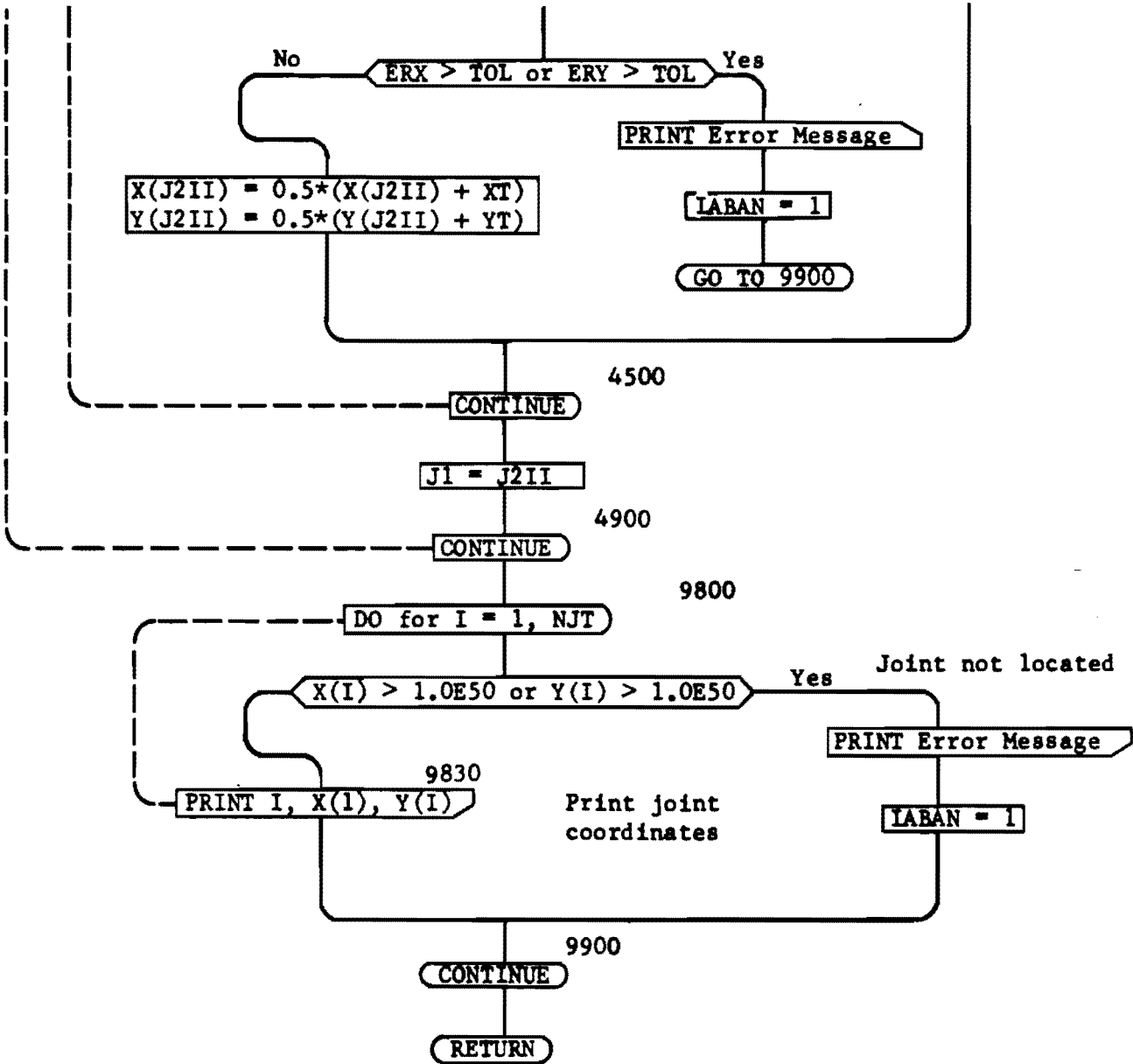






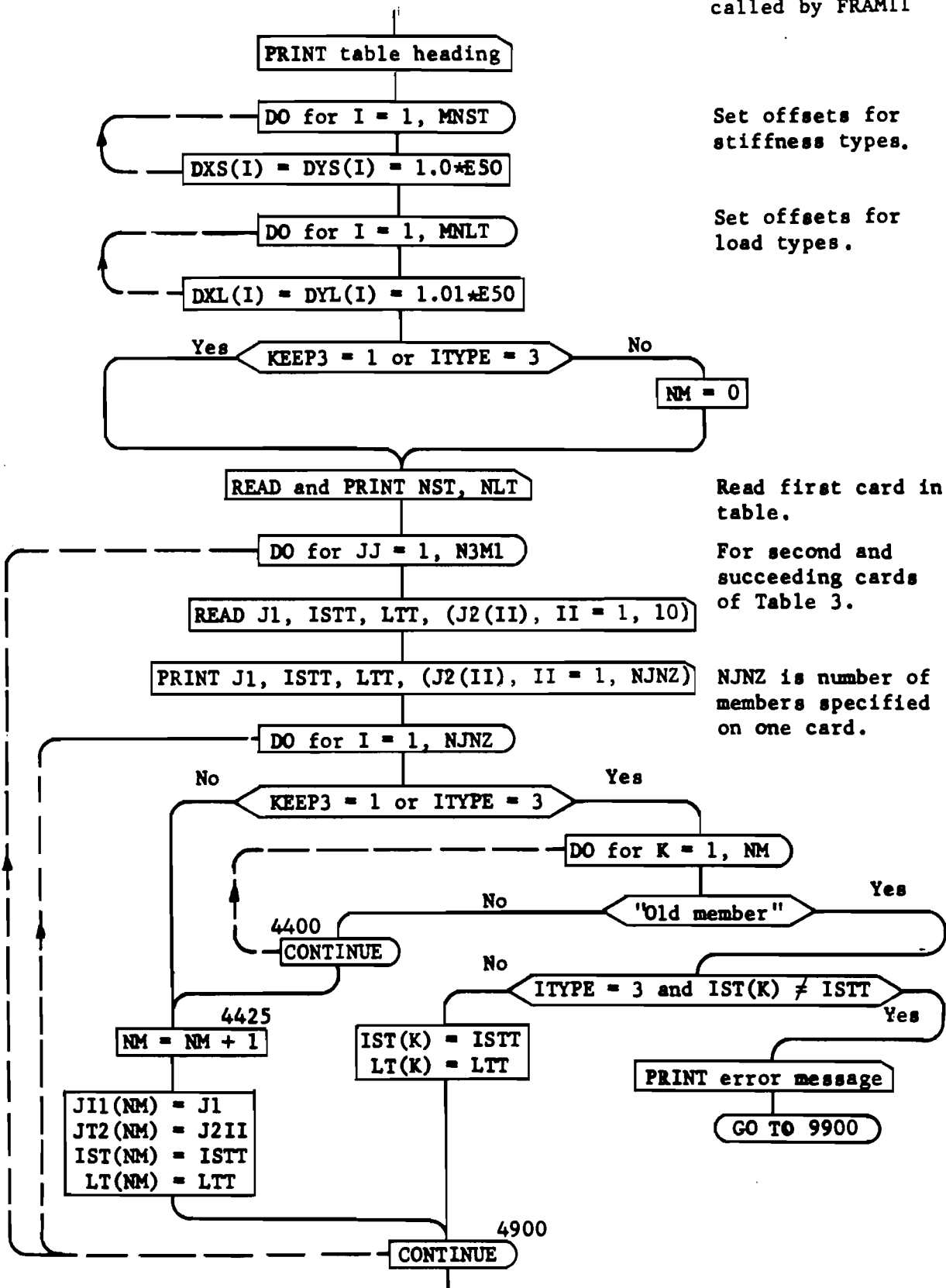
FLOW DIAGRAM FOR SUBROUTINE JNTCORD





SUBROUTINE MEMLOC

Subroutine MEMLOC
called by FRAM11



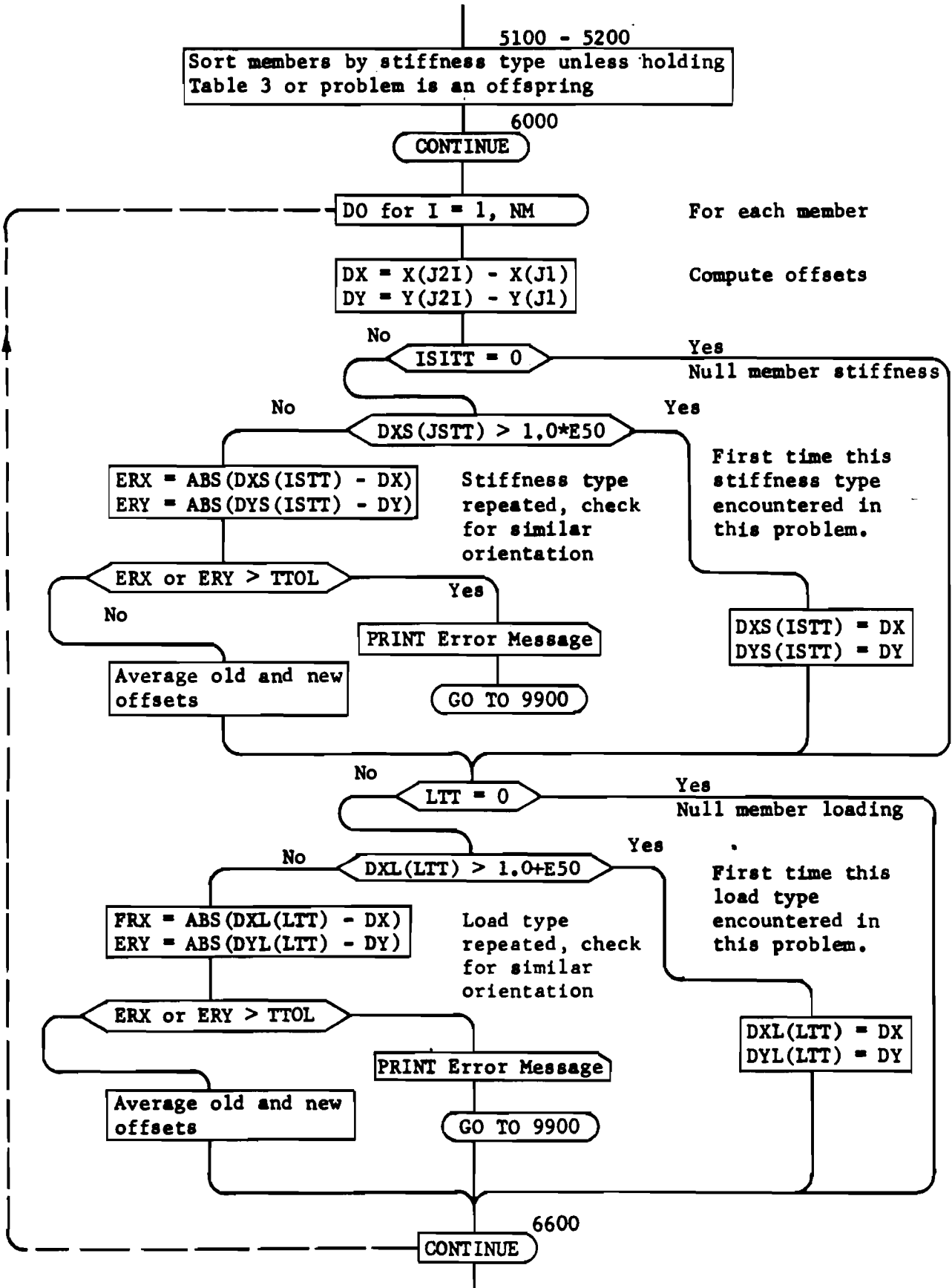
Set offsets for
stiffness types.

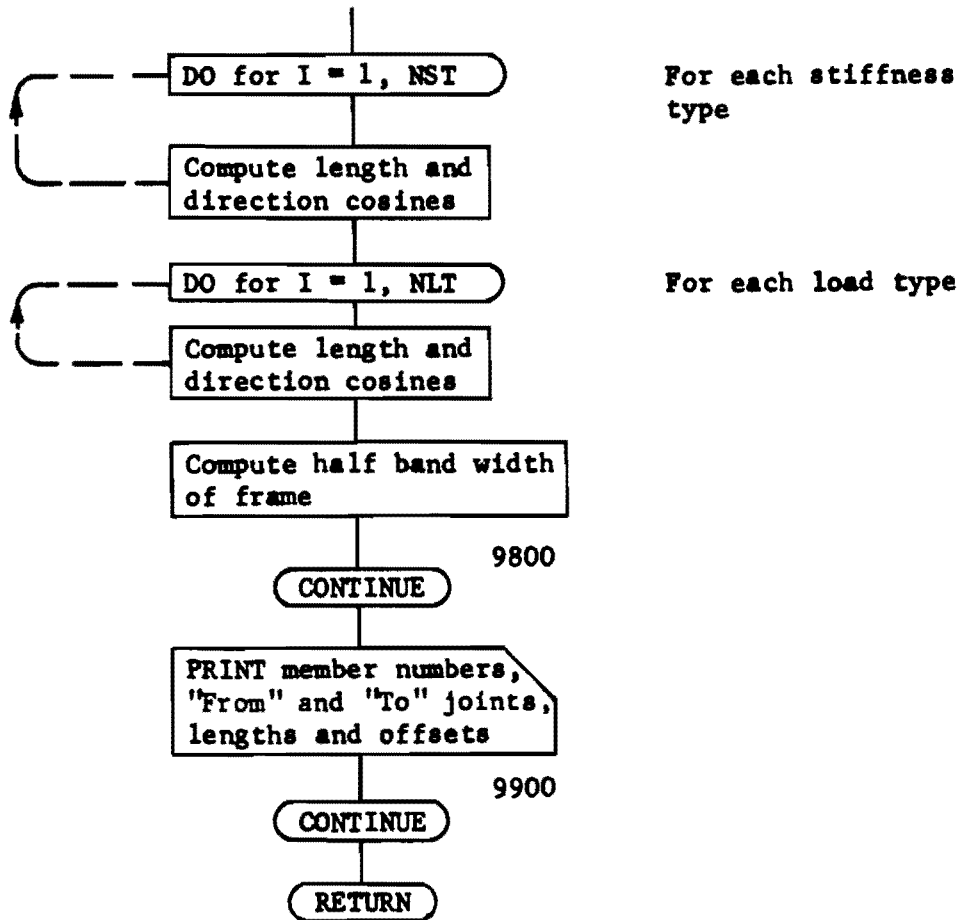
Set offsets for
load types.

Read first card in
table.

For second and
succeeding cards
of Table 3.

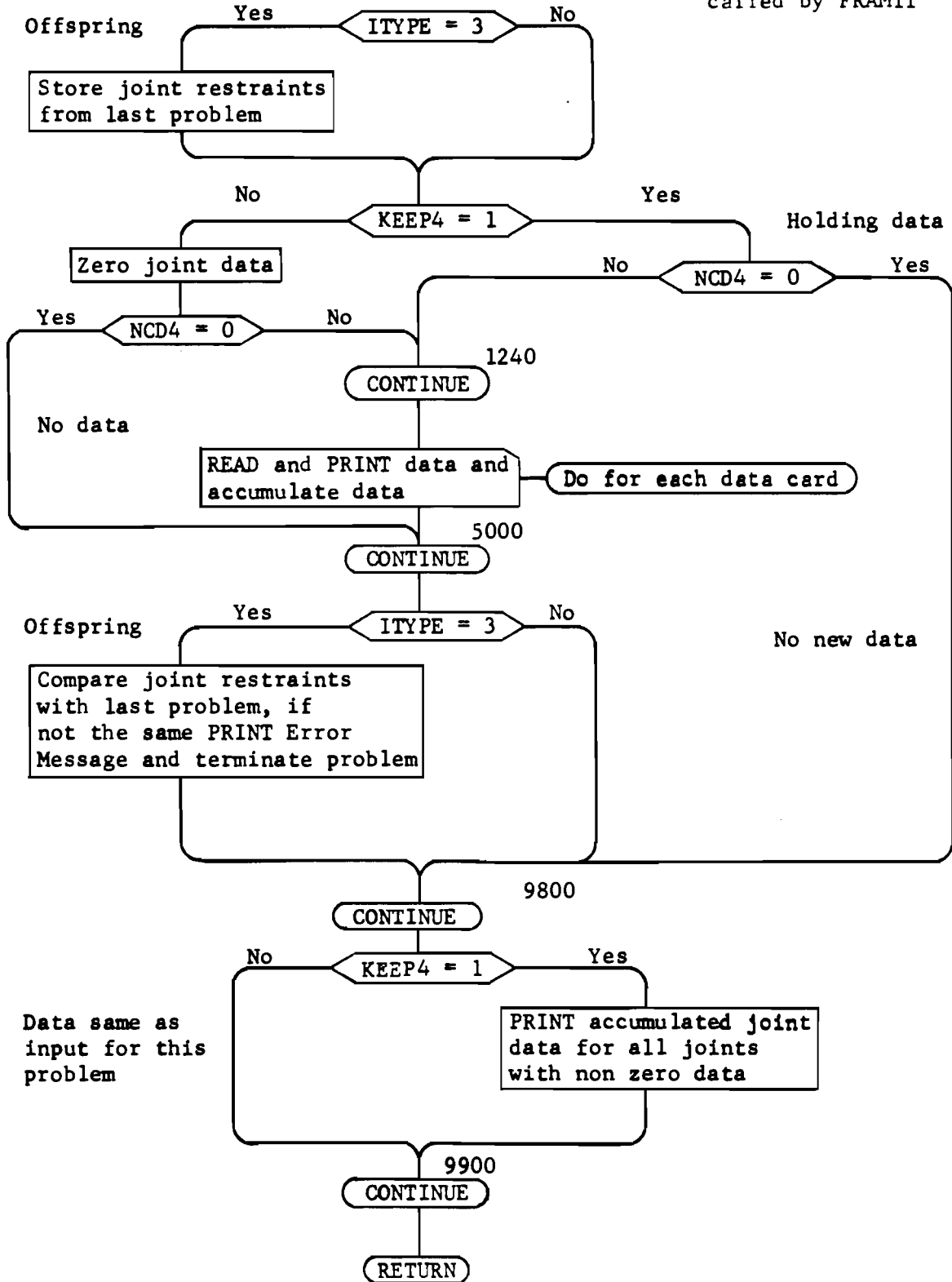
NJNZ is number of
members specified
on one card.





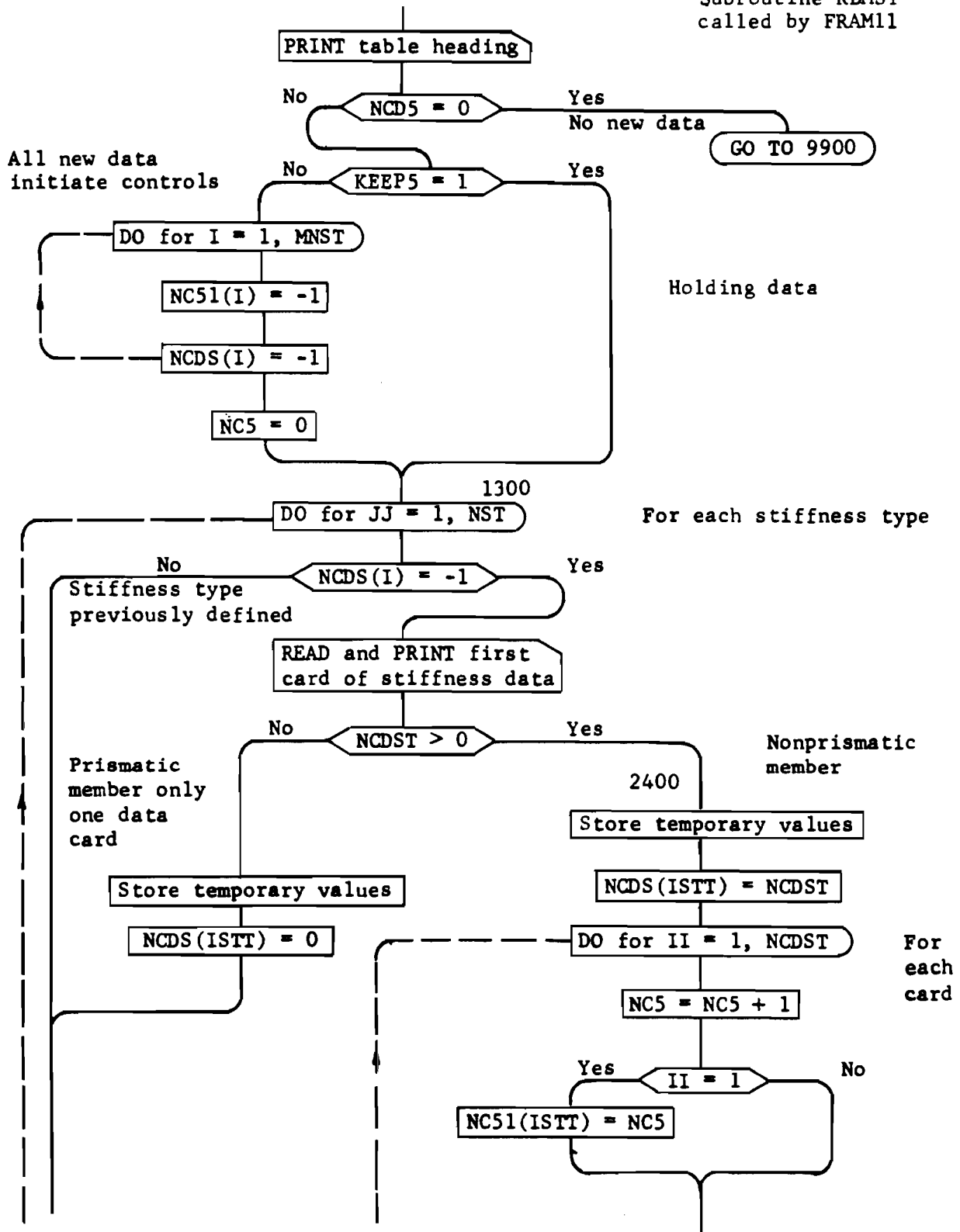
SUBROUTINE JNTDAT

Subroutine JNTDAT called by FRAM11



SUBROUTINE RDMST

Subroutine RDMST
called by FRAM11



All new data
initiate controls

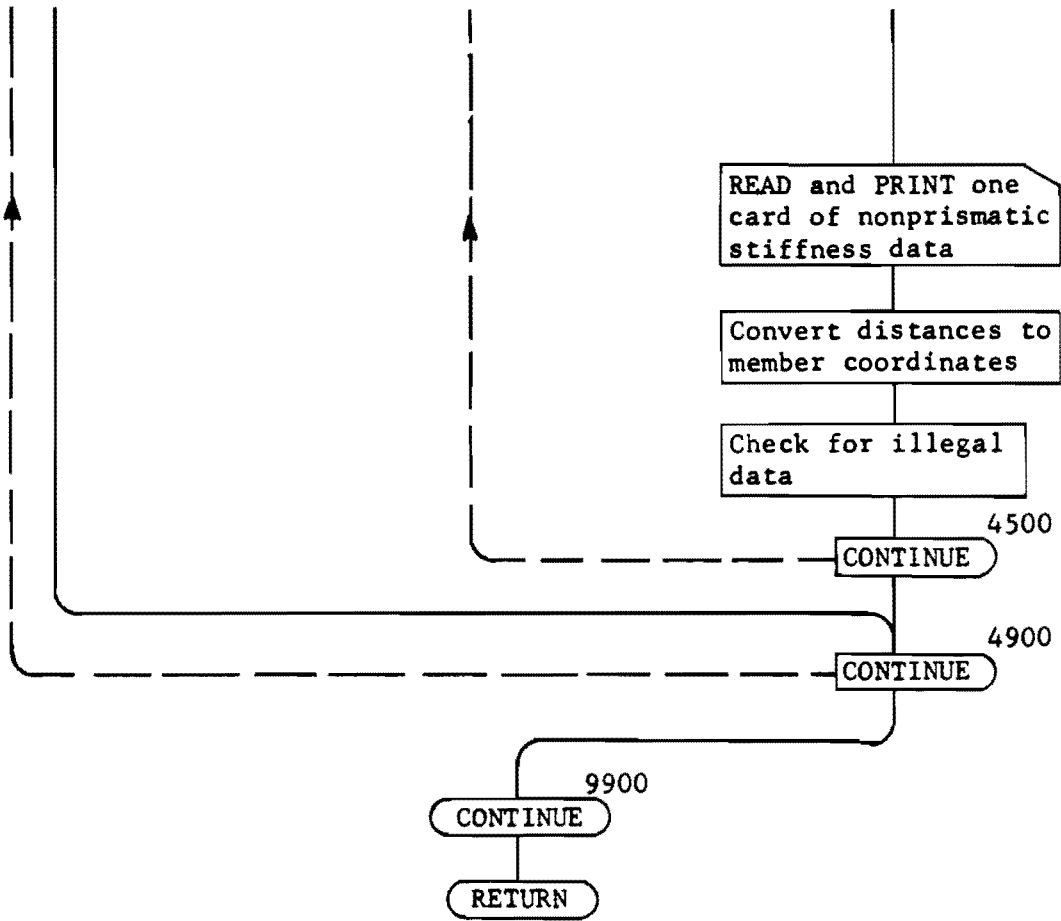
Holding data

For each stiffness type

Prismatic
member only
one data
card

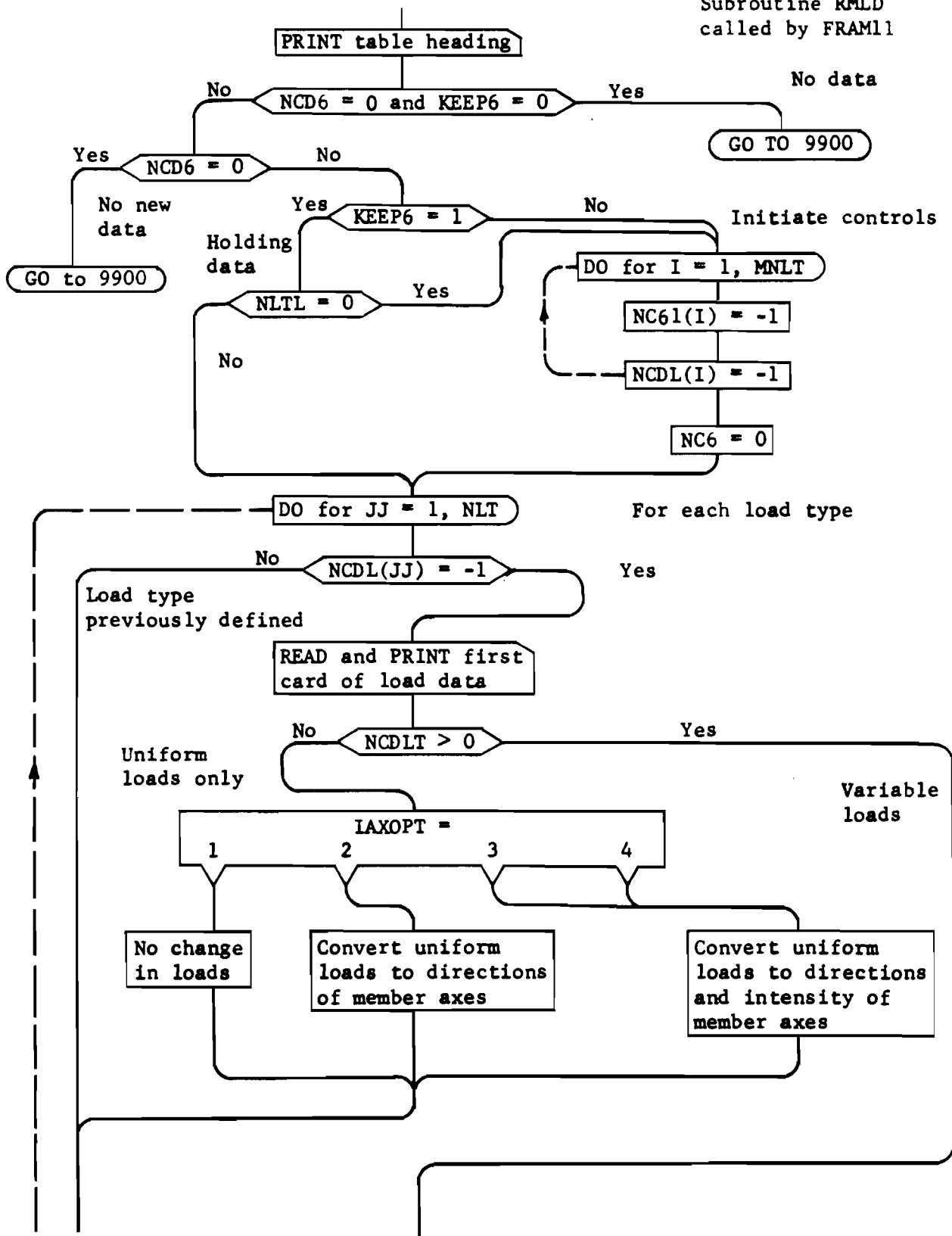
Nonprismatic
member

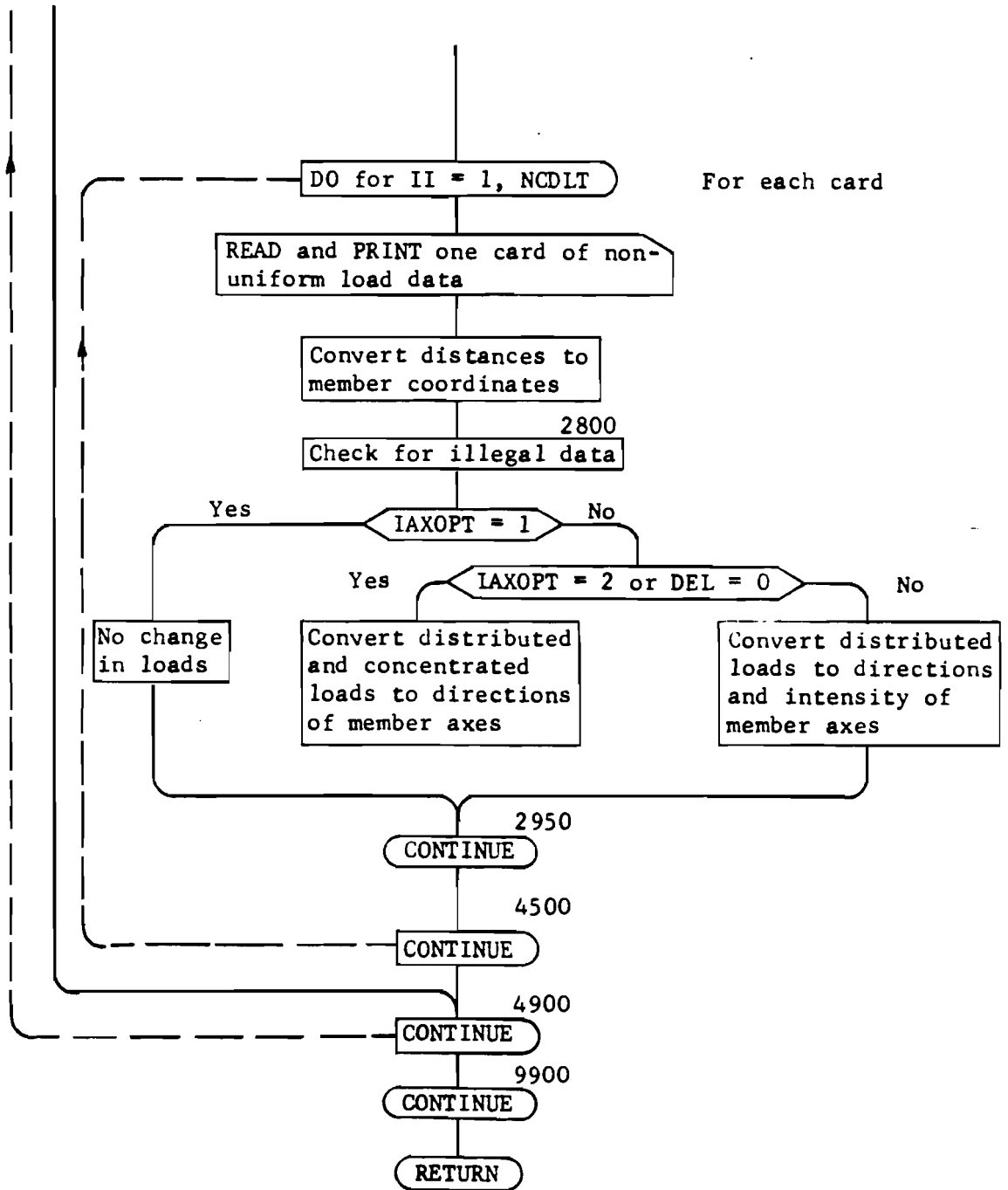
For
each
card



SUBROUTINE RDMLD

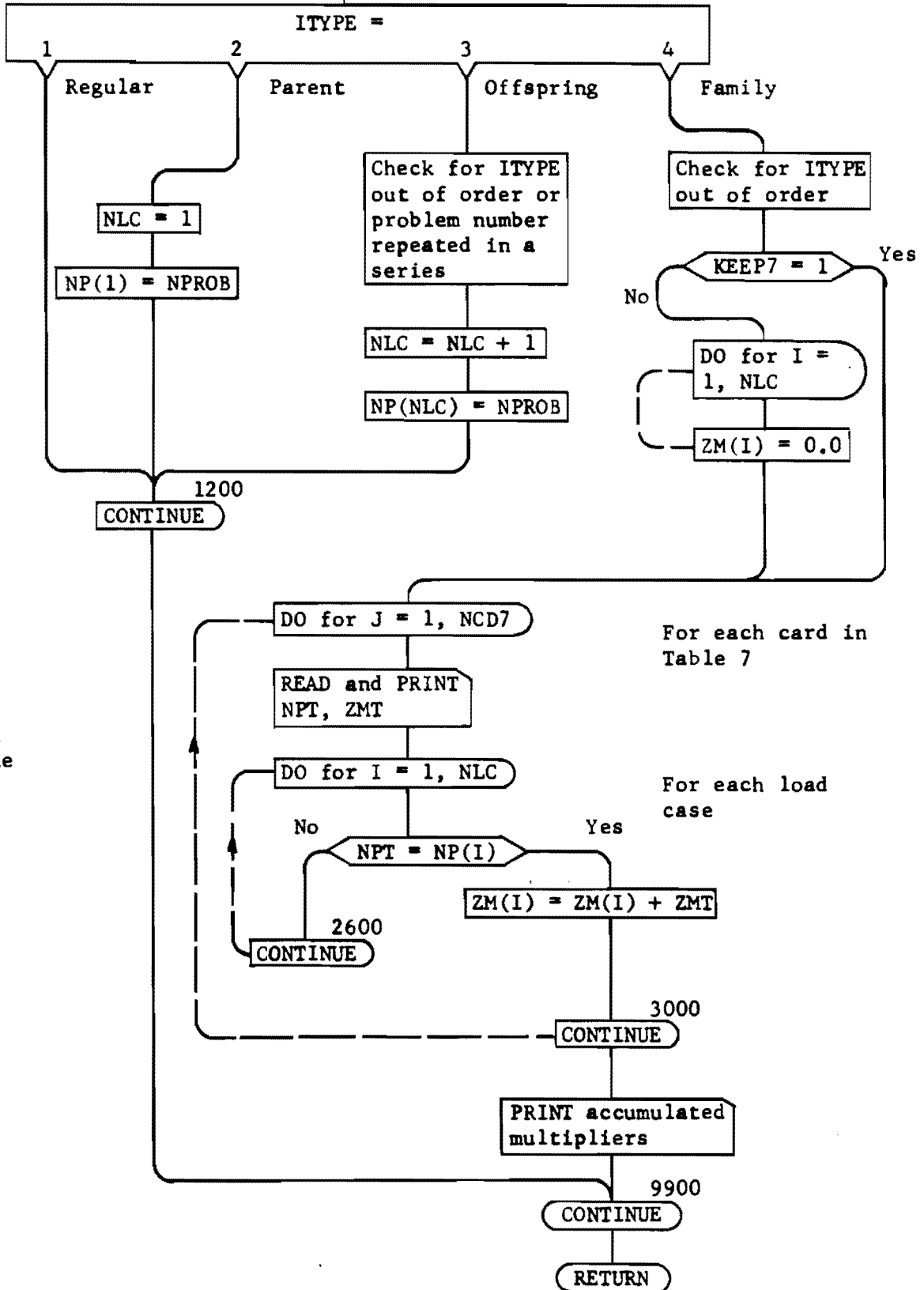
Subroutine RMLD
called by FRAM11





SUBROUTINE COMP

Subroutine COMP
called by FRAM11



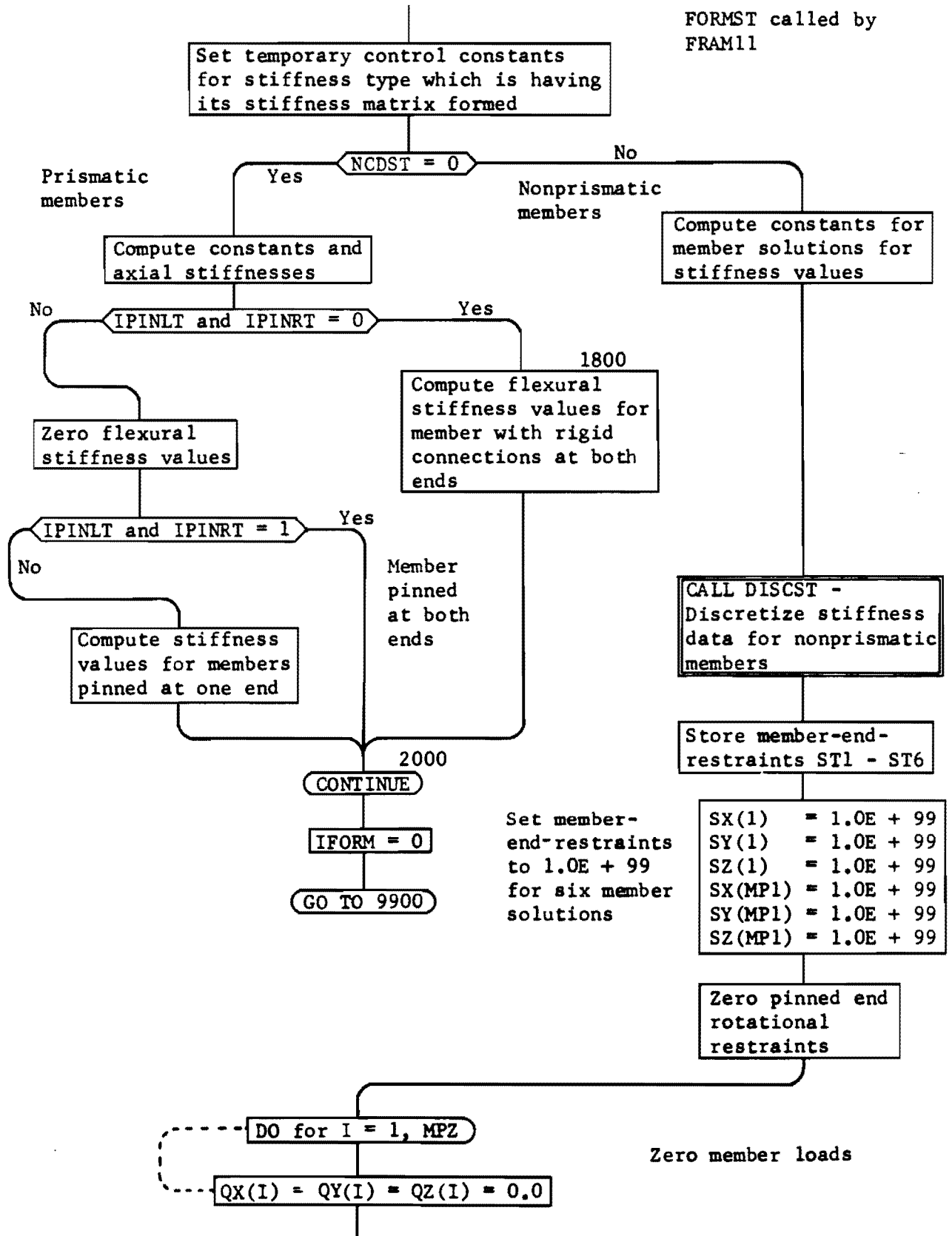
No data
in Table

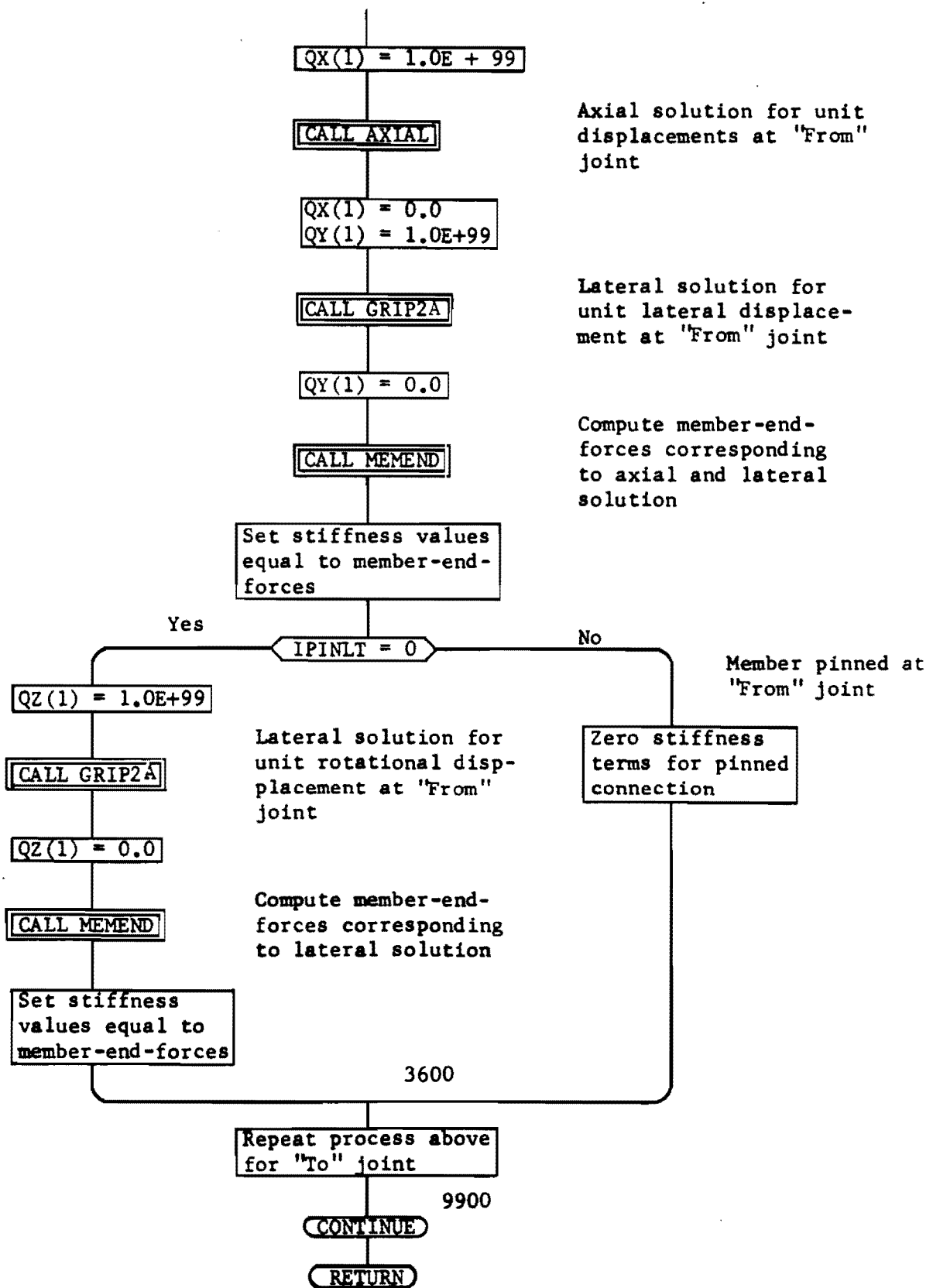
For each card in
Table 7

For each load
case

SUBROUTINE FORMST

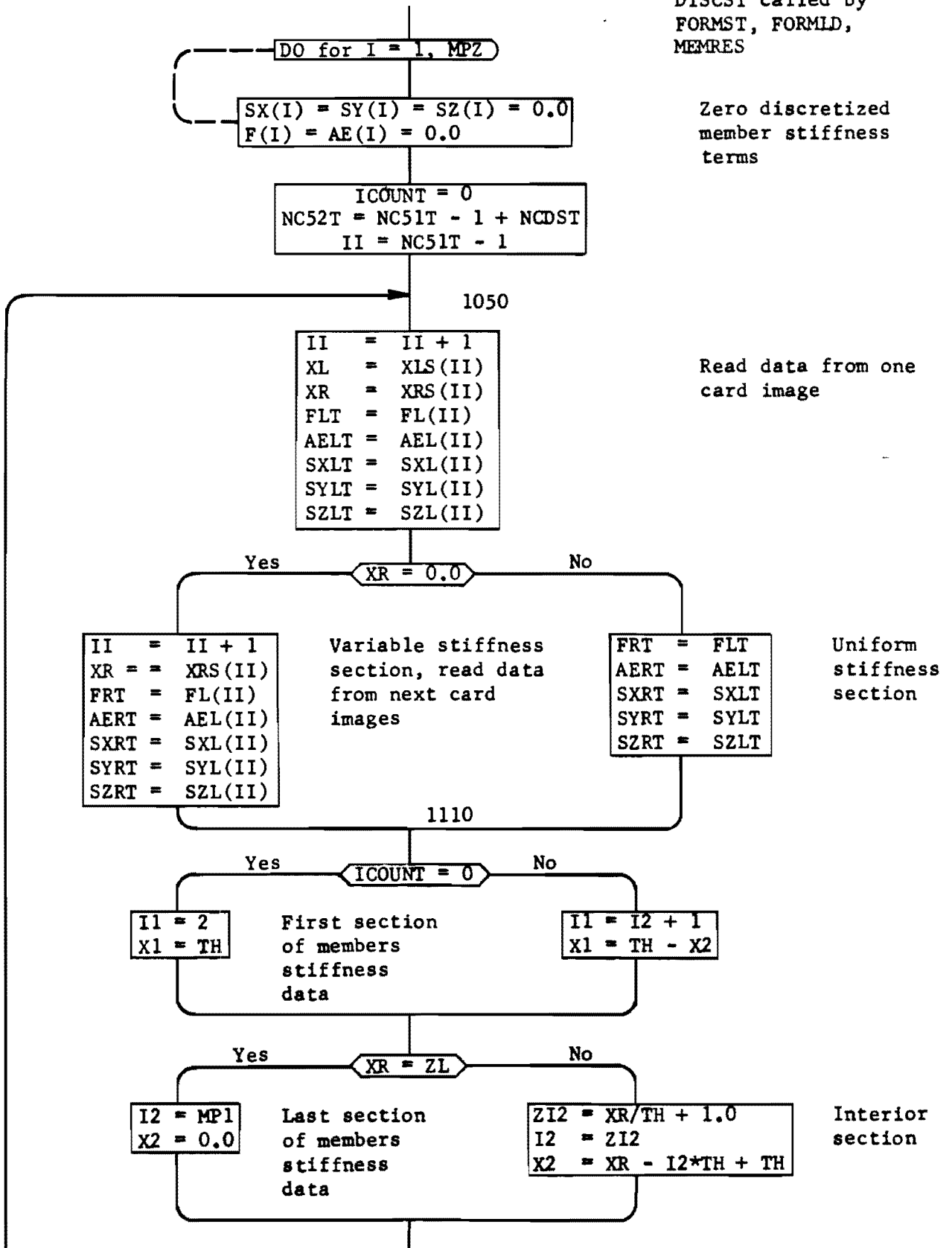
FORMST called by
FRAM11

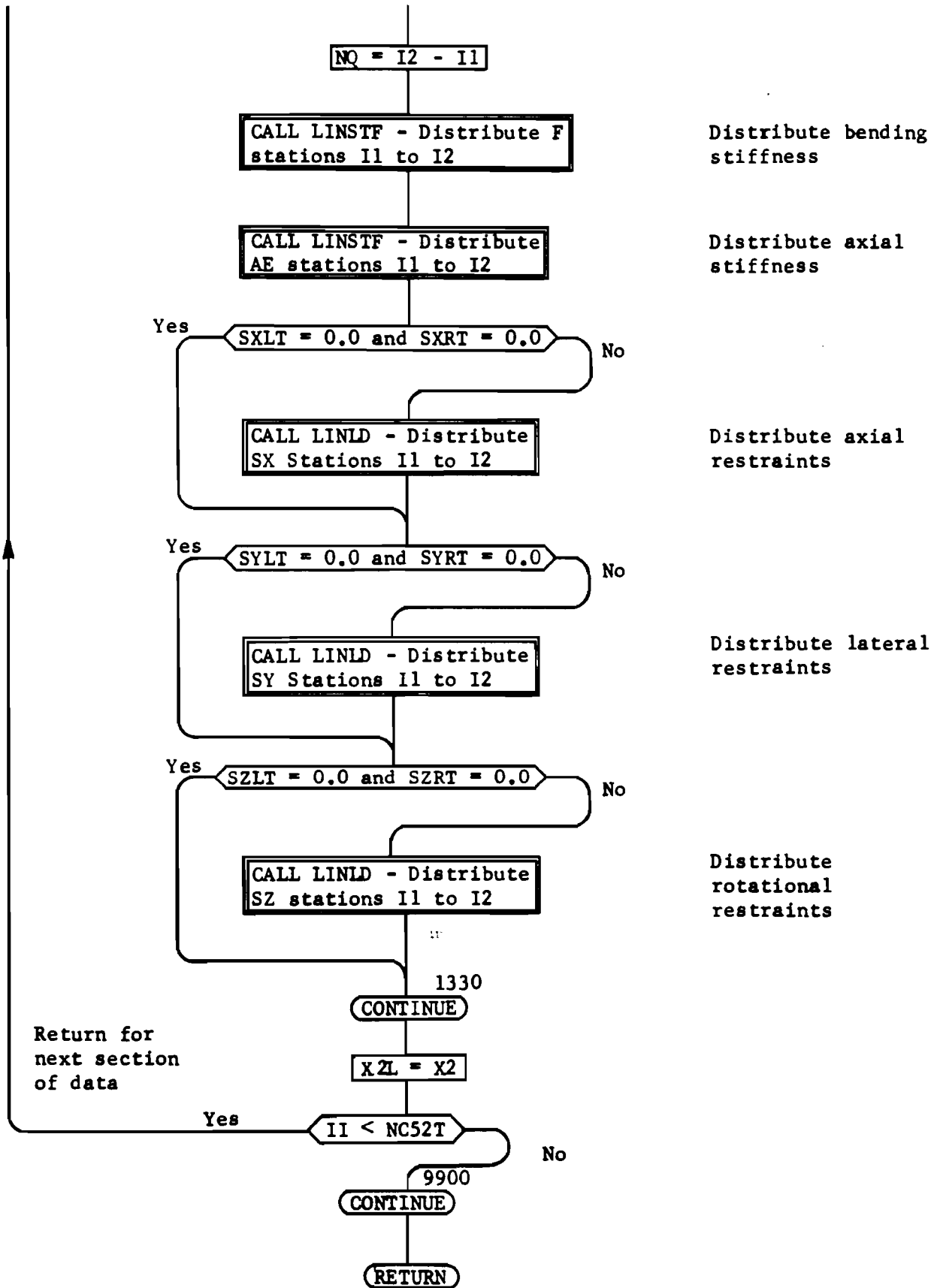




SUBROUTINE DISCST

DISCST called by
FORMST, FORMLD,
MEMRES



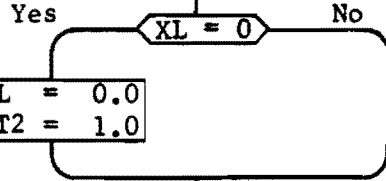


SUBROUTINE LINSTF

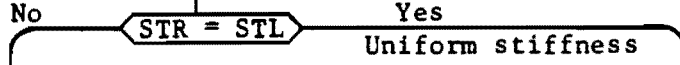
LINSTF called by
DISCST

First section
of members
stiffness
variation

X2L = 0.0
STT2 = 1.0



Linear variation
of stiffness



Compute slope of stiffness
variation (DS)

Compute effective stiffness
of first element in section
considering jump in
stiffness

Compute effective stiffness of
first element in section
considering jump in stiffness

I1P1 = I1 + 1
I1PNQ = I1 + NQ

I1P1 = I1 + 1
I1PNQ = I1 + NQ

DO for I = I1P1, I1PNQ

DO for I = I1P1, I1PNQ

Set stiffness equal to
uniform value

Compute stiffness at
midpoint of remaining
NQ elements

1350

CONTINUE

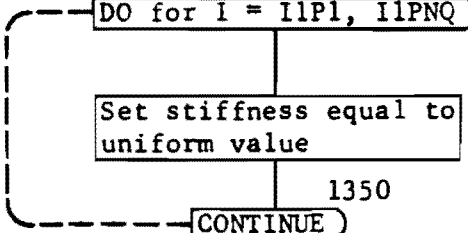
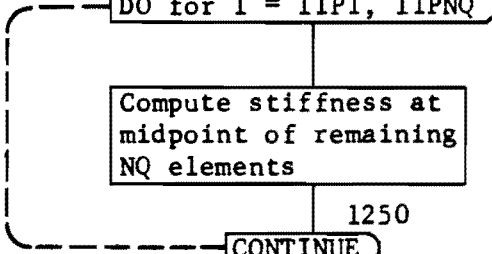
1250

CONTINUE

1800

CONTINUE

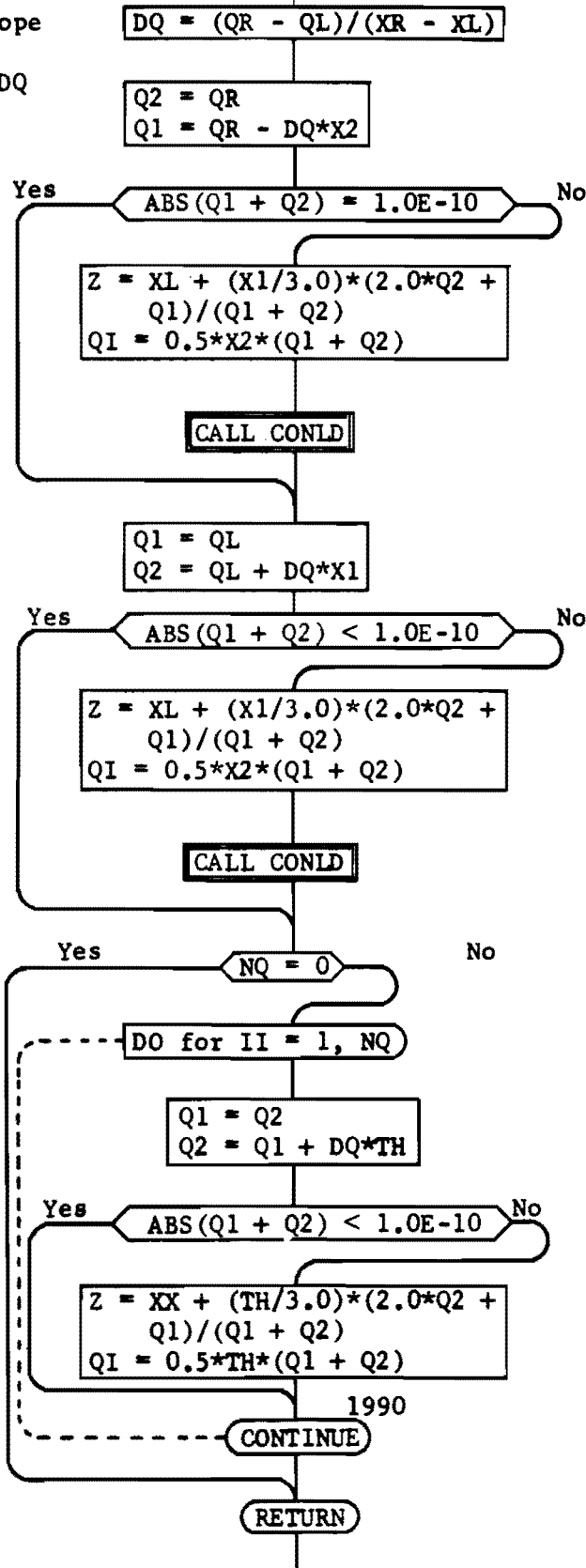
RETURN



SUBROUTINE LINLD

Compute slope
of linear
variation DQ

LINLD called by
DISCLD, DISCST



Compute concentrated
load or restraint
for element at right
end of section QI,
distance to line of
action Z and call
CONLD to distribute
to adjacent stations

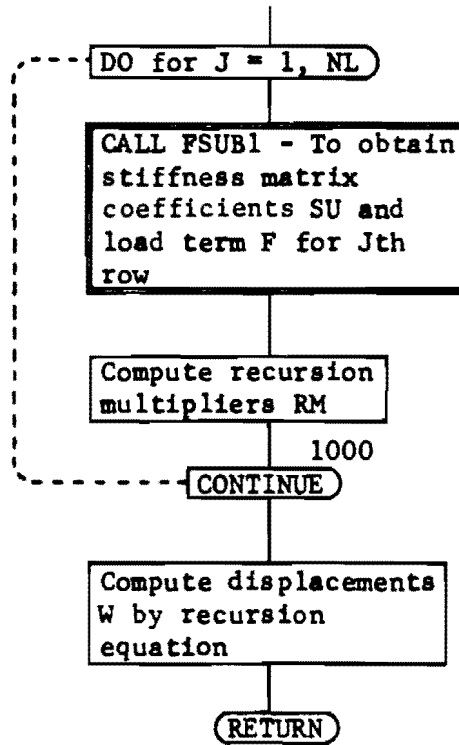
Same as above for
element at left end
of section

Same as above for
remaining NQ
elements

(XX is distance to
left of element
from the "From"
joint)

SUBROUTINE GRIP2A

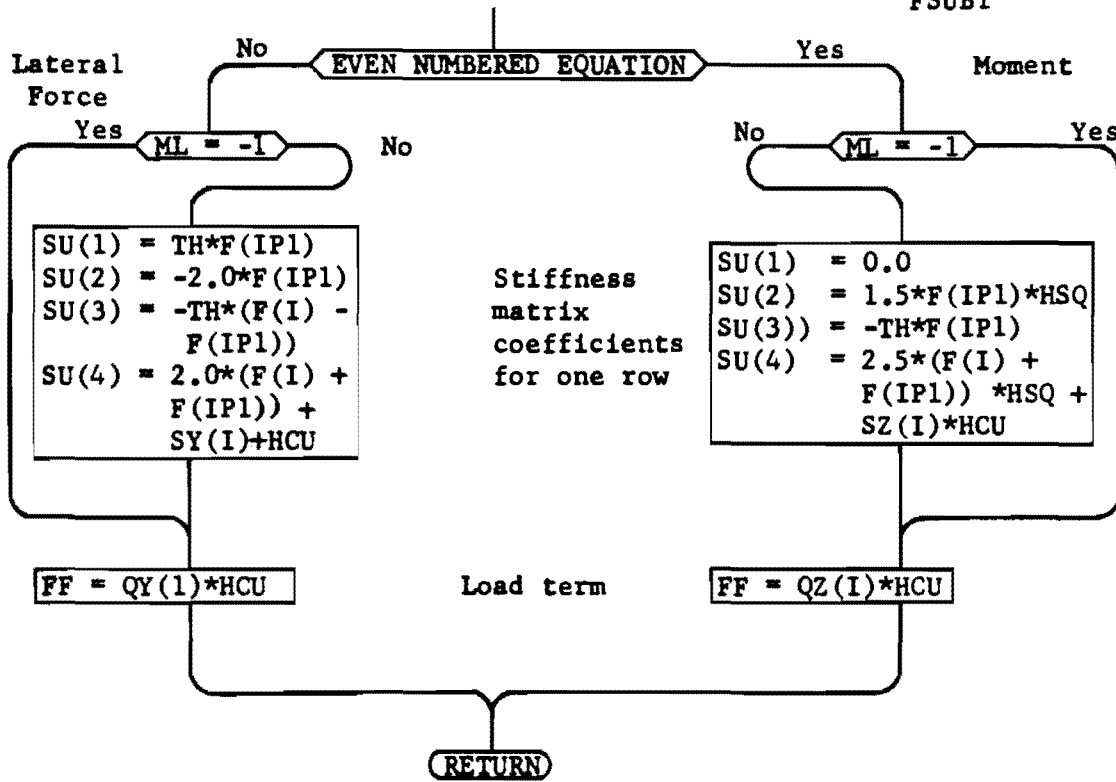
GRIP2A called by
FRAM11, FORMST,
FORMLD, MEMRES



FSUB1 calls FSUB11 to furnish SU and F for frame solution or FSUB12 to furnish SU and F for member solutions

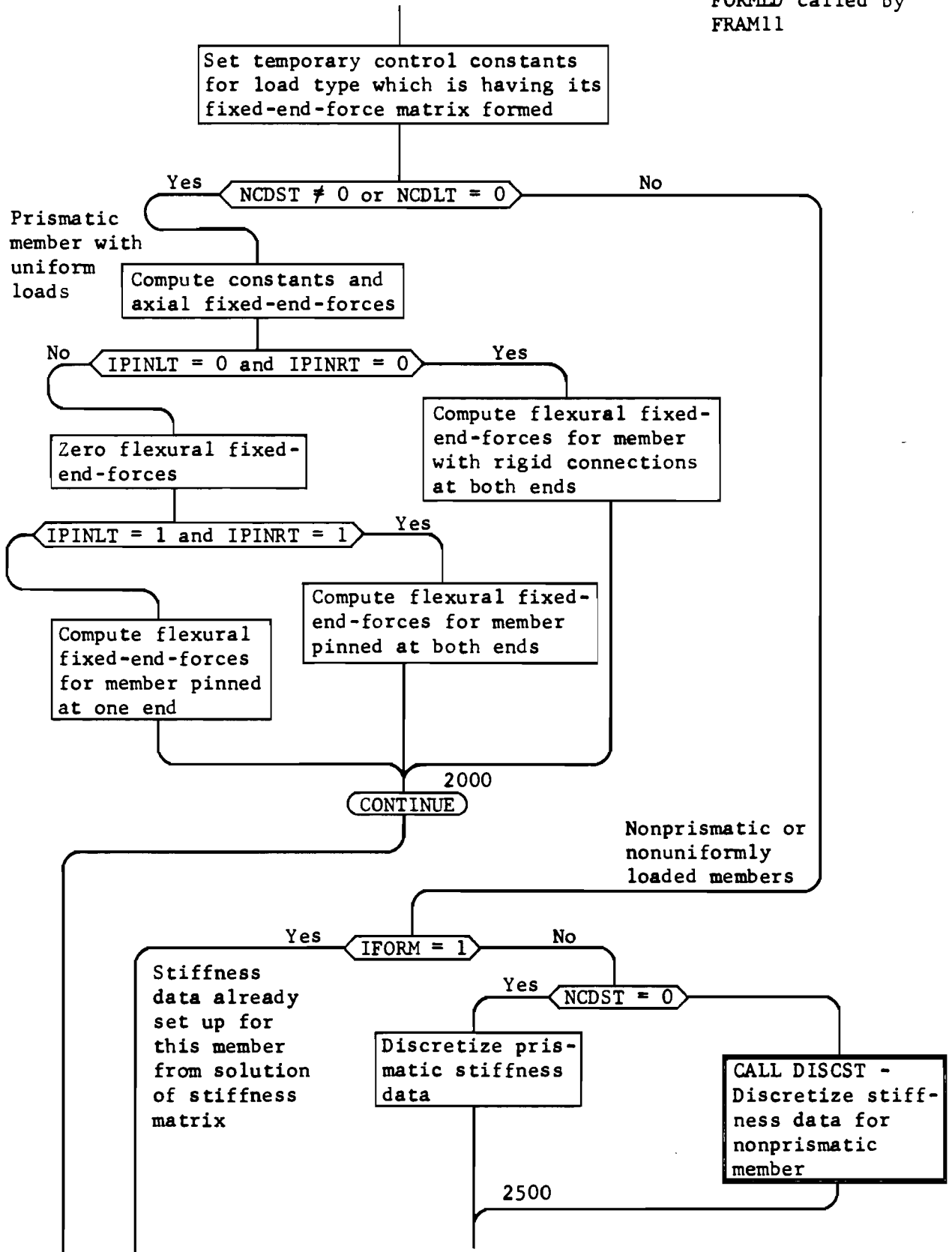
SUBROUTINE FSUB12

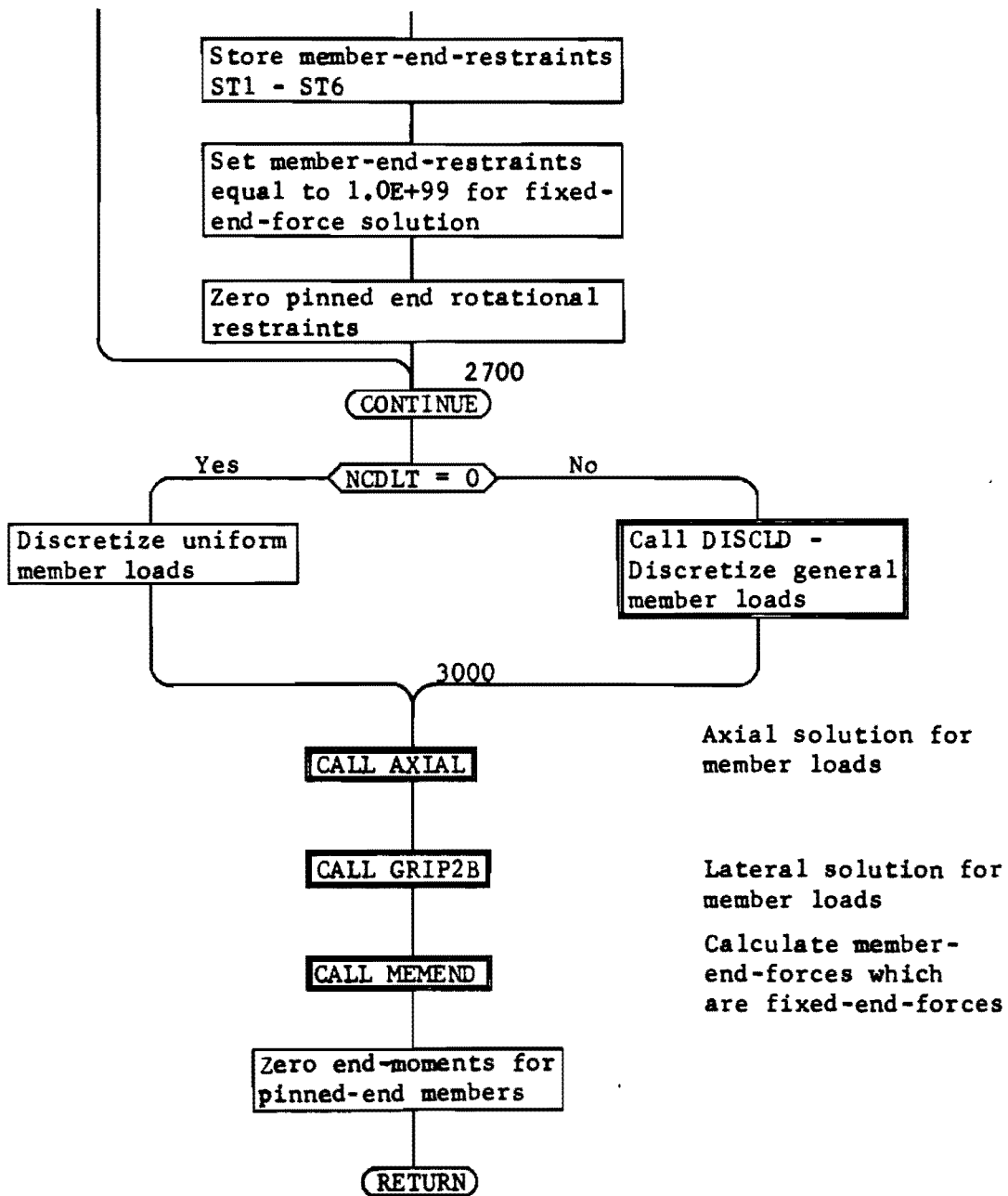
FSUB12 called by
FSUB1



SUBROUTINE FORMLD

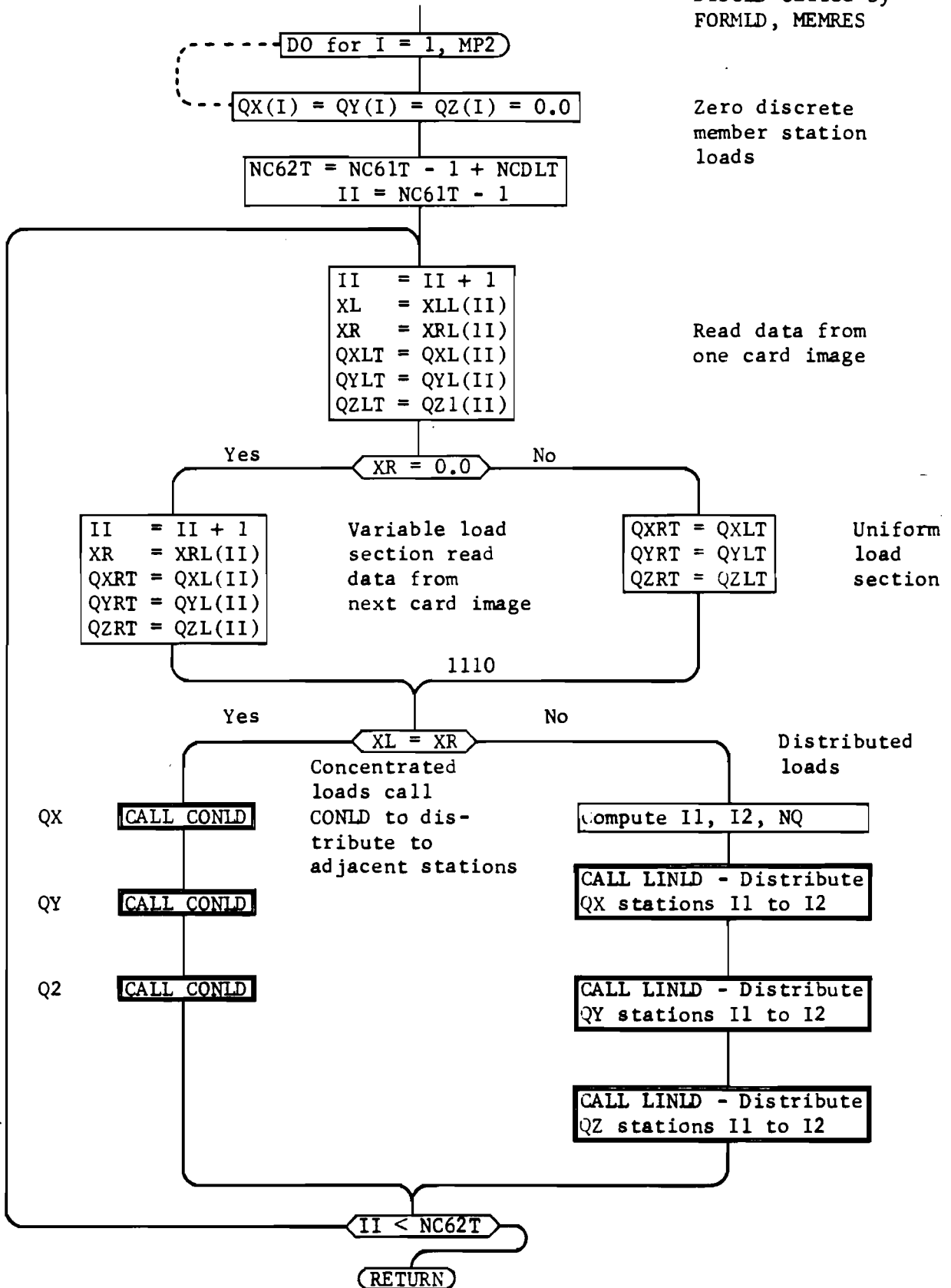
FORMLD called by
FRAM11





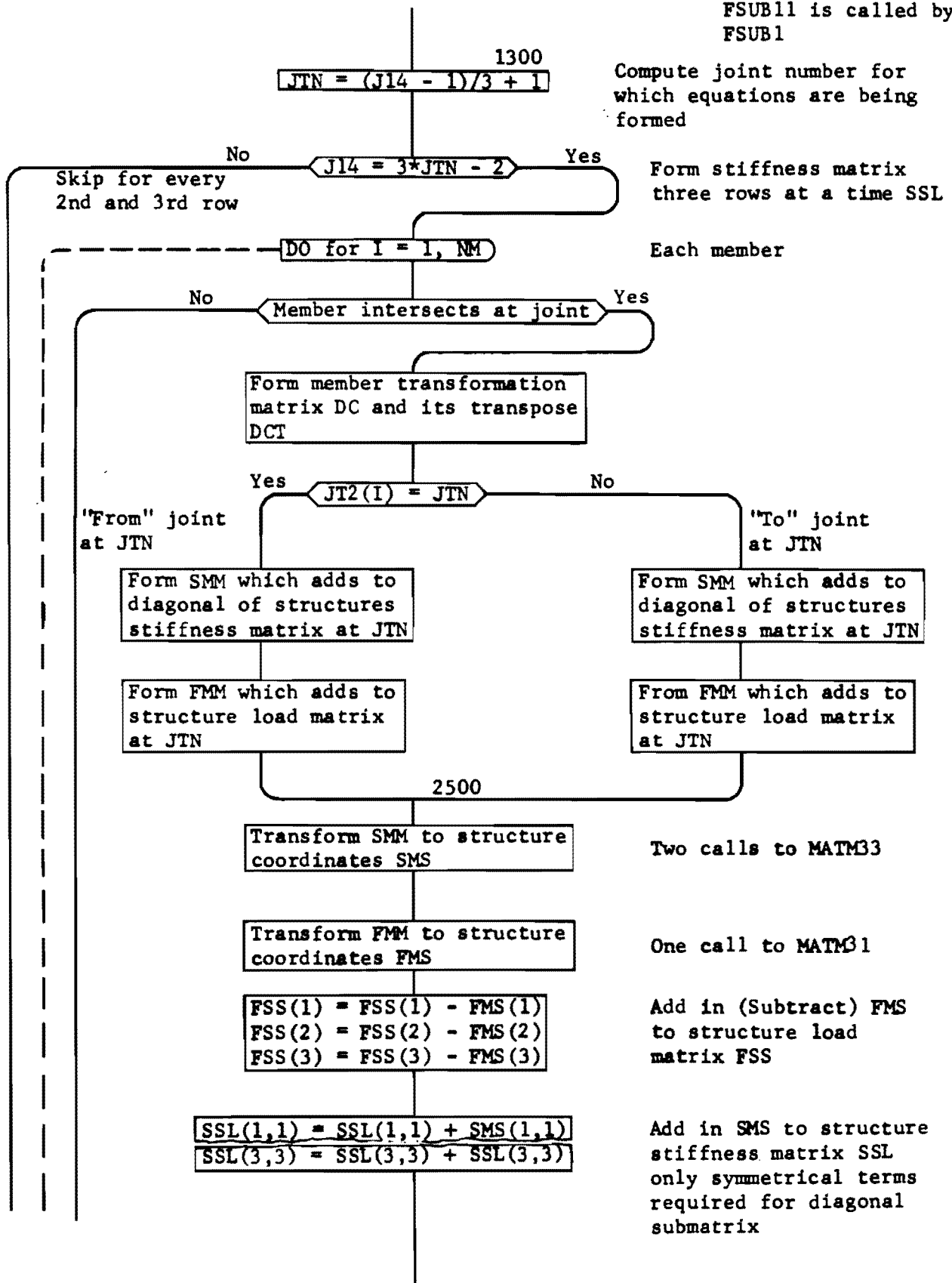
SUBROUTINE DISCLD

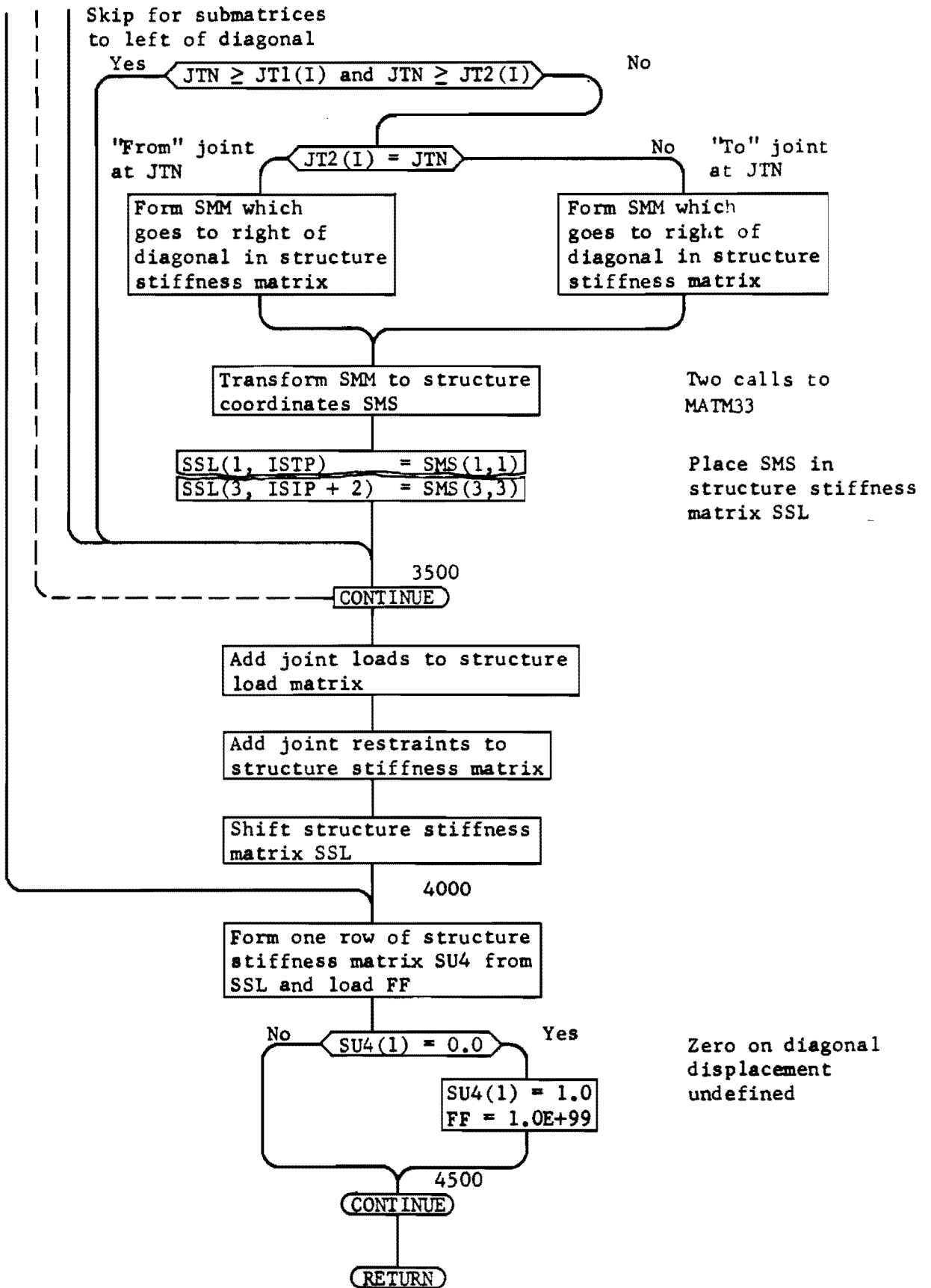
DISCLD called by
FORMLD, MEMRES



SUBROUTINE FSUB11

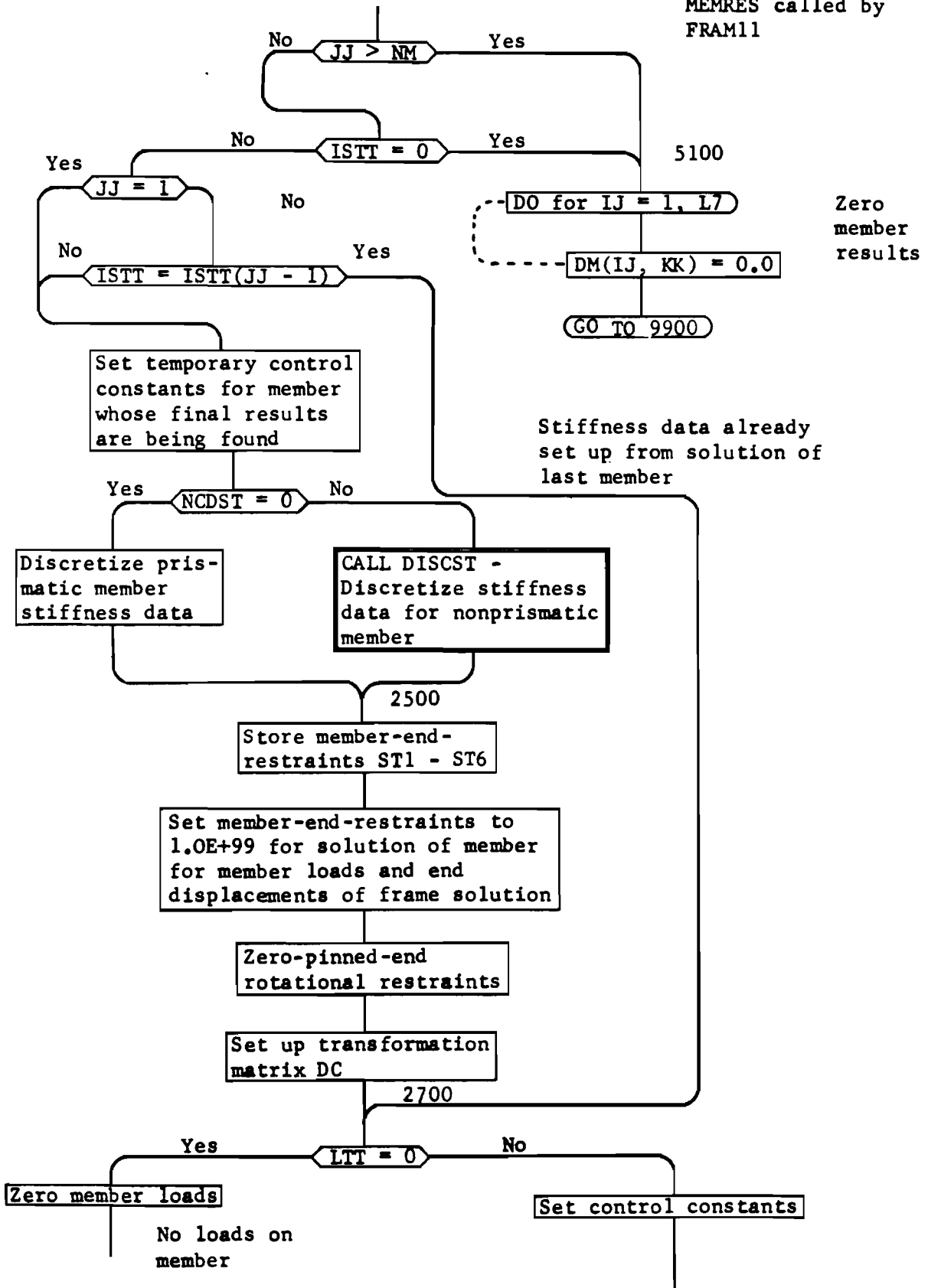
FSUB11 is called by FSUB1

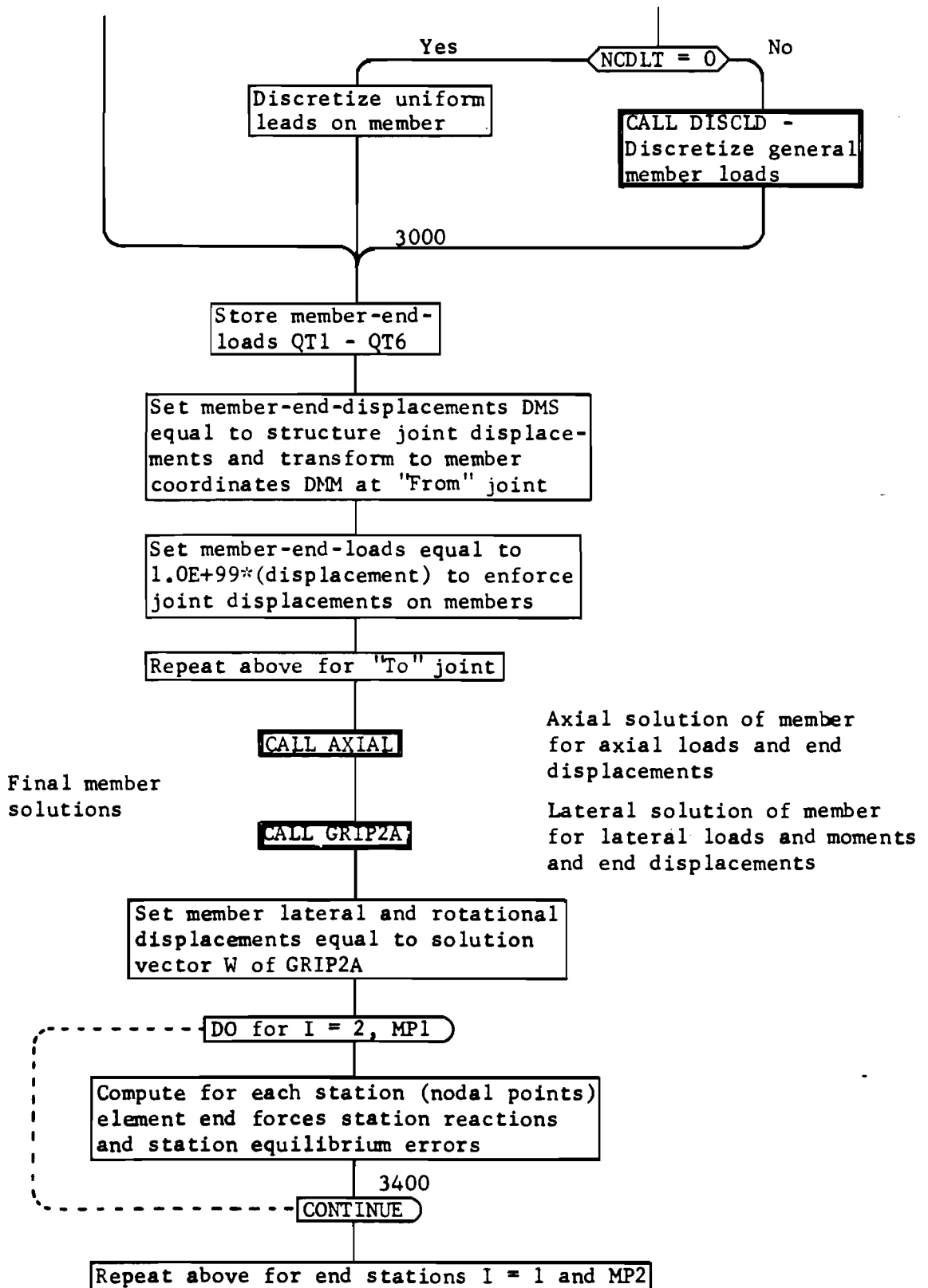


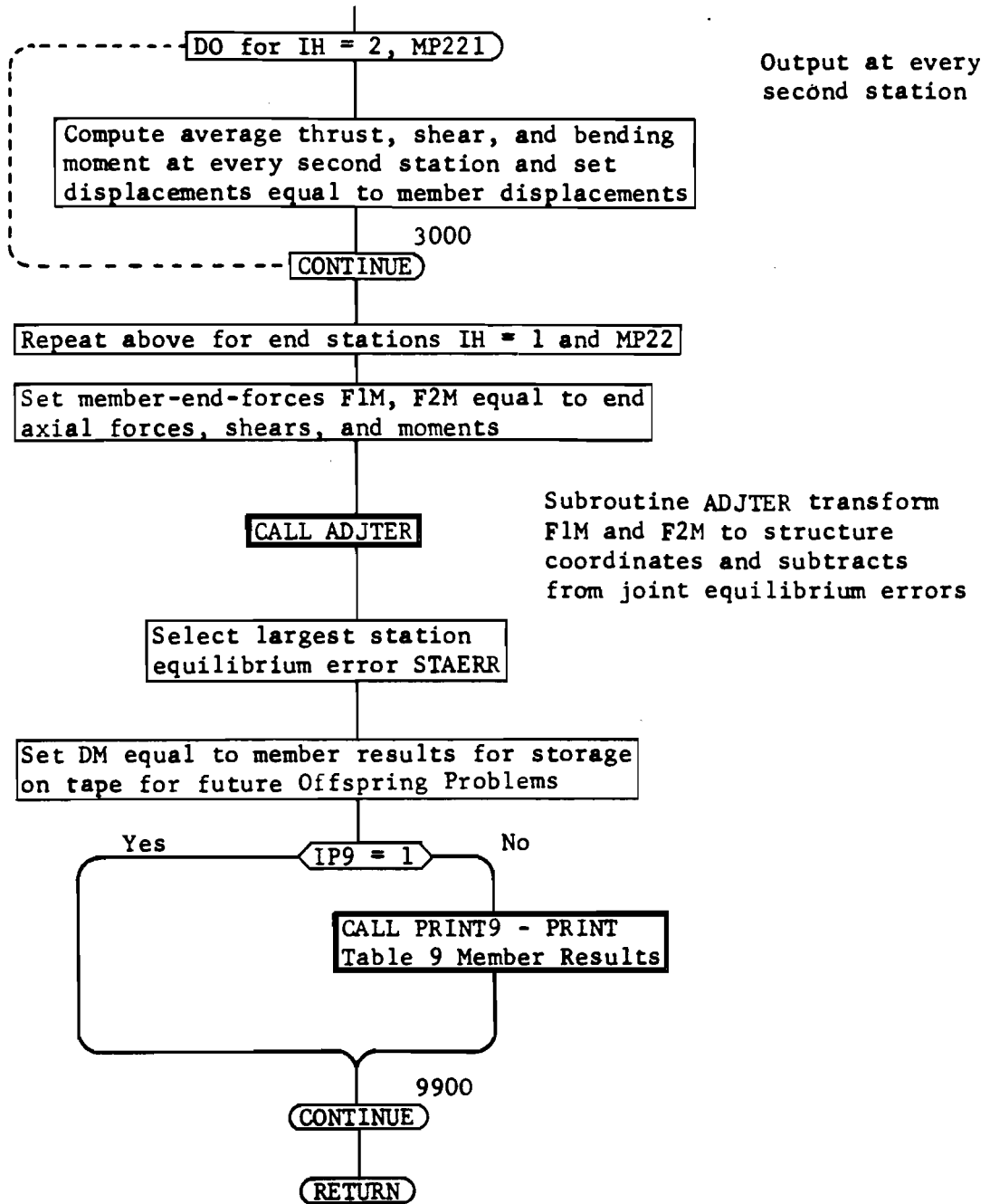


SUBROUTINE MEMRES

MEMRES called by
FRAM11







APPENDIX 4

GLOSSARY OF FORTRAN NOTATION

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C-----NOTATION FOR FRAME 11

C		02JLO	C	ERX, ERY	ERROR IN JOINT COORDINATES OR MEMBER	02JLO
C		2JLO	C		OFFSETS	02JLO
C		7JLO	C	ERRLN	ERROR IN LENGTH OF MEMBER	02JLO
C		02JLO	C	ERXX(), ERY(),	JOINT EQUILIBRIUM ERRORS	02JLO
C	AEI)	02JLO	C	ERZZ()		02JLO
C	AEI)	02JLO	C	FI)	MOMENT OF INERTIA TIMES MODULUS OF ELAS	02JLO
C	AEL()	02JLO	C	FF	COEFFICIENT IN LOAD MATRIX	02JLO
C	AELT	02JLO	C	FL()	VALUES OF FI) AT EDGES OF SECTIONS	02JLO
C	AERT	02JLO	C	FLT	VALUE OF FL() AT LEFT (START) OF SECTION	02JLO
C	AETT2	02JLO	C	FMM()	MEMBER END FORCES IN MEMBER COORDINATES	02JLO
C		02JLO	C	FMS()	MEMBER END FORCES IN STRUCTURE	02JLO
C	AMI)	02JLO	C		COORDINATES	02JLO
C	AM1(), AN2(),	02JLO	C	FMM(),	MEMBER FIXED END FORCES	02JLO
C	AM3()	02JLO	C	FMMT()	MEMBER FIXED END FORCES FOR ONE MEMBER	02JLO
C		02JLO	C	FRT	VALUE OF FI) AT RIGHT (END) OF SECTION	02JLO
C	A1(), A2	02JLO	C	FSS()	STRUCTURE LOAD MATRIX	02JLO
C		02JLO	C	FTT2	VALUE OF FI) AT MIDDLE OF PARTIAL	02JLO
C		02JLO	C		ELEMENT ON RIGHT (END) OF SECTION	02JLO
C	B()	02JLO	C	FIM()	MEMBER END FORCES AT FROM JOINT IN	02JLO
C	BB	02JLO	C		MEMBER COORDINATES	02JLO
C	BM()	02JLO	C	F2M()	MEMBER END FORCES AT TO JOINT IN MEMBER	02JLO
C	BM(), BV()	02JLO	C		COORDINATES	02JLO
C	C	02JLO	C	F1S()	MEMBER END FORCES AT FROM JOINT IN	02JLO
C		02JLO	C		STRUCTURE COORDINATES	02JLO
C	CC	02JLO	C	F2S()	MEMBER END FORCES AT TO JOINT IN	02JLO
C	CM(), CV()	02JLO	C		STRUCTURE COORDINATES	02JLO
C	DI)	02JLO	C	M	ONE HALF OF ELEMENTS LENGTH	02JLO
C	DC()	02JLO	C	MCU	MINIMUM	02JLO
C	DC1(),	02JLO	C	MSU	MAXIMUM	02JLO
C	DC1, DC2	02JLO	C	I	INTEGER INDEX	02JLO
C	DC1L(), DC2L()	02JLO	C	IABAN	FATAL ERROR FLAG	02JLO
C	DC1S(), DC2S()	02JLO	C	IAXOPL()	AXIS OPTIONS FOR LOAD TYPES	02JLO
C	DD	02JLO	C	IAXOPS()	AXIS OPTIONS FOR STIFFNESS TYPES	02JLO
C	DD18	02JLO	C	IAXOPT	TEMPORARY VALUE OF AXIS OPTION	02JLO
C	DDZ	02JLO	C	IC	PRINTER CONTROL	02JLO
C	DEL	02JLO	C	IB	DECREMENTING INTEGER	02JLO
C	DENOM()	02JLO	C	ICOUNT	CONTROL CONSTANT	02JLO
C	DIS	02JLO	C	IDJ	MAXIMUM DIFFERENCE IN JOINT NUMBERS	02JLO
C		02JLO	C		CONNECTED BY MEMBERS	02JLO
C	DM(), DMT(),	02JLO	C	IFORM	CONTROL CONSTANT	02JLO
C	DM1()	02JLO	C	IM	INTEGER INDEX FOR OUTPUT STATIONS	02JLO
C		02JLO	C	IMB	(BANDWIDTH OF EQ - 11/2	02JLO
C	DM2()	02JLO	C	IMBP1	IMB + 1	02JLO
C		02JLO	C	IMB1	IMB - 1	02JLO
C	DO	02JLO	C	I1, I11, IJ, IJT,	INTEGER INDICES	02JLO
C		02JLO	C	IJ1, IJ6		02JLO
C	DB	02JLO	C	IM1	I - 1	02JLO
C	DX(), DY(), DZ()	02JLO	C	IOP1	PRINTER CONTROL	02JLO
C	DX, DY	02JLO	C	IOPUP1(), IOPOPT	MEMBER OUTPUT OPTION	02JLO
C	DXL(), DY1()	02JLO	C	IPINL1(), IPINL2	PIN AT LEFT (FROM) JOINT OPTION	02JLO
C	DXB1(), DYB1()	02JLO	C	IPINR1(), IPINR2	PIN AT RIGHT (TO) JOINT OPTION	02JLO
C	DXT(), DYT()	02JLO	C	IP1	I + 1	02JLO
C	DZT()	02JLO	C	IP8, IP9, IP10	PRINT OPTIONS FOR TABLES 8, 9, 10	02JLO
C	DXX(), DYY(),	02JLO	C	IST(), ISTT	STIFFNESS TYPE	02JLO
C	DZZ()	02JLO	C	ISTP	3*J21 + 1	02JLO
C	E	02JLO	C	ITEST()	ALPHANUMERIC CONSTANTS	02JLO
C	ERR	02JLO				
C	ERX(), ERY(),	02JLO				
C	ERZ()	02JLO				

C	ITYPE	PROBLEM TYPE	02.JL0	C	NJNZ	NUMBER OF NON ZERO JOINTS ON DATA CARD IN	02.JL0
C	ITYPEL	PROBLEM TYPE OF PREVIOUS PROBLEM	02.JL0	C		TABLES 2,3	02.JL0
C	I1, I2	FIRST AND LAST STATION INSIDE SECTION	02.JL0	C	NJT	NUMBER OF FRAME JOINTS	02.JL0
C	I1PND	I1 + ND	02.JL0	C	NL,MLA	NUMBER OF SIMULTANEOUS EQUATIONS	02.JL0
C	I1P1	I1 + 1	02.JL0	C	NLC	NUMBER OF LOAD CASES	02.JL0
C	J, JJ, JJJ	INTEGER INDICIES	02.JL0	C	NLT	NUMBER OF LOAD TYPES	02.JL0
C	J1J	SWITCH TO ALTERNATE FORCE AND MOMENT	02.JL0	C	NLTL	NLT FOR LAST PROBLEM	02.JL0
C		EQUATIONS	02.JL0	C	NM	NUMBER OF FRAME MEMBERS	02.JL0
C	JM1	J - 1	02.JL0	C	NM6	NM/6	02.JL0
C	JNTL	ERROR FLAG FOR JOINT NUMBER TO LARGE	02.JL0	C	NPI (,)	LIST OF PROBLEM NUMBERS RESULTS SAVED FOR	02.JL0
C	JTR	JOINT NUMBER	02.JL0	C	NPUB1 (,), NPT1 ()	PROBLEM NUMBER (ALPHA NUMERIC)	02.JL0
C	JT1(), JT1T, J1,	FROM JOINT	02.JL0	C	NQ	NUMBER OF ELEMENTS REMAINING IN SECTION	02.JL0
C	J11		2.JL0	C	NST	NUMBER OF STIFF TYPES	02.JL0
C	J21	ABSOLUTE VALUE OF DIFFERENCE IN JOINT	02.JL0	C	NSTL	NST FOR LAST PROBLEM	02.JL0
C		NUMBERS OF MEMBERS	02.JL0	C	N2M1	NCD2 - 1	02.JL0
C	K	INTEGER INDEX	02.JL0	C	N2M1	NCD3 - 1	02.JL0
C	KEEP2-KEEP7	HOLD OPTIONS FOR TABLES 2-7	02.JL0	C	N123	CONTROL WHICH CYCLES 1,2,3	02.JL0
C	KEKE	CHECK FOR INDEPENDENT PROBLEM	02.JL0	C	N2, N3	ALTERNATING SWITCHES FOR TAPES 2,3	02.JL0
C	KK	INTEGER INDEX	02.JL0	C	PRAE(), PRAET	PRISMATIC AEI ()	02.JL0
C	L	INTEGER INDEX	02.JL0	C	PRAT	PRISMATIC AREA	02.JL0
C	LT(), LTT	LOAD TYPE	02.JL0	C	PRF(), PRFT	PRISMATIC F()	02.JL0
C	L1-L4, L6, L7	DIMENSION LIMITS	02.JL0	C	PRIT	PRISMATIC MOMENT OF INERTIA	02.JL0
C	L7M1	L7 - 1	02.JL0	C	QU	CONCENTRATED STATION LOAD OR SPRING	02.JL0
C	L7M2	L7 - 2	02.JL0	C	UI	CONCENTRATED LOAD OR SPRING BETWEEN	02.JL0
C	L7M3	L7 - 3	02.JL0	C		STATIONS	02.JL0
C	M	NUMBER OF ELEMENT IN MEMBER	02.JL0	C	QL, OR	INTENSITY OF LOADING OR RESTRAINT AT LEFT	02.JL0
C	MDJT	MAXIMUM PERMITTED VALUE OF IDJ	02.JL0	C		(START) AND RIGHT (END) OF SECTION	02.JL0
C	MDB	MAXIMUM PERMITTED VALUE OF INB	02.JL0	C	UT1-UT6	LOADS ON MEMBER END STATIONS	02.JL0
C	MDB1	MDB + 1	02.JL0	C	UX(), OY(), OZ()	MEMBER STATION LOADS	02.JL0
C	ML, MLT, MLA	CONTROL FOR MULTIPLE LOAD OPTION	02.JL0	C	UXL(), OYL(),	MEMBER LOADS AT EDGES OF SECTIONS	02.JL0
C	MM1	M - 1	02.JL0	C	OZL()		2.JL0
C	MNC5	MAXIMUM VALUE PERMITTED FOR NCS	02.JL0	C	UXLT,OYLT,	VALUES OF OXLI (),OYLI (), AT LEFT (START)	02.JL0
C	MNC6	MAXIMUM VALUE PERMITTED FOR NC6	02.JL0	C	OZLT	OF SECTION	02.JL0
C	MNJT	MAXIMUM VALUE PERMITTED FOR NJT	02.JL0	C	UXRT, OYRT,	VALUES OF OXLI (), OYLI (), OZLI () AT RIGHT	02.JL0
C	MNLC	MAXIMUM VALUE PERMITTED FOR NLC	02.JL0	C	UZRT	(END) OF SECTION	02.JL0
C	MNLT	MAXIMUM VALUE PERMITTED FOR NLT	02.JL0	C	UXX(), OYY(),	JOINT LOADS	02.JL0
C	MNM	MAXIMUM VALUE PERMITTED FOR NM	02.JL0	C	OZZ()		*2.JL0
C	MNST	MAXIMUM VALUE PERMITTED FOR NST	02.JL0	C	QXXT, OYYT, QZZT	TEMPORARY VALUES OF OXX(), OYY(), OZZ	02.JL0
C	MP1	M + 1	02.JL0	C	Q1, Q2	INTENSITY OF LOADING OR RESTRAINTS AT	02.JL0
C	MP2	M + 2	02.JL0	C		BEGINNING AND END OF ELEMENT	02.JL0
C	MP22	(M + 2)/2	02.JL0	C	RM(,), RO()	RECURSION MULTIPLIERS	02.JL0
C	MP221	MP22 - 1	02.JL0	C	RX(), RY(), RZ()	STATION REACTIONS	02.JL0
C	MCDL(), MCDLT	NUMBER OF CARDS THAT FOLLOW FOR LOAD TYPE	02.JL0	C	RXX(), RYY(),	JOINT REACTIONS	02.JL0
C	MCD5(), MCDST	NUMBER OF CARDS THAT FOLLOW FOR STIF TYPE	02.JL0	C	RZZ()		02.JL0
C	MCD2-MCD7	NUMBER OF CARDS IN TABLES 2-7	02.JL0	C	SL()	VECTOR OF STIFFNESS MATRIX	02.JL0
C	MCR5, MCR6	NUMBER OF CARDS READ IN TABLES 5 AND 6	02.JL0	C	SMC(,)	MEMBER STIFFNESS MATRICES IN COMPACT FORM	02.JL0
C	MCS1(), MCS	NUMBER OF CARDS IN TABLE 5 ABOVE THE	02.JL0	C	SMM(,)	MEMBER STIFFNESS MATRIX (3X3) IN MEMBER	02.JL0
C		NUMBER OF STIFF TYPES (VARIABLE STIFF)	02.JL0	C		COORDINATES	02.JL0
C	MCS1T, MCS2T	FIRST AND LAST CARD NUMBER OF VARIABLE	02.JL0	C	SMT()	SINGLE MEMBERS STIFFNESS MATRIX IN	02.JL0
C		STIFF DATA FOR MEMBER	02.JL0	C		COMPACT VECTOR FORM	02.JL0
C	MC61(), MC6	NUMBER OF CARDS IN TABLE 6 ABOVE THE	02.JL0	C	SMS	MEMBER STIFFNESS MATRIX (3X3) IN	02.JL0
C		NUMBER OF LOADS TYPES (VARIABLE LOADS)	02.JL0	C		STRUCTURE COORDINATES	02.JL0
C	MC61T, MC62T	FIRST AND LAST CARD NUMBER OF VARIABLE	02.JL0	C	SSL	STRUCTURE STIFFNESS MATRIX	02.JL0
C		LOAD DATA FOR MEMBER	02.JL0	C	ST	STATION VALUE OF STIFFNESS	02.JL0
C	MFSUB	SWITCH TO CHOOSE APPROPRIATE FSUM	02.JL0	C	STAERR	LARGEST STATION EQUILIBRIUM ERROR IN	02.JL0
C	M1	INTEGER INDEX	02.JL0	C		MEMBER	02.JL0

C	STL, STR	STIFFNESS AT LEFT (START) AND RIGHT (END) OF SECTION	02JLO
C	STT1, STT2	STIFFNESS AT MID POINTS OF PARTIAL ELEMENTS AT BEGINNING AND END OF ADJACENT SECTIONS	02JLO
C	ST1, ST6	RESTRAINTS AT MEMBER END STATIONS	02JLO
C	SU1), SU4()	COEFF OF STIFF MATRIX (ONE ROW)	02JLO
C	SX1), SY1), SZ1)	MEMBER STATION ELASTIC RESTRAINTS	02JLO
C	SXL1), SYL1), SZL1)	VALUES OF SX1), SY1), SZ1), AT EDGES OF SECTIONS	02JLO
C	SXLT, SYLT, SZLT	VALUES OF SXL1), SYL1), SZL1) AT LEFT (START) OF SECTION	02JLO
C	SXRT, SYRT, SZRT	VALUES OF SXL1), SYL1), SZL1) AT RIGHT (END) OF SECTION	02JLO
C	SXX1), SYX1), SZX1)	JOINT RESTRAINTS	02JLO
C	SXXT, SYXT, SZXT	TEMPORARY VALUES OF SXX1), SYX1), SZX1)	02JLO
C	TI)	AXIAL THRUST OUTPUT VALUE	02JLO
C	TAU1, TAU2	CONCENTRATED ANGLE CHANGES IN ELEMENT	02JLO
C	TEMP1, TEMP2	TEMPORARY VALUES	02JLO
C	TM	ELEMENT LENGTH	02JLO
C	TOL	JOINT LOCATION TOLERANCE	02JLO
C	TTOL	2*TOL	02JLO
C	T33	TEMPORARY MATRIX USED TO OBTAIN TRIPLE PRODUCT	02JLO
C	UQX1), UQY1)	UNIFORM MEMBER LOADS	02JLO
C	UQXT, UQYT	TEMPORARY VALUES OF UQX1), UQY1)	02JLO
C	U11), U21)	AXIAL FORCES ON ENDS OF ELEMENT	02JLO
C	V	SHEAR FORCE OUTPUT VALUE	02JLO
C	V11), V21)	SHEAR FORCES ON ENDS OF ELEMENT	02JLO
C	W1)	DISPLACEMENT VECTOR FROM GRIP2A	02JLO
C	W11), W21)	MOMENTS ON ENDS OF ELEMENT	02JLO
C	X1), Y1)	JOINT COORDINATES	02JLO
C	XL	DISTANCE TO LEFT (START) OF SECTION	02JLO
C	XL1)	DISTANCE TO LEFT (START) OF LOAD SECTION	02JLO
C	XLS1)	DISTANCE TO LEFT (START) OF STIFF SECTION	02JLO
C	XR	DISTANCE TO RIGHT (END) OF SECTION	02JLO
C	XRL	DISTANCE TO RIGHT (END) OF LOAD SECTION	02JLO
C	XRS1)	DISTANCE TO RIGHT (END) OF STIFF SECTION	02JLO
C	XT, YT	TEMPORARY JOINT COORDINATES	02JLO
C	XX	DISTANCE TO CONCENTRATED LOAD FROM STATION	02JLO
C	X1, X2	LENGTH OF PARTIAL ELEMENTS AT ENDS OF SECTIONS	02JLO
C	X2L	X2 FROM LAST SECTION	02JLO
C	Z1	FLOATING POINT 1	02JLO
C	Z11, Z12	FLOATING POINT 11 AND 12	02JLO
C	ZL	MEMBERS LENGTH	02JLO
C	ZLL1)	LENGTH OF MEMBERS BY LOAD TYPE	02JLO
C	ZLS1)	LENGTH OF MEMBERS BY STIFF TYPE	02JLO
C	ZL2	ZL*ZL	02JLO
C	ZL3	ZL2*ZL	02JLO
C	ZM1), ZMT	LOAD MULTIPLIERS	02JLO

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APPENDIX 5

FORTRAN LISTING OF PROGRAM

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COMMON /BLOCK5/ XLS( 75), XRS( 75), FL( 75), AEL( 75),
2 SZL( 75), SYL( 75), SZL( 75)
COMMON /BLOCK6/ XLL(150), XRL(150), QXL(150), QYL(150),
2 QZL(150)
COMMON /BLOCK7/ F1(42), AE(42), SK(42), SY(42),
2 SZ(42), OK(42), OY(42), OZ(42), AI(42),
3 BI(42), AI(42), DI(42), OK(42), DY(42),
4 OZ(42), UI(42), VI(42), WI(42), UZ(42),
5 VZ(42), WZ(42), ERX(42), ERY(42), ERZ(42),
6 RX(42), RY(42), RZ(42)
COMMON /BLOCK9/ T(21), V(21), BN(21), DNT(21),
2 DYT(21), DZT(21)
COMMON /BLK1/ KEEP2, KEEP3, KEEP4, KEEP5, KEEP6, KEEP7,
2 ITYPE, NCD2, NCD3, NCD4, NCD5, NCD6, NCD7,
3 TABAN, IFORM, NH, NJT, NST, NLT, TOL,
4 N, NP1, NP2, ISTT, LTT, IYPEL, IDJ,
5 NLC, IP8, IP9, IP10
COMMON /BLK2/ NHJT, NHST, NHLT, NHM, NHCS, NHCC, NDJT, NHLC
COMMON /BLK3/ NPSUB
COMMON / RI / RL, RL, J1
1 FORMAT ( 50H PROGRAM FRAME II - DEV DECK - MATLOCK-HAYS )
2 20HREVISION DATE = 27 AUG 70)
10 FORMAT ( 5H 80X 10H1-----TRIM )
11 FORMAT ( 5H1 80X 10H1-----TRIM )
12 FORMAT ( 20AA )
13 FORMAT ( 5X, 20AA )
14 FORMAT ( A1, A4, 5X, 17AA, A2 )
15 FORMAT ( ///10H PROB ( /5X, A1, A4, 5X, 17AA, A2 )
16 FORMAT ( ///17H PROB (CONTD), /5X, A1, A4, 5X, 17AA, A2, // )
17 FORMAT ( A1, A4, A1, A4, 17AA, A2 )
50 FORMAT ( 50H SOLUTION ABANDONED IN SEARCH OF AN INDEPENDENT
2 10H PROBLEM )
3 40H THE FOLLOWING CARDS WERE DISCARDED IN SEARCH,
4 // )
51 FORMAT ( ///50H NO HOLD OPTIONS MAY BE EXERCISED ON FIRST PRO.
2 15HBLEM OF RUN )
52 FORMAT ( 40H PROBLEM MUST BE TYPE 1, 2, 3, OR 4 )
53 FORMAT ( 55H HOLD OPTIONS MUST BE 1 OR 0 )
54 FORMAT ( 35H PRINT OPTIONS MUST BE 1 OR 0 )
55 FORMAT ( 46H NUMBER OF CARDS ADDED CAN NOT BE NEGATIVE )
100 FORMAT ( 5X, 15, 5X, 615, 5X, 315, //, 15X, 615 )
101 FORMAT ( 81//, 35H TABLE 1 - PROGRAM CONTROL DATA, //,
2 17H PROBLEM TYPE, 15, //, //, 25X, 12HINPUT TABLES, //,
3 10X, 45H TABLE HOLD DATA FROM NUMBER OF CARDS, //,
4 10X, 45H NUMBER LAST PROBLEM ADDED FOR THIS, //,
5 10X, 45H (1 - YES, 0 - NO) PROBLEM, //,
6 10X, 5H 2, 10X, 15, 15X, 15, //, 10X, 5H 3, 10X, 15, 15X, 15, //,
7 10X, 5H 4, 10X, 15, 15X, 15, //, 10X, 5H 5, 10X, 15, 15X, 15, //,
8 10X, 5H 6, 10X, 15, 15X, 15, //, 10X, 5H 7, 10X, 15, 15X, 15, //,
9 25X, 15HOUTPUT TABLES, //,
1 10X, 25H TABLE SUPPRESS OUTPUT, //,
2 10X, 25H NUMBER (1 = YES, 0 = NO), //,
3 10X, 5H 8, 10X, 15, //, 10X, 5H 9, 10X, 15, //, 10X, 5H 10, 10X, 15 )
151 FORMAT ( 50H TABLE B - JOINT DISPLACEMENTS AND REACTIONS ,
2 ///, 20X, 15HDISPLACEMENTS , 16X, 10H REACTIONS, //,
3 5X, 35HJOINT DISPL(X) DISPL(Y) ROTATION(Z) ,

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26JAO 4 31HREACT(X) REACT(Y) REACT(Z) // )
26JAO 152 FORMAT ( 5X, 15, 6E11, 3 )
26JAO 162 FORMAT ( 45H TABLE 10 - JOINT EQUILIBRIUM ERRORS , //, //,
2 40H JOINT ERR(X) ERR(Y) ERR(Z), //,
3 40H FORCE FORCE MOMENT, // )
26JAO DATA ITEST(1), ITEST(2), ITEST(3), ITEST(4) / 1MC, AMEASE, 1M ,
2 4H /
ITYPEL = 0
COMMENT - HEAD RUN ID, PRINT PROGRAM ID AND RUN ID
READ 12, (AM1(1)), 11 = 1, 40)
PRINT 11
PRINT 1
PRINT 12, (AM1(1)), 11 = 1, 40)
COMMENT - RETURN HERE TO READ NEW PROBLEM
1010 READ 14, NPROB, (AM2(1)), 11 = 1, 10)
COMMENT - IF NPROB = CEASE, TERMINATE RUN
IF (NPROB(1)) .EQ. ITEST(1) .AND. NPROB(2) .EQ. ITEST(2)
2 GO TO 9900
COMMENT - INPUT AND ECHO PRINT PROGRAM CONTROL DATA (TABLE 1)
READ 100, ITYPE, KEEP2, KEEP3, KEEP4, KEEP5, KEEP6, KEEP7, IP8, IP9, IP10,
2 NCD2, NCD3, NCD4, NCD5, NCD6, NCD7
1050 PRINT 11
PRINT 12, (AM1(1)), 11 = 1, 40)
PRINT 15, NPROB, (AM2(1)), 11 = 1, 10)
PRINT 1=1, ITYPE, KEEP2, NCD2, KEEP3, NCD3, KEEP4, NCD4, KEEP5, NCD5,
2 KEEP6, NCD6, KEEP7, NCD7, IP8, IP9, IP10
COMMENT - CHECK FOR ILLEGAL DATA IN TABLE 1
IF (ITYPE .LT. 1 .OR. ITYPE .GT. 4) GO TO 1210
IF (KEEP2 .LT. 0 .OR. KEEP2 .GT. 1) GO TO 1220
IF (KEEP3 .LT. 0 .OR. KEEP3 .GT. 1) GO TO 1220
IF (KEEP4 .LT. 0 .OR. KEEP4 .GT. 1) GO TO 1220
IF (KEEP5 .LT. 0 .OR. KEEP5 .GT. 1) GO TO 1220
IF (KEEP6 .LT. 0 .OR. KEEP6 .GT. 1) GO TO 1220
IF (KEEP7 .LT. 0 .OR. KEEP7 .GT. 1) GO TO 1220
IF (IP8 .LT. 0 .OR. IP8 .GT. 1) GO TO 1230
IF (IP9 .LT. 0 .OR. IP9 .GT. 1) GO TO 1230
IF (IP10 .LT. 0 .OR. IP10 .GT. 1) GO TO 1230
IF (NCD2 .LT. 0) GO TO 1240
IF (NCD3 .LT. 0) GO TO 1240
IF (NCD4 .LT. 0) GO TO 1240
IF (NCD5 .LT. 0) GO TO 1240
IF (NCD6 .LT. 0) GO TO 1240
IF (NCD7 .LT. 0) GO TO 1240
KEKE = KEEP2 + KEEP3 + KEEP4 + KEEP5 + KEEP6 + KEEP7
IF ( IYPEL .EQ. 0 .AND. KEKE .NE. 0 ) GO TO 1200
GO TO 1300
COMMENT - ABORT PROBLEM, SEARCH FOR INDEPENDENT PROBLEM
1240 PRINT 51
GO TO 9800
1210 PRINT 52
GO TO 9800
1220 PRINT 53
GO TO 9800
1230 PRINT 54
GO TO 9800
1240 PRINT 55

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1300      GO TO 9800
CONTINUE
      IABAN = 0
COMMENT - IABAN = 1 INDICATES FATAL ERROR FOUND IN SUBROUTINE
COMMENT - PROBLEM ABANDONED IN SEARCH OF AN INDEPENDENT PROBLEM
      IF (KEEPS .EQ. 0) NSTL = 0
COMMENT - MAIN PROGRAM STARTS HERE
COMMENT - SKIP TABLES 2 - 6 FOR FAMILY PROBLEM
      IF ( ITYPE .EQ. 4 ) GO TO 4000
      PRINT 11
      PRINT 16, NPROB, (AMZ(11), 11 = 1, 18)
COMMENT - SUBROUTINE JTCORD INPUTS JOINT GEOMETRY DATA (TABLE 2)
COMMENT - CHECKS FOR BAD DATA, COMPUTES JOINT COORDINATES, ECHO PRINTS
COMMENT - DATA AND PRINTS COMPUTED JOINT COORDINATES
      CALL JTCORD
      IF ( IABAN .EQ. 1 ) GO TO 9800
      PRINT 11
      PRINT 16, NPROB, (AMZ(11), 11 = 1, 18)
COMMENT - SUBROUTINE MEMLOC INPUTS LOCATION OF STIFFNESS AND LOAD
COMMENT - TYPES IN FRAME (TABLE 3), CHECKS FOR BAD DATA, COMPUTES MEMBER
COMMENT - NUMBERS, LENGTHS, OFFSETS AND DIRECTION COSINES, ECHO PRINTS DATA
COMMENT - AND PRINTS COMPUTED MEMBER NUMBERS, LENGTHS AND OFFSETS
      CALL MEMLOC
      IF ( IABAN .EQ. 1 ) GO TO 9800
      PRINT 11
      PRINT 16, NPROB, (AMZ(11), 11 = 1, 18)
COMMENT - SUBROUTINE JNTDAT INPUTS JOINT LOADS AND RESTRAINTS
COMMENT - (TABLE 4), CHECKS FOR BAD DATA, ACCUMULATES JOINT LOADS AND
COMMENT - RESTRAINTS, ECHO PRINTS DATA AND PRINTS ACCUMULATED DATA
      CALL JNTDAT
      IF ( IABAN .EQ. 1 ) GO TO 9800
      PRINT 11
      PRINT 16, NPROB, (AMZ(11), 11 = 1, 18)
COMMENT - SUBROUTINE RDMST INPUTS MEMBER STIFFNESS DATA (TABLE 5),
COMMENT - CHECKS FOR BAD DATA, CONVERTS INPUT DISTANCES TO MEMBER
COMMENT - COORDINATES AND ECHO PRINTS DATA
      CALL RDMST
      IF ( IABAN .EQ. 1 ) GO TO 9800
      PRINT 11
      PRINT 16, NPROB, (AMZ(11), 11 = 1, 18)
COMMENT - FOR BAD DATA, CONVERTS LOADS AND DISTANCES TO MEMBER
COMMENT - COORDINATES AND ECHO PRINTS DATA
      CALL RDMPLD
      IF ( IABAN .EQ. 1 ) GO TO 9800
4D0D      CONTINUE
      PRINT 11
      PRINT 16, NPROB, (AMZ(11), 11 = 1, 18)
COMMENT - SUBROUTINE COMP INPUTS SUPERPOSITION DATA (TABLE 7) FOR
COMMENT - FAMILY PROBLEMS, CHECKS FOR BAD DATA, SETS UP STORAGE FOR FAMILY
COMMENT - SOLUTIONS, ACCUMULATES PROBLEM MULTIPLIERS, ECHO PRINTS DATA
COMMENT - AND PRINTS ACCUMULATED PROBLEM MULTIPLIERS
      CALL COMP ( NPROB )
      IF ( IABAN .EQ. 1 ) GO TO 9800
      ITYPEL = ITYPE
COMMENT - SKIP FOR FAMILY PROBLEM
      IF ( ITYPE .EQ. 4 ) GO TO 6800

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04MYO
09FEO
26JAO
24APO
24APO
26JAO
24APO
24APO
09FEO
26JAO
09JEO
24APO
24APO
24APO
26JAO
26JAO
09JEO
24APO
24APO
26JAO
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24APO
26JAO
26JAO
09FEO

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COMMENT - FORM MEMBER STIFFNESS MATRICES AND MEMBER FIXED-END-FORCE
COMMENT - MATRICES
      DO 5800 JJ = 1, NM
          ISTT = IST(IJJ)
          LTT = LT(IJJ)
COMMENT - SKIP FOR NULL MEMBER
          IF (ISTT .EQ. 0) GO TO 5750
COMMENT - SKIP FOR OFFSPRING PROBLEMS
          IF ( ITYPE .EQ. 3 ) GO TO 5550
COMMENT - SKIP FOR STIFFNESS TYPES HELD FROM LAST PROBLEM - NOTE THAT
COMMENT - IF TABLE 5 IS NOT KEPT NSTL = 0.
          IF (ISTT .LE. NSTL) GO TO 5550
          IF ( IJJ .EQ. 1 ) GO TO 5500
COMMENT - SKIP FOR STIFFNESS TYPE REPEATED IN THIS PROBLEM
          IF (ISTT .EQ. IST(IJJ - 1)) GO TO 5700
COMMENT - SUBROUTINE FORMST CALCULATES STIFFNESS MATRIX FOR BOTH
COMMENT - PRISMATIC AND VARIABLE CROSS SECTION MEMBERS AND STORES IN
COMMENT - COMPACT VECTOR SMNT(1) = 1, 131
          5500 CALL FORMST ( RM, RO, W, SL, SU, SMNT, L1, L3, L4, L6 )
          DO 5510 I = 1, 13
              5510 SMC(ISTT, I) = SMNT(I)
          GO TO 5700
          5550 IFORM = 0
          5700 CONTINUE
          IF (LTT .EQ. 0) GO TO 5750
COMMENT - SUBROUTINE FORMLD CALCULATES FIXED-END-FORCE MATRIX FOR
COMMENT - BOTH PRISMATIC UNIFORMLY LOADED MEMBERS AND ALL OTHERS
          CALL FORMLD ( RM, RO, W, SL, SU, FOMMT, L1, L3, L4, L6 )
          DO 5710 I = 1, 6
              5710 FOMM(IJJ, I) = FOMMT(I)
          GO TO 5800
COMMENT - SET FIXED END-FORCE-MATRIX TO NULL MATRIX FOR NULL LOADING
          5750 DO 5780 I = 1, 6
              5780 FOMM(IJJ, I) = 0.0
          5800 CONTINUE
          NSTL = NST
COMMENT - START SOLUTION OF FRAME JOINT EQUILIBRIUM EQUATIONS
COMMENT - SET CONTROL CONSTANTS FOR FRAME SOLUTION
          IMB = 3*IDJ + 2
          ML = 3*NJT
          IF ( ITYPE .EQ. 1 ) ML = 0
          IF ( ITYPE .EQ. 2 ) ML = 1
          IF ( ITYPE .EQ. 3 ) ML = -1
          NFSUB = 11
          REWIND 2
COMMENT - READ RM AND RO OFF TAPE FOR OFFSPRING PROBLEM
          IF (ML .NE. -1) GO TO 6100
          READ (2) ((RM(I, J), I = 1, IMB), RO(I), J = 1, NL)
          6100 CONTINUE
COMMENT - CALL GRIP2A FOR SOLUTION OF FRAME JOINT EQUILIBRIUM EQUATIONS
COMMENT - GRIP2A SOLVES BOTH FRAME JOINT EQUILIBRIUM EQUATIONS AND
COMMENT - MEMBER EQUILIBRIUM EQUATIONS - GRIP2A CALLS FSUB1 WHICH CALLS
COMMENT - FSUB11 TO SET UP FRAME EQUATIONS OR FSUB12 TO SET UP MEMBER
COMMENT - EQUATIONS
          CALL GRIP2A ( RM, RO, W, SL, SU, L3, L4, L6, IMB )
COMMENT - WRITE RM AND RO ON TAPE FOR PARENT PROBLEM - THEY WOULD BE

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RETURN                                26JAO
END                                  26JAO
*****
SUBROUTINE          SUBROUTINE          SUBROUTINE          SUBROUTINE
*****
SUBROUTINE JTCORD                                26JAO
COMMENT - SUBROUTINE JTCORD INPUTS JOINT GEOMETRY DATA (TABLE 2) 24AP0
COMMENT - CHECKS FOR BAD DATA, COMPUTES JOINT COORDINATES, ECHO PRINTS 24AP0
COMMENT - DATA AND PRINTS COMPUTED JOINT COORDINATES 24AP0
DIMENSION J2( 7 )
COMMON /BLOCK1/ X( 75), Y( 75), OXX( 75), OYY( 75), 13FE0
2 OZZ( 75), SXX( 75), SYI( 75), SZZ( 75), DXX( 75), 13FE0
3 DYY( 75), DZZ( 75), RXX( 75), RYY( 75), RZZ( 75), 13FE0
4 ERXX( 75), ERYI( 75), ERZZ( 75) 13FE0
COMMON /BLK1/ KEEP2, KEEP3, KEEP4, KEEP5, KEEP6, KEEP7, 26JAO
2 ITYPE, NCD2, NCD3, NCD4, NCD5, NCD6, NCD7, 26JAO
3 IABAN, IFORM, NM, NJT, NST, NLT, TOL, 26JAO
4 M, MP1, MP2, ISTT, LTT, IYPEL, IDJ, 12FE0
5 NLC, IP8, IP9, IP10 13FE0
COMMON /BLK3/ MNJT, MNST, MNLT, MNH, MNC5, MNC6, MDJT, MNL 07FE0
9 FORMAT ( 35H TABLE 2 - FRAME GEOMETRY DATA , /// ) 26JAO
10 FORMAT ( 10X, I5, 5X, I5, 5X, 2E10.3, 10X, E10.3 ) 26JAO
11 FORMAT ( 32H NUMBER OF JOINTS IN FRAME = , I5, // ) 26JAO
2 30H REFERENCE JOINT IS JOINT , I5, 5H AT , 26JAO
3 5H X = , E12.3, 10H AND Y = , E10.3, //, 5X, 26JAO
4 25H JOINT TOLERANCE IS , E10.3, // ) 26JAO
12 FORMAT ( 10X, I5, 5X, 2E10.3, 5X, 7I5 ) 26JAO
13 FORMAT ( 10X, I5, 5X, 2E11.3, 5X, 7I5 ) 26JAO
14 FORMAT ( 25X, 23H INPUT OF JOINT OFFSETS , //, 26JAO
2 10X, 35H FROM X-OFFSET Y-OFFSET , 5X, 26JAO
3 35H TO TO TO TO TO TO TO TO , //, 30MR0
4 10X, 5HJOINT, 32X, 5HJOINT , // ) 30MR0
15 FORMAT ( 47X, 7I5 ) 26JAO
16 FORMAT ( 10X, I5 ) 26JAO
17 FORMAT ( 148H HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS, 26JAO
2 15H THE FOLLOWING , // ) 26JAO
18 FORMAT ( 35H NUMBER OF JOINTS IN FRAME = , I5, // ) 26JAO
19 FORMAT ( 31//, 10X, 26HCOMPUTED JOINT COORDINATES, //, 10X, 30MR0
2 25HJOINT X Y , // ) 26JAO
20 FORMAT ( 45H JOINT NUMBERS MUST BE POSITIVE , I 26JAO
21 FORMAT ( 10X, I5, 2E11.3 ) 26JAO
22 FORMAT ( 148H JOINT NUMBER , I5, 17H NOT LOCATED ) 26JAO
23 FORMAT ( 10H NONE ) 26JAO
24 FORMAT ( 40H NO DATA HELD OR READ IN TABLE 2 ) 26JAO
25 FORMAT ( 50H NUMBER OF CARDS IN TABLE 2 MAY NOT EQUAL 1 ) 09MY0
26 FORMAT ( 51H TYPE 3 PROBLEM SHOULD HAVE NO CHANGES IN JOINT, 26JAO
2 15H COORDINATES , //, 26JAO
3 35H NO CARDS ALLOWED IN TABLE 2 ) 26JAO
27 FORMAT ( 43H JOINT NUMBER ABOVE GREATER THAN NUMBER, 26JAO
2 20H OF JOINTS IN FRAME ) 26JAO
28 FORMAT ( 43H NUMBER OF JOINTS IN FRAME GREATER THAN, 26JAO
2 15H STORAGE ALLOWED ) 26JAO

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70 FORMAT ( 35H X AND Y OFFSETS FOR JOINT, I7, 26JAO
2 15H ARE BOTH ZERO ) 26JAO
80 FORMAT ( 10H JOINT, I5, 30H HAS NOT PREVIOUSLY BEEN SPEC, 26JAO
2 5HIFIED ) 26JAO
90 FORMAT ( 32H ERROR IN LOCATION OF JOINT , I5, 26JAO
2 40H EXCEEDS THE TOLERANCE SPECIFIED ABOVE , //, 4X, 26JAO
3 30H THE ERROR IN X DIRECTION IS , E10.3, //, 4X, 26JAO
4 30H THE ERROR IN Y DIRECTION IS , E10.3 ) 26JAO
100 FORMAT ( 40H TOLERANCE MUST BE A POSITIVE NUMBER ) 04MY0
PRINT 9 26JAO
IF ( NCD2 .EQ. 1 ) GO TO 8100 09MY0
IF ( NCD2 .LE. 0 .AND. KEEP2 .LE. 0 ) GO TO 8300 26JAO
IF ( ITYPE .EQ. 3 .AND. NCD2 .NE. 0 ) GO TO 8400 26JAO
IF ( NCD2 .NE. 0 ) GO TO 1150 26JAO
COMMENT - NO NEW DATA 29AP0
PRINT 11 26JAO
PRINT 23 26JAO
GO TO 9800 13FE0
1150 CONTINUE 26JAO
JNTL = 0 26JAO
IF ( KEEP2 .EQ. 1 ) GO TO 1230 26JAO
COMMENT - ALL NEW DATA - SET COORDINATES EQUAL TO 1.0E50 29AP0
DO 1200 I = 1, MNJT 26JAO
1200 X(I) = Y(I) = 1.0E50 26JAO
COMMENT - READ FIRST CARD OF TABLE 2 29AP0
READ 10, NJT, J1, DX, DY, TOL 26JAO
PRINT 11, NJT, J1, DX, DY, TOL 26JAO
IF ( TOL .LE. 0.0 ) GO TO 9000 04MY0
IF ( J1 .LE. 0 ) GO TO 8200 26JAO
IF ( J1 .GT. NJT ) GO TO 8500 26JAO
COMMENT - COMPUTE COORDINATES OF REFERENCE JOINT 29AP0
X(J1) = DX 26JAO
Y(J1) = DY 26JAO
GO TO 1240 26JAO
COMMENT - HOLDING DATA 29AP0
COMMENT - READ FIRST CARD OF TABLE 2 29AP0
1230 READ 16, NJT 26JAO
PRINT 17 26JAO
PRINT 18, NJT 26JAO
1240 CONTINUE 26JAO
IF ( NJT .GT. MNJT ) GO TO 8600 26JAO
PRINT 14 26JAO
N2M1 = NCD2 - 1 26JAO
COMMENT - JJ FOR SECOND AND SUCCEEDING CARDS OF TABLE 2 29AP0
DO 4900 JJ = 1, N2M1 26JAO
READ 17, J1, DX, DY, ( J2( I1 ), I1 = 1, T ) 26JAO
IF ( J1 .GT. NJT ) JNTL = 1 26JAO
NJM2 = 0 26JAO
DO 1270 I1 = 1, 7 26JAO
IF ( J2( I1 ) .GT. NJT ) JNTL = 1 26JAO
1270 IF ( J2( I1 ) .NE. 0 ) NJM2 = NJM2 + 1 26JAO
PRINT 13, J1, DX, DY, ( J2( I1 ), I1 = 1, NJM2 ) 26JAO
IF ( J1 .LE. 0 .OR. J2( I1 ) .LE. 0 ) GO TO 8200 26JAO
COMMENT - CHECK IF FROM JOINT HAS BEEN LOCATED 29AP0
1300 IF ( X(J1) .GT. 1.0E50 ) GO TO 8800 26JAO
IF ( DX .EQ. 0.0 .AND. DY .EQ. 0.0 ) GO TO 8700 26JAO

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COMMENT - IF (JNTL .EQ. 1) GO TO 8500
DO FOR ALL JOINTS SPECIFIED ON THIS CARD
DO 4600 I1= 1,NJNZ
COMMENT - COMPUTE TEMPORARY VALUES OF COORDINATES
5250 XT = X(J1) + DX
YT = Y(J1) + DY
J211 = J2(11)
IF (J211 .LE. 0) GO TO 8200
IF (X(J211) .GT. 1.0E50) GO TO 4000
COMMENT - JOINT PREVIOUSLY LOCATED COMPUTE DIFFERENCE BETWEEN OLD
COMMENT - LOCATION AND NEW LOCATION ERX AND ERY
ERX = ABS(X(J211) - XT)
ERY = ABS(Y(J211) - YT)
IF ( ERX .GT. TOL .OR. ERY .GT. TOL) GO TO 8900
COMMENT - AVERAGE OLD AND NEW COORDINATES
X(J211) = 0.5*(X(J211) + XT)
Y(J211) = 0.5*(Y(J211) + YT)
GO TO 4500
COMMENT - JOINT NOT PREVIOUSLY LOCATED
4000 X(J211) = XT
Y(J211) = YT
4500 CONTINUE
J1 = J211
4600 CONTINUE
4900 CONTINUE
GO TO 9800
8100 PRINT 51
GO TO 9700
8200 PRINT 20
GO TO 9700
8300 PRINT 30
GO TO 9700
8400 PRINT 40
GO TO 9700
8500 PRINT 50
GO TO 9700
8600 PRINT 60
GO TO 9700
8700 PRINT 70, J1
GO TO 9700
8800 PRINT 80, J1
GO TO 9700
8900 PRINT 90, J211, ERX, ERY
GO TO 9700
9000 PRINT 100
9700 IABAN = 1
9800 CONTINUE
PRINT 19
COMMENT - PRINT JOINT COORDINATES AND CHECK FOR JOINT NOT SPECIFIED
DO 9850 I = 1, NJT
IF (X(I) .GT. 1.0E50 .OR. Y(I) .GT. 1.0E50) GO TO 9840
9850 PRINT 21, I, X(I), Y(I)
GO TO 9845
9840 PRINT 22, I
IABAN = 1

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9845 CONTINUE
9850 CONTINUE
9900 CONTINUE
RETURN
END
C
C *****
C
C SUBROUTINE SUBROUTINE SUBROUTINE SUBROUTINE
C *****
C
SUBROUTINE MEMLOC
COMMENT - SUBROUTINE MEMLOC INPUTS LOCATION OF STIFFNESS AND LOAD
COMMENT - TYPES IN FRAME (TABLE 3), CHECKS FOR BAD DATA, COMPUTES MEMBER
COMMENT - NUMBERS, LENGTHS, OFFSETS AND DIRECTION COSINES, ECHO PRINTS DATA
COMMENT - AND PRINTS COMPUTED MEMBER NUMBERS, LENGTHS AND OFFSETS
DIMENSION J2( 10 )
COMMON /BLOCK1/ X( 75), Y( 75), QXX( 75), QYY( 75),
2 QZZ( 75), SXX( 75), SYY( 75), SZZ( 75), DXX( 75),
3 DYY( 75), DZZ( 75), RXX( 75), RYY( 75), RZZ( 75),
4 ERXX( 75), ERY( 75), ERZZ( 75)
COMMON /BLOCK2/ DXS( 50), DYS( 50), ZLS( 50), DCIS( 50),
2 DCZS( 50), PRF( 50), PRAE( 50), MDS( 50), IAROPS( 50),
3 IOPOP( 50), IPINL( 50), IPINR( 50), MCS( 50), SMC( 50, 13)
COMMON /BLOCK3/ DXL( 50), DYL( 50), ZLL( 50), DCIL( 50),
2 DCZL( 50), UOX( 50), UOY( 50), MCDL( 50), IAROPL( 50),
3 MCDL( 50)
COMMON /BLOCK4/ JT1(150), JT2(150), IST(150), LT(150),
2 FORM(150, 6)
COMMON /BLK1/ KEEP2, KEEP3, KEEP4, KEEP5, KEEP6, KEEP7,
2 IYPE, MCD2, MCD3, MCD4, MCD5, MCD6, MCD7,
3 IABAN, IFORM, MM, NJT, NST, NLT, TOL,
4 M, MP1, MP2, ISTT, LTT, IYPEL, IDJ,
5 MLC, IPB, IP9, IP10
COMMON /BLK3/ MJT, MNST, MMLT, MNN, MNC5, MNC6, MDJT, MMLC
6 FORMAT (5X, 3(I5, 1X), 2X, 2I5)
7 FORMAT (5X, 3(I5, 1X), 2X, 2I5, 3E11, 3)
8 FORMAT ( ///, 10X, 40M COMPUTED MEMBER NUMBERS, LENGTHS, AND OFF,
2 ANSETS, //, 40M MEMBER FROM TO STIFF LOAD LENGTH ,
3 25M X-OFFSET Y-OFFSET , /,
4 35M NUMB JOINT JOINT TYPE TYPE, //)
9 FORMAT ( 40M TABLE 3 - MEMBER LOCATION DATA , ///)
10 FORMAT (10X, I5, 3X, I5)
11 FORMAT ( 40M NUMBER OF MEMBER STIFFNESS TYPES = , I5, //,
1 40M NUMBER OF MEMBER LOAD TYPES = , I5, ///)
12 FORMAT ( 5X, I5, 3X, 2I5, 5X, 10I5)
13 FORMAT ( 5X, I5, 5X, 2I5, 5X, 10I5)
14 FORMAT (25X, 26M INPUT OF MEMBER LOCATIONS , //,
2 50M FROM STIFF LOAD TO TO TO TO ,
3 30M TO TO TO TO TO TO , /,
4 35M JOINT TYPE TYPE JOINT, //)
17 FORMAT (40M HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS,
2 15M THE FOLLOWING , //)
18 FORMAT ( ///, 47M *** COMPUTED MEMBER NUMBERS MAY NOT AGREE WITH ,
2 20M LAST PROBLEM *** )

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19 FORMAT (///,50H *** COMPUTED MEMBER NUMBERS AGREE WITH LAST PROBL. 01MYO
2 10HEM *** ) 01MYO
20 FORMAT ( 45H JOINT NUMBERS MUST BE POSITIVE ) 26JAO
23 FORMAT ( 10H NONE ) 26JAO
25 FORMAT ( 32H MEMBER WITH STIFFNESS TYPE ,15, 9H AND LOAD, 26JAO
2 5H TYPE,15,/,32H WAS SPECIFIED AS GOING FROM, 26JAO
3 7H JOINT ,15, 9H TO JOINT,15,/,17H PROGRAM DOES, 26JAO
4 36H NOT ALLOW THIS ORDER TO BE REVERSE) 26JAO
27 FORMAT ( 51H TYPE 3 PROBLEM DOES NOT ALLOW ANY CHANGE IN ST,26JAO
2 31H STIFFNESS TYPES FROM LAST PROBLEM,/, 26JAO
3 38H LAST PROBLEM DID NOT SPECIFY TYPE,15, 26JAO
4 20H STIFF BETWEEN JOINT,15,10H AND JOINT,15 ) 26JAO
30 FORMAT ( 40H NO DATA HELD OR READ IN TABLE 3 ) 26JAO
31 FORMAT ( 50H NUMBER OF CARDS IN TABLE 3 MAY NOT EQUAL 1 ) 09MYO
50 FORMAT ( 43H JOINT NUMBER ABOVE GREATER THAN NUMBER, 26JAO
2 20H OF JOINTS IN FRAME ) 26JAO
61 FORMAT ( 51H NUMBER OF STIFFNESS TYPES GREATER THAN STORAGE,26JAO
2 7H ALLOWS) 26JAO
62 FORMAT ( 46H NUMBER OF LOAD TYPES GREATER THAN STORAGE, 26JAO
2 7H ALLOWS) 26JAO
71 FORMAT ( 46H STIFFNESS AND LOAD TYPES MUST BE POSITIVE, 26JAO
2 8 NUMBERS) 26JAO
72 FORMAT ( 51H STIFFNESS OR LOAD TYPE ABOVE GREATER THAN TOTAL,26JAO
2 50H NUMBER OF STIFFNESS OR LOAD TYPES SPECIFIED ABOVE) 26JAO
73 FORMAT ( 51H YOU CANNOT HOLD UP THE LOAD WITHOUT SOME STIFF,26JAO
2 50HNESS - IF STIFF TYPE = 0 - LOAD TYPE MUST = 0 ) 26JAO
74 FORMAT ( 5H 74) 26JAO
91 FORMAT ( 50H ERROR IN OFFSETS FOR MEMBER OF STIFFNESS TYPE, 26JAO
2 15,/,47H THE X AND Y OFFSETS FOR THE MEMBER BETWEEN, 26JAO
3 7H JOINTS,15, 5H AND ,15,/, 26JAO
4 49H DO NOT AGREE WITH PREVIOUSLY DEFINED OFFSETS, 26JAO
5 46H FOR A MEMBER OF THIS TYPE, WITHIN THE ALLOWED,/, 26JAO
6 51H ERROR OF TWO TIMES THE JOINT LOCATION TOLERANC,26JAO
7 1HE) 26JAO
92 FORMAT ( 45H ERROR IN OFFSETS FOR MEMBER OF LOAD TYPE, 26JAO
2 15,/,47H THE X AND Y OFFSETS FOR THE MEMBER BETWEEN, 26JAO
3 7H JOINTS,15, 5H AND ,15,/, 26JAO
4 49H DO NOT AGREE WITH PREVIOUSLY DEFINED OFFSETS, 26JAO
5 46H FOR A MEMBER OF THIS TYPE, WITHIN THE ALLOWED,/, 26JAO
6 51H ERROR OF TWO TIMES THE JOINT LOCATION TOLERANC,26JAO
7 1HE) 26JAO
COMMENT - PRINT TABLE HEADINGS 26JAO
PRINT 9 01MYO
IF (INCD3 .EQ. 1) GO TO 8100 26JAO
IF (INCD3 .LE. 0 .AND. KEEPS .LE. 0 ) GO TO 8300 09MYO
TT0 = 2.0*TM. 26JAO
COMMENT - SET OFFSETS FOR STIFF TYPES 26JAO
DO 1100 I = 1, MNST 03APO
1100 DXS(I) = DYS(I) = 1.01E50 03APO
COMMENT - SET OFFSETS FOR LOAD TYPES 01MYO
DO 1110 I = 1, MNLT 03APO
1110 DXL(I) = DYL(I) = 1.01E50 03APO
IF (KEEPS .NE. 1) GO TO 1150 25APO
PRINT 17 26JAO
1150 CONTINUE 26JAO
COMMENT - NM IS NUMBER OF MEMBERS ACCUMULATED AND MUST NOT CHANGE 01MYO

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COMMENT - FROM PREVIOUS PROBLEM FOR OFFSPRING PROBLEMS 01MYO
IF (KEEPS .EQ. 1 .OR. ITYPE .EQ. 3) GO TO 1160 25APO
NM = 0 25APO
1160 CONTINUE 25APO
IF (INCO3 .NE. 0) GO TO 1180 03APO
PRINT 23 26JAO
GO TO 6000 3APO
1180 JNLT = 0 26JAO
1250 CONTINUE 26JAO
COMMENT - READ FIRST CARD IN TABLE 3 01MYO
READ 10,NST,NLT 26JAO
PRINT 11, NST,NLT 26JAO
IF (NST .GT. MNST) GO TO 8610 26JAO
IF (NLT .GT. MNLT) GO TO 8620 26JAO
PRINT 14 26JAO
N3M) = NCD3 - 1 26JAO
DO 4900 JJ = 1,N3M) 26JAO
COMMENT - READ 2ND AND SUCCEEDING CARDS IN TABLE 3 01MYO
READ 12,J1,ISTT,LTT,(J2(11),I1 = 1,10) 26JAO
IF (J1 .GT. NJT) JNLT = 1 26JAO
NJNZ = 0 26JAO
DO 1270 I1 = 1,10 26JAO
IF (J2(11) .GT. NJT) JNLT = 1 26JAO
IF (J2(11) .NE. 0) NJNZ = NJNZ + 1 30APO
CONTINUE 30APO
1270 CONTINUE 01MYO
COMMENT - PRINT 2ND AND SUCCEEDING CARDS IN TABLE 3 26JAO
PRINT 13,J1,ISTT,LTT, (J2(11),I1 = 1,NJNZ ) 26JAO
IF (J1 .LE. 0) GO TO 8200 30APO
IF (JNLT .EQ. 1) GO TO 8500 26JAO
IF (11STT .LT. 0 .OR. LTT .LT. 0) GO TO 8710 26JAO
IF (11STT .GT. NST .OR. LTT .GT. NLT) GO TO 8720 26JAO
IF (11STT .EQ. 0 .AND. LTT .NE. 0) GO TO 8730 26JAO
COMMENT - DO FOR NUMBER OF MEMBERS SPECIFIED ON ONE CARD 01MYO
DO 4500 I1 = 1,NJNZ 26JAO
J211 = J2(11) 26JAO
IF (J211 .LE. 0) GO TO 8200 30APO
IF (KEEPS .NE. 1 .AND. ITYPE .NE. 3) GO TO 4425 25APO
NML = NM 25APO
COMMENT - DO FOR EACH MEMBER 01MYO
DO 4400 K = 1,NM 26JAO
IF (J1 .EQ. JT1(K) .AND. J211 .EQ. JT2(K) ) GO TO 4410 26JAO
IF (J1 .EQ. JT2(K) .AND. J211 .EQ. JT1(K) ) GO TO 8750 26JAO
4400 CONTINUE 26JAO
GO TO 4425 26JAO
COMMENT - OLD MEMBER (PREVIOUSLY GIVEN STIFF AND LOAD TYPE) 01MYO
COMMENT - CAN NOT CHANGE STIFF TYPE FOR OFFSPRING PROBLEM 01MYO
4410 IF (ITYPE .EQ. 3 .AND. 1ST(K) .NE. 1STT) GO TO 8270 26JAO
1ST(K) = 1STT 26JAO
LTT(K) = LTT 26JAO
GO TO 4450 26JAO
COMMENT - NEW MEMBER INCREASE NM 01MYO
4425 NM = NM + 1 26JAO
JT1(NM) = J1 26JAO
JT2(NM) = J211 26JAO
1ST(NM) = 1STT 26JAO
LTT(NM) = LTT 26JAO

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      GO TO 9700
8570 PRINT 57
      GO TO 9700
8580 PRINT 58
      GO TO 9700
8650 PRINT 65
      GO TO 9700
8660 PRINT 60
      GO TO 9700
8670 PRINT 67
      GO TO 9700
8710 PRINT 71
      GO TO 9700
8720 PRINT 72
9700     IABAN = 1
9900     CONTINUE
      NSTL = NST
RETURN
END
C
C *****
C
C SUBROUTINE SUBROUTINE SUBROUTINE SUBROUTINE
C *****
C
SUBROUTINE RMDL
COMMENT - SUBROUTINE RMDL INPUTS MEMBER LOAD DATA (TABLE 6) CHECKS
COMMENT - FOR BAD DATA, CONVERTS LOADS AND DISTANCES TO MEMBER
COMMENT - COORDINATES AND ECHO PRINTS DATA
COMMON /BLOCK3/ DRL( 50), DYL( 50), ZLL( 50), DCL( 50),
2 DCZL( 50), UOX( 50), UOY( 50), MCDL( 50), IAXOPL( 50),
3 NC6( 50)
COMMON /BLOCK6/ XLL(150), XRL(150), OXL(150), OYL(150),
2 OZL(150)
COMMON /BLK1/ KEEP2, KEEP3, KEEP4, KEEP5, KEEP6, KEEP7,
2 ITYPE, MCD2, MCD3, MCD4, MCD5, MCD6, MCD7,
3 IABAN, IFORM, NM, NJT, NST, MLT, TOL,
4 N, NP1, NP2, ISTT, LTT, IYPEL, IDJ,
5 MLC, IP8, IP9, IP10
COMMON /BLK3/ MNJT, MNST, MNLT, MNM, MNCS, MNC6, MNJT, MNLC
9 FORMAT ( 40H TABLE 6 - MEMBER LOAD DATA , ///)
12 FORMAT ( 5X, I5, 20X, ZE10.3, 5X, 215 )
13 FORMAT ( 5X, I5, 2E11.3, 215 )
14 FORMAT ( ///, 50H LOAD UNIFORM UNIFORM NO AXIS , 26JAD
2 /, 50H TYPE OX OY CAPDS OPT , 26JAD
3 //)
15 FORMAT (10X, 5E10.3)
16 FORMAT ( 5X, 5E11.3)
17 FORMAT ( 40H HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS,
2 15H THE FOLLOWING , //)
18 FORMAT ( ///, 20H LOAD TYPE , 15, 6H CONTD., //, 2X,
2 49H FROM TO OX OY , 26JAD
3 30H OZ , //)
23 FORMAT ( 10H NONE )
24 FORMAT ( 20H NO DATA )

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52 FORMAT ( 50H LOAD SPECIFIED BEYOND THE END OF MEMBER ) 31MR0
53 FORMAT ( 50H LOAD SEQUENCE MUST BE LONGER THAN 1/40 SPAN ) 31MR0
54 FORMAT ( 51H LOAD SPECIFIED AT NEGATIVE DISTANCE ALONG MEMB ) 31MR0
55 FORMAT ( 49H TO DISTANCE OF ZERO IMPLIES THAT IT IS FIRST,
2 55H CARD OF TWO CARD SEQUENCE AND NEXT CARD WILL HAVE FROM, //,
3 20H DISTANCE OF ZERO ) 21AP0
56 FORMAT ( 50H ALL CARDS SPECIFIED FOR TABLE & READ BUT ALL ,
2 25HLOAD TYPES NOT SPECIFIED , //, 30AP0
3 46H CHECK CARD COUNT AND NUMBER OF LOAD TYPES ) 30AP0
57 FORMAT ( 48H NO CARDS IN TABLE & BUT ALL LOAD TYPES NOT ,
2 10H SPECIFIED ) 29AP0
58 FORMAT ( 51H ALL LOAD TYPES SPECIFIED BUT ALL CARDS NOT READ ) 30AP0
2 //, 46H CHECK CARD COUNT AND NUMBER OF LOAD TYPES ) 30AP0
59 FORMAT ( 40H AXIS OPTION MUST BE 1, 2, 3, OR 4 ) 10AP0
60 FORMAT ( 51H NUMBER OF CARDS TO FOLLOW MUST NOT BE NEGATIVE ) 10AP0
61 FORMAT ( 48H CONCENTRATED LOADS AT 0.0 ARE NOT PERMITTED ) 04MY0
65 FORMAT ( 45H LOAD TYPES MUST BE IN ASCENDING ORDER ) 31MR0
67 FORMAT ( 50H IF 2ND CARD USED FOR LOAD TYPE, UNIFORM LOAD ,
2 20HVALUES MUST BE 0.0 ) 31MR0
71 FORMAT ( 36H LOAD TYPES MUST NOT BE NEGATIVE ) 31MR0
72 FORMAT ( 48H LOAD TYPE GREATER THAN TOTAL NUMBER OF LOAD,
2 16H TYPES SPECIFIED ) 31MR0
COMMENT - PRINT TABLE HEADING
PRINT 9
IF (KEEP6 .EQ. 0) MLTL = 0
IF (MCD6 .EQ. 0 .AND. KEEP6 .EQ. 0) GO TO 1110
GO TO 1120
1110 PRINT 24
IF (MLT .NE. 0) GO TO 8570
GO TO 9900
1120 IF (MCD6 .NE. 0) GO TO 1150
IF (MLT .NE. MLTL) GO TO 8570
PRINT 17
PRINT 23
GO TO 9900
CONTINUE
IF (KEEP6 .EQ. 1) GO TO 1240
COMMENT - ALL NEW DATA
1160 DO 1200 I = 1, MNLT
MCD(1:1) = -1
1200 MCDL(1) = -1
MCD6 = 0
GO TO 1250
1240 PRINT 17
IF (MLTL .EQ. 0) GO TO 1160
CONTINUE
PRINT 14
MCD6 = 0
COMMENT - DO FOR EACH LOAD TYPE
DO 490, JJ = 1, MLT
COMMENT - SKIP FOR LOAD TYPES HELD FROM PREVIOUS PROBLEM
IF (MCDL(JJ) .NE. -1) GO TO 4900
IF (JJ .EQ. 1) GO TO 1300
IF (JJ .EQ. MLTL + 1) GO TO 1300
IF (MCDL(JJ) = 1) .GT. 0) PRINT 14
1300 CONTINUE

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IF INCR6 .EQ. MCD6) GO TO 8560
COMMENT - READ AND PRINT FIRST CARD FOR LOAD TYPE
READ 12, LTT, UOXT, UOYT, MCDLT, IAXOPT
PRINT 13, LTT, UOXT, UOYT, MCDLT, IAXOPT
IF (IAXOPT .LT. 1 .OR. IAXOPT .GT. 4) GO TO 8590
  NCR6 = NCR6 + 1
  IF (LTT .GT. NLT) GO TO 8720
  IF (LTT .LT. 0) GO TO 8710
  IF (IJJ .NE. LTT) GO TO 8650
  IF (MCDLT .LT. 0) GO TO 8600
  IF (MCDLT .GT. 0) GO TO 2400
COMMENT - UNIFORM LOADS ONLY
IF (IAXOPT .EQ. 1) GO TO 1500
IF (IAXOPT .EQ. 2) GO TO 1400
COMMENT - AXIS OPTION 3 OR 4 - CONVERT UNIFORM LOADS TO DIRECTIONS
COMMENT - AND INTENSITY OF MEMBER AXES
  TEMP1 = ABS(DC1L(LTT))
  TEMP2 = ABS(DC2L(LTT))
  UOX(LTT) = UOXT*DC1L(LTT)*TEMP2 +
  UOYT*DC2L(LTT)*TEMP1
  UOY(LTT) = -UOXT*DC2L(LTT)*TEMP2 +
  UOYT*DC1L(LTT)*TEMP1
  GO TO 1600
COMMENT - AXIS OPTION 2 - CONVERT UNIFORM LOADS TO DIRECTIONS OF
COMMENT - MEMBER AXES
1400  UOX(LTT) = UOXT*DC1L(LTT) + UOYT*DC2L(LTT)
      UOY(LTT) = -UOXT*DC2L(LTT) + UOYT*DC1L(LTT)
      GO TO 1600
COMMENT - AXIS OPTION 1 - LOADS ALLREADY IN MEMBER AXES
1500  UOX(LTT) = UOXT
      UOY(LTT) = UOYT
      MCDL(LTT) = 0
      IAXOPT(LTT) = IAXOPT
      GO TO 4900
COMMENT - VARIABLE LOADING
2400  CONTINUE
      IF (UOXT .NE. 0 .OR. UOYT .NE. 0) GO TO 8670
      MCDL(LTT) = MCDLT
      IAXOPT(LTT) = IAXOPT
      PRINT 18, LTT
COMMENT - DO FOR EACH ADDITIONAL CARD OF LOAD TYPE
DO 4500 II = 1, MCDLT
  NCR6 = NCR6 + 1
  IF (III .EQ. 1) MCR6(LTT) = NCR6
  IF (INCR6 .EQ. MCD6) GO TO 8560
COMMENT - READ AND PRINT UNIFORM LOAD DATA
READ 15, XLL( INC6), XRL( INC6), OXLT, OYLT, OZL( INC6)
PRINT 16, XLL( INC6), XRL( INC6), OXLT, OYLT, OZL( INC6)
  NCR6 = NCR6 + 1
  TH = ZLL(LTT)/H
COMMENT - CONVERT DISTANCES TO MEMBER COORDINATES
GO TO (2800, 2800, 2700, 2600), IAXOPT
2600  XLL( INC6) = XLL( INC6)/DC2L(LTT)
      XRL( INC6) = XRL( INC6)/DC2L(LTT)
      GO TO 2800
2700  XLL( INC6) = XLL( INC6)/DC1L(LTT)

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29APO  XRL( INC6) = XRL( INC6)/DC1L(LTT)
05MYO  CONTINUE
04MYO  CHECK FOR ILLEGAL DATA
26JAO  IF (XLL( INC6) .LT. 0.0) GO TO 8540
30APO  IF (XRL( INC6) .GT. ZLL(LTT) + 0.1*TH) GO TO 8520
29APO  IF (XRL( INC6) .EQ. 0.0) GO TO 2838
26JAO  IF (III .EQ. 1) GO TO 2820
26JAO  IF (XRL( INC6 - 1) .NE. 0.0) GO TO 2820
04MYO  IF (XLL( INC6) .NE. 0.0) GO TO 8550
21APO  DEL = XRL( INC6) - XLL( INC6 - 1)
21APO  GO TO 2830
21APO  DEL = XRL( INC6) - XLL( INC6)
21APO  IF (DEL .EQ. 0.0) GO TO 2840
21APO  IF (DEL .LE. TH) GO TO 8530
21APO  GO TO 2840
2820  DEL = 1.0
21APO  IF (III .EQ. 1) GO TO 2840
21APO  IF (XLL( INC6) .EQ. 0.0 .AND. XRL( INC6 - 1) .EQ. 0.0) GO TO 8610
21APO  CONTINUE
21APO  IF (IAXOPT .EQ. 1) GO TO 2900
21APO  IF (IAXOPT .EQ. 2 .OR. DEL .EQ. 0.0) GO TO 2850
21APO  COMMENT - AXIS APTIONS 3 OR 4 - CONVERT DISTRIBUTED LOADS TO DIRECTIONS
05MYO  COMMENT - AND INTENSITY OF MEMBER AXES
      TEMP1 = ABS(DC1L(LTT))
      TEMP2 = ABS(DC2L(LTT))
      OXL( INC6) = OXLT*DC1L(LTT)*TEMP2 +
      OYLT*DC2L(LTT)*TEMP1
      OYL( INC6) = -OXLT*DC2L(LTT)*TEMP2 +
      OYLT*DC1L(LTT)*TEMP1
      GO TO 2950
2850  CONTINUE
COMMENT - AXIS OPTION 2 OR CONCENTRATED LOADS - CONVERT DISTRIBUTED
COMMENT - AND CONCENTRATED LOADS TO DIRECTIONS OF MEMBER AXES
      OXL( INC6) = OXLT*DC1L(LTT) + OYLT*DC2L(LTT)
      OYL( INC6) = -OXLT*DC2L(LTT) + OYLT*DC1L(LTT)
      GO TO 2950
2900  CONTINUE
COMMENT - AXIS OPTION 1 - LOADS ALLREADY IN MEMBER AXES
      OXL( INC6) = OXLT
      OYL( INC6) = OYLT
2950  CONTINUE
4500  CONTINUE
4900  CONTINUE
      IF (INCR6 .LT. MCD6) GO TO 8580
      GO TO 9900
8520  PRINT 52
      GO TO 9700
8530  PRINT 53
      GO TO 9700
8540  PRINT 54
      GO TO 9700
8550  PRINT 55
      GO TO 9700
8560  PRINT 56
      GO TO 9700
8570  PRINT 57
26JAO  26JAO
26JAO  26JAO
05MYO  05MYO
21APO  21APO
21APO  21APO
29APO  29APO
29APO  29APO
26JAO  26JAO
26JAO  26JAO

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          SMNT(13) = FMM(16)                26JAO
9900    CONTINUE                            26JAO
        RETURN                               26JAO
        END                                  26JAO
C
C *****
C
C          SUBROUTINE          SUBROUTINE          SUBROUTINE          SUBROUTINE
C          *****
C
SUBROUTINE DISCST ( NC51T, NCDST, ZL, L1)    11FE0
COMMENT - SUBROUTINE DISCST DISCRETIZES MEMBER STIFFNESS DATA F, AE. 22MY0
COMMENT - SX, SY, AND SZ
COMMON /BLOCK9/ XLS( 75), XRS( 75), FL( 75), AEL( 75), 26JAO
              SYL( 75), SYL( 75), SZL( 75) 26JAO
COMMON /BLOCK7/ FI( 42), AEI( 42), SX( 42), SY( 42), 26JAO
              OXI( 42), OYI( 42), OZI( 42), AI( 42), 26JAO
              BI( 42), AI( 42), DI( 42), DX( 42), DY( 42), 26JAO
              DZ( 42), U1( 42), V1( 42), W1( 42), U2( 42), 13MR0
              V2( 42), W2( 42), ERX( 42), ERY( 42), ERZ( 42), 13MR0
              RX( 42), RY( 42), RZ( 42) 13MR0
COMMON /BLK1/ KEEP2, KEEP3, KEEP4, KEEP5, KEEP6, KEEP7, 26JAO
              ITYPE, NCD2, NCD3, NCD4, NCD5, NCD6, NCD7, 26JAO
              IABAN, IFORM, NH, NJT, NST, NLT, TOL, 26JAO
              M, MP1, MP2, ISTT, LTT, IYPEL, IDJ, 12FE0
              MLC, IP8, IP9, IP10 13FE0
COMMON /BLK2/ XL, XR, X1, X2, I1, I2, NO, H, TH, MSQ, MCU, XZL 26JAO
COMMENT - ZERO MEMBER STIFFNESS DATA 13MY0
DO 102D I = 1, MP2 26JAO
      SX(I) = 0.0 26JAO
      SY(I) = 0.0 26JAO
      SZ(I) = 0.0 26JAO
      FI(I) = 0.0 26JAO
      AE(I) = 0.0 26JAO
1020    CONTINUE 26JAO
      ICOUNT = 0 26JAO
      NC52T = NC51T - 1 + NCDST 26JAO
COMMENT - I1 GOES FROM NC51T TO NC52T 13MY0
      I1 = NC51T - 1 26JAO
1050    I1 = I1 + 1 26JAO
COMMENT - READ DATA FROM ONE CARD IMAGE (STIFFNESS AT LEFT OF SECTION) 13MY0
      XL = XLS(I1) 26JAO
      XR = XRS(I1) 26JAO
      FL = FL(I1) 26JAO
      AEL = AEL(I1) 26JAO
      SYL = SYL(I1) 26JAO
      SYLT = SYL(I1) 26JAO
      SZLT = SZL(I1) 26JAO
      IF (XR .NE. 0.0) GO TO 1100 26JAO
COMMENT - VARIABLE STIFFNESS SECTION READ ONE CARD IMAGE (STIFFNESS AT 13MY0
COMMENT - RIGHT OF SECTION) 13MY0
      I1 = I1 + 1 26JAO
      XR = XRS(I1) 26JAO
      FL = FL(I1) 26JAO
      AERT = AEL(I1) 26JAO

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      SKRT = SKL(I1) 26JAO
      SYRT = SYL(I1) 26JAO
      SZRT = SZL(I1) 26JAO
      GO TO 1110 26JAO
COMMENT - UNIFORM STIFFNESS SECTION SET STIFFNESS ON RIGHT EQUAL TO 13MY0
COMMENT - STIFFNESS ON LEFT 13MY0
1100    FRT = FLT 26JAO
      AERT = AEL 26JAO
      SKRT = SKLT 26JAO
      SYRT = SYLT 26JAO
      SZRT = SZLT 26JAO
1110    CONTINUE 26JAO
      IF (ICOUNT .NE. 0) GO TO 1210 26JAO
COMMENT - FIRST SECTION OF MEMBERS STIFFNESS DATA 13MY0
      ICOUNT = 1 26JAO
      I1 = 2 26JAO
      X1 = TH 26JAO
      GO TO 1250 26JAO
1210    CONTINUE 11MY0
      I1 = I2 + 1 26JAO
      X1 = TH - X2 26JAO
1250    CONTINUE 26JAO
      IF (XR .NE. ZL) GO TO 1260 26JAO
COMMENT - LAST SECTION OF MEMBERS STIFFNESS DATA 13MY0
      I2 = MP1 26JAO
      X2 = 0.0 26JAO
      GO TO 1270 26JAO
1260    ZI2 = XR/TH + 1.0 26JAO
      I2 = ZI2 26JAO
      X2 = XR - I2*TH + TH 26JAO
      NO = I2 - I1 26JAO
1270    COMMENT - SUBROUTINE LINSTF DISTRIBUTES F AND AE 13MY0
      CALL LINSTF ( FLT, FRT, F, FIT2, L1 ) 11FE0
      CALL LINSTF ( AELT, AERT, AE, AETT2, L1 ) 11FE0
COMMENT - SUBROUTINE LINLD DISTRIBUTES SX, SY, SZ, OX, OY, AND OZ 13MY0
      IF (SKLT .EQ. 0.0 .AND. SKRT .EQ. 0.0) GO TO 1280 11MY0
      CALL LINLD ( SKLT, SKRT, SX, L1 ) 11FE0
1280    IF (SYLT .EQ. 0.0 .AND. SYRT .EQ. 0.0) GO TO 1290 11MY0
      CALL LINLD ( SYLT, SYRT, SY, L1 ) 11FE0
1290    IF (SZLT .EQ. 0.0 .AND. SZRT .EQ. 0.0) GO TO 1330 11MY0
      CALL LINLD ( SZLT, SZRT, SZ, L1 ) 11FE0
1330    CONTINUE 26JAO
      XZL = XZL 26JAO
COMMENT - RETURN FOR IMAGE OF NEXT DATA CARD IF I1 LESS THAN NC52T 13MY0
9000    IF (I1 .LT. NC52T) GO TO 1050 26JAO
9900    CONTINUE 26JAO
        RETURN 26JAO
        END 26JAO
C
C *****
C
C          SUBROUTINE          SUBROUTINE          SUBROUTINE          SUBROUTINE
C          *****
C
SUBROUTINE LINSTF (STL, STR, ST, STT2, L1)    11FE0

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COMMENT - SUBROUTINE LINSTF DISTRIBUTES F AND AE          22MYO
DIMENSION ST(11)                                         11FE0
COMMON /BLK2/ XL,XR,X1,X2,I1,I2,NO,H,TH,MSQ,MCU,XZL      26JAO
IF ( XL .EQ. 0.0 ) GO TO 1150                            26JAO
COMMENT - FIRST SECTION OF MEMBER                        13MYO
XZL = 0.0                                                 26JAO
STT2 = 1.0                                               26JAO
1150 CONTINUE                                           26JAO
IF ( STR .EQ. STL ) GO TO 1310                            26JAO
COMMENT - LINEAR STIFFNESS SECTION                       13MYO
COMMENT - CALCULATE SLOPE OF LINEAR STIFFNESS VARIATION 13MYO
DS = (STR - STL)/(XR-XL)                                  26JAO
COMMENT - FIRST ELEMENT (TH LONG) OF SECTION             13MYO
COMMENT - COMPUTE EFFECTIVE STIFFNESS OF ELEMENT CONSIDERING JUMP AT 13MYO
COMMENT - START OF SECTION                               15MYO
ST1 = STL                                                26JAO
ST2 = STL + DS*X1                                         26JAO
STT1 = 0.5*(ST1 + ST2)                                   26JAO
ST(I1) = (TH*STT1*STT2)/(XZL*STT1 + X1*STT2)           26JAO
IF (NO .EQ. 0 ) GO TO 1250                               26JAO
IIP1 = I1 + 1                                             26JAO
IIPNO = I1 + NO                                          26JAO
COMMENT - REMAINING NO ELEMENTS                          13MYO
COMMENT - COMPUTE STIFFNESS AT MID POINT OF ELEMENT     15MYO
DO 1210 I = IIP1, IIPNO                                  26JAO
ST1 = ST2                                                26JAO
ST2 = ST1 + DS*TH                                         26JAO
1210 ST(I) = 0.5*(ST1 + ST2)                             26JAO
1250 CONTINUE                                           26JAO
ST1 = ST2                                                26JAO
ST2 = STR                                                26JAO
STT2 = 0.5*(ST1 + ST2)                                   26JAO
1300 GO TO 1000                                           26JAO
COMMENT - UNIFORM STIFFNESS SECTION                     13MYO
COMMENT - FIRST ELEMENT (TH LONG) OF SECTION            13MYO
COMMENT - COMPUTE EFFECTIVE STIFFNESS OF ELEMENT CONSIDERING JUMP AT 13MYO
COMMENT - START OF SECTION                               15MYO
1310 STT1 = STL                                           26JAO
ST(I1) = (TH*STT1*STT2)/(XZL*STT1 + X1*STT2)           26JAO
IF (NO .EQ. 0 ) GO TO 1360                               26JAO
IIP1 = I1 + 1                                             26JAO
IIPNO = I1 + NO                                          26JAO
COMMENT - REMAINING NO ELEMENTS HAVE CONSTANT STIFFNESS 15MYO
DO 1350 I = IIP1, IIPNO                                  26JAO
1350 ST(I) = STL                                         26JAO
1360 STT2 = STL                                           26JAO
1000 CONTINUE                                           26JAO
RETURN                                                  26JAO
END                                                       26JAO

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C
C *****
C SUBROUTINE SUBROUTINE SUBROUTINE SUBROUTINE
C *****
C

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SUBROUTINE LIMLD ( QL, OR, O, L1)                        11FE0
COMMENT - SUBROUTINE LIMLD DISTRIBUTES SX, SY, SZ, QX, QY, AND QZ 15MYO
DIMENSION Q(11)                                         11FE0
COMMON /BLK2/ XL,XR,X1,X2,I1,I2,NO,H,TH,MSQ,MCU,XZL    26JAO
COMMENT - COMPUTE SLOPE OF LINEAR VARIATION             15MYO
DO = (OR - Q1)/XR - XL                                  26JAO
COMMENT - COMPUTE CONCENTRATED LOAD OR RESTRAINT FOR ELEMENT AT RIGHT 26JAO
COMMENT - END OF SECTION Q1, DISTANCE TO LINE OF ACTION Z AND CALL 15MYO
COMMENT - COMULD TO DISTRIBUTE TO ADJACENT STATIONS     15MYO
Q2 = OR                                                  26JAO
Q1 = OR - DO*X2                                         26JAO
IF ( ABS(Q1 + Q2) .LE. 1.0E-10 ) GO TO 1005             26JAO
Z = XR - X2 + (X2/3.0)*(2.0*Q2 + Q1)/(Q1 + Q2)         26JAO
Q1 = 0.5*X2*(Q1 + Q2)                                   26JAO
IF ( ABS(Q1) .LE. 1.0E-10 ) GO TO 1005                 26JAO
CALL COMULD ( Q1, Z, O, L1 )                            11FE0
COMMENT - SAME AS ABOVE FOR ELEMENT AT LEFT END OF SECTION 15MYO
1005 Q1 = QL                                             26JAO
Q2 = QL + DO*X1                                         26JAO
IF ( ABS(Q1 + Q2) .LE. 1.0E-10 ) GO TO 1009            26JAO
Z = XL + (X1/3.0)*(2.0*Q2 + Q1)/(Q1 + Q2)               26JAO
Q1 = 0.5*X1*(Q1 + Q2)                                   26JAO
IF ( ABS(Q1) .LE. 1.0E-10 ) GO TO 1009                 26JAO
CALL COMULD ( Q1, Z, O, L1 )                            11FE0
1009 IF ( NO .EQ. 0 ) GO TO 2000                         26JAO
1020 KX = XL + X1                                         26JAO
COMMENT - SAME AS ABOVE FOR REMAINING NO ELEMENTS       15MYO
DO 1990 I1 = 1,NO                                       26JAO
Q1 = Q2                                                  26JAO
Q2 = Q1 + DO*TH                                         26JAO
IF ( ABS(Q1 + Q2) .LE. 1.0E-10 ) GO TO 1990            26JAO
Z = KX + (TH/3.0)*(2.0*Q2 + Q1)/(Q1 + Q2)               26JAO
KX = KX + TH                                             26JAO
Q1 = 0.5*TH*(Q1 + Q2)                                   26JAO
CALL COMULD ( Q1, Z, O, L1 )                            11FE0
1990 CONTINUE                                           26JAO
2000 CONTINUE                                           26JAO
RETURN                                                  26JAO
END                                                       26JAO

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C
C *****
C SUBROUTINE SUBROUTINE SUBROUTINE SUBROUTINE
C *****
C
SUBROUTINE COMULD ( Q1, Z, O, L1)                        11FE0
COMMENT - SUBROUTINE COMULD DISTRIBUTES CONCENTRATED LOAD OR ELASTIC 15MYO
COMMENT - SPRING RESTRAINT TO ADJACENT STATIONS I AND IP1 15MYO
DIMENSION Q(11)                                         11FE0
COMMON /BLK2/ XL,XR,X1,X2,I1,I2,NO,H,TH,MSQ,MCU,XZL    26JAO
Z1 = Z/TH + 1.0                                         26JAO
I = Z1                                                  26JAO
C = Z - I*TH + TH                                       01JLO
IP1 = I + 1                                             01JLO
QO(I) = QO(I) + Q1*(TH - C)/TH                          01JLO

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299      IZ = IZ - 1
        IZ = IZ + 1
299      II = IX + IZ
        IE = IS
DO 300 I = 12, M
      RM(I,II) = SU(I+1)
      II = II - 1
300      CONTINUE
        RO(J) = SU(I)
DO 400 L = 1, J1
      TEMP = RM(M, J-L) * RM(M-L, J)
      IF (J-M-L, LE, ML) RM(L, M+J-L) = RM(L, M+J-L) + SL(L) * TEMP
      LXX = 1
      IF (IE.GT.1) LXX = IE
      IF (IE.GT.1) IE = IE - 1
      LXX = LXX + L
DO 350 I = LXX, M
      RM(I, J+M-1) = RM(I, J+M-1) + RM(I-L, J+M-1) * TEMP
350      CONTINUE
400      CONTINUE
      IF (J.GT.M) RM(M, J) = RM(M, J) + SL(M) * RM(M, J-M) * SL(M)
      RM(M, J) = -1.0 / RM(M, J)
C      COMPUTE PRELIMINARY VALUE FOR W(J)
750      W(J) = 0.0
DO 800 I = 1, J1
      W(J) = W(J) + RM(M-I, J) * W(J-I)
800      CONTINUE
      IF (J.GT.M) W(J) = W(J) + RO(J-M) * W(J-M)
      W(J) = RM(M, J) * ( W(J) - F )
1000     CONTINUE
C*****
C      CALCULATE RECURSION EQUATION
C*****
      K = 0
DO 3000 L = 1, NLM1
      J = ML - L
      TEMP = W(J)
      W(J) = 0
      K = K + 1
DO 2100 I = 1, M1
      W(J) = W(J) + RM(M-1, J+1) * W(J+1)
      IF (I.EQ.K) GO TO 2200
2100     CONTINUE
2200     IF (J.LE.NLMM) W(J) = W(J) + RO(J) * W(J+M)
      W(J) = RM(M, J) * W(J) + TEMP
3000     CONTINUE
RETURN
END

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C*****
C      SUBROUTINE          SUBROUTINE          SUBROUTINE          SUBROUTINE
C*****
SUBROUTINE FSUB1 ( SU, FF, M, L4 )

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COMMENT - SUBROUTINE FSUB1 CALLS FSUB11 FOR FRAME SOLUTION AND FSUB12
COMMENT - FOR MEMBER SOLUTIONS
DIMENSION SU(L4)
COMMON /BLK5/ NFSUB
      IF (NFSUB.EQ. 11) CALL FSUB11 (SU, FF, L4, M)
      IF (NFSUB.EQ. 12) CALL FSUB12 (SU, FF, L4)
RETURN
END
C*****
C      SUBROUTINE          SUBROUTINE          SUBROUTINE          SUBROUTINE
C*****
SUBROUTINE FSUB12 ( SU, FF, L4 )
COMMENT - SUBROUTINE FSUB12 FURNISHES RIGHT SIDE OF SYMMETRIC STIFFNESS
COMMENT - MATRIX SU AND LOAD TERM F TO GRIP2A FOR MEMBER SOLUTION
DIMENSION SU (L4 )
COMMENT - JJ IS EQUATION NUMBER - SU(I) IS LAST TERM IN BAND ON RIGHT
COMMON /BLOCK7/ F( 42), AE( 42), SX( 42), SY(42),
2 SZ( 42), OX( 42), OY( 42), QZ( 42), AI( 42),
3 BI( 42), AI( 42), DI( 42), DX( 42), DY( 42),
4 UZ(42), U1(42), V1(42), W1(42), U2(42),
5 V2(42), W2(42), ERX(42), ERY(42), ERZ(42),
6 RX(42), RY(42), RZ(42)
COMMON /BLK2/ XL, XR, X1, X2, I1, I2, NO, H, TH, HSO, HCU, X2L
COMMON / RI / ML, ML, J1
      JJ = J1
      JJJ = JJ/2
      JIJ = 2 *JJJ
      IF (JJ.EQ. J1) GO TO 600
COMMENT - UDU NUMBERED EQUATION FOR LATERAL FORCE EQUILIBRIUM
      I = JJ/2 + 1
      IP1 = I + 1
COMMENT - SU NOT REQUIRED FOR OFFSPRING
      IF (ML.EQ. - 1) GO TO 50
      SU(1) = TH*F(IP1)
      SU(2) = -2.0*F(IP1)
      SU(3) = -TH*F(I) - F(IP1)
      SU(4) = 2.0*F(I) + F(IP1) + SY(I)*HCU
      FF = OY(I)*HCU
      GO TO 800
600     CONTINUE
COMMENT - EVEN NUMBERED EQUATION FOR MOMENT EQUILIBRIUM
      I = JJ/2
      IP1 = I + 1
COMMENT - SU NOT REQUIRED FOR OFFSPRING
      IF (ML.EQ. - 1) GO TO 650
      SU(1) = 0.0
      SU(2) = 1.5*F(IP1)*HSO
      SU(3) = -TH*F(IP1)
      SU(4) = 2.5*F(I) + F(IP1)*HSO + SZ(I) *HCU
      FF = QZ(I)*HCU
650     CONTINUE
800     RETURN

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C      END                    26JAO
C      *****
C      SUBROUTINE            SUBROUTINE            SUBROUTINE            SUBROUTINE
C      *****
C      SUBROUTINE FORMLD ( RM, RO, W, SL, SU, FOMM, L1, L3, L4, L6 )      08APO
COMMENT - SUBROUTINE FORMLD CALCULATES FIXED-END-FORCE MATRIX FOR      24APO
COMMENT - BOTH PRISMATIC UNIFORMLY LOADED MEMBERS AND ALL OTHERS      24APO
DIMENSION RML(3,L6), ROLL(4), W(L6), SL(L3), SU(L4)                  08APO
DIMENSION FOMM(6)                                                   08APO
COMMON /BLOCK2/ DXS( 50), DYS( 50), ZLS( 50), DCIS( 50),           26JAO
2 DCZS( 50), PRF( 50), PRAE( 50), NCD5( 50), IAXDPS( 50),          26JAO
3 IOPPI( 50), IPINL( 50), IPINR( 50), NCS1( 50), SMC( 50,13)      20MYO
COMMON /BLOCK3/ DKL( 50), DYL( 50), ZLL( 50), DCIL( 50),          26JAO
2 DCZL( 50), UQX( 50), UQY( 50), NCDL( 50), IAXOPL( 50),          26JAO
3 NCS1( 50)                                                           26JAO
COMMON /BLOCK5/ XLS( 75), XRS( 75), FL( 75), AEL( 75),            26JAO
2 SXL( 75), SYL( 75), SZL( 75)                                       26JAO
COMMON /BLOCK6/ XLL(150), XRL(150), QXL(150), OYL(150),            26JAO
2 QZL(150)                                                             26JAO
COMMON /BLOCK7/ F( 42), AE( 42), SX( 42), SY(42),                 26JAO
2 SZ( 42), QX( 42), OY( 42), QZ( 42), AI( 42),                    26JAO
3 BI( 42), AI( 42), DI( 42), DX( 42), DY( 42),                    26JAO
4 U1(42), U1(42), V1(42), W1(42), U2(42),                          13MRD
5 V2(42), W2(42), ERX(42), ERY(42), ERZ(42),                       13MRD
6 RX(42), RV(42), RZ(42)                                             13MRD
COMMON /BLK1/ KEEP2, KEEP3, KEEP4, KEEP5, KEEP6, KEEP7,            26JAO
2 ITYPE, NCD2, NCD3, NCD4, NCD5, NCD6, NCD7,                          26JAO
3 IABAR, IFORM, MN, NJT, NST, NLT, TOL,                               26JAO
4 M, MP1, MP2, ISTT, LTT, ITYPEL, IDJ,                                12FEO
5 NLC, IPB, IP9, IP10                                                13FEO
COMMON /BLK2/ XL, XR, X1, X2, I1, I2, MO, H, TH, HSO, HCU, XZL        26JAO
COMMON /BLK3/ MNL, MNST, MNLT, MNH, MNC5, MNC6, MDJT, MNLG            07FEO
COMMON /BLK4/ ST1, ST2, ST3, ST4, ST5, ST6                            26JAO
COMMON /BLK5/ MFSUB                                                  08APO
COMMON / R1 / R2 / R3 / R4 / J1                                       08APC
COMMENT - SET TEMPORARY CONTROL CONSTANTS FOR LOAD TYPE WHICH IS    16MYO
COMMENT - HAVING ITS FIXED-END-FORCE MATRIX FORMED                   16MYO
IPINLT = IPINL(ISTT)                                                  26JAO
IPINRT = IPINR(ISTT)                                                  26JAO
ZL = ZLS(ISTT)                                                         26JAO
PRFT = PRF(ISTT)                                                       26JAO
PRAET = PRAE(ISTT)                                                     26JAO
NCDST = NCD5(ISTT)                                                     26JAO
NCS1T = NCS1(ISTT)                                                    26JAO
UQXT = UQX(LTT)                                                        26JAO
UQYT = UQY(LTT)                                                        26JAO
NCDLT = NCDL(LTT)                                                      26JAO
NCS1T = NCS1(LTT)                                                     26JAO
IF (NCDST.NE.0 .OR. NCDLT.NE.0) GO TO 2100                            26JAO
COMMENT - PRISMATIC MEMBER WITH UNIFORM LOADS                       16MYO
COMMENT - COMPUTE CONSTANTS AND AXIAL FIXED-END-FORCES              16MYO
ZL2 = ZL*ZL                                                            26JAO

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FOMM(1) = - C.5*UQXT*ZL                                             26JAO
FOMM(4) = FOMM(1)                                                   26JAO
IF (IPINLT.EQ.0 .AND. IPINRT.EQ.0) GO TO 1800                        26JAO
COMMENT - ZERO FLEXURAL FIXED END FORCES                             16MYO
FOMM(3) = FOMM(6) = 0.0                                             26FEO
IF (IPINLT.EQ.1 .AND. IPINRT.EQ.1) GO TO 1600                       26FEO
IF (IPINRT.EQ.1) GO TO 1700                                          26JAO
COMMENT - COMPUTE FLEXURAL FIXED-END-FORCES FOR MEMBER PINNED AT    16MYO
COMMENT - FROM JOINT                                                16MYO
FOMM(2) = -0.375*UQYT*ZL                                             26JAO
FOMM(5) = -0.625*UQYT*ZL                                             26JAO
FOMM(6) = UQYT*ZL2/8.0                                              26JAO
GO TO 2000                                                            16MYO
COMMENT - COMPUTE FLEXURAL FIXED-END-FORCES FOR MEMBER PINNED AT    16MYO
COMMENT - BOTH JOINTS                                               16MYO
FOMM(2) = -0.5*UQYT*ZL                                             26FEO
FOMM(5) = FOMM(2)                                                  26FEO
GO TO 2000                                                            16MYO
COMMENT - COMPUTE FLEXURAL FIXED-END-FORCES FOR MEMBER PINNED AT    16MYO
COMMENT - TO JOINT                                                  16MYO
FOMM(2) = -0.625*UQYT*ZL                                             26JAO
FOMM(3) = -UQYT*ZL2/8.0                                             26JAO
FOMM(5) = -0.375*UQYT*ZL                                             26JAO
GO TO 2000                                                            16MYO
COMMENT - COMPUTE FLEXURAL FIXED-END-FORCES FOR MEMBER WITH RIGID    16MYO
COMMENT - CONNECTIONS AT BOTH JOINTS                                16MYO
FOMM(2) = -0.5*UQYT*ZL                                             26JAO
FOMM(3) = -UQYT*ZL2/12.0                                           26JAO
FOMM(5) = FOMM(2)                                                  26JAO
FOMM(6) = - FOMM(3)                                                26JAO
2000 CONTINUE                                                         26JAO
GO TO 9900                                                            26JAO
2100 CONTINUE                                                         26JAO
COMMENT - NONPRISMATIC OR NONUNIFORMLY LOADED MEMBER              16MYO
IF (IFORM.EQ.1) GO TO 2700                                           26JAO
TH = ZL/M                                                            26JAO
M = 8.5*TH                                                           26JAO
MSO = MN                                                              26JAO
HCU = HSO*M                                                          26JAO
NL = Z*MP1                                                            26JAO
IF (NCDST.NE.0) GO TO 2400                                           26JAO
COMMENT - DISCRETIZE PRISMATIC STIFFNESS DATA                     16MYO
DO 2300 I = 1,MP2                                                    26JAO
SX(I) = SY(I) = SZ(I) = 0.0                                         26JAO
AE(I) = PRAET                                                         26JAO
F(I) = PRFT                                                            26JAO
AE(I) = F(I) = AE(MP2) = F(MP2) = 0.0                               26JAO
GO TO 2500                                                            26JAO
COMMENT - NONPRISMATIC MEMBER                                       16MYO
COMMENT - SUBROUTINE DISCT DISCRETIZES MEMBER STIFFNESS DATA F, AE 16MYO
COMMENT - SX, SY, AND SZ                                           16MYO
2400 CALL DISCT( NCS1T, NCDST, ZL, L1)                                11FEO
2500 CONTINUE                                                         26JAO
COMMENT - STORE MEMBER END RESTRAINTS ST1-ST6                       16MYO
ST1 = SX(I)                                                            26JAO
ST2 = SY(I)                                                            26JAO

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      ST3 = SZ(11)          26JAO
      ST4 = SX(MP1)        26JAO
      ST5 = SY(MP1)        26JAO
      ST6 = SZ(MP1)        26JAO
      ML = 1                26JAO
      MLT = 1              26JAO
COMMENT - SET MEMBER-END-RESTRAINTS EQUAL TO 1.0E+99 FOR FIXED-END-FORCE16MYO
COMMENT - SOLUTION 16MYO
      SX(11) = SX(MP1) = 1.0E99 26JAO
      SY(11) = SY(MP1) = 1.0E99 26JAO
      SZ(11) = SZ(MP1) = 1.0E99 26JAO
COMMENT - ZERO PINNED END ROTATIONAL RESTRAINTS 16MYO
      IF (IPINLT .EQ. 1) SZ(11) = 0.0 26JAO
      IF (IPINRT .EQ. 1) SZ(MP1) = 0.0 26JAO
2700 CONTINUE 26JAO
      IF (MCDLT .NE. 0) GO TO 2900 26JAO
COMMENT - DISCRETIZE UNIFORM MEMBER LOADS 16MYO
      DO 2800 I = 2,M 26JAO
      OX(I) = UOXT*TM 26JAO
      OY(I) = UOYT*TM 26JAO
      OZ(I) = 0.0 26JAO
      OX(MP2) = OY(MP2) = OZ(MP2) = 0.0 26JAO
      OX(11) = OX(MP1) = 0.5*UOXT*TM 26JAO
      OY(11) = OY(MP1) = 0.5*UOYT*TM 26JAO
      OZ(11) = OZ(MP1) = 0.0 26JAO
      GO TO 3000 26JAO
COMMENT - NONUNIFORM LOADS 16MYO
COMMENT - SUBROUTINE DISCLD DISCRETIZES GENERAL MEMBER LOADS OX, OY, OZ 16MYO
2900 CALL DISCLD ( NC61T, MCDLT, ZL, LI ) 11FEO
3000 CONTINUE 26JAO
COMMENT - AXIAL SOLUTION FOR MEMBER LOADS 16MYO
      CALL AXIAL ( MLT ) 26JAO
      MFSUB = 12 08APO
COMMENT - LATERAL SOLUTION FOR MEMBER LOADS 16MYO
      CALL GRIP2A ( RM, RO, W, SL, SU, L3, L4, L6, 3 ) 15MYO
COMMENT - CALCULATE MEMBER-END-FORCES WHICH ARE EQUAL TO FIXED-END- 16MYO
COMMENT - FORCES 16MYO
      CALL MEMEND ( W, FOMH, L6 ) 08APO
COMMENT - ZERO END-MOMENTS FOR PINNED END MEMBERS 16MYO
      IF (IPINLT .EQ. 1) FOMH(3) = 0.0 29JAO
      IF (IPINRT .EQ. 1) FOMH(6) = 0.0 29JAO
9900 CONTINUE 26JAO
      RETURN 26JAO
      END 26JAO
C
C *****
C SUBROUTINE SUBROUTINE SUBROUTINE SUBROUTINE
C *****
C
      SUBROUTINE DISCLD ( MC61T, MCDLT, ZL, LI ) 11FEO
COMMENT - SUBROUTINE DISCLD DISCRETIZES GENERAL MEMBER LOADS OX, OY, OZ 22MYO
COMMON /BLOCK6/ XLL(150), XRL(150), OXL(150), OYL(150), 26JAO
2 OZL(150) 26JAO
COMMON /BLOCK7/ P( 42), AE( 42), SX( 42), SY(42), 26JAO

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2 SZ( 42), OX( 42), OY( 42), OZ( 42), A( 42), 26JAO
3 BI( 42), A1( 42), D( 42), DX( 42), DY( 42), 26JAO
4 DZ(42), U1(42), V1(42), W1(42), U2(42), 15MRO
5 V2(42), W2(42), ERX(42), ERY(42), ERZ(42), 15MRO
6 RX(42), RY(42), RZ(42) 15MRO
COMMON /BLK1/ KEEP2, KEEP3, KEEP4, KEEP5, KEEP6, KEEP7, 26JAO
2 ITYPE, MCD2, MCD3, MCD4, MCD5, MCD6, MCD7, 26JAO
3 IABAN, IFORM, NM, NJT, NST, NLT, TOL, 26JAO
4 M, MP1, MP2, ISTD, LTT, IYPEL, IDJ, 12FEO
5 NLC, IPB, IP9, IP10 13FEO
COMMON /BLK2/ XL, XR, X1, X2, I1, I2, M0, H, TH, MSQ, HCU, X2L 26JAO
COMMENT - ZERO MEMBER LOAD DATA 15MYO
      DO 1020 I = 1,MP2 26JAO
      OX(I) = OY(I) = OZ(I) = 0.0 26JAO
      NC62T = NC61T - I + MCDLT 26JAO
      I1 = NC61T - 1 26JAO
COMMENT - I1 GOES FROM NC61T TO NC62T 15MYO
      1050 I1 = I1 + 1 26JAO
COMMENT - READ DATA FROM ONE CARD IMAGE (LOADS AT LEFT OF SECTION) 15MYO
      XL = XLL(I1) 26JAO
      XR = XRL(I1) 26JAO
      OXLT = OXL(I1) 26JAO
      OYLT = OYL(I1) 26JAO
      OZLT = OZL(I1) 26JAO
      IF (XR .NE. 0.0) GO TO 1100 26JAO
COMMENT - VARIABLE LOADING SECTION READ ONE CARD IMAGE (LOADS AT 15MYO
COMMENT - RIGHT OF SECTION) 15MYO
      I1 = I1 + 1 26JAO
      XR = XRL(I1) 26JAO
      OXRT = OXL(I1) 26JAO
      OYRT = OYL(I1) 26JAO
      OZRT = OZL(I1) 26JAO
      GO TO 1110 26JAO
1100 OXRT = OXLT 26JAO
      OYRT = OYLT 26JAO
      OZRT = OZLT 26JAO
1110 CONTINUE 26JAO
      IF ( XL .NE. XR) GO TO 2100 26JAO
COMMENT - CONCENTRATED LOADS CALL CONLD TO DISTRIBUTE CONCENTRATED 15MYO
COMMENT - LOADS TO ADJACENT STATIONS 15MYO
      CALL CONLD ( OXLT, XL, OX, L1 ) 11FEO
      CALL CONLD ( OYLT, XL, OY, L1 ) 11FEO
      CALL CONLD ( OZLT, XL, OZ, L1 ) 11FEO
      GO TO 2200 26JAO
2100 CONTINUE 26JAO
      Z11 = XL/TH + 2.0 26JAO
      I1 = Z11 26JAO
      X1 = I1*TH - XL - TH 26JAO
      Z12 = XR/TH + 1.0 26JAO
      I2 = Z12 26JAO
      X2 = XR - I2*TH + TH 26JAO
      M0 = I2 - I1 26JAO
COMMENT - DISTRIBUTION LOADS CALL LINLD TO DISTRIBUTE LOADS STATIONS 15MYO
COMMENT - I1 TO I2 15MYO
      IF (OXLT .EQ. 0.0 .AND. OXRT .EQ. 0.0) GO TO 2150 11MYO
      CALL LINLD ( OXLT, OXRT, OX, L1 ) 11FEO

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2150 IF IQVLT.EQ. 0.0 .AND. QYRT.EQ. 0.0) GO TO 2160      11MYO
CALL LIND ( QYLT, QYRT, QY, L1 )                          11FEO
2160 IF (QZLT.EQ. 0.0 .AND. QZRT.EQ. 0.0) GO TO 2200      11MYO
CALL LIND ( QZLT, QZRT, QZ, L1 )                          11FEO
2200 CONTINUE                                             26JAO
9000 IF (III.LT. NCG2T) GO TO 1050                       26JAO
9900 CONTINUE                                             26JAO
RETURN                                                    26JAO
END                                                        26JAO

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SUBROUTINE MEMEND ( N, FMM, L6 )                          08APO
COMMENT - SUBROUTINE MEMEND CALCULATES FORCES ON END OF MEMBER USING 15MYO
COMMENT - DISPLACEMENTS FOUND FROM MEMBER SOLUTION      15MYO
DIMENSION FMM(6), W(6)                                    08APO
COMMON /BLOCK7/ F( 42), AE( 42), SX( 42), SY(42),      26JAO
2 SZ( 42), OX( 42), OY( 42), OZ( 42), A( 42),          26JAO
3 B( 42), AI( 42), DI( 42), DX( 42), DY( 42),          26JAO
4 DZ(42), U(42), V(42), W(42), U2(42),                 13MRO
5 V2(42), W2(42), ERX(42), ERY(42), ERZ(42),           13MRO
6 RX(42), RY(42), RZ(42)                                13MRO
COMMON /BLK1/ KEEP2, KEEP3, KEEP4, KEEP5, KEEP6, KEEP7, 26JAO
2 ITYPE, NCD2, NCD3, NCD4, NCD5, NCD6, NCD7,            26JAO
3 IABAN, IFORM, NM, NJT, NST, MLT, TOL,                 26JAO
4 M, MP1, MP2, ISTT, LTT, ITYPEL, IDJ,                  12FEO
5 NLC, IP8, IP9, IP10                                    13FEO
COMMON /BLK2/ XL, XR, X1, X2, I1, I2, NQ, N, TH, HSO, HCU, XZL 26JAO
COMMON /BLK4/ ST1, ST2, ST3, ST4, ST5, ST6                26JAO
COMMENT - COMPUTE AXIAL END FORCES                        15MYO
DOX = DX(2) - DX(1)                                     26JAO
FMM (1) = -AE(2)*DOX/TH + ST1*DX(1) - OX(1)              26JAO
DOX = DX(MP1) - DX(M)                                   26JAO
FMM (4) = AE(MP1)*DOX/TH + ST4*DX(MP1) - OX(MP1)         26JAO
COMMENT - CONVERT DISPLACEMENTS FROM GRIP2B (M) TO LATERAL AND 15MYO
COMMENT - ROTATIONAL DISPLACEMENTS DY AND DZ             15MYO
DY(1) = W(1)                                            26JAO
OZ(1) = W(2)                                            26JAO
DY(2) = W(3)                                            26JAO
DZ(2) = W(4)                                            26JAO
DY(M) = W(2*M - 1)                                       26JAO
DZ(M) = W(2*M)                                           26JAO
DY(MP1) = W(2*M + 1)                                     26JAO
DZ(MP1) = W(2*M + 2)                                     26JAO
COMMENT - COMPUTE CURVATURES                              15MYO
TAU1 = (DY(2) - DY(1))/H - (1.5*DZ(1) + 0.5*DZ(2))      26JAO
TAU2 = -(DY(2) - DY(1))/H + (0.5*DZ(1) + 1.5*DZ(2))      26JAO
COMMENT - COMPUTE LATERAL AND ROTATIONAL END FORCES      15MYO
FMM (2) = F(2)*(TAU2 - TAU1)/HSQ + ST2*OY(1) - OY(1)     26JAO
FMM (3) = F(2)*( -1.5*TAU1 + 0.5*TAU2)/H + ST3*DZ(1) - 26JAO
OZ(1)                                                    26JAO
2 TAU1 = (DY(MP1) - DY(M))/H - (1.5*DZ(M) + 0.5*DZ(MP1)) 26JAO

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TAU2 = -(DY(MP1) - DY(M))/H + (0.5*DZ(M) + 1.5*DZ(MP1)) 26JAO
FMM (5) = F(MP1)*(TAU1 - TAU2)/HSQ + ST5*DY(MP1) -      26JAO
OY(MP1)                                                  26JAO
FMM (6) = F(MP1)*( -0.5*TAU1 + 1.5*TAU2)/H +          26JAO
ST6*DZ(MP1) - OZ(MP1)                                    26JAO
RETURN                                                    26JAO
END                                                        26JAO

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SUBROUTINE SUBROUTINE SUBROUTINE SUBROUTINE
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SUBROUTINE FSUB11 ( SU4, FF, L4, IHB )                    25JLO
COMMENT - SUBROUTINE FSUB11 FURNISHES RIGHT SIDE OF SYMMETRIC STIFFNESS 21MYO
COMMENT - MATRIX SU AND LOAD TERM F TO GRIP2A FOR FRAME SOLUTION          21MYO
COMMENT - SU IS ONE ROW OF STIFFNESS MATRIX AND F IS CORRESPONDING LOAD 21MYO
COMMENT - FSUB11 FORMS SSL (3 ROWS OF SU) AND FSS (3 LOADS) EVERY THIRD 21MYO
COMMENT - CALL FROM GRIP2A AND FURNISHES SU AND F FOR EACH CALL          21MYO
DIMENSION SU4(L4)                                        21MYO
DIMENSION SMM(3,3), SMS(3,3), DC(3,3), DCT(3,3), T33(3,3), 21MYO
2 FMM(3), FSS(3), FMS(3)                                 21MYO
COMMON /BLOCK1/ X( 75), Y( 75), OX( 75), OY( 75),      13FEO
2 UZZ( 75), SXX( 75), SYY( 75), SZZ( 75), DXX( 75),    13FEO
3 DYY( 75), DZZ( 75), RXX( 75), RYY( 75), RZZ( 75),    13FEO
4 ERXX( 75), ERY( 75), ERZ( 75)                        13FEO
COMMON /BLOCK2/ DXS( 50), DYS( 50), ZLS( 50), DC1S( 50), 26JAO
2 DC2S( 50), PRF( 50), PRAE( 50), NCD5( 50), IAKDPS( 50), 26JAO
3 IOPUP( 50), IPINL( 50), IPINR( 50), NCS1( 50), SMC( 50,13) 20MYO
COMMON /BLOCK3/ DALS( 50), DYL( 50), ZLL( 50), DC1L( 50), 26JAO
2 DC2L( 50), UOX( 50), UOY( 50), NCDL( 50), IAKOPL( 50), 26JAO
3 NCG1( 50)                                               26JAO
COMMON /BLOCK4/ JT1(150), JT2(150), IST(150), LT(150), 26JAO
2 FOMM(150,6)                                             26JAO
COMMON / BLOCIO / SSL (3,30)                              08APO
COMMON /BLK1/ KEEP2, KEEP3, KEEP4, KEEP5, KEEP6, KEEP7, 26JAO
2 ITYPE, NCD2, NCD3, NCD4, NCD5, NCD6, NCD7,            26JAO
3 IABAN, IFORM, NM, NJT, NST, MLT, TOL,                 26JAO
4 M, MP1, MP2, ISTT, LTT, ITYPEL, IDJ,                  12FEO
5 NLC, IP8, IP9, IP10                                    13FEO
COMMON / RI / NL, ML, JI                                  08APO
NL4 = NL                                                 08APO
ML4 = ML                                                 08APO
JI4 = JI                                                 08APO
IF (JI4.NE. 1) GO TO 1300                                26JAO
COMMENT - SET CONSTANTS ON FIRST CALL FROM GRIP2A        21MYO
IHB1 = IHB + 1                                           26JAO
IHB1 = IHB - 1                                           26JAO
SMM(1,2) = SMM(1,3) = SMM(2,1) = SMM(3,1) = 0.0        20MYO
DC(1,3) = DC(2,3) = DC(3,1) = DC(3,2) = 0.0           26JAO
DCT(1,3) = DCT(2,3) = DCT(3,1) = DCT(3,2) = 0.0        26JAO
DC(3,3) = DCT(3,3) = 1.0                                 26JAO
COMMENT - COMPUTE JOINT NUMBER FOR WHICH EQUATIONS ARE BEING FORMED 21MYO
1300 JTN = (JI4 - 1)/3 + 1                               26JAO
COMMENT - SKIP FOR EVERY SECOND AND THIRD EQUATION (CALL FROM GRIP2A) 21MYO

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COMMENT - UP SSL                                21MYO
DO 3600 I = 1,IMB                               27JAO
3600   SSL(2,I) = SSL(2, I + 1)                 27JAO
DO 3700 I = 1,IMB1                              27JAO
3700   SSL(3,I) = SSL(3, I + 2)                 27JAO
4000   SSL(2,IMB1) = SSL(3,IMB1) = SSL(3,IMB) = 0.0  28JAO
CONTINUE                                         26JAO
N123 = J14 - 3*JTM = 3                          05MRO
IF ( N14 .EQ. -1 ) GO TO 4400                   05MRO
COMMENT - SKIP FOR OFFSPRING                    21MYO
IB = IMB1                                       01JAO
COMMENT - FURN SU FROM ROW(N123) OF SSL         21MYO
DO 4300 I = 1,IMB1                              27JAO
4300   SUM(I) = SSL(N123,IB)                    01JAO
4400   IB = IB - 1                              01JAO
4400   FF = FSS(N123)                            27JAO
COMMENT - SKIP FOR ALL BUT OFFSPRING           21MYO
IF ( N14 .GT. -1 ) GO TO 4450                   05MRO
COMMENT - CHECK FOR UNDEFINED DISPLACEMENT IN PARENT OF THIS OFFSPRING 21MYO
IF ( N123 .EQ. 1 .AND. DXX(JTM) .EQ. 1.0E99 ) GO TO 4480 08APO
IF ( N123 .EQ. 2 .AND. OYY(JTM) .EQ. 1.0E99 ) GO TO 4480 08APO
IF ( N123 .EQ. 3 .AND. DZZ(JTM) .EQ. 1.0E99 ) GO TO 4480 08APO
GO TO 4500                                       05MRO
4450   K = IMB1                                  05MRO
IF (SUM(K) .NE. 0.0) GO TO 4500                 05MRO
COMMENT - ZERO ON DIAGONAL OF MATRIX - DISPLACEMENT UNDEFINED - SET 21MYO
COMMENT - DISPLACEMENT EQUAL TO 1.0E-99        21MYO
SUM(K) = 1.0                                    05MRO
4480   FF = 1.0E99                              05MRO
4500   CONTINUE                                  05MRO
RETURN                                           26JAO
END                                               26JAO
C
C *****
C
C SUBROUTINE SUBROUTINE SUBROUTINE SUBROUTINE
C *****
C
SUBROUTINE MATM3 ( AM, BM, CM )                 26JAO
COMMENT - THIS SUBROUTINE MULTIPLIES A 3X3 MATRIX ,AM, TIMES A 26JAO
COMMENT - 3X3 MATRIX ,BM, TO PRODUCE A 3X3 MATRIX ,CM 26JAO
DIMENSION AM(3,3),BM(3,3),CM(3,3)             26JAO
DO 23 I = 1,3                                   26JAO
DO 23 J = 1,3                                   26JAO
CM(I,J) = 0.0                                  26JAO
DO 23 K = 1,3                                   26JAO
CM(I,J) = AM(I,K)*BM(K,J) + CM(I,J)           26JAO
23 CONTINUE                                     26JAO
RETURN                                          26JAO
END                                             26JAO
C
C *****
C
C SUBROUTINE SUBROUTINE SUBROUTINE SUBROUTINE
C *****

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C *****
C
SUBROUTINE MATM3 ( AM, BV, CV )                 26JAO
COMMENT - THIS SUBROUTINE MULTIPLIES A 3X3 MATRIX ,AM, TIMES A 26JAO
COMMENT - 3X1 MATRIX ,BV TO PRODUCE A 3X1 MATRIX ,CV 26JAO
DIMENSION AM(3,3),BV(3),CV(3)                26JAO
DO 23 I = 1,3                                   26JAO
CV(I) = 0.0                                    26JAO
DO 23 K = 1,3                                   26JAO
CV(I) = AM(I,K)*BV(K) + CV(I)                 26JAO
23 CONTINUE                                     01JAO
RETURN                                          26JAO
END                                             26JAO
C
C *****
C
C SUBROUTINE SUBROUTINE SUBROUTINE SUBROUTINE
C *****
C
SUBROUTINE SUM1                                12FEO
COMMENT - SUBROUTINE SUM1 DOES SUPERPOSITION SOLUTION FOR FRAME 24APO
COMMENT - JOINT DISPLACEMENTS AND REACTIONS FOR FAMILY PROBLEMS 24APO
COMMON /BLOCK1/ X1(75), Y1(75), OXX1(75), DYY(75), 13FEO
2 DZZ(75), SXX1(75), SYX1(75), SZZ(75), OXX(75), 13FEO
3 DYY(75), OZZ(75), RXX1(75), RYY(75), RZZ(75), 13FEO
4 ERXX(75), ERYX(75), ERZZ(75)               13FEO
COMMON /BLOCK4/ JT1(150), JT2(150), IST(150), LTI(150), 26JAO
2 FOMM(150,6)                                  26JAO
COMMON /BLOCK8/ MP(21,21), ZM(21)             09JEO
COMMON /BLK1/ KEEP2, KEEP3, KEEP4, KEEP5, KEEP6, KEEP7, 12FEO
2 ITYPE, MCD2, MCD3, MCD4, MCD5, MCD6, MCD7, 12FEO
3 LABAN, IFORM, NM, NJT, MST, NLT, TOL, 12FEO
4 M, MP1, MP2, IST, LTI, IYPEL, IDJ, 12FEO
5 NLC, IP8, IP9, IP10                          13FFO
REWIND 1                                       16JFO
COMMENT - ZERO JOINT DISPLACEMENTS, REACTIONS, AND LOADS 20MYO
DO 1100 I = 1, NJT                             12FEO
DXX(I) = DYY(I) = DZZ(I) = 0.0                 12FFO
RXX(I) = RYY(I) = RZZ(I) = 0.0                 12FEO
OXX(I) = OYY(I) = OZZ(I) = 0.0                 12FEO
1100 COMMENT - DO FOR EACH PROBLEM RESULTS ARE STORED FOR 20MYO
DO 2400 J = 1, NLC                              17FEO
COMMENT - ERXX, ERYX, ERZZ, AND FOMM ARE NOT NEEDED FOR TYPE 4 PROBLEM AND 20MYO
COMMENT - ARE USED AS DUMMIES TO READ IN VALUES OF DISPLACEMENTS. 20MYO
COMMENT - REACTIONS AND LOADS FROM TAPF 20MYO
KLAO (I) ( LRXX(I), LRYX(I), LERZZ(I), FOMM(I,1), FOMM(I,2), 16JEO
2 FOMM(I,3), FOMM(I,4), FOMM(I,5), FOMM(I,6) ), I = 1, NJT 1 13FEO
COMMENT - SKIP FOR ZERO MULTIPLIER 20MYO
IF ( ZM(I) .EQ. 0.0) GO TO 2400                 12FFO
COMMENT - MULTIPLY AND SUM 20MYO
DO 2100 I = 1, NJT                              13FEO
DXX(I) = DXX(I) + ZM(I)*ERXX(I)                13FEO
DYY(I) = DYY(I) + ZM(I)*ERYX(I)                13FEO
DZZ(I) = DZZ(I) + ZM(I)*ERZZ(I)                13FEO
RXX(I) = RXX(I) + ZM(I)*FOMM(I,1)              12FFO

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RYY(I) = RYY(I) + ZH(J)*FOMH(I,2) 12FEO
RZZ(I) = RZZ(I) + ZH(J)*FOMH(I,3) 12FEO
QXX(I) = QXX(I) + ZH(J)*FOMH(I,4) 12FEO
QYY(I) = QYY(I) + ZH(J)*FOMH(I,5) 12FEO
QZZ(I) = QZZ(I) + ZH(J)*FOMH(I,6) 12FEO
2100 CONTINUE 12FEO
2400 CONTINUE 12FEO
RETURN 12FEO
END 12FEO

*****
SUBROUTINE SUBROUTINE SUBROUTINE SUBROUTINE
*****

SUBROUTINE MEMBES ( DM, JJ, KK, RM, RO, W, SL, SU, NPROB, ANZ, L1, 08APO
2 L3, L4, L6, L7 ) 08APO
COMMENT - SUBROUTINE MEMBES COMPUTES MEMBER RESULTS FOR ALL MEMBERS 24APO
COMMENT - IN FRAME, SUBTRACTS APPROPRIATE MEMBER END FORCES TO COMPLETE 24APO
COMMENT - CALCULATION OF JOINT EQUILIB ERRORS AND PRINTS OUT MEMBER 24APO
COMMENT - RESULTS 24APO
DIMENSION RM(L3),L6), RO(L6), W(L6), SL(L3), SU(L4) 08APO
DIMENSION DC(3,3), DMH(3), DMS(3), DM(L7,6) 08APO
DIMENSION FJM(3), FZM(3) 21MRO
DIMENSION ANZ(18), NPROB(2) 11FEO
COMMON /BLOCK1/ X( 75), Y( 75), QXX( 75), QYY( 75), 13FEO
2 UZZ( 75), SXX( 75), SYX( 75), SYZ( 75), DXX( 75), 13FEO
3 DYY( 75), DZZ( 75), RXX( 75), RYY( 75), RZZ( 75), 13FEO
4 ERXX( 75), ERYX( 75), ERZZ( 75) 13FEO
COMMON /BLOCK2/ DKS( 50), DYS( 50), ZLS( 50), DC(51 50), 26JAO
2 DCZ( 50), PRF( 50), PRAE( 50), NCDST( 50), IAXOP( 50), 26JAO
3 IOPOP( 50), IPINL( 50), IPINR( 50), NCS( 50,13), SMC( 50,13) 20MYO
COMMON /BLOCK3/ DKL( 50), DYL( 50), ZLL( 50), DCIL( 50), 26JAO
2 DCZL( 50), UOR( 50), UOY( 50), MCDL( 50), IAXOPL( 50), 26JAO
3 MCL( 50) 26JAO
COMMON /BLOCK4/ JT1(150), JT2(150), IST(150), LTI(150), 26JAO
2 FOMH(150,4) 26JAO
COMMON /BLOCK5/ XLS( 75), XRS( 75), FL( 75), AEL( 75), 26JAO
2 SXL( 75), SYL( 75), SZL( 75) 26JAO
COMMON /BLOCK6/ XLL(150), XRL(150), QXL(150), QYL(150), 26JAO
2 QZL(150) 26JAO
COMMON /BLOCK7/ F( 42), AE( 42), SX( 42), SY(42), 13MRO
2 SE( 42), QX( 42), QY( 42), QZ( 42), AI( 42), 13MRO
3 BI( 42), AI( 42), DI( 42), DX( 42), DY( 42), 13MRO
4 DZ(42), U(42), V(42), W(42), UZ(42), 13MRO
5 VZ(42), WZ(42), ERX(42), ERY(42), ERZ(42), 13MRO
6 RX(42), RY(42), RZ(42) 13MRO
COMMON /BLOCK9/ T(21), V(21), BM(21), DXT(21), 16MRO
2 DYT(21), DZT(21) 16MRO
COMMON /BLK1/ KEEP2, KEEP3, KEEP4, KEEP5, KEEP6, KEEP7, 13MRO
2 ITYPE, NCD2, NCD3, NCD4, NCD5, NCD6, NCD7, 13MRO
3 IABAN, IFORM, RM, NJT, NST, NLT, NLT, TOL, 13MRO
4 M, MP1, MP2, IST, LTI, IYPEL,10J, 13MRO
5 NLC, IP8, IP9, IP10 13MRO
COMMON /BLK2/ XL,XR,XI,X2,I1,I2,NO,H,TH,HSQ,MCU,XZL 13MRO

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COMMON /BLK3/ MNJT,MNST,MNLT,MNH,MNC5,MNC6,MDJT,MMLC 13MRO
COMMON /BLK4/ ST1,ST2,ST3,ST4,ST5,ST6 13MRO
COMMON /BLK5/ MF,SUB 08APO
COMMON / R / RL, ML, J1 08APO
MP22 = MP2/2 16MRO
MP221 = MP22 - 1 20MRO
MM1 = M - 1 17MRO
IF (JJ .GT. NM) GO TO 5100 16MRO
ISTT = IST(JJ) 16MRO
LTI = LTI(JJ) 16MRO
IF (ISTT .EQ. 0) GO TO 5100 13MRO
IF (JJ .EQ. 1) GO TO 1300 16MRO
IF (ISTT .NE. IST(JJ - 1)) GO TO 1300 16MRO
COMMENT - SKIP FOR MEMBER WHICH HAS SAME STIFFNESS TYPE AS LAST MEMBER 16MYO
ML = -1 17MRO
MLT = -1 17MRO
GO TO 2700 17MRO
1300 CONTINUE 13MRO
COMMENT - SET TEMPORARY CONTROL CONSTANTS 16MYO
IPINLT = IPINL(ISTT) 13MRO
IPINRT = IPINR(ISTT) 13MRO
ZL = ZLS(ISTT) 13MRO
PRFT = PRF(ISTT) 13MRO
PRAET = PRAE(ISTT) 13MRO
NCDST = NCDST(ISTT) 13MRO
NCS11 = NCS1(ISTT) 13MRO
TH = ZL/M 13MRO
M = 0.5*TH 13MRO
HSQ = MMH 13MRO
MCU = HSQ*H 13MRO
ML = 2*MP1 13MRO
IF (NCDST .NE. 0) GO TO 2400 13MRO
COMMENT - PRISMATIC MEMBER DISCRETIZE MEMBER STIFFNESS DATA 16MYO
DU 2300 I = 1,MP2 13MRO
SX(I) = SY(I) = SZ(I) = 0.0 13MRO
AE(I) = PRAET 13MRO
F(I) = PRFT 13MRO
2300 AE(1) = F(1) = AE(MP2) = F(MP2) = 0.0 13MRO
GO TO 2500 13MRO
COMMENT - NONPRISMATIC MEMBER DISCRETIZE MEMBER STIFFNESS DATA 16MYO
2400 CALL DISCST ( NCS1T, NCDST, ZL, L1 ) 13MRO
2500 CONTINUE 13MRO
COMMENT - STORE MEMBER-END-RESTRAINTS ST1 - ST6 16MYO
ST1 = SX(1) 13MRO
ST2 = SY(1) 13MRO
ST3 = SZ(1) 13MRO
ST4 = SX(MP1) 13MRO
ST5 = SY(MP1) 13MRO
ST6 = SZ(MP1) 13MRO
ML = 1 13MRO
MLT = 1 13MRO
COMMENT - SET MEMBER-END-RESTRAINTS EQUAL TO 1.0E+99 FOR FINAL MEMBER 16MYO
COMMENT - SOLUTION 16MYO
SX(1) = SX(MP1) = 1.0E99 13MRO
SY(1) = SY(MP1) = 1.0E99 13MRO
SZ(1) = SZ(MP1) = 1.0E99 13MRO

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ERZ(IM1) = QZ(IM1) + RZ(IM1) - WZ(IM1) - W1(I)
3400 CONTINUE
COMMENT - REPEAT ABOVE FOR END STATIONS
U1(2) = U1(2) + ST1*DX(I) - DT1
V1(2) = V1(2) + ST2*DY(I) - DT2
W1(2) = W1(2) + ST3*DZ(I) - DT3
U2(MP1) = U2(MP1) + ST4*DX(MP1) - DT4
V2(MP1) = V2(MP1) + ST5*DY(MP1) - DT5
W2(MP1) = W2(MP1) + ST6*DZ(MP1) - DT6
I = I + 1
COMMENT - DO FOR EVERY SECOND INTERIOR STATION
DO 3600 IM = 2,MP221
I = I + 2
IP1 = I + 1
COMMENT - COMPUTE AVERAGE AXIAL FORCE, SHEAR, AND BENDING MOMENT
T(IM) = 0.5*(U2(I) - U1(IP1))
V(IM) = -0.5*(V2(I) - V1(IP1))
BM(IM) = 0.5*(W2(I) - W1(IP1))
COMMENT - OUTPUT DISPLACEMENTS
DXT(IM) = DX(I)
DYT(IM) = DY(I)
DZT(IM) = DZ(I)
3600 CONTINUE
COMMENT - REPEAT ABOVE FOR END STATIONS
T(1) = -U1(2)
V(1) = V1(2)
BM(1) = -W1(2)
DXT(1) = DX(1)
DYT(1) = DY(1)
DZT(1) = DZ(1)
T(MP22) = U2(MP1)
V(MP22) = -V2(MP1)
BM(MP22) = W2(MP1)
DXT(MP22) = DX(MP1)
DYT(MP22) = DY(MP1)
DZT(MP22) = DZ(MP1)
COMMENT - COMPUTE MEMBER END FORCES
F1M(1) = -T(1)
F1M(2) = V(1)
F1M(3) = -BM(1)
F2M(1) = T(MP22)
F2M(2) = -V(MP22)
F2M(3) = BM(MP22)
COMMENT - SUBROUTINE ADJTER TRANSFORMS MEMBER END FORCES AND SUBTRACTS
COMMENT - FROM JOINT EQUILIBRIUM ERRORS TO ACCUMULATE JOINT EQUILIBRIUM
COMMENT - ERRORS FOR FRAME
CALL ADJTER ( F1M, F2M, J1(JJ), J2(JJ), DC1, DC2),
2 DC25(I,ST) I
COMMENT - COMPUTE MAXIMUM EQUILIBRIUM ERROR IN MEMBER
STAERR = 0.0
DO 3800 I = 2,M
ERR = ABS(ERX(I))
IF (ERR .LE. STAERR) GO TO 3740
STAERR = ERR
3740 ERR = ABS(ERY(I))
IF (ERR .LE. STAERR) GO TO 3760

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13MR0 STAERR = ERR
13MR0 ERR = ABS(ERZ(I))
16MY0 IF (ERR .LE. STAERR) GO TO 3780
23MR0 STAERR = ERR
23MR0 CONTINUE
23MR0 CONTINUE
3800 IJ = 0
COMMENT - DM IS STORAGE VARIABLE FOR SIX MEMBER RESULTS
DO 4100 IM = 1,MP22
IJ = IJ + 1
DM(IJ,KK1) = DXT(IM)
IJ = IJ + 1
DM(IJ,KK1) = DYT(IM)
IJ = IJ + 1
DM(IJ,KK1) = DZT(IM)
IJ = IJ + 1
DM(IJ,KK1) = T(IM)
IJ = IJ + 1
DM(IJ,KK1) = V(IM)
IJ = IJ + 1
DM(IJ,KK1) = BM(IM)
IJ = IJ + 1
DM(IJ,KK1) = STAERR
4100 COMMENT - PRINT TABLE 9 MEMBER RESULTS IF REQUESTED
IF (IP9 .EQ. 1) GO TO 9900
CALL PRINT9 ( DM, L7, AN2, JJ, KK, NPROB )
GO TO 9900
5100 CONTINUE
COMMENT - ZERO MEMBER RESULTS FOR ZERO STIFFNESS TYPE AND DUMMY MEMBERS
COMMENT - USED TO FILL OUT GROUPS OF SIX
DO 5200 IJ = 1,L7
5200 DM(IJ,KK) = 0.0
5900 CONTINUE
RETURN
END
C
C *****
C SUBROUTINE SUBROUTINE SUBROUTINE SUBROUTINE
C *****
C
SUBROUTINE ADJTER (F1M, F2M, J1, J2, DC1, DC2)
COMMENT - SUBROUTINE ADJTER TRANSFORMS MEMBER-END-FORCES TO STRUCTURE
COMMENT - COORDINATES AND SUBTRACTS FROM APPROPRIATE JOINT EQUILIBRIUM
COMMENT - ERROR FOR FRAME
DIMENSION F1M(3), F2M(3), F1S(3), F2S(3)
DIMENSION DCT(3,3)
COMMON /BLUCK1/ XI(75), Y1(75), QXX(75), QYY(75),
2 QZZ(75), SXX(75), SYY(75), SZZ(75), DXX(75),
3 DYY(75), DZZ(75), RXX(75), RYY(75), RZZ(75),
4 ERXX(75), ERYX(75), ERZ(75)
COMMENT - FORM TRANSPOSE OF MEMBER TRANSFORMATION MATRIX
DCT(1,3) = DCT(2,3) + DCT(3,1) + DCT(3,2) = 0.0
DCT(3,3) = 1.0
DCT(1,1) = DC1

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REVIND 2
REVIND 3
310 CONTINUE
RETURN
END
*****
SUBROUTINE SUBROUTINE SUBROUTINE SUBROUTINE
*****
SUBROUTINE PRINT9 ( DM, L7, AM2, JJ, KK, NPROB )
COMMENT - SUBROUTINE PRINT9 PRINTS MEMBER RESULTS FOR ALL MEMBERS
COMMENT - PRINT9 IS CALLED FOR EACH INDIVIDUAL MEMBER (NUMBER JJ)
DIMENSION DM(17,6)
DIMENSION AM2(12), NPROB(12)
COMMON /BLOCK2/ DSI( 50), DYS( 50), ZLS( 50), DCIS( 50),
2 DC2S( 50), PRF( 50), PRAE( 50), NCDS( 50), IAXOPS( 50),
3 IOPOP( 50), IPTNL( 50), IPINR( 50), NCS( 50), SMM( 50,13)
COMMON /BLOCK4/ JT1(130), JT2(130), IST(130), LT(130),
2 FOMM(130,4)
COMMON /BLK1/ KEEP2, KEEP3, KEEP4, KEEP5, KEEP6, KEEP7,
2 ITYPE, MCD2, MCD3, MCD4, MCD5, MCD6, MCD7,
3 TABAR, IFORM, NM, NJT, NST, MLT, TOL,
4 M, MP1, MP2, ISTD, LTT, ITYPEL, IJD,
5 NLC, IPB, IP9, IP10
11 FORMAT ( 3H1 'BOR 18H1-----TRIN )
16 FORMAT ( ///17H PROB (CONTD), /5X, A1, A4, 5X, 17A4, A2, // )
51 FORMAT ( 30H TABLE 9 - MEMBER RESULTS ,/// )
52 FORMAT ( 40H TABLE 9 - MEMBER RESULTS (CONTD) ,/// )
61 FORMAT ( 18H MEMBER NUMBER,15,15H STIFF TYPE,15,
2 15H LOAD TYPE , 15 )
62 FORMAT ( 18H MEMBER NUMBER,15,15H STIFF TYPE,15)
71 FORMAT ( 14H LENGTH = ,E11.3, 13H ALPHA = ,E11.3,
2 13H BETA = ,E11.3 )
81 FORMAT ( 20H GOES FROM JOINT, 15 , 9H TO JOINT,15)
91 FORMAT ( 34H OUTPUT DISTANCES ARE FROM JOINT, 15,
2 25H ALONG THE MEMBER AXIS )
92 FORMAT ( 34H OUTPUT DISTANCES ARE FROM JOINT, 15,
2 27H ALONG THE STRUCTURE X-AXIS )
93 FORMAT ( 34H OUTPUT DISTANCES ARE FROM JOINT, 15,
2 27H ALONG THE STRUCTURE Y-AXIS )
101 FORMAT ( 49H ALL OUTPUT FORCES AND DISPLACEMENTS ARE WITH,
2 30H RESPECT TO THE MEMBER AXES ,///,
3 26X, 15H DISPLACEMENTS , 21X, 7H FORCES,///,
4 30H DISTANCE AXIAL LATERAL ROTATIONAL ,
5 30H AXIAL SHEAR MOMENT ,// )
111 FORMAT ( 5X,7E11.3)
201 FORMAT ( 44H ALL OUTPUT FORCES ARE WITH RESPECT TO THE,
2 15H MEMBER AXES , // , 15X,
3 10H AT JOINT , 15, 12X, 10H AT JOINT , 15,///,5X,
4 15H AXIAL FORCE = ,E11.3,5X, 15H AXIAL FORCE = ,E11.3,25HRO
5 /, 5X, 15H SHEAR = ,E11.3,5X, 15H SHEAR = ,E11.3,30HRO
6 /, 5X, 15H MOMENT = ,E11.3,5X, 15H MOMENT = ,E11.3,24APD
301 FORMAT ( /,50H THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE, 25HRO

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2 10H MEMBER IS, E11.3, /// )
MP22 = MP2/2
L7M1 = L7 - 1
L7M2 = L7 - 2
L7M3 = L7 - 3
COMMENT - SKIP FOR COMPLETE OUTPUT
IF ( IOPOP(ISTT) .EQ. 0 ) GO TO 1600
IF ( JJ .EQ. 1 ) GO TO 1500
COMMENT - PRINT PARTIAL RESULTS FOR 3 MEMBERS ON 1 SHEET
IF ( IOPL .NE. 1 ) GO TO 1500
IC = IC + 1
IF ( IC .EQ. 4 ) GO TO 1500
GO TO 2100
1500 IC = 1
1600 CONTINUE
COMMENT - PRINT HEADINGS
PRINT 11
PRINT 14, NPROB, (AM2(11), I1 = 1, 18)
IF ( JJ .EQ. 1 ) GO TO 1700
PRINT 52
GO TO 2100
1700 PRINT 51
2100 CONTINUE
IF ( ITYPE .NE. 4 ) GO TO 2500
PRINT 62, JJ, ISTD
GO TO 2600
2500 PRINT 61, JJ, ISTD, LTT
2600 CONTINUE
PRINT 71, ZLS(ISTT), DCIS(ISTT), DC2S(ISTT)
PRINT 81, JT1( JJ ), JT2( JJ )
IF ( IOPOP(ISTT) .EQ. 1 ) GO TO 5100
IAXOPT = IAXOPS(ISTT)
GO TO ( 2800, 2900, 3000 ) , IAXOPT
2800 PRINT 91, JT1( JJ )
GO TO 3100
2900 PRINT 92, JT1( JJ )
GO TO 3100
3000 PRINT 93, JT1( JJ )
3100 PRINT 101
DDIS = 2.0*ZLS(ISTT)/M
IF ( IAXOPS(ISTT) .EQ. 2 ) DDIS = DCIS(ISTT)*DDIS
COMMENT - CONVERT OUTPUT DISTANCES TO BE COMPATIBLE WITH STIFFNESS INPUT
IF ( IAXOPS(ISTT) .EQ. 3 ) DDIS = DC2S(ISTT)*DDIS
DIS = - DDIS
J6 = 0
DO 3600 I = 1, MP22
DIS = DIS + DDIS
J1 = J6 + 1
J2 = J1 + 1
J3 = J2 + 1
J4 = J3 + 1
J5 = J4 + 1
J6 = J5 + 1
COMMENT - PRINT COMPLETE MEMBER RESULTS
PRINT 111, DIS, DM(IJ1, KK), DM(IJ2, KK), DM(IJ3, KK), DM(IJ4, KK),
2 DM(IJ5, KK), DM(IJ6, KK)

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3600	CONTINUE	25MRO
	GO TO 7100	25MRO
5100	CONTINUE	75MRO
COMMENT -	PRINT PARTIAL MEMBER RESULTS	19MRO
	PRINT 201, JT1(JJ), JT2(JJ), DM(4, KK), DM(L7M3, KK), DM(5, KK),	75MRO
2	DM(L7M2, KK), DM(6, KK), DM(L7M1, KK)	25MRO
7100	CONTINUE	25MRO
	PRINT 301, DM(L7, KK)	25MRO
	IOPL = IOPOP(ISTT)	25MRO
	RETURN	25MRO
	END	25MRO

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APPENDIX 6

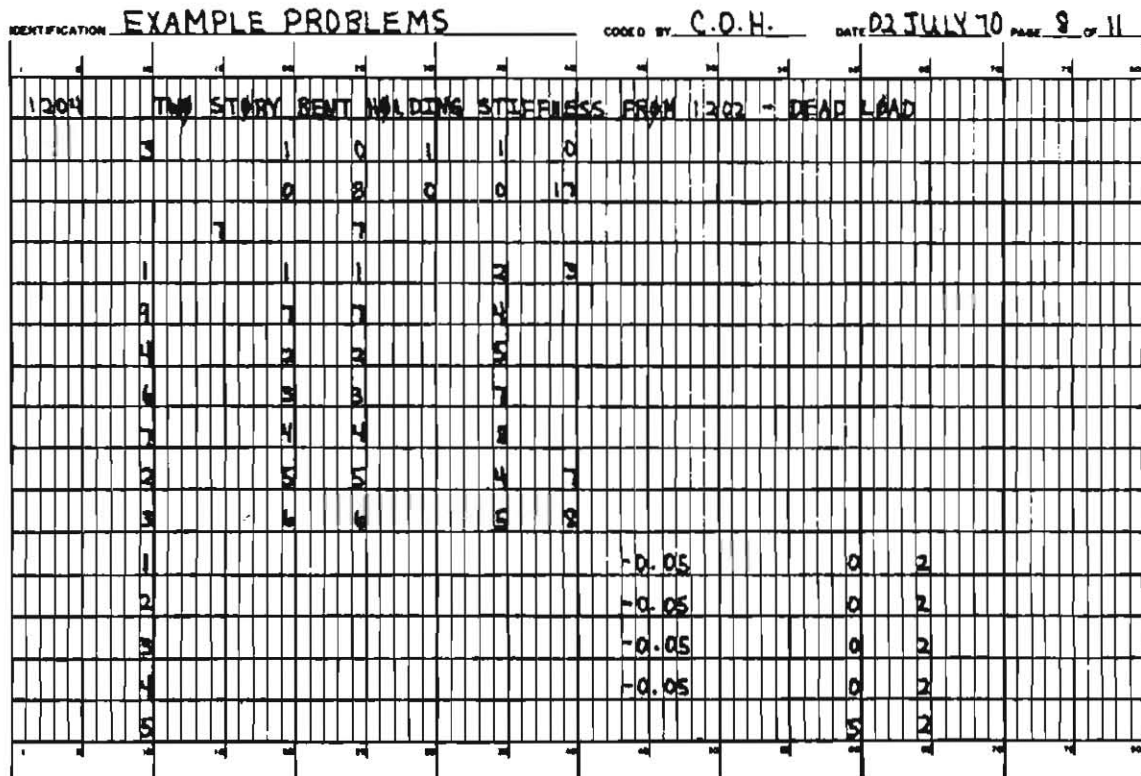
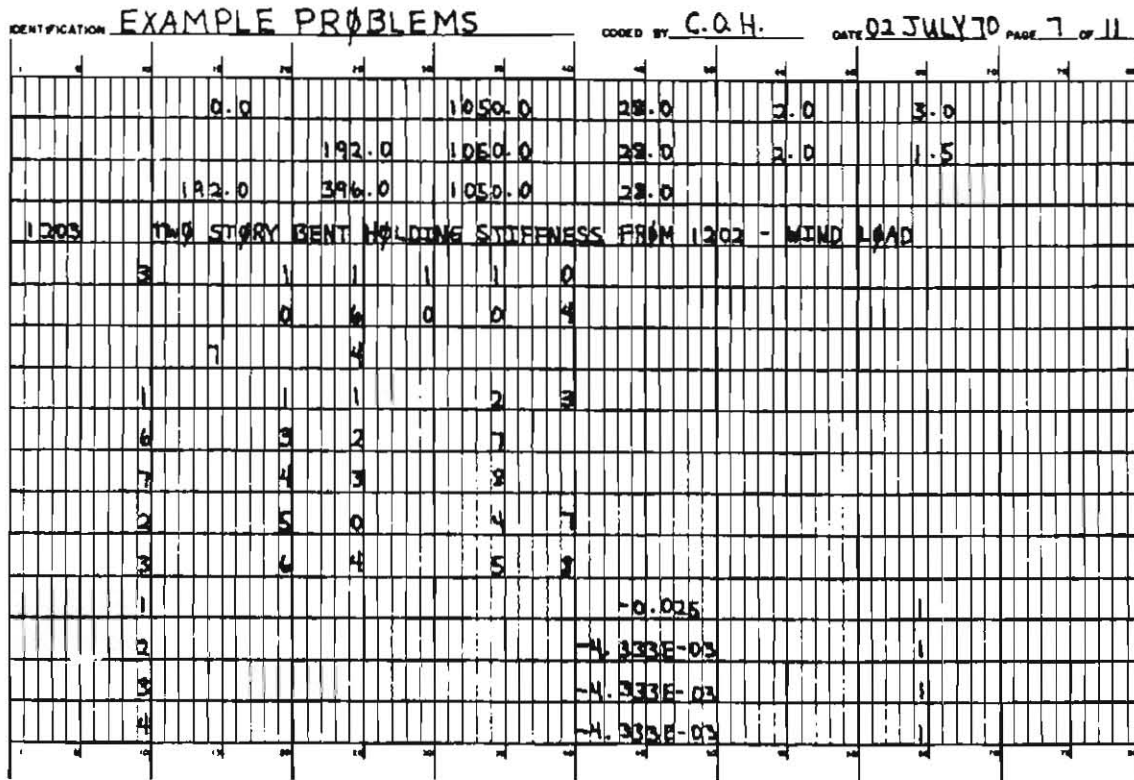
INPUT FOR EXAMPLE PROBLEMS

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IDENTIFICATION		EXAMPLE PROBLEMS	CODED BY C.O.H.		DATE 02 JULY 70		PAGE 3 OF 11	
2	3.000E+04	2.5	2.5	1.0	1	1	1	1
3	3.000E+04	2.25	2.25	1.0	1	1	1	1
4	3.000E+04	1.0	1.0	2.0	1	1	1	1
1102 EFFECT OF RIGID CONNECTIONS ON TRUSS								
1	3.000E+04	4.0	4.0	4.0	1	1		
2	3.000E+04	2.25	2.25	1.0	1	1		
3	3.000E+04	2.25	2.25	1.0	1	1		
4	3.000E+04	1.0	1.0	2.0	1	1		
1103 EFFECT OF RIGID CONNECTIONS AND ROLLER FREEZING ON TRUSS								
1.2								
			1.000E+00					
1201 TWO STORY RENT WITHOUT INTERIOR COLUMN - LIVE LOAD								

IDENTIFICATION		EXAMPLE PROBLEMS	CODED BY C.O.H.		DATE 02 JULY 70		PAGE 4 OF 11	
1	1.000E+00	0	0	0	0	0	0.00	
2	0.0	227.75	0	0	0	0		
3	120.0	0.0	10	0	0	0		
4	120.0	0	0	0	0	0		
5	0.0	375.0	10	0	0	0		
6	160.0	175.24	10	0	0	0		
1.000E+00 1.000E+00								
1.000E+00 1.000E+00 1.000E+00								



APPENDIX 7

SELECTED OUTPUT FOR EXAMPLE PROBLEMS

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02 JULY 78 - COM - MM
 EXAMPLE PROBLEMS FOR REPORT

PROB
 1101 TRUSS WITH UNSYMT LOADS - PG 270 STRUCTURAL ANALYSIS - MCCORMAC

TABLE 1 - PROGRAM CONTROL DATA
 PROBLEM TYPE 1

INPUT TABLES		
TABLE NUMBER	HOLD DATA FROM LAST PROBLEM (1 = YES, 0 = NO)	NUMBER OF CARDS ADDED FOR THIS PROBLEM
2	-0	5
3	-0	14
4	-0	7
5	-0	4
6	-0	-0
7	-0	-0

OUTPUT TABLES	
TABLE NUMBER	SUPPRESS OUTPUT (1 = YES, 0 = NO)
8	-0
9	-0
10	-0

PROB (CONTD)
 1101 TRUSS WITH UNSYMT LOADS - PG 270 STRUCTURAL ANALYSIS - MCCORMAC

TABLE 2 - FRAME GEOMETRY DATA

NUMBER OF JOINTS IN FRAME = 12
 REFERENCE JOINT IS JOINT 1 AT X = 0. AND Y = 0.
 JOINT TOLERANCE IS 5.000E-02

INPUT OF JOINT OFFSETS

FROM JOINT	X-OFFSET	Y-OFFSET	TO JOINT	TO	TO	TO	TO	TO	TO
1	2.400E+02	0.	3	5	7	9	11	12	
1	2.400E+02	2.400E+02	2						
2	2.400E+02	0.	4	6	8	10			
10	2.400E+02	-2.400E+02	12						

COMPUTED JOINT COORDINATES

JOINT	X	Y
1	0.	0.
2	2.400E+02	2.400E+02
3	2.400E+02	0.
4	4.800E+02	2.400E+02
5	4.800E+02	0.
6	7.200E+02	2.400E+02
7	7.200E+02	0.
8	9.600E+02	2.400E+02
9	9.600E+02	0.
10	1.200E+03	2.400E+02
11	1.200E+03	0.
12	1.440E+03	0.

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17	3	2	*	0	2.400E+02	0.	2.400E+02
18	5	4	*	0	2.400E+02	0.	2.400E+02
19	7	6	*	0	2.400E+02	0.	2.400E+02
20	9	8	*	0	2.400E+02	0.	2.400E+02
21	11	10	*	0	2.400E+02	0.	2.400E+02

TABLE 3 - MEMBER LOCATION DATA

NUMBER OF MEMBER STIFFNESS TYPES = 4
NUMBER OF MEMBER LOAD TYPES = 0

INPUT OF MEMBER LOCATIONS

FROM JOINT	STIFF TYPE	LOAD TYPE	TO JOINT	TO	TO	TO	TO	TO	TO	TO
1	1	0	3	5	7	9	11	12		
2	1	0	4	6	8	10				
3	2	0	2							
4	2	0	0							
5	2	0	10							
6	3	0	5							
7	3	0	7							
8	3	0	12							
9	4	0	2							
10	4	0	4							
11	4	0	6							
12	4	0	8							
13	4	0	10							

COMPUTED MEMBER NUMBERS, LENGTHS, AND OFFSETS

MEMBER NUMB	FROM JOINT	TO JOINT	STIFF TYPE	LOAD TYPE	LENGTH	X-OFFSET	Y-OFFSET
1	1	3	1	0	2.400E+02	2.400E+02	0.
2	3	5	1	0	2.400E+02	2.400E+02	0.
3	5	7	1	0	2.400E+02	2.400E+02	0.
4	7	9	1	0	2.400E+02	2.400E+02	0.
5	9	11	1	0	2.400E+02	2.400E+02	0.
6	11	12	1	0	2.400E+02	2.400E+02	0.
7	2	4	1	0	2.400E+02	2.400E+02	0.
8	4	6	1	0	2.400E+02	2.400E+02	0.
9	6	8	1	0	2.400E+02	2.400E+02	0.
10	8	10	1	0	2.400E+02	2.400E+02	0.
11	1	2	2	0	3.394E+02	2.400E+02	2.400E+02
12	7	6	2	0	3.394E+02	2.400E+02	2.400E+02
13	9	10	2	0	3.394E+02	2.400E+02	2.400E+02
14	2	5	3	0	3.394E+02	2.400E+02	-2.400E+02
15	4	7	3	0	3.394E+02	2.400E+02	-2.400E+02
16	10	12	3	0	3.394E+02	2.400E+02	-2.400E+02

*** COMPUTED MEMBER NUMBERS MAY NOT AGREE WITH LAST PROBLEM ***

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TABLE 4 - JOINT DATA

INPUT OF JOINT DATA

JOINT	FORCE (X)	FORCE (Y)	MOMENT (Z)	SPRING (K)	SPRING (V)	SPRING (Z)
1	-0.	-0.	-0.	1.000E+00	1.000E+00	-0.
3	-0.	-2.000E+01	-0.	-0.	-0.	-0.
5	-0.	-2.000E+01	-0.	-0.	-0.	-0.
7	-0.	-2.000E+01	-0.	-0.	-0.	-0.
9	-0.	-1.000E+01	-0.	-0.	-0.	-0.
11	-0.	-1.000E+01	-0.	-0.	-0.	-0.
12	-0.	-0.	-0.	-0.	1.000E+00	-0.

ACCUMULATED JOINT DATA

SAME AS INPUT FOR THIS PROBLEM

PROB (CONTD)
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TABLE 5 - MEMBER STIFFNESS DATA

STIFF TYPE	MOD OF ELAST	PRISMATIC I	PHISMATIC A	MU CARUS	AXIS OPT	OUTPUT OPT	PIN FROM	PIN TO
1	3.000E+04	4.000E+00	4.000E+00	-0	1	1	1	1
2	3.000E+04	2.250E+00	3.000E+00	-0	1	1	1	1
3	3.000E+04	2.250E+00	3.000E+00	-0	1	1	1	1
4	3.000E+04	1.000E+00	2.000E+00	-0	1	1	1	1

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TABLE 6 - MEMBER LOAD DATA

NO DATA

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TABLE 7 - COMPILATION TABLE

NO DATA

PROB (CONTD)
1101 TRUSS WITH UNSYMT LOADS - PG 270 STRUCTURAL ANALYSIS - MCCORMAC

TABLE 8 - JOINT DISPLACEMENTS AND REACTIONS

JOINT	DISPLACEMENTS			REACTIONS		
	DISP(X)	DISP(Y)	ROTATION(Z)	REACT(X)	REACT(Y)	REACT(Z)
1	1.692E-10	-4.500E-08	1.000E+99	-1.592E-11	4.500E+01	0.
2	5.967E-01	-9.361E-01	1.000E+99	0.	0.	0.
3	0.000E+02	-1.016E+00	1.000E+99	0.	0.	0.
4	4.347E-01	-1.541E+00	1.000E+99	0.	0.	0.
5	1.000E-01	-1.541E+00	1.000E+99	0.	0.	0.
6	3.067E-01	-1.736E+00	1.000E+99	0.	0.	0.
7	3.200E-01	-1.736E+00	1.000E+99	0.	0.	0.
8	1.567E-01	-1.459E+00	1.000E+99	0.	0.	0.
9	4.400E-01	-1.399E+00	1.000E+99	0.	0.	0.
10	3.067E-02	-8.073E-01	1.000E+99	0.	0.	0.
11	5.100E-01	-8.473E-01	1.000E+99	0.	0.	0.
12	5.800E-01	-3.500E-00	1.000E+99	0.	3.500E+01	0.

PROB (CONTD)
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TABLE 9 - MEMBER RESULTS

MEMBER NUMBER 1 STIFF TYPE 1 LOAD TYPE 0
LENGTH = 2.400E+02 ALPHA = 1.000E+00 BETA = 0.
GOES FROM JOINT 1 TO JOINT 3
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 1		AT JOINT 3	
AXIAL FORCE =	4.500E+01	AXIAL FORCE =	4.500E+01
SHEAR =	-1.036E-10	SHEAR =	1.865E-10
MOMENT =	-4.441E-12	MOMENT =	4.441E-11

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 4.641E-10

MEMBER NUMBER 2 STIFF TYPE 1 LOAD TYPE 0
LENGTH = 2.400E+02 ALPHA = 1.000E+00 BETA = 0.
GOES FROM JOINT 3 TO JOINT 5
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 3		AT JOINT 5	
AXIAL FORCE =	4.500E+01	AXIAL FORCE =	4.500E+01
SHEAR =	-5.240E-10	SHEAR =	4.130E-10
MOMENT =	-5.320E-11	MOMENT =	1.465E-10

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 1.006E-09

MEMBER NUMBER 3 STIFF TYPE 1 LOAD TYPE 0
LENGTH = 2.400E+02 ALPHA = 1.000E+00 BETA = 0.
GOES FROM JOINT 5 TO JOINT 7
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 5		AT JOINT 7	
AXIAL FORCE =	7.000E+01	AXIAL FORCE =	7.000E+01
SHEAR =	-8.200E-10	SHEAR =	5.715E-10
MOMENT =	-2.662E-11	MOMENT =	3.616E-10

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 1.255E-09

PROB (CONTD)
1101 TRUSS WITH UNSYMT LOADS - PG 270 STRUCTURAL ANALYSIS - MCCORMAC

TABLE 6 - MEMBER RESULTS (CONTD)

MEMBER NUMBER 4 STIFF TYPE 1 LOAD TYPE A
LENGTH = 2.400E+02 ALPHA = 1.000E+00 BETA = 0.
GOES FROM JOINT 7 TO JOINT 9
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 7		AT JOINT 9	
AXIAL FORCE =	6.000E+01	AXIAL FORCE =	4.000E+01
SHEAR =	-7.268E-10	SHEAR =	4.824E-10
MOMENT =	-4.485E-10	MOMENT =	2.034E-10

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 1.033E-09

MEMBER NUMBER 5 STIFF TYPE 1 LOAD TYPE 0
LENGTH = 2.400E+02 ALPHA = 1.000E+00 BETA = 0.
GOES FROM JOINT 9 TO JOINT 11
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 9		AT JOINT 11	
AXIAL FORCE =	3.500E+01	AXIAL FORCE =	3.500E+01
SHEAR =	-5.773E-10	SHEAR =	3.271E-10
MOMENT =	-2.398E-10	MOMENT =	1.221E-10

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 1.008E-09

MEMBER NUMBER 6 STIFF TYPE 1 LOAD TYPE A
LENGTH = 2.400E+02 ALPHA = 1.000E+00 BETA = 0.
GOES FROM JOINT 11 TO JOINT 12
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 11		AT JOINT 12	
AXIAL FORCE =	3.500E+01	AXIAL FORCE =	3.500E+01
SHEAR =	-3.249E-10	SHEAR =	8.845E-11
MOMENT =	-7.327E-11	MOMENT =	4.106E-12

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 4.563E-10

PROB (CONTD)
1101 TRUSS WITH UNSYMT LOADS - PG 270 STRUCTURAL ANALYSIS - MCCORMAC

TABLE 4 - MEMBER RESULTS (CONTD)

MEMBER NUMBER 7 STIFF TYPE 1 LOAD TYPE 0
LENGTH = 2.400E+02 ALPHA = 1.000E+00 BETA = 0.
GOES FROM JOINT 2 TO JOINT 4
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 2		AT JOINT 4	
AXIAL FORCE =	-7.000E+01	AXIAL FORCE =	-7.000E+01
SHEAR =	-5.003E-10	SHEAR =	4.989E-10
MOMENT =	-1.066E-10	MOMENT =	4.186E-10

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 1.004E-09

MEMBER NUMBER 8 STIFF TYPE 1 LOAD TYPE A
LENGTH = 2.400E+02 ALPHA = 1.000E+00 BETA = 0.
GOES FROM JOINT 4 TO JOINT 6
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 4		AT JOINT 6	
AXIAL FORCE =	-7.500E+01	AXIAL FORCE =	-7.500E+01
SHEAR =	-8.051E-10	SHEAR =	5.647E-10
MOMENT =	3.497E-11	MOMENT =	3.370E-10

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 1.002E-09

MEMBER NUMBER 9 STIFF TYPE 1 LOAD TYPE A
LENGTH = 2.400E+02 ALPHA = 1.000E+00 BETA = 0.
GOES FROM JOINT 6 TO JOINT 8
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 6		AT JOINT 8	
AXIAL FORCE =	-7.500E+01	AXIAL FORCE =	-7.500E+01
SHEAR =	-8.293E-10	SHEAR =	5.642E-10
MOMENT =	1.887E-11	MOMENT =	4.211E-10

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 1.010E-09

PROB (CONTU)
1101 TRUSS WITH UNSYMT LOADS - PB 270 STRUCTURAL ANALYSIS - MCCORMAC

TABLE 9 - MEMBER RESULTS (CONTO)

MEMBER NUMBER 10 STIFF TYPE 1 LOAD TYPE 0
LENGTH = 2.400E+02 ALPHA = 1.000E+00 BETA = 0.
GOES FROM JOINT 8 TO JOINT 10
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 8 AT JOINT 10

AXIAL FORCE = -6.000E+01 AXIAL FORCE = -6.000E+01
SHEAR = -6.666E-10 SHEAR = 3.666E-10
MOMENT = -1.000E-10 MOMENT = 2.620E-10

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 0.016E-10

MEMBER NUMBER 11 STIFF TYPE 2 LOAD TYPE 0
LENGTH = 3.394E+02 ALPHA = 7.071E-01 BETA = 7.071E-01
GOES FROM JOINT 1 TO JOINT 2
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 1 AT JOINT 2

AXIAL FORCE = -6.364E+01 AXIAL FORCE = -6.364E+01
SHEAR = 5.329E-11 SHEAR = -7.223E-11
MOMENT = -1.766E-12 MOMENT = 1.457E-10

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 7.839E-10

MEMBER NUMBER 12 STIFF TYPE 2 LOAD TYPE 0
LENGTH = 3.394E+02 ALPHA = 7.071E-01 BETA = 7.071E-01
GOES FROM JOINT 7 TO JOINT 8
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 7 AT JOINT 8

AXIAL FORCE = 2.121E+01 AXIAL FORCE = 7.121E+01
SHEAR = 1.039E-10 SHEAR = -1.455E-10
MOMENT = -6.000E-11 MOMENT = 7.000E-11

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 9.300E-10

PROB (CONTU)
1101 TRUSS WITH UNSYMT LOADS - PB 270 STRUCTURAL ANALYSIS - MCCORMAC

TABLE 9 - MEMBER RESULTS (CONTO)

MEMBER NUMBER 13 STIFF TYPE 2 LOAD TYPE 0
LENGTH = 3.394E+02 ALPHA = 7.071E-01 BETA = 7.071E-01
GOES FROM JOINT 9 TO JOINT 10
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 9 AT JOINT 10

AXIAL FORCE = 3.536E+01 AXIAL FORCE = 3.536E+01
SHEAR = 1.374E-10 SHEAR = -1.039E-10
MOMENT = -7.065E-12 MOMENT = 7.728E-11

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 2.343E-10

MEMBER NUMBER 14 STIFF TYPE 3 LOAD TYPE 0
LENGTH = 3.394E+02 ALPHA = 7.071E-01 BETA = -7.071E-01
GOES FROM JOINT 2 TO JOINT 5
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 2 AT JOINT 5

AXIAL FORCE = 3.536E+01 AXIAL FORCE = 3.536E+01
SHEAR = 6.287E-11 SHEAR = -9.263E-11
MOMENT = 7.065E-12 MOMENT = 5.785E-11

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 2.262E-10

MEMBER NUMBER 15 STIFF TYPE 3 LOAD TYPE 0
LENGTH = 3.394E+02 ALPHA = 7.071E-01 BETA = -7.071E-01
GOES FROM JOINT 4 TO JOINT 7
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 4 AT JOINT 7

AXIAL FORCE = 7.071E+00 AXIAL FORCE = 7.071E+00
SHEAR = 1.062E-10 SHEAR = -1.090E-10
MOMENT = -1.000E-11 MOMENT = 1.196E-10

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 7.261E-10

PROB (CONTD)
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TABLE 8 - MEMBER RESULTS (CONTD)

MEMBER NUMBER 16 STIFF TYPE 3 LOAD TYPE 0
 LENGTH = 3.394E+02 ALPHA = 7.071E-01 BETA = -7.071E-01
 GOES FROM JOINT 10 TO JOINT 12
 ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 10	AT JOINT 12
AXIAL FORCE = -4.950E+01	AXIAL FORCE = -4.950E+01
SHEAR = 2.248E-11	SHEAR = 8.431E-12
MOMENT = -1.413E-11	MOMENT = -2.318E-11

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 1.132E-10

MEMBER NUMBER 17 STIFF TYPE 4 LOAD TYPE 0
 LENGTH = 2.400E+02 ALPHA = 0. BETA = 1.000E+00
 GOES FROM JOINT 3 TO JOINT 2
 ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 3	AT JOINT 2
AXIAL FORCE = 2.000E+01	AXIAL FORCE = 2.000E+01
SHEAR = -2.442E-11	SHEAR = 3.095E-11
MOMENT = -0.	MOMENT = 3.428E-11

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 2.842E-10

MEMBER NUMBER 18 STIFF TYPE 4 LOAD TYPE 0
 LENGTH = 2.400E+02 ALPHA = 0. BETA = 1.000E+00
 GOES FROM JOINT 5 TO JOINT 4
 ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 5	AT JOINT 4
AXIAL FORCE = -5.000E+00	AXIAL FORCE = -5.000E+00
SHEAR = -2.077E-11	SHEAR = 7.247E-11
MOMENT = -8.604E-12	MOMENT = 2.567E-11

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 3.553E-10

PROB (CONTD)
1101 TRUSS WITH UNSYMT LOADS - PG 270 STRUCTURAL ANALYSIS - MCCORMAC

TABLE 9 - MEMBER RESULTS (CONTD)

MEMBER NUMBER 19 STIFF TYPE 4 LOAD TYPE 0
 LENGTH = 2.400E+02 ALPHA = 0. BETA = 1.000E+00
 GOES FROM JOINT 7 TO JOINT 6
 ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 7	AT JOINT 6
AXIAL FORCE = 2.132E-09	AXIAL FORCE = -1.402E-09
SHEAR = -4.441E-11	SHEAR = 3.101E-11
MOMENT = 1.112E-11	MOMENT = 2.944E-11

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 3.553E-10

MEMBER NUMBER 20 STIFF TYPE 4 LOAD TYPE 0
 LENGTH = 2.400E+02 ALPHA = 0. BETA = 1.000E+00
 GOES FROM JOINT 9 TO JOINT 8
 ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 9	AT JOINT 8
AXIAL FORCE = -1.500E+01	AXIAL FORCE = -1.500E+01
SHEAR = -4.746E-11	SHEAR = 2.239E-11
MOMENT = 2.776E-13	MOMENT = 1.360E-11

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 2.842E-10

MEMBER NUMBER 21 STIFF TYPE 4 LOAD TYPE 0
 LENGTH = 2.400E+02 ALPHA = 0. BETA = 1.000E+00
 GOES FROM JOINT 11 TO JOINT 10
 ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 11	AT JOINT 10
AXIAL FORCE = 1.000E+01	AXIAL FORCE = 1.000E+01
SHEAR = -4.959E-11	SHEAR = 1.527E-11
MOMENT = -1.554E-11	MOMENT = 2.982E-12

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 1.421E-10

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TABLE 10 - JOINT EQUILIBRIUM ERRORS

JOINT	ERR(1) FORCE	ERR(2) FORCE	ERR(3) MOMENT
1	4.229E-11	1.344E-10	-6.207E-12
2	-2.517E-09	2.338E-09	-2.793E-10
3	-4.004E-10	2.014E-09	-9.770E-11
4	-3.447E-09	4.036E-09	-4.281E-10
5	-1.906E-09	4.124E-09	-2.374E-10
6	-1.417E-09	2.886E-09	-3.473E-10
7	-1.498E-09	5.326E-09	-9.628E-10
8	1.924E-10	3.191E-09	-6.195E-10
9	-1.132E-09	3.382E-09	-4.508E-10
10	-1.526E-10	2.043E-09	-3.355E-10
11	-2.394E-09	1.693E-09	-2.109E-10
12	-1.397E-09	4.818E-10	1.700E-11

02 JULY 78 - COM - MM
 EXAMPLE PROBLEMS FOR REPORT

PROB (CONTD)
 1102 EFFECT OF RIGID CONNECTIONS ON TRUSS

PROB
 1102 EFFECT OF RIGID CONNECTIONS ON TRUSS

TABLE 2 - FRAME GEOMETRY DATA

HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS THE FOLLOWING

NONE

TABLE 1 - PROGRAM CONTROL DATA
 PROBLEM TYPE 1

COMPUTED JOINT COORDINATES

INPUT TABLES

TABLE NUMBER	HOLD DATA FROM LAST PROBLEM (1 = YES, 0 = NO)	NUMBER OF CARDS ADDED FOR THIS PROBLEM
2	1	-0
3	1	-0
4	1	-0
5	-0	4
6	-0	-0
7	-0	-0

JOINT	X	Y
1	0.	0.
2	2.400E+02	2.400E+02
3	2.400E+02	0.
4	4.800E+02	2.400E+02
5	4.800E+02	0.
6	7.200E+02	2.400E+02
7	7.200E+02	0.
8	9.600E+02	2.400E+02
9	9.600E+02	0.
10	1.200E+03	2.400E+02
11	1.200E+03	0.
12	1.440E+03	0.

OUTPUT TABLES

TABLE NUMBER	SUPPRESS OUTPUT (1 = YES, 0 = NO)
8	-0
9	-0
10	-0

PROB (CONTU)
1102 EFFECT OF RIGID CONNECTIONS ON TRUSS

TABLE 3 - MEMBER LOCATION DATA

HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS THE FOLLOWING

NONE

COMPUTED MEMBER NUMBERS, LENGTHS, AND OFFSETS

MEMBER NUMB	FROM JOINT	TO JOINT	STIFF TYPE	LOAD TYPE	LENGTH	X-OFFSET	Y-OFFSET
1	1	3	1	0	2.400E+02	2.400F+02	0.
2	3	5	1	0	2.400E+02	2.400F+02	0.
3	5	7	1	0	2.400E+02	2.400F+02	0.
4	7	9	1	0	2.400E+02	2.400F+02	0.
5	9	11	1	0	2.400E+02	2.400F+02	0.
6	11	12	1	0	2.400E+02	2.400F+02	0.
7	2	4	1	0	2.400E+02	2.400F+02	0.
8	4	6	1	0	2.400E+02	2.400F+02	0.
9	6	8	1	0	2.400E+02	2.400F+02	0.
10	8	10	1	0	2.400E+02	2.400F+02	0.
11	1	2	2	0	3.394E+02	2.400F+02	2.400E+02
12	7	8	2	0	3.394E+02	2.400F+02	2.400E+02
13	9	10	2	0	3.394E+02	2.400F+02	2.400E+02
14	2	5	J	0	3.394E+02	2.400F+02	-2.400E+02
15	4	7	J	0	3.394E+02	2.400F+02	-2.400E+02
16	10	12	J	0	3.394E+02	2.400F+02	-2.400E+02
17	3	2	4	0	2.400E+02	0.	2.400F+02
18	5	4	4	0	2.400E+02	0.	2.400F+02
19	7	6	4	0	2.400E+02	0.	2.400F+02
20	9	8	4	0	2.400E+02	0.	2.400F+02
21	11	10	4	0	2.400E+02	0.	2.400F+02

*** COMPUTED MEMBER NUMBERS AGREE WITH LAST PROBLEM ***

PROB (CONTU)
1102 EFFECT OF RIGID CONNECTIONS ON TRUSS

TABLE 4 - JOINT DATA

HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS THE FOLLOWING

NONE

ACCUMULATED JOINT DATA

JOINT	FORCE (X)	FORCE (Y)	MOMENT (Z)	SPRING (X)	SPRING (Y)	SPRING (Z)
1	0.	0.	0.	1.000E+99	1.000E+99	0.
3	0.	-2.000E+01	0.	0.	0.	0.
5	0.	-2.000E+01	0.	0.	0.	0.
7	0.	-2.000E+01	0.	0.	0.	0.
9	0.	-1.000E+01	0.	0.	0.	0.
11	0.	-1.000E+01	0.	0.	0.	0.
12	0.	0.	0.	0.	1.000E+99	0.

PROB (CONTU)
1102 EFFECT OF RIGID CONNECTIONS ON TRUSS

TABLE 5 - MEMBER STIFFNESS DATA

STIFF TYPE	MOD OF ELAST	PRISMATIC I	PRISMATIC A	NU CARUS	AXIS OPT	OUTPUT OPT	PIN FROM	PIN TO
1	3.000E+04	4.000E+00	4.000E+00	-0	1	1	-0	-0
2	3.000E+04	2.250E+00	3.000E+00	-0	1	1	-0	-0
3	3.000E+04	2.250E+00	3.000E+00	-0	1	1	-0	-0
4	3.000E+04	1.000E+00	2.000E+00	-0	1	1	-0	-0

PROB (CONTU)
1102 EFFECT OF RIGID CONNECTIONS ON TRUSS

TABLE 6 - MEMBER LOAD DATA

NO DATA

PROB (CONTU)
1102 EFFECT OF RIGID CONNECTIONS ON TRUSS

TABLE 7 - COMPILATION TABLE

NO DATA

PROB (CONTU)
1102 EFFECT OF RIGID CONNECTIONS ON TRUSS

TABLE 8 - JOINT DISPLACEMENTS AND REACTIONS

JOINT	DISPLACEMENTS			REACTIONS		
	DISP(X)	DISP(Y)	ROTATION(Z)	REACT(X)	REACT(Y)	REACT(Z)
1	4.343-110	-4.500E-98	-4.391E-03	-4.343E-11	4.500E+01	0.
2	5.966E-01	-9.359E-01	-2.670E-03	0.	0.	0.
3	8.998E-02	-1.016E+00	-3.172E-03	0.	0.	0.
4	4.566E-01	-1.561E+00	-1.683E-03	0.	0.	0.
5	1.800E-01	-1.541E+00	-1.513E-03	0.	0.	0.
6	3.066E-01	-1.735E+00	2.246E-04	0.	0.	0.
7	3.200E-01	-1.735E+00	2.702E-04	0.	0.	0.
8	1.567E-01	-1.459E+00	1.922E-03	0.	0.	0.
9	4.400E-01	-1.399E+00	1.853E-03	0.	0.	0.
10	3.067E-02	-8.072E-01	2.604E-03	0.	0.	0.
11	5.100E-01	-8.472E-01	2.859E-03	0.	0.	0.
12	5.799E-01	-3.500E-98	3.596E-03	0.	3.500E+01	0.

PROB (CONTU)
1102 EFFECT OF RIGID CONNECTIONS ON TRUSS

TABLE 9 - MEMBER RESULTS

MEMBER NUMBER 1 STIFF TYPE 1 LOAD TYPE 1
LENGTH = 2.400E+02 ALPHA = 1.000E+00 BETA = 0.
GOES FROM JOINT 1 TO JOINT 3
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 1		AT JOINT 3	
AXIAL FORCE	= 4.499E+01	AXIAL FORCE	= 4.499E+01
SHEAR	= 1.129E-02	SHEAR	= 1.129E-02
MOMENT	= -7.451E-1	MOMENT	= 1.963E+00

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 7.775E-10

MEMBER NUMBER 2 STIFF TYPE 1 LOAD TYPE 1
LENGTH = 2.400E+02 ALPHA = 1.000E+00 BETA = 0.
GOES FROM JOINT 3 TO JOINT 5
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 3		AT JOINT 5	
AXIAL FORCE	= 4.500E+01	AXIAL FORCE	= 4.500E+01
SHEAR	= -3.861E-03	SHEAR	= -3.861E-03
MOMENT	= -3.861E-03	MOMENT	= -3.861E-03

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 7.775E-10

MEMBER NUMBER 21 STIFF TYPE 1 LOAD TYPE 1
LENGTH = 2.400E+02 ALPHA = 1.000E+00 BETA = 0.
GOES FROM JOINT 11 TO JOINT 10
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 11		AT JOINT 10	
AXIAL FORCE	= -1.498E+01	AXIAL FORCE	= -1.498E+01
SHEAR	= 4.423E-03	SHEAR	= 4.423E-03
MOMENT	= -5.221E-1	MOMENT	= 5.221E-01

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 2.062E-10

MEMBER NUMBER 21 STIFF TYPE 1 LOAD TYPE 1
LENGTH = 2.400E+02 ALPHA = 1.000E+00 BETA = 0.
GOES FROM JOINT 11 TO JOINT 10
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 11		AT JOINT 10	
AXIAL FORCE	= 9.991E+00	AXIAL FORCE	= 9.991E+00
SHEAR	= 4.747E-03	SHEAR	= 4.747E-03
MOMENT	= -6.016E-01	MOMENT	= 6.016E-01

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 1.776E-10

PROB (CONTD)
1102 EFFECT OF RIGID CONNECTIONS ON TRUSS

TABLE 10 - JOINT EQUILIBRIUM ERRORS

JOINT	ERR(X) FORCE	ERR(Y) FORCE	ERR(Z) MOMENT
1	-2.621E-07	-1.501E-06	-1.467E-04
2	1.022E-06	-1.960E-06	-4.970E-05
3	-7.916E-07	2.369E-06	-4.425E-05
4	4.775E-07	1.733E-06	-1.360E-04
5	-4.002E-07	-1.305E-06	2.950E-05
6	-1.007E-07	2.957E-07	1.552E-05
7	2.001E-07	1.000E-06	2.507E-05
8	-8.309E-07	1.606E-06	1.321E-04
9	8.106E-07	-1.667E-06	2.046E-05
10	-1.005E-06	-1.007E-06	4.071E-05
11	7.394E-07	1.400E-06	2.601E-05
12	2.210E-07	-9.506E-07	8.040E-05

02 JULY 70 - COM - MM
 EXAMPLE PROBLEMS FOR REPORT

PROB
 1103 EFFECT OF RIGID CONNECTIONS AND ROLLER FREEZING ON TRUSS

TABLE 1 - PROGRAM CONTROL DATA
 PROBLEM TYPE 1

INPUT TABLES

TABLE NUMBER	HOLD DATA FROM LAST PROBLEM (1 = YES, 0 = NO)	NUMBER OF CARDS ADDED FOR THIS PROBLEM
2	1	-0
3	1	-0
4	1	1
5	1	-0
6	1	-0
7	-0	-0

OUTPUT TABLES

TABLE NUMBER	SUPPRESS OUTPUT (1 = YES, 0 = NO)
8	-0
9	-0
10	-0

PROB (CONTD)
 1103 EFFECT OF RIGID CONNECTIONS AND ROLLER FREEZING ON TRUSS

TABLE 2 - FRAME GEOMETRY DATA

HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS THE FOLLOWING

NONE

COMPUTED JOINT COORDINATES

JOINT	X	Y
1	0.	0.
2	2.400E+02	2.400E+02
3	2.400E+02	0.
4	4.800E+02	2.400E+02
5	4.800E+02	0.
6	7.200E+02	2.400E+02
7	7.200E+02	0.
8	9.600E+02	2.400E+02
9	9.600E+02	0.
10	1.200E+03	2.400E+02
11	1.200E+03	0.
12	1.440E+03	0.

PROB (CONTU)
1103 EFFECT OF RIGID CONNECTIONS AND ROLLER FREEZING ON TRUSS

TABLE 3 - MEMBER LOCATION DATA

HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS THE FOLLOWING

NONE

COMPUTED MEMBER NUMBERS, LENGTHS, AND OFFSETS

MEMBER NUMB	FROM JOINT	TO JOINT	STIFF TYPE	LOAD TYPE	LENGTH	X-OFFSET	Y-OFFSET
1	1	3	1	0	2.400E+02	2.400E+02	0.
2	3	5	1	0	2.400E+02	2.400E+02	0.
3	5	7	1	0	2.400E+02	2.400E+02	0.
4	7	9	1	0	2.400E+02	2.400E+02	0.
5	9	11	1	0	2.400E+02	2.400E+02	0.
6	11	12	1	0	2.400E+02	2.400E+02	0.
7	2	4	1	0	2.400E+02	2.400E+02	0.
8	4	6	1	0	2.400E+02	2.400E+02	0.
9	6	8	1	0	2.400E+02	2.400E+02	0.
10	8	10	1	0	2.400E+02	2.400E+02	0.
11	1	2	2	0	3.394E+02	2.400E+02	2.400E+02
12	7	8	2	0	3.394E+02	2.400E+02	2.400E+02
13	9	10	2	0	3.394E+02	2.400E+02	2.400E+02
14	2	5	3	0	3.394E+02	2.400E+02	-2.400E+02
15	4	7	3	0	3.394E+02	2.400E+02	-2.400E+02
16	10	12	3	0	3.394E+02	2.400E+02	-2.400E+02
17	3	2	4	0	2.400E+02	0.	2.400E+02
18	5	4	4	0	2.400E+02	0.	2.400E+02
19	7	6	4	0	2.400E+02	0.	2.400E+02
20	9	8	4	0	2.400E+02	0.	2.400E+02
21	11	10	4	0	2.400E+02	0.	2.400E+02

*** COMPUTED MEMBER NUMBERS AGREE WITH LAST PROBLEM ***

PROB (CONTU)
1103 EFFECT OF RIGID CONNECTIONS AND ROLLER FREEZING ON TRUSS

TABLE 4 - JOINT DATA

HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS THE FOLLOWING

INPUT OF JOINT DATA

JOINT	FORCE (X)	FORCE (Y)	MOMENT (Z)	SPRING (X)	SPRING (Y)	SPRING (Z)
12	-0.	-0.	-0.	1.000E+99	-0.	-0.

ACCUMULATED JOINT DATA

JOINT	FORCE (X)	FORCE (Y)	MOMENT (Z)	SPRING (X)	SPRING (Y)	SPRING (Z)
1	0.	0.	0.	1.000E+99	1.000E+99	0.
3	0.	-2.000E+01	0.	0.	0.	0.
5	0.	-2.000E+01	0.	0.	0.	0.
7	0.	-2.000E+01	0.	0.	0.	0.
9	0.	-1.000E+01	0.	0.	0.	0.
11	0.	-1.000E+01	0.	0.	0.	0.
12	0.	0.	0.	1.000E+99	1.000E+99	0.

PROB (CONTU)
1103 EFFECT OF RIGID CONNECTIONS AND ROLLER FREEZING ON TRUSS

TABLE 5 - MEMBER STIFFNESS DATA

HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS THE FOLLOWING

NONE

PROB (CONTD)
1103 EFFECT OF RIGID CONNECTIONS AND ROLLER FREEZING ON TRUSS

TABLE 6 - MEMBER LOAD DATA

HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS THE FOLLOWING

NONE

PROB (CONTD)
1103 EFFECT OF RIGID CONNECTIONS AND ROLLER FREEZING ON TRUSS

TABLE 7 - COMPILATION TABLE

NO DATA

PROB (CONTD)
1103 EFFECT OF RIGID CONNECTIONS AND ROLLER FREEZING ON TRUSS

TABLE 8 - JOINT DISPLACEMENTS AND REACTIONS

JOINT	DISPLACEMENTS			REACTIONS		
	DISP(X)	DISP(Y)	ROTATION(Z)	REACT(X)	REACT(Y)	REACT(Z)
1	-4.833E-08	-4.500E-08	-2.967E-03	4.833E+01	4.500E+01	0.
2	3.066E-01	-6.460E-01	-1.998E-03	0.	0.	0.
3	-6.673E-03	-7.259E-01	-2.354E-03	0.	0.	0.
4	1.666E-01	-1.174E+00	-1.532E-03	0.	0.	0.
5	-1.334E-02	-1.154E+00	-1.349E-03	0.	0.	0.
6	1.665E-02	-1.349E+00	2.246E-04	0.	0.	0.
7	2.999E-02	-1.349E+00	2.702E-04	0.	0.	0.
8	-1.333E-01	-1.072E+00	1.772E-03	0.	0.	0.
9	5.332E-02	-1.012E+00	1.689E-03	0.	0.	0.
10	-2.533E-01	-5.173E-01	1.932E-03	0.	0.	0.
11	2.667E-02	-5.572E-01	2.041E-03	0.	0.	0.
12	4.833E-08	-3.500E-08	2.172E-03	-4.833E+01	3.500E+01	0.

PROB (CONTD)
1103 EFFECT OF RIGID CONNECTIONS AND ROLLER FREEZING ON TRUSS

TABLE 9 - MEMBER RESULTS

MEMBER NUMBER 1 STIFF TYPE 1 LOAD TYPE 0
LENGTH = 2.400E+02 ALPHA = 1.000E+00 BETA = 0.
GOES FROM JOINT 1 TO JOINT 3
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 1 AT JOINT 3

AXIAL FORCE = -3.337E+10 AXIAL FORCE = -3.337E+00
SHEAR = 9.108E-03 SHEAR = 9.108E-03
MOMENT = -7.867E-01 MOMENT = 1.399E+00

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 7.220E-14

MEMBER NUMBER 2 STIFF TYPE 1 LOAD TYPE 0
LENGTH = 2.400E+02 ALPHA = 1.000E+00 BETA = 0.
GOES FROM JOINT 3 TO JOINT 5
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 3 AT JOINT 5

AXIAL FORCE = -3.331E+00 AXIAL FORCE = 1.000E+00
SHEAR = -1.667E-03 SHEAR = 1.000E+00
MOMENT = 0.000E+00 MOMENT = 1.000E+00

RESPECT TO THE MEMBER AXES

AT JOINT 5 AT JOINT 9

AXIAL FORCE = -1.498E+01 AXIAL FORCE = -1.498E+01
SHEAR = 5.956E-03 SHEAR = 5.956E-03
MOMENT = -7.044E-01 MOMENT = 7.251E-01

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 7.553E-14

MEMBER NUMBER 21 STIFF TYPE 4 LOAD TYPE 0
LENGTH = 2.400E+02 ALPHA = 0. BETA = 1.000E+00
GOES FROM JOINT 11 TO JOINT 10
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 11 AT JOINT 10

AXIAL FORCE = 9.995E+00 AXIAL FORCE = 9.995E+00
SHEAR = 5.126E-03 SHEAR = 5.126E-03
MOMENT = -6.288E-01 MOMENT = 6.014E-01

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 1.776E-14

PROB (CONTD)
1103 EFFECT OF RIGID CONNECTIONS AND ROLLER FREEZING ON TRUSS

TABLE 10 - JOINT EQUILIBRIUM ERRORS

JOINT	ERR(X) FORCE	ERR(Y) FORCE	ERR(Z) MOMENT
1	-3.866E-07	-1.036E-06	-7.796E-05
2	1.195E-06	-2.130E-06	-1.946E-05
3	-8.904E-07	1.683E-06	-3.765E-05
4	6.602E-07	1.613E-06	-1.605E-04
5	-6.348E-07	-1.269E-06	-2.421E-05
6	-1.874E-07	8.829E-07	1.552E-05
7	2.817E-07	2.636E-06	2.988E-05
8	-1.011E-06	1.286E-06	1.566E-04
9	1.040E-06	-1.631E-06	7.424E-05
10	-1.256E-06	-2.037E-06	1.047E-05
11	8.008E-07	7.210E-07	1.941E-05
12	3.469E-07	-4.942E-07	1.767E-05

82 JULY 78 - CON - MH
 EXAMPLE PROBLEMS FOR REPORT

PROB (CONTD)
 1201 TWO STORY BENT WITHOUT INTERIOR COLUMN - LIVE LOAD

PROB
 1201 TWO STORY BENT WITHOUT INTERIOR COLUMN - LIVE LOAD

TABLE 1 - PROGRAM CONTROL DATA
 PROBLEM TYPE 1

INPUT TABLES		
TABLE NUMBER	HOLD DATA FROM LAST PROBLEM (1 = YES, 0 = NO)	NUMBER OF CARDS ADDED FOR THIS PROBLEM
2	-0	6
3	-0	7
4	-0	2
5	-0	13
6	-0	18
7	-0	6

OUTPUT TABLES	
TABLE NUMBER	SUPPRESS OUTPUT (1 = YES, 0 = NO)
8	-0
9	-0
10	-0

TABLE 2 - FRAME GEOMETRY DATA

NUMBER OF JOINTS IN FRAME = 8
 REFERENCE JOINT IS JOINT 1 AT X = 0. AND Y = 0.
 JOINT TOLERANCE IS 3.000E-02

INPUT OF JOINT OFFSETS						
FROM JOINT	X-OFFSET	Y-OFFSET	TO JOINT	TO	TO	TO
1	0.	2.278E+02	2	3		
2	4.800E+02	-0.	4	7		
3	4.800E+02	3.312E+01	5	8		
7	0.	-2.760E+02	6			
1	9.600E+02	-4.824E+01	6			

COMPUTED JOINT COORDINATES

JOINT	X	Y
1	0.	0.
2	0.	2.278E+02
3	0.	4.555E+02
4	4.800E+02	2.278E+02
5	4.800E+02	4.866E+02
6	9.600E+02	-4.824E+01
7	9.600E+02	2.278E+02
8	9.600E+02	5.218E+02

PROB (CONTD)
1201 TWO STORY BEAM WITHOUT INTERIOR COLUMN - LIVE LOAD

TABLE 3 - MEMBER LOCATION DATA

NUMBER OF MEMBER STIFFNESS TYPES = 6
NUMBER OF MEMBER LOAD TYPES = 2

INPUT OF MEMBER LOCATIONS

FROM JOINT	STIFF TYPE	LOAD TYPE	TO JOINT	TO	TO	TO	TO	TO	TO	TO	TO
1	1	0	2	3							
4	2	0	5								
6	3	0	7								
7	4	0	8								
3	5	2	8	0							
2	6	1	4	7							

COMPUTED MEMBER NUMBERS, LENGTHS, AND OFFSETS

MEMBER NUMB	FROM JOINT	TO JOINT	STIFF TYPE	LOAD TYPE	LENGTH	X-OFFSET	Y-OFFSET
1	1	2	1	0	2.278E+02	0.	2.278E+02
2	2	3	1	0	2.278E+02	0.	2.278E+02
3	4	5	2	0	2.409E+02	0.	2.409E+02
4	6	7	3	0	2.740E+02	0.	2.740E+02
5	7	8	4	0	2.940E+02	0.	2.940E+02
6	2	4	5	1	4.800E+02	4.800E+02	0.
7	4	7	5	1	4.800E+02	4.800E+02	0.
8	3	5	6	2	4.811E+02	4.800E+02	3.312E+01
9	5	8	6	2	4.811E+02	4.800E+02	3.312E+01

*** COMPUTED MEMBER NUMBERS MAY NOT AGREE WITH LAST PROBLEM ***

PROB (CONTD)
1201 TWO STORY BEAM WITHOUT INTERIOR COLUMN - LIVE LOAD

TABLE 4 - JOINT DATA

INPUT OF JOINT DATA

JOINT	FORCE (X)	FORCE (Y)	MOMENT (Z)	SPRING (X)	SPRING (Y)	SPRING (Z)
1	-0.	-0.	-0.	1.000E+00	1.000E+99	-0.
6	-0.	-0.	-0.	1.000E+00	1.000E+99	1.000E+99

ACCUMULATED JOINT DATA

SAME AS INPUT FOR THIS PROBLEM

PROB (CONTD)
1201 TWO STORY BEAM WITHOUT INTERIOR COLUMN - LIVE LOAD

TABLE 5 - MEMBER STIFFNESS DATA

MEMBER TYPE	MOD OF ELAST	PRISMATIC I	PRISMATIC A	NO CARUS	AXIS OPT	OUTPUT OPT	PIN FROM	PIN TO
1	2.900E+04	1.050E+03	2.800E+01	-0	1	1	-0	-0
2	2.900E+04	0.000E+02	2.000E+01	-0	1	1	-0	1
3	2.900E+04	1.050E+03	2.800E+01	-0	1	1	-0	-0
4	2.900E+04	1.050E+03	2.800E+01	-0	1	1	-0	-0
5	2.900E+04	1.000E+03	2.400E+01	-0	1	-0	-0	-0
6	2.900E+04	-0.	-0.	7	2	-0	-0	-0

STIFF TYPE 6 CONID RESTRAINTS ARE IN MEMBER PRIMED AXES

FROM	TO	I	A	RX	SY	SZ
0.	-0.	4.000E+03	7.000E+01	-0.	-0.	-0.
-0.	6.300E+01	3.400E+03	4.200E+01	-0.	-0.	-0.
6.300E+01	1.000E+02	3.400E+03	4.200E+01	-0.	-0.	-0.
1.000E+02	2.940E+02	5.000E+03	5.000E+01	-0.	-0.	-0.
2.940E+02	4.164E+02	3.400E+03	4.200E+01	-0.	-0.	-0.
4.164E+02	-0.	3.400E+03	4.200E+01	-0.	-0.	-0.
-0.	4.800E+02	4.000E+03	7.000E+01	-0.	-0.	-0.

PROB (CONTD)
1201 TWO STORY BEAM WITHOUT INTERIOR COLUMN - LIVE LOAD

TABLE 6 - MEMBER LOAD DATA

LOAD TYPE	UNIFORM QX	UNIFORM QY	NO CARDS	AXIS OPT
1	-0.	-0.	4	3

1 -0. -0. 4 3

LOAD TYPE FROM	UNIFORM QX	UNIFORM QY	NO CARDS	AXIS OPT
1	9.600E+01	9.600E+01	-0.	-1.000E+01 -0.
2	1.920E+02	1.920E+02	-0.	-1.000E+01 -0.
3	2.880E+02	2.880E+02	-0.	-1.000E+01 -0.
4	3.840E+02	3.840E+02	-0.	-1.000E+01 -0.

LOAD TYPE	UNIFORM QX	UNIFORM QY	NO CARDS	AXIS OPT
2	-0.	-0.	4	3

2 -0. -0. 4 3

LOAD TYPE FROM	UNIFORM QX	UNIFORM QY	NO CARDS	AXIS OPT
1	9.600E+01	9.600E+01	-0.	-2.000E+01 -0.
2	1.920E+02	1.920E+02	-0.	-2.000E+01 -0.
3	2.880E+02	2.880E+02	-0.	-2.000E+01 -0.
4	3.840E+02	3.840E+02	-0.	-2.000E+01 -0.

PROB (CONTD)
1201 TWO STORY BEAM WITHOUT INTERIOR COLUMN - LIVE LOAD

TABLE 7 - COMPILATION TABLE

NO DATA

PROB (CONTD)
1201 TWO STORY BEAM WITHOUT INTERIOR COLUMN - LIVE LOAD

TABLE 8 - JOINT DISPLACEMENTS AND REACTIONS

JOINT	DISPLACEMENTS			REACTIONS		
	DISP(X)	DISP(Y)	ROTATION(Z)	REACT(X)	REACT(Y)	REACT(Z)
1	-1.840E-08	-1.205E-07	3.181E-03	1.000E+01	1.205E+02	0.
2	-3.377E-02	-3.312E-02	-5.918E-03	0.	0.	0.
3	7.278E-01	-5.390E-02	-1.590E-02	0.	0.	0.
4	-6.341E-03	-5.920E+00	-7.025E-04	0.	0.	0.
5	1.114E+00	-5.934E+00	3.291E-04	0.	0.	0.
6	1.040E-08	-1.195E-07	-1.021E-06	-1.000E+01	1.195E+02	1.021E+03
7	2.246E-02	-3.979E-02	4.291E-03	0.	0.	0.
8	6.917E-01	-6.477E-02	1.463E-02	0.	0.	0.

PROB (CONTD)
1201 TWO STORY HEAT WITHOUT INTERIOR COLUMN - LIVE LOAD

TABLE 4 - MEMBER RESULTS

MEMBER NUMBER 1 STIFF TYPE 1 LOAD TYPE A
LENGTH = 2.278E+02 ALPHA = 0. BETA = 1.000E+00
GOES FROM JOINT 1 TO JOINT 2
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 1		AT JOINT 2	
AXIAL FORCE	= -1.205E+02	AXIAL FORCE	= -1.205E+02
SHEAR	= -1.091E+01	SHEAR	= -1.091E+01
MOMENT	= 1.940E+01	MOMENT	= -2.484E+03

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 4.787E-08

MEMBER NUMBER 2 STIFF TYPE 1 LOAD TYPE A
LENGTH = 2.278E+02 ALPHA = 0. BETA = 1.000E+00
GOES FROM JOINT 2 TO JOINT 3
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 2		AT JOINT 3	
AXIAL FORCE	= -7.562E+01	AXIAL FORCE	= -7.562E+01
SHEAR	= -5.440E+01	SHEAR	= -5.440E+01
MOMENT	= 4.833E+03	MOMENT	= -7.557E+03

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 0.820E-08

MEMBER NUMBER 3 STIFF TYPE 2 LOAD TYPE A
LENGTH = 2.609E+02 ALPHA = 0. BETA = 1.000E+00
GOES FROM JOINT 4 TO JOINT 5
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 4		AT JOINT 5	
AXIAL FORCE	= -1.395E+01	AXIAL FORCE	= -1.395E+01
SHEAR	= 3.740E+00	SHEAR	= 3.740E+00
MOMENT	= -9.757E+02	MOMENT	= 3.950E+00

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 0.192E-08

PROB (CONTD)
1201 TWO STORY HEAT WITHOUT INTERIOR COLUMN - LIVE LOAD

TABLE 4 - MEMBER RESULTS (CONTD)

MEMBER NUMBER 4 STIFF TYPE 3 LOAD TYPE A
LENGTH = 2.760E+02 ALPHA = 0. BETA = 1.000E+00
GOES FROM JOINT 6 TO JOINT 7
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 6		AT JOINT 7	
AXIAL FORCE	= -1.195E+02	AXIAL FORCE	= -1.195E+02
SHEAR	= 1.091E+01	SHEAR	= 1.091E+01
MOMENT	= -1.022E+03	MOMENT	= 1.940E+03

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 1.969E-08

MEMBER NUMBER 5 STIFF TYPE 4 LOAD TYPE A
LENGTH = 2.966E+02 ALPHA = 0. BETA = 1.000E+00
GOES FROM JOINT 7 TO JOINT 8
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 7		AT JOINT 8	
AXIAL FORCE	= -7.042E+01	AXIAL FORCE	= -7.042E+01
SHEAR	= 5.066E+01	SHEAR	= 5.066E+01
MOMENT	= -6.353E+03	MOMENT	= 4.540E+03

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 1.556E-07

PROB (CONTD)
1201 TWO STORY BEAT WITHOUT INTERIOR COLUMN - LIVE LOAD

PROB (CONTD)
1201 TWO STORY BEAT WITHOUT INTERIOR COLUMN - LIVE LOAD

TABLE 6 - MEMBER RESULTS (CONTD)

TABLE 9 - MEMBER RESULTS (CONTD)

MEMBER NUMBER 6 STIFF TYPE 5 LOAD TYPE I
LENGTH = 4.800E+02 ALPHA = 1.000E+00 BETA = 0.
BOES FROM JOINT 2 TO JOINT 4
OUTPUT DISTANCES ARE FROM JOINT 2 ALONG THE MEMBER AXIS
ALL OUTPUT FORCES AND DISPLACEMENTS ARE WITH RESPECT TO THE MEMBER AXES

MEMBER NUMBER 7 STIFF TYPE 5 LOAD TYPE I
LENGTH = 4.800E+02 ALPHA = 1.000E+00 BETA = 0.
BOES FROM JOINT 4 TO JOINT 7
OUTPUT DISTANCES ARE FROM JOINT 4 ALONG THE MEMBER AXIS
ALL OUTPUT FORCES AND DISPLACEMENTS ARE WITH RESPECT TO THE MEMBER AXES

DISTANCE	DISPLACEMENTS			FORCES		
	AXIAL	LATERAL	ROTATIONAL	AXIAL	SHEAR	MOMENT
0.	-3.377E-02	-3.312E-02	-5.910E-03	4.349E+01	4.490E+01	-7.316E+03
2.400E+01	-3.236E-02	-2.127E-01	-8.971E-03	4.349E+01	4.490E+01	-8.239E+03
4.800E+01	-3.084E-02	-4.597E-01	-1.154E-02	4.349E+01	4.490E+01	-5.161E+03
7.200E+01	-2.937E-02	-7.625E-01	-1.302E-02	4.349E+01	4.490E+01	-4.084E+03
9.600E+01	-2.790E-02	-1.109E+00	-1.522E-02	4.349E+01	3.990E+01	-3.006E+03
1.200E+02	-2.643E-02	-1.489E+00	-1.638E-02	4.349E+01	3.490E+01	-2.189E+03
1.440E+02	-2.496E-02	-1.893E+00	-1.717E-02	4.349E+01	3.490E+01	-1.331E+03
1.680E+02	-2.349E-02	-2.311E+00	-1.758E-02	4.349E+01	3.490E+01	-4.936E+02
1.920E+02	-2.202E-02	-2.734E+00	-1.762E-02	4.349E+01	2.990E+01	-3.439E+02
2.160E+02	-2.055E-02	-3.153E+00	-1.733E-02	4.349E+01	2.490E+01	9.414E+02
2.400E+02	-1.908E-02	-3.563E+00	-1.677E-02	4.349E+01	2.490E+01	1.539E+03
2.640E+02	-1.761E-02	-3.956E+00	-1.594E-02	4.349E+01	2.490E+01	2.136E+03
2.880E+02	-1.614E-02	-4.326E+00	-1.484E-02	4.349E+01	1.990E+01	2.734E+03
3.120E+02	-1.468E-02	-4.687E+00	-1.353E-02	4.349E+01	1.490E+01	3.091E+03
3.360E+02	-1.321E-02	-4.974E+00	-1.206E-02	4.349E+01	1.490E+01	3.449E+03
3.600E+02	-1.174E-02	-5.244E+00	-1.042E-02	4.349E+01	1.490E+01	3.806E+03
3.840E+02	-1.027E-02	-5.473E+00	-8.629E-03	4.349E+01	0.896E+00	4.164E+03
4.080E+02	-8.799E-03	-5.657E+00	-6.726E-03	4.349E+01	4.896E+00	4.281E+03
4.320E+02	-7.329E-03	-5.796E+00	-4.771E-03	4.349E+01	4.896E+00	4.399E+03
4.560E+02	-5.860E-03	-5.886E+00	-2.763E-03	4.349E+01	4.896E+00	4.516E+03
4.800E+02	-4.391E-03	-5.928E+00	-7.025E-04	4.349E+01	4.896E+00	4.634E+03

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 3.200E-07

DISTANCE	DISPLACEMENTS			FORCES		
	AXIAL	LATERAL	ROTATIONAL	AXIAL	SHEAR	MOMENT
0.	-4.391E-03	-5.928E+00	-7.025E-04	3.975E+01	-8.065E+00	5.610E+03
2.400E+01	-3.048E-03	-5.915E+00	1.775E-03	3.975E+01	-8.065E+00	5.392E+03
4.800E+01	-1.706E-03	-5.843E+00	4.155E-03	3.975E+01	-8.065E+00	5.175E+03
7.200E+01	-3.629E-04	-5.716E+00	8.437E-03	3.975E+01	-8.065E+00	4.957E+03
9.600E+01	9.799E-04	-5.535E+00	8.621E-03	3.975E+01	-1.406E+01	4.740E+03
1.200E+02	2.323E-03	-5.303E+00	1.065E-02	3.975E+01	-1.906E+01	4.202E+03
1.440E+02	3.665E-03	-5.025E+00	1.248E-02	3.975E+01	-1.906E+01	3.825E+03
1.680E+02	5.008E-03	-4.706E+00	1.418E-02	3.975E+01	-1.906E+01	3.367E+03
1.920E+02	6.351E-03	-4.356E+00	1.551E-02	3.975E+01	-2.406E+01	2.909E+03
2.160E+02	7.694E-03	-3.964E+00	1.667E-02	3.975E+01	-2.906E+01	2.212E+03
2.400E+02	9.037E-03	-3.553E+00	1.751E-02	3.975E+01	-2.906E+01	1.514E+03
2.640E+02	1.038E-02	-3.126E+00	1.803E-02	3.975E+01	-2.906E+01	8.167E+02
2.880E+02	1.172E-02	-2.690E+00	1.824E-02	3.975E+01	-3.406E+01	1.192E+02
3.120E+02	1.306E-02	-2.254E+00	1.808E-02	3.975E+01	-3.906E+01	-8.184E+02
3.360E+02	1.441E-02	-1.826E+00	1.759E-02	3.975E+01	-3.906E+01	-1.756E+03
3.600E+02	1.575E-02	-1.417E+00	1.650E-02	3.975E+01	-3.906E+01	-2.693E+03
3.840E+02	1.709E-02	-1.037E+00	1.508E-02	3.975E+01	-4.406E+01	-3.631E+03
4.080E+02	1.844E-02	-6.971E-01	1.318E-02	3.975E+01	-4.906E+01	-4.809E+03
4.320E+02	1.978E-02	-4.090E-01	1.075E-02	3.975E+01	-4.906E+01	-5.986E+03
4.560E+02	2.112E-02	-1.857E-01	7.783E-03	3.975E+01	-4.906E+01	-7.164E+03
4.800E+02	2.246E-02	-3.979E-02	4.291E-03	3.975E+01	-4.906E+01	-8.341E+03

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 3.845E-07

PROB (CONTD)
1201 TWO STORY BEAT WITHOUT INTERIOR COLUMN - LIVE LOAD

TABLE 9 - MEMBER RESULTS (CONTD)

MEMBER NUMBER 8 STIFF TYPE 6 LOAD TYPE 2
LENGTH = 4.811E+02 ALPHA = 9.976E-01 BETA = 6.884E-02
GOES FROM JOINT 3 TO JOINT 8
OUTOUT DISTANCES ARE FROM JOINT 3 ALONG THE STRUCTURE X-AXIS
ALL OUTPUT FORCES AND DISPLACEMENTS ARE WITH RESPECT TO THE MEMBER AXES

DISTANCE	DISPLACEMENTS			FORCES		
	AXIAL	LATERAL	ROTATIONAL	AXIAL	SHEAR	MOMENT
0.	7.223E-01	-1.039E-01	-1.590E-02	-5.947E-01	7.170E+01	-7.556E+03
2.400E+01	7.216E-01	-4.947E-01	-1.658E-02	-5.947E-01	7.170E+01	-5.831E+03
4.800E+01	7.207E-01	-9.620E-01	-1.728E-02	-5.947E-01	7.170E+01	-4.104E+03
7.200E+01	7.196E-01	-1.326E+00	-1.796E-02	-5.947E-01	7.170E+01	-2.381E+03
9.600E+01	7.184E-01	-1.763E+00	-1.832E-02	-5.878E-01	6.172E+01	-6.564E+02
1.200E+02	7.173E-01	-2.205E+00	-1.833E-02	-5.809E-01	5.175E+01	5.885E+02
1.440E+02	7.162E-01	-2.643E+00	-1.804E-02	-5.809E-01	5.175E+01	1.893E+03
1.680E+02	7.151E-01	-3.078E+00	-1.746E-02	-5.809E-01	5.175E+01	3.078E+03
1.920E+02	7.140E-01	-3.461E+00	-1.665E-02	-5.740E-01	4.177E+01	4.323E+03
2.160E+02	7.131E-01	-3.872E+00	-1.588E-02	-5.671E-01	3.180E+01	5.088E+03
2.400E+02	7.121E-01	-4.244E+00	-1.499E-02	-5.671E-01	3.180E+01	5.853E+03
2.640E+02	7.112E-01	-4.593E+00	-1.398E-02	-5.671E-01	3.180E+01	6.618E+03
2.880E+02	7.103E-01	-4.915E+00	-1.284E-02	-5.602E-01	2.182E+01	7.383E+03
3.120E+02	7.093E-01	-5.205E+00	-1.119E-02	-5.534E-01	1.184E+01	7.668E+03
3.360E+02	7.082E-01	-5.452E+00	-9.318E-03	-5.534E-01	1.184E+01	7.953E+03
3.600E+02	7.071E-01	-5.653E+00	-7.383E-03	-5.534E-01	1.184E+01	8.238E+03
3.840E+02	7.061E-01	-5.807E+00	-5.380E-03	-5.445E-01	1.866E+00	8.522E+03
4.080E+02	7.050E-01	-5.912E+00	-3.366E-03	-5.396E-01	-8.110E+00	8.327E+03
4.320E+02	7.040E-01	-5.971E+00	-1.607E-03	-5.396E-01	-8.110E+00	8.132E+03
4.560E+02	7.032E-01	-5.995E+00	-4.769E-04	-5.396E-01	-8.110E+00	7.937E+03
4.800E+02	7.025E-01	-5.996E+00	3.291E-04	-5.396E-01	-8.110E+00	7.742E+03

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 1.283E-06

PROB (CONTD)
1201 TWO STORY BEAT WITHOUT INTERIOR COLUMN - LIVE LOAD

TABLE 9 - MEMBER RESULTS (CONTD)

MEMBER NUMBER 9 STIFF TYPE 6 LOAD TYPE 2
LENGTH = 4.811E+02 ALPHA = 9.976E-01 BETA = 6.884E-02
GOES FROM JOINT 5 TO JOINT 8
OUTOUT DISTANCES ARE FROM JOINT 5 ALONG THE STRUCTURE X-AXIS
ALL OUTPUT FORCES AND DISPLACEMENTS ARE WITH RESPECT TO THE MEMBER AXES

DISTANCE	DISPLACEMENTS			FORCES		
	AXIAL	LATERAL	ROTATIONAL	AXIAL	SHEAR	MOMENT
0.	7.025E-01	-5.996E+00	3.291E-04	-5.119E+01	6.067E+00	7.742E+03
2.400E+01	7.019E-01	-5.979E+00	1.133E-03	-5.119E+01	6.067E+00	7.888E+03
4.800E+01	7.011E-01	-5.939E+00	2.252E-03	-5.119E+01	6.067E+00	8.034E+03
7.200E+01	7.002E-01	-5.865E+00	3.985E-03	-5.119E+01	6.067E+00	8.180E+03
9.600E+01	6.992E-01	-5.746E+00	5.958E-03	-5.050E+01	-3.909E+00	8.326E+03
1.200E+02	6.982E-01	-5.579E+00	7.908E-03	-4.981E+01	-1.389E+01	7.992E+03
1.440E+02	6.972E-01	-5.366E+00	9.778E-03	-4.981E+01	-1.389E+01	7.658E+03
1.680E+02	6.963E-01	-5.109E+00	1.157E-02	-4.981E+01	-1.389E+01	7.324E+03
1.920E+02	6.954E-01	-4.811E+00	1.314E-02	-4.912E+01	-2.386E+01	6.990E+03
2.160E+02	6.946E-01	-4.481E+00	1.421E-02	-4.844E+01	-3.384E+01	6.176E+03
2.400E+02	6.938E-01	-4.128E+00	1.515E-02	-4.844E+01	-3.384E+01	5.362E+03
2.640E+02	6.930E-01	-3.754E+00	1.598E-02	-4.844E+01	-3.384E+01	4.548E+03
2.880E+02	6.922E-01	-3.381E+00	1.663E-02	-4.775E+01	-4.381E+01	3.734E+03
3.120E+02	6.913E-01	-2.953E+00	1.730E-02	-4.746E+01	-5.379E+01	2.440E+03
3.360E+02	6.904E-01	-2.531E+00	1.773E-02	-4.746E+01	-5.379E+01	1.145E+03
3.600E+02	6.895E-01	-2.102E+00	1.789E-02	-4.746E+01	-5.379E+01	-1.486E+02
3.840E+02	6.886E-01	-1.675E+00	1.766E-02	-4.677E+01	-6.377E+01	-1.443E+03
4.080E+02	6.877E-01	-1.258E+00	1.710E-02	-4.548E+01	-7.374E+01	-3.217E+03
4.320E+02	6.869E-01	-8.545E-01	1.624E-02	-4.448E+01	-7.374E+01	-4.991E+03
4.560E+02	6.862E-01	-4.737E-01	1.542E-02	-4.548E+01	-7.374E+01	-6.765E+03
4.800E+02	6.856E-01	-1.122E-01	1.463E-02	-4.548E+01	-7.374E+01	-8.539E+03

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 1.175E-06

PROB (CONTD)
1201 TWO STORY BEAM WITHOUT INTERIOR COLUMN - LIVE LOAD

TABLE 10 - JOINT EQUILIBRIUM ERRORS

JOINT	ERR(X) FORCE	ERR(Y) FORCE	ERR(Z) MOMENT
1	-1.704E-03	1.801E-10	1.940E-01
2	-6.796E-03	-3.890E-03	2.284E-01
3	8.590E-03	1.546E-09	9.680E-01
4	1.461E-04	8.432E-03	1.182E-01
5	-1.461E-04	1.691E-07	-1.221E-06
6	1.704E-03	1.764E-10	-2.351E-01
7	6.211E-03	-4.541E-03	-3.088E-01
8	-7.915E-03	1.055E-08	-1.164E-00

02 JULY 70 - CON - MM
 EXAMPLE PROBLEMS FOR REPORT

PROB 1202 TWO STORY BENT - ADD INTERIOR COLUMN - LIVE LOAD

PROB (CONTD)
 1202 TWO STORY BENT - ADD INTERIOR COLUMN - LIVE LOAD

TABLE 2 - FRAME GEOMETRY DATA

HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS THE FOLLOWING

NUMBER OF JOINTS IN FRAME = 9

TABLE 1 - PROGRAM CONTROL DATA
 PROBLEM TYPE 2

INPUT TABLES		
TABLE NUMBER	HOLD DATA FROM LAST PROBLEM (1 = YES, 0 = NO)	NUMBER OF CARDS ADDED FOR THIS PROBLEM
2	1	2
3	1	2
4	1	1
5	1	4
6	1	0
7	-0	-0

OUTPUT TABLES	
TABLE NUMBER	SUPPRESS OUTPUT (1 = YES, 0 = NO)
8	-0
9	-0
10	-0

INPUT OF JOINT OFFSETS

FROM JOINT	X-OFFSET	Y-OFFSET	TO JOINT	TO	TO	TO	TO	TO	TO
4	0.	-3.950E+02	9						

COMPUTED JOINT COORDINATES

JOINT	X	Y
1	0.	0.
2	0.	2.278E+02
3	0.	4.555E+02
4	4.800E+02	2.278E+02
5	4.800E+02	4.888E+02
6	9.600E+02	-4.824E+01
7	9.600E+02	2.278E+02
8	9.600E+02	5.218E+02
9	4.800E+02	-1.682E+02

PROB (CONTD)
1202 TWO STORY BENT - ADD INTERIOR COLUMN - LIVE LOAD

TABLE 3 - MEMBER LOCATION DATA

HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS THE FOLLOWING

NUMBER OF MEMBER STIFFNESS TYPES = 7
NUMBER OF MEMBER LOAD TYPES = 2

INPUT OF MEMBER LOCATIONS

FROM JOINT	STIFF LOAD TYPE	TO JOINT	TO	TO	TO	TO	TO	TO	TO	TO
9	7	8	4							

COMPUTED MEMBER NUMBERS, LENGTHS, AND OFFSETS

MEMBER NUMB	FROM JOINT	TO JOINT	STIFF LOAD TYPE	LENGTH	X-OFFSET	Y-OFFSET
1	1	2	1	8	2.278E+02	0.
2	2	3	1	8	2.278E+02	0.
3	4	5	2	8	2.609E+02	0.
4	6	7	3	8	2.740E+02	0.
5	7	8	4	8	2.940E+02	0.
6	2	4	5	1	4.800E+02	4.000E+02
7	4	7	5	1	4.800E+02	4.000E+02
8	3	5	6	2	4.811E+02	4.000E+02
9	5	8	6	2	4.811E+02	4.000E+02
10	9	4	7	0	3.940E+02	0.

*** COMPUTED MEMBER NUMBERS AGREE WITH LAST PROBLEM ***

PROB (CONTD)
1202 TWO STORY BENT - ADD INTERIOR COLUMN - LIVE LOAD

TABLE 4 - JOINT DATA

HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS THE FOLLOWING

INPUT OF JOINT DATA

JOINT	FORCE (X)	FORCE (Y)	MOMENT (Z)	SPRING (X)	SPRING (Y)	SPRING (Z)
9	-0.	-0.	-0.	-0.	2.000E+01	-0.

ACCUMULATED JOINT DATA

JOINT	FORCE (X)	FORCE (Y)	MOMENT (Z)	SPRING (X)	SPRING (Y)	SPRING (Z)
1	0.	0.	0.	1.000E+99	1.000E+99	0.
6	0.	0.	0.	1.000E+99	1.000E+99	1.000E+99
9	0.	0.	0.	0.	2.000E+01	0.

PROB (CONTD)
1202 TWO STORY BENT - ADD INTERIOR COLUMN - LIVE LOAD

TABLE 5 - MEMBER STIFFNESS DATA

HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS THE FOLLOWING

STIFF TYPE	MOD OF ELAST	PRISMATIC I	PRISMATIC A	NO CARDS	AXIS OPT	OUTPUT NOT	FIN FROM	PIN TO
7	2.960E+04-0.	-0.		3	1	-0	-0	-0

STIFF TYPE 7 CONTD RESTRAINTS ARE IN MEMBER PRIMED AREAS

FROM	TO	I	A	XX	YY
9					

0.	-0.	1.050E+03	2.400E+01	2.000E+00	3.000E+00	-0.
-0.	1.920E+02	1.050E+03	2.400E+01	2.000E+00	1.500E+00	-0.
1.920E+02	3.960E+02	1.050E+03	2.900E+01	-0.	-0.	-0.

PROB (CONTO)
1202 TWO STORY BENT - ADD INTERIOR COLUMN - LIVE LOAD

TABLE 6 - MEMBER LOAD DATA

HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS THE FOLLOWING

NONE

PROB (CONTO)
1202 TWO STORY BENT - ADD INTERIOR COLUMN - LIVE LOAD

TABLE 7 - COMPILATION TABLE

NO DATA

PROB (CONTO)
1202 TWO STORY BENT - ADD INTERIOR COLUMN - LIVE LOAD

TABLE 8 - JOINT DISPLACEMENTS AND REACTIONS

JOINT	DISPLACEMENTS			REACTIONS		
	DISP(X)	DISP(Y)	ROTATION(Z)	REACT(X)	REACT(Y)	REACT(Z)
1	-2.706E-09	-5.439E-08	7.556E-04	2.706E+00	5.439E+01	0.
2	-6.516E-04	-1.495E-02	-1.503E-03	0.	0.	0.
3	1.803E-01	-2.444E-02	-3.445E-03	0.	0.	0.
4	5.677E-03	-3.685E-01	-7.806E-05	0.	0.	0.
5	2.028E-01	-4.090E-01	8.050E-05	0.	0.	0.
6	2.785E-09	-5.411E-08	-2.655E-07	-2.785E+00	5.411E+01	2.655E+02
7	1.147E-02	-1.802E-02	1.054E-03	0.	0.	0.
8	1.725E-01	-2.988E-02	3.118E-03	0.	0.	0.
9	1.167E-04	-3.202E-01	3.337E-06	0.	6.405E+00	0.

PROB (CONTO)
1202 TWO STORY BENT - ADD INTERIOR COLUMN - LIVE LOAD

TABLE 9 - MEMBER RESULTS

MEMBER NUMBER 1 STIFF TYPE 1 LOAD TYPE 0
LENGTH = 2.278E+02 ALPHA = 0. BETA = 1.000E+00
GOES FROM JOINT 1 TO JOINT 2
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 1		AT JOINT 2	
AXIAL FORCE	= -5.439E+01	AXIAL FORCE	= -5.439E+01
SHEAR	= -2.706E+00	SHEAR	= -2.706E+00
MOMENT	= 4.616E-02	MOMENT	= -4.164E-02

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 1.191E-08

MEMBER NUMBER 2 STIFF TYPE 1 LOAD TYPE 0
LENGTH = 2.278E+02 ALPHA = 0. BETA = 1.000E+00
GOES FROM JOINT 2 TO JOINT 3
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 2		AT JOINT 3		SHEAR	MOMENT
AXIAL FORCE	SHEAR	AXIAL FORCE	SHEAR		
AXIAL FORCE	= -3.454E+01	AXIAL FORCE	= -1.454E+01	4.567E+01	-4.662E+03
SHEAR	= -1.207E+01	SHEAR	= -1.454E+01	4.567E+01	-3.563E+03
MOMENT	= 1.110E-01	MOMENT	= -1.454E+01	4.567E+01	-2.464E+03
THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 1.234E-07					

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 1.234E-07

PROB (CONTD)
1202 TWO STORY BENT - ADD INTERIOR COLUMN - LIVE LOAD

02 JULY 70 - COM - MM
EXAMPLE PROBLEMS FOR REPORT

TABLE 9 - MEMBER RESULTS (CONTD)

MEMBER NUMBER 10 STIFF TYPE 7 LOAD TYPE 0
LENGTH = 3.960E+02 ALPHA = 0.0 BETA = 1.000E+00
80% FROM JOINT 9 TO JOINT 4
OUTPUT DISTANCES ARE FROM JOINT 9 ALONG THE MEMBER AXIS
ALL OUTPUT FORCES AND DISPLACEMENTS ARE WITH RESPECT TO THE MEMBER AXES

PROB 1203 TWO STORY BENT HOLDING STIFFNESS FROM 1202 - WIND LOAD

TABLE 1 - PROGRAM CONTROL DATA
PROBLEM TYPE 3

DISTANCE	DISPLACEMENTS			FORCES		
	AXIAL	LATERAL	ROTATIONAL	AXIAL	SHEAR	MOMENT
0.	-3.202E-01	-1.167E-04	3.337E-06	-6.405E+00	4.475E-14	-1.532E-12
1.980E+01	-3.206E-01	-5.055E-05	3.351E-06	-1.949E+01	4.851E-03	5.798E-02
3.960E+01	-3.212E-01	1.642E-05	3.424E-06	-3.190E+01	5.817E-03	1.728E-01
5.940E+01	-3.221E-01	8.544E-05	3.567E-06	-4.453E+01	3.215E-03	2.707E-01
7.920E+01	-3.233E-01	1.580E-04	3.749E-06	-5.731E+01	-2.673E-03	2.838E-01
9.900E+01	-3.248E-01	2.338E-04	3.894E-06	-7.014E+01	-1.157E-02	1.499E-01
1.188E+02	-3.266E-01	3.112E-04	3.891E-06	-8.304E+01	-2.313E-02	-1.876E-01
1.386E+02	-3.288E-01	3.858E-04	3.595E-06	-9.602E+01	-3.688E-02	-7.771E-01
1.584E+02	-3.312E-01	4.503E-04	2.832E-06	-1.091E+02	-5.212E-02	-1.656E+00
1.782E+02	-3.340E-01	4.935E-04	1.411E-06	-1.223E+02	-4.791E-02	-2.844E+00
1.980E+02	-3.371E-01	5.804E-04	-8.638E-07	-1.312E+02	-7.839E-02	-4.328E+00
2.178E+02	-3.402E-01	4.526E-04	-4.117E-06	-1.315E+02	-7.868E-02	-5.886E+00
2.376E+02	-3.433E-01	3.306E-04	-8.364E-06	-1.315E+02	-7.868E-02	-7.444E+00
2.574E+02	-3.465E-01	1.147E-04	-1.360E-05	-1.315E+02	-7.868E-02	-9.002E+00
2.772E+02	-3.496E-01	-2.148E-04	-1.903E-05	-1.315E+02	-7.868E-02	-1.056E+01
2.970E+02	-3.528E-01	-6.775E-04	-2.706E-05	-1.315E+02	-7.868E-02	-1.212E+01
3.168E+02	-3.559E-01	-1.243E-03	-3.527E-05	-1.315E+02	-7.868E-02	-1.368E+01
3.366E+02	-3.590E-01	-2.081E-03	-4.448E-05	-1.315E+02	-7.868E-02	-1.523E+01
3.564E+02	-3.622E-01	-3.061E-03	-5.468E-05	-1.315E+02	-7.868E-02	-1.679E+01
3.762E+02	-3.653E-01	-4.253E-03	-6.587E-05	-1.315E+02	-7.868E-02	-1.835E+01
3.960E+02	-3.685E-01	-5.677E-03	-7.806E-05	-1.315E+02	-7.868E-02	-1.991E+01

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 2.983E-10

PROB (CONTD)
1202 TWO STORY BENT - ADD INTERIOR COLUMN - LIVE LOAD

TABLE 10 - JOINT EQUILIBRIUM ERRORS

JOINT	ERR(X) FORCE	ERR(Y) FORCE	ERR(Z) MOMENT
1	-4.220E-04	7.481E-11	4.816E-02
2	-1.464E-03	2.334E-05	2.686E-01
3	1.886E-03	1.920E-09	2.148E-01
4	2.762E-05	8.160E-05	2.358E-02
5	-2.762E-05	1.482E-08	-9.907E-08
6	4.352E-04	8.872E-11	-6.005E-02
7	1.341E-03	-1.049E-04	-2.959E-01
8	-1.776E-03	1.877E-09	-2.611E-01
9	4.475E-14	2.729E-10	-1.532E-12

INPUT TABLES

TABLE NUMBER	HOLD DATA FROM LAST PROBLEM (1 = YES 0 = NO)	NUMBER OF CARDS ADDED FOR THIS PROBLEM
2	1	0
3	1	6
4	1	0
5	1	0
6	0	4
7	0	0

OUTPUT TABLES

TABLE NUMBER	SUPPRESS OUTPUT (1 = YES 0 = NO)
8	0
9	0
10	0

PROB (CONTD)
1203 TWO STORY BEAT HOLDING STIFFNESS FROM 1202 - WIND LOAD

TABLE 2 - FRAME GEOMETRY DATA

HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS THE FOLLOWING

NONE

COMPUTED JOINT COORDINATES

JOINT	X	Y
1	0.	0.
2	0.	2.278E+02
3	0.	4.556E+02
4	4.800E+02	2.278E+02
5	4.800E+02	4.556E+02
6	9.600E+02	-4.829E+01
7	9.600E+02	2.278E+02
8	9.600E+02	5.218E+02
9	4.800E+02	-1.682E+02

PROB (CONTD)
1203 TWO STORY BEAT HOLDING STIFFNESS FROM 1202 - WIND LOAD

TABLE 4 - MEMBER LOCATION DATA

HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS THE FOLLOWING

NUMBER OF MEMBER STIFFNESS TYPES = 7
NUMBER OF MEMBER LOAD TYPES = 4

INPUT OF MEMBER LOCATIONS

FROM JOINT	STIFF TYPE	LOAD TYPE	TO JOINT	T ₀	T ₁	T ₂	T ₃	T ₄	T ₅	T ₆	T ₇	T ₈	T ₉
1	1	1	2										
6	3	2	7										
7	4	3	8										
2	5	0	4										
3	6	4	5										

COMPUTED MEMBER NUMBERS, LENGTHS, AND OFFSETS

MEMBER NUMB	FROM JOINT	TO JOINT	STIFF TYPE	LOAD TYPE	LENGTH	X-OFFSET	Y-OFFSET
1	1	2	1	1	2.278E+02	0.	2.278E+02
2	2	3	1	1	2.278E+02	0.	2.278E+02
3	4	5	2	0	2.409E+02	0.	2.609E+02
4	6	7	3	2	2.740E+02	0.	2.740E+02
5	7	8	4	3	2.940E+02	0.	2.940E+02
6	2	4	5	0	4.400E+02	4.800E+02	0.
7	4	7	5	0	4.400E+02	4.800E+02	0.
8	3	5	6	4	4.811E+02	4.800E+02	3.312E+01
9	5	8	6	4	4.811E+02	4.800E+02	3.312E+01
10	9	4	7	0	3.460E+02	0.	3.940E+02

*** COMPUTED MEMBER NUMBERS AGREE WITH LAST PROBLEM ***

PROB (CONTD)
1203 TWO STORY BENT HOLDING STIFFNESS FROM 1202 - WIND LOAD

TABLE 4 - JOINT DATA

HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS THE FOLLOWING

NONE

ACCUMULATED JOINT DATA

JOINT	FORCE (X)	FORCE (Y)	MOMENT (Z)	SPRING (X)	SPRING (Y)	SPRING (Z)
1	0.	0.	0.	1.000E+99	1.000E+99	0.
6	0.	0.	0.	1.000E+99	1.000E+99	1.000E+99
9	0.	0.	0.	0.	2.000E+01	0.

PROB (CONTD)
1203 TWO STORY BENT HOLDING STIFFNESS FROM 1202 - WIND LOAD

TABLE 5 - MEMBER STIFFNESS DATA

HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS THE FOLLOWING

NONE

PROB (CONTD)
1203 TWO STORY BENT HOLDING STIFFNESS FROM 1202 - WIND LOAD

TABLE 6 - MEMBER LOAD DATA

LOAD TYPE	UNIFORM GX	UNIFORM GY	NO CARDS	AXIS OPT
1	-0.	-2.580E-02	-0	1
2	-0.	-4.333E-03	-0	1
3	-0.	-4.333E-03	-0	1
4	-0.	-4.333E-03	-0	1

PROB (CONTD)
1203 TWO STORY BENT HOLDING STIFFNESS FROM 1202 - WIND LOAD

TABLE 7 - COMPILATION TABLE

NO DATA

PROB (CONTD)
1203 TWO STORY BENT HOLDING STIFFNESS FROM 1202 - WIND LOAD

TABLE 8 - JOINT DISPLACEMENTS AND REACTIONS

JOINT	DISPLACEMENTS			REACTIONS		
	DISP (X)	DISP (Y)	ROTATION (Z)	REACT (X)	REACT (Y)	REACT (Z)
1	3.794E-09	1.603E-99	-2.567E-03	-3.794E+00	-1.603E+00	0.
2	4.344E-01	4.405E-04	-9.644E-04	0.	0.	0.
3	6.684E-01	3.623E-04	-4.053E-04	0.	0.	0.
4	4.296E-01	-7.009E-03	-4.069E-04	0.	0.	0.
5	6.688E-01	-8.211E-03	1.479E-04	0.	0.	0.
6	5.510E-09	-3.261E-99	-8.283E-97	-5.510E+00	3.261E+00	8.283E+02
7	4.274E-01	-1.086E-03	-1.092E-03	0.	0.	0.
8	6.683E-01	-1.493E-03	-1.525E-04	0.	0.	0.
9	-2.683E-02	-6.091E-03	-2.862E-04	0.	1.218E-01	0.

PROB (CONTO)
1203 TWO STORY BENT HOLDING STIFFNESS FROM 1202 - WIND LOAD

TABLE 8 - MEMBER RESULTS

MEMBER NUMBER 1 STIFF TYPE 1 LOAD TYPE 1
LENGTH = 2.278E+02 ALPHA = 0. BETA = 1.000E+00
GOES FROM JOINT 1 TO JOINT 2
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 1 AT JOINT 2

AXIAL FORCE = 1.603E+00 AXIAL FORCE = 1.603E+00
SHEAR = 3.794E+00 SHEAR = -1.990E+00
MOMENT = 5.070E-02 MOMENT = 2.157E+02

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 5.916E-04

MEMBER NUMBER 2 STIFF TYPE 1 LOAD TYPE 1
LENGTH = 2.278E+02 ALPHA = 0. BETA = 1.000E+00
GOES FROM JOINT 2 TO JOINT 3
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER

AT JOINT 2 SHEAR MOMENT

AXIAL FORCE = -2.400E+02 1.479E-04 -1.301E-01 0.263E-01 -1.368E-02
SHEAR = 6.667E-01 1.350E-04 -1.301E-01 5.221E-01 -1.197E-02
MOMENT = 1.201E-04 -1.301E-01 7.179E-01 -9.720E+01
1.440E+02 6.667E-01 -4.509E-02 1.012E-04 -1.301E-01 6.136E-01 -8.118E+01
1.680E+02 6.667E-01 -4.287E-02 8.342E-05 -1.301E-01 5.094E-01 -6.768E+01
1.920E+02 6.667E-01 -3.956E-02 6.859E-05 -1.301E-01 4.051E-01 -5.668E+01
2.160E+02 6.667E-01 -3.734E-02 5.610E-05 -1.301E-01 3.009E-01 -4.818E+01
2.400E+02 6.667E-01 -3.666E-02 3.845E-05 -1.301E-01 2.242E-02 -3.872E+01
2.640E+02 6.667E-01 -3.590E-02 2.403E-05 -1.301E-01 -1.161E-01 -3.929E+01
2.880E+02 6.667E-01 -3.541E-02 1.734E-05 -1.301E-01 -2.203E-01 -4.334E+01
3.120E+02 6.667E-01 -3.500E-02 9.785E-06 -1.301E-01 -3.245E-01 -4.909E+01
3.360E+02 6.667E-01 -3.498E-02 2.191E-06 -1.301E-01 -4.288E-01 -5.895E+01
3.600E+02 6.667E-01 -3.521E-02 -1.763E-05 -1.301E-01 -5.330E-01 -7.052E+01
3.840E+02 6.666E-01 -3.585E-02 -3.613E-05 -1.301E-01 -6.373E-01 -8.460E+01
4.080E+02 6.666E-01 -3.698E-02 -5.830E-05 -1.301E-01 -7.415E-01 -1.012E+02
4.320E+02 6.666E-01 -3.869E-02 -8.473E-05 -1.301E-01 -8.457E-01 -1.203E+02
4.560E+02 6.666E-01 -4.107E-02 -1.126E-04 -1.301E-01 -9.500E-01 -1.419E+02
4.800E+02 6.666E-01 -4.404E-02 -1.341E-04 -1.301E-01 -1.054E+00 -1.660E+02
4.800E+02 6.666E-01 -4.749E-02 -1.525E-04 -1.301E-01 -1.158E+00 -1.926E+02

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 1.037E-04

PROB (CONTO)
1203 TWO STORY BENT HOLDING STIFFNESS FROM 1202 - WIND LOAD

TABLE 9 - MEMBER RESULTS (CONTO)

MEMBER NUMBER 10 STIFF TYPE 7 LOAD TYPE 0
LENGTH = 3.900E+02 ALPHA = 0. BETA = 1.000E+00
GOES FROM JOINT 9 TO JOINT 4
OUTPUT DISTANCES ARE FROM JOINT 9 ALONG THE MEMBER AXIS
ALL OUTPUT FORCES AND DISPLACEMENTS ARE WITH RESPECT TO THE MEMBER AXES

DISTANCE	DISPLACEMENTS			FORCES		
	AXIAL	LATHEAL	ROTATIONAL	AXIAL	SHEAR	MOMENT
0.	-6.091E-03	2.863E-02	-2.862E-04	-1.218E-01	0.228E-11	-5.664E-10
1.900E+01	-6.097E-03	2.294E-02	-2.900E-04	-3.632E-01	-1.491E+00	-1.573E+01
3.900E+01	-6.109E-03	1.702E-02	-3.123E-04	-6.040E-01	-2.590E+00	-5.709E+01
5.900E+01	-6.126E-03	1.035E-02	-3.670E-04	-8.471E-01	-3.384E+00	-1.164E+02
7.920E+01	-6.149E-03	2.202E-03	-4.631E-04	-1.090E+00	-3.618E+00	-1.859E+02
9.900E+01	-6.178E-03	-8.296E-03	-6.044E-04	-1.334E+00	-3.491E+00	-2.575E+02
1.188E+02	-6.213E-03	-2.203E-02	-7.894E-04	-1.580E+00	-2.861E+00	-3.217E+02
1.386E+02	-6.254E-03	-3.980E-02	-1.010E-03	-1.826E+00	-1.658E+00	-3.680E+02
1.584E+02	-6.300E-03	-6.217E-02	-1.251E-03	-2.075E+00	1.802E-01	-3.842E+02
1.782E+02	-6.353E-03	-8.933E-02	-1.489E-03	-2.325E+00	2.687E+00	-3.575E+02
1.980E+02	-6.411E-03	-1.209E-01	-1.694E-03	-2.594E+00	4.771E+00	-2.773E+02
2.178E+02	-6.471E-03	-1.560E-01	-1.840E-03	-2.501E+00	4.841E+00	-1.815E+02
2.376E+02	-6.531E-03	-1.934E-01	-1.925E-03	-2.501E+00	4.841E+00	-6.560E+01
2.574E+02	-6.590E-03	-2.318E-01	-1.949E-03	-2.501E+00	4.841E+00	1.026E+01
2.772E+02	-6.650E-03	-2.701E-01	-1.912E-03	-2.501E+00	4.841E+00	1.061E+02
2.970E+02	-6.710E-03	-3.071E-01	-1.814E-03	-2.501E+00	4.841E+00	2.020E+02
3.168E+02	-6.770E-03	-3.415E-01	-1.654E-03	-2.501E+00	4.841E+00	2.978E+02
3.366E+02	-6.829E-03	-3.722E-01	-1.434E-03	-2.501E+00	4.841E+00	3.937E+02
3.564E+02	-6.889E-03	-3.979E-01	-1.153E-03	-2.501E+00	4.841E+00	4.805E+02
3.762E+02	-6.949E-03	-4.174E-01	-8.103E-04	-2.501E+00	4.841E+00	5.854E+02
3.960E+02	-7.009E-03	-4.296E-01	-4.609E-04	-2.501E+00	4.841E+00	6.813E+02

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 1.712E-04

PROB (CONTD)
1203 TWO STORY RENT HOLDING STIFFNESS FROM 1982 - WIND LOAD

TABLE 10 - JOINT EQUILIBRIUM ERRORS

JOINT	ERR (X) FORCE	ERR (Y) FORCE	ERR (Z) MOMENT
1	1.480E-04	-2.352E-12	5.070E-02
2	2.257E-04	2.949E-04	1.137E-02
3	-3.738E-04	1.152E-10	-1.101E-01
4	2.079E-05	3.550E-05	1.447E-01
5	-2.081E-05	1.759E-09	-3.263E-08
6	7.676E-04	4.846E-12	-8.874E-02
7	-6.353E-04	-3.304E-04	-4.375E-02
8	-1.323E-04	1.248E-09	-3.895E-02
9	8.226E-11	2.588E-12	-5.664E-10

PROB (CONTO)
1204 TWO STORY BEAT HOLDING STIFFNESS FROM 1202 - DEAD LOAD

TABLE 3 - MEMBER LOCATION DATA

NUMBER OF MEMBER STIFFNESS TYPES = 7
NUMBER OF MEMBER LOAD TYPES = 7

INPUT OF MEMBER LOCATIONS

FROM JOINT	STIFF TYPE	LOAD TYPE	TO JOINT	TO	TO	TO	TO	TO	TO	TO
1	1	1	2	3						
9	7	7	4							
4	2	2	5							
6	3	3	7							
7	4	4	8							
2	5	5	4	7						
3	6	6	5	8						

COMPUTED MEMBER NUMBERS, LENGTHS, AND OFFSETS

MEMBER NUMB	FROM JOINT	TO JOINT	STIFF TYPE	LOAD TYPE	LENGTH	X-OFFSET	Y-OFFSET
1	1	2	1	1	2.278E+02	0.	2.278E+02
2	2	3	1	1	2.278E+02	0.	2.278E+02
3	4	5	2	2	2.609E+02	0.	2.609E+02
4	6	7	3	3	2.740E+02	0.	2.740E+02
5	7	8	4	4	2.940E+02	0.	2.940E+02
6	2	4	5	5	4.800E+02	4.800E+02	0.
7	4	7	5	5	4.800E+02	4.800E+02	0.
8	3	5	6	6	4.811E+02	4.800E+02	3.312E+01
9	5	8	6	6	4.411E+02	4.800E+02	3.312E+01
10	9	4	7	7	3.966E+02	0.	3.966E+02

*** COMPUTED MEMBER NUMBERS AGREE WITH LAST PROBLEM ***

PROB (CONTO)
1204 TWO STORY BEAT HOLDING STIFFNESS FROM 1202 - DEAD LOAD

TABLE 4 - JOINT DATA

HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS THE FOLLOWING

NONE

ACCUMULATED JOINT DATA

JOINT	FORCE (X)	FORCE (Y)	MOMENT (Z)	SPRING (X)	SPRING (Y)	SPRING (Z)
1	0.	0.	0.	1.000E+99	1.000E+99	0.
6	0.	0.	0.	1.000E+99	1.000E+99	1.000E+99
9	0.	0.	0.	0.	2.000E+01	0.

PROB (CONTO)
1204 TWO STORY BEAT HOLDING STIFFNESS FROM 1202 - DEAD LOAD

TABLE 5 - MEMBER STIFFNESS DATA

HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS THE FOLLOWING

NONE

02 JULY 70 - COM - MM
 EXAMPLE PROBLEMS FOR REPORT

PROB
 1204 TWO STORY BENT HOLDING STIFFNESS FROM 1202 - DEAD LOAD

TABLE 1 - PROGRAM CONTROL DATA
 PROBLEM TYPE 3

INPUT TABLES		
TABLE NUMBER	HOLD DATA FROM LAST PROBLEM (1 = YES, 0 = NO)	NUMBER OF CARDS ADDED FOR THIS PROBLEM
2	1	0
3	0	0
4	1	0
5	1	0
6	0	17
7	-0	-0

OUTPUT TABLES	
TABLE NUMBER	SUPPRESS OUTPUT (1 = YES, 0 = NO)
8	-0
9	-0
10	-0

PROB (CONTU)
 1204 TWO STORY BENT HOLDING STIFFNESS FROM 1202 - DEAD LOAD

TABLE 2 - FRAME GEOMETRY DATA

HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS THE FOLLOWING

NONE

COMPUTED JOINT COORDINATES

JOINT	X	Y
1	0.	0.
2	0.	2.270E+02
3	0.	4.555E+02
4	4.800E+02	2.270E+02
5	4.800E+02	4.806E+02
6	9.600E+02	-4.824E+01
7	9.600E+02	2.270E+02
8	4.800E+02	5.210E+02
9	4.800E+02	-1.062E+02

PROB (CONTD)
1204 TWO STORY BENT HOLDING STIFFNESS FROM 1902 - DEAD LOAD

TABLE 6 - MEMBER LOAD DATA

LOAD TYPE	UNIFORM QX	UNIFORM QY	NO CARDS	AXIS OPT
1 -0.		-5.000E-02	-0	2
2 -0.		-5.000E-02	-0	2
3 -0.		-5.000E-02	-0	2
4 -0.		-5.000E-02	-0	2
5 -0.		-0.	5	2

LOAD TYPE FROM	UNIFORM QX	UNIFORM QY	NO CARDS	AXIS OPT
0.	4.800E+02	-0.	-2.000E-01	-0.
9.600E+01	9.600E+01	-0.	-4.000E+00	-0.
1.920E+02	1.920E+02	-0.	-4.000E+00	-0.
2.880E+02	2.880E+02	-0.	-4.000E+00	-0.
3.840E+02	3.840E+02	-0.	-4.000E+00	-0.

LOAD TYPE	UNIFORM QX	UNIFORM QY	NO CARDS	AXIS OPT
6 -0.		-0.	5	2

LOAD TYPE FROM	UNIFORM QX	UNIFORM QY	NO CARDS	AXIS OPT
0.	4.800E+02	-0.	-2.000E-01	-0.
9.600E+01	9.600E+01	-0.	-4.000E+00	-0.
1.920E+02	1.920E+02	-0.	-4.000E+00	-0.
2.880E+02	2.880E+02	-0.	-4.000E+00	-0.
3.840E+02	3.840E+02	-0.	-4.000E+00	-0.

LOAD TYPE	UNIFORM QX	UNIFORM QY	NO CARDS	AXIS OPT
7 -0.		-5.000E-02	-0	2

PROB (CONTD)
1204 TWO STORY BENT HOLDING STIFFNESS FROM 1902 - DEAD LOAD

TABLE 7 - COMPIIATION TABLE

NO DATA

PROB (CONTD)
1204 TWO STORY BENT HOLDING STIFFNESS FROM 1902 - DEAD LOAD

TABLE 8 - JOINT DISPLACEMENTS AND REACTIONS

JOINT	DISPLACEMENTS			REACTIONS		
	DISP(X)	DISP(Y)	ROTATION(Z)	REACT(X)	REACT(Y)	REACT(Z)
1	-7.093E-09	-1.301E-07	2.101E-03	7.093E+00	1.301E+02	0.
2	-2.913E-02	-3.419E-02	-3.818E-03	0.	0.	0.
3	2.757E-01	-5.025E-02	-4.634E-03	0.	0.	0.
4	-1.989E-02	-7.438E-01	-1.203E-04	0.	0.	0.
5	3.203E-01	-7.999E-01	1.519E-04	0.	0.	0.
6	7.590E-09	-1.350E-07	-6.886E-07	-7.590E+00	1.350E+02	6.886E+02
7	-1.183E-02	-4.265E-02	3.186E-03	0.	0.	0.
8	2.625E-01	-6.123E-02	3.997E-03	0.	0.	0.
9	2.274E-03	-6.506E-01	2.688E-05	0.	1.301E+01	0.

PROB (CONTD)
1204 TWO STORY BENT HOLDING STIFFNESS FROM 1202 - DEAD LOAD

TABLE 9 - MEMBER RESULTS

MEMBER NUMBER 1 STIFF TYPE 1 LOAD TYPE 1
LENGTH = 2.278E+02 ALPHA = 0. BETA = 1.000E+00
GOES FROM JOINT 1 TO JOINT 2
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 1	AT JOINT 2
AXIAL FORCE = -1.301E+02	AXIAL FORCE = -1.107E+02
SHEAR = -7.094E+00	SHEAR = -7.094E+00
MOMENT = 1.262E-01	MOMENT = -1.616E+03

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 3.356E-08

MEMBER NUMBER 2 STIFF TYPE 1 LOAD TYPE 1
LENGTH = 2.278E+02 ALPHA = 0. BETA = 1.000E+00
GOES FROM JOINT 2 TO JOINT 3
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 2	AT JOINT 3
AXIAL FORCE = -6.410E+01	AXIAL FORCE = 4.504E+01
SHEAR = -2.076E+01	SHEAR = 4.024E+01
MOMENT = 2.221E+01	MOMENT = -2.156E+03

MEMBER NUMBER 1	MEMBER NUMBER 2
2.221E+01	4.504E+01
-9.23E-04	4.844E+01
-7.020E-04	3.061E+01
-2.344E+01	2.922E+01
-9.363E-01	3.185E+01
-9.360E-01	1.562E+03
-9.242E-01	2.225E+01
-9.811E-01	2.631E+03
-8.461E-01	1.538E+01
-8.192E-01	2.831E+03
-7.589E-01	8.662E+00
-6.839E-01	2.897E+03
-5.949E-01	3.047E+03
-4.982E-01	3.082E+03
-3.911E-01	3.082E+03
-2.844E-01	2.999E+03
-1.803E-01	2.708E+03
-8.113E-02	2.300E+03
	1.778E+03
	1.136E+03
	2.861E+02
	-6.790E+02
	-1.760E+03
	-2.956E+03

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 1.201E-07

PROB (CONTD)
1204 TWO STORY BENT HOLDING STIFFNESS FROM 1202 - DEAD LOAD

TABLE 9 - MEMBER RESULTS (CONTD)

MEMBER NUMBER 10 STIFF TYPE 7 LOAD TYPE 7
LENGTH = 3.960E+02 ALPHA = 0. BETA = 1.000E+00
GOES FROM JOINT 9 TO JOINT 4
OUTPUT DISTANCES ARE FROM JOINT 9 ALONG THE MEMBER AXIS
ALL OUTPUT FORCES AND DISPLACEMENTS ARE WITH RESPECT TO THE MEMBER AXES

	DISPLACEMENTS				FORCES	
DISTANCE	AXIAL	LATERAL	ROTATIONAL	AXIAL	SHEAR	MOMENT
0.	-6.506E-01	-2.274E-03	2.688E-05	-1.301E+01	-1.007E+11	1.191E-10
1.980E+01	-6.513E-01	-1.740E-03	2.719E-05	-3.740E+01	1.161E-01	1.238E+00
3.960E+01	-6.525E-01	-1.187E-03	2.892E-05	-6.262E+01	1.966E-01	4.419E+00
5.940E+01	-6.542E-01	-6.776E-04	3.312E-05	-8.750E+01	2.428E-01	8.854E+00
7.920E+01	-6.566E-01	1.447E-04	4.034E-05	-1.125E+02	2.540E-01	1.386E+01
9.900E+01	-6.596E-01	1.042E-03	5.074E-05	-1.375E+02	2.279E-01	1.873E+01
1.188E+02	-6.632E-01	2.174E-03	6.399E-05	-1.637E+02	1.606E-01	2.268E+01
1.386E+02	-6.674E-01	3.589E-03	7.922E-05	-1.881E+02	4.810E-02	2.486E+01
1.584E+02	-6.722E-01	5.315E-03	9.502E-05	-2.138E+02	1.127E-01	2.435E+01
1.782E+02	-6.776E-01	7.342E-03	1.044E-04	-2.394E+02	-3.225E-01	2.016E+01
1.980E+02	-6.836E-01	9.618E-03	1.196E-04	-2.548E+02	-4.918E-01	1.157E+01
2.178E+02	-6.897E-01	1.204E-02	1.249E-04	-2.541E+02	-4.974E-01	1.724E+00
2.376E+02	-6.958E-01	1.448E-02	1.218E-04	-2.541E+02	-4.974E-01	-8.124E+00
2.574E+02	-7.019E-01	1.682E-02	1.135E-04	-2.442E+02	-4.974E-01	-1.797E+01
2.772E+02	-7.080E-01	1.893E-02	9.894E-05	-2.532E+02	-4.974E-01	-2.782E+01
2.970E+02	-7.140E-01	2.070E-02	7.808E-05	-2.552E+02	-4.974E-01	-3.767E+01
3.168E+02	-7.200E-01	2.198E-02	5.044E-05	-2.412E+02	-4.974E-01	-4.752E+01
3.366E+02	-7.260E-01	2.267E-02	1.753E-05	-2.582E+02	-4.974E-01	-5.738E+01
3.564E+02	-7.320E-01	2.293E-02	-2.215E-05	-2.492E+02	-4.974E-01	-6.721E+01
3.762E+02	-7.379E-01	2.175E-02	-6.810E-05	-2.442E+02	-4.974E-01	-7.706E+01
3.960E+02	-7.438E-01	1.989E-02	-1.243E-04	-2.472E+02	-4.974E-01	-8.691E+01

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 1.205E-09

PROB (CONTD)

1204 TWO STORY BENT HOLDING STIFFNESS FROM 1902 - DEAD LOAD

TABLE 10 - JOINT EQUILIBRIUM ERRORS

JOINT	ERR(X) FORCE	ERR(Y) FORCE	ERR(Z) MOMENT
1	-1.108E-03	2.001E-10	1.262E-01
2	-2.136E-03	2.129E-04	5.468E-01
3	3.244E-03	3.720E-09	3.694E-01
4	4.827E-05	-2.441E-04	3.099E-02
5	-4.827E-05	2.593E-08	-2.195E-07
6	1.186E-03	2.046E-10	-1.637E-01
7	1.865E-03	3.129E-05	-6.197E-01
8	-3.051E-03	6.947E-09	-4.485E-01
9	-1.067E-11	7.595E-10	1.191E-10

02 JULY 78 - CON - MM
EXAMPLE PROBLEMS FOR REPORT

PROB
1205 TWO STORY BENT HOLDING STIFFNESS FROM 1202 = DEAD+LIVE+WIND LOADS

TABLE 1 - PROGRAM CONTROL DATA
PROBLEM TYPE 4

INPUT TABLES

TABLE NUMBER	HOLD DATA FROM LAST PROBLEM (1 = YES, 0 = NO)	NUMBER OF CARDS ADDED FOR THIS PROBLEM
2	-0	-0
3	-0	-0
4	-0	-0
5	-0	-0
6	-0	-0
7	-0	3

OUTPUT TABLES

TABLE NUMBER	SUPPRESS OUTPUT (1 = YES, 0 = NO)
8	-0
9	-0
10	-0

PROB (CONTD)
1205 TWO STORY BENT HOLDING STIFFNESS FROM 1202 = DEAD+LIVE+WIND LOADS

TABLE 7 - COMPILATION TABLE

TABLES (2 - 6) OMITTED

INPUT OF PROBLEM NUMBERS AND MULTIPLIERS

NPROB	MULTIPLIER
1202	1.250E+00
1204	1.250E+00
1203	1.250E+00

PROBLEM NUMBERS AND MULTIPLIERS USED FOR THIS PROBLEM
PROBLEM NUMBERS IN ORDER PROBLEMS WERE INPUT

NPROB	MULTIPLIER
1202	1.250E+00
1203	1.250E+00
1204	1.250E+00

PROB (CONTD)
1205 TWO STORY BENT HOLDING STIFFNESS FROM 1202 = DEAD+LIVE+WIND LOADS

TABLE 8 - JOINT DISPLACEMENTS AND REACTIONS

JOINT	DISPLACEMENTS			REACTIONS		
	DISP(X)	DISP(Y)	ROTATION(Z)	REACT(X)	REACT(Y)	REACT(Z)
1	-7.506E-99	-2.206E-97	3.625E-04	7.506E+00	2.206E+02	0.
2	5.458E-01	-8.086E-02	-7.882E-03	0.	0.	0.
3	1.406E+00	-9.290E-02	-1.060E-02	0.	0.	0.
4	5.192E-01	-1.399E+00	-7.566E-04	0.	0.	0.
5	1.990E+00	-1.521E+00	4.753E-04	0.	0.	0.
6	1.986E-98	-2.404E-97	-2.228E-96	-1.986E+00	2.404E+02	2.228E+03
7	5.341E-01	-7.719E-02	3.935E-03	0.	0.	0.
8	1.379E+00	-1.183E-01	8.703E-03	0.	0.	0.
9	-3.280E-02	-1.221E+00	-3.199E-04	0.	2.442E+01	0.

PROB (CONTD)
1205 TWO STORY BEAT HOLDING STIFFNESS FROM 1202 - DEAD,LIVE,WIND LOADS

TABLE 9 - MEMBER RESULTS

MEMBER NUMBER 1 STIFF TYPE 1
LENGTH = 2.278E+02 ALPHA = 0. BETA = 1.000E+00
GOES FROM JOINT 1 TO JOINT 2
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 1 AT JOINT 2

AXIAL FORCE = -2.206E+02 AXIAL FORCE = -2.144E+02
SHEAR = -7.500E+00 SHEAR = -1.463E+01
MOMENT = 2.813E+01 MOMENT = -2.320E+03

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 1.308E-07

MEMBER NUMBER 2 STIFF TYPE 1
LENGTH = 2.278E+02 ALPHA = 0. BETA = 1.000E+00
GOES FROM JOINT 2 TO JOINT 3
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 2

SHEAR MOMENT

AXIAL FORCE = -1.237E+02
SHEAR = -1.599E+00
MOMENT = -1.747E+00
1.378E+00 -1.699E+00 -1.619E+03 -4.714E+01 1.269E+02 -8.877E+03
1.680E+02 1.376E+00 -1.945E+00 -1.369E+03 -4.398E+01 6.021E+01 4.627E+03
1.920E+02 1.375E+00 -1.984E+00 -2.050E+04 -4.155E+01 9.902E+01 5.999E+03
2.160E+02 1.375E+00 -1.957E+00 8.129E+04 -4.011E+01 1.802E+01 6.506E+03
2.400E+02 1.374E+00 -1.924E+00 1.901E+03 -3.989E+01 1.189E+01 6.866E+03
2.640E+02 1.373E+00 -1.865E+00 3.036E+03 -3.970E+01 5.755E+00 7.078E+03
2.880E+02 1.373E+00 -1.778E+00 4.193E+03 -3.782E+01 -1.548E+01 7.140E+03
3.120E+02 1.372E+00 -1.640E+00 5.473E+03 -3.639E+01 -9.642E+01 6.337E+03
3.360E+02 1.371E+00 -1.586E+00 7.077E+03 -3.497E+01 -4.256E+01 5.387E+03
3.600E+02 1.371E+00 -1.321E+00 8.235E+03 -3.556E+01 -4.869E+01 4.289E+03
3.840E+02 1.370E+00 -1.112E+00 9.114E+03 -3.410E+01 -4.998E+01 3.039E+03
4.080E+02 1.369E+00 -8.862E+01 8.590E+03 -3.247E+01 -9.088E+01 9.268E+02
4.320E+02 1.369E+00 -6.550E+01 9.541E+03 -3.225E+01 -9.761E+01 -1.334E+03
4.560E+02 1.368E+00 -4.287E+01 9.213E+03 -3.184E+01 -1.031E+02 -3.741E+03
4.800E+02 1.368E+00 -2.129E+01 8.703E+03 -3.144E+01 -1.090E+02 -6.296E+03

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 3.173E-07

PROB (CONTD)
1205 TWO STORY BEAT HOLDING STIFFNESS FROM 1202 - DEAD,LIVE,WIND LOADS

TABLE 9 - MEMBER RESULTS (CONTD)

MEMBER NUMBER 10 STIFF TYPE 7
LENGTH = 3.960E+02 ALPHA = 0. BETA = 1.000E+00
GOES FROM JOINT 9 TO JOINT 4
OUTPUT DISTANCES ARE FROM JOINT 9 ALONG THE MEMBER AXIS
ALL OUTPUT FORCES AND DISPLACEMENTS ARE WITH RESPECT TO THE MEMBER AXES

DISPLACEMENTS

FORCES

DISTANCE	AXIAL	LATERAL	ROTATIONAL	AXIAL	SHEAR	MOMENT
0.	-1.221E+00	3.280E-02	-3.199E-04	-2.442E+01	4.957E+11	-5.610E-10
1.980E+01	-1.222E+00	2.644E-02	-3.243E-04	-7.157E+01	-1.713E+09	-1.805E+01
3.960E+01	-1.225E+00	1.981E-02	-3.580E-04	-1.188E+02	-2.944E+08	-6.562E+01
5.940E+01	-1.228E+00	1.232E-02	-4.129E-04	-1.641E+02	-3.822E+07	-1.341E+02
7.920E+01	-1.233E+00	3.131E-03	-5.237E-04	-2.136E+02	-4.260E+06	-2.148E+02
9.900E+01	-1.238E+00	4.775E-03	-6.672E-04	-2.613E+02	-4.094E+05	-2.983E+02
1.188E+02	-1.245E+00	2.443E-02	-9.019E-04	-3.092E+02	-3.404E+04	-3.740E+02
1.386E+02	-1.253E+00	4.478E-02	-1.159E-03	-3.574E+02	-2.058E+03	-4.298E+02
1.584E+02	-1.262E+00	7.081E-02	-1.442E-03	-4.060E+02	1.919E+02	-4.519E+02
1.782E+02	-1.272E+00	1.019E-01	-1.723E-03	-4.549E+02	2.871E+00	-4.232E+02
1.980E+02	-1.284E+00	1.385E-01	-1.969E-03	-4.879E+02	5.251E+00	-3.376E+02
2.178E+02	-1.296E+00	1.794E-01	-2.150E-03	-4.877E+02	5.332E+00	-2.320E+02
2.376E+02	-1.307E+00	2.232E-01	-2.264E-03	-4.864E+02	5.332E+00	-1.265E+02
2.574E+02	-1.319E+00	2.680E-01	-2.311E-03	-4.852E+02	5.332E+00	-2.889E+01
2.772E+02	-1.330E+00	3.142E-01	-2.291E-03	-4.840E+02	5.332E+00	6.447E+01
2.970E+02	-1.342E+00	3.588E-01	-2.203E-03	-4.827E+02	5.332E+00	1.902E+02
3.168E+02	-1.353E+00	4.018E-01	-2.048E-03	-4.815E+02	5.332E+00	2.958E+02
3.366E+02	-1.365E+00	4.395E-01	-1.826E-03	-4.802E+02	5.332E+00	4.014E+02
3.564E+02	-1.376E+00	4.729E-01	-1.537E-03	-4.780E+02	5.332E+00	5.069E+02
3.762E+02	-1.388E+00	4.999E-01	-1.180E-03	-4.778E+02	5.332E+00	6.125E+02
3.960E+02	-1.399E+00	5.192E-01	-7.566E-04	-4.765E+02	5.332E+00	7.181E+02

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 2.338E-08

PROB (CONTD)
1205 TWO STORY BENT HOLDING STIFFNESS FROM 1962 - DEAD,LIVE,WIND LOADS

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TABLE 10 - JOINT EQUILIBRIUM ERRORS

JOINT	ERR (X) FORCE	ERR (Y) FORCE	ERR (Z) MOMENT
1	-1.729E-03	3.465E-10	2.013E-01
2	-4.217E-03	6.639E-04	1.033E+00
3	5.946E-03	7.195E-09	5.927E-01
4	1.208E-04	-1.508E-04	2.490E-01
5	-1.209E-04	5.313E-08	-4.640E-07
6	2.986E-03	3.620E-10	-3.906E-01
7	3.213E-03	-5.051E-04	-1.199E+00
8	-6.199E-03	1.259E-08	-9.357E-01
9	8.957E-11	1.294E-09	-5.610E-10

02 JULY 79 - CON - MM
EXAMPLE PROBLEMS FOR REPORT

PROB
1206 TWO STORY BENT HOLDING STIFFNESS FROM 1202 - DEAD+LIVE LOADS

TABLE 1 - PROGRAM CONTROL DATA
PROBLEM TYPE 4

INPUT TABLES		
TABLE NUMBER	HOLD DATA FROM LAST PROBLEM (1 = YES, 0 = NO)	NUMBER OF CARDS ADDED FOR THIS PROBLEM
2	-0	-0
3	-0	-0
4	-0	-0
5	-0	-0
6	-0	-0
7	-0	2

OUTPUT TABLES	
TABLE NUMBER	SUPPRESS OUTPUT (1 = YES, 0 = NO)
8	-0
9	-0
10	-0

PROB (CONTD)
1206 TWO STORY BENT HOLDING STIFFNESS FROM 1202 - DEAD+LIVE LOADS

TABLE 7 - COMPILATION TABLE

TABLES (2 - 6) OMITTED

INPUT OF PROBLEM NUMBERS AND MULTIPLIERS
NPROB MULTIPLIER

1202 1.800E+00
1204 1.500E+00

PROBLEM NUMBERS AND MULTIPLIERS USED FOR THIS PROBLEM
PROBLEM NUMBERS IN ORDER PROBLEMS WERE INPUT

NPROB MULTIPLIER

1202 1.800E+00
1203 0.
1204 1.500E+00

PROB (CONTD)
1206 TWO STORY BENT HOLDING STIFFNESS FROM 1202 - DEAD+LIVE LOADS

TABLE 8 - JOINT DISPLACEMENTS AND REACTIONS

JOINT	DISPLACEMENTS			REACTIONS		
	DISP(X)	DISP(Y)	ROTATION(Z)	REACT(X)	REACT(Y)	REACT(Z)
1	-1.551E-08	-2.931E-07	4.511E-03	1.551E+01	2.931E+02	0.
2	-4.487E-02	-7.820E-02	-8.432E-03	0.	0.	0.
3	7.382E-01	-1.194E-01	-1.315E-02	0.	0.	0.
4	-1.962E-02	-1.779E+00	-3.210E-04	0.	0.	0.
5	8.454E-01	-1.936E+00	3.727E-04	0.	0.	0.
6	1.640E-08	-2.999E-07	-1.511E-06	-1.640E+01	2.999E+02	1.511E+03
7	2.906E-03	-9.641E-02	6.677E-03	0.	0.	0.
8	7.043E-01	-1.486E-01	1.161E-02	0.	0.	0.
9	3.621E-03	-1.552E+00	4.633E-05	0.	3.105E+01	0.

PROB (CONTO) 1206 TWO STORY BENT HOLDING STIFFNESS FROM 1202 - DEAD+LIVE LOADS

TABLE 9 - MEMBER RESULTS

MEMBER NUMBER 1 STIFF TYPE 1
LENGTH = 2.278E+02 ALPHA = 0. BETA = 1.000E+00
GOES FROM JOINT 1 TO JOINT 2
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 1 AT JOINT 2
AXIAL FORCE = -2.931E+02 AXIAL FORCE = -2.760E+02
SHEAR = -1.551E+01 SHEAR = -1.551E+01
MOMENT = 2.760E+01 MOMENT = -3.533E+03

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 7.178E-08

MEMBER NUMBER 2 STIFF TYPE 1
LENGTH = 2.278E+02 ALPHA = 0. BETA = 1.000E+00
GOES FROM JOINT 2 TO JOINT 3
ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 2 AT JOINT 3
AXIAL FORCE = -1.583E+02 AXIAL FORCE = 1.720E+02
SHEAR = -2.200E+01 SHEAR = 1.648E+02
MOMENT = 2.760E+01 MOMENT = -1.155E+04
... (table of internal member results) ...

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 4.022E-07

PROB (CONTO) 1206 TWO STORY BENT HOLDING STIFFNESS FROM 1202 - DEAD+LIVE LOADS

TABLE 9 - MEMBER RESULTS (CONTO)

MEMBER NUMBER 10 STIFF TYPE 7
LENGTH = 3.960E+02 ALPHA = 0. BETA = 1.000E+00
GOES FROM JOINT 9 TO JOINT 4
OUTPUT DISTANCES ARE FROM JOINT 9 ALONG THE MEMBER AXIS
ALL OUTPUT FORCES AND DISPLACEMENTS ARE WITH RESPECT TO THE MEMBER AXES

DISPLACEMENTS FORCES
DISTANCE AXIAL LATERAL ROTATIONAL AXIAL SHEAR MOMENT
0. -1.552E+00 -3.621E-03 4.633E-05 -3.105E+01 -1.592E-11 1.759E-10
1.980E+01 -1.554E+00 -2.701E-03 4.681E-05 -9.106E+01 1.828E-01 1.961E+00
... (table of displacements and forces) ...

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 2.465E-09

PROB (CONTO) 1206 TWO STORY BENT HOLDING STIFFNESS FROM 1202 - DEAD+LIVE LOADS

TABLE 10 - JOINT EQUILIBRIUM ERRORS

JOINT ERR(X) ERR(Y) ERR(Z)
FORCE FORCE MOMENT
1 -2.424E-03 4.438E-10 2.760E-01
2 -5.838E-03 3.613E-04 1.304E-00
... (table of joint equilibrium errors) ...

92 JULY 70 - COM - MM
 EXAMPLE PROBLEMS FOR REPORT

PROB
 1207 ADD JOINT 10 AT GROUND LINE TO SUBDIVIDP PILE - DEAD LOAD

TABLE 1 - PROGRAM CONTROL DATA
 PROBLEM TYPE 1

TABLE NUMBER	INPUT TABLES	
	HOLD DATA FROM LAST PROBLEM (1 = YES, 0 = NO)	NUMBER OF CARDS ADDED FOR THIS PROBLEM
2	1	2
3	1	4
4	1	6
5	1	4
6	1	6
7	-0	-0

TABLE NUMBER	OUTPUT TABLES
	SUPPRESS OUTPUT (1 = YES, 0 = NO)
6	-0
9	-0
10	-0

PROB (CONID)
 1207 ADD JOINT 10 AT GROUND LINE TO SUBDIVIDP PILE - DEAD LOAD

TABLE 2 - FRAME GEOMETRY DATA

HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS THE FOLLOWING

NUMBER OF JOINTS IN FRAME = 10

INPUT OF JOINT OFFSETS

FROM JOINT	X-OFFSET	Y-OFFSET	TO JOINT	TO	TO	TO	TO	TO	TO
9	0.	1.920E+02	10						

COMPUTED JOINT COORDINATES

JOINT	X	Y
1	0.	0.
2	0.	2.278E+02
3	0.	4.555E+02
4	4.800E+02	2.278E+02
5	4.800E+02	4.866E+02
6	9.600E+02	-4.824E+01
7	9.600E+02	2.278E+02
8	9.600E+02	5.218E+02
9	4.800E+02	-1.682E+02
10	4.800E+02	2.376E+01

PROB (CONTD)
1207 ADD JOINT 10 AT GROUND LINE TO SUBDIVID PILE - DEAD LOAD

TABLE 3 - MEMBER LOCATION DATA

HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS THE FOLLOWING

NUMBER OF MEMBER STIFFNESS TYPES = 9
NUMBER OF MEMBER LOAD TYPES = 7

INPUT OF MEMBER LOCATIONS

FROM JOINT	STIFF TYPE	LOAD TYPE	TO JOINT	TO 1	TO 2	TO 3	TO 4	TO 5	TO 6	TO 7	TO 8	TO 9	TO 10
9	8	8	4										
9	8	8	10										
10	8	8	4										

COMPUTED MEMBER NUMBERS, LENGTHS, AND OFFSETS

MEMBER NUMB	FROM JOINT	TO JOINT	STIFF TYPE	LOAD TYPE	LENGTH	X-OFFSET	Y-OFFSET
1	1	2	1	1	2.278E+02	0.	2.278E+02
2	2	3	1	1	2.278E+02	0.	2.278E+02
3	4	5	2	2	2.989E+02	0.	2.989E+02
4	6	7	3	3	2.760E+02	0.	2.760E+02
5	7	8	4	4	2.940E+02	0.	2.940E+02
6	2	4	5	5	4.800E+02	4.800E+02	0.
7	4	7	5	5	4.800E+02	4.800E+02	0.
8	3	5	6	6	4.811E+02	4.800E+02	3.312E+01
9	5	8	6	6	4.811E+02	4.800E+02	3.312E+01
10	9	4	8	8			
11	9	10	8	8	1.920E+02	0.	1.920E+02
12	10	4	9	9	2.848E+02	0.	2.848E+02

*** COMPUTED MEMBER NUMBERS AGREE WITH LAST PROBLEM ***

PROB (CONTD)
1207 ADD JOINT 10 AT GROUND LINE TO SUBDIVID PILE - DEAD LOAD

TABLE 4 - JOINT DATA

HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS THE FOLLOWING

NONE

ACCUMULATED JOINT DATA

JOINT	FORCE (X)	FORCE (Y)	MOMENT (Z)	SPRING (X)	SPRING (Y)	SPRING (Z)
1	0.	0.	0.	1.000E+99	1.000E+99	0.
6	0.	0.	0.	1.000E+99	1.000E+99	1.000E+99
9	0.	0.	0.	0.	2.000E+81	0.

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1207 ADD JOINT 10 AT GROUND LINE TO SUBDIVID PILE - DEAD LOAD

TABLE 5 - MEMBER STIFFNESS DATA

HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS THE FOLLOWING

STIFF TYPE	MOD OF ELAST	PRISMATIC I	PRISMATIC A	NO CURVS	AXIS OPT	OUTPUT OPT	PIN FROM	PIN TO
8	2.980E+04=0.	-0.	-0.	2	1	-0	-0	-0
STIFF TYPE FROM	TO	MEMBER I	RESTRAINTS ARE IN MEMBER AXES A	MEMBER KX	MEMBER KY	MEMBER KZ	MEMBER PX	MEMBER PY
0.	-0.	1.050E+03	2.800E+01	2.800E+00	1.000E+00	-0.		
-0.	1.920E+02	1.050E+03	2.800E+01	2.800E+00	1.500E+00	-0.		
STIFF TYPE FROM	MOD OF ELAST	PRISMATIC I	PRISMATIC A	NO CURVS	AXIS OPT	OUTPUT OPT	PIN FROM	PIN TO
9	2.980E+04	1.050E+03	2.800E+01	0	1	-0	-0	-0

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1207 ADD JOINT 10 AT GROUND LINE TO SUBDIVIDE PILE - DEAD LOAD

TABLE 6 - MEMBER LOAD DATA

HOLDING DATA FROM THE PREVIOUS PROBLEM PLUS THE FOLLOWING

NONE

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1207 ADD JOINT 10 AT GROUND LINE TO SUBDIVIDE PILE - DEAD LOAD

TABLE 7 - COMPILATION TABLE

NO DATA

PROB (CONTD)

1207 ADD JOINT 10 AT GROUND LINE TO SUBDIVIDE PILE - DEAD LOAD

TABLE 8 - JOINT DISPLACEMENTS AND REACTIONS

JOINT	DISPLACEMENTS			REACTIONS		
	DISP(X)	DISP(Y)	ROTATION(Z)	REACT(X)	REACT(Y)	REACT(Z)
1	-7.024E-99	-1.296E-97	2.082E-03	7.024E+00	1.296E+02	0.
2	-2.923E-02	-3.404E-02	-3.770E-03	0.	0.	0.
3	2.705E-01	-4.999E-02	-4.526E-03	0.	0.	0.
4	-2.019E-02	-6.959E-01	-1.163E-04	0.	0.	0.
5	3.120E-01	-7.522E-01	1.497E-04	0.	0.	0.
6	7.516E-99	-1.344E-97	-6.814E-97	-7.516E+00	1.344E+02	6.814E+02
7	-1.231E-02	-4.244E-02	3.159E-03	0.	0.	0.
8	2.376E-01	-6.293E-02	3.897E-03	0.	0.	0.
9	2.278E-03	-8.048E-01	2.692E-05	0.	1.210E+01	0.
10	-8.894E-03	-6.348E-01	1.172E-04	0.	0.	0.

PROB (CONTD)

1207 ADD JOINT 10 AT GROUND LINE TO SUBDIVIDE PILE - DEAD LOAD

TABLE 9 - MEMBER RESULTS

MEMBER NUMBER 1 STIFF TYPE 1 LOAD TYPE 1
 LENGTH = 2.278E+02 ALPHA = 0. BETA = 1.000E+00
 GOES FROM JOINT 1 TO JOINT 2
 ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 1 AT JOINT 2

AXIAL FORCE = -1.296E+02 AXIAL FORCE = -1.192E+02
 SHEAR = -7.025E+00 SHEAR = -7.025E+00
 MOMENT = 1.250E+01 MOMENT = -1.600E+03

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 2.901E-08

MEMBER NUMBER 2 STIFF TYPE 1 LOAD TYPE 1
 LENGTH = 2.278E+02 ALPHA = 0.
 GOES FROM JOINT 2 TO JOINT 3
 ALL OUTPUT FORCES ARE WITH RESPECT TO THE MEMBER AXES

AT JOINT 2 AT JOINT 3

AXIAL FORCE = -2.338E+01 AXIAL FORCE = 6.018E+01
 SHEAR = -2.305E+01 SHEAR = 5.538E+01
 MOMENT = -2.272E+01 MOMENT = -3.521E+03
 1.680E+02 2.567E-01 -8.972E-01 -3.634E-04 -2.679E+01 2.259E+01 2.105E+03
 1.920E+02 2.564E-01 -8.997E-01 1.522E-04 -2.892E+01 1.372E+01 2.589E+03
 2.160E+02 2.560E-01 -8.997E-01 5.966E-04 -1.985E+01 9.003E+00 2.864E+03
 2.400E+02 2.557E-01 -8.707E-01 1.076E+03 -1.452E+01 4.203E+00 3.023E+03
 2.640E+02 2.554E-01 -8.388E-01 1.572E+03 -1.919E+01 -8.971E-01 3.066E+03
 2.880E+02 2.551E-01 -7.951E-01 2.066E+03 -1.871E+01 -7.586E+00 2.991E+03
 3.120E+02 2.547E-01 7.380E-01 2.693E+03 -1.825E+01 -1.419E+01 2.707E+03
 3.360E+02 2.544E-01 -6.658E-01 3.294E+03 -1.792E+01 -1.899E+01 2.308E+03
 3.600E+02 2.540E-01 -5.805E-01 3.786E+03 -1.749E+01 -3.379E+01 1.794E+03
 3.840E+02 2.537E-01 -4.848E-01 4.141E+03 -1.711E+01 -4.073E+01 1.160E+03
 4.080E+02 2.534E-01 -3.827E-01 4.319E+03 -1.645E+01 -3.738E+01 3.188E+02
 4.320E+02 2.531E-01 -2.787E-01 4.292E+03 -1.432E+01 -4.218E+01 -6.381E+02
 4.560E+02 2.528E-01 -1.772E-01 4.131E+03 -1.499E+01 -4.698E+01 -1.711E+03
 4.800E+02 2.526E-01 -8.051E-02 3.897E+03 -1.547E+01 -8.155E+01 -2.898E+03

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 1.333E-07

PROB (CONTD)
1207 ADD JOINT 10 AT GROUND LINE TO SUBDIVING PILE - DEAD LOAD

TABLE 6 - MEMBER RESULTS (CONTD)

MEMBER NUMBER 11 STIFF TYPE 8 LOAD TYPE 0
LENGTH = 1.920E+02 ALPHA = 0. BETA = 1.000E+00
GOES FROM JOINT 9 TO JOINT 10
OUTPUT DISTANCES ARE FROM JOINT 9 ALONG THE MEMBER AXIS
ALL OUTPUT FORCES AND DISPLACEMENTS ARE WITH RESPECT TO THE MEMBER AXES

DISTANCE	DISPLACEMENTS			FORCES		
	AXIAL	LATERAL	ROTATIONAL	AXIAL	SHEAR	MOMENT
0.	-6.048E-01	-2.278E-03	2.692E-05	-1.210E+01	-1.745E-11	1.741E-10
9.600E+00	-6.051E-01	-2.020E-03	2.695E-05	-2.371E+01	9.108E-02	3.035E-01
1.920E+01	-6.054E-01	-1.760E-03	2.717E-05	-3.533E+01	1.135E-01	1.152E+00
2.880E+01	-6.059E-01	-1.487E-03	2.772E-05	-4.696E+01	1.575E-01	2.443E+00
3.840E+01	-6.065E-01	-1.227E-03	2.873E-05	-5.860E+01	1.933E-01	4.150E+00
4.800E+01	-6.072E-01	-9.436E-04	3.032E-05	-7.025E+01	2.211E-01	6.155E+00
5.760E+01	-6.081E-01	-6.423E-04	3.256E-05	-8.192E+01	2.409E-01	8.383E+00
6.720E+01	-6.091E-01	-3.160E-04	3.552E-05	-9.360E+01	2.529E-01	1.076E+01
7.680E+01	-6.103E-01	4.214E-05	3.922E-05	-1.053E+02	2.558E-01	1.321E+01
8.640E+01	-6.116E-01	4.395E-04	4.360E-05	-1.178E+02	2.504E-01	1.565E+01
9.600E+01	-6.130E-01	8.632E-04	4.868E-05	-1.288E+02	2.360E-01	1.800E+01
1.056E+02	-6.146E-01	1.380E-03	5.477E-05	-1.496E+02	2.121E-01	2.015E+01
1.152E+02	-6.162E-01	1.937E-03	6.136E-05	-1.594E+02	1.782E-01	2.205E+01
1.248E+02	-6.181E-01	2.559E-03	6.835E-05	-1.642E+02	1.338E-01	2.356E+01
1.344E+02	-6.201E-01	3.251E-03	7.579E-05	-1.761E+02	7.855E-02	2.459E+01
1.440E+02	-6.222E-01	4.015E-03	8.347E-05	-1.841E+02	1.201E-02	2.504E+01
1.536E+02	-6.244E-01	4.853E-03	9.118E-05	-2.000E+02	-4.605E-02	2.479E+01
1.632E+02	-6.268E-01	5.765E-03	9.869E-05	-2.170E+02	-1.557E-01	2.374E+01
1.728E+02	-6.293E-01	6.747E-03	1.057E-04	-2.241E+02	-2.569E-01	2.177E+01
1.824E+02	-6.320E-01	7.793E-03	1.120E-04	-2.326E+02	-3.693E-01	1.878E+01
1.920E+02	-6.348E-01	8.894E-03	1.172E-04	-2.404E+02	-4.926E-01	1.466E+01

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 1.445E-09

PROB (CONTD)
1207 ADD JOINT 10 AT GROUND LINE TO SUBDIVING PILE - DEAD LOAD

TABLE 7 - MEMBER RESULTS (CONTD)

MEMBER NUMBER 12 STIFF TYPE 9 LOAD TYPE 0
LENGTH = 2.040E+02 ALPHA = 0. BETA = 1.000E+00
GOES FROM JOINT 10 TO JOINT 4
OUTPUT DISTANCES ARE FROM JOINT 10 ALONG THE MEMBER AXIS
ALL OUTPUT FORCES AND DISPLACEMENTS ARE WITH RESPECT TO THE MEMBER AXES

DISTANCE	DISPLACEMENTS			FORCES		
	AXIAL	LATERAL	ROTATIONAL	AXIAL	SHEAR	MOMENT
0.	-6.348E-01	0.894E-03	1.172E-04	-2.404E+02	-4.926E-01	1.467E+01
1.020E+01	-6.378E-01	1.011E-02	1.212E-04	-2.404E+02	-4.926E-01	9.641E+00
2.040E+01	-6.409E-01	1.136E-02	1.235E-04	-2.404E+02	-4.926E-01	4.616E+00
3.060E+01	-6.440E-01	1.263E-02	1.242E-04	-2.404E+02	-4.926E-01	-4.087E-01
4.080E+01	-6.470E-01	1.389E-02	1.233E-04	-2.404E+02	-4.926E-01	-5.434E+00
5.100E+01	-6.501E-01	1.513E-02	1.207E-04	-2.404E+02	-4.926E-01	-1.046E+01
6.120E+01	-6.531E-01	1.634E-02	1.164E-04	-2.404E+02	-4.926E-01	-1.548E+01
7.140E+01	-6.562E-01	1.750E-02	1.105E-04	-2.404E+02	-4.926E-01	-2.051E+01
8.160E+01	-6.592E-01	1.859E-02	1.029E-04	-2.404E+02	-4.926E-01	-2.553E+01
9.180E+01	-6.623E-01	1.960E-02	9.374E-05	-2.404E+02	-4.926E-01	-3.056E+01
1.020E+02	-6.654E-01	2.050E-02	8.289E-05	-2.404E+02	-4.926E-01	-3.558E+01
1.122E+02	-6.684E-01	2.128E-02	7.039E-05	-2.404E+02	-4.926E-01	-4.061E+01
1.224E+02	-6.715E-01	2.193E-02	5.624E-05	-2.404E+02	-4.926E-01	-4.563E+01
1.326E+02	-6.745E-01	2.242E-02	4.044E-05	-2.404E+02	-4.926E-01	-5.066E+01
1.428E+02	-6.776E-01	2.279E-02	2.299E-05	-2.404E+02	-4.926E-01	-5.568E+01
1.530E+02	-6.806E-01	2.289E-02	3.887E-06	-2.404E+02	-4.926E-01	-6.071E+01
1.632E+02	-6.837E-01	2.242E-02	-1.888E-05	-2.404E+02	-4.926E-01	-6.573E+01
1.734E+02	-6.868E-01	2.254E-02	-3.920E-05	-2.404E+02	-4.926E-01	-7.076E+01
1.836E+02	-6.898E-01	2.202E-02	-6.330E-05	-2.404E+02	-4.926E-01	-7.578E+01
1.938E+02	-6.929E-01	2.124E-02	-9.900E-05	-2.404E+02	-4.926E-01	-8.081E+01
2.040E+02	-6.959E-01	2.019E-02	-1.103E-04	-2.404E+02	-4.926E-01	-8.583E+01

THE MAXIMUM EQUILIBRIUM ERROR INTERNAL TO THE MEMBER IS 4.081E-09

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TABLE 8 - JOINT EQUILIBRIUM ERRORS

JOINT	ERR(X) FORCE	ERR(Y) FORCE	ERR(Z) MOMENT
1	-1.048E-03	1.992E-10	1.250E-01
2	-2.009E-03	2.467E-04	5.471E-01
3	3.107E-03	2.511E-09	3.629E-01
4	1.242E-04	-3.161E-04	3.808E-02
5	-4.117E-05	2.161E-08	-8.149E-08
6	1.175E-03	2.837E-10	-1.621E-01
7	1.824E-03	6.947E-05	-6.195E-01
8	-2.994E-03	3.809E-09	-4.407E-01
9	-1.745E-11	1.414E-09	1.741E-10
10	-7.697E-05	1.664E-08	7.851E-03

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